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Comparison of the Differential Evolution and Simulated Annealing Algorithms Applied to the Constructal Design of Double-T Shaped Cavities

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Abstract. In this paper, it is analyzed the application of two meta-heuristic methods, Differential Evolution and Simulated Annealing, for the geometric optimization in a heat transfer problem. The optimization problem is defined by the Constructal Design method, that determines the objective and constraints in the optimization process, as well as, the degrees of freedom of the geometry that is optimized. The main purpose of this work is the comparison between the results of this two meta-heuristics, mainly the reproduction of the degrees of freedom effect over the optimal geometry, and the thermal performance of the computational domain. The experiment consists in performing thirty runs of each algorithm, with different values for the configuration parameters, and also four versions of Differential Evolution and five versions of Simulated Annealing. The optimization results for four degrees of freedom show that the meta-heuristic algorithm and its parameter configuration can lead to the wrong interpretation of the effect of degrees of freedom over optimal geometry. It is possible to conclude that the algorithm of Differential Evolution with a specific parameter configuration achieved the best and most robust results than the others algorithm. Therefore, the significant contribution here is the recommendation of the more reliable meta-heuristic, and its correct parameters for the heat transfer problem considered.

Keywords: Heat Transfer, Constructal Design, Simulated Annealing, Differential Evolution, Geometric Optimization

1. INTRODUCTION

The geometric optimization of cooling cavities inside a solid with heat generation was first proposed by Biserni *et al.* (2004). Through by the Constructal Design (CD) method, the elementary geometry of C-shaped and T-shaped cavities was evaluated. The CD method is based on the Constructal Theory. According to its author, (Bejan, 1997), there is a physical principle that defines the geometries in flow systems, and this principle determines the shapes and structures that emerge in nature. The Constructal Law explains that for a finite-size flow system to persist in time (to survive) its configuration must evolve in such a way that it provides easier access to the currents that flow through it (Bejan, 1997). The CD method is not an optimization method, and determines the degrees of freedom which varies in the geometric optimization process and depends on the constraints to which this optimization is submitted to. The optimization is performed by an optimization method, e.g., Exhaustive Search (ES).

Employing the CD method to determine the objectives and constraints of the optimization process, more complex cavities have been evaluated (Biserni *et al.*, 2007). Comparing the results between different shapes of cavities, kept the same constraints, it is noted that a more complex cavity geometry leads to the gain in the thermal performance, as can be seen in the complex and multi-cavity studies (Lorenzini *et al.*, 2012; Xie *et al.*, 2010). However, complex cavities need more degrees of freedom and require more computational effort in the optimization process. Therefore, for complex cavities, meta-heuristic methods are used as alternative methods to ES. Lorenzini *et al.* (2014) use Genetic Algorithm (GA) to evaluate the more complex problem in the Y-shaped cavity optimization including a convective parameter. The GA also was performed with CD to the optimization and analyses of morphing fins coupled with a heat generating body in Biserni *et al.* (2017). Gonzales *et al.* (2015b) analyses the performance of different parameters of Simulated Annealing (SA) algorithm in the geometric optimization of a Y-shaped cavity. The work of Gonzales *et al.* (2015b) compares various

parameters of the Cooling Schedule (CS) and proposes a hybrid parameter that showed best results than GA for a Y-shaped cavity. In the paper of Gonzales *et al.* (2017), was performed a comparison between the Luus-Jaakola and SA algorithms with hybrid parameters of CS applied in the Double-T shaped cavity optimization. The results showed the best performance of the SA algorithm configured with the hybrid parameter of Cooling Schedule.

In the present work, the meta-heuristic of SA is compared with the Differential Evolution (DE) algorithm, applied jointly with CD, to the geometric optimization of the Double-T shaped cavity. The Double-T Shaped cavity was first proposed by Gonzales *et al.* (2015a), and the SA was used in the geometric optimization. The two algorithms are performed with distinct parameters, and then the comparison is made between the variations of SA and DE. The best versions for each meta-heuristic are then compared. The SA variations differ in the Cooling Schedule (CS) parameter, and five distinct CS are performed. Different versions of the DE algorithm are compared too. The DE variations differ in the crossover and mutation operators, and four DE algorithms are performed. The double-T shaped cavity has five degrees of freedom (DOFs) that define the cavity geometry $(H/L, H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0 . Four and five DOFs are optimized, and each algorithm achieves the curve of the effect of DOF over optimal geometry and thermal performance. The results for each algorithm are registered in a database and analyzed. Therefore, the comparison between the results of the algorithms is performed through the effect of DOF over optimal geometry, not just the minimum temperature. Then, the better algorithm is the one that reproduces this effect with more reliable results.

2. MATHEMATICAL AND NUMERICAL MODEL

Figure 1 shows the heat transfer problem of interest, that consists in a conducting body in the two-dimensional configuration, with the third dimension, with length W, perpendicular to the plane of the figure. The solid domain has a constant and uniform internal heat generation with the volumetric rate given by $q'''(Wm^{-3})$. The solid has a constant thermal conductivity k. The outer surfaces of the solid are perfectly insulated, corresponding to adiabatic conditions. In this case, the heat can only be removed through the double-T shaped cavity, which is kept at a minimum temperature (θ_{min}) . The minimal temperature of the cavity may be kept with the flow of refrigerant fluid through the cavity, changing phase at a low temperature. For the sake of simplicity, the heat transfer coefficient on the cavity wall is assumed to be sufficiently large so that the convective resistance can be neglected in comparison to the solid conduction resistance.

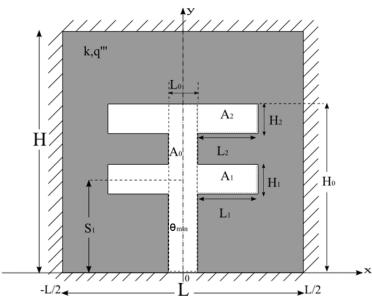


Figure 1: Computational domain of Double-T shaped cavity.

The objective of the analysis is to determine the optimal geometry $(H/L, H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0) that is characterized by the minimum global thermal resistance $(\theta_{max} - \theta_{min})/(q'''A)$. According to the CD this optimization can be subjected to the constraints of the total area and cavity area, represented respectively by:

$$A = HL \tag{1}$$

$$A_c = A_0 + 2A_1 + 2A_2 \tag{2}$$

The fraction of the cavity area with respect to the total area is given by:

$$\phi_c = A_c/A \tag{3}$$

For the determination of the temperature field in the solid domain, it is necessary to solve the heat conduction equation given by:

$$\frac{\partial^2 \theta}{\partial \tilde{x}^2} + \frac{\partial^2 \theta}{\partial \tilde{y}^2} + 1 = 0 \tag{4}$$

For the sake of brevity, the equations with dimensionless variables, the equations of boundary conditions of null flux in the solid outer surfaces, as well as, the equations of boundary conditions of minimal temperature in the cavity wall can be seen in the study by Gonzales *et al.* (2015a), and are not reproduced here. The aim is to minimize the maximal excess temperature represented by the following equation:

$$\tilde{\theta}_{max} = \frac{\theta_{max} - \theta_{min}}{q''' \cdot \frac{A}{k}} \tag{5}$$

The determination of $\tilde{\theta}_{max}$ is needed to optimize the five degrees of freedom $(H/L, H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0) submitted to the corresponding constraints of the cavity area (ϕ_c, ϕ_1) and ϕ_2 and the total solid area. In this paper are optimized the five DOFs $(H/L, H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0 . The function represented by Eq. (5) is determined numerically by solving Eq. (4) for the temperature field in every assumed configuration $(H/L, H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0 , and calculating $\tilde{\theta}_{max}$ to see whether its value may be minimized by varying the configuration. The numerical solution is performed with the Finite Element Method (FEM)(Reddy and Gartling, 1994), based on linear triangular elements, available in the The MATLAB® environment, with the PDE (partial-differential-equations) toolbox. The grid was non-uniform in both x and y directions, and varied from one geometry to the next. The appropriate mesh size was determined by successive refinements (h-adaptively), increasing the number of elements four times from one current mesh size to the next one. For the sake of brevity, the grid independence test can be seen in Gonzales *et al.* (2015a), and, therefore, it is note repeated here.

2.1 GEOMETRIC OPTIMIZATION

The CD method is employed to determine the values for the objectives and constraints chosen, as well as, the search space and the degrees of freedom (DOFs). With the optimization problem defined, the search for the optimal geometry was performed with two optimization algorithms, SA and DE. The parameters of these two heuristic methods applied are also investigated. Therefore, the results are compared in order to indicate the best method for the case studies in this work. For the SA algorithm, five different functions of Cooling Schedules (CS) are tested, the traditional schedules Boltz and Exponential present in the MATLAB® optimization toolbox, and the hybrid CS (BoltzExp, ConstExp1, and ConstExp2) proposed by Gonzales *et al.* (2015a,b). The DE algorithm is also varied in four versions with different parameters for the crossover and mutation operators.

The geometric optimization of double-t shaped cavities is realized by the variation of the parameters that define the geometry cavity and, according to the CD method, this process must be submitted to constraints. However, the cavity studied here has nine variables $(H, L, H_0, L_0, H_1, L_1, H_2, L_2 \text{ and } S1)$ and four constraints $(A, \phi_c, \phi_1 \text{ and } \phi_2)$. Then, five degrees of freedom are needed to complete the equations and allow the determination of the geometry $(H/L, H_0/L_0, H_1/L_1, H_2/L_2 \text{ and } S_1/H_0)$. The optimization process is concentrated in four and five DOFs. Firstly, the four DOFs optimization is performed by the optimization of the three DOFs $(H_1/L_1, H_2/L_2 \text{ and } S_1/H_0)$ for ten different values of H_0/L_0 , keeping fixed H/L = 1 and the four study constraints are kept as A = 1, $\phi_c = 0.1$; $\phi_1 = 0.015$; and $\phi_2 = 0.015$. The five DOFs optimization is performed by the optimization of four DOFs $(H_0/L_0, H_1/L_1, H_2/L_2)$ and $(H_0/L_0, H_1/L_1, H_2/L_2)$ and (H

3. RESULTS

The results of the four DOFs optimization process performed by thirty runs of each algorithm, with two versions of SA and three versions of DE algorithm, were performed and saved in a database. All the algorithms conduct to the optimal geometry and maximum excess of temperature three times minimized. However, because of the stochastic nature, the algorithms do not produce the same results in all thirty rounds, and the average value is obtained to construct the graphics shown in Figure 2, allowing to observe the algorithm version that is closer to the correct result.

4. CONCLUSIONS

The results show that the versions of the DE algorithms represent the results that come closest to the right effect. Morover, the version DE1 has the most robust results with minor average and more approximated results to the curve

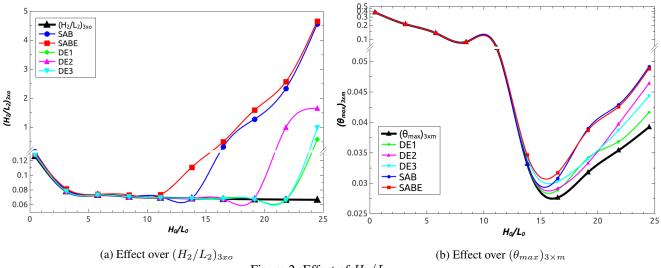


Figure 2: Effect of H_0/L_0

of the effect of H_0/L_0 over $(H_2/L_2)_{3xo}$ optimal, and to the curve of the maximum excess of temperature three times minimized $(\theta_{max})_{3\times m}$.

5. ACKNOWLEDGEMENTS

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