

Self-Driving Cars

Lecture 6 – Vehicle Control

Prof. Dr.-Ing. Andreas Geiger

Autonomous Vision Group

University of Tübingen / MPI-IS

EBERHARD KARLS
UNIVERSITÄT
TÜBINGEN



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European Laboratory for Learning and Intelligent Systems

Agenda

6.1 Introduction

6.2 Black Box Control

6.3 Geometric Control

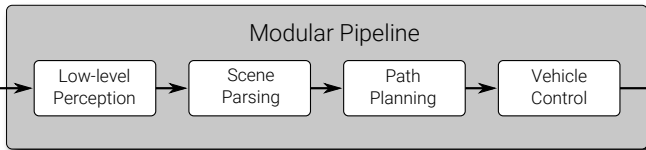
6.4 Optimal Control

6.1

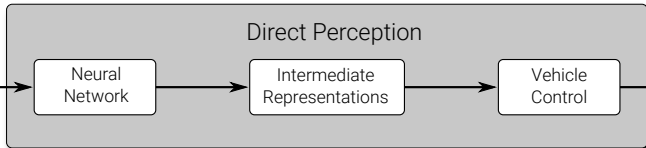
Introduction

Approaches to Self-Driving

Sensory Input

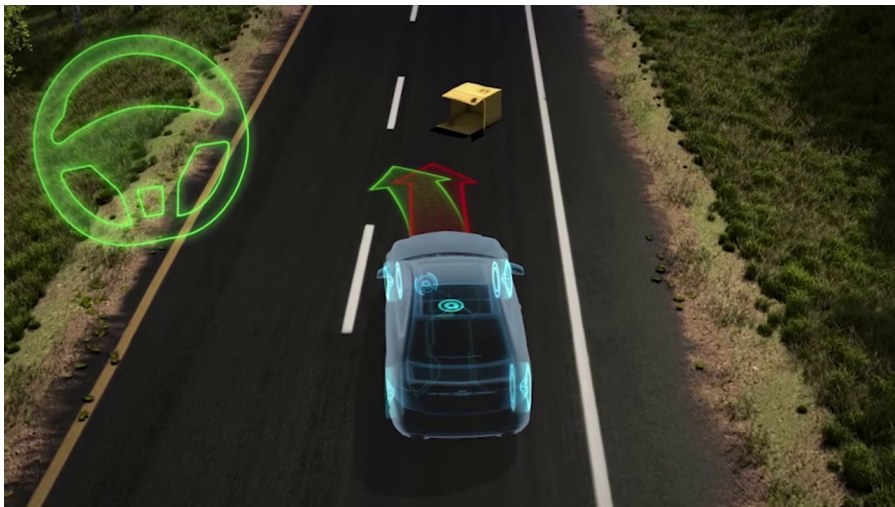


Sensory Input



- Self-driving cars and driver assistance systems require **vehicle control**

Electronic Stability Program

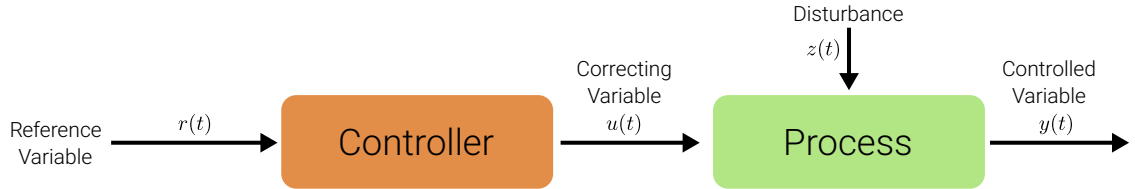


Sophisticated controllers are already in your car today!

Brief History of Driver Assistance Systems

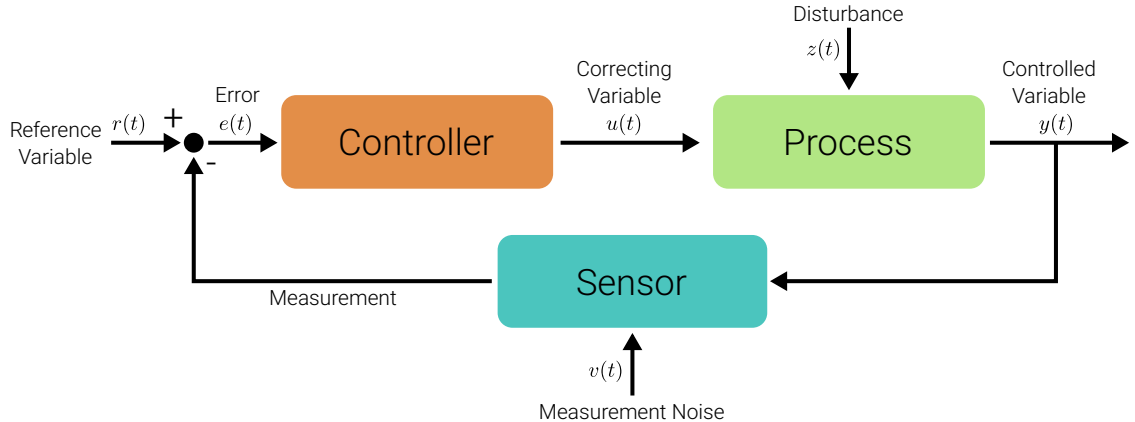
- ▶ 1926: Servo braking (Pierce-Arrow)
- ▶ 1951: Servo steering (Chrysler)
- ▶ 1958: Cruise control (Chrysler)
- ▶ 1978: Anti-lock braking system ABS (Bosch)
- ▶ 1986: Traction control system ASR (Bosch)
- ▶ 1995: Electronic stability program ESP (Bosch/BMW)
- ▶ 2000: Adaptive cruise control ACC (Mitsubishi/Toyota/Bosch)
- ▶ 2002: Emergency brake assistant (Mercedes Benz)
- ▶ 2003: Lane-keeping assistant (Honda)
- ▶ 2007: Automatic park assistant (Valeo)

Open-Loop Control



- ▶ Requires **precise knowledge** of the plant and the influence factors
- ▶ **No feedback** about the controlled variable
- ▶ Cannot handle unknown disturbances, resulting in **drift**

Closed-Loop Control



- Exploit feedback loop to minimize error between reference and measurement

Centrifugal Governor



Closed-Loop Control

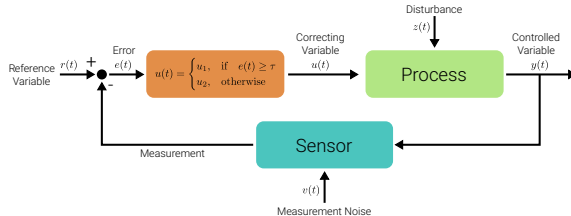
- ▶ We will be considering **closed-loop control** in this lecture
- ▶ A vehicle needs to be controlled both **longitudinally** and **laterally**
- ▶ We consider 3 different types of controllers:
 - ▶ **Black box controllers** don't require knowledge about the process
 - ▶ **Geometric controllers** exploit geometric relationships between the vehicle and the path, resulting in compact control laws for path tracking
 - ▶ **Optimal controllers** use knowledge of the system and minimize an objective function over future time steps

Bang-Bang Control

Bang-Bang Control

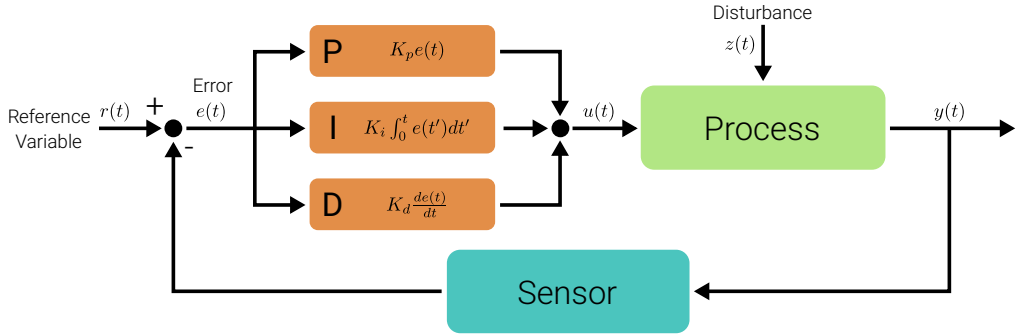
- ▶ Also called: hysteresis controller
- ▶ Often applied, e.g. in household thermostats
- ▶ Switches abruptly between two states
- ▶ Mathematical formulation:

$$u(t) = \begin{cases} u_1, & \text{if } e(t) \geq \tau \\ u_2, & \text{otherwise} \end{cases}$$



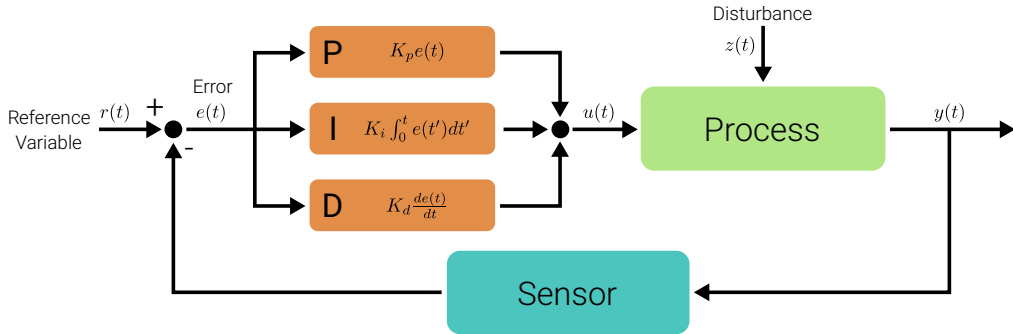
PID Control

PID Control



- ▶ Most common controller in current industrial applications
- ▶ Doesn't require knowledge about plant/process to be controlled
- ▶ Comprises proportional (**P**), integration (**I**) and differential (**D**) element
- ▶ Reference variable $r(t)$, correcting variable $u(t)$, controlled variable $y(t)$, error $e(t)$

PID Control

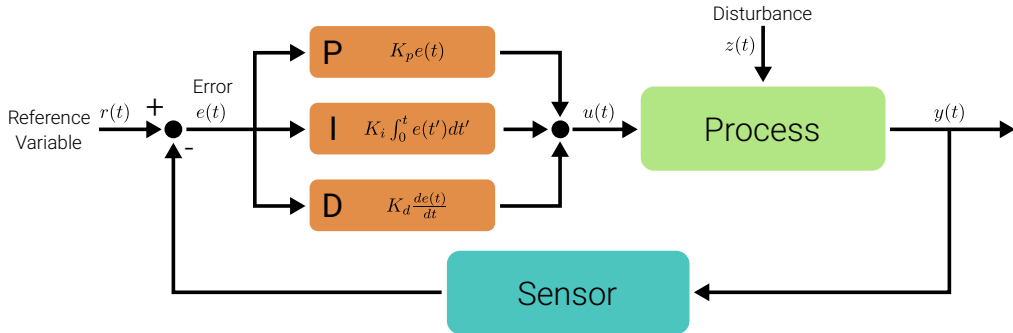


Mathematical formulation:

$$u(t) = K_p e(t) + K_i \int_0^t e(t') dt' + K_d \frac{de(t)}{dt}$$

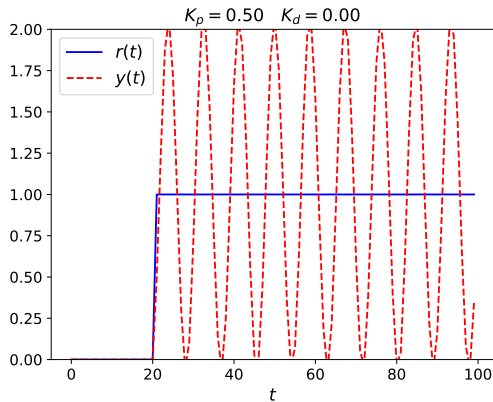
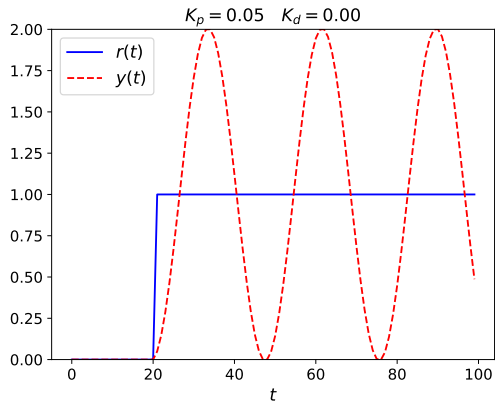
with parameters K_p , K_i and K_d

PID Control



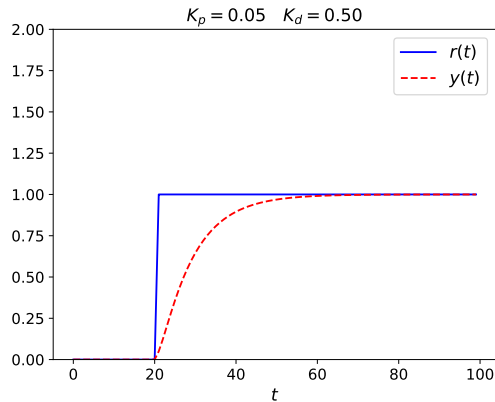
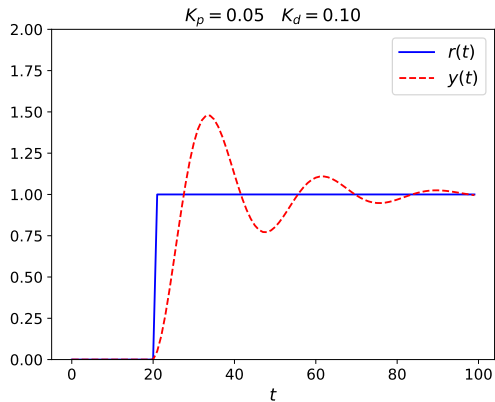
- ▶ Using the **P** element alone leads to overshooting / oscillation
- ▶ Adding a **D** element alleviates this problem by introducing damping behavior
- ▶ The **I** element corrects residual errors by integrating past error measurements

P Control



- Controlled Variable: Position $y(t) = x(t)$
- Correcting Variable: Acceleration $u(t) = a(t) = \ddot{x}(t)$

PD Control



- Controlled Variable: Position $y(t) = x(t)$
- Correcting Variable: Acceleration $u(t) = a(t) = \ddot{x}(t)$

PID Control

As PID controllers are often used to control black box processes, various heuristics have been proposed for setting the parameters. **Ziegler-Nichols** is most common:

- ▶ Set $K_i = K_d = 0$
- ▶ Increase K_p until the ultimate gain $K_p = K_u$ where the system oscillates
- ▶ Measure the oscillation period T_u at K_u
- ▶ Set $K_p = 0.6K_u$, $K_i = 1.2K_u/T_u$ and $K_d = 3K_uT_u/40$

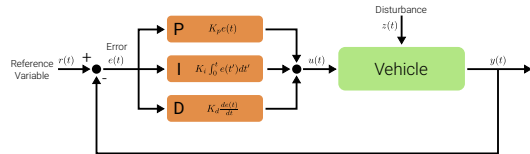
These heuristics have been derived empirically using computer simulations. While they provide a good starting point, manual fine-tuning is often required in practice. Note: Ziegler-Nichols does not apply to double integrators as in the previous example.

PID Control

Example: Longitudinal Vehicle Control

$$v(t) = v_{\max} (1 - \exp(-\theta_1 d(t) - \theta_2))$$

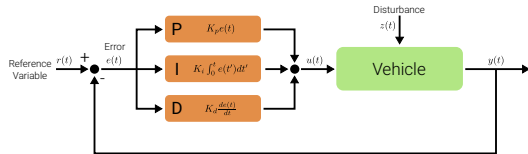
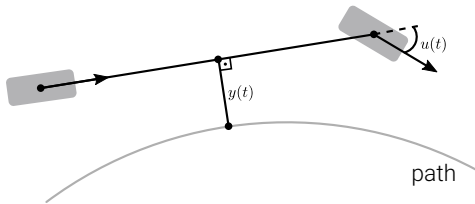
- ▶ $v(t)$: target velocity at time t
- ▶ $d(t)$: distance to preceding car



- ▶ Reference variable: $r(t) = v(t) =$ target velocity
- ▶ Correcting variable: $u(t) =$ gas/brake pedal
- ▶ Controlled variable: $y(t) =$ current velocity
- ▶ Error: $e(t) = v(t) - y(t)$

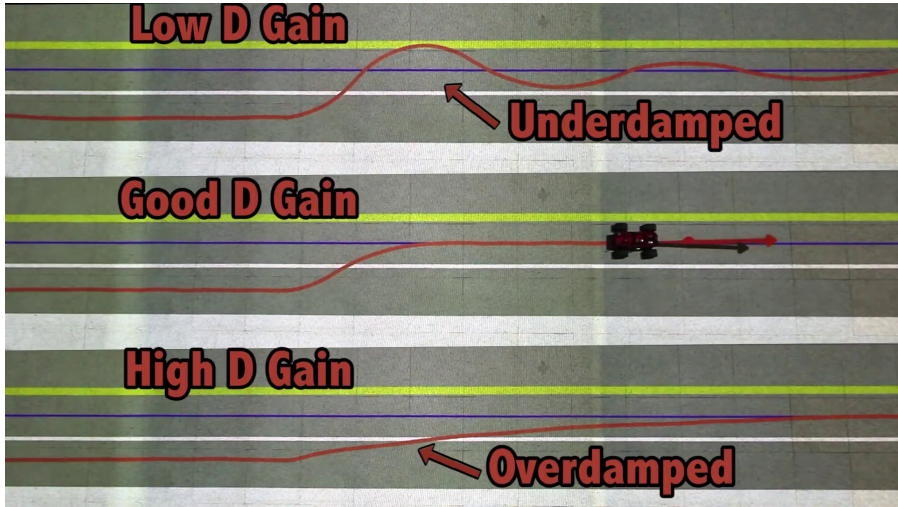
PID Control

Example: Lateral Vehicle Control



- ▶ Reference variable: $r(t) = 0$ = no cross track error
- ▶ Correcting variable: $u(t) = \delta$ = steering angle
- ▶ Controlled variable: $y(t)$ = cross track error
- ▶ Error: $e(t) = -y(t)$ = cross track error

PID Control



PID Control

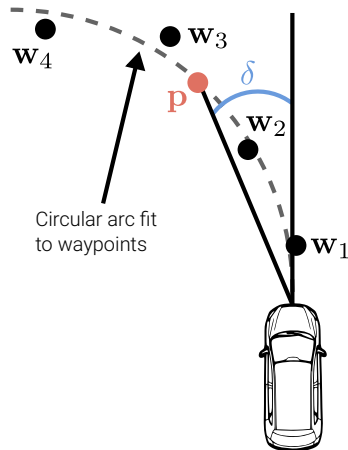
Example: Waypoint-based Vehicle Control

- **Input:** Waypoints $\mathbf{w} = \{\mathbf{w}_1, \dots, \mathbf{w}_K\}$
- **Velocity:** (Longitudinal PID control)

$$v = \frac{1}{K} \sum_{k=1}^K \frac{\|\mathbf{w}_k - \mathbf{w}_{k-1}\|_2}{\Delta t}$$

- **Steering angle:** (Lateral PID control)

$$\delta = \tan^{-1} \left(\frac{p_y}{p_x} \right)$$



PID Control



6.3

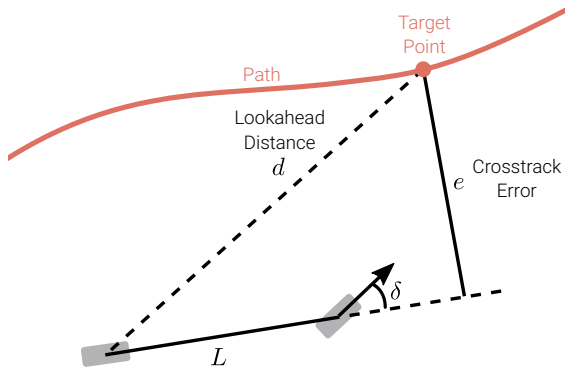
Geometric Control

Pure Pursuit Control

Pure Pursuit Control

Goal:

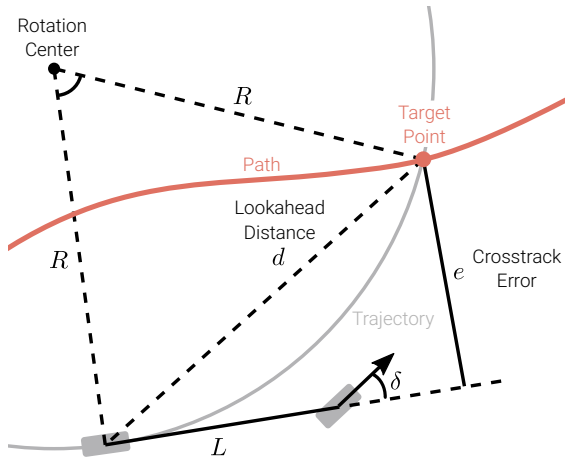
- ▶ Track **target point** at **lookahead distance** d to follow path
- ▶ Exploit geometric relationship between vehicle and path to follow



Pure Pursuit Control

Goal:

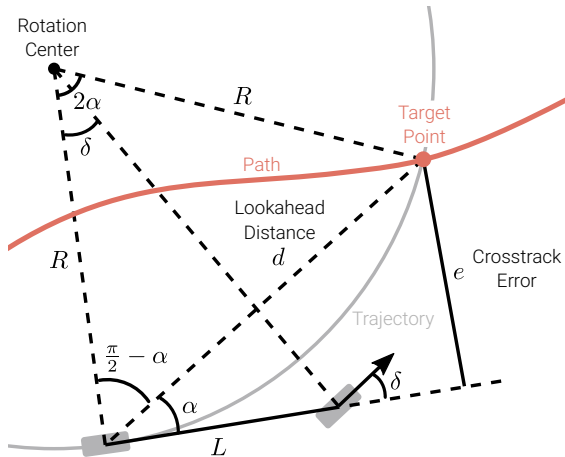
- ▶ Track **target point** at **lookahead distance** d to follow path
- ▶ Exploit geometric relationship between vehicle and path to follow
- ▶ Minimize **crosstrack error** e by following circular trajectory



Pure Pursuit Control

Goal:

- ▶ Track **target point** at **lookahead distance** d to follow path
- ▶ Exploit geometric relationship between vehicle and path to follow
- ▶ Minimize **crosstrack error** e by following circular trajectory
- ▶ Steering angle δ determined by angle α between vehicle heading direction and lookahead direction: $\delta(\alpha) = ?$



Pure Pursuit Control

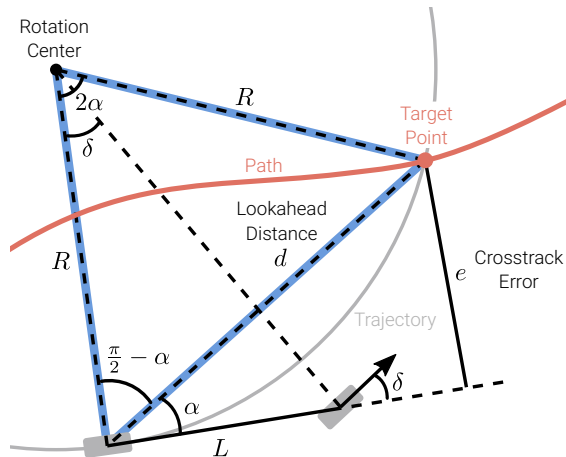
From the **law of sines**:

$$\frac{d}{\sin(2\alpha)} = \frac{R}{\sin\left(\frac{\pi}{2} - \alpha\right)}$$

$$\frac{d}{2 \sin \alpha \cos \alpha} = \frac{R}{\cos \alpha}$$

$$\kappa = \frac{1}{R} = \frac{2 \sin \alpha}{d}$$

with κ the curvature of trajectory.



Pure Pursuit Control

The **steering angle** is calculated as:

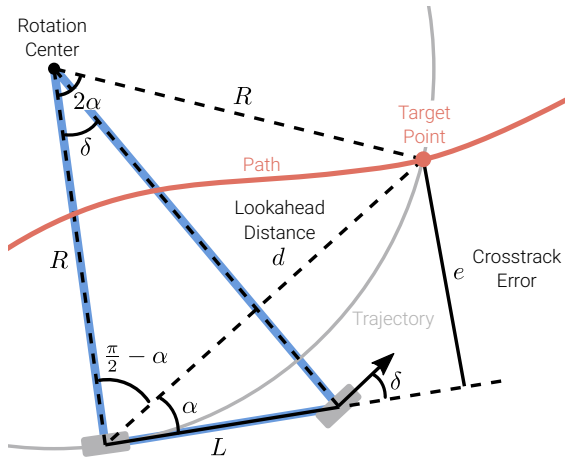
$$\tan(\delta) = \frac{L}{R} = \frac{2L \sin(\alpha)}{d}$$

$$\delta = \tan^{-1} \left(\frac{2L \sin(\alpha)}{d} \right)$$

$$\delta \approx \frac{2L \sin(\alpha)}{d}$$

► d is often based on vehicle speed v :

$$d = Kv \quad \text{with constant } K$$



Pure Pursuit Control

In terms of **cross track error** we obtain:

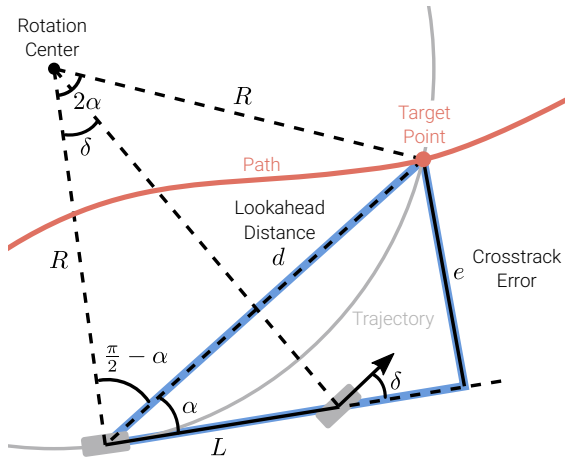
$$\sin \alpha = \frac{e}{d}$$

$$\delta = \tan^{-1} \left(\frac{2L \sin(\alpha)}{d} \right)$$

$$\delta = \tan^{-1} \left(\frac{2Le}{d^2} \right) \approx \frac{2L}{d^2} e$$

- Pure pursuit acts as a **proportional** controller wrt. the crosstrack error
- d is often based on vehicle speed v :

$$d = Kv \quad \text{with constant } K$$



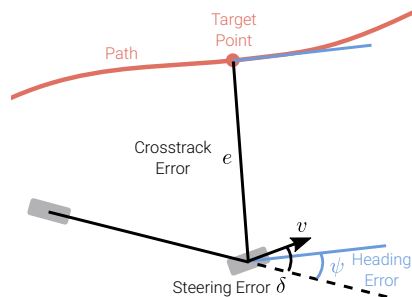
Stanley Control

Stanley Control

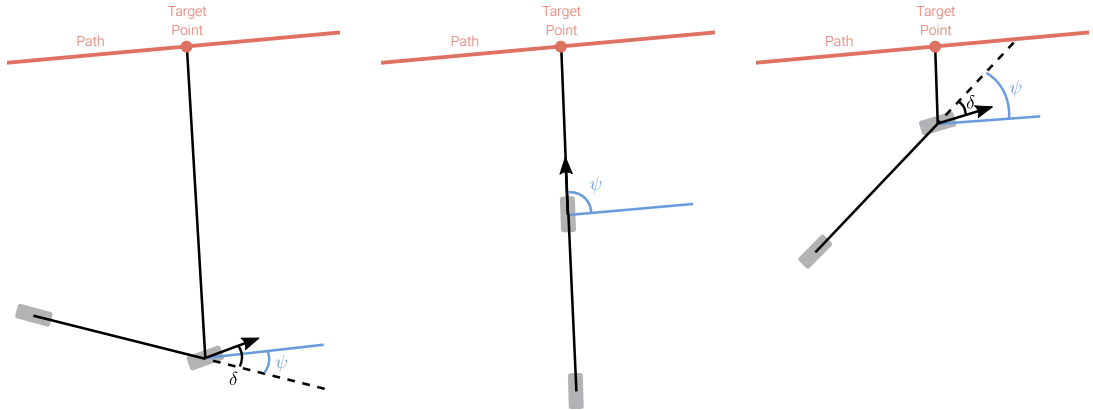
Control Law used by **Stanley** in **DARPA Challenge**:

$$\delta = \psi + \tan^{-1} \left(\frac{k e}{v} \right)$$

- ▶ v = speed, ψ = heading err., e = crosstrack err.
- ▶ Reference at **front axle, no lookahead**
- ▶ Combines **heading** and **crosstrack error**
- ▶ It can be shown that the crosstrack error **converges** exponentially to 0 (indep. of v)
- ▶ Works for small velocities w/o disturbances

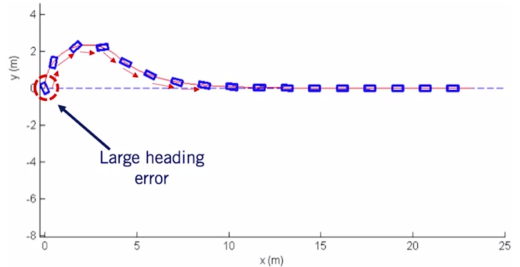
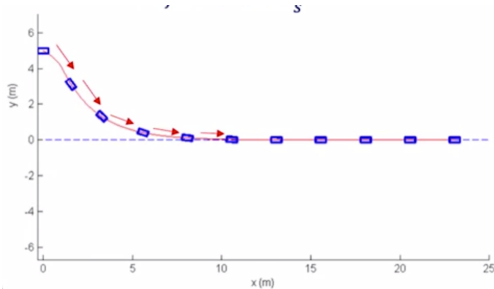


Stanley Control



- As heading changes, heading correction **counteracts** crosstrack correction

Stanley Control



- ▶ The Stanley controller can correct **large crosstrack** and **large heading errors**
- ▶ **Global stability:** Independent of initial conditions guides vehicle back (proven)
- ▶ But does not consider noisy observations, actuator dynamics, tire force effects
- ▶ Softening/dampening terms and curvature information can be added

Stanley Control



6.4

Optimal Control

Optimal Control

Recap: Dynamic Bicycle Model

$$\begin{bmatrix} \dot{v}_y \\ \dot{\psi} \\ \dot{\omega} \end{bmatrix} = \begin{bmatrix} -\frac{c_r+c_f}{mv_x} & 0 & \frac{c_rl_r-c_fl_f}{mv_x} - v_x \\ 0 & 0 & 1 \\ \frac{l_rc_r-l_fc_f}{I_zv_x} & 0 & -\frac{l_f^2c_f+l_r^2c_r}{I_zv_x} \end{bmatrix} \underbrace{\begin{bmatrix} v_y \\ \psi \\ \omega \end{bmatrix}}_{\text{State } \mathbf{x}} + \underbrace{\begin{bmatrix} \frac{c_f}{m} \\ 0 \\ \frac{c_f}{I_z}l_f \end{bmatrix} \delta}_{\text{Input}}$$

With state \mathbf{x} and front steering angle δ .

We can rewrite this equation as the following linear system

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{b}\delta \quad \text{with} \quad \mathbf{x} = (v_y, \psi, \omega)^T$$

Optimal Control

Linear Quadratic Regulator (LQR): For the continuous-time linear system

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{b}\delta \quad \text{with} \quad \mathbf{x}(0) = \mathbf{x}_{\text{init}}$$

and quadratic cost functional defined as (\mathbf{Q} is a diagonal weight matrix)

$$J = \frac{1}{2} \int_0^\infty \mathbf{x}^T(t) \mathbf{Q} \mathbf{x}(t) + q \delta(t)^2 dt$$

the feedback control $\delta(t)$ that minimizes J is given by

$$\delta(t) = -\mathbf{k}^T(t) \mathbf{x}(t)$$

with $\mathbf{k}(t) = \frac{1}{q} \mathbf{b}^T \mathbf{P}(t)$ and $\mathbf{P}(t)$ the solution to a Ricatti equation (no details here).

Model Predictive Control

Model Predictive Control

Generalizes LQR to:

- ▶ **Non-linear** cost function and dynamics (consider straight road leading into turn)
- ▶ **Flexible:** allows for receding window & incorporation of constraints
- ▶ **Expensive:** non-linear optimization required at every iteration

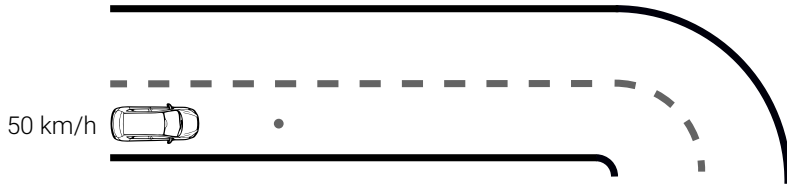
Formally:

$$\begin{array}{lll} \underset{\delta_1, \dots, \delta_T}{\operatorname{argmin}} & \sum_{t=1}^T C_t(\mathbf{x}_t, \delta_t) & \text{(sum of costs)} \\ \text{s.t.} & \mathbf{x}_1 = \mathbf{x}_{\text{init}} & \text{(initialization)} \\ & \mathbf{x}_{t+1} = f(\mathbf{x}_t, \delta_t) & \text{(dynamics model)} \\ & \underline{\delta} \leq \delta_t \leq \bar{\delta} & \text{(constraints)} \end{array}$$

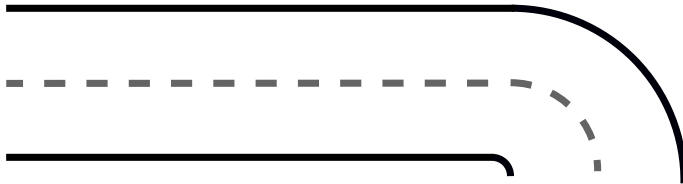
- ▶ Unroll dynamic model T times \Rightarrow apply non-linear optimization to find $\delta_1, \dots, \delta_T$

Model Predictive Control

PID Control / Path Tracking

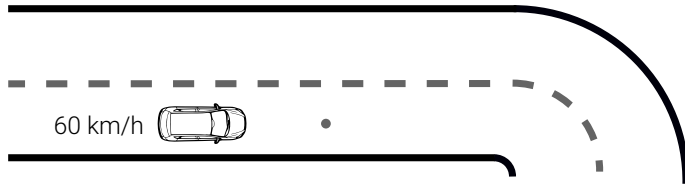


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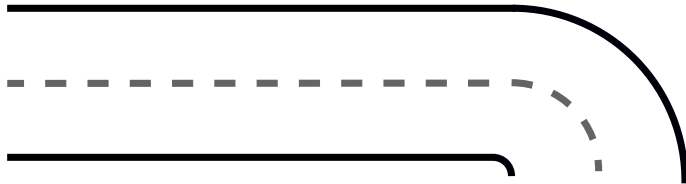


Model Predictive Control

PID Control / Path Tracking

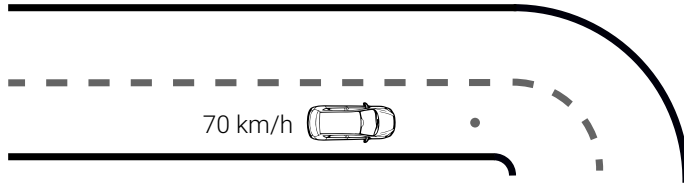


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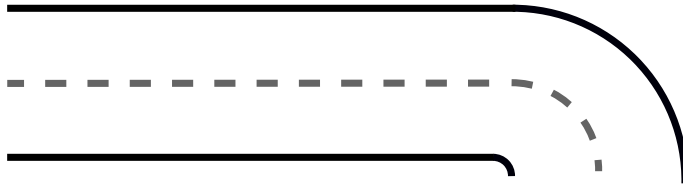


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PID Control / Path Tracking

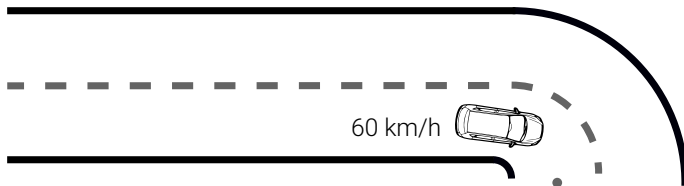


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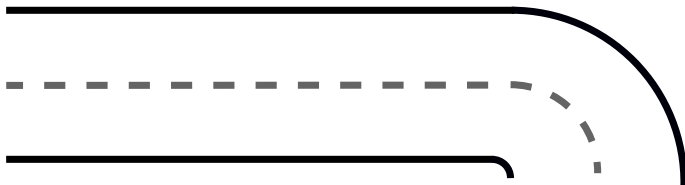


Model Predictive Control

PID Control / Path Tracking

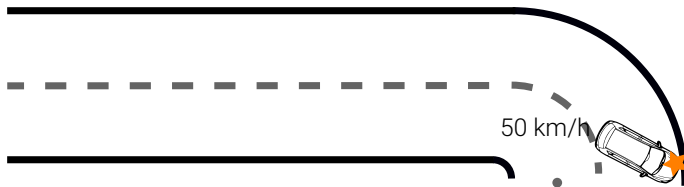


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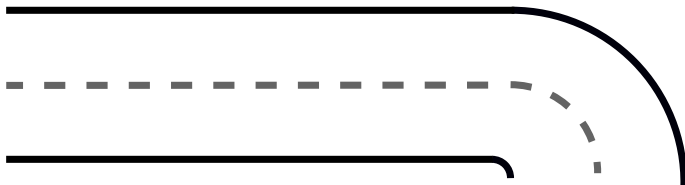


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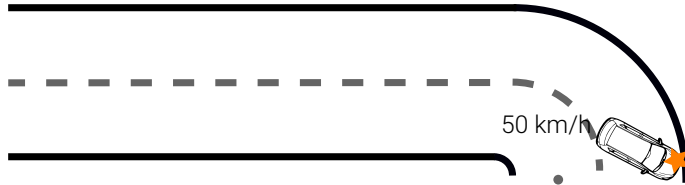


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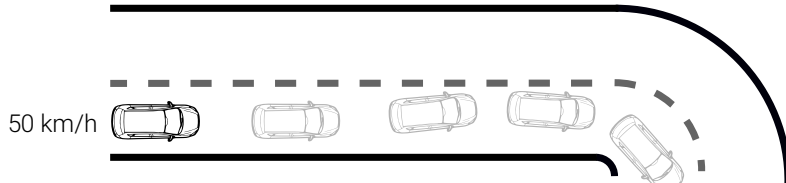


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PID Control / Path Tracking

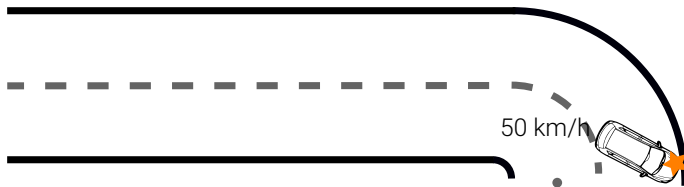


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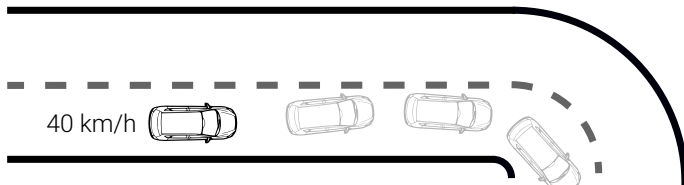


Model Predictive Control

PID Control / Path Tracking

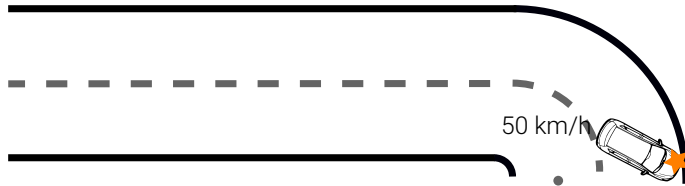


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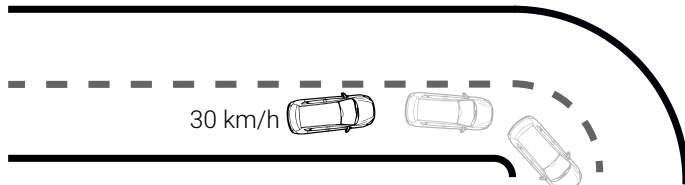


Model Predictive Control

PID Control / Path Tracking

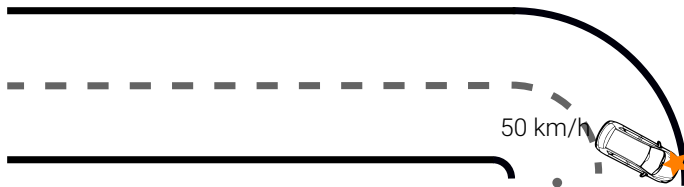


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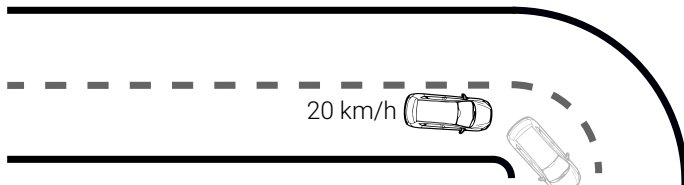


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PID Control / Path Tracking

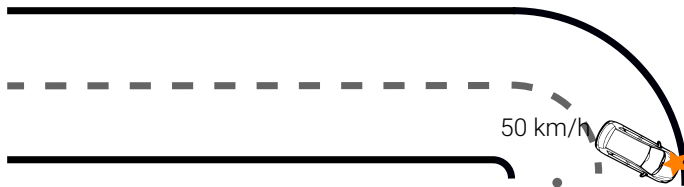


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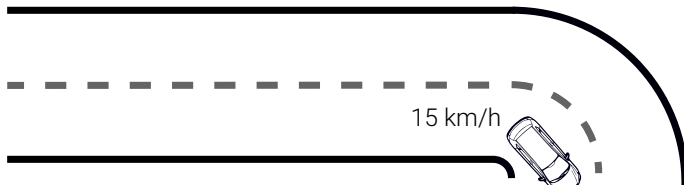


Model Predictive Control

PID Control / Path Tracking



Model Predictive Control



Summary

- ▶ Open-loop controllers cannot handle unknown disturbances
- ▶ In practice, we thus require closed-loop control with sensor feedback
- ▶ Black box controllers don't require knowledge about the process
- ▶ Most popular black box controller: PID controller
- ▶ Geometric controllers exploit geometric relationships for path tracking
- ▶ Optimal controllers use a vehicle model and optimize a cost function
- ▶ MPC is the most flexible and powerful approach
- ▶ However, MPC requires solving an optimization problem at every time step