

thermal performance and definition of system design. Results also indicated that one of the Differential evolution algorithms led to the best and most robust performance. Therefore, the significant contribution here is the recommendation of the more reliable meta-heuristic, and its correct parameters for the studied problem.

Keywords: Heat Transfer, Constructal Design, Simulated Annealing, Differential Evolution, Geometric Optimization

INTRODUCTION

The geometric optimization of cooling cavities inside a solid with heat generation using Constructal Design for geometric evaluation was first proposed by Biserni et al. (2004). In this

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study, C and T-shaped cavities were investigated. Constructal Design is a method used to prove that the design of all flow systems of finite dimensions can be predicted by a physical principle named Constructal Law (Bejan, 1997) This principle determines the shapes and structures that emerge in nature. The Constructal Law explains that for a finite-size flow system to persist in time (to survive) its configuration must evolve in such a way that it provides easier access to the currents that flow through it (Bejan, 1997). The CD method is not an optimization method but a method for geometrical evaluation based on performance indicators and constraints (physical, geometrical). More precisely, the method has been used to definition of search space, which must be investigated with an optimization method, e.g., Exhaustive Search (ES). In this method, all geometric possibilities are evaluated considering an increment of variation of the geometric parameters. However, its employment becomes prohibitive for complex shapes (with several degrees of freedom) due to high computational effort (Gonzales et al., 2015b).

After the pioneer work of Biserni et al. (2004), more complex shapes were investigated using CD associated with ES. For instance, Biserni et al. (2007) studied H-shaped cavities and Lorenzini et al. (2011) investigated Y-shaped ones. The comparison between different shapes of cavities, kepting the same constraints, has indicated that as more complex is the cavity shape the best is the thermal performance of the system (for systems with high intensity). Examples of this behavior can be seen in recent studies of complex and several cavities (Xie et al., 2010; Lorenzini et al., 2012, 2014). However, complex cavities need more degrees of freedom and require more computational effort in the optimization process. Therefore, for complex cavities, meta-heuristic methods have been used as alternative for geometrical evaluation, allowing the study of many degrees of freedom. Lorenzini et al. (2014) used Genetic Algorithm (GA) to optimize a complex Y-shaped cavity considering the effect of a convective parameter over the problem design. The GA was also associated with CD for geometric optimization of morphing fins coupled with a trapezoidal heat generating body in the study of Biserni et al. (2017). Gonzales et al. (2015b) analyzed the performance of different parameters of Simulated Annealing (SA) algorithm in the geometric optimization of a Y-shaped cavity. The work of Gonzales et al. (2015b) compared various parameters of the Cooling Schedule (CS) for SA algorithm. It was noticed that hybrid parameters led to the best thermal performances and improve the representation of the geometric parameters effect over thermal performance of the studied problem. Recently, Gonzales et al. (2017) performed a comparison between the Luus-Jaakola and SA algorithms with hybrid parameters of CS applied in the Double-T shaped cavity optimization. Results showed that the best performance was reached with the SA algorithm with hybrid parameter for Cooling Schedule.

In the present work, the meta-heuristic of SA is compared with the Differential Evolution (DE) algorithm to the geometric optimization of the Double-T shaped cavity. The main purpose of the system is to minimize the maximal excess of temperature in the solid domain. The Double-T Shaped cavity was first proposed by Gonzales *et al.* (2015a), and the SA was used in the geometric optimization. The two algorithms are performed with distinct parameters, and then the comparison is made between the variations of SA and DE. The best versions for each meta-heuristic are then compared. The SA variations differ in the Cooling Schedule (CS) parameter, and five distinct CS are investigated. Different versions of the DE algorithm are also compared. The DE variations differ in the crossover and mutation operators, and four DE algorithms are performed. The double-T shaped cavity has five degrees of freedom (DOFs) that define the cavity geometry $(H/L, H_0/L_0, H_1/L_1, H_2/L_2 \text{ and } S_1/H_0)$. Four and five DOFs are optimized, and each algorithm achieves the curve of the effect of DOF over optimal geometry and thermal performance. The results for each algorithm are registered in a database and an-

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alyzed. Therefore, the comparison between the results of the algorithms is performed through the effect of DOF over optimal geometry, not only using the minimal maximum excess of temperature. Then, it is evaluated here the algorithm that is more able to reproduce the effect of the degrees of freedom over the performance of the system.

2. MATHEMATICAL AND NUMERICAL MODELS



Figure 1 shows the heat transfer problem of interest, that consists in a conducting body in the two-dimensional configuration, with the third dimension, with length W, perpendicular to the plane of the figure. The solid domain has a constant and uniform internal heat generation with the volumetric rate given by $q'''(Wm^{-3})$. The solid has a constant thermal conductivity k. The outer surfaces of the solid are perfectly insulated, corresponding to adiabatic conditions. In this case, the heat can only be removed through the Double-T shaped cavity, which is kept at a minimum temperature (θ_{min}). The minimal temperature of the cavity may be kept with the flow of refrigerant fluid through the cavity, changing phase at a low temperature. For the sake of simplicity, the heat transfer coefficient on the cavity wall is assumed to be sufficiently large so that the convective resistance can be neglected in comparison to the solid conduction resistance.

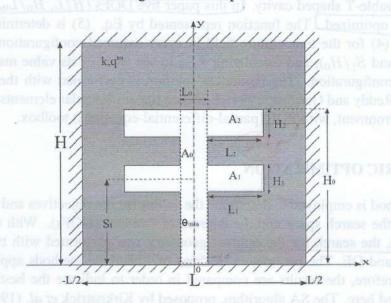


Figure 1- Computational domain of Double-T shaped cavity.

The objective of the analysis is to determine the optimal geometry $(H/L, H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0) that is characterized by the minimum global thermal resistance $(\theta_{max} - \theta_{min})/(q'''A)$. According to the CD the geometric evaluation can be subjected to total and cavity areas constraints, represented respectively by:

$$A = HL \tag{1}$$

$$A_c = A_0 + 2A_1 + 2A_2 \tag{2}$$

The fraction of the cavity area with respect to the total area is given by:

$$\phi_c = A_c/A \tag{3}$$

For the determination of the temperature field in the solid domain, it is necessary to solve the heat conduction equation given by:

$$rac{\partial^2 heta}{\partial ilde{x}^2} + rac{\partial^2 heta}{\partial ilde{y}^2} + 1 = 0$$
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For the sake of brevity, the equations with dimensionless variables and the equations of boundary conditions of minimum temperature are not reproduced here. More details can be seen in the work of Gonzales *et al.* (2015a). The aim is to minimize the maximal excess temperature represented by the following equation:

$$\tilde{\theta}_{max} = \frac{\theta_{max} - \theta_{min}}{q''' \cdot \frac{A}{k}}$$
 $(j: \ell. \quad \phi_1 = \frac{2A_1}{A} \quad omd \quad \phi_2 = \frac{2A_2}{A}.$

The determination of $\tilde{\theta}_{max}$ is needed to optimize the five degrees of freedom $(H/L, H_0/L_0, H_1/L_1, H_2/L_2 \text{ and } S_1/H_0)$ submitted to the corresponding constraints of the cavity area $(\phi_c, \phi_1 \text{ and } \phi_2)$ and the total solid area. Where ϕ_1 and ϕ_2 represent the dimensionless area of each branch of the Double-T shaped cavity. In this paper five DOFs $(H/L, H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0 are optimized. The function represented by Eq. (5) is determined numerically by solving Eq. (4) for the temperature field in every assumed configuration $(H/L, H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0 , and calculating $\tilde{\theta}_{max}$ to see whether its value may be minimized by varying the configuration. The numerical solution is performed with the Finite Element Method (FEM)(Reddy and Gartling, 1994), based on linear triangular elements, available in the MATLAB® environment, with PDE (partial-differential-equations) toolbox.

3. GEOMETRIC OPTIMIZATION

The CD method is employed to determine the values for the objectives and constraints chosen, as well as, the search space and the degrees of freedom (DOFs). With the optimization problem defined, the search for the optimal geometry was performed with two optimization algorithms, SA and DE. The parameters of these two heuristic methods applied are also investigated. Therefore, the results are compared in order to indicate the best method for the problem evaluated here. The SA algorithm, proposed by Kirkpatrick et al. (1983), has as main parameter the function that controls the temperature decay, named Cooling Schedule. Five different functions of Cooling Schedules (CS) are tested in this study, the traditional schedules Boltz and Exponential, implemented in the MATLAB® optimization toolbox, and named here respectively as SAEX and SABO. Also hybrid CS are investigates (BoltzExp, ConstExp1, and ConstExp2) proposed by Gonzales et al. (2015a,b), named respectively as SABE, SAC1 and SAC2. The DE is an evolutionary-based algorithm proposed by Storn and Price (1997) for continuous spaces. In this work, the DE algorithm is also varied in four versions, two versions with basic DE strategy for mutation operator named DE/rand/1/bin and two version with the DE/best/2/bin variant of mutation operator (Storn and Price, 1997). The algorithm named DE1 and DE2 are variant of basic DE strategy, and the versions named here as DE3 and DE4 have the approach of DE/best/2/bin. The DE heuristic have more two parameters varied in this paper, the Crossover (CR) rate and the factor F, which controls the amplification of differential variation. The DE1 and DE4 have the factor F = 1.5 and CR = 0.7, while the DE2 and DE3 versions factor, F = 2 and CR = 0.9.

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The geometric optimization of Double-T shaped cavity is realized by the variation of the parameters that define its geometry and, according to the CD method, this process must be sub- L_2 and S1) and four constraints $(A, \phi_c, \phi_1 \text{ and } \phi_2)$. Then, five degrees of freedom are required to closure the equation system which defines the geometry $(H/L, H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0). The optimization process is concentrated in four and five DOFs since these conditions represent the most critical situation of evaluation. Firstly, the four DOFs optimization is performed by the optimization of the three DOFs (H_1/L_1 , H_2/L_2 and S_1/H_0) for ten different values of H_0/L_0 , keeping fixed H/L = L and the four study constraints are kept as A = 1, $\phi_c =$ 0.1; $\phi_1 = 0.015$; and $\phi_2 = 0.015$. The five DOFs optimization is performed by the optimization of four DOFs $(H_0/L_0, H_1/L_1, H_2/L_2)$ and S_1/H_0 for nineteen different values of H/L. The constraints are kept with the same values used in the four DOFs optimization. Therefore, at the end of the four DOFs optimization process, each algorithm conducts to results of the effect of the ratio H_0/L_0 over the four times minimized maximal excess of temperature, $(\theta_{max})_{b \neq m}$, and their respective optimal shapes. The curve of effect of the DOF H/L over optimal geometry, is also obtained in five DOFs optimization process. More details about the definition of geometric variables as function of restrictions and degrees of freedom can be seen in Gonzales et al. (2015a).

RESULTS AND DISCUSSIONS

The results of the four DOFs optimization process performed by thirty runs of each algorithm, with different versions of SA and DE algorithms, were achieved and saved in a database. The algorithms have the number of iterations limited to 150. All the algorithms conduct to the optimal geometry and maximum excess of temperature three times minimized. However, because of the stochastic nature, the algorithms do not produce the same results in all thirty founds, and the average value is obtained to construct the results seen in Fig. 2, allowing to observe the algorithm version that is closer to the benchmark solution. The benchmark solution is the composition of the best results achieved between all algorithm and contain the minimum values of $(\theta_{max})_{3\times m}$, $(\theta_{max})_{4\times m}$ and its optimal geometries.

In the Figure 2 (a), it is possible to observe the effect of H_0/L_0 over $(\theta_{max})_{3\times m}$ obtained with all algorithms. It can be observed that all methods led to similar results for $H_0/L_0 = 10$. For ratios $H_0/L_0 > 10$ same differences are noticed. The results reached with DE algorithms have the best agreement with the benchmark solution, mainly the DE1 version. In spite of some differences, results reached with this version properly reproduced the effect of H_0/L_0 over $(\theta_{max})_{3\times m}$. The SA versions with the cooling schedule functions Exponential (SAEX), Boltzmann (SABO) and hybrid BoltExp (SABE) led to the worst performance for reproduction of the effect of H_0/L_0 over $(\theta_{max})_{3\times m}$.

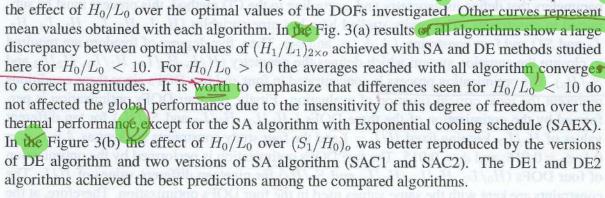
Figure 2(b) shows the results of each algorithm analysed in this paper for the effect of H_0/L_0 over H_2/L_2 three times optimized, $(H_2/L_2)_{3\times o}$. A similar tendency observed in the effect of of H_0/L_0 over $(\theta_{max})_{3\times m}$ is reached for the effect of H_0/L_0 over the $(H_2/L_2)_{3\times n}$. The best agreements where found for $H_0/L_0 \ll 10$, with exception of SA Exponential where poor agreement is found in all range of H_0/L_0 . For $H_0/L_0 > 10$ the best representation of the effect. of H_0/L_0 over $(H_2/L_2)_{3\times 0}$ is obtained with DE1 and DE3. Some discrepancies are seen for the highest magnitudes of H_0/L_0 , probably by the achievement of some local optimal shapes.

The effect of H_0/L_0 over $(H_1/L_1)_{2\times o}$ and $(S_1/H_0)_o$ is showed in the Fig. 3(a) and Fig.

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3(b), respectively. The curve represented by the black line in the Fig/3 (a) and (b) reproduce mean values obtained with each algorithm. In the Fig. 3(a) results of all algorithms show a large to correct magnitudes. It is worth to emphasize that differences seen for H_0/L_0 < 10 do not affected the global performance due to the insensitivity of this degree of freedom over the thermal performance, except for the SA algorithm with Exponential cooling schedule (SAEX). of DE algorithm and two versions of SA algorithm (SAC1 and SAC2). The DE1 and DE2



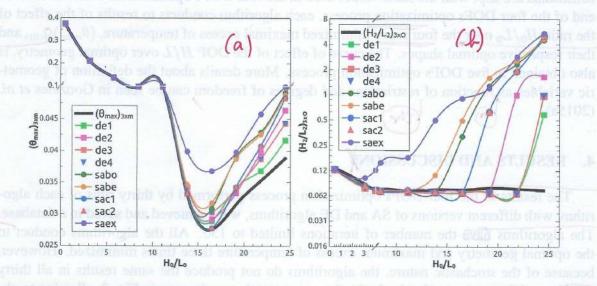


Figure 2- Effect of H_0/L_0 over $(\theta_{max})_{3\times m}$ and $(H_2/L_2)_{3\times o}$ obtained with different DE and SA algorithms over: a) over $(\theta_{max})_{3\times m}$. b) over $(H_2/L_2)_{3\times o}$

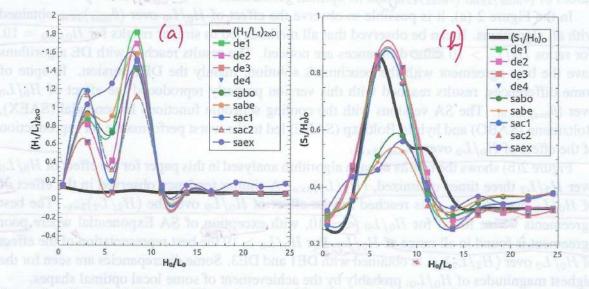


Figure 3- Effect of H_0/L_0 over $(H_1/L_1)_{2\times o}$ and $(S_1/H_0)_o$ obtained with different DE and SA algorithms over: a) $(H_1/L_1)_{2\times o}$, b) $(S_1/H_0)_o$

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For the evaluation of 5 DOFs, the number of iterations was changed to 300 in order to improve the seek for the effect of H/L over the thermal performance and optimal shapes. For each ratio of H/L 30 runs of each algorithm were performed and the maximum excess of temperature four times minimized are recorded for different magnitudes of H/L studied here. Then, it is possible to reproduce the effect of H/L over $(\theta_{max})_{4\times m}$ and respective optimal shapes. Statistical measures were also stored in a database, as well as, the minimal values of the maximum excess of temperature four times minimized and its respective optimal geometries.

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Figure 4 shows the minimal values of the maximum excess of temperature four times minimized represented by the black line, and other curves represent the average of this value reached for each algorithm in the geometric optimization process of five DOFs $(H/L, H_0/L_0, H_1/L_1, H_2/L_2 \text{ and } S_1/H_0)$. Results of Fig. 4(a) and 4(b) show that the effect of H/L over $(\theta_{max})_{4\times m}$ is better reproduced with DE algorithm versions than SA ones. For values of H/L > 0.5 all algorithms can reproduce the effect of H/L of $(\theta_{max})_{4\times m}$ with precision. The same behavior was observed for degree of freedom H/L over H/L o

Figure 5 shows the comparison between the results of the versions of DE algorithm and variants of the SA algorithm to predict effect of H/L over $(H_2/L_2)_{3\times o}$. In this case, variations of DE algorithm have a good agreement with benchmark solution, with exception of DE2 version which showed some oscillations for H/L < 0.5. The results of the SA algorithm, Fig. 5(b), show some differences for H/L < 5.0 in spite of representation of a similar tendency. For the lowest values of ratio H/L, none of SA algorithms reproduced the magnitudes of $(H_2/L_2)_{3\times o}$. For values of H/L > 5 the versions of SA algorithm SAC1 and SAC2 achieved the most similar magnitudes in comparison with $(H_2/L_2)_{3\times o}$. The difficulty of the versions of SA algorithm to reach the optimal configuration of H_2/L_2 for the lowest magnitudes of H/L explain the differences found for $(\theta_{max})_{4\times m}$ showed in the Fig.4(b).

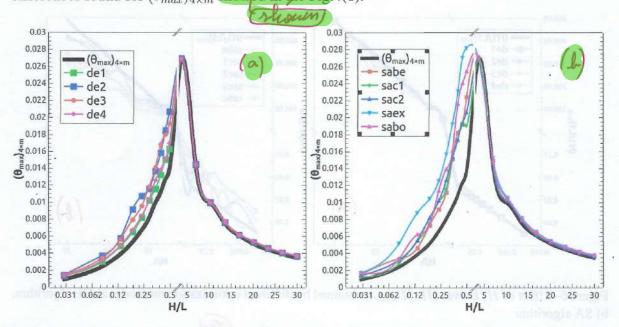
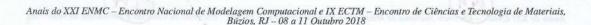


Figure 4- Effect of H/L over $(\theta_{max})_{4\times m}$ obtained by different optimization methods: a) DE algorithm, b) SA algorithm













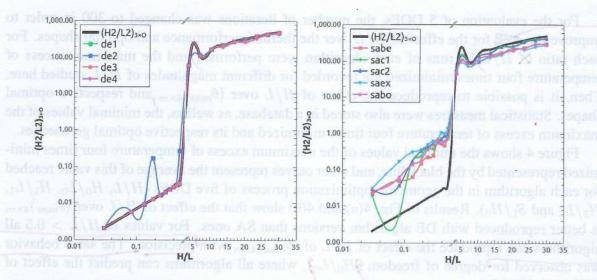


Figure 5- Effect of H/L over $(H_2/L_2)_{3\times o}$ obtained by different optimization methods: a) DE algorithm, b) SA algorithm

The effect of H/L over $(H_1/L_1)_{2\times o}$ is shown in the Figure 6. Fig. 6(a) shows the results of DE algorithms, while Fig. 6(b) illustrate the results achieved with SA algorithms. As can be seen, results reached with DE led to a closer concordance with benchmark solution than those reached with SA. Among the DE algorithms, the best results are obtained with DE1 and DE2. The versions of the SA algorithm achieve divergent results for this DOF, where the worst performance is reached with SAEX, followed by SABE and SABO. The other versions (SAC1 and SAC2) are more similar to those reached with DE3 and DE4 for higher magnitudes of H/L.

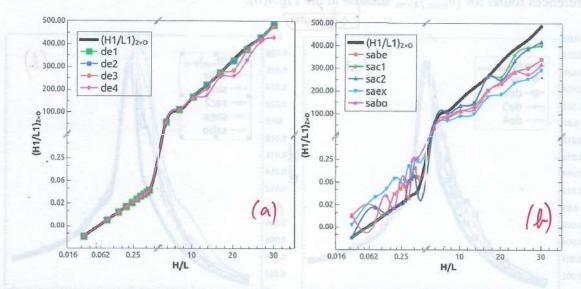


Figure 6- Effect of H/L over $(H_1/L_1)_{2\times o}$ obtained by different optimization methods: a) DE algorithm, b) SA algorithm

In Figure 7/it is possible to note that the results of all algorithms diverge from the optimal values of $(S_1/H_0)_{2\times o}$ for all ratios of H/L. The results in Fig 7(a) shows that all versions of DE algorithm do not reach the optimal values of $(S_1/H_0)_{2\times o}$ for lower ratios of H/L. In spite of this fact, DE1 results have a similar tendency to that predicted with benchmark solution. The

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7(b) shows that versions of SA algorithm have a high difficult to represent the optimal values of $(S_1/H_0)_{2\times 0}$ for any ratio of H/L analyzed.

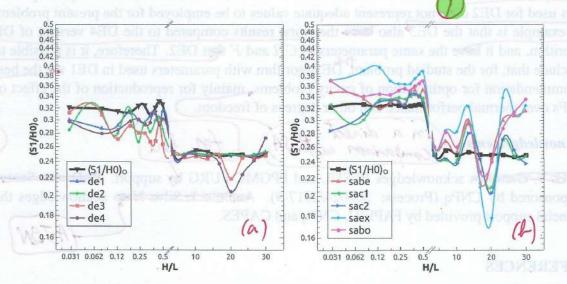


Figure 7- Effect of H/L over $(S_1/H_0)_o$ obtained by different optimization methods: a) DE algorithm, b) SA algorithm.

5. CONCLUSIONS



The present work studied the employment of two meta-heuristic, Simulated Annealing (SA) and Differential Evolution (DE), combined with Constructal Design (CD) to perform a geometric optimization in a heat transfer problem. The problem consists of a cooled double-T shaped cavity intruded into a conductive solid wall with constant heat generation. The geometric optimization problem is conducted by the CD method that defines the Degrees of Freedom (DOFs) and constraints. The purpose is to compare the results of the two heuristics to evaluate the efficiency to reproduce the effect of geometric parameters over four and five times minimized maximal excess of temperature reached in the solid wall and respective optimal configurations for degrees of freedom. The evaluation of geometric configurations over the flow system performance is an important subject into Constructal Design framework. Then, the present study aimed to contribute with recommendations about the best optimization methods to be associated with Constructal Design to reproduction of geometry effect over system performance.

The SA algorithm versions led to inferior performance than that reached with DE methods. However, in an inner comparison between SA versions, the best performance was achieved with hybrid functions of Cooling Schedules (CS) in agreement with previous founds of Gonzales et al. (2015b,a).

Results of optimization for four degrees of freedom indicated that DE algorithms were successful for reproduction of the effect of degrees of freedom over thermal performance in most of the investigations performed, mainly the DE1 version. For the geometric optimization of five DOF the versions of DE algorithm also were superior to the results of SA algorithm. For the evaluation with 5 degrees of freedom, the DE1 version again was the most well succeeded to find the best shapes. Results also showed that the DE2 algorithm achieved good results for some cases, as the effect of H/L over $(S_1/H_0)_o$ and over $(H_1/L_1)_{2\times o}$. However, this version of DE algorithm led to worst results on the effect of H/L over $(H_1/L_1)_{2\times o}$ which influenced









the results of $(\theta_{max})_{4\times m}$. Differences in the Crossover rate (CR) and amplification factor (F) can explain the differences found for two versions of DE (DE1 and DE2). Probably, the parameters used for DE2 does not represent adequate values to be employed for the present problem. An example is that the DE3 also the worst results compared to the DE4 version of DE algorithm, and it have the same parameters of CR and F that DE2. Therefore, it is possible to conclude that, for the studied problem, DE algorithm with parameters used in DE1 are the best recommendation for optimization of cavity problems, mainly for reproduction of the effect of DOFs over thermal performance and other degrees of freedom.

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