

Full Name : \_\_\_\_\_

Time : 40 minutes**Qu 1**

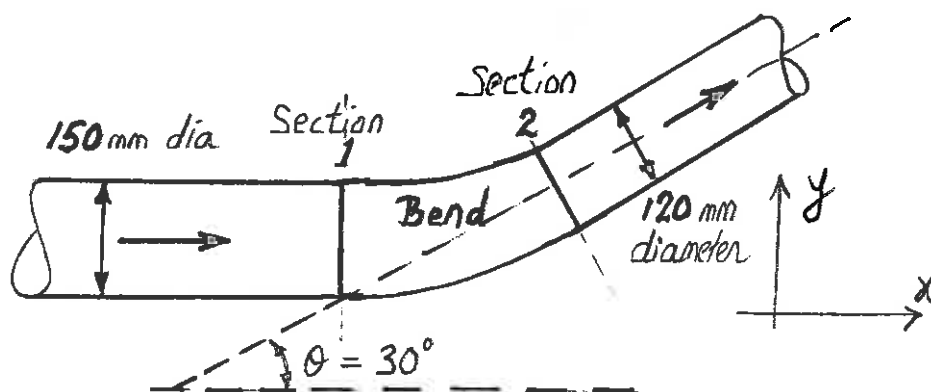
Water flows through a reducing pipe bend as shown. The flow rate is  $0.02 \text{ m}^3/\text{s}$ . Pressure at section 1 and 2 is 280 kPa and 260 kPa respectively. Determine the resultant force and direction to hold the pipe bend stationary.

Weight of nozzle and water inside the nozzle may be neglected. There is no height difference between 1 and 2.

(Write down the governing equations)

(10 marks)

This is internal flow  
Hence, there is pressure  
 $P_1$  &  $P_2$  acting on CV.



- 1) Apply BE to determine  $V_1$  and  $V_2$ ,  $P_1$  &  $P_2$  are given
- 2) Find  $\dot{m}$  kg/s.
- 3) Apply momentum equation in  $x$  and  $y$  direction

$$(\sum F)_x = \dot{m}(V_{2x} - V_{1x}).$$

$$F_x + P_1 A_1 - P_2 A_2 \cos 30^\circ = \dot{m}(V_2 \cos 30^\circ - V_1).$$

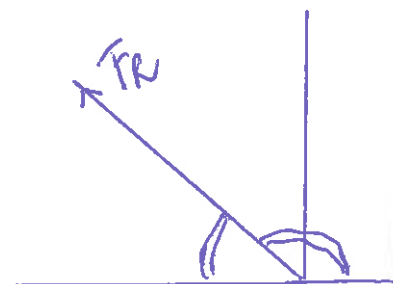
Note:  $P_2$  acting in opposite direction at angle  $30^\circ$

$$(\sum F)_y = \dot{m}(V_{2y} - V_{1y}).$$

$$F_y - P_2 A_2 \sin 30^\circ = \dot{m}(V_{2y})$$

$$V_{1y} = 0.$$

$$\text{Resultant force } F_R = \sqrt{F_x^2 + F_y^2}$$



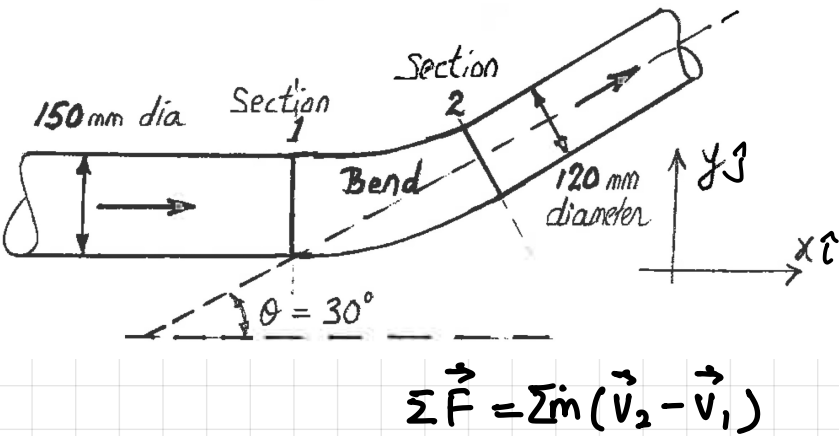
### Qu 1

Water flows through a reducing pipe bend as shown. The flow rate is  $0.02 \text{ m}^3/\text{s}$ . Pressure at section 1 and 2 is  $280 \text{ kPa}$  and  $260 \text{ kPa}$  respectively. Determine the resultant force and direction to hold the pipe bend stationary.

Weight of nozzle and water inside the nozzle maybe neglected. There is no height difference between 1 and 2.

( Write down the governing equations )

( 10 marks )



At 1:  $Q = 0.02 \text{ m}^3/\text{s}$  At 2:

$P_1 = 280 \text{ kPa}$   $\dot{m} = 20 \text{ kg/s}$   $P_2 = 260 \text{ kPa}$

$V_1 =$   $V_2 =$

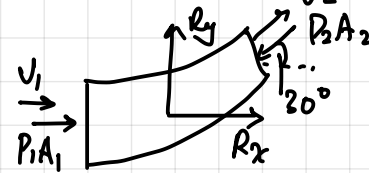
$A_1 = \frac{\pi}{4} (150 \text{ mm})^2 = 0.0177 \text{ m}^2$   $A_2 = \frac{\pi}{4} (120 \text{ mm})^2 = 0.01131 \text{ m}^2$

$\dot{m}_1 = \dot{m}_2$

Continuity  $\Rightarrow Q_1 = Q_2$

$V_1 =$  m/s

$V_2 =$  m/s



$$P_1 + \frac{1}{2} \rho g h_1 + \frac{1}{2} \rho V_1^2 = P_2 + \frac{1}{2} \rho g h_2 + \frac{1}{2} \rho V_2^2$$

$$P_1 - P_2 = \frac{1}{2} \rho (V_2^2 - V_1^2)$$

$$\frac{2(P_1 - P_2)}{\rho} = V_2^2 - V_1^2$$

$$A_1 V_1 = A_2 V_2$$

$$V_2 = \frac{A_1}{A_2} V_1$$

$$[R_x + P_1 A_1 - P_2 A_2 \cos 30^\circ] \hat{i} + [R_y - P_2 A_2 \sin 30^\circ] \hat{j} = 20 [(V_2 \cos 30^\circ - V_1) \hat{i} + (V_2 \sin 30^\circ) \hat{j}]$$

In  $\hat{j}$ :

$$R_y = (260 \text{ k})(0.01131) \sin 30^\circ + 20 (8.22 \sin 30^\circ)$$

$$R_y = 1552.5 \text{ N}$$

In  $\hat{i}$ :

$$R_x = -(280 \text{ k})(0.0177) + (260 \text{ k})(0.0113) \sin 30^\circ + 20 [(8.22 \cos 30^\circ - 5.25)]$$

$$R_x = -4956 + 1470.3 + 37.37 = -3448.33 \text{ N}$$

$$\frac{2(P_1 - P_2)}{\rho} = \left[ \left( \frac{A_1}{A_2} \right)^2 - 1 \right] V_1^2$$

$$V_1 = 5.25 \text{ m/s}$$

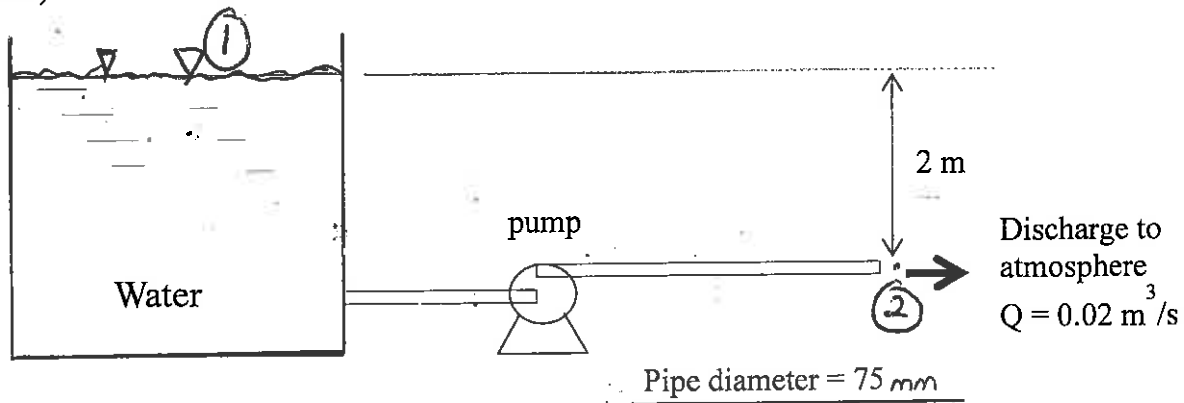
$$V_2 = 8.22 \text{ m/s}$$

**Qu.2**

Water is delivered from an open tank through a pump and piping system with diameter of 75mm.. Water exits the pipe to the atmosphere at  $0.02 \text{ m}^3/\text{s}$ . The total loss in the piping system is,  $h_L = 6.5V^2/2g$  where  $V$  is the velocity of water in the pipe. What is the power required by the pump?

It is proposed to double the flow rate to  $Q = 0.04 \text{ m}^3/\text{s}$  by sealing the water tank completely and pressurizing the air space

**Determine the air pressure required.** (The pump remains in the piping system and piping system loss is,  $h_L = 6.5V^2/2g$ .)  
(10 marks)



(i) Apply energy equation from ①  $\rightarrow$  ②

$$\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + z_1 - h_L + h_p = \frac{P_2}{\rho g} + \frac{V_2^2}{2g} + z_2$$

$\downarrow$   
 $6.5 \frac{V_2^2}{2g}$

$$h_p = 9.842 \text{ m}$$

$Q = 0.02 \text{ m}^3/\text{s}$ , hence  $V_2 = 4.527 \text{ m/s}$  hence  $h_p = \frac{7.5(4.527^2)}{19.6} + 2$

power =  $\rho g Q h_p$  W

(ii).  $Q = 0.04 \text{ m}^3/\text{s}$ , hence, new  $V_2$

Same pump head applies

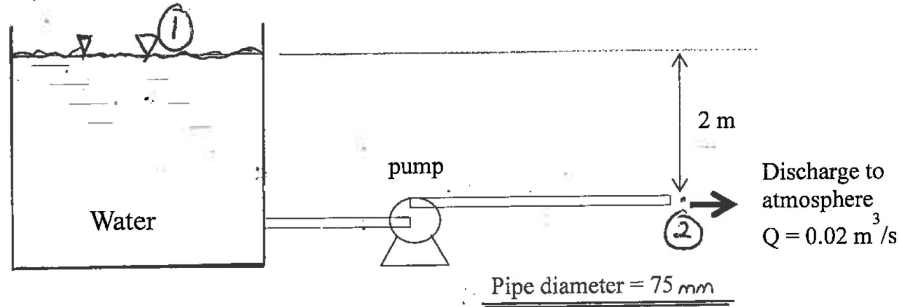
$P_1$  is not zero,  $P_1$  is now pressurized, (find  $P_1$ ).

Apply energy equation from ①  $\rightarrow$  ② to find  $P_1$

### Qu.2

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It is proposed to double the flow rate to  $Q = 0.04 \text{ m}^3/\text{s}$  by sealing the water tank completely and pressurizing the air space. Determine the air pressure required. (The pump remains in the piping system and piping system loss is,  $h_L = 6.5V^2/2g$ .) (10 marks)



$$e = \frac{P}{\rho} + \frac{1}{2}v^2 + gz$$

$\check{u} \Rightarrow$  internal energy

$h_L$

$$\frac{u_{out} - u_{in} = q h_L}{m}$$

Energy equation:

$$\sum \dot{W}_{in} + \sum \dot{m}(\check{e} + \check{u})_{in} = \sum \dot{W}_{out} + \sum \dot{m}(\check{e} + \check{u})_{out}$$

$$\dot{W}_{in} = \dot{m}[\check{u}_{out} - \check{u}_{in}] + \dot{m}\left[\frac{P_2 - P_1}{\rho} + \frac{1}{2}(v_2^2 - v_1^2) + g(z_2 - z_1)\right]$$

$$(i) \quad \check{u}_{out} - \check{u}_{in} = \frac{6.5V^2}{2}$$

$$P_2 - P_1 = 0, \quad v_1 = 0, \quad z_2 - z_1 = 2 \text{ m}$$

$$v_2 = \frac{Q}{A_2} = \frac{0.02}{\frac{\pi}{4}(75 \text{ mm})^2} = 4.527 \text{ m/s}$$

$$\check{u}_{out} - \check{u}_{in} = 6.8$$

$$\dot{W}_{in} = 20(6.8)^{9.8} + 20\left[\frac{1}{2}(4.527)^2 + (9.8)(2)\right]$$

$$\dot{W}_{in} = 1332.8 + 596.94 = 1929.74 \text{ W}$$

$$(ii) \quad v_{2 \text{ new}} = 9.054 \text{ m/s}$$

$$\dot{W}_{in} = \left[\frac{6.5(9.054)^2}{2}\right] + \left[\frac{\Delta P}{1000} + \frac{1}{2}(9.054)^2 + g(2)\right] = \frac{1929.74}{20}$$

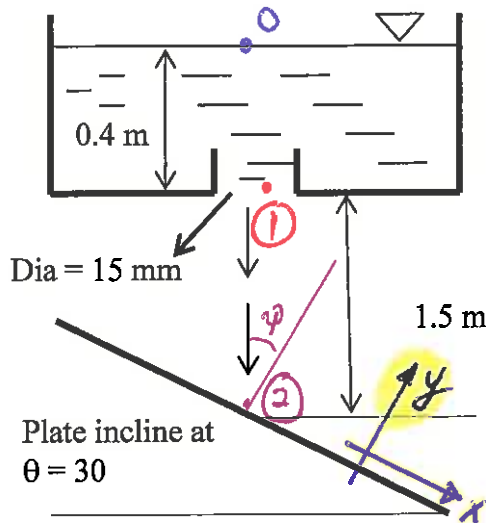
$$266.42 + \frac{\Delta P}{1000} + 60.59 = 96.49$$

$$\Delta P = -230.52 \text{ kPa} \Rightarrow P_1 = 230.52 \text{ kPa}$$

Name : \_\_\_\_\_

Time : 40 minutes

Qu 1 : A jet of water leaves a 15 mm diameter opening at the bottom of the tank and strikes an inclined plate and the water split equally with half of the water flows upwards and the other half downwards. Find the resultant force  $F$  to hold the plate stationary. Neglect mass of plate and assume water level in tank remains constant. (10 marks)  
Hint : Find velocity at exit 15 mm diameter opening.



(1) Find velocity at exit point 1. Apply B.E between (0)  $\rightarrow$  (1)

(2) With  $V_1$ , find  $Q$  flow rate and hence  $\dot{m}$  kg/s.

(3) Jet exit (1) will strike plate at point (2).

Find  $V_2$ .

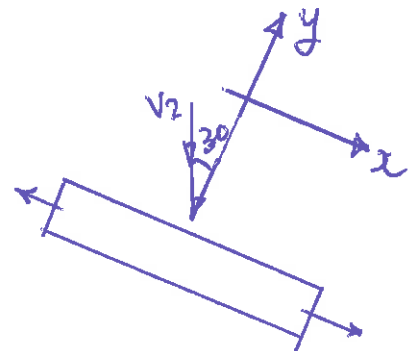
Momentum along the plate (x-direction) = 0

Apply momentum equation along y-direction (perpendicular to plate)

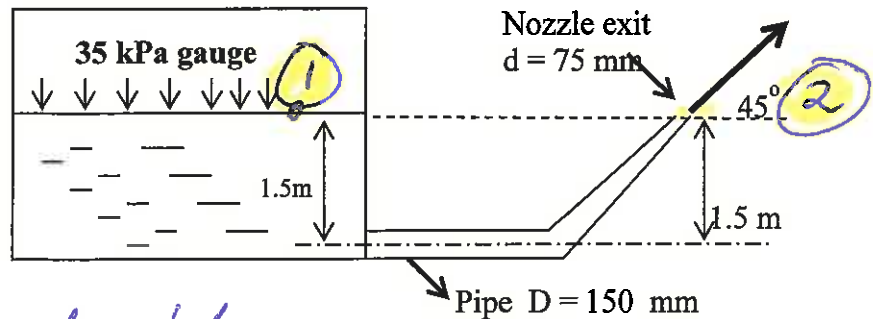
$$\sum F_y = \dot{m}(V_{2y} - V_{1y})$$

$$\sum F_y = (\sum mV)_{out} - (\sum mV)_{in}$$

$$\underline{F_y} = \underline{\dot{m}} (0 - V_2 \cos 30)$$



Qu 2 : Water is discharged from a pressurised tank (35 kPa) thru a pipe and a nozzle. Total loss for the pipe and nozzle is :  $h_L = 1.4 V_2^2/2g$ , where  $V_2$  is the velocity at nozzle exit. a) Find the discharge flow rate. b) if a pump is added to triple the discharge flow rate, find the fluid power required by the pump. Pressure in the tank and  $h_L$  remains unchanged ( $h_L = 1.4 V_2^2/2g$ ) (10 marks)



(a) Apply energy equation between  
①  $\rightarrow$  nozzle exit ②.

$$\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + Z_1 - \text{losses} = \frac{P_2}{\rho g} + \frac{V_2^2}{2g} + Z_2$$

$$P_1 = 35 \text{ kPa}$$

$$\text{losses} = 1.4 \frac{V_2^2}{2g}$$

$V_2$  can be found, hence flow rate  $Q = A_2 V_2$ .

(2). triple flow rate with pump.

$Q \times 3$ , hence find new velocity  $V_2$

Apply energy equation from ①  $\rightarrow$  2

$$\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + Z_1 - \text{losses} + h_p = \frac{P_2}{\rho g} + \frac{V_2^2}{2g} + Z_2$$

$$P_1 = 35 \text{ kPa}$$

$$\left(1.4 \frac{V_2^2}{2g}\right)$$

Find  $h_p$  = m.

$$\text{Power} = \rho g Q h_p \text{ (W)}$$