

A multirate variational approach to simulation and optimal control for flexible spacecraft

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Outline



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- Short overview
- Mathematical model
- Multirate DMOC
- Computational results and discussion

Overview

- **Flexible spacecraft**
 - Stringent performance and positioning requirements
 - Efficient operation
 - Lightweight structure designs and induced vibrations- potentially degrading the performance, causing loss of pointing accuracy or even structural damage
- > Need for efficient control method maximizing the system's performance while respecting all hard safety-critical constraints
- **Multirate Discrete Mechanics and Optimal Control (Multirate DMOC)¹⁻⁴ :**
 - ✓ **High fidelity simulation at a reduced computational cost**

Advantages



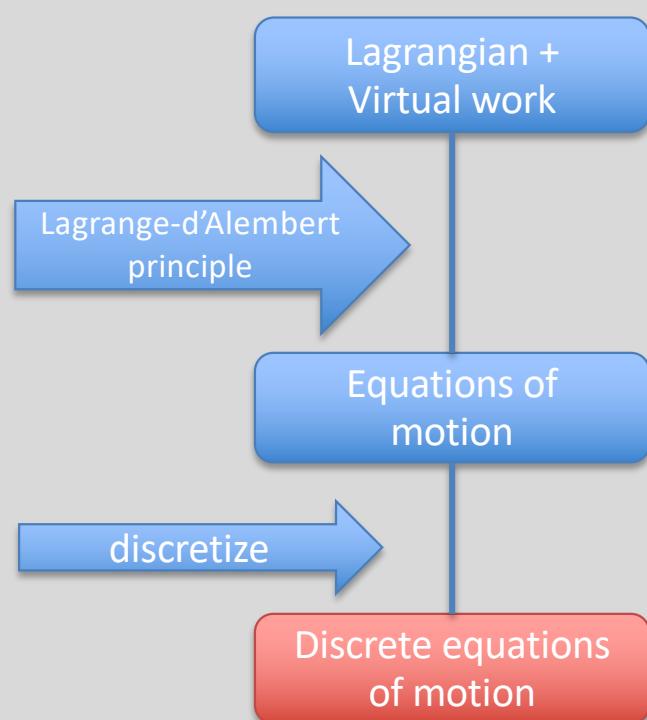
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- Multirate Discrete Mechanics and Optimal Control (Multirate DMOC):

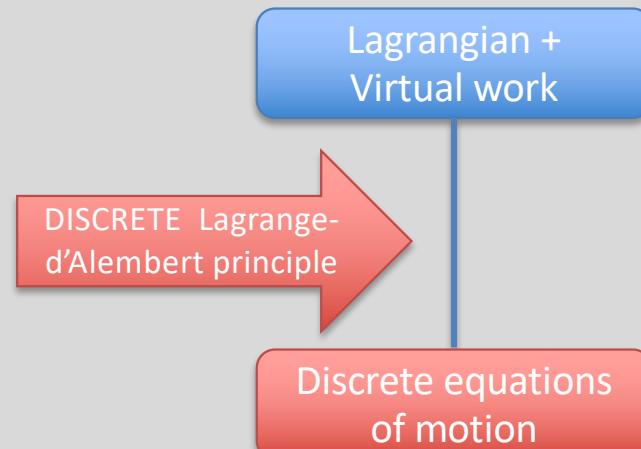
High fidelity simulation at a reduced computational cost

- ✓ Structure-preserving -> *conservation of energy and/or momenta of the system*

Standard direct methods



DMOC

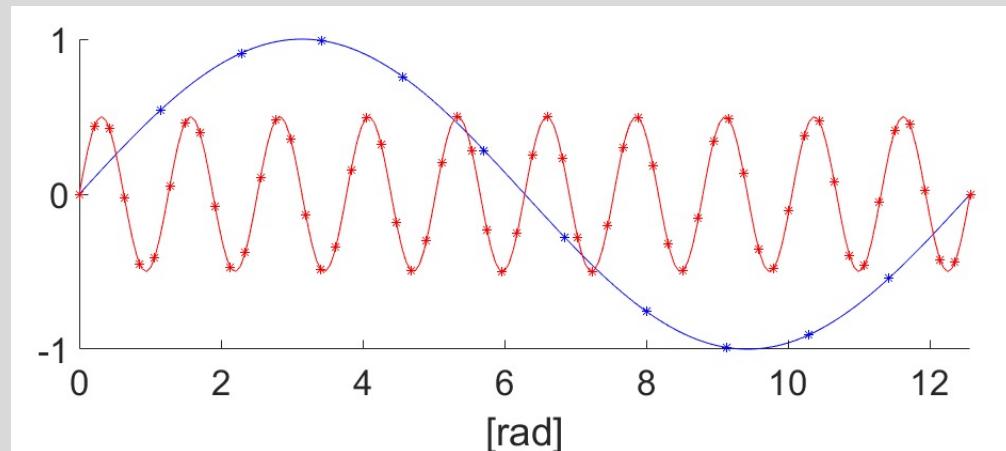
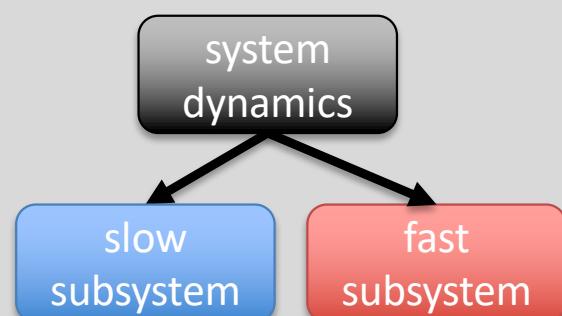


Advantages

- Multirate Discrete Mechanics and Optimal Control (Multirate DMOC):

High fidelity simulation at a reduced computational cost

- ✓ Structure-preserving -> *conservation of energy and/or momenta of the system*
- ✓ **Multirate discretization** -> *computational efficiency*⁵



- Reduces the number of optimization variables
- Reduces the number of equality constraints

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- ✓ Multirate discretization -> *computational efficiency*
- ✓ Unified control methodology -> *potential improvement in optimality and constraint-handling capabilities**

*For comparison see: M. Azadi, M. Eghtesad, S. Fazelzadeh, and E. Azadi, "Dynamics and control of a smart flexible satellite moving in an orbit," *Multibody System Dynamics*, Vol. 35, No. 1, 2015, pp. 1–23.



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- ✓ **No need for decoupling of the equations of motion** -> *simple application to a variety of linear and nonlinear models**

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 - ✓ **Straightforward selection of slow and fast subsystems -> capability to tailor the solution to the time scales present in the problem to obtain further reductions in computational cost***

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Mathematical model



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- **General linear model neglecting dissipation effects**

- single-axis rest-to-rest rotational maneuver
- control torque applied at the hub

- **Spatial discretization - The Assumed Modes Method⁷**

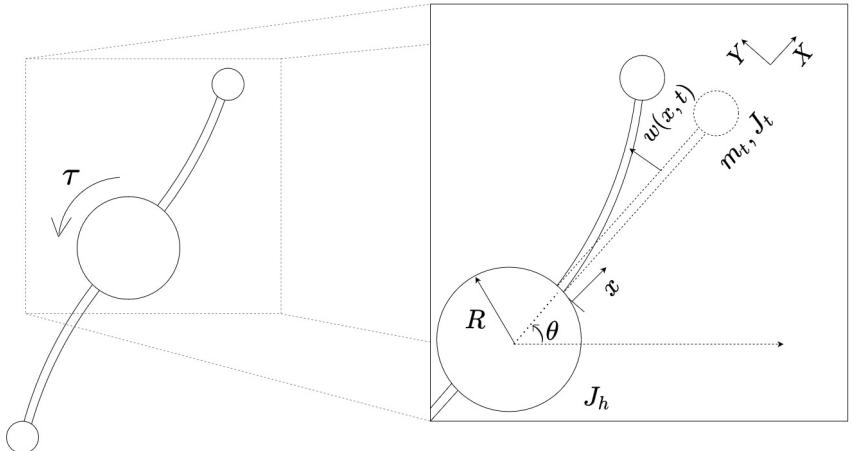
$$w(x, t) = \sum_{j=1}^N \phi_j(x) \eta_j(t), \quad x \in [0, L]$$

ϕ_j - assumed spatial mode shapes

η_j - time varying modal amplitudes

N - number of modes retained in the approximation

L - length of the beam



- **System description⁷**

$$\underline{\xi} = \begin{bmatrix} \theta \\ \underline{\eta} \end{bmatrix}, \quad \underline{\eta} = [\eta_1, \eta_2, \dots, \eta_N]^T \in R^{N \times 1}$$

- **Lagrange - d'Alembert principle**

$$\mathcal{L}(\underline{\xi}, \dot{\underline{\xi}}) = \frac{1}{2} \dot{\underline{\xi}}^T \mathbf{M} \dot{\underline{\xi}} - \frac{1}{2} \underline{\xi}^T \mathbf{K} \underline{\xi}, \quad \delta W = \underline{f} \cdot \delta \underline{\xi} = \tau \delta \theta$$

$$\delta \int_{t_0}^{t_f} \mathcal{L}(\underline{\xi}, \dot{\underline{\xi}}) dt + \int_{t_0}^{t_f} (\underline{f} \cdot \delta \underline{\xi}) dt = 0 \text{ for all variations } \delta \underline{\xi} \text{ with } \delta \underline{\xi}(t_0) = \delta \underline{\xi}(t_f) = 0$$

$$\mathbf{M} \ddot{\underline{\xi}} + \mathbf{K} \underline{\xi} = \mathbf{D} \tau, \quad \mathbf{M} = \begin{bmatrix} M_{\theta\theta} & M_{\theta\eta}^T \\ M_{\theta\eta} & M_{\eta\eta} \end{bmatrix}, \quad \mathbf{K} = \begin{bmatrix} 0 & 0 \\ 0 & K_{\eta\eta} \end{bmatrix}, \quad \mathbf{D} = [1, 0, \dots, 0]^T$$

Transformation to modal coordinates



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$$\begin{aligned}\underline{\xi} &= \begin{bmatrix} \theta \\ \eta \end{bmatrix} \\ \mathcal{L}(\underline{\xi}, \dot{\underline{\xi}}) &= \frac{1}{2} \dot{\underline{\xi}}^T \mathbf{M} \dot{\underline{\xi}} - \frac{1}{2} \underline{\xi}^T \mathbf{K} \underline{\xi} \\ \underline{f} &= \mathbf{D} \tau\end{aligned}$$

transformation to
modal coordinates⁷

$$\begin{aligned}\underline{\xi} &= \mathbf{E} \underline{q} \\ \mathcal{L} &= \frac{1}{2} (\dot{\underline{q}}^T \dot{\underline{q}} - \underline{q}^T \mathbf{\Lambda} \underline{q})\end{aligned}$$

$$\underline{f} = \mathbf{E}^T \mathbf{D} \tau$$

$$\mathbf{M} \ddot{\underline{\xi}} + \mathbf{K} \underline{\xi} = \mathbf{D} \tau$$

$$\ddot{\underline{q}} + \mathbf{\Lambda} \underline{q} = \mathbf{E}^T \mathbf{D} \tau$$

$$\mathbf{E}^T \mathbf{M} \mathbf{E} = \mathbf{I}, \quad \mathbf{E}^T \mathbf{K} \mathbf{E} = \mathbf{\Lambda},$$

$$w_1 \leq w_2 \leq \dots \leq w_{N+1},$$

$$\mathbf{\Lambda} = \begin{bmatrix} \omega_1^2 & \cdots & 0 \\ \vdots & \ddots & \vdots \\ 0 & \cdots & \omega_{N+1}^2 \end{bmatrix}$$

Table 1: Structural parameters used for the simulations

Hub radius	R	1.0	ft
Hub rotary inertia	J_h	8.0	slug-ft ²
Tip mass	m_t	0.156941	slug
Tip mass rotary inertia	J_t	0.0018	slug-ft ²
Beam length	L	4.0	ft
Beam height	h	6.0	in.
Beam thickness	t	0.125	in.
Beam linear density	ρA	0.0271875	slug/ft
Beam elastic modulus	E	0.1584×10^{10}	lb/ft ²

Table 2: Natural frequencies

w_1	0	rad/s
w_2	6.454	rad/s
w_3	52.41	rad/s
w_4	1.607×10^2	rad/s
w_5	3.381×10^2	rad/s
w_6	5.78×10^2	rad/s

$$N = 5$$

Multirate formulation

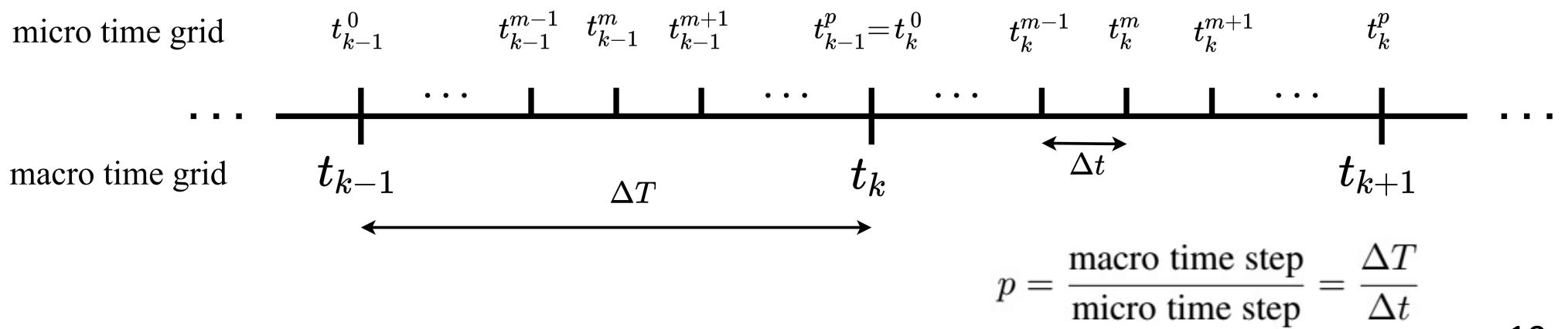


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$$\underline{q} = \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \end{bmatrix} \begin{cases} \underline{q}_s \\ \underline{q}_f \end{cases} \longrightarrow \begin{array}{l} w_1 = 0 \text{ rad/s} \\ w_2 = 6.454 \text{ rad/s} \\ w_3 = 52.41 \text{ rad/s} \\ w_4 = 1.607 \times 10^2 \text{ rad/s} \\ w_5 = 3.381 \times 10^2 \text{ rad/s} \\ w_6 = 5.78 \times 10^2 \text{ rad/s} \end{array}$$

$$\mathcal{L} = \frac{1}{2} \left((\dot{\underline{q}}^s)^T \dot{\underline{q}}^s - (\underline{q}^s)^T \boldsymbol{\Lambda}_s \underline{q}^s \right) + \frac{1}{2} \left((\dot{\underline{q}}^f)^T \dot{\underline{q}}^f - (\underline{q}^f)^T \boldsymbol{\Lambda}_f \underline{q}^f \right)$$

where $\boldsymbol{\Lambda}_s = \text{diag}(\omega_1^2, \dots, \omega_3^2)$, $\boldsymbol{\Lambda}_f = \text{diag}(\omega_4^2, \dots, \omega_6^2)$



Multirate formulation

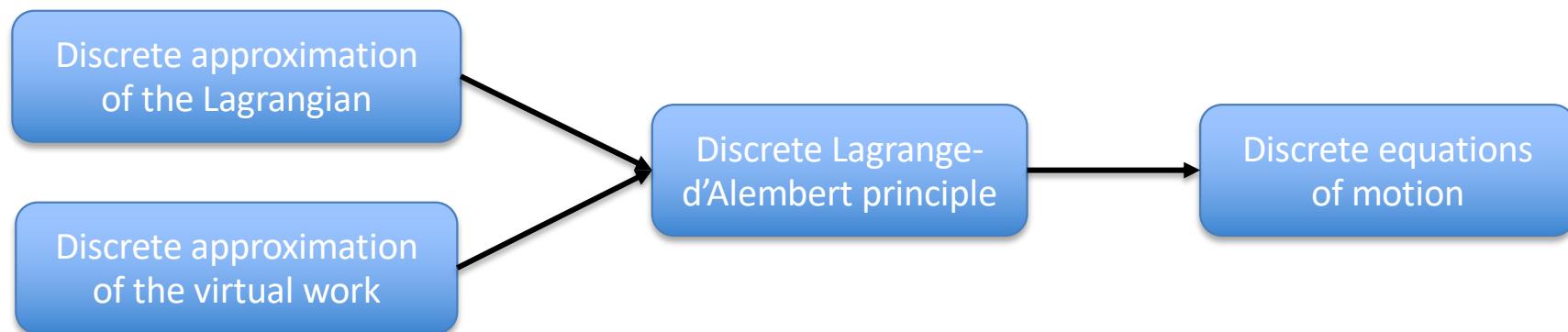


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Forward simulation without control



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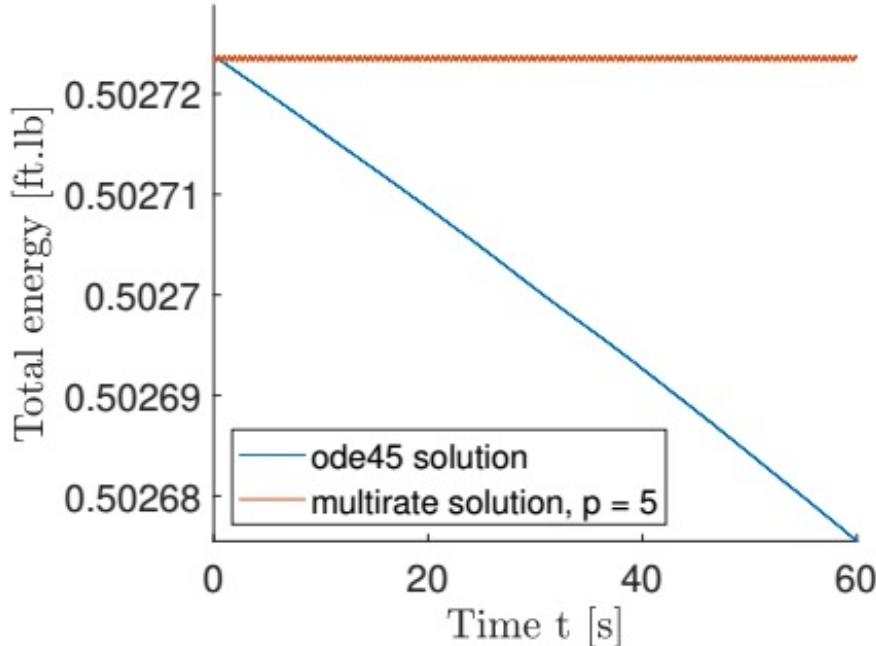


Figure 1: Numerical dissipation of total energy for simulations with $\Delta t = 10^{-4}$ and $t_f = 60\text{s}$

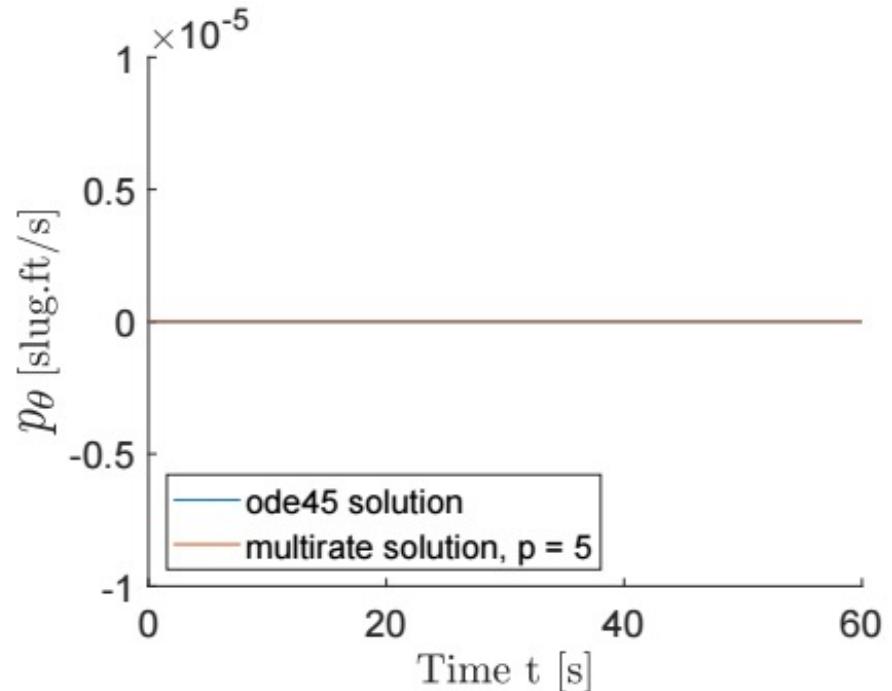


Figure 2: Momentum preservation for simulations with $\Delta t = 10^{-4}$ and $t_f = 60\text{s}$

$$p_\theta = \frac{\partial \mathcal{L}}{\partial \dot{\theta}} = M_{\theta\theta} \dot{\theta} + (M_{\theta\eta})^T \underline{\dot{\eta}}$$

Optimal control problem formulation



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$$J(\underline{x}, u) = \int_{t_0}^{t_f} C(\underline{x}(t), u(t)) dt = \frac{1}{2} \int_{t_0}^{t_f} [\underline{x}(t)^T \mathbf{W} \underline{x}(t) + u(t)^2] dt$$

subject to

$$\left\{ \begin{array}{l} \dot{\underline{x}}(t) = \mathbf{A}\underline{x}(t) + \mathbf{B}\tau(t) \\ \underline{q}(t_0) = \mathbf{E}^{-1} \underline{\xi}_{t_0}, \quad \underline{\xi}_{t_0} = [0, \dots, 0]^T \\ \underline{q}(t_f) = \mathbf{E}^{-1} \underline{\xi}_{t_f}, \quad \underline{\xi}_{t_f} = [\theta_{t_f}, 0, \dots, 0]^T \\ \dot{\underline{q}}(t_0) = \dot{\underline{q}}(t_f) = [0, \dots, 0]^T \end{array} \right.$$

where $\underline{x}(t) = \begin{bmatrix} q(t) \\ \dot{q}(t) \end{bmatrix}$, $u(t) = \tau(t)$, $\mathbf{A} = \begin{bmatrix} 0 & \mathbf{I} \\ -\boldsymbol{\Lambda} & 0 \end{bmatrix}$, $\mathbf{B} = \begin{bmatrix} 0 \\ \mathbf{E}^T \mathbf{D} \end{bmatrix}$, $\mathbf{W} = \mathbf{I}$

Example solution

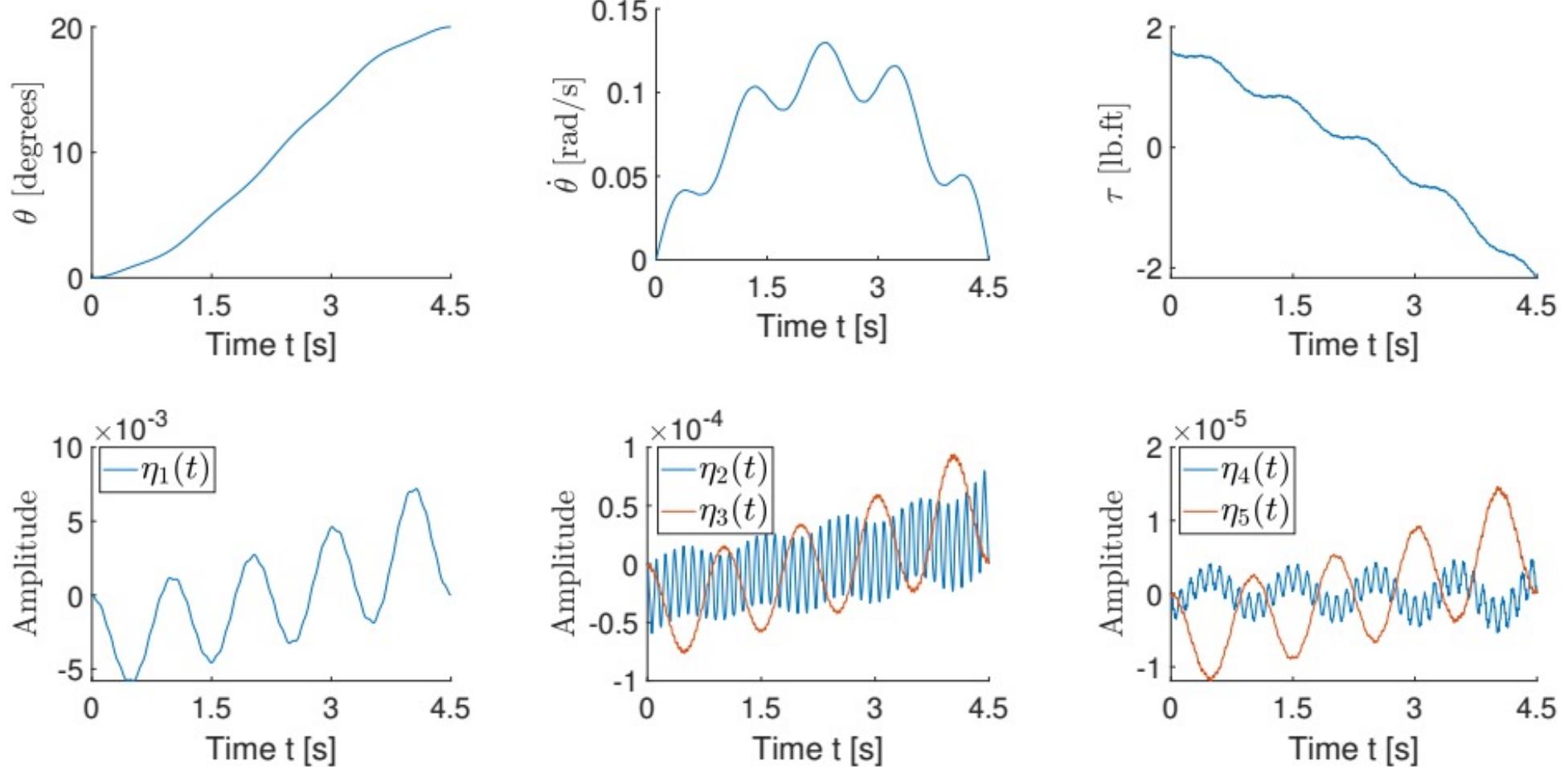


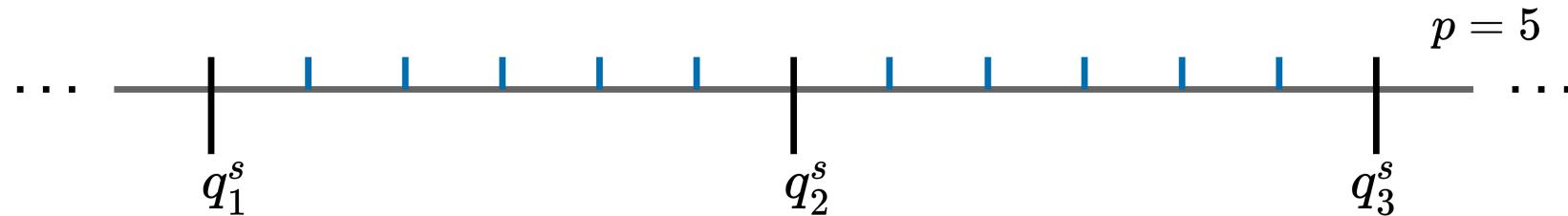
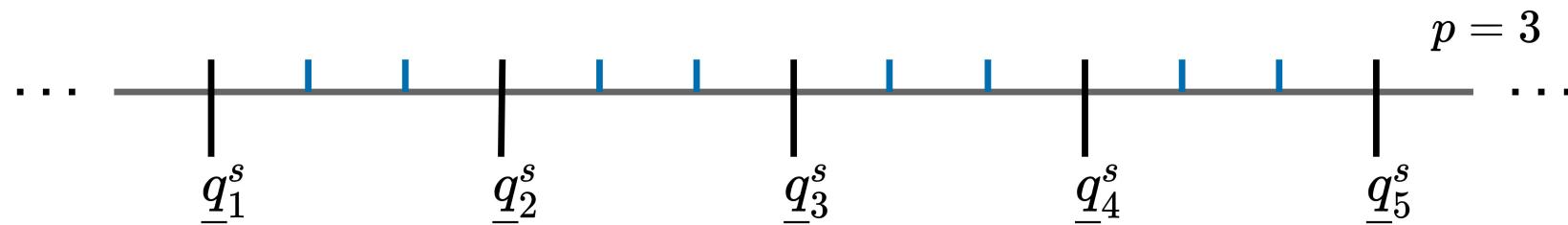
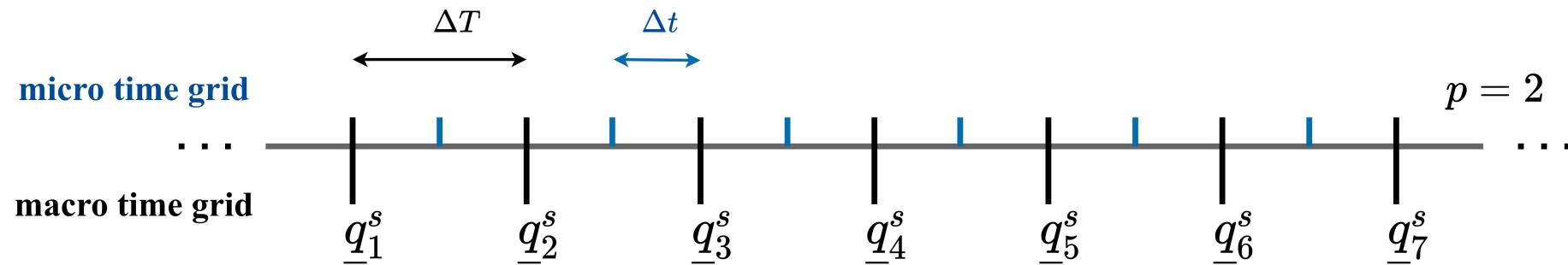
Figure 3: OCP solution with Multirate DMOC for $\theta_{t_f} = 20^\circ$, $t_f = 4.5s$, $\Delta t = 10^{-3}$ and $p = 5$

Problem size



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$$p = \frac{\text{macro time step}}{\text{micro time step}} = \frac{\Delta T}{\Delta t}$$



Problem size



$n_{eq\ con}$ - number of equality constraints

$n_{slow\ var}$ - number of optimization variables resulting from discretization on the macro grid

$n_{fast\ var}$ - number of optimization variables resulting from discretization on the micro grid

$$n_{total\ var} = n_{slow\ var} + n_{fast\ var}$$

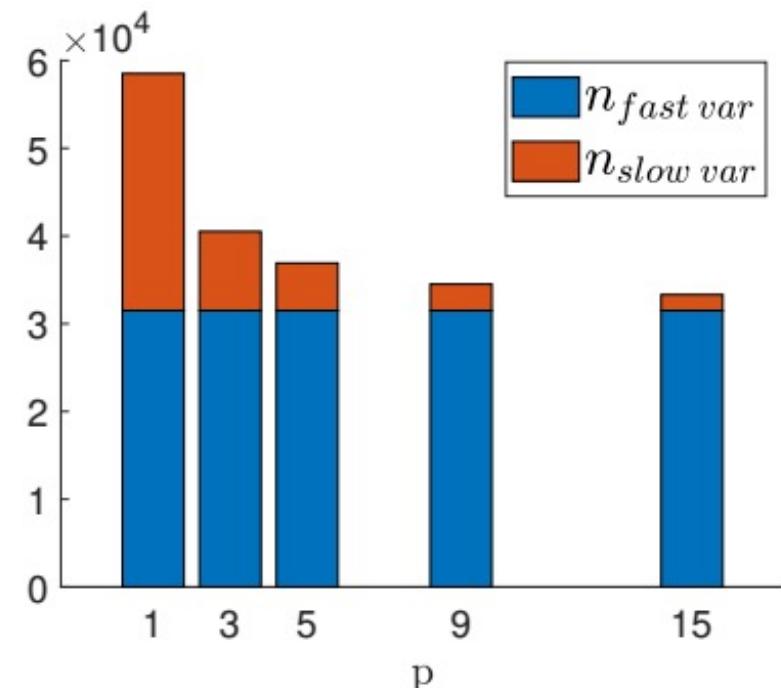
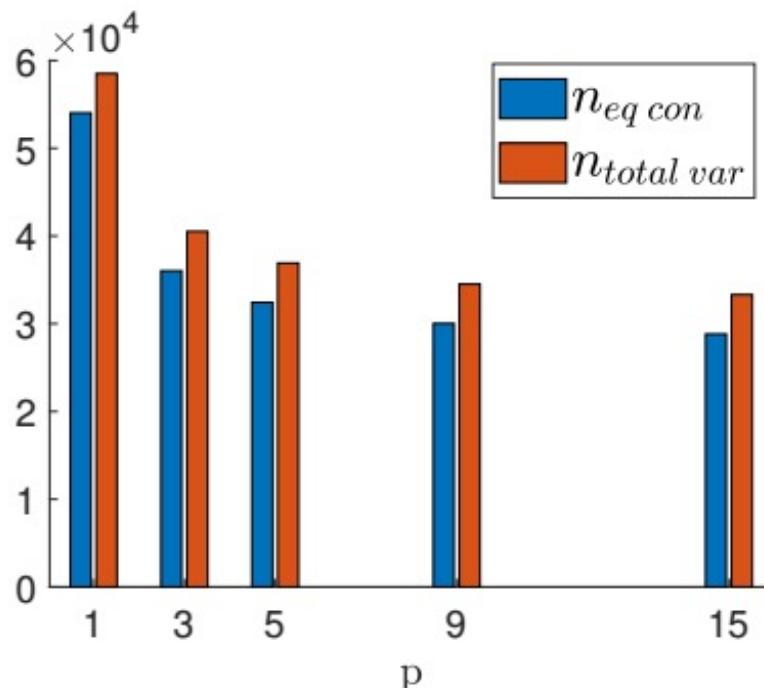


Figure 4: Size of OCP based on Multirate DMOC for a simulations with $\Delta t = 10^{-3}$ and $t_f = 4.5s$

Main result- the trade-off



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$$e_{\underline{\xi}}^{rel}(\underline{\xi}_d, \underline{\xi}) = \frac{\max_{k=0, \dots, n_s} (\|\underline{\xi}_k - \underline{\xi}(t_k)\|_\infty)}{\max_{k=0, \dots, n_s} (\|\underline{\xi}(t_k)\|_\infty)}$$

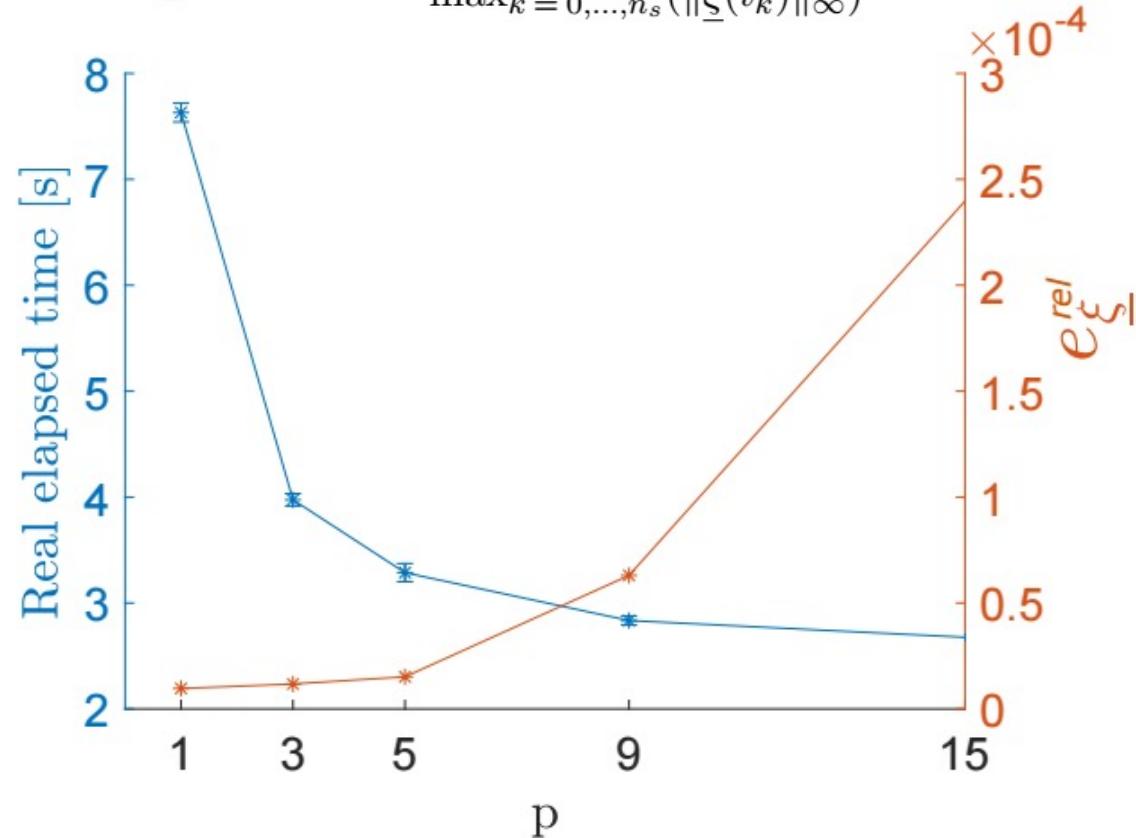


Figure 5: Mean computational time with standard deviation and relative error in $\underline{\xi}$ versus p for a constant micro time step of 10^{-3} , $t_f = 4.5s$ and $\theta_{t_f} = 20^\circ$

Main result- the trade-off



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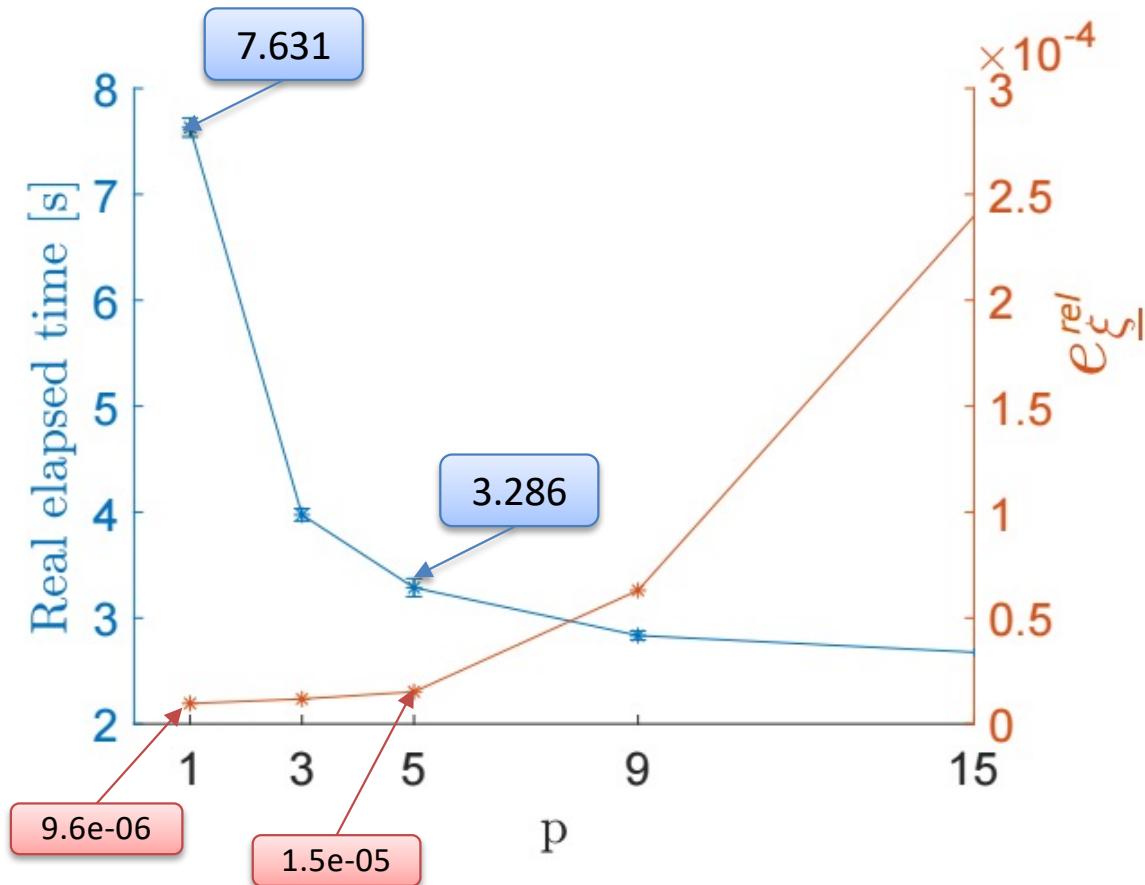


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Further customization



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$$\underline{q}^s \in R^{r \times 1}$$

r - degrees of freedom of the slow subsystem

$$\underline{q}^f \in R^{(N+1-r) \times 1}$$

so far in the examples $r = 3, N = 5$

$$n_{total\ var} = n_{slow\ var} + n_{fast\ var}$$

$$n_{slow\ var}(p, r, N, t_f, \Delta t) = 2 r \left(\frac{t_f}{p \Delta t} + 1 \right)$$

$$n_{fast\ var}(r, N, t_f, \Delta t) = 2 (N + 1 - r) \left(\frac{t_f}{\Delta t} + 1 \right) + \frac{t_f}{\Delta t}$$

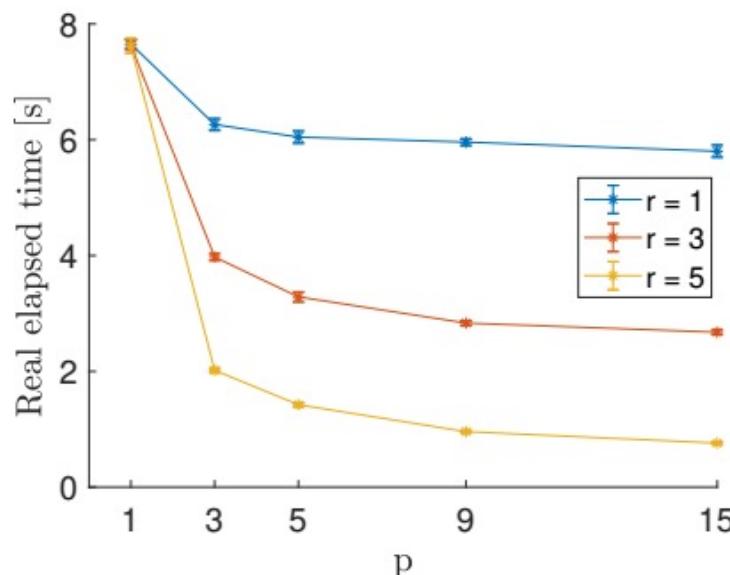


Figure 6: Mean computational time with standard deviation versus p for a constant micro time step of 10^{-3} , $t_f = 4.5s$ and $\theta_{t_f} = 20^\circ$

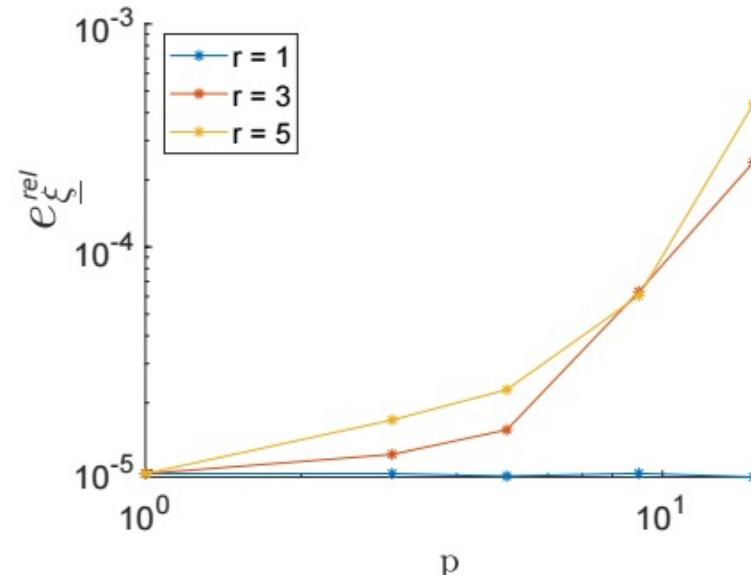


Figure 7: Relative error in ξ versus p for a constant micro time step of 10^{-3} , $t_f = 4.5s$ and $\theta_{t_f} = 20^\circ$



Conclusions

- **Multirate Discrete Mechanics and Optimal Control (Multirate DMOC):**

High fidelity simulation at a reduced computational cost

- ✓ Structure-preserving -> *conservation of energy and/or momenta of the system*
- ✓ Multirate discretization -> *computational efficiency*
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Flexibility to tailor the method to the time scales of the problem and obtain the required fidelity at a reduced computational cost!

- **Future work** – methods for obtaining optimal p and r and extending the work to models including kinematic nonlinearities and dissipation effects



THANK YOU FOR YOUR ATTENTION!

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Main references:

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