Time Allowed: 2½ hours

### NANYANG TECHNOLOGICAL UNIVERSITY

### SEMESTER 1 EXAMINATION 2019-2020

### MA3002- SOLID MECHANICS AND VIBRATION

November/December 2019

## **INSTRUCTIONS**

- 1. This paper contains **FOUR (4)** questions and comprises **EIGHT (8)** pages.
- 2. Answer **ALL** questions.
- 3. All questions carry equal marks.
- 4. This is a **RESTRICTED-OPEN BOOK** examination. One double-sided A4 size reference sheet of paper is allowed.
- 1(a) A spring loaded mechanism is made up of rigid bars AB, BC and CD pin jointed and subject to a vertical force, **P**, applied at point C. The mechanism is in equilibrium with the two springs shown in Figure 1. The rigid bars have uniform length of L. The mechanism has an angle,  $\theta$ , relative to the horizontal surface. The two linear spring stiffness are indicated as  $k_1$  and  $k_2$ . When the load, P, is removed, the springs will return the mechanism back to its initial position on the vertical plane when  $\theta = 90^{\circ}$ .

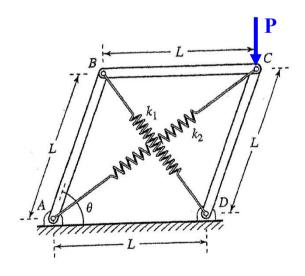


Figure 1

(i) Determine the relationship between the applied force (P) with respect to the spring stiffness  $k_1$  and  $k_2$ , angle ( $\theta$ ), and the rigid bar length L. Use the Principle of Virtual Work (PVW) method. Draw and specify your datum. Neglect friction and selfweight of the bars.

(4 marks)

Note: Question 1 continues on page 2.

- (ii) Calculate the applied force P in newton (N), if the bar dimension L = 1.0 m, the angle,  $\theta = 60^{\circ}$ , the spring stiffness,  $k_1 = 1000$  N/m and  $k_2 = 1000$  N/m respectively. (2 marks)
- (b) A cantilevered curved beam (A-B-C-D) is subjected to a load, W, at the free end (D) as shown in Figure 2. Using the Unit Load method (consider only bending effects and state clearly your datum), determine the following:
  - (i) Vertical Displacement expression of the free end at D.
  - (ii) Horizontal Displacement expression of the free end at D.

Draw your REAL and VIRTUAL load diagrams and specify the relevant Bending Moment equations to be used. Calculate the Vertical and Horizontal Displacements in millimeters (mm) if the Point Load W = 100N, R = 1.0 m, and EI = 21,000 N/m<sup>2</sup>.

(6 marks)

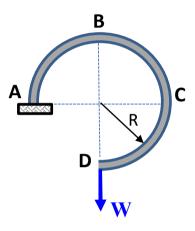
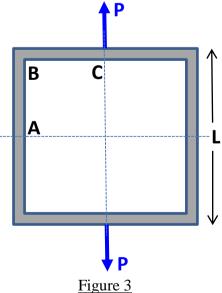


Figure 2

(c) A statically indeterminate structure is in a form of a square steel member is subjected to symmetrical Point Loads, P, as shown in Figure 3. A quarter symmetry segment is indicated by the position of A-B-C. The length of the square edge is given as L.



Note: Question 1 continues on page 3.

(i) Draw the free body diagram for a Quarter Symmetry segment A-B-C, and indicate the equilibrium forces and moments in a free body diagram.

(4 marks)

- (ii) Show that this problem is Statically Indeterminate to what degree of Indeterminacy. (2 marks)
- (iii) Determine the Vertical Displacement expression at the Load Point, C, for the Quarter Symmetry segment A-B-C using the Unit Load and Unit Moment methods. Consider bending effects only. Neglect friction and self-weight of the bars.

(5 marks)

(iv) Calculate the Vertical Displacement at the Load Point, C, for the Quarter Symmetry segment ABC in millimeters (mm) if the Point Load P = 1000N, L = 1.0 m, and  $EI = 26,000 \text{ N/m}^2$ 

(2 marks)

2(a) A Plane Strain fracture toughness test was conducted using a Compact Tension (CT) Specimen (with a thickness of B = 50mm) and the test data is recorded in Table 1.

<u>Table 1</u>: Compact Tension (CT) Specimen Test Data

 $P_{\text{max}} = 170 \text{ kN}$   $P_{\text{Q}} = 160 \text{ kN}$   $K_Q = \frac{P_Q}{BW^{1/2}} f_2 \left(\frac{a}{W}\right)$   $K_Q = \frac{P_Q}{BW^{1/2}} f_2 \left(\frac{a}{W}\right)$   $F_2 \left(\frac{a}{W}\right) = 9.6$ 

W = 100mm (Width)

Yield Stress,  $\sigma_y = 1200 \text{ MN/m}^2$ .

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(i) Calculate the Qualifying Stress Intensity Factor,  $K_Q$  value from the test data given in Table 1.

(4 marks)

(ii) Determine if the CT specimen test result meet the Plane Strain Fracture Toughness three-pronged checking criteria. Conclude if,  $K_Q$  is a valid  $K_{IC}$  (plane strain fracture toughness) test result?

(4 marks)

(b) An edge crack in an infinite width plate has a flaw in the form of an edge crack of length, a, as shown in Figure 4. The geometric factor, Y = 1.12, can be used in calculating the Stress Intensity Factor, solution.

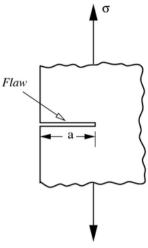


Figure 4

(i) Calculate the applied stress intensity factor,  $K_I$ , if the crack length, a=1.0 mm (or 0.001m), and the applies stress,  $\sigma=250$  MN/m<sup>2</sup>.

(2 marks)

(ii) Calculate the flaw size or critical crack length,  $a_{cr}$ , due to brittle fracture, if the plane strain fracture toughness of the material,  $K_{IC} = 56 \text{ MN/m}^{3/2}$ .

(2 marks)

(iii) Calculate the crack propagation life,  $N_f$ , for a cyclic stress varying from zero to 250 MN/m<sup>2</sup> giving a stress range,  $S_R = 250$  MN/m<sup>2</sup>. The initial crack length,  $a_o = 1.0$  mm (or 0.001m) and the final critical crack length  $a_f$  can be obtained from part (ii). The fatigue crack propagation life expression and the Paris Law constants are given below.

$$N_f = \frac{2}{C(YS_R)^m \pi^{m/2} (2-m)} \left( a_f^{1-m/2} - a_0^{1-m/2} \right)$$

where,  $C = 4 \times 10^{-12}$  (m/cycle); m = 3.3

(6 marks)

Note: Question 2 continues on page 5.

(c) A steel specimen was tested under fatigue stress range conditions and the fatigue S-N curve test data is given below in Table 2.

<u>Table 2</u>: Fatigue S-N Curve Test Data

Test Specimen No.1

Stress Range :  $S_R = 200 \text{ MN/m}^2$ . Fatigue Life :  $N_f = 900,000 \text{ cycles}$ 

Test Specimen No.2

Stress Range :  $S_R = 310 \text{ MN/m}^2$ . Fatigue Life :  $N_f = 12,000 \text{ cycles}$ 

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(i) The steel structure is subjected to two stress range level exposures every day and strain gage measurement show that there are 30 cycles of stress range of  $S_R = 200 \ MN/m^2$  and 3 cycles of stress range of  $S_R = 310 \ MN/m^2$ , in one day of loading operation.

Using Miners Rule, calculate the number of years it will take for the structure to fail by fatigue when the Cumulative Damage Summation index is equal to D = 1.0. Use the Fatigue test data provided in Table 2.

(4 marks)

(ii) The S-N Curve equation can be curve-fitted to the fatigue test data when analyzed on a Log Stress versus Log Cycles plot.

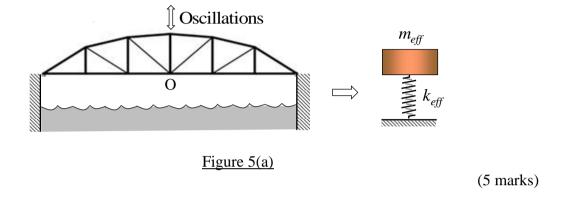
Determine the S-N curve equation constants A and b using the fatigue test data from Table 2.

Note: A typical S-N Curve equation has two constants **A** and **-b**.

$$S_R = A \cdot (N_f)^{-b}$$

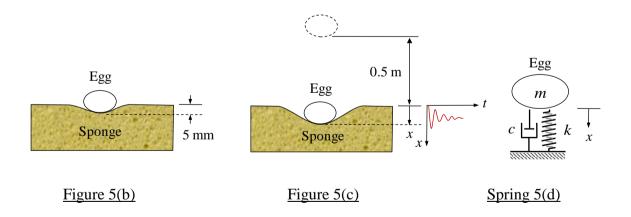
(3 marks)

3(a) It is intended to analyse the vertical oscillations of a small bridge by modelling it as a spring-mass system as shown in Figure 5(a) where  $k_{eff}$  is the effective stiffness of the bridge and  $m_{eff}$  is the corresponding effective mass. When the bridge is deflected down by imposing a certain vertical displacement at O (the mid-section of the beam) and allowed to oscillate freely, the bridge executes natural oscillations in the vertical direction as indicated. The natural period of vertical oscillations is found to be 0.4 s. When a mass 8000 kg is placed at the mid-section of the bridge at O, the natural period of the oscillation increases to 0.42 s. Determine  $k_{eff}$  and  $m_{eff}$  of the spring-mass system. Ignore damping effects.



- (b) When a chicken egg of mass m = 50 g is placed gently on a block of sponge material, it produces a static deflection of 5 mm as shown in Figure 5(b).
  - (i) Assuming that the sponge material behaves like a spring, determine the stiffness (*k*) of the sponge material.

(2 marks)



Note: Question 3 continues on page 7.

(ii) When the egg is dropped from a height of 0.5 m above the surface of sponge block (as shown in Figure 5(c)), the motion of egg after landing on sponge block is underdamped with successive vibration amplitudes reducing to 0.1 times that of the previous. Assume the egg does not rebound or break during the drop. Ignore the mass of sponge material. Determine the damping ratio, damping coefficient, undamped natural frequency (rad/s) and damped natural frequency (rad/s).

(5 marks)

(iii) As the egg is falling from the height of 0.5 m, the time t is taken as 0 when the egg just touches the sponge material. Using an appropriate free body diagram, derive the equation of motion of egg for time t > 0 by modelling the sponge-egg combination as a spring-mass-damper system as shown in Figure 5(d), and determine the initial conditions x(0) and  $\dot{x}(0)$ .

(4 marks)

(iv) Determine the solution to the equation of motion derived in part (iii).

(7 marks)

(v) What should be the value of damping coefficient of sponge material for which the motion of egg is critically damped? Assume that sponge material has the same stiffness as calculated in part (i).

(2 marks)

4. Figure 6 shows a 2-DOF vibrating system in its static equilibrium configuration. The system consists of a rigid beam ABC of length l and mass  $m_1$ , pin-jointed at A and supported by two springs of same stiffness  $k_1$  at the mid-section of the beam at B. The beam carries a spring-mass system of mass  $m_2$  and spring stiffness  $k_2$  at C. Mass  $m_2$  is subjected to a harmonic force excitation of amplitude  $F_2$  at excitation frequency  $\omega$ .

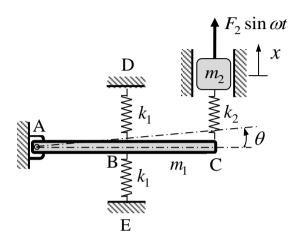


Figure 6

(a) Applying Newton's  $2^{nd}$  law, derive the equations of motion for this 2-DOF system in terms of the coordinates x and  $\theta$ , and represent them in matrix form. Draw neat free body diagrams to aid your derivation. Static forces need not be shown on the free body diagrams as they are balanced.

(8 marks)

- (b) Taking l = 1 m,  $m_1 = 1$  kg,  $k_1 = 1000$  N/m,  $m_2 = 1$  kg,  $k_2 = 1000$  N/m, determine the natural frequencies (in Hz) and the corresponding amplitude ratios of the system. (10 marks)
- (c) Taking  $F_2 = 20$  N, determine the vibration amplitude of the beam ( $\Theta$ ) at the excitation frequency  $\omega = 15$  rad/s. (5 marks)
- (d) What is the amplitude of force transmitted to each of the supports at E and D. (2 marks)

# **End of Paper**