

NON-LINEAR CONTROL OF WHEELED MOBILE ROBOTS

Maria Isabel Ribeiro Pedro Lima

mir@isr.ist.utl.pt

pal@isr.ist.utl.pt

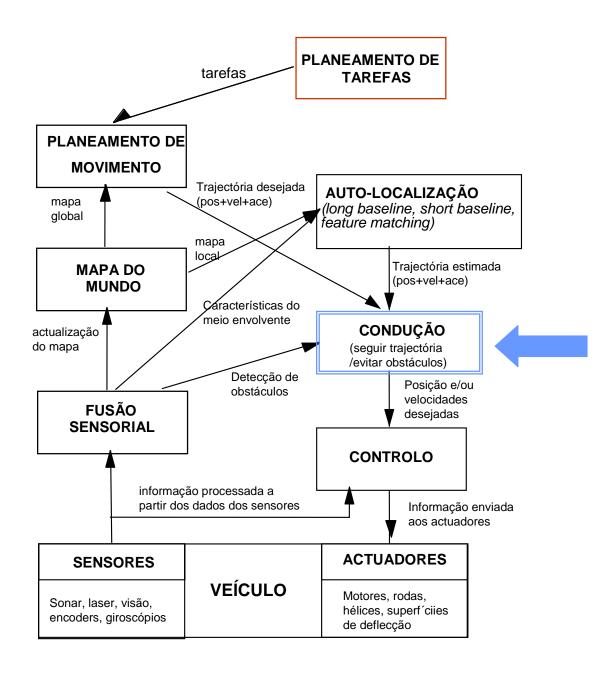
Instituto Superior Técnico (IST)
Instituto de Sistemas e Robótica (ISR)
Av.Rovisco Pais, 1
1049-001 Lisboa
PORTUGAL

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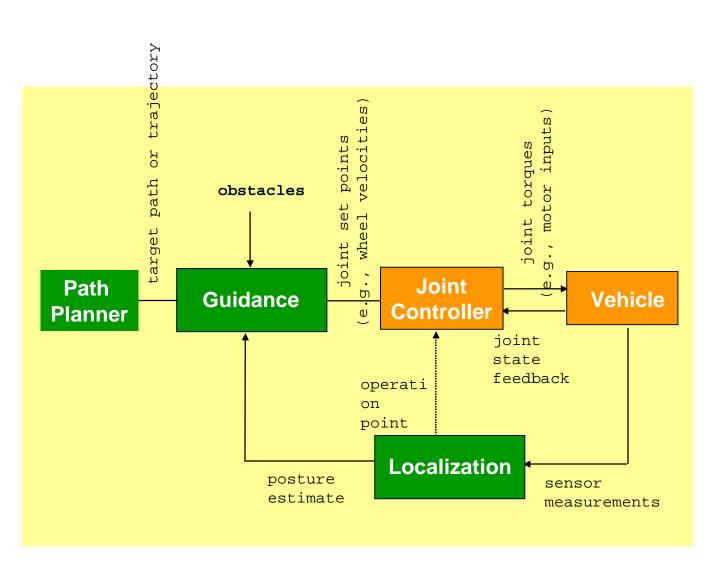


Course Outline





Guidance



GUIDANCE

 take the robot from the current posture to the desired posture, possibly following a pre-determined path or trajectory, while avoiding obstacles



Guidance Methodologies

Some Guidance methodologies

- State(posture)-feedback methods:
 - **posture stabilization** (initial and final postures given; no path or trajectory pre-determined; obstacles not considered; may lead to large unexpected paths)
 - trajectory tracking (requires pre-planned path)
 - virtual vehicle tracking (requires pre-planned trajectory)
- Potential-Field like methods
 - potential fields (holonomic vehicles)
 - generalized potential fields (non-holonomic vehicles)
 - modified potential fields (non-holonomic vehicles)
- Vector Field Histogram (VHF) like methods
 - nearness diagram navigation (holonomic vehicles)
 - freezone (non-holonomic vehicles)

NON-LINEAR CONTROL DESIGN FOR MOBILE ROBOTS



Control of Mobile Robots

- Three distintic problems:
 - Trajectory Tracking or Posture Tracking
 - Path Following
 - Point Stabilization



Vehicle of unicycle type

$$\dot{x} = v \cos \theta$$

$$\dot{y} = v \sin \theta$$

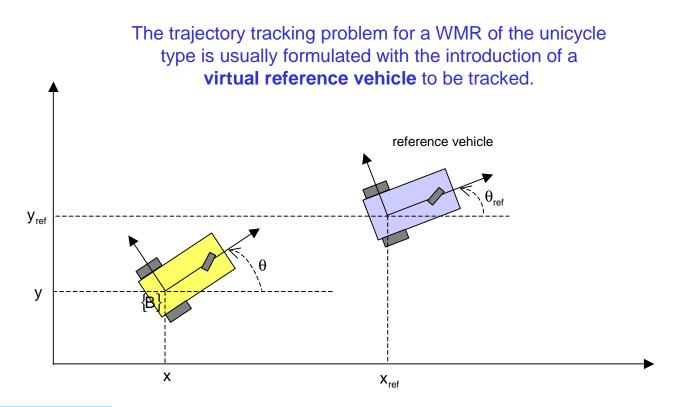
$$\dot{\Theta} = \omega$$

kinematic model



- $z = (x, y, \theta)$ position and orientation with respect to a fixed frame
 - V linear velocity
 - O angular velocity

- Is a simplified model, but
- Captures the nonholonomy property which characterizes most WMR and is the core of the difficulties involved in the control of these vehicles





Kinematic model of the Reference Vehicle

$$\begin{split} \dot{\textbf{x}}_{\text{ref}} &= \textbf{v}_{\text{ref}} \cos \theta_{\text{ref}} \\ \dot{\textbf{y}}_{\text{ref}} &= \textbf{v}_{\text{ref}} \sin \theta_{\text{ref}} \\ \dot{\theta}_{\text{ref}} &= \omega_{\text{ref}} \end{split}$$

$$z_{ref} = (x_{ref}, y_{ref}, \theta_{ref})$$

$$V_{ref}(t)$$
 $\omega_{ref}(t)$ • Bounded derivatives

- Bounded
- Do not tend to zero as t tends to infinity

Control Objective

Drive the errors $X - X_{ref}$, $y - y_{ref}$, $\theta - \theta_{ref}$ to zero

Express the errors in the {B} frame

$$\mathbf{e} = \begin{bmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \\ \mathbf{e}_3 \end{bmatrix} = \begin{bmatrix} \cos \theta & \sin \theta & 0 \\ -\sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \mathbf{x} - \mathbf{x}_{ref} \\ \mathbf{y} - \mathbf{y}_{ref} \\ \theta - \theta_{ref} \end{bmatrix}$$



Differentiating Introducing the change of inputs

$$u_1 = -V + V_{ref} \cos e_3$$

$$u_2 = \omega_{ref} - \omega$$



$$\dot{e} = \begin{bmatrix} 0 & \omega & 0 \\ -\omega & 0 & 0 \\ 0 & 0 & 1 \end{bmatrix} e + \begin{bmatrix} 0 \\ \sin e_3 \\ 0 \end{bmatrix} v_{ref} + \begin{bmatrix} 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$
non-linear dynamic system

Question?

- Is it possible to design a feedback law u=f(e) such that the error converges to zero?
- Is this law linear or non-linear?

Two different solutions:

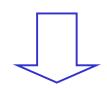
- Linear feedback control
- Nonlinear feedback control

control variables



LINEAR FEEDBACK CONTROL

$$\dot{\mathbf{e}} = \begin{bmatrix} 0 & \omega & 0 \\ -\omega & 0 & 0 \\ 0 & 0 & 1 \end{bmatrix} \mathbf{e} + \begin{bmatrix} 0 \\ \sin \mathbf{e}_3 \\ 0 \end{bmatrix} \mathbf{v}_{ref} + \begin{bmatrix} 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} \mathbf{u}_1 \\ \mathbf{u}_2 \end{bmatrix}$$



linearize about the e = 0equilibrium point

$$u = 0$$

$$\dot{e} = \begin{bmatrix} 0 & \omega_{ref}(t) & 0 \\ -\omega_{ref}(t) & 0 & 0 \\ 0 & 0 & 1 \end{bmatrix} e + \begin{bmatrix} 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

linear time-varying dynamic system

Assuming

$$\omega_{ref}(t) = \omega_{ref}$$

$$V_{ref}(t) = V_{ref}$$

linear time invariant dynamic system

Is it possible to design a linear feedback law u=f(e) such that the error converges to zero?



Is the dynamic system controllable?



$$\Gamma_{c} = \begin{bmatrix} 1 & 0 & 0 & 0 & -\omega_{ref}^{2} & v_{ref}\omega_{ref} \\ 0 & 0 & -\omega_{ref} & v_{ref} & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \end{bmatrix}$$

If $V_{ref} = \omega_{ref} = 0$ the SLIT is non-controllable



the reference robot at rest



 $u_1 = -k$ the error cannot be taken to zero in finite time

otherwise

$$\mathbf{u} = \mathbf{K}\mathbf{e}$$

$$\begin{bmatrix} \mathbf{u}_1 \\ \mathbf{u}_2 \end{bmatrix} = \begin{bmatrix} \mathbf{K}_{11} & \mathbf{K}_{12} & \mathbf{K}_{13} \\ \mathbf{K}_{21} & \mathbf{K}_{22} & \mathbf{K}_{23} \end{bmatrix} \begin{bmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \\ \mathbf{e}_3 \end{bmatrix}$$

K_{ii} Chosen by pole placement

undetermined system

closed loop poles

$$(s+2\xi a)(s^2+2\xi as+a^2)=0$$

$$u_1 = -k_1 e_1$$

 $u_2 = -k_2 \operatorname{sgn}(v_{ref}) e_2 - k_3 e_3$

$$k_1 = 2\xi a$$

$$k_2 = \frac{a^2 - \omega_r^2}{|v_{ref}|}$$

$$k_3 = 2\xi a$$

If v_r tends to zero, k₂ increases withouth bound



To avoid the previous problem:

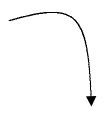
The closed-loop poles depend on the values of \boldsymbol{v}_{r} and \boldsymbol{w}_{r}

closed loop poles
$$(s+2\xi a)(s^2+2\xi as+a^2)=0$$

$$a = \sqrt{w_{ref}^2 + bv_{ref}^2}$$

$$u_1 = -k_1 e_1$$

 $u_2 = -k_2 sgn(v_r)e_2 - k_3 e_3$



$$k_{\scriptscriptstyle 1} = 2\xi \sqrt{w_{\scriptscriptstyle ref}^2 + b v_{\scriptscriptstyle ref}^2}$$

$$k_2 = b | v_{ref} |$$

$$k_3 = 2\xi \sqrt{w_{ref}^2 + bv_{ref}^2}$$



NONLINEAR FEEDBACK CONTROL

$$\dot{e} = \begin{bmatrix} 0 & \omega & 0 \\ -\omega & 0 & 0 \\ 0 & 0 & 1 \end{bmatrix} e + \begin{bmatrix} 0 \\ \sin e_3 \\ 0 \end{bmatrix} v_{ref} + \begin{bmatrix} 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

$$u_{1} = -k_{1}(v_{ref}, w_{ref})e_{1}$$

$$u_{2} = -k_{4}v_{ref} \frac{\sin e_{3}}{e_{3}}e_{2} - k_{3}(v_{ref}, w_{ref})e_{3}$$

k₄ positive constantk₁ continuos function strictly positive in RxR-(0,0)

k₃ continuos function strictly positive in RxR-(0,0)

See the analogy with the linear contol

Property: This control globally asymptotically stabilizies the origin e=0

demonstration using Lyapunov stability theory

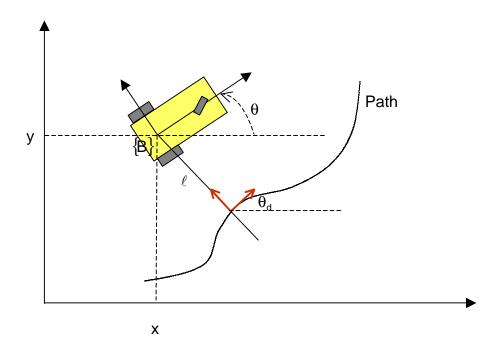
$$k_{1}(v_{ref}, w_{ref}) = k_{3}(v_{ref}, w_{ref}) = 2\xi \sqrt{w_{ref}^{2} + bv_{ref}^{2}}$$

 $k_{4} = b$



Objective:

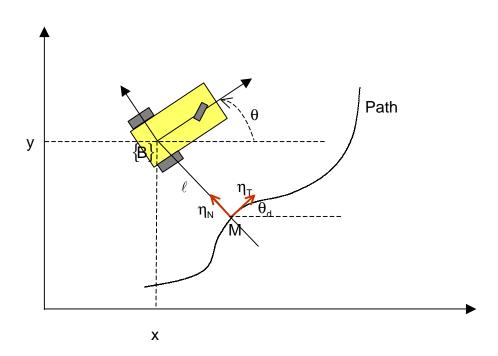
 Steer the vehicle at a constant forward speed along a predefined geometric path that is given in a time-free parametrization.



Approach:

- The controller should compute
 - The distance of the vehicle to the path
 - The orientation error between the vehicle's main axis and the tangent to the path
- The controller should act on the angular velocity to drive both to zero





- M is the orthogonal projection of the robot's position P on the path
 - M exists and is uniquely defined if the path satisfies some conditions

 (P, η_T, η_N)

- Serret-Frenet frame moving along the path.
 - The vector $\eta_{\scriptscriptstyle T}$ $\,$ is the tangent vector to the path in the closest point to the vehicle
 - The vector η_N is the normal
- ℓ is the distance between P and M
- **s** is the signed curvilinear distance along the path, from some initial path to the point M
- θ_d(s) is the angle betwen the vehicle's x-axis and the tange to the path at the point M.
- c(s) is the path's curvature at the point M, assumed uniformly bounded and differentiable
- $\widetilde{\theta} = \theta \theta_d$ is the orientation error



A new set of state coordinates for the mobile robot

$$(s, \ell, \widetilde{\theta})$$

They coincide with x,y,θ in the particular case where the path coincides with the x-axis

 The error dynamics can be derived writing the vehicle kinematic model in the Serret-Frenet frame:

$$\dot{s} = \frac{v \cos \tilde{\theta}}{1 - c(s)\ell}$$

$$\dot{\ell} = v \sin \tilde{\theta}$$

$$\dot{\tilde{\theta}} = w - \frac{v \cos \tilde{\theta}c(s)}{1 - c(s)\ell}$$

PROBLEM FORMULATION

Given a path in the x-y plane and the mobile robot translational velocity, v(t), (assumed to be bounded) together with its time-derivative dv(t)/dt, the path following problem consists of finding a (smooth) feedback control law

$$\omega = k(s,\ell,\widetilde{\theta},v(t))$$

such that

$$\lim_{t\to\infty}\ell(t)=0$$

$$\lim_{t\to\infty}\widetilde{\theta}(t)=0$$



$$\dot{s} = \frac{v \cos \tilde{\theta}}{1 - c(s)\ell}$$

$$\dot{\ell} = v \sin \tilde{\theta}$$

$$\dot{\tilde{\theta}} = w - \frac{v \cos \tilde{\theta}c(s)}{1 - c(s)\ell}$$

$$u = w - \frac{v \cos \widetilde{\theta} c(s)}{1 - c(s)\ell}$$

$$\dot{s} = \frac{v \cos \widetilde{\theta}}{1 - c(s)\ell}$$

$$\dot{\ell} = v \sin \widetilde{\theta}$$

$$\dot{\widetilde{\theta}} = u$$

Two different solutions:

- * Linear feedback control
- Nonlinear feedback control

LINEAR FEEDBACK CONTROL

Linearize the dynamics around the equilibrium point $\ (\ell=0,\,\widetilde{\theta}=0)$



$$\dot{\ell}(t) = v(t)\widetilde{\theta}(t)$$
 $\dot{\widetilde{\theta}}(t) = u(t)$

Linearization of the last two equations



lf

$$v(t)=v=cte \neq 0$$

The linear system is

CONTROLLABLE



ASYMPTOTICALY STABILIZABLE BY LINEAR STATE FEEDBACK

Linear stabilizing feedback

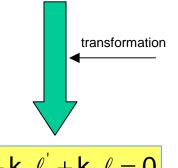
$$\mathbf{u} = -\mathbf{k}_2 \mathbf{v}\ell - \mathbf{k}_3 \mid \mathbf{v} \mid \widetilde{\mathbf{\theta}}$$

$$k_2 > 0, k_3 > 0$$



$$\ddot{\ell} + k_3 | v | \dot{\ell} + k_2 v^2 \ell = 0$$

Closed-loop differential equation



transformation
$$\ell' = \frac{\partial \ell}{\partial \gamma}$$

$$\gamma = \int\limits_0^t |\mathbf{V}| \, d\tau \quad \text{Distance gone by point M along the path}$$

$$\ell + K_3 \ell + K_2 \ell = 0$$

$$s^2 + 2\xi as + a^2 = 0$$

Desired closed-loop characeristic equation



NONLINEAR FEEDBACK CONTROL

$$\dot{s} = \frac{v \cos \widetilde{\theta}}{1 - c(s)\ell}$$

$$\dot{\ell} = v \sin \widetilde{\theta}$$

$$\dot{\widetilde{\theta}} = u$$

Nonlinear control law

$$u = -k_2 v \ell \frac{\sin \widetilde{\theta}}{\widetilde{\theta}} - k(v)\widetilde{\theta}$$

with

$$k_2 > 0$$

k(.) continuous function strictly positive

In order to have the two (linear and nonlinear) controllers behave similarly near $\ell=0,\ \widetilde{\theta}=0$ chose

$$k(v) = k_3 | v |$$

with
$$k_3 = 2\xi a$$

 $k_2 = a^2$



Property

Under the assumption

$$\lim_{t\to\infty}v(t)\neq 0$$

the non-linear control

$$u = -k_2 v \ell \frac{\sin \widetilde{\theta}}{\widetilde{\theta}} - k(v)\widetilde{\theta}$$

asymptotically stabilizes $(\tilde{\ell} = 0, \tilde{\theta} = 0)$

provided that the vehicle's initial configuration is such that

$$\ell(0)^2 + \frac{1}{k_2} \widetilde{\theta}(0)^2 < \frac{1}{\limsup(c(s)^2)}$$

Condition to guarantee that 1-c(s)l remains positive

The vehicle's location along the path is characterized by the value of **s** (the distance gone along the path)

Depends on v(t)

This degree of freedom can be used to stabilize $\bf s$ about a prescribed value $\bf s_d$



Point Stabilization

- Given
 - An arbitrary posture

$$z_d = (x, y, \theta)$$

- Find
 - A control law

$$\begin{pmatrix} v \\ \omega \end{pmatrix} = k(z-z_d,t)$$

which stabilizes asymptotically z-z_d about zero, whatever the initial robot's posture z(0)

COROLLARY

There is **no** smooth control law k(z) that can solve the point stabilization problem for the considered class of systems.

3 ALTERNATIVES

- Smooth (differentiable) time-varying nonlinear feedback k(z,t)
- Piecewise continuous control laws k(z)
- Time-varying piecewise continuous control laws k(z,t)



References

- C. Canudas de Wit, H. Khennouf, C. Samson, O. Sordalen, "Nonlinear Control Design for Mobile Robots" in Recent Developments in Mobile Robots, World Scientific, 1993.
 - Reading assignment
- Carlos Canudas de Wit, Bruno Siciliano and Georges Bastin (Eds), "Theory of Robot Control", 1996.