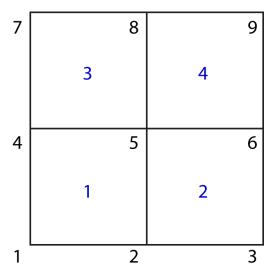
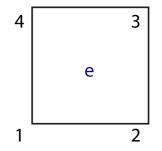
Multidimensional elastic solutions: FEM Implementation

MAE 404/598 - Oswald

3/26/2015

Organization of nodal degrees of freedom

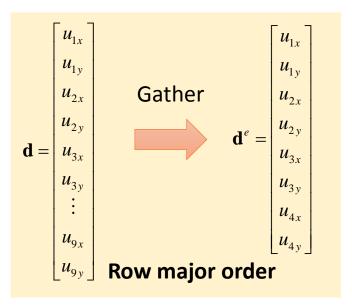


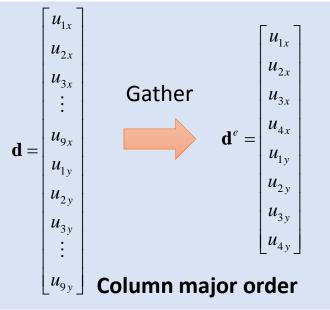


Element node numbering

$$\mathbf{d} = \begin{bmatrix} u_{1x} & u_{1y} \\ u_{1y} & u_{2y} \\ u_{2x} & u_{3y} \\ \vdots & \vdots \\ u_{9x} & u_{9y} \end{bmatrix}$$

Nodal displacement matrix





Weak form and discretization

$$\int_{\Omega} \nabla_{S} \mathbf{w} : \mathbf{C}^{e} : \nabla_{S} \mathbf{u} \, d\Omega = \int_{\Omega} \mathbf{w} \cdot \mathbf{b} \, d\Omega + \int_{\Gamma_{t}} \mathbf{w} \cdot \overline{\mathbf{t}} \, d\Gamma$$

$$\nabla_{s} = \begin{bmatrix} \frac{\partial}{\partial x} & 0\\ 0 & \frac{\partial}{\partial y}\\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} \end{bmatrix}$$

$$\mathbf{w} = N_I \mathbf{w}_I \qquad \nabla_S \mathbf{w} = \mathbf{B}_I \mathbf{w}_I$$

$$\mathbf{u} = N_I \mathbf{u}_I \qquad \nabla_S \mathbf{u} = \mathbf{B}_I \mathbf{u}_I$$

Symmetric gradient operator

$$\mathbf{B}_{I} = \begin{bmatrix} \frac{\partial N_{I}}{\partial x} & 0 \\ 0 & \frac{\partial N_{I}}{\partial y} \\ \frac{\partial N_{I}}{\partial y} & \frac{\partial N_{I}}{\partial x} \end{bmatrix}$$

$$\mathbf{B}_{I} = \begin{bmatrix} \frac{\partial N_{I}}{\partial x} & 0 \\ 0 & \frac{\partial N_{I}}{\partial y} \\ \frac{\partial N_{I}}{\partial y} & \frac{\partial N_{I}}{\partial x} \end{bmatrix} \qquad \mathbf{B} = \begin{bmatrix} \frac{\partial N_{1}}{\partial x} & 0 & \frac{\partial N_{2}}{\partial x} & 0 & \frac{\partial N_{3}}{\partial x} & 0 & \frac{\partial N_{4}}{\partial x} & 0 \\ 0 & \frac{\partial N_{1}}{\partial y} & 0 & \frac{\partial N_{2}}{\partial y} & 0 & \frac{\partial N_{3}}{\partial y} & 0 & \frac{\partial N_{4}}{\partial y} \\ \frac{\partial N_{I}}{\partial y} & \frac{\partial N_{I}}{\partial x} & \frac{\partial N_{1}}{\partial x} & \frac{\partial N_{2}}{\partial y} & \frac{\partial N_{2}}{\partial x} & \frac{\partial N_{3}}{\partial y} & \frac{\partial N_{3}}{\partial x} & \frac{\partial N_{4}}{\partial y} & \frac{\partial N_{4}}{\partial x} \end{bmatrix}$$

General 2D B-matrix

B-Matrix for a 4-node element.

Weak form – stiffness matrix

$$\int_{\Omega} \nabla_{S} \mathbf{w} : \mathbf{C}^{e} : \nabla_{S} \mathbf{u} \, d\Omega = \int_{\Omega} \mathbf{w} \cdot \mathbf{b} \, d\Omega + \int_{\Gamma_{t}} \mathbf{w} \cdot \overline{\mathbf{t}} \, d\Gamma$$

$$\int_{\Omega} \nabla_{S} \mathbf{w} : \mathbf{C}^{e} : \nabla_{S} \mathbf{u} \, d\Omega \Rightarrow \mathbf{w}^{T} \mathbf{K} \mathbf{d} \qquad \mathbf{K} = \sum_{e} \mathbf{L}^{eT} \mathbf{K}^{e} \, \mathbf{L}^{e}$$

$$\mathbf{K} = \sum_{e} \mathbf{L}^{eT} \mathbf{K}^{e} \ \mathbf{L}^{e}$$

$$\mathbf{K}^e = \int_{\Omega} \mathbf{B}^{eT} \mathbf{C}^e \mathbf{B}^e \, d\Omega$$

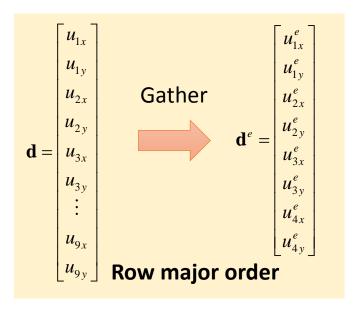
LHS of weak form Stiffness matrix assembly

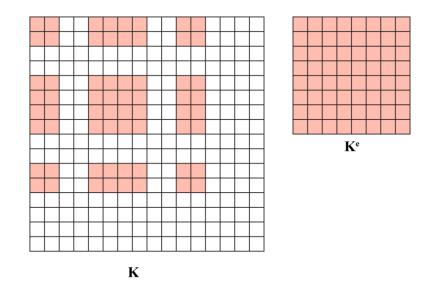
Element stiffness matrix

```
for c = mesh.conn
  xe = mesh.x(:,c);
  Ke = zeros(8);
  for q = quadrature_points % [3xNQ] matrix, each column gives [\xi;\eta;wt]
    dNdp = grad shape(q);
                              % compute gradient of shape functions
                              % J_{ij} = dx_i/d\xi_j; J is <u>not</u> generally symmetric.
    J = xe * dNdp;
    dNdx = dNdp / J;
    B = zeros(3,8);
    B(1,1:2:end) = dNdx(:,1);
                               % assigns dNdx to odd columns in first row
    B(2,2:2:end) = dNdx(:,2);
                               % assigns dNdx to even columns in second row
    B(3,1:1:end) = dNdx(:,2); % assigns dNdy to odd columns in third row
    B(3,2:2:end) = dNdx(:,1); % assigns dNdx to even columns in third row
    Ke = Ke + B'*C*B * det(J)*q(3);
  end
  % scatter operation goes here...
end
```

Scattering of element stiffness matrix

$$K_{IJ} \leftarrow K_{ij}^e$$
 What is the relationship between (i,j) and (I,J)?





```
cc = [2*conn-1; 2*conn];
% e.g. c=[1;2;3;4]; becomes
% cc = [1;2;3;4;5;6;7;8];
K(cc,cc) = K(cc,cc) + Ke;
```

Procedure for row-major ordering (2D)

Procedure for row-major ordering (3D)

Vector operations vs for loops (scatter in 3D)

Where c is an array of the nodes within an element.

```
METHOD 1
                                    METHOD 2
cc = [3*c-2; 3*c-1; 3*c];
                                 for i = 1:4
K(cc,cc) = K(cc,cc) + Ke;
                                      for di = 1:3
                                         for j = 1:4
                                           for dj = 1:3
                                             I = 3*(c[i]-1) + di;
                                             J = 3*(c[j]-1) + dj;
                                             ii = 3*(i-1) + di;
                                             jj = 3*(j-1) + dj;
                                             K(I,J) = K(I,J) + Ke(ii,jj);
                                           end
                                        end
                                      end
                                    end
```

Timing for 1000 elements

METHOD 1: 15 ms METHOD 2: 850 ms

METHOD 2 is 60X slower!

Integration of applied traction

Simple case

When the element edge is straight (and nodes I, J are the corner nodes on the edge)

$$\left\| \frac{\partial \mathbf{x}}{\partial \xi} \right\| = \frac{\left\| \mathbf{x}_{I}^{e} - \mathbf{x}_{J}^{e} \right\|}{2}$$

General case (where c are the nodes on the edge)

 $dNdp = shape1D(\xi);$ % 1D edge shape function $J_edge = norm(xe(:,c) * dNdp);$

