



Generalized Newton's methods for the approximation and resolution of frictional contact problems in elasticity



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ARTICLE INFO

Article history:

Received 31 July 2012

Received in revised form 6 December 2012

Accepted 23 December 2012

Available online 31 December 2012

Keywords:

Contact

Friction

Augmented Lagrangian

Generalized Newton's method

Nitsche's method

ABSTRACT

In this paper, some new generalized Newton's methods for the resolution of elastostatic frictional contact problem approximated by finite elements are presented and compared to existing ones. A numerical experimentation is performed to compare the different methods, especially with respect to the sensitivity to the method parameter. Two different strategies to approximate the contact and friction condition are considered: a nodal and an integral one. Existence and uniqueness results of the solution to the discretized problem are also discussed.

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1. Introduction

The numerical resolution of elastostatic or elastodynamic contact problems is already the subject of an important literature. The algorithms proposed to solve the approximated problem are quite varied. It includes interior points methods [31,37], specific methods for linear complementarity problems such as the Lemke algorithm [26,8], projected conjugate gradient [34], SSOR with projection [8,24] and finally generalized Newton's methods. This paper focuses on generalized Newton's methods which are widely used, especially in commercial software, and proved their efficiency and robustness in the framework of large scale elasticity problems with contact and friction.

The pioneering work in this area is the one of Alart and Curnier [2,3]. The Newton's method is applied to the optimality system of an augmented Lagrangian whose saddle point is the solution to the contact problem with Tresca friction. The optimality system can be easily extended to Coulomb friction. The term "generalized Newton's method" comes from the fact that the optimality system is not continuously differentiable but only Lipschitz-continuous and piecewise continuously differentiable. However, no special treatment is really needed by this non-differentiable character mainly because from a numerical viewpoint, it is quite improbable to come across a non-differentiable point. Note that in these references, the augmented Lagrangian is written on a nodal discretization of the contact problem, not on the continuous problem.

A second important work is the one of Laursen and Simo [39,30]. It is based on the optimality system of the same augmented Lagrangian. The main difference is that the proposed algorithm is an Uzawa algorithm on the contact stress, with a descent parameter equal to the augmentation parameter of the Lagrangian. The generalized Newton's method is only applied on the optimization step of the Uzawa algorithm which only concerns the displacement. This method is widely used, especially in commercial software. This might be due to the fact that it does not require a coupled resolution of the displacement and contact stress. Another characteristic is the fact that the first iteration, if the initial contact stress is set to zero, corresponds to the solving of a penalized contact problem with a penalization coefficient equal to the augmentation parameter of the Lagrangian. It allows to optionally stop after the first iteration if a penalization approach is chosen or to perform the Uzawa iterations to converge toward the non-penalized problem. Moreover, when the augmentation parameter is large enough, the Uzawa algorithm is converging in a few iterations.

Alart in [1] gives a global convergence result for a generalized Newton's method applied on a simplified non-symmetric version of the optimality system of the augmented Lagrangian. This global convergence occurs only on a model problem (membrane with obstacle) without the addition of any line search strategy. He also gives and analyzes some examples of instabilities for the case of a two-dimensional elasticity problem with contact on the boundary.

Christensen and Pang in [11] prove that the optimality system of the problem approximated by finite elements and extended to Coulomb friction is semi-smooth. A Lipschitz-continuous map f is

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said to be semi-smooth at a point y when it satisfies $\sup_{M \in \partial f(y+s)} \|f(y+s) - f(y) - Ms\| = o(\|s\|)$ where ∂f is its generalized gradient. The Newton method applied to a semi-smooth map converges super-linearly for a starting point sufficiently close to the solution (see [33,41]).

Hüber and Wohlmuth in [21] present an active set strategy. The set of active constraints is updated at each iteration. This algorithm can be re-interpreted as a generalized Newton's method for Alart–Curnier augmented Lagrangian.

Kunish and Stadler in [40,28] state that the optimality system of Alart–Curnier augmented Lagrangian is not semi-smooth in the continuous framework. They deduce that the scalability of Newton's method cannot be guaranteed. They also prove that the Uzawa algorithm introduced by Simo and Laursen converges whatever the value of the augmentation parameter for the contact with Tresca friction (in a continuous framework, which prove its scalability) and with a number of iterations which decreases when the augmentation parameter increases.

Recently, several attempts have been made to adapt Nitsche's method, initially proposed to prescribe a Dirichlet boundary condition, to contact with or without friction (see [17,42,9]). The advantage of Nitsche-based methods is to prevent the use of a multiplier while being consistent.

In this paper, some generalized Newton's methods based on new formulations of the contact problem are presented. These formulations do not derive from some augmented Lagrangians. The goal of the paper is to compare these new strategies to the existing ones from a numerical experiment viewpoint, especially to test the sensitivity to the augmentation parameter, to the applied load and the efficiency to solve two and three-dimensional problems.

The outline of the paper is the following. The so-called Signorini with Coulomb friction problem is recalled in Section 2, in strong and weak formulations. In Section 3, two versions of the finite element discretization are presented: one with a nodal discretization of the contact condition and another one with an integral one. Additionally, the numerical tests performed in the next sections are described. Section 4 is devoted to Alart–Curnier generalized Newton's method, Section 5 to the Simo–Laursen method, Section 6 to some new Nitsche-based methods for contact with friction and Section 7 to some other new methods based on augmented multipliers. Finally some conclusions are given in Section 8.

2. Signorini's problem with Coulomb friction

For the sake of simplicity, we consider the classical simple situation of the equilibrium of a linearly elastic body in contact with (static) Coulomb friction with a rigid foundation. Note that the main interest of this problem is to be similar to the one obtained by applying a time integration scheme on a quasi-static problem.

Let $\Omega \subset \mathbb{R}^d$, $d \in \{2, 3\}$ be a bounded regular domain which represents the reference configuration of a linearly elastic body submitted to a Neumann condition on Γ_N , a Dirichlet condition on Γ_D and a unilateral contact condition with friction on Γ_C with a rigid foundation, where Γ_N, Γ_D and Γ_C are non-overlapping open parts of $\partial\Omega$, the boundary of Ω (see Fig. 1). The part Γ_D is supposed of non-zero measure in $\partial\Omega$.

The displacement $u(t, x)$ of the body obeys the following equations:

$$\begin{aligned} -\operatorname{div} \sigma(u) &= f, \text{ in } \Omega, \quad \sigma(u) = \mathcal{A} \varepsilon(u), \text{ in } \Omega, \\ \sigma(u)n &= k, \text{ on } \Gamma_N, \quad u = 0, \text{ on } \Gamma_D, \end{aligned} \quad (1)$$

where $\sigma(u)$ is the stress tensor, $\varepsilon(u) = (\nabla u + \nabla u^T)/2$ is the linearized strain tensor, \mathcal{A} is the fourth order elasticity tensor which satisfies usual conditions of symmetry, coercivity and boundedness, n is the outward unit normal to Ω on $\partial\Omega$ and f, k are given force den-

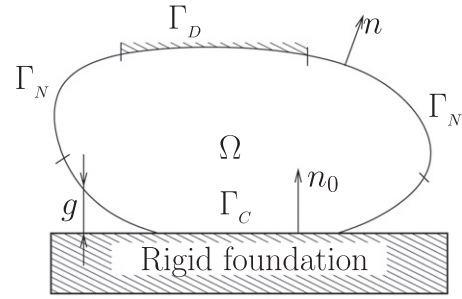


Fig. 1. Linearly elastic body Ω in contact with a rigid foundation.

sities. On Γ_C , it is usual to decompose the displacement and the stress in normal and tangential components as follows, assuming the shape of the rigid foundation to have the C^1 regularity:

$$\begin{aligned} u_N &= -u \cdot n_0, \quad u_T = u + u_N n_0, \quad \sigma_N(u) = -(\sigma(u)n) \cdot n_0, \\ \sigma_T(u) &= \sigma(u)n + \sigma_N(u)n_0, \end{aligned}$$

where n_0 is the unit outward normal to the obstacle (see Fig. 1). Denoting by g the initial normal gap between the solid and the rigid obstacle, the unilateral contact condition is expressed by the following complementary condition:

$$u_N - g \leq 0, \quad \sigma_N(u) \leq 0, \quad (u_N - g)\sigma_N(u) = 0, \quad (2)$$

while the static Coulomb friction condition is expressed as follows for \mathcal{F} the friction coefficient:

$$|\sigma_T| \leq \mathcal{F} \sigma_N, \quad \text{if } u_T \neq 0 \text{ then } \sigma_T = -\mathcal{F} \sigma_N \frac{u_T}{|u_T|}. \quad (3)$$

A classical weak formulation (see [14]) can be obtained introducing

$$V = \{v \in H^1(\Omega; \mathbb{R}^d) : v = 0 \text{ on } \Gamma_D\},$$

$$K = \{v \in V : v_N - g \leq 0 \text{ on } \Gamma_C\},$$

$$a(u, v) = \int_{\Omega} \mathcal{A} \varepsilon(u) : \varepsilon(v) dx, \quad l(v) = \int_{\Omega} f \cdot v dx + \int_{\Gamma_N} k \cdot v d\Gamma.$$

$$W_N = \{v \in L^2(\Gamma_C) : \exists w \in V, v = w_N\},$$

$$W_T = \{v \in L^2(\Gamma_C; \mathbb{R}^{d-1}) : \exists w \in V, v = w_T\},$$

$$W = W_N \times W_T, \quad j(s, v) = \langle s, |v_T| \rangle_{W'_N, W_N}.$$

The space W_N (resp. W_T) is the space of the normal (resp. tangential) traces and W'_N (resp. W'_T) will denote its topological dual space. Note that the duality pairing $\langle \cdot, \cdot \rangle_{W'_N, W_N}$ reduces to the integral on Γ_C when the contact stress $\sigma_N(u)$ is sufficiently regular. Problem (1)–(3) is then formally equivalent to the variational inequality (see [14])

$$\begin{cases} \text{Find } u \in K \text{ satisfying} \\ a(u, v - u) + j(-\mathcal{F} \sigma_N(u), u) - j(-\mathcal{F} \sigma_N(u), v) \geq l(v - u), \quad \forall v \in K. \end{cases} \quad (4)$$

A generalized Newton's method cannot be directly applied to this formulation. Classically, one introduces some multipliers representing the contact and friction stresses and obtains the following hybrid formulation (see [4] for instance):

$$\begin{cases} \text{Find } u \in V, \lambda_N \in \Lambda_N \text{ satisfying} \\ a(u, v) - \langle \lambda_N, v_N \rangle_{W'_N, W_N} - \langle \lambda_T, v_T \rangle_{W'_T, W_T} = l(v), \quad \forall v \in V, \\ \langle \mu_N - \lambda_N, u_N - g \rangle_{W'_N, W_N} \geq 0, \quad \forall \mu_N \in \Lambda_N, \\ \langle \mu_T - \lambda_T, u_T \rangle_{W'_T, W_T} \geq 0, \quad \forall \mu_T \in \Lambda_T(-\mathcal{F} \lambda_N), \end{cases} \quad (5)$$

with Λ_N the cone of non-positive contact stresses

$$\Lambda_N = \left\{ \lambda_N \in W'_N : \langle \lambda_N, v_N \rangle_{W'_N, W_N} \geq 0 \quad \forall v \in K \right\},$$

and $\Lambda_T(s)$ the set of admissible friction stresses defined by

$$\Lambda_T(s) = \left\{ \lambda_T \in W'_T : -\langle \lambda_T, v_T \rangle_{W'_T, W_T} + \langle s, |v_T| \rangle_{W'_N, W_N} \geq 0 \quad \forall v_T \in W_T \right\}.$$

In the case of Tresca friction, the (weakly) non-negative threshold $s \in W'_N$ is given and one considers the problem

$$\begin{cases} \text{Find } u \in V, \lambda_N \in \Lambda_N \text{ satisfying} \\ a(u, v) - \langle \lambda_N, v_N \rangle_{W'_N, W_N} - \langle \lambda_T, v_T \rangle_{W'_T, W_T} = l(v), \quad \forall v \in V, \\ \langle \mu_N - \lambda_N, u_N - g \rangle_{W'_N, W_N} \geq 0, \quad \forall \mu_N \in \Lambda_N, \\ \langle \mu_T - \lambda_T, u_T \rangle_{W'_T, W_T} \geq 0, \quad \forall \mu_T \in \Lambda_T(s). \end{cases} \quad (6)$$

The formulation (6) is the optimality system of the Lagrangian

$$\mathcal{L}(u, \lambda) = \frac{1}{2} a(u, u) - l(u) - j(s, u) - \langle \lambda, u_N - g \rangle_{W'_N, W_N} - I_{\Lambda_N}(\lambda_N) - I_{\Lambda_T(s)}(\lambda_T), \quad (7)$$

where I_{Λ_N} (resp. $I_{\Lambda_T(s)}$) is the indicator function of Λ_N (resp. $\Lambda_T(s)$) (see for instance [29] for more details). This last term means that the saddle point has to be chosen with the constraints $\lambda_N \in \Lambda_N$ and $\lambda_T \in \Lambda_T(s)$. The fact that the Lagrangian $\mathcal{L}(u, \lambda)$ is under constraint poses practical difficulties for the numerical resolution. This is one of the reason why augmented Lagrangian are considered.

The existence and uniqueness of the solution to Problem (6) is obtained by standard techniques (see [25] for instance). Unfortunately, as it is well known, the problem with Coulomb friction (5) is not a variational problem and thus do not derive from a Lagrangian. Some existence results for the problem with Coulomb friction have been proved in [22,15] with some reasonable assumptions on the regularity of the boundary and for a sufficiently small friction coefficient. The uniqueness of the solution for a small friction coefficient is an open problem. A criterion has been given in [35] which states that the uniqueness is reached under a kind of regularity condition on the solution and for a small enough friction coefficient. Examples of non-uniqueness has been exhibited in [18,19] for a large friction coefficient.

3. Finite element approximation

In the framework of the finite element method, different principles can be used to approximate a contact condition with or without friction (see [25,24] for instance). The contact condition can be applied on each finite element node of the contact boundary (or alternatively on the Gauss points of a quadrature method defined on the contact boundary). In that case, generally, the non-interpenetration condition is prescribed on each node. The contact condition can also be weakened by the use of a multiplier of lower degree and a contact condition on each finite element node of the multiplier. In that case, this is generally the non-positivity on each node of the multiplier which is considered. A third possibility, used more or less implicitly is some references (for instance in [29,38]) is to use directly a non constrained integral weak formulation of the contact condition. From the resolution with a Newton algorithm point of view, the two first alternatives are very close. Thus, the choice is made to test only the first and the third alternatives. In the following, we will denote by V^h a family of finite dimensional vector spaces (see [7]) indexed by h coming from a family \mathcal{T}^h of meshes of the domain Ω ($h = \max_{T \in \mathcal{T}^h} h_T$ where h_T is the diameter of T).

3.1. Nodal discretization of the contact condition

This is probably the most commonly used method to approximate the contact condition. It consists to prescribe the non-pene-

tration and friction conditions on each finite element node of a Lagrange finite element method. This *a priori* restricts V^h to be built from a Lagrange finite element method. This means that denoting $\phi_i(x)$, $i = 1..N_d$ the (scalar) shape functions of V^h and $a_i \in \mathbb{R}^d$ the corresponding finite element nodes, one has $\phi_i(a_j) = \delta_{ij}$ and the following Lagrange interpolation principle:

$$u^h(x) = \sum_{i=1}^{N_d} u^h(a_i) \phi_i(x).$$

Let us denote \mathcal{N}_{Γ_C} the set of node indices on the contact boundary. The approximation of Problem (5) with a nodal contact condition reads as:

$$\begin{cases} \text{Find } u^h \in V^h, \lambda_N^i \in \mathbb{R}, \lambda_T^i \in \mathbb{R}^{d-1}, i \in \mathcal{N}_{\Gamma_C}, \text{ satisfying} \\ a(u^h, v^h) - \sum_{i \in \mathcal{N}_{\Gamma_C}} \lambda_N^i v_N^h(a_i) - \sum_{i \in \mathcal{N}_{\Gamma_C}} \lambda_T^i \cdot v_T^h(a_i) = l(v^h), \quad \forall v \in V^h, \\ \lambda_N^i \leq 0, \quad u_N^h(a_i) - g(a_i) \leq 0, \lambda_N^i (u_N^h(a_i) - g(a_i)) = 0 \quad \forall i \in \mathcal{N}_{\Gamma_C}, \\ |\lambda_T^i| \leq -\mathcal{F} \lambda_N^i, \quad \text{if } u_T^h(a_i) \neq 0 \text{ then } \lambda_T^i = \mathcal{F} \lambda_N^i \frac{u_T^h(a_i)}{|u_T^h(a_i)|} \quad \forall i \in \mathcal{N}_{\Gamma_C}. \end{cases} \quad (8)$$

An algebraic formulation can be obtained by defining the stiffness matrix K and the right-hand side L as

$$K_{(i-1)d+j, (k-1)d+l} = a(\phi_i e_j, \phi_k e_l), \quad L_{(i-1)d+j} = \ell(\phi_i e_j), \\ 1 \leq i, k \leq N_d, \quad 1 \leq j, l \leq d,$$

where e_0, \dots, e_{d-1} is the canonical basis of \mathbb{R}^d . Let also U be the vector of degrees of freedom such that

$$U_{(i-1)d+j} = u^h(a_i) \cdot e_j,$$

and $N_i \in \mathbb{R}^{dN_d}$, $T_i \in \mathcal{M}_{dN_d, (d-1)}(\mathbb{R})$ for $i \in \mathcal{N}_{\Gamma_C}$ be defined such that

$$N_i^T U = u_N^h(a_i), \quad T_i^T U = u_T^h(a_i).$$

Let also B_N be the matrix whose lines are the vectors N_i , B_T be the matrix whose lines are the columns of T_i , Λ_N the vector of all λ_N^i and Λ_T the vector of all λ_T^i . Then the algebraic formulation of Problem (8) read as

$$\begin{cases} \text{Find } U \in \mathbb{R}^{dN_d}, \lambda_N^i \in \mathbb{R}, \lambda_T^i \in \mathbb{R}^{d-1}, i \in \mathcal{N}_{\Gamma_C}, \text{ satisfying} \\ KU - B_N^T \Lambda_N - B_T^T \Lambda_T = L, \\ \lambda_N^i \leq 0, \quad N_i^T U - g(a_i) \leq 0, \quad \lambda_N^i (N_i^T U - g(a_i)) = 0, \quad \forall i \in \mathcal{N}_{\Gamma_C}, \\ |\lambda_T^i| \leq -\mathcal{F} \lambda_N^i, \quad \text{if } T_i^T U \neq 0 \text{ then } \lambda_T^i = \mathcal{F} \lambda_N^i \frac{T_i^T U}{|T_i^T U|} \quad \forall i \in \mathcal{N}_{\Gamma_C}. \end{cases} \quad (9)$$

This is a classical result that Problem (8) admits a solution whatever the value of the friction coefficient $\mathcal{F} \geq 0$ and that this solution is unique for \mathcal{F} small enough of order $O(\sqrt{h})$ (see [16]).

It is also well known that in the two-dimensional case, Problem (9) is equivalent to a linear complementarity problem (LCP, see [12]). The three dimensional frictional problem cannot be expressed as a linear complementarity problem without a polyhedral approximation of the Coulomb friction cone [27]. However, it can be expressed as a second order cone linear complementarity problem (SOCLCP, see [23]).

3.2. Integral approximation of the contact condition

In that case, V^h is an arbitrary finite element space and, additionally, we denote by $W_N^h \subset L^2(\Gamma_D)$, $W_T^h \subset L^2(\Gamma_D; \mathbb{R}^{d-1})$, $W^h = W_N^h \times W_T^h$ some finite element spaces for the approximation of contact forces. We use the following equivalent expression of contact and friction condition (other choices are presented in the next sections):

$$\lambda_N^h = -(\lambda_N^h - r(u_N^h - g))_-, \quad (10)$$

$$\lambda_T^h = P_{B(0, \mathcal{F}(\lambda_N^h - r(u_N^h - g))_-)}(\lambda_T^h - ru_T^h), \quad (11)$$

where $r > 0$ is an augmentation parameter, $P_B(0, \rho)$ is the orthogonal projection on the closed ball of center 0 and radius ρ and $(\cdot)_-$ is the negative part ($(x)_- = 0$ for $x \geq 0$ and $(x)_- = -x$ for $x \leq 0$). In particular, these expressions can be derived from Alart–Curnier Augmented Lagrangian which will be described later on.

This allows to give the following approximation of Problem (5) with an unconstrained integral contact condition:

$$\begin{cases} \text{Find } u^h \in V^h, \lambda_N^h \in W_N^h \text{ and } \lambda_T^h \in W_T^h \text{ satisfying} \\ a(u^h, v^h) - \int_{\Gamma_C} \lambda_N^h v_N^h d\Gamma - \int_{\Gamma_C} \lambda_T^h \cdot v_T^h d\Gamma = l(v^h), \quad \forall v \in V^h, \\ -\frac{1}{r} \int_{\Gamma_C} (\lambda_N^h + (\lambda_N^h - r(u_N^h - g))_-) \mu_N^h d\Gamma \\ -\frac{1}{r} \int_{\Gamma_C} (\lambda_T^h - P_{B(0, \mathcal{F}(\lambda_N^h - r(u_N^h - g))_-)}(\lambda_T^h - ru_T^h)) \cdot \mu_T^h d\Gamma = 0 \quad \forall \mu^h \in W^h. \end{cases} \quad (12)$$

The existence and uniqueness of a solution to Problem (12) is discussed in [29]. The inf–sup (or LBB) condition

$$\inf_{\mu^h \in W^h} \sup_{v^h \in V^h} \frac{\int_{\Gamma_C} \mu^h \cdot v^h d\Gamma}{\|v^h\|_V \|\mu^h\|_W} \geq \gamma > 0, \quad (13)$$

is necessary to ensure the uniqueness of the multiplier. A constant $\gamma > 0$ independent of h is necessary to obtain an optimal numerical convergence. Note that, unlike Problems (8) and (12) is not equivalent to a LCP or a SOCLCP problem.

3.3. Description of the numerical experiments

The aim of the numerical experiments presented in the next sections is to compare the efficiency of the different generalized Newton's method on a small set of simple contact situations in two and three dimensions. Three different two-dimensional situations are considered: an elastic punch clamped on its top submitted to two different prescribed vertical compressions and in contact with a flat rigid obstacle at the bottom (see the mesh in Fig. 2 and the two deformations in Fig. 3 and a disk with no Dirichlet condition ($\Gamma_D = \emptyset$) submitted to a vertical volumic load and also in contact with a flat rigid obstacle at the bottom (mesh and deformation are also in Fig. 2 and Fig. 3, respectively). Both are in plane stress approximation. The deformations, especially for the second case, are obviously non-physical since they are too important for the linear elasticity model to remain valid. This is done intentionally here to test the algorithms both with a small and a large real contact area and to test the influence of the size of the load.

In 3D, we consider the counterpart situations for a 3D punch and a sphere (see Figs. 4 and 5 for the meshes and the deformations, respectively).

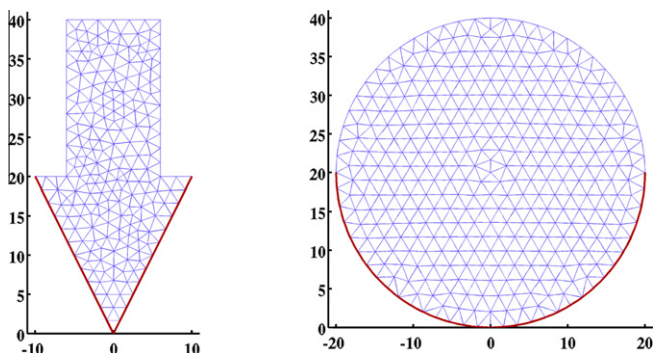


Fig. 2. 2D meshes.

For all the experiment a quadratic isoparametric finite element is considered. The number of displacement degrees of freedom for the different meshes used in the numerical experiments are summarized in Table 1.

Note that the two cases corresponding to the disc and the sphere are semi-coercive cases in the sense that the bilinear form $a(\cdot, \cdot)$ is not strictly coercive due to free rigid body motions. Nevertheless, there exists a unique solution because the load is chosen compatible.

The different parameters used for the model are a Poisson ratio $\nu = 0.25$, a Young modulus $E = 2660$ MPa in 2D cases and $E = 2500$ MPa in 3D cases and a uniform (excessive) volumic load of 20×10^6 N/m³ for the disc and the sphere.

The initial iteration for the different (generalized Newton or Uzawa) algorithms is the reference configuration (zero displacement). A very basic line search is considered for generalized Newton's method which can be summarized as follows:

- Test a full Newton step. Since the norm of the residual is greater than the one of the previous iteration, divide the step by a factor 2, with a minimal step of 10^{-10} .
- If there is more than three Newton iterations with a decreasing of the residual less than 1% then allow a step with an increase of the residual of a factor 2.

Let us recall that generalized Newton's method means that Newton's method is applied on a discrete system like (12) which is not C^1 -regular but only Lipschitz-continuous and piecewise C^1 -regular. No special treatment is applied when a point of non-differentiability is reached by Newton's method. Such a point is on the frontier between two or more zones of differentiability and the tangent matrix corresponding to one of these zones is selected arbitrarily.

In the sections that follow, the graphs represent the number of iterations of Newton's method to achieve convergence. Each of the six graphs of a set corresponds to one of the six situations described previously. The number of iterations is limited to 100. A number of iteration of 100 means that the algorithm has been stopped.

4. Alart–Curnier generalized Newton's method

Most of generalized Newton's methods used to solve contact problem in elasticity are based on the augmented Lagrangian formulation presented in [2,3] on the finite element approximation of the contact problem with a nodal contact with friction condition. This formulation can be interpreted as a proximal Lagrangian in the sense of Rockafellar for the problem with Tresca friction (see [29,36] for more details). In this section, the presentation is made for a contact condition with Coulomb friction for both a nodal and an integral approximation.

First, let us recall that an augmented Lagrangian formulation is only possible for the frictionless case or for the case with Tresca friction since the Coulomb friction law do not derive from a potential. However, the optimality system of the augmented Lagrangian is classically extended to the Coulomb friction law.

In the continuous framework, the augmented Lagrangian has the following expression for the problem with Tresca friction:

$$\begin{aligned} \mathcal{L}_r(u, \lambda) = & \frac{1}{2} a(u, u) - \ell(u) - \frac{1}{2r} \int_{\Gamma_C} |\lambda|^2 d\Gamma + \frac{1}{2r} \int_{\Gamma_C} (\lambda_N - r(u_N \\ & - g))_-^2 d\Gamma + \frac{1}{2r} \int_{\Gamma_C} |P_{B(0, s)}(\lambda_T - ru_T)|^2 d\Gamma, \end{aligned}$$

where s is the Tresca threshold. Note that this expression is only valid for sufficiently regular Tresca threshold and friction stresses in

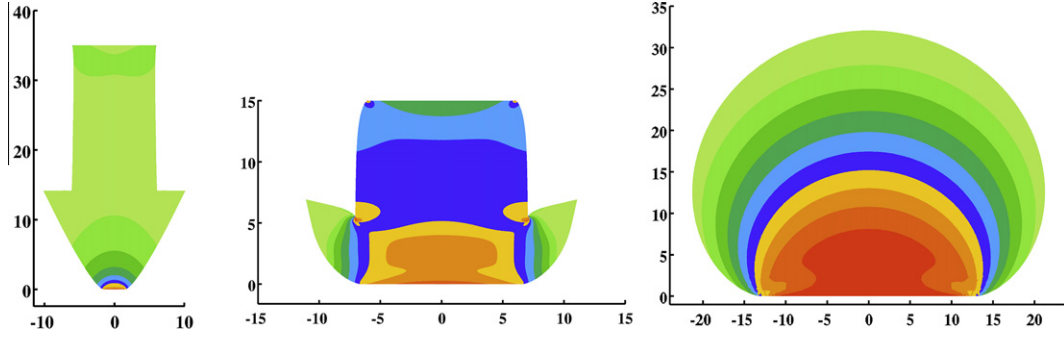


Fig. 3. 2D deformed configurations with color plot of the Von Mises stress (and with friction, $\mathcal{F} = 1$).

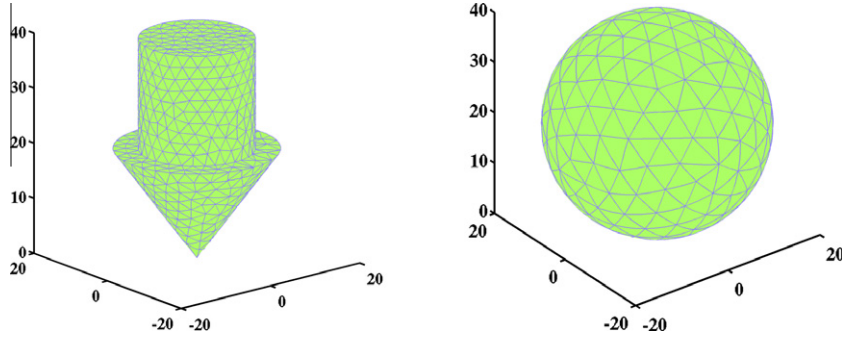


Fig. 4. 3D meshes.

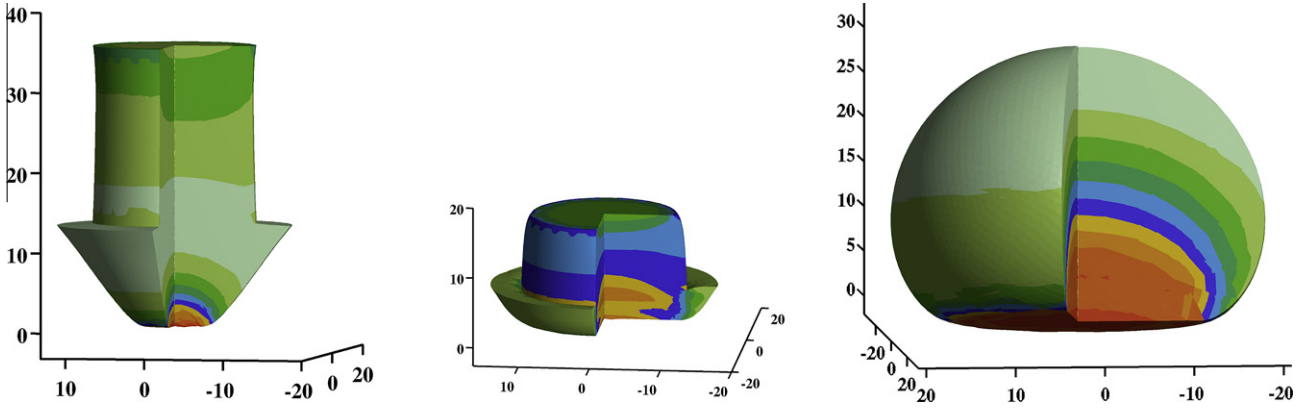


Fig. 5. 3D deformed configurations with color plot of the Von Mises stress (and with friction, $\mathcal{F} = 1$).

Table 1
Number of displacement degrees of freedom for the different meshes used.

2D punch	Disc	3D punch	Sphere
$h = 4$	424	$h = 5$	1182
$h = 1$	6226	$h = 3$	4305
$h = 0.25$	97,390	$h = 1.7$	24,375
		$h = 20$	75
		$h = 6$	2232
		$h = 2.3$	26,259

$L^2(\Gamma_C)$. The advantage compared to (7) is that there is no constraint on the augmented Lagrangian variables. With $H = L^2(\Gamma_C; \mathbb{R}^d)$, the optimality system (still for Tresca friction) read as:

$$\begin{cases} a(u, v) = \ell(v) + \int_{\Gamma_C} (\lambda_N - r(u_N - g))_- v_N d\Gamma + \int_{\Gamma_C} P_{B(0,s)} (\lambda_T - ru_T) \cdot v_T d\Gamma \quad \forall v \in V, \\ -\frac{1}{r} \int_{\Gamma_C} (\lambda_N + (\lambda_N - r(u_N - g))_-) \mu_N d\Gamma - \frac{1}{r} \int_{\Gamma_C} (\lambda_T - P_{B(0,s)} (\lambda_T - ru_T)) \cdot \mu_T d\Gamma = 0 \quad \forall \mu \in H. \end{cases}$$

This optimality system can be adapted to the Coulomb friction law simply by replacing the threshold s by $\mathcal{F}\lambda_N$. It is in fact more usual to replace it by the so called augmented multiplier

$-\mathcal{F}(\lambda_N - r(u_N - g))_-$ (see [3]). It is also possible, as for instance in [1], to simplify the first line of the optimality system by using the two other lines and then to obtain the less symmetric following formulation of the problem:

$$\begin{cases} a(u, v) = \ell(v) + \int_{\Gamma_C} \lambda_N v_N d\Gamma + \int_{\Gamma_C} \lambda_T \cdot v_T d\Gamma \quad \forall v \in V, \\ -\frac{1}{r} \int_{\Gamma_C} (\lambda_N + (\lambda_N - r(u_N - g))_-) \mu_N d\Gamma \\ -\frac{1}{r} \int_{\Gamma_C} (\lambda_T - P_{B(0,\mathcal{F}(\lambda_N - r(u_N - g))_-)} (\lambda_T - ru_T)) \cdot \mu_T d\Gamma = 0 \quad \forall \mu \in H. \end{cases} \quad (14)$$

It is easy to check that this formulation is equivalent to Problem (5) when the solution is assumed to be regular enough (λ_N and λ_T in H).

4.1. Nodal approximation of the contact condition, unsymmetric version

The first numerical experiment concerns the approximation of the less symmetric system (14) with a nodal approximation of

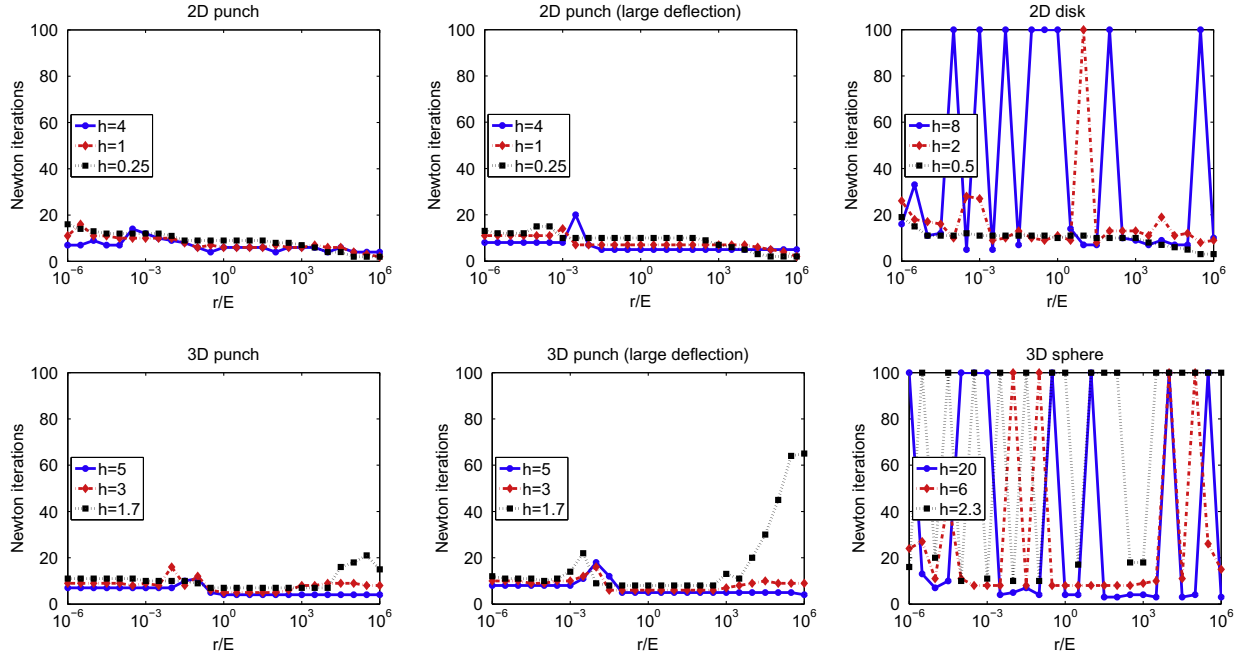


Fig. 6. Number of iterations for Alart–Curnier generalized Newton's method with nodal frictionless contact approximation. Unsymmetric version.

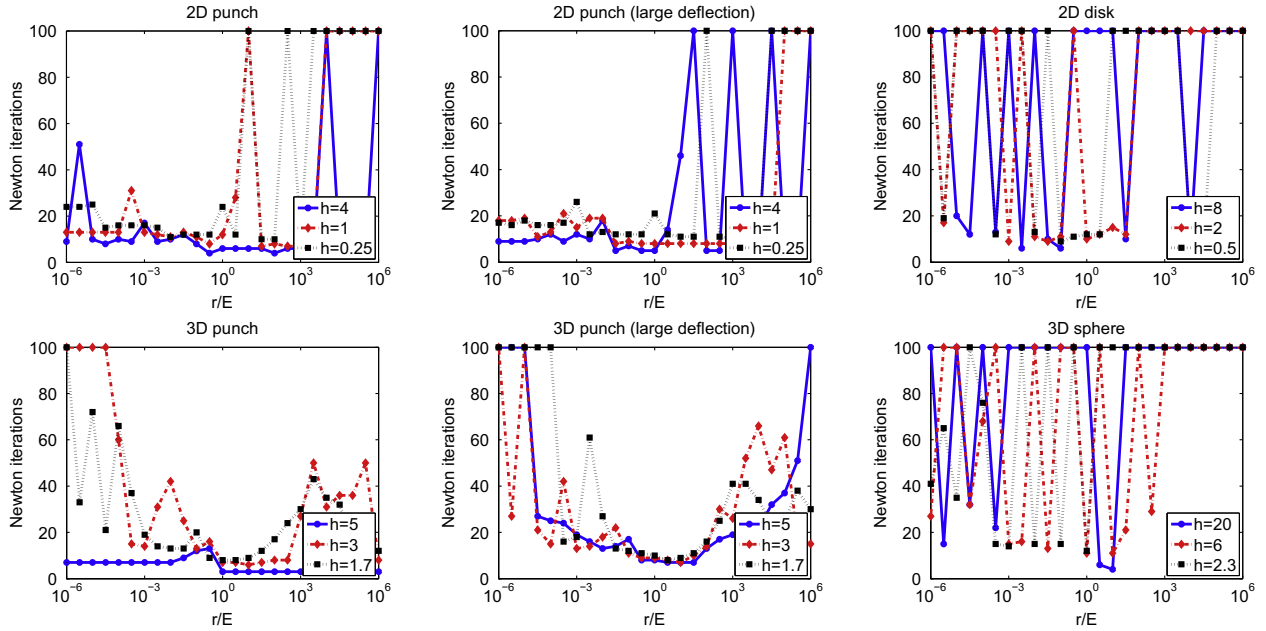


Fig. 7. Number of iterations for Alart–Curnier generalized Newton's method with nodal contact with friction approximation. Unsymmetric version.

the contact and friction conditions. The generalized Newton's method is applied on the following system:

$$\begin{cases} a(u^h, v^h) = \ell(v^h) + \sum_{a \in \mathcal{N}_{\Gamma_C}} \lambda_N^a v_N^h(a) + \sum_{a \in \mathcal{N}_{\Gamma_C}} \lambda_T^a \cdot v_T^h(a) & \forall v^h \in V^h, \\ -\frac{1}{r} (\lambda_N^a + (\lambda_N^a - r(u_N^h(a) - g(a)))_-) = 0, & \forall a \in \mathcal{N}_{\Gamma_C}, \\ -\frac{1}{r} (\lambda_T^a - P_{B(0, \mathcal{F}(\lambda_N^a - r(u_N^h(a) - g(a)))_-)}(\lambda_T^a - r u_T^h(a))) = 0, & \forall a \in \mathcal{N}_{\Gamma_C}. \end{cases}$$

The results are shown in Fig. 6 for the frictionless case ($\mathcal{F} = 0$) and in Fig. 7 for the frictional one ($\mathcal{F} = 1$). Concerning the frictionless case (Fig. 6), in the coercive cases (2D and 3D punches) the number of iterations is less than 20 regardless the value of the augmentation parameter r , except for one experiment corresponding

to the finest mesh of the 3D punch with large deflection. Note that there is a slight increase of the number of iterations when the mesh becomes finer which is probably due to the deterioration of the conditioning of the system. Conversely, there is no dependence of the iterations number on the size of the load. Nevertheless, in the non-coercive cases, the method is performing poorly, especially in the case of coarse meshes.

Concerning now the coercive cases with friction (Fig. 7), the convergence is less uniform with respect to the augmentation parameter than for the frictionless cases. However, the convergence occurs in less than 20 iterations for a value of the augmentation parameter close to the value of the Young modulus. In the non-coercive cases, the convergence occurs only for a few experiments.

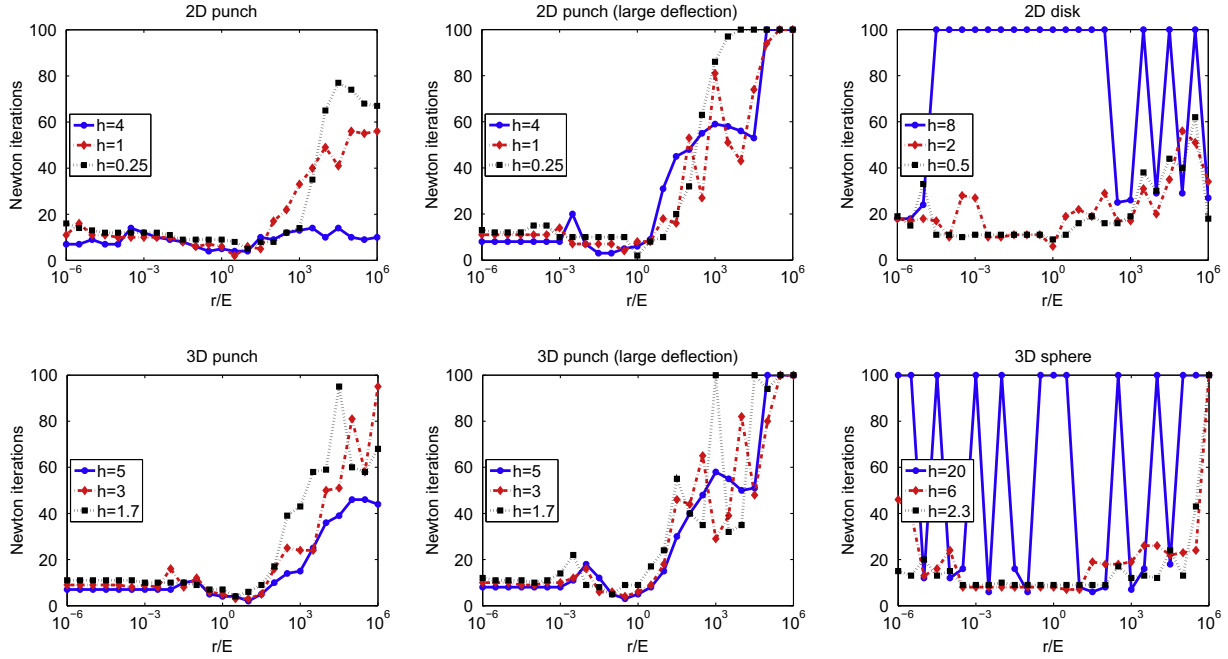


Fig. 8. Number of iterations for Alart–Curnier generalized Newton's method with nodal frictionless contact approximation. Symmetric version.

4.2. Nodal approximation of the contact condition, symmetric version

In this section, instead of using the less symmetric system, we test the system without the simplification of the first line. This leads to the following formulation of the nodal approximation:

$$\begin{cases} a(u^h, v^h) = \ell(v^h) + \sum_{a \in \mathcal{N}_{\Gamma_C}} (\lambda_N^a - r(u_N^h(a) - g(a)))_- v_N^h(a) \\ \quad + \sum_{a \in \mathcal{N}_{\Gamma_C}} P_{B(0, \mathcal{F}(\lambda_N^a - r(u_N^h(a) - g(a))))_-} (\lambda_T^a - ru_T^h(a)) \cdot v_T^h(a) \quad \forall v^h \in V^h, \\ -\frac{1}{r} (\lambda_N^a + (\lambda_N^a - r(u_N^h(a) - g(a)))_-) = 0, \quad \forall a \in \mathcal{N}_{\Gamma_C}, \\ -\frac{1}{r} (\lambda_T^a - P_{B(0, \mathcal{F}(\lambda_N^a - r(u_N^h(a) - g(a))))_-} (\lambda_T^a - ru_T^h(a))) = 0, \quad \forall a \in \mathcal{N}_{\Gamma_C}. \end{cases}$$

The corresponding numerical experiments are presented in Figs. 8 and 9. There is a clear deterioration of the convergence for the frictionless case (Fig. 8) compared to the less symmetric system except for the non-coercive case. In the frictional case, shown in Fig. 9 the deterioration is less important but is still present especially in 3D. We did not made further investigation to identify the reasons for this deterioration. The fact that there is twice as much projections in the symmetric formulation can be a reason of this deterioration.

4.3. Integral approximation of the contact condition

We turns now to the integral approximation of the contact with friction condition. The approximation of the less symmetric system (14) leads to the following system:

$$\begin{cases} a(u^h, v^h) = \ell(v^h) + \int_{\Gamma_C} \lambda_N^h v_N^h d\Gamma + \int_{\Gamma_C} \lambda_T^h \cdot v_T^h d\Gamma \quad \forall v^h \in V^h, \\ -\frac{1}{r} \int_{\Gamma_C} (\lambda_N^h + (\lambda_N^h - r(u_N^h - g))_-) \mu_N^h d\Gamma = 0 \quad \forall \mu^h \in W^h, \\ -\frac{1}{r} \int_{\Gamma_C} (\lambda_T^h - P_{B(0, \mathcal{F}(\lambda_N^h - r(u_N^h - g))_-)} (\lambda_T^h - ru_T^h)) \cdot \mu_T^h d\Gamma = 0 \quad \forall \mu^h \in W^h. \end{cases} \quad (15)$$

An existence result of a solution to Problem (15) for arbitrary $r > 0$ and $\mathcal{F} \geq 0$ and a uniqueness result for both r and \mathcal{F} small enough can be found in [29].

The corresponding numerical experiments are presented in Figs. 10 and 11 for the cases without or with friction, respectively.

The method is more sensitive to the value of the augmentation parameter in the frictionless case if one compare with Fig. 6 for the nodal approximation, except for the non-coercive case which behaves slightly better. For the case with friction, results are more similar. However, one may conclude that globally, Newton's method on the nodal discretization is more robust.

Now, still with an integral approximation of the contact with friction condition, it is also possible to consider the more symmetric system:

$$\begin{cases} a(u^h, v^h) = \ell(v^h) + \int_{\Gamma_C} (\lambda_N^h - r(u_N^h - g))_- v_N^h d\Gamma \\ \quad + \int_{\Gamma_C} P_{B(0, -\mathcal{F}(\lambda_N^h - r(u_N^h - g))_-)} (\lambda_T^h - ru_T^h) \cdot v_T^h d\Gamma \quad \forall v^h \in V^h, \\ -\frac{1}{r} \int_{\Gamma_C} (\lambda_N^h + (\lambda_N^h - r(u_N^h - g))_-) \mu_N^h d\Gamma = 0 \quad \forall \mu^h \in W^h, \\ -\frac{1}{r} \int_{\Gamma_C} (\lambda_T^h - P_{B(0, \mathcal{F}(\lambda_N^h - r(u_N^h - g))_-)} (\lambda_T^h - ru_T^h)) \cdot \mu_T^h d\Gamma = 0 \quad \forall \mu^h \in W^h. \end{cases}$$

The number of Newton iterations are shown in Fig. 12 for the frictional case. The convergence is very similar to the one of the unsymmetric version with only a slight degradation in 3D cases. The frictionless case being similar, the graphs are not shown for shortness.

5. Simo–Laursen method

5.1. The original method

The Simo–Laursen method corresponds to an Uzawa algorithm (in the case of Tresca friction). Still for shortness, only the integral approximation of contact is considered. The nodal approximation give similar results. The first step, for a contact stress $\lambda^{h,i}$ given is to find $u^{h,i+1}$ solution to

$$\begin{aligned} a(u^{h,i+1}, v^h) = & \ell(v^h) + \int_{\Gamma_C} (\lambda_N^{h,i} - r(u_N^{h,i+1} - g))_- v_N^h d\Gamma \\ & + \int_{\Gamma_C} P_{B(0, -\mathcal{F}(\lambda_N^{h,i} - r(u_N^{h,i+1} - g))_-)} (\lambda_T^{h,i} - ru_T^{h,i+1}) \cdot v_T^h d\Gamma \quad \forall v^h \in V^h. \end{aligned}$$

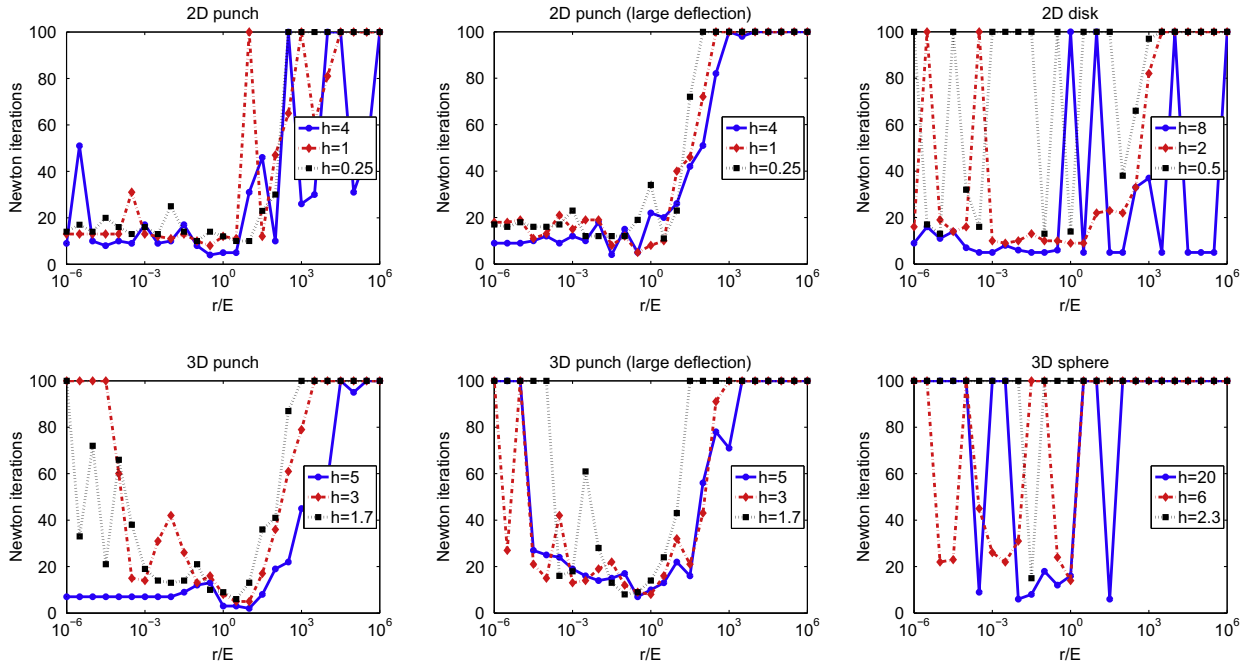


Fig. 9. Number of iterations for Alart–Curnier generalized Newton's method with nodal contact with friction approximation. Symmetric version.

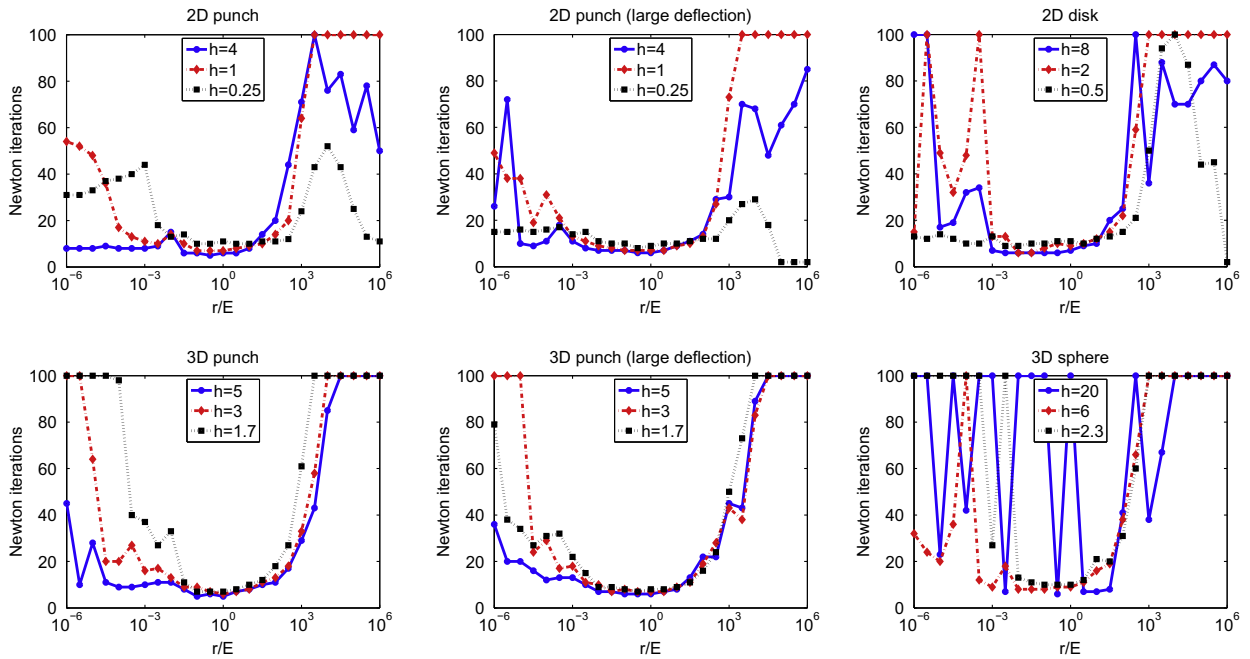


Fig. 10. Number of iterations for Alart–Curnier generalized Newton's method with integral frictionless contact approximation. Unsymmetric version.

This first step is solved with a generalized Newton's method. Then, the second step consists in computing $\lambda^{h,i+1}$ thanks to

$$\begin{aligned} \int_{\Gamma_c} (\lambda_N^{h,i+1} + (\lambda_N^{h,i} - r(u_N^{h,i+1} - g))_-) \mu_N^h d\Gamma &= 0 \quad \forall \mu^h \in W^h, \\ \int_{\Gamma_c} (\lambda_T^{h,i+1} - P_{B(0, -\mathcal{F}(\lambda_N^{h,i} - r(u_N^{h,i+1} - g))_-)}(\lambda_T^{h,i} - r u_T^{h,i+1})) \cdot \mu_T^h d\Gamma &= 0 \quad \forall \mu^h \in W^h. \end{aligned}$$

In the frictionless case, Figs. 13 and 14 present the total number of Newton iterations (summed over the Uzawa iterations) and the number of Uzawa iterations, respectively. The analysis of the two

series of graphs indicates that the number of Newton iterations inside an Uzawa iteration increases with the augmentation parameter r . This is a classical result when the contact condition is approached by penalization which is found here certainly because of the similarity between this formulation and the penalized one. Conversely, the largest is the augmentation parameter, the fastest the Uzawa method converges (on the graphs, remember that the algorithm is stopped after 100 cumulated Newton iterations, whatever the number of Uzawa iterations). The main difficulty of the Simo–Laursen method is to find the good compromise to allow both a reasonable number of iterations for Newton's method and

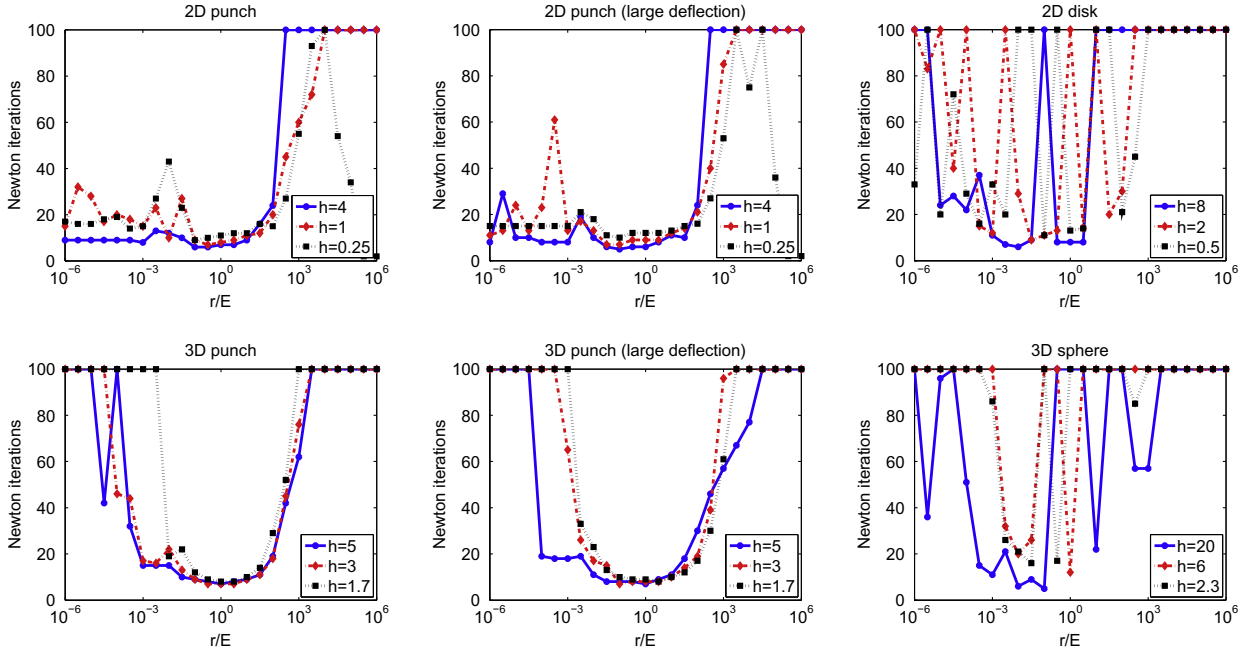


Fig. 11. Number of iterations for Alart–Curnier generalized Newton's method with integral contact with friction approximation. Unsymmetric version.

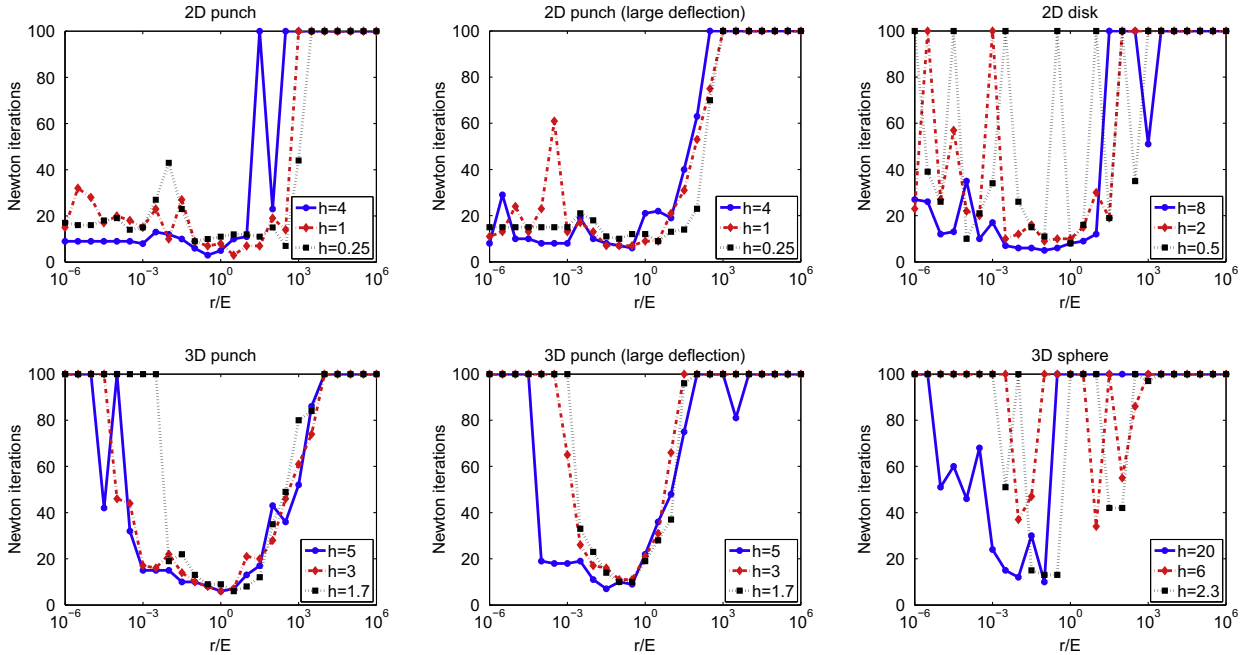


Fig. 12. Number of iterations for Alart–Curnier generalized Newton's method with integral contact with friction approximation. Symmetric version.

for the Uzawa algorithm. Figs. 13 and 14 shows that this may not be easy. The difficulty of the convergence of Newton's method for the penalized problem is addressed in some works such as [43] where the contact condition is modified in the first Newton iterations in order to facilitate the convergence. These techniques can also be applied to the Simo–Laursen method. However, this difficulty is a drawback of the method. The same conclusion can be applied to the case with friction on analyzing Fig. 15. Note also the significant sensitivity to the size of the load and the shift of the optimal augmentation parameter when the mesh is refined which have not been observed for Alart–Curnier method.

5.2. Using De Saxcé's projection

Another possibility, coming from De Saxcé bipotential theory (see [13,29]) is to replace the two separate projections (10) and (11) for the contact and friction conditions by the following single one:

$$\lambda^h = P_{\Lambda_{\mathcal{F}}}(\lambda^h - r((u_N^h - g) - \mathcal{F}|u_T^h|)n_0 + u_T^h), \quad (16)$$

where $\Lambda_{\mathcal{F}}$ is the friction cone defined by

$$\Lambda_{\mathcal{F}} = \{\lambda \in \mathbb{R}^d : |\lambda_T| \leq -\mathcal{F}\lambda_N\},$$

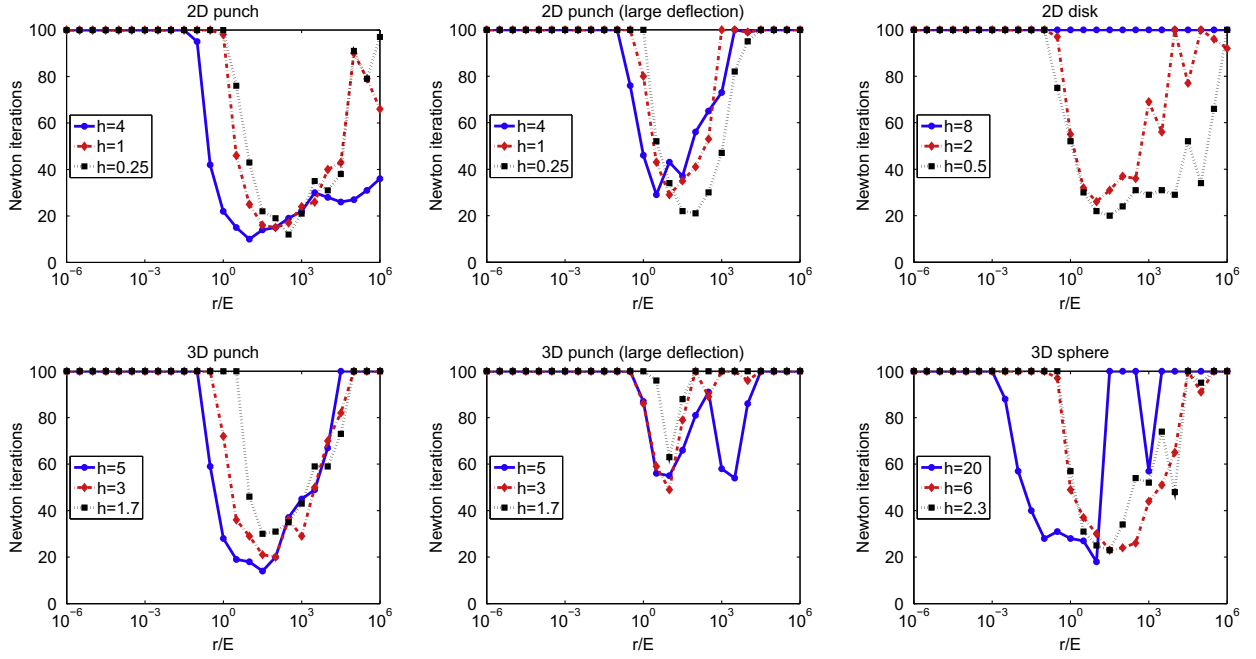


Fig. 13. Cumulative number of Newton iterations for the Simo–Laursen method with integral frictionless contact approximation.

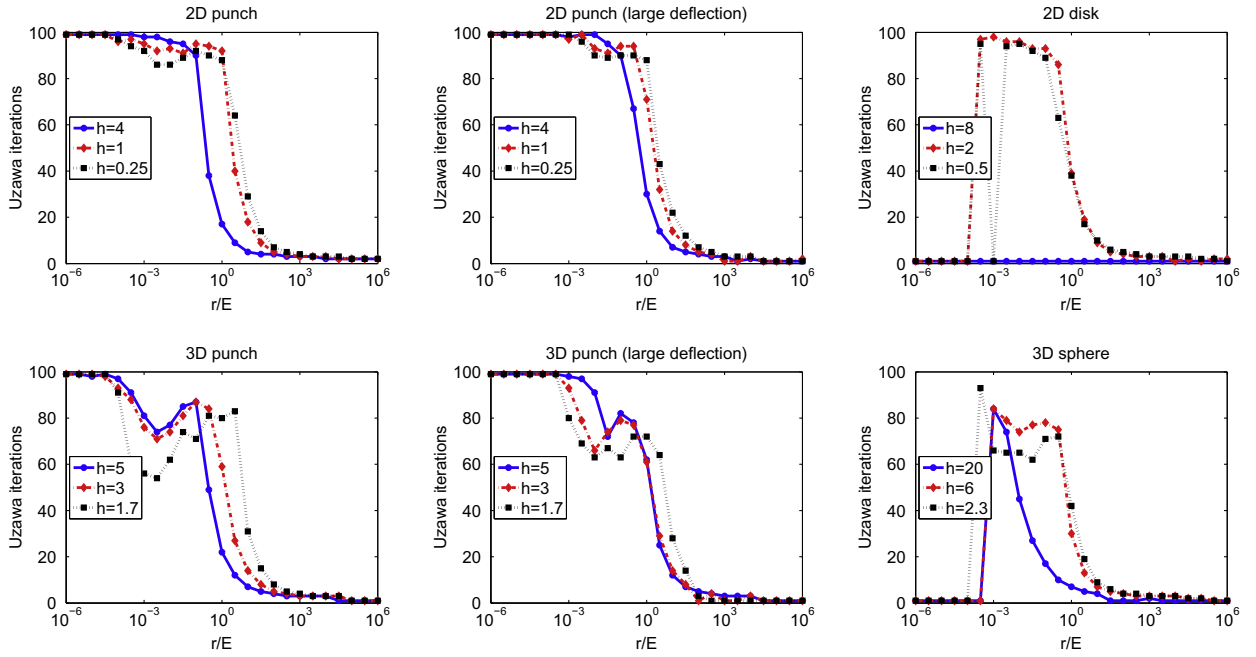


Fig. 14. Number of Uzawa iterations for the Simo–Laursen method with integral frictionless contact approximation.

and P_{Λ_F} the orthogonal projection which reads

$$P_{\Lambda_F}(\lambda) = \begin{cases} 0 & \text{if } \mathcal{F}|\lambda_T| \leq \lambda_N, \\ \lambda & \text{if } |\lambda_T| \leq -\mathcal{F}\lambda_N, \\ \frac{\lambda_N - \mathcal{F}|\lambda_T|}{\mathcal{F}^2 + 1} (n_0 - \mathcal{F} \frac{\lambda_T}{|\lambda_T|}) & \text{otherwise.} \end{cases}$$

Using this unique projection, the first step of Uzawa's algorithm corresponds now to find $u^{h,i+1}$ solution to

$$a(u^{h,i+1}, v^h) = \ell(v^h) + \int_{\Gamma_C} P_{\Lambda_F}(\lambda^{h,i} - r((u_N^{h,i+1} - g) - \mathcal{F}|u_T^{h,i+1}|)n_0 + u_T^{h,i+1})) \cdot \mu d\Gamma, \\ + u_T^{h,i+1} \cdot \nu d\Gamma,$$

for a given $\lambda^{h,i}$, and the second step corresponds to compute $\lambda^{h,i+1}$ thanks to

$$\int_{\Gamma_C} (\lambda^{h,i+1} - P_{\Lambda_F}(\lambda^{h,i} - r((u_N^{h,i+1} - g) - \mathcal{F}|u_T^{h,i+1}|)n_0 + u_T^{h,i+1})) \cdot \mu d\Gamma \\ = 0 \quad \forall \mu^h \in W^h.$$

A significant aspect of this formulation is the non-differentiability of the term $|u_T^{h,i+1}|$ occurring for $u_T^{h,i+1} = 0$ which corresponds of course to non-improbable situations (since it corresponds to a sticking solution). One may consider that this could affect the convergence of Newton's method since order two of convergence can be lost at non-differentiable points (see [33,41]). Such a

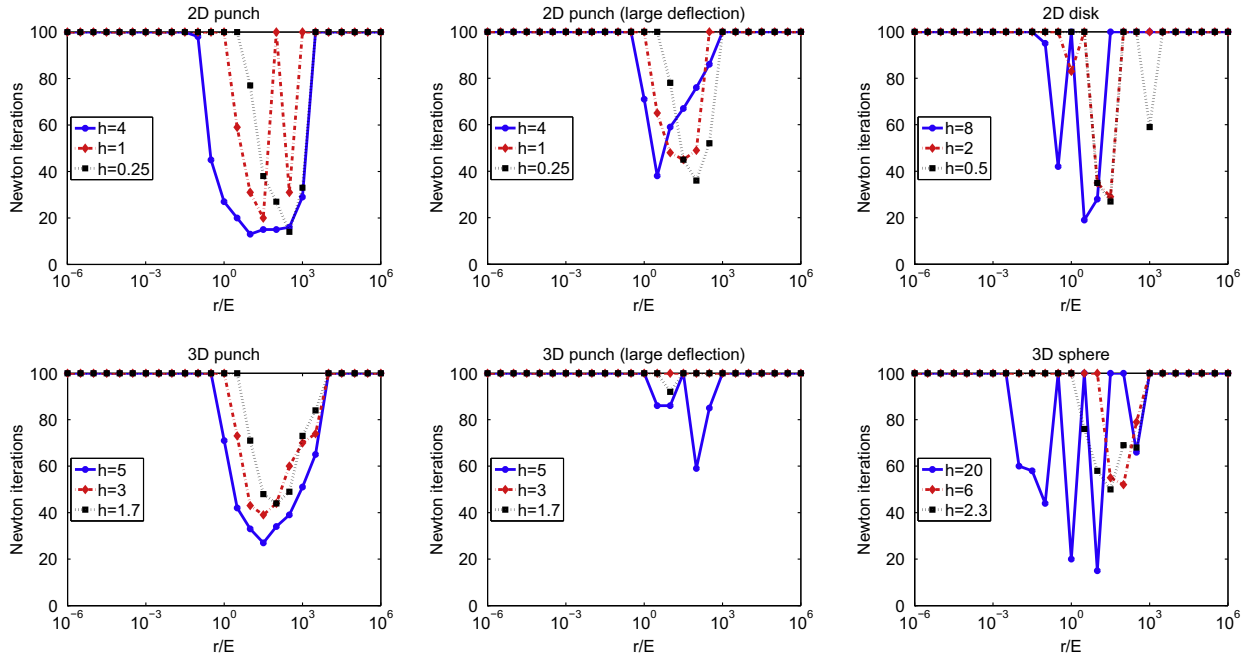


Fig. 15. Cumulative number of Newton iterations for the Simo–Laursen method with integral contact with friction approximation.

degradation of the convergence is not clearly visible on graphs of Fig. 16 which represent the case with friction (De Saxcé's projection has of course no interest in the frictionless case). However, we can conclude that the use of De Saxcé's projection do not improve the convergence here.

Note also that the use of De Saxcé's projection has also been tested on Alart–Curnier generalized Newton's method with also very few differences on the number of iterations for convergence. Numerical results are again not presented for shortness of the paper.

6. Nitsche's method adapted to contact with friction

Original Nitsche's method [32] allows to prescribe a Dirichlet condition on a boundary in a consistent way without the use of a multiplier. The extension to the contact condition is a recent concern. In [17] an extension to bilateral (persistent) contact is proposed. In [42] the method is extended to large strain bilateral frictionless contact. In [9] the method is extended to unilateral contact and a numerical analysis is performed which shows the optimality of the method. An interesting result of [9] is the full optimality of the *a priori* error estimate without any additional assumption on the solution than having the appropriate regularity. This is all the more remarkable that this is the first approximation of contact condition in elasticity for which an optimal *a priori* error estimate has been proven. See for instance [20] and the references there in for *a priori* error estimates of standard approximation of contact.

6.1. Symmetric version of Nitsche's method

We present here the Nitsche-based method presented in [9] and extended to Coulomb friction. For frictionless problem, the tangent system is symmetric (of course there is an unavoidable unsymmetric term when Coulomb friction is considered). In the formalism of the rest of the paper, the corresponding problem reads as

$$\begin{cases} \text{Find } u^h \in V^h \text{ satisfying} \\ a(u^h, v^h) - \frac{1}{r} \int_{\Gamma_C} (\sigma(u)n) \cdot (\sigma(v)n) d\Gamma - \frac{1}{r} \int_{\Gamma_C} (\sigma_N(u^h) - r(u_N^h - g))_+ (\sigma_N(v) - r v_N) d\Gamma \\ + \frac{1}{r} \int_{\Gamma_C} P_{B(0, \mathcal{F}(\sigma_N(u^h) - r(u_N^h - g))_+)} (\sigma_T(u^h) - r u_T^h) \cdot (\sigma_T(v^h) - r v_T^h) d\Gamma = l(v^h), \forall v \in V^h. \end{cases} \quad (17)$$

There is at least two main advantages of Nitsche's method. It leads to a weak formulation which consists in an equation (not an inequality) on which a generalized Newton's method can be directly applied. Similarly to a penalized formulation, the only unknown is the displacement field. The method is consistent which means that the non-penetration condition holds for any parameter $r > 0$. However, the parameter r should be chosen sufficiently large to keep the coercivity of the tangent matrix (see [9]). In fact the resolution of Problem (17) is very similar to the resolution of the first step of Simo–Laursen method presented in the previous section. A main difference is that the terms λ_N and λ_T are replaced by $\sigma_N(u^h)$ and $\sigma_T(u^h)$, respectively, and there is an additional term ensuring the consistency. Of course, the advantage compared to Simo–Laursen method is that no Uzawa iteration is needed. A small disadvantage of the method is the need of the material constitutive law for the computation of $\sigma(u)n$ which makes the implementation dependent on it.

The number of iterations of the corresponding generalized Newton's method are presented in Fig. 17 for the frictionless case and Fig. 18 for the case with friction. The result are very similar to the ones for the Simo–Laursen method with the same kind of difficulties. One may think that a strategy like the one proposed in [43] may be applied to improve the convergence.

Additionally here, when the parameter r is too small and the coercivity is lost, the Newton algorithm may converge toward a non-physical solution. This minimal parameter can be estimated by the computation of the smallest eigenvalue of the tangent matrix. The minimal parameter r was found to be about $15E$ for the largest meshes and $60E$ for the finest one in the presented numerical results.

6.2. An unsymmetric version of Nitsche's method

A rather simpler version can be obtained from (17) by remarking that the term $\sigma(v)n$ is not necessary for the consistency of the method. The following formulation is then obtained:

$$\begin{cases} \text{Find } u^h \in V^h \text{ satisfying} \\ a(u^h, v^h) + \int_{\Gamma_C} (\sigma_N(u^h) - r(u_N^h - g))_+ v_N d\Gamma \\ - \int_{\Gamma_C} P_{B(0, \mathcal{F}(\sigma_N(u^h) - r(u_N^h - g))_+)} (\sigma_T(u^h) - r u_T^h) \cdot v_T^h d\Gamma = l(v^h), \forall v \in V^h. \end{cases} \quad (18)$$

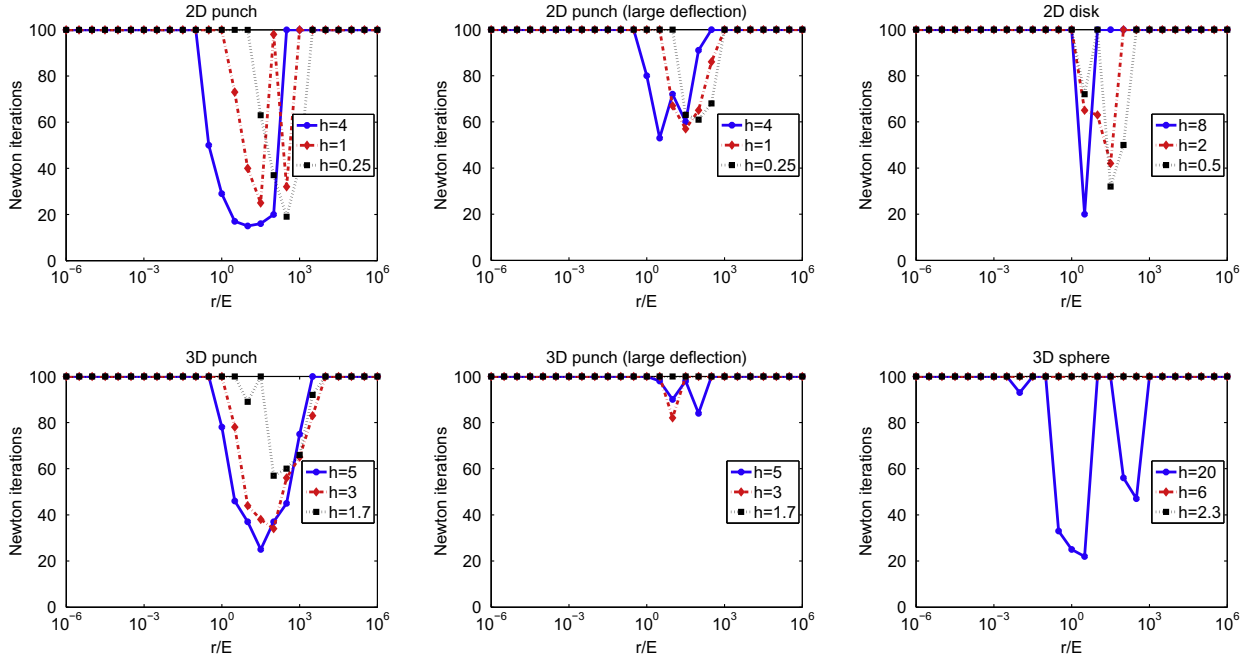


Fig. 16. Cumulative number of Newton iterations for the Simo–Laursen method using De Saxcé projection and with integral contact with friction approximation.

Compared to (17) this formulation is non-symmetric even in the frictionless case. The number of iterations of the corresponding generalized Newton's method are presented in Fig. 19 for the frictionless case and Fig. 20 for the case with friction. The convergence of Newton's method is greatly improved in comparison to the symmetric version. Another significant advantage is that the method seems to converge toward a physical solution for much smaller values of the parameter r . As a conclusion, for contact with friction problems, unsymmetric Nitsche's method behave much better than the symmetric one. This method is also discussed in [10].

7. A new generalized Newton's method based on augmented multipliers

Unlike previously presented methods, the method proposed here does not derive from an augmented Lagrangian. The principle is to consider an auxiliary variable instead of the multiplier representing the contact forces. For the contact condition, let us consider ξ_N a real variable whose negative part will store the normal gap and whose positive part will store the contact pressure:

$$(\xi_N)_+ = r(g - u_N), \quad (\xi_N)_- = -\lambda_N,$$

where $r > 0$ represent a scaling coefficient which allows the variable ξ to be described with an homogeneous unit (N/m²). The advantage of this auxiliary variable, the so-called augmented multiplier, is that the complementarity relation is automatically satisfied since ξ_N is either negative or positive. Similarly, for the friction condition, one can consider the augmented multiplier $\xi_T \in \mathbb{R}^{d-1}$ satisfying

$$P_{\bar{B}(0, \mathcal{F}(\xi_N)_-)}(\xi_T) = \lambda_T,$$

$$\xi_T - P_{\bar{B}(0, \mathcal{F}(\xi_N)_-)}(\xi_T) = -ru_T.$$

There is some similarities with the formulation of Ben Dhia and Zarroug [6,5] which is however rather different since, for instance, an additional discontinuous function with an active set strategy is used.

7.1. Nodal approximation of the contact condition

Introducing some augmented multipliers for each finite element nodes leads to the following system:

$$\begin{cases} a(u^h, v^h) = \ell(v^h) - \sum_{a \in \mathcal{N}_{\Gamma_C}} (\xi_N^a)_- v_N^h(a) + P_{\bar{B}(0, \mathcal{F}(\xi_N)_-)}(\xi_T^a) \cdot v_T^h(a) \quad \forall v^h \in V^h, \\ -\frac{1}{r}(r(u_N^h(a) - g(a)) + (\xi_N^a)_+) = 0, \quad \forall a \in \mathcal{N}_{\Gamma_C}, \\ -\frac{1}{r}(ru_T^h(a) + \xi_T^a - P_{\bar{B}(0, \mathcal{F}(\xi_N)_-)}(\xi_T^a)) = 0, \quad \forall a \in \mathcal{N}_{\Gamma_C}. \end{cases} \quad (19)$$

The discrete problem (19) is still equivalent to the standard nodal contact approximation (8). Thus, problems (8) and (19) share the same properties for existence and uniqueness of the solution.

The numerical experiments are presented in Figs. 21 and 22 for the contact without and with friction respectively. In the frictionless case, one sees that the number of iterations is nearly independent of the parameter r (except for the 3D cases and a large r). The novelty is that the method works also without difficulty for the non-coercive case. The situation is less favorable in the frictional case, especially for the 3D cases where the range of parameter r over which the number of Newton iterations is less than 20 is rather narrower compared to Fig. 7 for the Alart–Curnier approach (except for the non-coercive cases).

7.2. Nodal approximation of the contact condition with De Saxcé's projection

The same principle can be applied to De Saxcé's formulation of contact with friction (16) introducing $\xi \in \mathbb{R}^d$ an augmented multiplier satisfying

$$P_{\Lambda_{\mathcal{F}}}(\xi) = \lambda,$$

$$P_{\Lambda_{\mathcal{F}}}(\xi) - \xi = -(r(u_N - g) - \mathcal{F}|u_T|)n_0 - ru_T.$$

The numerical experiments made on the corresponding discrete problem with nodal approximation of contact and friction condition is shown in Fig. 23. This method is the less sensitive to the parameter r compared to all other tested methods when friction is consid-

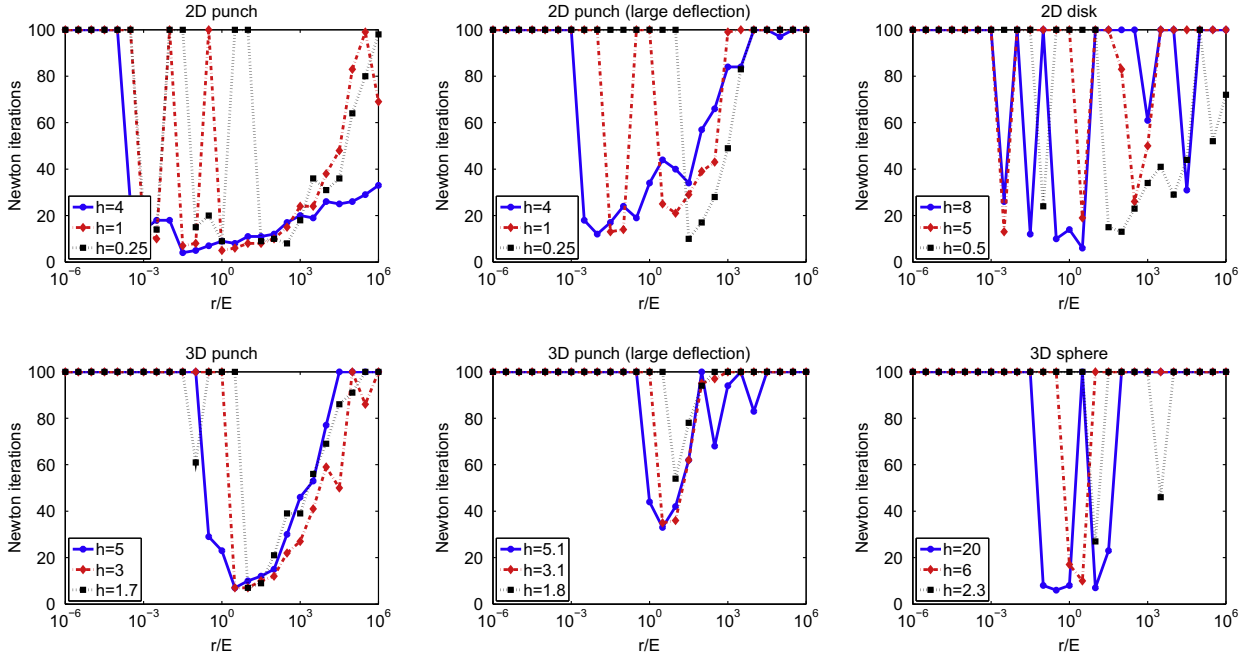


Fig. 17. Number of Newton iterations for symmetric Nitsche's method adapted to frictionless contact and an integral approximation.

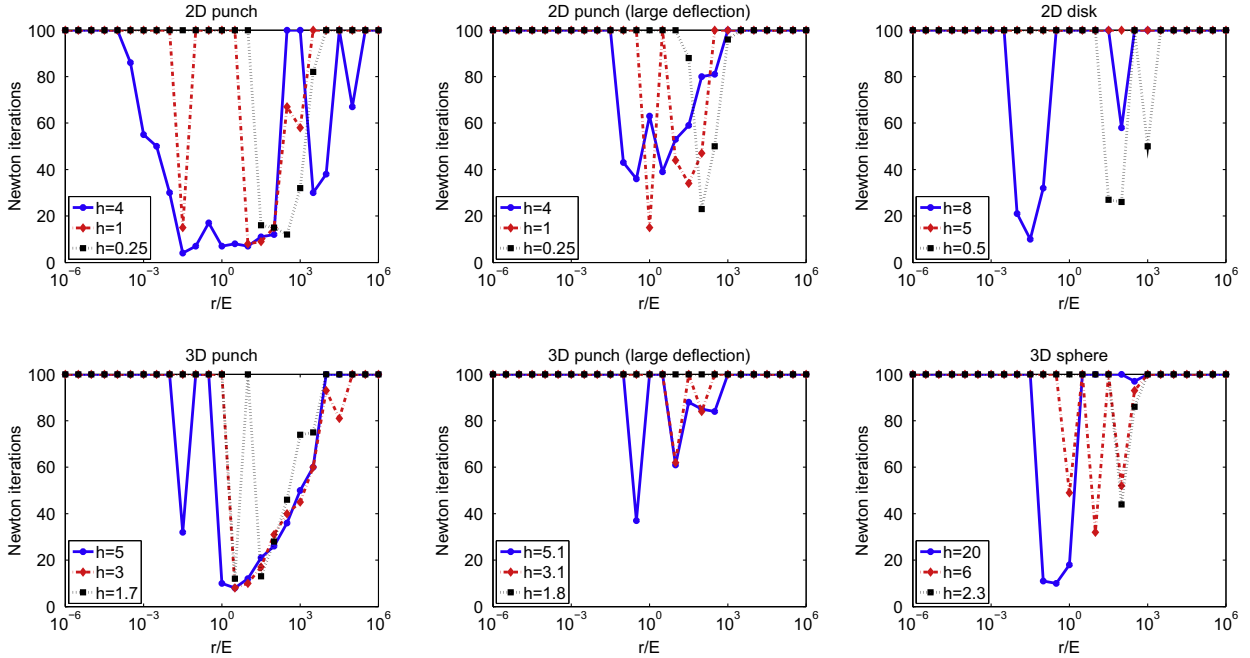


Fig. 18. Number of Newton iterations for symmetric Nitsche's method adapted to contact with friction and an integral approximation.

ered, especially for 2D cases. The results are also very satisfactory for 3D and non-coercive cases.

7.3. Integral approximation of the contact condition

The counterpart of the nodal formulation (19) in term of integral approximation of the contact condition reads as follows:

$$\begin{cases} a(u^h, v^h) = \ell(v^h) - \int_{\Gamma_C} (\zeta_N^h)_- v_N^h d\Gamma + \int_{\Gamma_C} P_{B(0, \mathcal{F}(\zeta_N^h)_-)}(\zeta_T^h) \cdot v_T^h d\Gamma \quad \forall v^h \in V^h, \\ - \int_{\Gamma_C} ((u_N^h - g) + \frac{1}{r} (\zeta_N^h)_-) \mu_N^h d\Gamma = 0 \quad \forall \mu^h \in W^h, \\ - \int_{\Gamma_C} (u_T^h + \frac{1}{r} \zeta_T^h - \frac{1}{r} P_{B(0, \mathcal{F}(\zeta_N^h)_-)}(\zeta_T^h)) \cdot \mu_T^h d\Gamma = 0 \quad \forall \mu^h \in W^h. \end{cases} \quad (20)$$

The existence of a solution to Problem (20) for any value of $r > 0$ and $\mathcal{F} \geq 0$ can be obtained classically with Brouwer's fixed point theorem applied on $\mathcal{P} : W^h \rightarrow W^h$ defined by

$$\mathcal{P}(\zeta^h) = P_{W^h} \left((\zeta_N^h)_- - r(u_N^h - g) \right) n_0 + P_{B(0, \mathcal{F}(\zeta_N^h)_-)}(\zeta_T^h) - r u_T^h,$$

where u is the solution to the first equation of (20) and P_{W^h} is the orthogonal projection on W^h (the technique of proof can be found for instance in [29]). The uniqueness for both $r > 0$ and $\mathcal{F} \geq 0$ small enough can be obtained as follows:

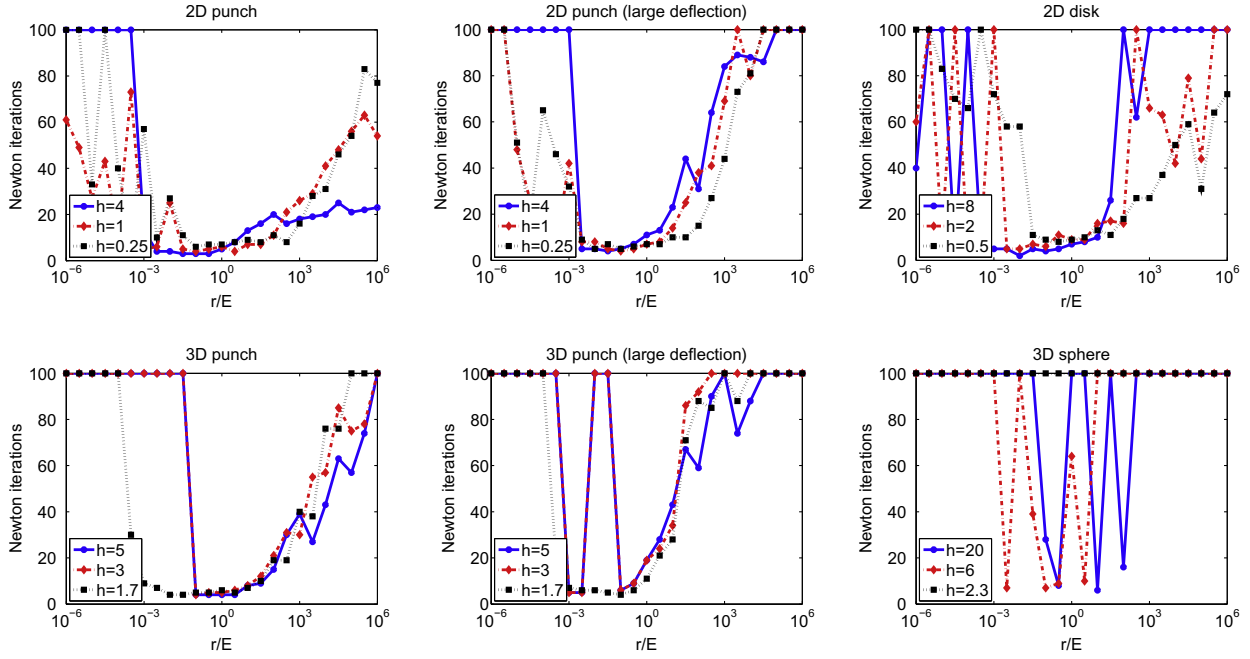


Fig. 19. Number of Newton iterations for unsymmetric Nitsche's method adapted to frictionless contact and an integral approximation.

Proposition 7.1. The solution (u^h, ξ^h) to Problem (20) is unique provided that $a(\cdot, \cdot)$ is coercive, the inf-sup condition (13) is satisfied and for $r > 0$ and $\mathcal{F} \geq 0$ small enough.

Proof. Let $(u^{h,1}, \xi^{h,1})$ and $(u^{h,2}, \xi^{h,2})$ be two solutions to Problem (20). Denoting $\lambda_N^{h,1} = (\xi_N^{h,1})_-$, $\lambda_N^{h,2} = (\xi_N^{h,2})_-$, $\lambda_T^{h,1} = P_{B(0, \mathcal{F}(\xi_N^{h,1})_-)}(\xi_T^{h,1})$ and $\lambda_T^{h,2} = P_{B(0, \mathcal{F}(\xi_N^{h,2})_-)}(\xi_T^{h,2})$, Problem (20) implies for $i = 1, 2$:

$$a(u^{h,i}, v^h) = l(v^h) + \int_{\Gamma_C} \lambda_N^{h,i} v_N^h d\Gamma + \int_{\Gamma_C} \lambda_T^{h,i} \cdot v_T^h d\Gamma \quad \forall v^h \in V^h,$$

$$\int_{\Gamma_C} (\xi^{h,i} + ru^{h,i} - \lambda^{h,i}) \cdot \mu^h d\Gamma = 0 \quad \forall \mu^h \in W^h.$$

In particular, this leads to $\xi^{h,i} = P_{W^h}(\lambda^{h,i} - ru^{h,i})$ and thus the following estimate holds:

$$\begin{aligned} \|\xi^{h,1} - \xi^{h,2}\|_{0,\Gamma_C}^2 &= \|P_{W^h}(\lambda^{h,1} - ru^{h,1}) - P_{W^h}(\lambda^{h,2} - ru^{h,2})\|_{0,\Gamma_C}^2 \\ &\leq \|\lambda^{h,1} - ru^{h,1} - \lambda^{h,2} + ru^{h,2}\|_{0,\Gamma_C}^2 \\ &= \|\lambda^{h,1} - \lambda^{h,2}\|_{0,\Gamma_C}^2 - 2r \int_{\Gamma_C} (\lambda^{h,1} - \lambda^{h,2})(u^{h,1} - u^{h,2}) d\Gamma \\ &\quad + r^2 \|u^{h,1} - u^{h,2}\|_{0,\Gamma_C}^2 \\ &= \|\lambda^{h,1} - \lambda^{h,2}\|_{0,\Gamma_C}^2 - 2ra(u^{h,1} - u^{h,2}, u^{h,1} - u^{h,2}) \\ &\quad + r^2 \|u^{h,1} - u^{h,2}\|_{0,\Gamma_C}^2. \end{aligned}$$

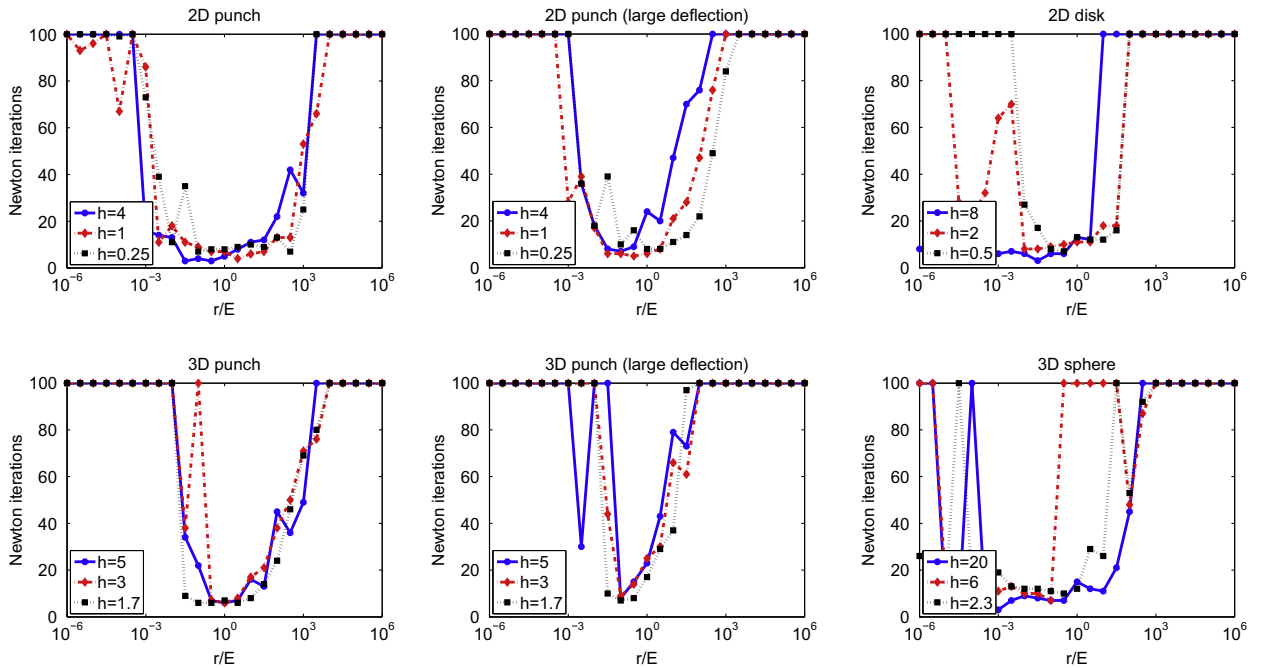


Fig. 20. Number of Newton iterations for unsymmetric Nitsche's method adapted to contact with friction and an integral approximation.

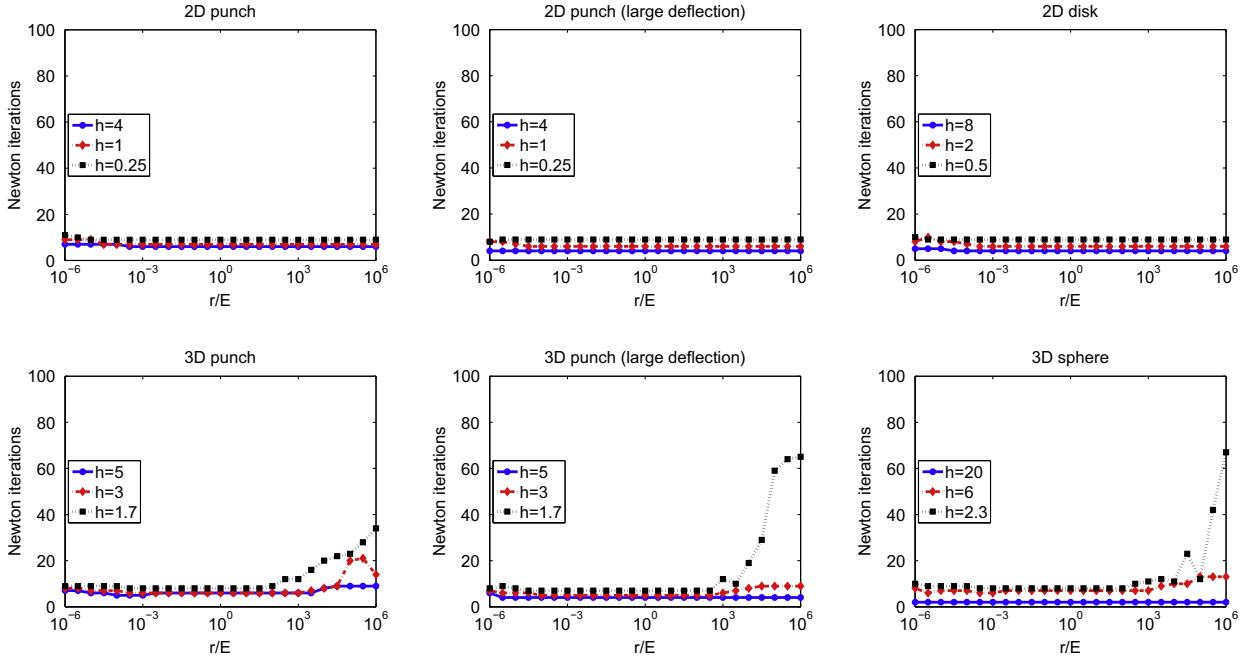


Fig. 21. Number of iterations for the new Newton's method with nodal frictionless contact approximation.

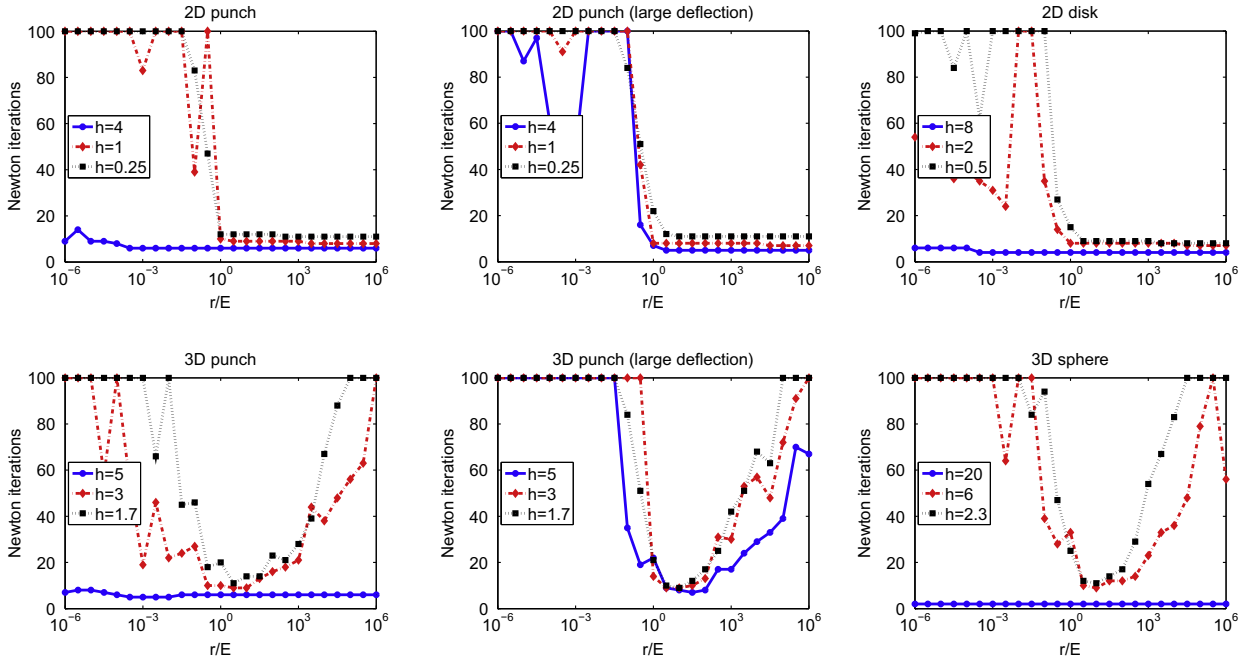


Fig. 22. Number of iterations for the new Newton's method with nodal contact with friction approximation.

Due to the coercivity of $a(\cdot, \cdot)$ there exist $\alpha > 0$ such that for $r > 0$ sufficiently small one has

$$\begin{aligned} & -2ra(u^{h,1} - u^{h,2}, u^{h,1} - u^{h,2}) + r^2 \|u^{h,1} - u^{h,2}\|_{0,\Gamma_C}^2 \\ & \leq -\alpha \|u^{h,1} - u^{h,2}\|_{0,\Gamma_C}^2, \end{aligned}$$

which together with the inf-sup condition (13) leads to the existence of $\beta > 0$ such that

$$\begin{aligned} & -2ra(u^{h,1} - u^{h,2}, u^{h,1} - u^{h,2}) + r^2 \|u^{h,1} - u^{h,2}\|_{0,\Gamma_C}^2 \\ & \leq -\beta \|\lambda^{h,1} - \lambda^{h,2}\|_{0,\Gamma_C}^2. \end{aligned}$$

Moreover one has

$$\begin{aligned} \|\lambda^{h,1} - \lambda^{h,2}\|_{0,\Gamma_C} & \leq \|\zeta^{h,1} - \zeta^{h,2}\|_{0,\Gamma_C} + \|P_{B(0,\mathcal{F}(\zeta_N^{h,1}))}(\zeta_T^{h,2}) \\ & \quad - P_{B(0,\mathcal{F}(\zeta_N^{h,2}))}(\zeta_T^{h,2})\|_{0,\Gamma_C}, \end{aligned}$$

and

$$\|P_{B(0,\mathcal{F}(\zeta_N^{h,1}))}(\zeta_T^{h,2}) - P_{B(0,\mathcal{F}(\zeta_N^{h,2}))}(\zeta_T^{h,2})\|_{0,\Gamma_C} \leq \mathcal{F} \|\zeta_N^{h,1} - \zeta_N^{h,2}\|_{0,\Gamma_C}.$$

Thus, the following estimate holds for $r > 0$ small enough:

$$\|\zeta^{h,1} - \zeta^{h,2}\|_{0,\Gamma_C} \leq \sqrt{1 - \beta(1 + \mathcal{F})} \|\zeta^{h,1} - \zeta^{h,2}\|_{0,\Gamma_C},$$

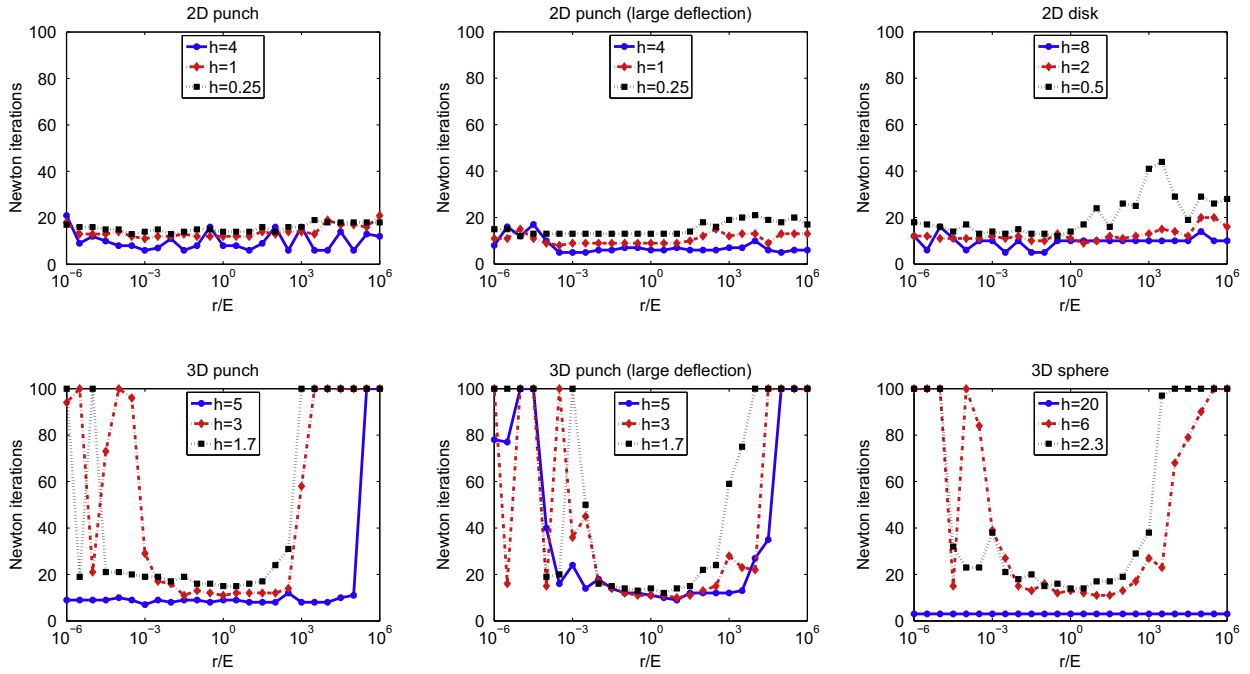


Fig. 23. Number of iterations for the new Newton's method using De Saxcé projection and with nodal contact with friction approximation.

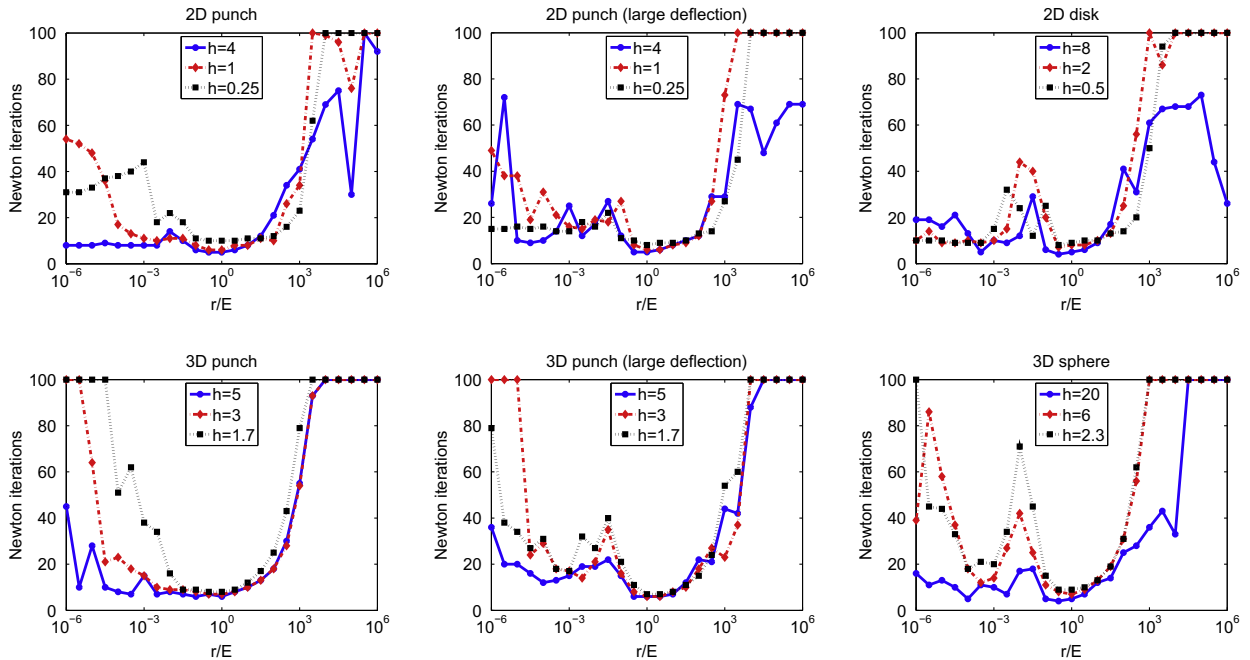


Fig. 24. Number of iterations for the new Newton's method with integral frictionless contact approximation.

which leads to $\zeta^{h,1} = \zeta^{h,2}$ for \mathcal{F} sufficiently small. \square

Note that since the solution to Problem (20) do depend on the parameter r , the uniqueness of the solution for an arbitrary $r > 0$ is an open question.

The corresponding numerical experiments are presented in Figs. 24 and 25 for the contact without and with friction, respectively. Similarly to the Alart–Curnier approach, generalized Newton's method on the integral approximation leads to a less robust method with respect to the parameter r . When considering addi-

tionally De Saxcé's projection for the case with friction, the graphs of Fig. 26 show also an improvement of the convergence similarly to what was noted for the nodal approximation.

8. Concluding remarks

In this paper, some new generalized Newton's method for contact with friction problems are introduced and compared to existing strategies on some basic 2D and 3D situations. Two kinds of approximations of the contact and friction conditions are con-

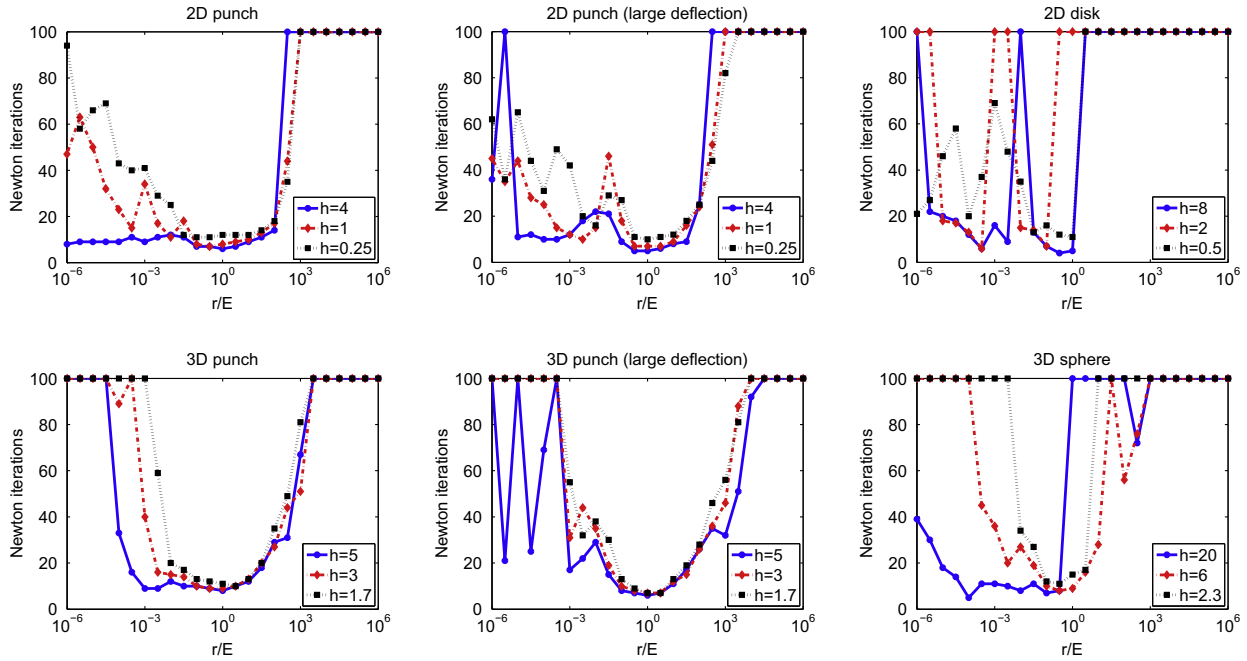


Fig. 25. Number of iterations for the new Newton's method with integral contact with friction approximation.

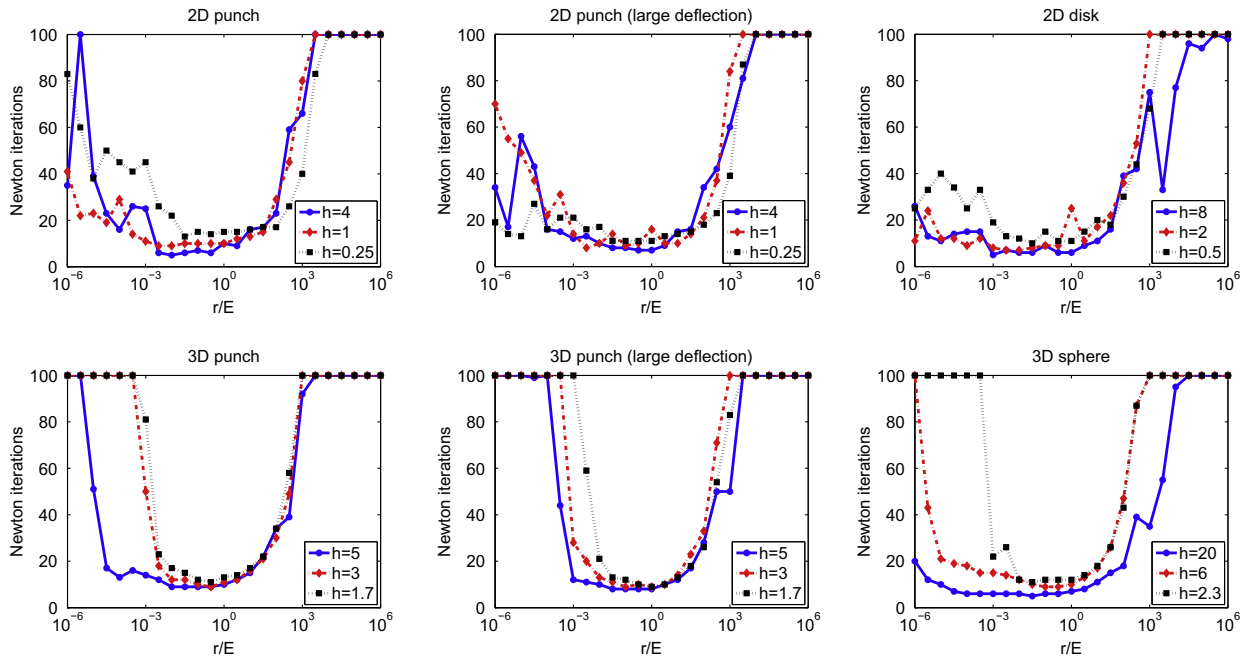


Fig. 26. Number of iterations for the new Newton's method using De Saxcé projection and with integral contact with friction approximation.

sidered: a nodal and an integral one. The advantage of the integral approximation is the fact that it is without constraints. Consequently, the set of admissible stress is not to be described. This may be a crucial advantage for instance in the framework of fictitious domain or Xfem methods. However, this approximation leads to Newton's methods a little bit more sensitive to the value of the augmentation parameter compared to the ones build on a nodal approximation.

The new generalized Newton's methods based on augmented multipliers have a better behavior than the other tested methods,

especially when using De Saxcé's projection. In particular it solves correctly the non-coercive cases. Of course, the method can be adapted to more complex contact situations such as two-body contact and large deformations. Despite the fact that it does not solve the non-coercive cases every time, non-symmetric Nitsche's method is also very promising and prevent the use of a multiplier which can be an advantage in some situations.

For all the methods, it is observed a slight increase of the number of Newton iterations when the mesh is refined. This variation indicates the non-scalability of the Newton's methods. However,

this increase is quite small and, *a priori*, does not require a specific treatment.

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