EE 264 SIST, ShanghaiTech

Model Reference Adaptive Control II

YW 10-1

Contents

- Direct MRAC with Unnormalized Adaptive Law
- Direct MRAC with Normalized Adaptive Law
- Indirect MRAC with Unnormalized Adaptive Laws
- Indirect MRAC with Normalized Adaptive Laws
- Case study: Adaptive Cruise Control Design

Unnormalized Adaptive Laws

Consider the plant with $n^* = 1$, the control law

$$\dot{\omega}_1 = F\omega_1 + gu_p, \quad \omega_1(0) = 0,$$

$$\dot{\omega}_2 = F\omega_2 + gy_p, \quad \omega_2(0) = 0,$$

$$u_p = \theta^\top \omega$$

where

$$\bullet \ \omega = \left[\omega_1^\top \omega_2^\top, y_p, r\right]^\top \in \mathbb{R}^{2n}$$

- $\theta(t)$ is the estimate of $\theta^* = \left[\theta_1^{*\top}, \theta_2^{*\top}, \theta_3^*, c_0^*\right]^\top$, calculated from the matching equation
- ullet (F,g) is the state-space realization of $rac{lpha(s)}{\Lambda(s)}$, i.e.

$$(sI - F)^{-1}g = \frac{\alpha(s)}{\Lambda(s)}$$

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Augmented state-space

$$\dot{Y}_c = A_0 Y_c + \mathbf{B}_c \theta^{*\top} \omega + B_c (u_p - \theta^{*\top} \omega), \quad Y_c(0) = Y_0$$
$$y_p = C_c^{\top} Y_c$$

where we have added and subtracted the desired input $B_c\theta^{*\top}\omega$

and
$$Y_c = \left[x_p^\intercal, \omega_1^\intercal, \omega_2^\intercal\right]^\intercal \in \mathbb{R}^{n_p+2n-2}$$

$$A_0 = \begin{bmatrix} A_p & 0 & 0 \\ 0 & F & 0 \\ gC_p^\top & 0 & F \end{bmatrix}, \quad B_c = \begin{bmatrix} B_p \\ g \\ 0 \end{bmatrix}$$
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$$A_0 = \begin{bmatrix} A_p & 0 & 0 \\ 0 & F & 0 \\ gC_p^\top & 0 & F \end{bmatrix}, \quad B_c = \begin{bmatrix} B_p \\ g \\ 0 \end{bmatrix},$$

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Absorb the term $B_c\theta^{*\top}\omega$ into homogeneous part

$$\dot{Y}_c = A_c Y_c + B_c c_0^* r + B_c \left(u_p - \theta^{*\top} \omega \right), \quad Y_c(0) = Y_0$$
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$$A_c = \begin{bmatrix} A_p + B_p \theta_3^* C_p^\top & B_p \theta_1^{*\top} & B_p \theta_2^{*\top} \\ g \theta_3^* C_p^\top & F + g \theta_1^{*\top} & g \theta_2^{*\top} \\ g C_p^\top & 0 & F \end{bmatrix}$$

is an Hurwitz matrix. Moreover, there exists nonminimal state space representation for reference model

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Define $e:=Y_c-Y_m$, $e_1:=y_p-y_m$, the tracking error equation

$$\dot{e} = A_c e + B_c \left(u_p - \theta^{*\top} \omega \right),$$

$$e_1 = C_c^{\top} e.$$

Since $C_c^{\top} (sI - A_c)^{-1} B_c c_0^* = W_m(s)$, we have

$$e_1 = W_m(s)\rho^* \left(u_p - \theta^{*\top}\omega\right)$$

where $\rho^* = \frac{1}{c_0^*}$.

Now, we can use above PM to generate a wide class of adaptive laws for estimating θ^* by using the results of parameter estimation.

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Alternatively, rewrite the tracking error equation

$$\dot{e} = A_c e + \bar{B}_c \rho^* \tilde{\theta}^\top \omega,$$

$$e_1 = C_c^\top e.$$

where $\bar{B}_c = B_c c_0^*$, Note

$$W_m(s) = C_c^{\top} (sI - A_c)^{-1} \bar{B}_c$$

is SPR and A_c is Hurwitz, it motivates the Lyapunov function

$$V(\tilde{\theta}, e) = \frac{e^{\top} P_c e}{2} + \frac{\tilde{\theta}^{\top} \Gamma^{-1} \tilde{\theta}}{2} |\rho^*|$$

where $\Gamma^{ op}=\Gamma>0$, $P_c=P_c^{ op}>0$ and satisfies equations

$$P_c A_c + A_c^{\top} P_c = -q q^{\top} - v_c L_c$$

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$$\dot{V} = -\frac{e^{\top}qq^{\top}e}{2} - \frac{v_c}{2}e^{\top}L_ce + e^{\top}P_c\bar{B}_c\rho^*\tilde{\theta}^{\top}\omega + \tilde{\theta}^{\top}\Gamma^{-1}\dot{\tilde{\theta}}|\rho^*|$$

Since $e^{\top}P_c\bar{B}_c=e^{\top}C_c=e_1$ and $\rho^*=|\rho^*|\operatorname{sgn}(\rho^*)$, choosing

$$\dot{\tilde{\theta}} = \dot{\theta} = -\Gamma e_1 \omega \operatorname{sgn}\left(\rho^*\right)$$

we make

$$\dot{V} = -\frac{e^{\top}qq^{\top}e}{2} - \frac{v_c}{2}e^{\top}L_ce \le 0$$

•
$$e, \tilde{\theta} \in \mathcal{L}_{\infty}, x_p, y_p, \omega_1, \omega_2 \in \mathcal{L}_{\infty}, u_p = \theta^{\top} \omega \in \mathcal{L}_{\infty}$$

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$$e, e_1 \rightarrow 0$$
 as $t \rightarrow \infty$

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implies

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$$e, \tilde{\theta} \in \mathcal{L}_{\infty}, x_p, y_p, \omega_1, \omega_2 \in \mathcal{L}_{\infty}, u_p = \theta^{\top} \omega \in \mathcal{L}_{\infty}$$

 \bullet $e, e_1 \to 0$ as $t \to \infty$.

Summary: Direct unnormalized MRAC for $n^* = 1$

Scheme
$$u_p = \theta^\top \omega$$

$$\dot{\omega}_1 = F\omega_1 + gu_p, \quad \omega_1(0) = \omega_{10} \in \mathbb{R}^{n-1}$$

$$\dot{\omega}_2 = F\omega_2 + gy_p, \quad \omega_1(0) = \omega_{10} \in \mathbb{R}^{n-1}$$

$$\dot{\theta} = -\Gamma e_1 \omega \operatorname{sgn}\left(\frac{k_p}{k_m}\right) \quad \theta(0) = \theta_0 \in \mathbb{R}^{2n}$$

Properties:

- (i) All signals in the closed-loop plant are bounded, and $e_1 \to 0$ asymptotically for any $r \in \mathcal{L}_{\infty}$.
- (ii) If r is S.R. of order $2n, \dot{r} \in \mathcal{L}_{\infty}$, and $Z_p(s), R_p(s)$ are relatively coprime, then $|\tilde{\theta}| \to 0, e_1 \to 0$ exponentially fast

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Problem : $W_m(s)$ is no longer SPR. Recall the error equation

$$\dot{\mathbf{e}} = A_c \mathbf{e} + \bar{B}_c \rho^* \tilde{\boldsymbol{\theta}}^\top \omega = A_c \mathbf{e} + \bar{B}_c (\mathbf{s} + \mathbf{p}_0) \rho^* \tilde{\boldsymbol{\theta}}^\top \phi,$$

$$\mathbf{e}_1 = C_c^\top \mathbf{e}.$$

with $\phi = \frac{1}{s+n_0}\omega$, $p_0 > 0$. In transfer function form, we have

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where
$$B_1 = A_c \bar{B}_c + \bar{B}_c p_0$$
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where
$$B_1=A_c\bar{B}_c+\bar{B}_cp_0$$
, $C_c^{\top}\bar{B}_c=C_p^{\top}B_pc_0^*=0$ due to $n^*=2$.

Now, we can apply the same adaptive law

$$\dot{\tilde{\theta}} = \dot{\theta} = -\Gamma e_1 \phi \operatorname{sgn}(\rho^*)$$

Scheme:

$$\dot{\omega}_1 = F\omega_1 + gu_p, \quad \omega_1(0) = 0 \in \mathbb{R}^{n-1},$$

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$$u_p = \theta^\top \omega + \dot{\theta}^\top \phi, \qquad \omega = \left[\omega_1^\top, \omega_1^\top, y_p, r\right]^\top,$$

Remark: When the relative degree is greater than 2 the procedure is very similar, but the control law becomes more complex, refer to Robust Adaptive Law Section 6.4.

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Recall control law

$$u_p = \theta_1^{\top}(t) \frac{\alpha(s)}{\Lambda(s)} u_p + \theta_2^{\top}(t) \frac{\alpha(s)}{\Lambda(s)} y_p + \theta_3(t) y_p + c_0(t) r$$

and the error equation

$$e_1 = W_m(s)\rho^* \left(u_p - \theta^{*\top}\omega\right)$$

where $\rho^* = \frac{1}{c_0^*} = \frac{k_p}{k_m}, \theta^* = \left[\theta_1^{*\top}, \theta_2^{*\top}, \theta_3^*, c_0^*\right]^{\top}$. Regardless of the relative degree, express it as a standard B-SPM

$$e_1 = \rho^* \left(\theta^{*\top} \phi + u_f \right)$$

where $\phi = -W_m(s)\omega$, $u_f = W_m(s)u_p$.

Use the result of Parameter Estimation, we have

$$\dot{\theta} = \Gamma \varepsilon \phi \operatorname{sgn}\left(\frac{k_p}{k_m}\right), \qquad \dot{\rho} = \gamma \varepsilon \xi$$

with
$$\varepsilon = \frac{e_1 - \rho \xi}{m_s^2}$$
, $m_s^2 = 1 + \phi^{\top} \phi + u_f^2$, $\xi = \theta^{\top} \phi + u_f$, $\phi = -W_m(s)\omega$, $u_f = W_m(s)u_p$.

The MRAC scheme applied to the plant with arbitrary n^* has the following properties:

- (i) All signals are uniformly bounded and the tracking error e_1 converges to zero as $t \to \infty$.
- (iii) If the reference input signal r is sufficiently rich of order $2n, \dot{r} \in \mathcal{L}_{\infty}$, and R_p, Z_p are coprime, the tracking error e_1 and parameter error $\tilde{\theta} = \theta \theta^*$ converge to zero exponentially fast.

Use the result of Parameter Estimation, we have

$$\dot{\theta} = \Gamma \varepsilon \phi \operatorname{sgn}\left(\frac{k_p}{k_m}\right), \qquad \dot{\rho} = \gamma \varepsilon \xi$$

with
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Alternatively, A linear parametric model for θ^* may be developed from

$$e_1 = W_m(s)\rho^* \left[u_p - \theta^{*\top}\omega \right]$$

as follows: Because $\rho^* = \frac{1}{c_0^*}$ and $\theta^{*\top}\omega = \theta_0^{*\top}\omega_0 + c_0^*r$ where $\theta_0^* = \left[\theta_1^{*\top}, \theta_2^{*\top}, \theta_3^*\right]^{\top}$ and $\omega_0 = \left[\omega_1^{\top}, \omega_2^{\top}, y_p\right]^{\top}$, we have

$$W_m(s)u_p = c_0^* e_1 + W_m(s)\theta_0^{*\top} \omega_0 + c_0^* W_m(s)r$$

Substituting for $e_1=y_p-y_m$ and using $y_m=W_m(s)r$ we obtain

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which may be written as

$$z = \theta^{*\top} \phi_p$$

where

$$z = W_m(s)u_p$$

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Note that: the adaptive laws based on the linear model generate estimates of c_0^* only, without any knowledge of the $\mathrm{sgn}\,(\rho^*)$ or lower bound for $|\rho^*|$. It turns out, however, that in the stability analysis of the MRAC schemes whose adaptive laws are based on linear model, $1/c_0(t)$ is required to be bounded.

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This can be guaranteed by modifying the adaptive laws for $c_0(t)$ using projection, so that

$$|c_0(t)| \ge \underline{c}_0 > 0, \forall t \ge 0$$

for some constant $\underline{c}_0 \leq |c_0^*| = \left|\frac{k_m}{k_p}\right|$.

Constraint:

$$g(\theta) = \underline{c}_0 - c_0 \operatorname{sgn} c_0 \le 0$$

Projection operator:

$$P_r[\Gamma x] \triangleq \begin{cases} \Gamma x & \text{if } |c_0(t)| > \underline{c}_0 \text{ or } \\ & \text{if } |c_0(t)| = \underline{c}_0 \text{ and } (\Gamma x)^\top \nabla g \leq 0 \\ \Gamma x - \Gamma rac{\nabla g \nabla g^\top}{\nabla g^\top \Gamma \nabla g} \Gamma x & \text{otherwise} \end{cases}$$

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 Model Reference Adaptive Control II

Example: Let us consider the second-order plant

$$y_p = \frac{k_p (s + b_0)}{(s^2 + a_1 s + a_0)} u_p$$

where $k_p > 0, b_0 > 0$, and k_p, b_0, a_1, a_0 are unknown constants.

The reference model is chosen as

$$y_m = \frac{3}{s+3}r$$

- Q1) Design a direct MRAC such that $y_p \to y_m$ as $t \to \infty$.
- Q2) Given a uniformly bounded reference signal, state(or Proof the boundedness and convergence property of related signals.
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Exercise: Let us consider the second-order plant

$$y_p = \frac{k_p}{s^2 + a_1 s + a_0} u_p$$

where $k_p > 0$ and a_1, a_0 are constants. The reference model is

$$y_m = \frac{9}{(s+3)^2}r$$

Contents

- Direct MRAC with Unnormalized Adaptive Law
- Direct MRAC with Normalized Adaptive Law
- Indirect MRAC with Unnormalized Adaptive Laws
- Indirect MRAC with Normalized Adaptive Laws
- Case study: Adaptive Cruise Control Design

In the case of $n^*=1$, control law admits the same form

$$\dot{\omega}_1 = F\omega_1 + gu_p, \quad \omega_1(0) = 0 \in \mathbb{R}^{n-1}$$
$$\dot{\omega}_2 = F\omega_2 + gy_p, \quad \omega_2(0) = 0 \in \mathbb{R}^{n-1}$$
$$u_p = \theta^\top \omega,$$

where $\omega = \left[\omega_1^\top, \omega_2^\top, y_p, r\right]^\top \in \mathbb{R}^{2n}$.

Rewrite the plant polynomials as

$$R_p(s) = s^n + a_{n-1}s^{n-1} + a_p^{\top}\alpha_{n-2}(s)$$
$$Z_p(s) = s^{n-1} + b_p^{\top}\alpha_{n-2}(s)$$

where $a_p, b_p \in \mathbb{R}^{n-1}$ and $\alpha_{n-2}(s) = [s^{n-2}, s^{n-3}, \cdots, 1]^{\top}$.

Express the plant parameter vector as

$$\theta_p^* = [k_p, b_p^\top, a_{n-1}, a_p^\top]^\top$$

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For indirect methods, the relation mapping the desired controller parameter θ^* and the plant parameter θ^*_p is critical. Recall the matching equation

$$\frac{c_0^* k_p Z_p \Lambda}{\left(\Lambda - \theta_1^{*\top} \alpha\right) R_p - k_p Z_p \left(\theta_2^{*\top} \alpha + \theta_3^* \Lambda\right)} = k_m \frac{Z_m}{R_m}$$

we can obtain the algebraic equations

$$c_0^* = \frac{k_m}{k_p}, \quad \theta_1^* = \lambda - b_p,$$

$$\theta_2^* = \frac{a_p - a_{n-1}\lambda + r_{n-1}\lambda - v}{k_p}, \quad \theta_3^* = \frac{a_{n-1} - r_{n-1}}{k_p}$$

where λ, r_{n-1}, v are the coefficients of

$$\Lambda(s) = s^{n-1} + \lambda^{\top} \alpha_{n-2}(s)$$

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$$\Lambda(s) = s^{n-1} + \lambda^{\top} \alpha_{n-2}(s)$$
$$\Lambda_0(s) R_m(s) = s^n + r_{n-1} s^{n-1} + v^{\top} \alpha_{n-2}(s)$$

If we let $\theta_p = [\hat{k}_p(t), \hat{b}_p(t)^\top, \hat{a}_{n-1}(t), \hat{a}_p(t)^\top]^\top$ be the estimates of θ_p^* , then $\theta(t) = \left[\theta_1^\top(t), \theta_2^\top(t), \theta_3(t), c_0(t)\right]^\top$ can be calculated as

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The adaptive law for generating $\hat{k}_p, \hat{b}_p, \hat{a}_p, \hat{a}_{n-1}$ is constructed by considering the parametric model

$$e_1 = W_m(s)\rho^* \left(u_p - \theta^{*\top}\omega\right)$$

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$$\begin{split} c_0(t) &= \frac{k_m}{\hat{k}_p(t)} \\ \theta_1(t) &= \lambda - \hat{b}_p(t) \\ \theta_2(t) &= \frac{\hat{a}_p(t) - \hat{a}_{n-1}(t)\lambda + r_{n-1}\lambda - v}{\hat{k}_p(t)} \\ \theta_3(t) &= \frac{\hat{a}_{n-1}(t) - r_{n-1}}{\hat{k}_p(t)} \end{split}$$

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Note, in the case of $n^* = 1$, $W_m(s)$ is SPR.

Express the error equation PM in terms of $\tilde{\theta}_p$:

$$e_1 = W_m(s) \frac{1}{k_m} \left(\tilde{k}_p \xi_1 + \tilde{a}_{n-1} \xi_2 + \tilde{a}_p^{\top} \omega_2 - k_p \tilde{b}_p^{\top} \omega_1 \right)$$

where $\xi_1 \triangleq \lambda^\top \omega_1 - u_p - \hat{b}_p^\top \omega_1, \xi_2 \triangleq y_p - \lambda^\top \omega_2$. A state-space representation is

$$\dot{e} = A_c e + B_c \left(\tilde{k}_p \xi_1 + \tilde{a}_{n-1} \xi_2 + \tilde{a}_p^{\top} \omega_2 - k_p \tilde{b}_p^{\top} \omega_1 \right)$$

$$e_1 = C_c^{\top} e$$

where $C_c^{\top}(sI-A_c)^{-1}B_c=\frac{1}{k_m}W_m(s)$. Motivating the Lyapunov-like function

$$V = \frac{e^{\top} P_c e}{2} + \frac{\tilde{k}_p^2}{2\gamma_p} + \frac{\tilde{a}_{n-1}^2}{2\gamma_1} + |k_p| \frac{\tilde{b}_p^{\top} \Gamma_1^{-1} \tilde{b}_p}{2} + \frac{\tilde{a}_p^{\top} \Gamma_2^{-1} \tilde{a}_p}{2}$$

where $P_c = P_c^{\top} > 0$ satisfies the MKY lemma. i.e. $P_c B_c = C_c^{\top}$.

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By rendering \dot{V} as

$$\dot{V} \le -v_c \frac{e^{\top} L_c e}{2}$$

we have the adaptive laws

$$\begin{split} &\dot{\hat{a}}_{n-1} = -\gamma_1 e_1 \xi_2 \\ &\dot{\hat{a}}_p = -\Gamma_2 e_1 \omega_2 \\ &\dot{\hat{b}}_p = \Gamma_1 e_1 \omega_1 \operatorname{sgn}\left(k_p\right) \\ &\dot{\hat{k}}_p = \left\{ \begin{array}{ll} & -\gamma_p e_1 \xi_1 & \text{if } \left|\hat{k}_p\right| > k_0 \text{ or if } \left|\hat{k}_p\right| = k_0 \text{ and } e_1 \xi_1 \operatorname{sgn}\left(k_p\right) \leq 0 \\ &0 & \text{otherwise} \end{array} \right. \end{split}$$

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Example: Let us consider the second-order plant

$$y_p = \frac{k_p}{s + a_0} u_p$$

where $k_p > k_0 > 0$ and a_0 are constants. The reference model is

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- Case study: Adaptive Cruise Control Design

The simplicity of Indirect MRAC with normalized adaptive law arises from the fact that the control and adaptive laws are designed independently.

Step 1: Starting with the plant equation

$$y_p = \frac{b_m s^m + b_{m-1} s^{m-1} + \dots + b_0}{s^n + a_{m-1} s^{m-1} + \dots + a_0} u_p$$

with $b_m = k_p$. Form the expression into a SPM

$$z = \theta_p^{*\top} \phi$$

$$z = \frac{s^n}{\Lambda_p(s)} y_p, \quad \phi = \left[\frac{\alpha_m^{\top}(s)}{\Lambda_p(s)} u_p, -\frac{\alpha_{n-1}^{\top}(s)}{\Lambda_p(s)} y_p \right]^{\top}$$

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$$Q_p^* = \left[b_m, \dots, b_0, a_{n-1}, \dots, a_0 \right]^{\top}$$

The simplicity of Indirect MRAC with normalized adaptive law arises from the fact that the control and adaptive laws are designed independently.

Step 1: Starting with the plant equation

$$y_p = \frac{b_m s^m + b_{m-1} s^{m-1} + \dots + b_0}{s^n + a_{n-1} s^{n-1} + \dots + a_0} u_p$$

with $b_m=k_p$. Form the expression into a SPM

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Step II: Control law

$$u_p = \theta_1^{\top} \frac{\alpha(s)}{\Lambda(s)} u_p + \theta_2^{\top} \frac{\alpha(s)}{\Lambda(s)} y_p + \theta_3 y_p + c_0 r = \theta^{\top} \omega$$

where

$$\omega = \left[\frac{\alpha^{\top}(s)}{\Lambda(s)} u_p, \frac{\alpha^{\top}(s)}{\Lambda(s)} y_p, y_p, r \right]^{\top}, \quad \alpha(s) = \alpha_{n-2}(s)$$

and $\theta = \left[\theta_1^{\intercal}, \theta_2^{\intercal}, \theta_3, c_0\right]^{\intercal}$ is calculated using the mapping $\theta(t) = f\left(\theta_p(t)\right)$, which obtained from the matching equation

$$c_0^* = \frac{k_m}{k_p}, \quad \theta_1^* = \lambda - b_p,$$

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Step III: Adaptive Law, e.g. the gradient algorithm

$$\begin{split} \dot{\theta}_p &= \Pr(\Gamma \varepsilon \phi) \\ \varepsilon &= \frac{z - \theta_p^T \phi}{m_s^2}, \quad m_s^2 = 1 + \phi^T \phi \end{split}$$

where $\Pr(\cdot)$ guarantees namely $\hat{b}_m = \hat{k}_p$, satisfies $\hat{b}_m \operatorname{sgn}(k_p) \ge k_0 > 0$, $\forall t \ge 0$, provided $\hat{b}_m(0) \operatorname{sgn}(k_p) \ge k_0 > 0$. The $\Pr(\cdot)$ operator does not affect the other elements of θ_p .

Property: The indirect MRAC with normalized gradient-based adaptive law guarantees that all signals are bounded and the tracking error $e_1=y_p-y_m$ converges to zero as $t\to\infty$.

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Contents

- Direct MRAC with Unnormalized Adaptive Law
- Direct MRAC with Normalized Adaptive Law
- Indirect MRAC with Unnormalized Adaptive Laws
- Indirect MRAC with Normalized Adaptive Laws
- Case study: Adaptive Cruise Control Design

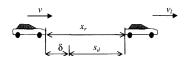


Figure: car following

The ACC system regulates the following vehicle's speed v towards the leading vehicle's speed v_l and maintains the intervehicle distance x_r close to the desired spacing s_d , that is given by

$$s_d = s_0 + hv$$

- ullet s_0 is a fixed safety intervehicle spacing
- h is constant time headway

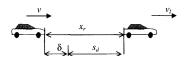


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$$v_r \to 0$$
, $\delta \to 0$ as $t \to \infty$

where $v_r:=v_l-v$ is the speed error or relative speed and $\delta:=x_r-s_d$ is the separation or spacing error.

In addition, we wish

- \bullet $a_{\min} < v < a_{\max}$
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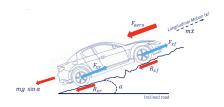
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The full order longitudinal vehicle model



$$m\ddot{x} = F_{xf} + F_{xr} - F_{aero} - R_{xf} - R_{xr} - mg\sin\alpha$$

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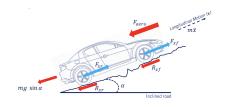
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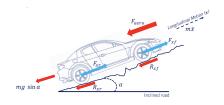
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Simplification is necessary and feasible because:

- the human driver can drive the vehicle without having in mind such a complex nonlinear system
- the brake/throttle commands are low frequency inputs

Step 0 Simplified first-order model:

$$\dot{v} = -av + bu + d$$

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- d is the modeling uncertainty
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$$v_m = \frac{a_m}{s + a_m} \left(v_l + k\delta \right)$$

where a_m,k are positive design constants, reference signal $v_l+k\delta$ represents the desired speed the following vehicle has to reach in order to match the speed of the lead vehicle and reduce the spacing error to zero.

Step II Check assumptions and calculate optimal control law:

$$u = k_1^* v_r + k_2^* \delta + k_3^*$$

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$$V = \frac{e^2}{2} + \sum_{i=1}^{3} \frac{b}{2\gamma_i} \tilde{k}_i^2$$

where $\gamma_i > 0$ and b, even though unknown, is always positive.

$$\dot{V} = -a_m e^2 + be\left(\tilde{k}_1 v_r + \tilde{k}_2 \delta + \tilde{k}_3\right) + \sum_{i=1}^3 \frac{b}{\gamma_i} \tilde{k}_i \dot{\tilde{k}}_i$$

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$$\dot{k}_1 = \Pr\left\{-\gamma_1 e v_r\right\}, \dot{k}_2 = \Pr\left\{-\gamma_2 e \delta\right\}, \dot{k}_3 = \Pr\left\{-\gamma_3 e\right\}$$

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where $\dot{k}_3^* = \frac{a\dot{v}_l - \dot{d}}{b}$ is a bounded disturbance term.

- $e \in \mathcal{L}_{\infty}$, $k_i \in \mathcal{L}_{\infty}$, which in turn implies that all the other signals in the closed-loop system are bounded
- ullet if v_l and d are constants, $e \in \mathcal{L}_2$ and e, δ converge to zero with time

Note that, the above adaptive control scheme is designed for the throttle subsystem. For the brake subsystem another controller is developed.

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Step VII Simulation verification

- Leading vehicle with constant speed
- Leading vehicle with slowly change speed
- Consider constant disturbance
- Consider slowly time-varying disturbance
- Consider measurement noise
- ...