System Dynamics Homework 5

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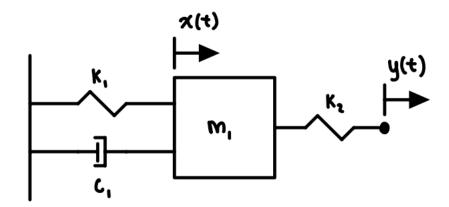
```
[1]: import control as ct
import sympy as sp
import numpy as np
import matplotlib.pyplot as plt
plt.style.use('../maroon_ipynb.mplstyle')
```

Contents

1	\mathbf{Pro}	blem 1	L																	
	1.1	Given															 			
	1.2	Find															 			
	1.3	Solution	on														 			
		1.3.1	Part A														 			
		1.3.2	Part B														 			
		1.3.3	Part C																	
		1.3.4	Part D																	
		1.3.5	Part E														 			

1 Problem 1

1.1 Given



The mass above is being controlled by the input position y(t). Take $k_1=10\frac{lbf}{in},\ k_2=100\frac{lbf}{in},\ c_1=1\frac{lbf\,s}{in},\ and\ m_1=0.0518\,goobs$ where $1\,goob=\frac{lbs\,s^2}{in}$. The number comes from the mass weighing $20\,lbf$.

$$W = mg$$

$$20 \, lbf = m \cdot 32.2 \, \frac{ft}{s^2}$$

$$m = \frac{20}{32.2} \, slugs = 0.621 \, \frac{lbf \, s^2}{ft} \cdot \frac{ft}{12 \, in} = 0.0518 \, \frac{lbf \, s^2}{in} = 0.0518 \, goobs$$

1.2 Find

For $y(t) = 1.5\sin(\omega_r t)$,

- a. Find the equation of motion.
- b. Find the transfer function $\frac{X(s)}{Y(s)}$.
- c. Plot the Magnitude $(M(\omega))$ not in decibels) and Phase Response for $1 \le \omega \le 1000 \, rad/s$. Use the bode() function for checking. Note, the bode() function will produce a log-log plot on the y and x-axis, so the usual plot of $M(\omega)$ will look different, since we use a linear scale for the y-axis.
- d. Find the resonant frequency ω_r .
- e. Find the steady state function $x_{ss}(t)$ at the resonant frequency and plot the forced response on top of the steady state response up to 1 second.

1.3 Solution

1.3.1 Part A

```
[2]: t, s, k1, k2, c1, m1 = sp.symbols('t s k1 k2 c1 m1')
x, y = sp.Function('x')(t), sp.Function('y')(t)

k1_, k2_ = 10, 100  # lbf/in
c1_ = 1  # lbs*s/in
m1_ = 0.0518  # goobs

eq = sp.Eq(m1*x.diff(t, 2), -k1*x - c1*x.diff() + k2*(y - x))
eq
```

[2]: $m_1 \frac{d^2}{dt^2} x(t) = -c_1 \frac{d}{dt} x(t) - k_1 x(t) + k_2 \left(-x(t) + y(t) \right)$

1.3.2 Part B

```
[3]: lp = lambda expr: sp.laplace_transform(expr, t, s)[0]
    eq_s = sp.Eq(lp(eq.lhs), lp(eq.rhs.expand()))

sub_ics = [
        (x.subs(t, 0), 0),
        (x.diff().subs(t, 0), 0)
]

eq_s = eq_s.subs(sub_ics)
    eq_s
```

$$\boxed{\mathbf{G3]}: m_1 s^2 \mathcal{L}_t\left[x(t)\right](s) = -c_1 s \mathcal{L}_t\left[x(t)\right](s) - k_1 \mathcal{L}_t\left[x(t)\right](s) - k_2 \mathcal{L}_t\left[x(t)\right](s) + k_2 \mathcal{L}_t\left[y(t)\right](s)}$$

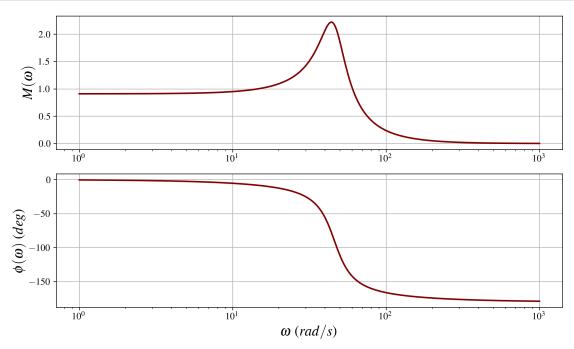
[4]:
$$\frac{k_2}{c_1s + k_1 + k_2 + m_1s^2}$$

[5]:

$$\frac{100}{0.0518s^2 + s + 110}$$

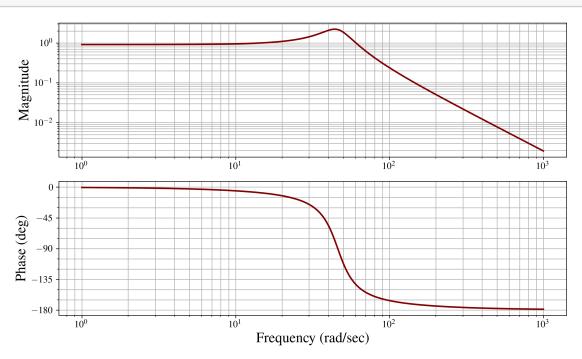
1.3.3 Part C

```
(c1, c1_),
    (m1, m1_)
]))
omegas = np.linspace(1, 1000, 100_000)
mags = np.abs(T_jw(omegas))
phase = np.angle(T_jw(omegas))
fig, (ax1, ax2) = plt.subplots(nrows=2, ncols=1)
ax1.set_xscale('log')
ax2.set_xscale('log')
# ax1.set_yscale('log')
ax1.plot(omegas, mags)
ax1.set_ylabel(r'$M(\omega)$')
ax2.plot(omegas, np.rad2deg(phase))
ax2.set_ylabel(r'$\phi(\omega)$ ($deg$)')
ax2.set_xlabel(r'$\omega$ ($rad/s$)')
# fig.savefig('p1.png')
plt.show()
```



[7]: # For checking

 $_$ = ct.bode(T, omega=omegas) # There is unwanted behavior to where the $_$ $_$ magnitude y-axis is logarithmic. I checked and there is no changing it.



1.3.4 Part D

Eigenvalue (pole) Damping Frequency
-9.653 +45.06j 0.2095 46.08
-9.653 -45.06j 0.2095 46.08

[9]: wr = wn*np.sqrt(1 - 2*zeta**2) wr # rad/s

[9]: 44.01375056012086

[10]: # Could also do something like this omegas [max(mags) == mags] [0]

[10]: 44.017370173701735

1.3.5 Part E

[11]: phi = np.angle(T_jw(wr))
 B = 1.5*np.abs(T_jw(wr))
 x_ss = lambda t_: B*np.sin(wr*t_ + phi)

```
t_array = np.linspace(0, 1, 1000)
_, x_ = ct.forced_response(T, T=t_array, U=1.5*np.sin(wr*t_array))

fig, ax = plt.subplots()

ax.plot(t_array, x_ss(t_array), label='$x_{ss}(t)$')
ax.plot(t_array, x_, label='$x(t)$')
ax.plot(t_array, 1.5*np.sin(wr*t_array), label='Input $y(t)$')

ax.legend()
ax.set_xlabel('Time ($s$)')
ax.set_ylabel('Position ($in$)')
plt.show()
```

