

# Machine Design Homework 4

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```
[1]: # Notebook Preamble
%matplotlib inline
import matplotlib.pyplot as plt
import sympy as sp
import numpy as np
from IPython.display import display, Markdown

plt.style.use('maroon_ipynb.mplstyle')
```

# Contents

<b>1</b>	<b>Problem 6-1</b>	<b>3</b>
1.1	Given . . . . .	3
1.2	Find . . . . .	3
1.3	Solution . . . . .	3
<b>2</b>	<b>Problem 6-3</b>	<b>5</b>
2.1	Given . . . . .	5
2.2	Find . . . . .	5
2.3	Solution . . . . .	5
<b>3</b>	<b>Problem 6-17</b>	<b>7</b>
3.1	Given . . . . .	7
3.2	Find . . . . .	7
3.3	Solution . . . . .	7

## 1 Problem 6-1

### 1.1 Given

A 10-mm steel drill rod was heat treated and ground. The measured hardness was found to be 300 Brinell.

### 1.2 Find

Estimate the endurance strength in *MPa* if the rod is used in rotating bending.

### 1.3 Solution

Eq. 6-10 on p. 305,

$$S'_e = \begin{cases} 0.5S_{ut} & S_{ut} \leq 200 \text{ ksi (1400 MPa)} \\ 100 & S_{ut} > 200 \text{ ksi} \\ 700 \text{ MPa} & S_{ut} > 1400 \text{ MPa} \end{cases}$$

The ultimate strength of steel comes from Eq. 2-36,

$$S_{ut} = 3.4H_B$$

```
[2]: H_B = 300
      S_ut = sp.S('3.4')*H_B

      if S_ut <= 1400:
          S_e_prime = 0.5*S_ut
      else:
          S_e_prime = sp.S(700)

      S_e_prime # ksi
```

```
[2]: 510.0
```

This value is not the final value. The relationship for the refined value is,

$$S_e = k_a k_b k_c k_d k_e S'_e$$

The only necessary *k* values used for this analysis is *k<sub>a</sub>* and *k<sub>b</sub>*, whose equations are at 6-18 and 6-19 respectfully.

```
[3]: # See Table 6-2
      k_a = sp.S('1.38')*S_ut**-(sp.S('0.067'))
      d = 10
```

```
k_b = sp.S('1.24')*d**-(sp.S('0.107'))  
# display(k_a, k_b)  
S_e = k_a*k_b*S_e_prime  
S_e # MPa
```

[3]: 428.839455736079

## 2 Problem 6-3

### 2.1 Given

A steel rotating beam test specimen has an ultimate strength of 120 *ksi*.

### 2.2 Find

Estimate the life of the specimen if it is tested at completely reversed stress amplitude of 70 *ksi*.

### 2.3 Solution

Find  $S_e$  first.

```
[4]: S_ut = sp.S(120) # ksi

if S_ut <= 200:
    S_e_prime = 0.5*S_ut
else:
    S_e_prime = sp.S(100)

S_e_prime # ksi
```

```
[4]: 60.0
```

The  $S'_e$  value will be used in place of  $S_e$  from Figure 6-23 description. We can use the following relationships to determine  $N$ .

$$N = \left( \frac{\sigma_{ar}}{a} \right)^{1/b}$$
$$a = \frac{(fS_{ut})^2}{S_e}$$
$$b = -\frac{1}{3} \log \left( \frac{fS_{ut}}{S_e} \right)$$

The value of  $f$  is 0.82 from Figure 6-23. The  $S_{ut}$  value is  $2(S_e) = 120$  *ksi*.

```
[5]: def log10(x_):
    return sp.log(x_)/sp.log(10)

f = sp.S('0.82')
a = (f*S_ut)**2/S_e_prime
b = -sp.Rational(1, 3)*log10(f*S_ut/S_e_prime)

display(sp.Eq(sp.Symbol('a'), a.n()),
        sp.Eq(sp.Symbol('b'), b.n()))
```

```
sig_ar = 70
N = ((sig_ar/a)**(1/b)).n()
N # cycles
```

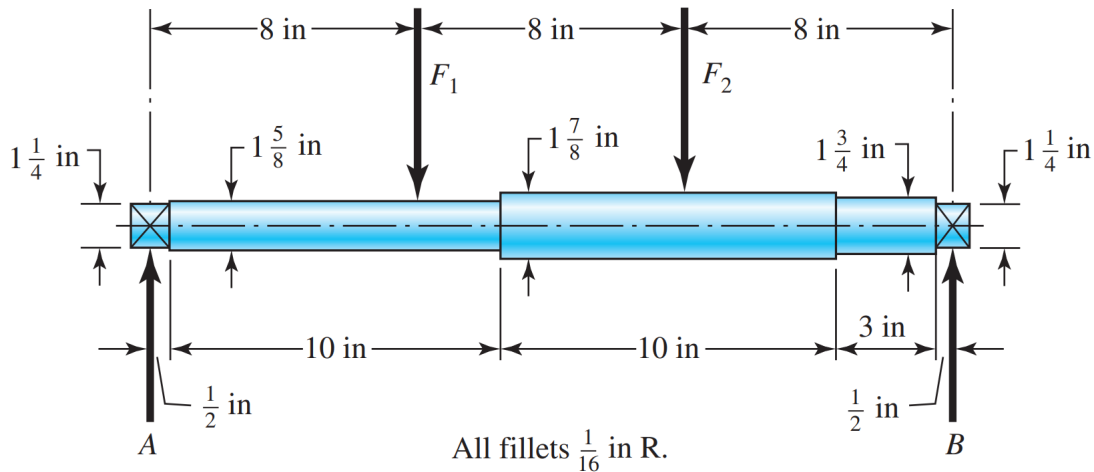
$a = 161.376$

$b = -0.0716146160158993$

[5]: 116192.956004683

### 3 Problem 6-17

#### 3.1 Given



The shaft shown in the figure above is machined from AISI 1040 CD steel. The shaft rotates at 1600 rpm and is supported in roller bearings at  $A$  and  $B$ . The applied forces are  $F_1 = 2500 \text{ lbf}$  and  $F_2 = 1000 \text{ lbf}$ .

#### 3.2 Find

Determine the minimum fatigue factor of safety based on achieving infinite life. If infinite life is not predicted, estimate the number of cycles to failure. Also check for yielding.

#### 3.3 Solution

The reaction forces need to be solved first.

```
[6]: A, B = sp.symbols('A B')
F1, F2 = 2500, 1000

eq1 = sp.Eq(A + B, F1 + F2)
eq2 = sp.Eq(B*24 - F1*8 - F2*16, 0)

sol = sp.solve([eq1, eq2], dict=True)[0]

display(eq1, eq2, Markdown('---'))
for key, value in sol.items():
    display(sp.Eq(key, value))
```

$$A + B = 3500$$

$$24B - 36000 = 0$$

$A = 2000$

$B = 1500$

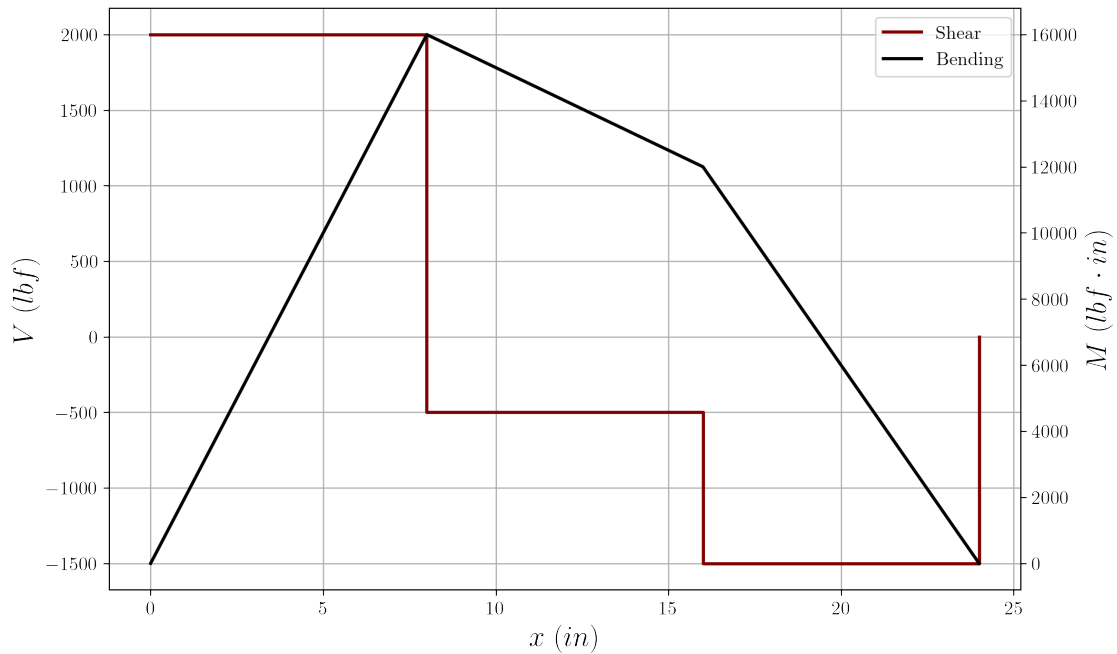
```
[7]: # Plotting Shear and Bending Moment Diagram
x = [0, 8, 16, 24]
x_shear = [0, 8, 8, 16, 16, 24, 24]
V1, V2, V3, V4 = [sol[A], sol[A] - F1, sol[A] - F1 - F2, sol[A] - F1 - F2 +
↳sol[B]]
V = [V1, V1, V2, V2, V3, V3, V4]
M = M1, M2, M3, M4 = [0, V1*8, V1*8 + V2*8, V1*8 + V2*8 + V3*8]

fig, ax = plt.subplots()
ax2 = ax.twinx()

ax.plot(x_shear, V, label='Shear')
ax2.plot(x, M, color='black', label='Bending')
ax2.grid(visible=False)

ax.legend(handles=[ax.lines[0], ax2.lines[0]])
ax.set_xlabel('$x$ ($in$)')
ax.set_ylabel('$V$ ($lbf$)')
ax2.set_ylabel(r'$M$ ($lbf \cdot in$)')

plt.show()
```





We are interested in the stress at the fillet radius in which the smaller diameter is used.

```
[8]: M_mid = (M3 - M2)/8*(sp.S('10.5') - 8) + M2  
M_mid # in lbf*in
```

```
[8]: 14750.0
```

```
[9]: c = sp.S('1.625')/2  
I = sp.pi.n()/4*c**4  
sig = M_mid*c/I  
sig # in psi
```

```
[9]: 35013.218176932
```

The yield strength is 71 *ksi*, and this stress is far below this value. The ultimate strength is  $S_{ut} = 0.5(H_B) = 0.5(170) = 85$  *ksi*.