

Mass and energy balance calculations for an artificial ice reservoir (Icestupa)

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2 ABSTRACT

Artificial Ice Reservoirs (AIRs) can successfully store water during winter and release the water during spring and summer. This makes them a reliable fresh water resource for irrigation in dry environments. Several AIRs have been built but studies of their water storage capacity and efficiency are scarce. This study models a cone-shaped AIR popularly called Icestupa. Processes involved in the development and temporal evolution of an Icestupa are calculated by a physically-based model using equations governing the heat transfer, vapour diffusion and transport of water that undergoes phase changes. These processes were quantified using meteorological data in conjunction with fountain spray information (mass input of an Icestupa) to estimate the quantity of frozen, melted, evaporated and runoff water at a location called 'Eispalast' in Fribourg, Switzerland. At this measurement site, an Icestupa was built for model validation purposes. The model was further tested by performing a sensitivity and uncertainty study showing that the most sensitive parameters are the ice emissivity and the temperature threshold used to determine precipitation phase. Model calculations estimate that the Eispalast Icestupa stored about 6% of the total water sprayed as ice. In addition, we found that reducing nozzle diameter of the fountain from 5 mm to 3 mm increases the water storage up to 78% without compromising on the storage duration.

Keywords: ictestupa, mass balance, water storage, climate change adaptation, geoengineering

1 INTRODUCTION

Seasonal snow cover, glaciers and permafrost are expected to change their water storage capacity due to climate change with major consequences for downriver water supply (Immerzeel et al., 2019). The challenges brought about by these changes are especially important for dry mountain environments such as in Central Asia or the Andes, which directly rely on the seasonal meltwater for their farming and drinking needs (Hoelzle et al., 2019; Apel et al., 2018; Buytaert et al., 2017; Chen et al., 2016; Unger-Shayesteh



Figure 1. Icestupa in Ladakh, India on March 2017 was 24 m tall and contained around 3700 m^3 of water. Picture Credits: Lobzang Dadul

25 et al., 2013). Some villages in Ladakh, India have already been forced to relocate due to glacial retreat and
26 the corresponding loss of their main fresh water resources (Grossman, 2015).

27 Artificial ice reservoirs (AIRs) have been considered to be a feasible way to adapt to these changes (Hock
28 et al., 2019; Nüsser et al., 2019b). An AIR is a human-made ice structure typically constructed during the
29 cold winter months and designed to slowly release freshwater during the warm and dry spring and summer
30 months. The main purpose of AIRs is irrigation. Therefore, AIRs are designed to store water in the form of
31 ice as long into the summer as possible. The energy required to construct an AIR is usually derived from
32 the gravitational head of the source water body. Some are constructed horizontally by freezing water using
33 a series of checkdams and others are built vertically by spraying water through fountain systems (Nüsser
34 et al., 2019a). The latter are colloquially referred to as Icestupas and are the subject of this study.

35 A typical Icestupa just requires a pipeline attached to a vertically mounted metal pipe with a fountain
36 nozzle for construction. Their water source is usually a high altitude lake or glacial stream. Due to the
37 altitude difference between the pipeline input and fountain output, water ejects from the fountain nozzle as
38 droplets that eventually lose their latent heat to the atmosphere and accumulate as ice around the metal pipe.
39 The fountain nozzle is raised through addition of further pipes as and when significant ice accumulates.
40 Typically, a dome of branches is constructed around the metal pipe so that such pipe extensions can be
41 done from within this dome. During the winter, the fountain is manually activated from sunset to sunrise.
42 Threads, tree branches and fishing nets are used to guide and accelerate the ice formation.

43 Since their invention in 2013 (Wangchuk, 2014), Icestupas have gained widespread publicity in the region
44 of Ladakh, Northern India since they require very little infrastructure, skills and energy to be constructed
45 in comparison to other water storage technologies. Compared to other AIR geometries, Icestupas (Fig.
46 1) can be built at lower altitudes and last much longer into the summer than other types of ice structures
47 (Wangchuk, 2014). However, to date, no reliable estimates exist about the amount of sprayed water that
48 is necessary to create them and the meltwater they provide (Nüsser et al., 2019a). Rough estimates of
49 Icestupa meltwater in Ladakh suggest that the water loss during the construction process is considerable
50 (see Appendix 8.1). A complete set of measurements of the water storage and energy balance are required
51 to understand the cause of the water losses better and increase the construction efficiency.

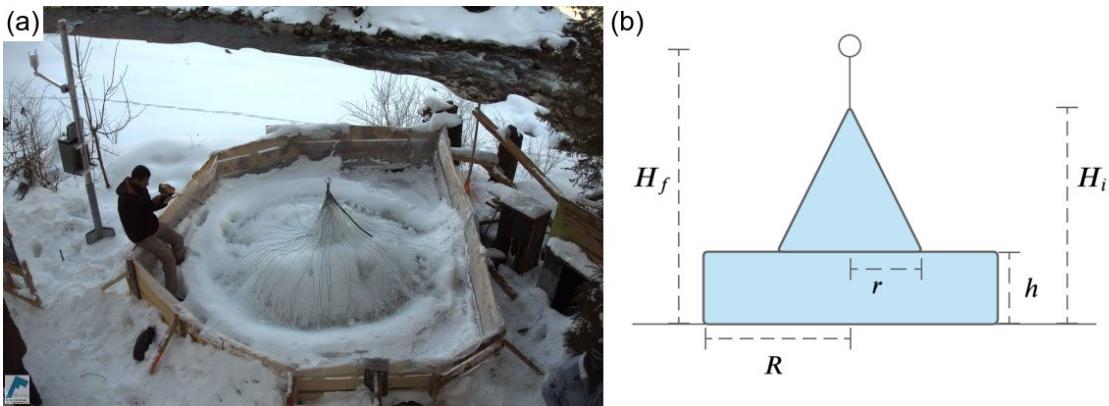


Figure 2. (a) The ice structure during the first validation measurement as seen on the webcam image of 14th Feb. (b) The corresponding cross section of the EP ice structure with the field estimates of r , R , h , H_i , H_f used to determine the Icestupa volume is shown on the right.

52 In this paper, we aim to develop a physically-based model of a vertical AIR (or Icestupa) that can quantify
 53 their storage efficiency using existing weather and water usage information. Mass and energy balance
 54 equations were used to estimate the quantity of water frozen, melted, evaporated and runoff. Sensitivity
 55 and uncertainty analysis were performed to identify the most critical parameters and the variance caused
 56 by them. For validation, we created an Icestupa at an accessible site (called Eispalast) near Schwarzsee
 57 close to the city of Fribourg, Switzerland, allowing easy maintenance and control of the measurements.
 58 Due to the low altitude of the site with relatively high winter temperatures, only a small Icestupa could
 59 be established during winter 2018/19 to provide us with model validation data. Our model and validation
 60 experiments provide first steps towards evaluating the effectiveness of a vertical AIR for irrigation and
 61 allow us to outline some preliminary guidelines for consideration when a construction of an Icestupa for
 62 water storage is envisaged.

2 STUDY SITE

63 The Eispalast (EP) site in the Schwarzsee region lies at 967 m a.s.l.. In the winter (Oct-Apr), mean daily
 64 maximum and minimum air temperatures vary between -4 and 14 °C. Clear skies are rare, averaging
 65 around 7 days, and precipitation amounts average 155 mm per month during winter (Meteoblue, 2020).
 66 The site was situated adjacent to a stream resulting in high humidity values across the study period. Within
 67 the EP site, an enclosure with a 1.8 m radius was constructed for the experiment. An automatic weather
 68 station (AWS) was adjacent to the wooden boundary as shown in Fig. 2. The fountain used for spraying
 69 water had a nozzle diameter of 5 mm and a height of 1.35 m, and was placed in the centre of the wooden
 70 enclosure. The water was transferred from a spring water source at 1267 m a.s.l. by pipeline and flowed
 71 via a flowmeter and an air escape valve to the nozzle, where it was sprinkled with a spray radius of around
 72 1.7 m. The air escape valve was installed to avoid errors in the flow measurements due to air bubbles. In
 73 addition, a webcam guaranteed a continuous survey of the site during the construction of the Icestupa.

74 2.1 Construction

75 From 30th January to 18th March 2019 the Icestupa was constructed through the fountain spray, which
 76 was manually switched on if measured air temperature was below -5 °C after sunset and was switched
 77 off as soon as the ice was exposed to daylight or temperatures were above 0 °C. The water spray of the
 78 fountain was initially adjusted so that most of the water droplets land within the wooden boundary zone.
 79 The ice formation was guided by adding a metal framework at the ice structure base after the first night of

80 operation. Several cotton threads were tied between the ice structure base and fountain pole for accelerating
 81 and further guiding the ice formation process.

82 2.2 Measurements and Data

83 Measurements comprising air temperature, relative humidity, water flow rate, wind speed and direction
 84 were made every 5 minutes throughout the construction period. The water flow rate or discharge was
 85 measured via an ultrasonic sensor attached to the fountain supply pipeline. Precipitation data was derived
 86 from the Plaffeien AWS (IDAWEB, 2019) located 8.8 km away from the measurement site at an altitude of
 87 1042 m a.s.l.

88 In addition, we used ERA5 reanalysis dataset (Copernicus Climate Change Service (C3S), 2017) for
 89 filling data gaps and adding data that were not measured directly at the EP site. We recognised during
 90 our data analysis that, except precipitation, all the other meteorological variables of the EP site correlated
 91 much better with the ERA5 dataset than the nearby Plaffeien AWS dataset. The 2 m temperature parameter
 92 correlated with air temperature ($r^2 = 0.9$), surface pressure parameter correlated with air pressure ($r^2 = 1$)
 93 and 10m wind speed parameter (derived from horizontal and vertical components) correlated with wind
 94 speed ($r^2 = 0.6$). The ERA5 reanalysis dataset has a good correlation with lower elevation sites in
 95 Switzerland (Scherrer, 2020). The hourly ERA5 data and the 10 minute Plaffeien AWS data were linearly
 96 interpolated to the 5 minute data frequency of the EP AWS.

97 Due to a power failure, all data from the EP AWS was lost from 27th February 15:20 2019 to 2nd March
 98 15:00 2019 (equivalent to around 7% of the measurement period). During heavy snowfall events, the
 99 ultrasonic wind sensor was blocked and recorded zero values. ERA5 was used to fill such errors and data
 100 gaps. Near-surface humidity is not provided directly in ERA5 dataset, but from near-surface (2 m from the
 101 surface) temperature (T_{ERA5}) and dew point temperature (Tw_{ERA5}) the relative humidity (RH) at 2 m
 102 was calculated as:

$$RH = 100 \cdot \frac{e_{sat}(Tw_{ERA5})}{e_{sat}(T_{ERA5})} \quad (1)$$

103 where the saturation vapour pressure function e_{sat} is expressed with the Teten's formula (Tetens, 1930):

$$e_{sat}(T) = a_1 \cdot e^{(a_3 \cdot \frac{T}{(T+273.16-a_4)})} \quad (2)$$

104 with T in °C and the parameters set for saturation over water ($a_1 = 611.21$ Pa, $a_3 = 17.502$ and $a_4 = 32.19$
 105 K) according to Buck (1981). Zero wind speed values were recorded whenever snow accumulated on the
 106 ultrasonic wind sensor. All such null values were replaced using the ERA5 dataset.

107 The ERA5 grid point chosen (Latitude 46° 38' 24" N, Longitude 7° 14' 24" E) for the EP site was around
 108 9 km away from the actual site. All the ERA5 variables were therefore fitted with the EP dataset via linear
 109 regressions. With the modified ERA5 dataset, we were also able to further extend the EP dataset and allow
 110 the model to run beyond 18th March 2019. Precipitation was filled as null values beyond 18th March 2019.

111 2.2.1 Field Measurements for Validation
 112 The volume was determined by decomposing the ice structure into a cylinder (length $2R$ and height h)
 113 and a cone (radius r and height $(H_i - h)$) through the following equation:

$$V = \pi \cdot R^2 \cdot h + 1/3 \cdot \pi \cdot r^2 \cdot (H_i - h) \quad (3)$$

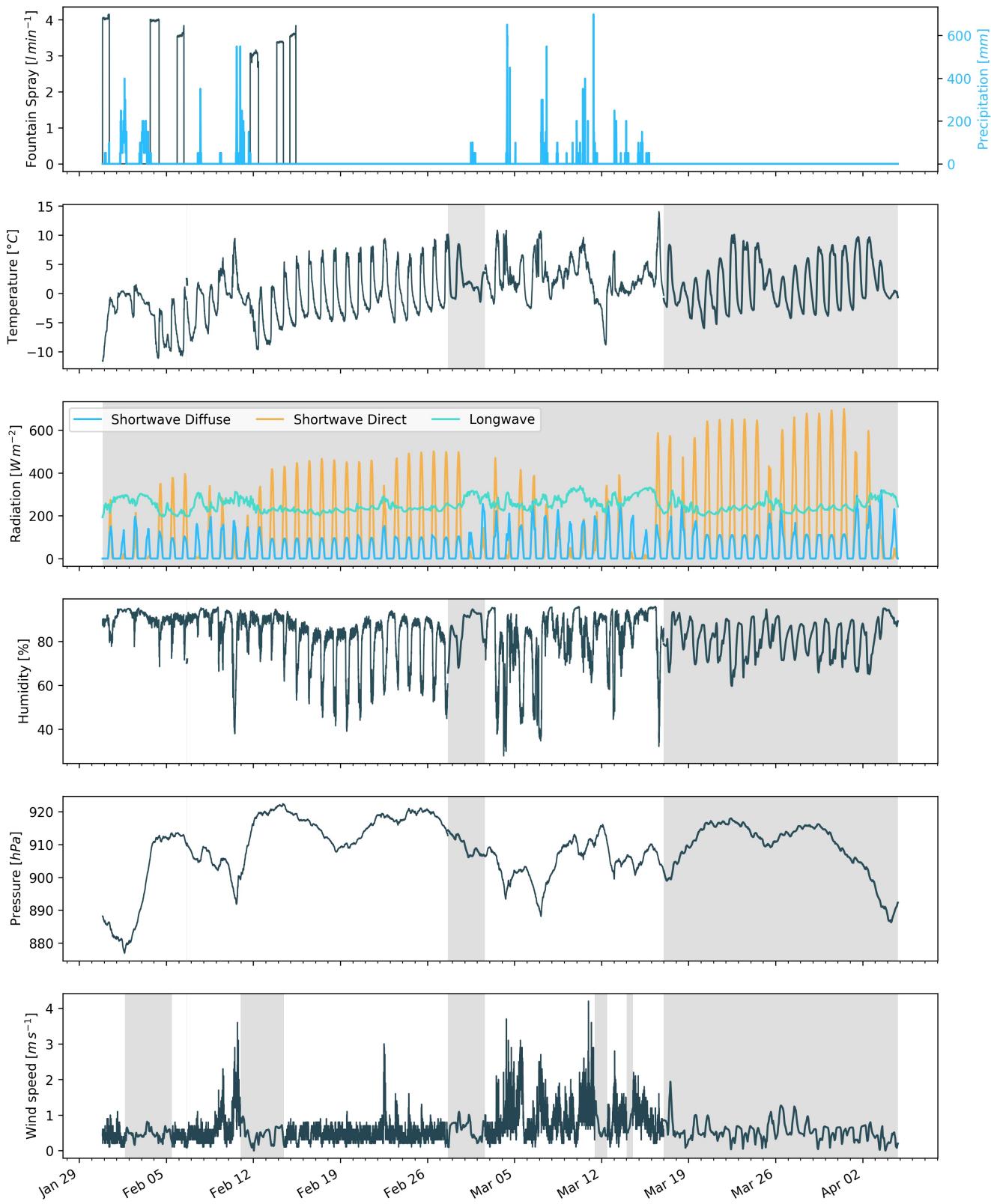


Figure 3. Measurements at the AWS of EP were used as main model input data in 5 minute frequency. Plaffeien AWS provided the precipitation data. Incoming shortwave and longwave radiation were obtained from ERA5 reanalysis dataset. Several data gaps and errors were also filled from the ERA5 dataset (shaded regions).

114 Manual measurements were performed at the end of the freezing period on 14th February 16:00 2019
 115 (only one more fountain run was possible after this date) to estimate r, R, h, H_i, H_f (see Fig. 2 for the
 116 different geometry components):

$$0.55 \leq r \leq 1m ; 1.1 \leq R \leq 1.2m ; 0.1 \leq h \leq 0.2m ; 0.6 \leq H_i \leq 0.8m ; 1.3 \leq H_f \leq 1.4m$$

117 The ranges of the variables show the variance of the Icestupa's dimensions across different compass
 118 orientations. Correspondingly, the volume range estimated for the first validation point was 0.857 ± 0.186
 119 m^3 on 14th February 16:00 2019.

120 The second validation point corresponds to the end of the melting process on 10th March 18:00 2019.
 121 Based on the webcam imagery and manual measurement, a thin layer of ice with an observed thickness
 122 between 0.01 to 0.06 m could be quantified. This results in the volume range for the second validation to
 123 be $0.13 \pm 0.09 m^3$ on 11th March 2019

124 In reality, the EP ice structure was more cylindrical until a height of 0.2 m and conical afterwards until a
 125 height of 0.6 m with a radius of 1.18 m. However, we assume a conical shape of this ice structure in order
 126 to apply the modelling strategy described below.

3 MODEL SETUP

127 The model consists of three parts which calculate, a) the geometric evolution of the Icestupa, b) the energy
 128 balance and c) the mass balance as shown schematically in Fig. 4. A bulk energy and mass balance model
 129 is used to calculate the amounts of ice, meltwater, water vapour and runoff water of the Icestupa every 5
 130 minutes.

3.1 Icestupa geometric evolution

131 Radius r_{ice}^i and height h_{ice}^i define the dimensions of the Icestupa assuming its geometry to be a cone as
 132 shown in Fig. 5. The surface area A^i exposed to the atmosphere and volume V^i are:

$$A = \pi \cdot r_{ice} \cdot \sqrt{r_{ice}^2 + h_{ice}^2} \quad (4)$$

$$V = \pi/3 \cdot r_{ice}^2 \cdot h_{ice} \quad (5)$$

134 Note that we do not specify the time step superscript i of the shape variables A, V, r_{ice} and h_{ice} for
 135 brevity. The equations used henceforth display model time step superscript i only if it is different from the
 136 current time step.

137 With the mass of the Icestupa M_{ice} , its current volume can also be expressed as:

$$V = M_{ice}/\rho_{ice} \quad (6)$$

138 where ρ_{ice} is the density of ice ($917 kg m^{-3}$). The model of the Icestupa is initialised with a thickness
 139 of Δx (defined in 3.2) and a circular area of radius r_F . The constant r_F represents the mean spray radius
 140 of the fountain. This fountain spray radius is determined by modelling the projectile motion of the water

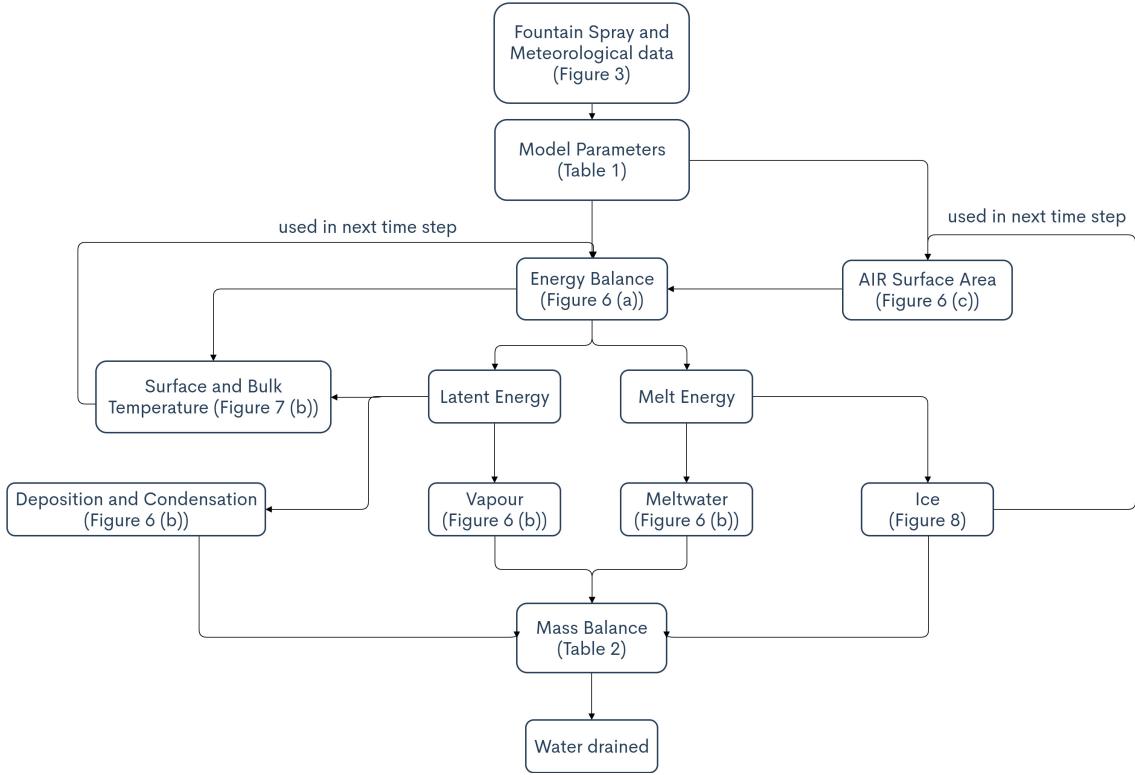


Figure 4. Model schematic showing the algorithm used in the model at every time step. Further details about the variables can be found in the associated tables and figures.

141 droplets. Using mass conservation, the droplet speed v_F can be determined from the spray rate d_F and the
 142 diameter dia_F of the nozzle as follows:

$$v_F = \frac{d_F}{\pi \cdot dia_F^2 / 4} \quad (7)$$

143 Afterwards, we assume that the water droplets move with an air friction free projectile motion from
 144 the fountain nozzle with a height h_F to the ice/ground surface. The resulting spray radius r_F was then
 145 determined from the projectile motion equation as follows:

$$r_F = \frac{v_F \cdot \cos\theta_F (v_F \cdot \sin\theta_F + \sqrt{(v_F \cdot \sin\theta_F)^2 + 2 \cdot g \cdot h_F})}{g} \quad (8)$$

146 where $g = 9.8 \text{ ms}^{-2}$ is the acceleration due to gravity and $\theta_F = 45^\circ$ is the angle of launch.

147 During subsequent time steps, the dimensions of the Icestupa evolve assuming a uniform ice formation
 148 and decay across its surface area with an invariant slope $s_{cone} = \frac{h_{ice}}{r_{ice}}$ as shown in Fig. 5. During these time
 149 steps, the volume is parameterised using Eqn. 5 as:

$$V = \pi / 3 \cdot r_{ice}^3 \cdot s_{cone} \quad (9)$$

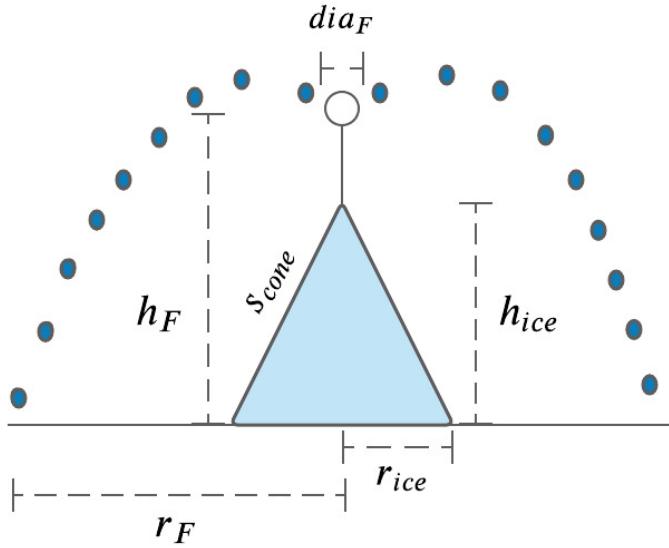


Figure 5. Shape variables and fountain constants of the EP Icestupa. r_{ice} is the radius, h_{ice} is the height and s_{cone} is the slope of the ice cone. r_F is the spray radius, h_F is the height and dia_F is the nozzle diameter of the fountain.

150 However, the Icestupa cannot outgrow the maximum range of the water droplets ($(r_{ice})_{max} = r_F$).
 151 Combining equations 5, 6 and 9, the geometric evolution of the Icestupa at each time step i can be
 152 determined by considering the following rules:

$$(r_{ice}, h_{ice}) = \begin{cases} (r_F, \Delta x) & \text{if } i = 0 \\ (r_{ice}^{i-1}, \frac{3 \cdot M_{ice}}{\pi \cdot \rho_{ice} \cdot (r_{ice}^{i-1})^2}) & \text{if } r_{ice}^{i-1} \geq r_F \text{ and } \Delta M_{ice} > 0 \text{ where } \Delta M_{ice} = M_{ice}^{i-1} - M_{ice}^{i-2} \\ (\frac{3 \cdot M_{ice}}{\pi \cdot \rho_{ice} \cdot s_{cone}})^{1/3} \cdot (1, s_{cone}) & \text{otherwise} \end{cases} \quad (10)$$

153 3.2 Energy Balance

154 The energy balance equation (Hock, 2005) for the Icestupa is formulated as follows:

$$q_{net} = q_{SW} + q_{LW} + q_L + q_S + q_F + q_G \quad (11)$$

155 where q_{net} is the net energy flux in $[W m^{-2}]$; q_{SW} is the net shortwave radiation; q_{LW} is the net longwave
 156 radiation; q_L and q_S are the turbulent latent and sensible heat fluxes. q_F represents the heat exchange of the
 157 fountain water droplets with the AIR ice surface during fountain on time steps. q_G represents ground heat
 158 flux between Icestupa surface and Icestupa interior. Energy transferred in the direction of the ice surface is
 159 always denoted as positive and away as negative.

160 Equation 11 is usually referred to as the energy budget for “the surface”, but practically it must apply to a
 161 surface layer of ice with a finite thickness Δx . The energy flux acts upon the Icestupa surface layer which
 162 has an upper and a lower boundary defined by the atmosphere and the ice body of the Icestupa, respectively.
 163 The parameter selection for Δx is based on the following two arguments: (a) the ice thickness Δx should

164 be small enough to represent the daily surface temperature variations and (b) Δx should be large enough
 165 for these temperature variations to not reach the bottom of the surface layer. Therefore, we introduced a 5
 166 mm thick ice surface layer, over which the energy balance is calculated. A sensitivity analysis was later
 167 performed to understand the influence of this factor. Here, we define the surface temperature T_{ice} to be
 168 the modelled average temperature of the Icestupa surface layer and the energy flux q_{net} is assumed to act
 169 uniformly across the Icestupa area A .

170 3.2.1 Net Shortwave Radiation q_{SW}

171 The net shortwave radiation q_{SW} is computed as follows:

$$q_{SW} = (1 - \alpha) \cdot (SW_{direct} \cdot f_{cone} + SW_{diffuse}) \quad (12)$$

172 where SW_{direct} and $SW_{diffuse}$ are the ERA5 direct and diffuse short wave radiation, α is the modelled
 173 albedo and f_{cone} is the area fraction of the ice structure exposed to the direct shortwave radiation.

174 We model the albedo using a scheme described in Oerlemans and Knap (1998). The scheme records the
 175 decay of albedo with time after fresh snow is deposited on the surface. δt records the number of time steps
 176 after the last snowfall event. After snowfall, albedo changes over a time step, δt , as

$$\alpha = \alpha_{ice} + (\alpha_{snow} - \alpha_{ice}) \cdot e^{(-\delta t)/\tau} \quad (13)$$

177 where α_{ice} is the bare ice albedo value (0.35), α_{snow} is the snow ice albedo value (0.85) and τ is a decay
 178 rate, which determines how fast the albedo of the ageing snow reaches this value. The decay rate τ is
 179 assumed to have a base value of 10 days similar to values obtained by Schmidt et al. (2017) for wet surfaces
 180 and its maximal value is set based on observations by Oerlemans and Knap (1998) as shown in Table 1.
 181 Furthermore, the albedo α varies depending on the water source that formed the current Icestupa surface.
 182 Correspondingly, the albedo is reset to the value of bare ice albedo if the fountain is spraying water onto
 183 the current ice surface and to the value of fresh snow albedo if a snowfall event occurred. Snowfall events
 184 are assumed if the air temperature is below $T_{ppt} = 1^{\circ}C$ (Fujita and Ageta, 2000).

185 The area fraction f_{cone} of the ice structure exposed to the direct shortwave radiation depends on the
 186 shape considered. The direct solar radiation incident on the AIR surface is first decomposed into horizontal
 187 and vertical components using the solar elevation angle θ_{sun} . For a conical shape, half of the total curved
 188 surface is exposed to the vertical component of the direct shortwave radiation and the projected triangle
 189 of the curved surface is exposed to the horizontal component of the direct shortwave radiation. The solar
 190 elevation angle θ_{sun} used is modelled using the parametrisation proposed by Woolf (1968). Accordingly,
 191 f_{cone} is determined as follows:

$$f_{cone} = \frac{(0.5 \cdot r_{ice} \cdot h_{ice}) \cdot \cos\theta_{sun} + (\pi \cdot r_{ice}^2 / 2) \cdot \sin\theta_{sun}}{\pi \cdot r_{ice} \cdot (r_{ice}^2 + h_{ice}^2)^{1/2}} \quad (14)$$

192 The ERA5 diffuse shortwave radiation is assumed to impact the conical Icestupa surface uniformly.

193 3.2.2 Net Longwave Radiation q_{LW}

194 The net longwave radiation q_{LW} , for which there were no direct measurements available at EP, is
 195 determined as follows:

$$q_{LW} = LW_{in} - \sigma \cdot \epsilon_{ice} \cdot (T_{ice} + 273.15)^4 \quad (15)$$

where T_a represents the measured air temperature, T_{ice} is the modelled surface temperature, both temperatures are given in $^{\circ}\text{C}$, $\sigma = 5.67 \cdot 10^{-8} \text{ J m}^{-2} \text{ s}^{-1} \text{ K}^{-4}$ is the Stefan-Boltzmann constant, LW_{in} denotes the incoming longwave radiation derived from the ERA5 dataset and ϵ_{ice} is the corresponding emissivity value for the Icestupa surface (see Table 1).

3.2.3 Turbulent sensible q_S and latent q_L heat fluxes

The turbulent sensible q_S and latent heat q_L fluxes are computed with the following expressions proposed by Garratt (1992):

$$q_S = c_a \cdot \rho_a \cdot p_a / p_{0,a} \cdot \frac{\kappa^2 \cdot v_a \cdot (T_a - T_{ice})}{(\ln \frac{h_{AWS}}{z_{ice}})^2} \quad (16)$$

$$q_L = 0.623 \cdot L_s \cdot \rho_a / p_{0,a} \cdot \frac{\kappa^2 \cdot v_a (p_{v,a} - p_{v,ice})}{(\ln \frac{h_{AWS}}{z_{ice}})^2} \quad (17)$$

where h_{AWS} is the measurement height above the ground surface of the AWS (in m), v_a is the wind speed in m s^{-1} and M_F denotes fountain water spray mass in kg . c_a is the specific heat of air at constant pressure ($1010 \text{ J kg}^{-1} \text{ K}^{-1}$), ρ_a is the air density at standard sea level (1.29 kg m^{-3}), $p_{0,a}$ is the air pressure at standard sea level (1013 hPa), κ is the von Karman constant (0.4), L_s is the heat of sublimation (2848 kJ kg^{-1}) and z_{ice} (1.7 mm) denotes the roughness length of ice (momentum and scalar). The vapor pressures over air ($p_{v,a}$) and ice ($p_{v,ice}$) was obtained using the following formulation given in WMO (2018):

$$\begin{aligned} p_{v,a} &= 6.107 \cdot 10^{(7.5 \cdot T_a / (T_a + 237.3))} \\ p_{v,ice} &= (1.0016 + 3.15 \cdot 10^{-6} \cdot p_a - 0.074 \cdot p_a^{-1}) \cdot (6.112 \cdot e^{(22.46 \cdot T_{ice} / (T_{ice} + 272.62))}) \end{aligned} \quad (18)$$

where p_a is the measured air pressure in hPa .

3.2.4 Fountain water heat flux q_F

The total energy flux is further influenced through the heat flux caused by the water that was additionally added to the surface of the Icestupa during the time the fountain was running. We take this interaction between the fountain water and the ice surface into account by assuming that the ice surface temperature remains constant at 0°C during time steps when the fountain is active. This process can be divided into two simultaneous steps: (a) the water temperature T_{water} is cooled to 0°C and (b) the ice surface temperature is warmed to 0°C . Process (a) transfers the necessary energy for process (b) throughout the fountain runtime. We further assume that this process is instantaneous, i.e. the ice temperature is immediately set to 0°C within just one time step Δt when the fountain is switched on. Thus, the heat flux caused by the fountain water is calculated as follows:

$$q_F = \begin{cases} 0 & \text{if } \Delta M_F = 0 \\ \frac{\Delta M_F \cdot c_{water} \cdot T_{water}}{\Delta t \cdot A} + \frac{\rho_{ice} \cdot \Delta x \cdot c_{ice} \cdot T_{ice}}{\Delta t} & \text{if } \Delta M_F > 0 \end{cases} \quad (19)$$

221 with c_{ice} as the specific heat of ice.

222 3.2.5 Bulk Icestupa heat flux q_G

223 The bulk Icestupa heat flux q_G corresponds to the ground heat flux in normal soils and is caused by the
224 temperature gradient between the surface layer (T_{ice}) and the ice body (T_{bulk}). It is expressed by using the
225 heat conduction equation as follows:

$$q_G = k_{ice} \cdot (T_{bulk} - T_{ice}) / l_{ice} \quad (20)$$

226 where k_{ice} is the thermal conductivity of ice ($2.123 \text{ W m}^{-1} \text{ K}^{-1}$), T_{bulk} is the mean temperature of the
227 ice body within the Icestupa and l_{ice} is the average distance of any point in the surface to any other point in
228 the ice body. T_{bulk} is initialised as 0°C and later determined from Eqn. 20 as follows:

$$T_{bulk}^{i+1} = T_{bulk} - (q_G \cdot A \cdot \Delta t) / (M_{ice} \cdot c_{ice}) \quad (21)$$

229 Since we assume a conical shape with $r_{ice} > h_{ice}$, l_{ice} cannot be greater than $2r_{ice}$ and also cannot
230 be less than Δx . Therefore, the average distance from any point on the surface to any point inside is
231 $\Delta x \leq l_{ice} \leq r_{ice}$. We calculate q_G here assuming $l_{ice} = r_{ice}/2$.

232 3.2.6 Surface temperature changes and melt energy q_{melt}

233 The available net energy q_{net} partly increases surface temperature, but also contributes to ice melt at the
234 surface of the Icestupa. q_T denotes the energy used on changing the surface temperature T_{ice} and q_{melt}
235 denotes the energy used to produce meltwater. So Eqn. 11 can be rewritten as:

$$q_{net} = q_{melt} + q_T \quad (22)$$

236 We define the freezing energy as $q_{freeze} = (q_{net} - q_L)$. This is because the latent heat always contributes
237 to temperature fluctuations. Now, the temperature fluctuates based on three scenarios, namely (1) the
238 freezing energy flux is negative but cannot freeze all the fountain water output; (2) the freezing energy flux
239 is negative and can freeze all the fountain water output; (3) the freezing energy is positive or the fountain is
240 inactive ($\Delta M_F = 0$). Therefore, we express the rate of change of temperature as follows:

$$\frac{\Delta T}{\Delta t} = \begin{cases} -T_{ice}^{i-1} / \Delta t & \text{if } q_{freeze} < 0 \text{ and } \Delta M_F \geq -q_{freeze} \cdot A \cdot \Delta t / L_f \\ (\Delta M_F \cdot L_f) / (\rho_{ice} \cdot c_{ice} \cdot \Delta x \cdot A \cdot \Delta t) & \text{if } q_{freeze} < 0 \text{ and } \Delta M_F < -q_{freeze} \cdot A \cdot \Delta t / L_f \\ q_{net} / (\rho_{ice} \cdot c_{ice} \cdot \Delta x) & \text{if } \Delta M_F = 0 \text{ or } q_{freeze} > 0 \end{cases} \quad (23)$$

241 Whenever the model predicts $T_{ice}^{i+1} > 0^\circ\text{C}$, then the surface temperature is set to 0°C in the corresponding
242 time step and additional energy contributes to q_{melt} . Combining these requirements, we get:

$$(q_T, q_{melt}) = \begin{cases} (\rho_{ice} \cdot c_{ice} \cdot \Delta x \cdot \frac{\Delta T}{\Delta t}, q_{net} - q_L - \rho_{ice} \cdot c_{ice} \cdot \Delta x \cdot \frac{\Delta T}{\Delta t}) & \text{if } T_{ice}^{i+1} \leq 0^\circ\text{C} \text{ and } \Delta M_F > 0 \\ (\rho_{ice} \cdot c_{ice} \cdot \Delta x \cdot \frac{\Delta T}{\Delta t}, q_{net} - \rho_{ice} \cdot c_{ice} \cdot \Delta x \cdot \frac{\Delta T}{\Delta t}) & \text{if } T_{ice}^{i+1} \leq 0^\circ\text{C} \text{ and } \Delta M_F = 0 \\ (-\rho_{ice} \cdot c_{ice} \cdot \Delta x \cdot \frac{T_{ice}^i}{\Delta t}, q_{net} - q_L + \rho_{ice} \cdot c_{ice} \cdot \Delta x \cdot \frac{T_{ice}^i}{\Delta t}) & \text{if } T_{ice}^{i+1} > 0^\circ\text{C} \text{ and } \Delta M_F > 0 \\ (-\rho_{ice} \cdot c_{ice} \cdot \Delta x \cdot \frac{T_{ice}^i}{\Delta t}, q_{net} + \rho_{ice} \cdot c_{ice} \cdot \Delta x \cdot \frac{T_{ice}^i}{\Delta t}) & \text{if } T_{ice}^{i+1} > 0^\circ\text{C} \text{ and } \Delta M_F = 0 \end{cases} \quad (24)$$

Table 1. Free parameters in the model categorised as constant, uncertain and site parameters. Base value (B) and uncertainty (U) were taken from the literature. For assumptions (assum.), the uncertainty was chosen to be relatively large (5 %). For measurements (meas.), the uncertainty due to parallax errors is chosen to be (1 %).

Constant Parameters	Symbol	Value	References
Van Karman constant	κ	0.4	B: Cuffey and Paterson
Stefan Boltzmann constant	σ	$5.67 \cdot 10^{-8} W m^{-2} K^{-4}$	B: Cuffey and Paterson
Air pressure at sea level	$p_{0,a}$	1013 hPa	B: Mölg and Hardy
Density of water	ρ_w	$1000 kg m^{-3}$	B: Cuffey and Paterson
Density of ice	ρ_{ice}	$917 kg m^{-3}$	B: Cuffey and Paterson
Density of air	ρ_a	$1.29 kg m^{-3}$	B: Mölg and Hardy
Specific heat of water	c_w	$4186 J kg^{-1} ^\circ C^{-1}$	B: Cuffey and Paterson
Specific heat of ice	c_{ice}	$2097 J kg^{-1} ^\circ C^{-1}$	B: Cuffey and Paterson
Specific heat of air	c_a	$1010 J kg^{-1} ^\circ C^{-1}$	B: Mölg and Hardy
Thermal conductivity of ice	k_{ice}	$2.123 W m^{-1} K^{-1}$	B: Bonales et al.
Latent Heat of Sublimation	L_s	$2848 kJ kg^{-1}$	B: Cuffey and Paterson
Latent Heat of Evaporation	L_e	$2514 kJ kg^{-1}$	B: Cuffey and Paterson
Latent Heat of Fusion	L_f	$334 kJ kg^{-1}$	B: Cuffey and Paterson
Gravitational acceleration	g	$9.81 m s^{-2}$	B: Cuffey and Paterson
Uncertain Parameters		Range	
Precipitation	T_{ppt}	$1 ^\circ C$	$\pm 1 ^\circ C$
Temperature threshold			B + U: Fujita and Ageta, Zhou et al.
Ice Emissivity	ϵ_{ice}	0.95	[0.949,0.993] B: Cuffey and Paterson; U: Hori et al.
Ice Albedo	α_{ice}	0.35	$\pm 5 \%$ B: Cuffey and Paterson; U: assum.
Snow Albedo	α_{snow}	0.85	$\pm 5 \%$ B: Cuffey and Paterson; U: assum.
Albedo Decay Rate	τ	10 days	[1, 22] days B: Schmidt et al.; U: Oerlemans and Knap assum.
Ice layer thickness	Δx	5 mm	[1, 10] mm
Site Parameters			
Fountain diameter	nozzle	dia_F	5 mm $\pm 1 \%$ B: meas. ; U: assum.
Fountain height		h_F	$\pm 1 \%$ B: meas. ; U: assum.
Fountain temperature	water	T_{water}	$[0, 9] ^\circ C$ B: meas. ; U: meas.
AWS height		h_{AWS}	$\pm 1 \%$ B: meas. ; U: assum.

243 **3.3 Mass Balance**

244 The mass balance equation is used to derive the water that drains away (M_{runoff}) as follows:

$$\frac{\Delta M_{runoff}}{\Delta t} = \frac{\Delta M_F + \Delta M_{ppt} + \Delta M_{dpt} + \Delta M_{cdt} - \Delta M_{ice} - \Delta M_{melt} - \Delta M_{vapour}}{\Delta t} \quad (25)$$

245 where $\Delta M = M^i - M^{i-1}$. Here $\frac{\Delta M_F}{\Delta t} = d_F$ where d_F is the spray of the fountain measured in $[kg s^{-1}]$;
 246 M_{ppt} is the cumulative precipitation; M_{dpt} is the cumulative accumulation through water vapour deposition;
 247 M_{cdt} is the cumulative accumulation through water vapour condensation; M_{ice} is the cumulative mass of
 248 ice; M_{melt} is the cumulative mass of melt water and M_{vapour} represents the cumulative water vapor loss
 249 by evaporation or sublimation.

250 Precipitation input is calculated as:

$$\frac{\Delta M_{ppt}}{\Delta t} = \begin{cases} \pi \cdot r_{ice}^2 \cdot \rho_w \cdot ppt & \text{if } T_a < T_{ppt} \\ 0 & \text{if } T_a \geq T_{ppt} \end{cases} \quad (26)$$

251 where ρ_w is the density of water ($1000 kg m^{-3}$), ppt is the measured precipitation rate in $[m s^{-1}]$ and
 252 T_{ppt} is the temperature threshold below which precipitation falls as snow. Here, snowfall events were
 253 identified using T_{ppt} as $1^\circ C$. Snow mass input is calculated by assuming a uniform deposition over the
 254 entire circular footprint of the Icestupa.

255 The latent heat flux is used to estimate either the evaporation and condensation processes or sublimation
 256 and deposition processes. To differentiate between these two possibilities, we classify the time steps into
 257 humid or non-humid depending on whether the corresponding relative humidity value is above or below
 258 60% (Stigter et al., 2018). On humid time steps, we assume condensation or evaporation to occur whereas
 259 on non-humid time steps deposition or sublimation can occur. Correspondingly, latent heat of evaporation
 260 (L_e) is used for humid time steps and latent heat of sublimation (L_s) is used for non-humid time steps.

$$\left(\frac{\Delta M_{vapour}}{\Delta t}, \frac{\Delta M}{\Delta t} \right) = \begin{cases} (-q_L \cdot A/L, 0) & \text{if } q_L < 0 \\ (0, q_L \cdot A/L) & \text{if } q_L \geq 0 \end{cases} \quad (27)$$

261 where $L = \begin{cases} L_s & \text{if } RH < 60 \\ L_e & \text{if } RH \geq 60 \end{cases}$ and $M = \begin{cases} M_{dpt} & \text{if } RH < 60 \\ M_{cdt} & \text{if } RH \geq 60 \end{cases}$

262 Deposition or sublimation process involves phase change between ice and vapour. Condensation or
 263 evaporation process involves phase change between water and vapour. To account for this difference, we
 264 define the following terms:

$$(\Delta M_{ice}^*, \Delta M_{melt}^*) = \begin{cases} (\Delta M_{ice} + \Delta M_{vapour} - \Delta M_{dpt}, \Delta M_{melt}) & \text{if } RH < 60 \\ (\Delta M_{ice}, \Delta M_{melt} + \Delta M_{vapour} - \Delta M_{cdt}) & \text{if } RH \geq 60 \end{cases} \quad (28)$$

265 Using the melt energy q_{melt} , we estimate the frozen and melted ice mass (ΔM_{ice} , ΔM_{melt}). Removing
 266 the contribution of precipitation and combining Eqn. 27 and Eqn. 28 we are left with the contribution from
 267 the melt energy as follows:

Table 2. Summary of mass balance components for the EP experiment after the fountain spray was stopped (on 15th February 2019) and at the end of the model run (on 5th April). All parameters except M_F were modelled.

	Mass Component	Fountain spray ends	Model ends
Input	M_F	18060 kg	18060 kg
	M_{ppt}	444 kg	466 kg
	M_{dpt} M_{cdt}	0 kg 7 kg	0 kg 32 kg
Output	M_{melt}	163 kg	1013 kg
	M_{ice}	809 kg	0 kg
	M_{vapour} M_{runoff}	11 kg 17529 kg	16 kg 17529 kg

$$\left(\frac{\Delta M_{ice}^* - \Delta M_{ppt}}{\Delta t}, \frac{\Delta M_{melt}^*}{\Delta t} \right) = \begin{cases} \frac{q_{melt} \cdot A}{L_f} \cdot (-1, 1) & \text{if } q_{melt} \geq 0 \\ \frac{q_{melt} \cdot A}{L_f} \cdot (-1, 0) & \text{if } q_{melt} < 0 \text{ and } \frac{\Delta M_F}{\Delta t} \geq -\frac{q_{melt} \cdot A}{L_f} \\ \left(\frac{\Delta M_F}{\Delta t}, 0 \right) & \text{if } q_{melt} < 0 \text{ and } 0 \leq \frac{\Delta M_F}{\Delta t} < -\frac{q_{melt} \cdot A}{L_f} \end{cases} \quad (29)$$

268 Now, with all the other terms known in Eqn. 25, the water drainage/runoff can be determined.

269 Considering AIRs as water reservoirs, we can quantify their potential through the amount of water they
270 store (storage quantity) and the length of time they store it (storage duration). Another means of comparing
271 different Icestupas is through their water storage efficiency defined accordingly as:

$$\text{Storage Efficiency} = \frac{M_{melt}}{(M_F + M_{ppt} + M_{dpt} + M_{cdt})} \cdot 100 \quad (30)$$

4 MODEL RESULTS

272 The model was forced with meteorological data from 30th January to 5th April 2019 (Fig. 3) and various
273 parameters (see Table 1) to calculate the mass and energy balance of the Icestupa.

4.1 Energy and mass balance calculation

275 Daily averages of some components of the energy balance are shown in Fig. 6 (b). On average during
276 the experiment duration, the total energy balance was almost zero. Net shortwave radiation (33 W m^{-2}),
277 sensible (10 W m^{-2}) and latent heat flux (2 W m^{-2}) with a mostly positive flux towards the surface of the
278 icestupa were compensated by the net longwave radiation (-34 W m^{-2}) and the melt energy (-12 W m^{-2}).
279 The contributions of other fluxes were negligible in comparison.

280 Net shortwave radiation is the main input to, and the most varying energy flux on the ice surface. Its
281 variability is controlled by the surface albedo α and the area fraction f_{cone} which therefore represent key
282 variables in the energy balance (Fig. 7 (b)). Although global radiation flux reached a daily maximum
283 value of 339 W m^{-2} , q_{sw} only went up to 98 W m^{-2} . This is caused by the fact that only about 30 % of
284 the direct solar radiation influenced the Icestupa surface as shown by the area fraction f_{cone} in Fig. 7 (a).

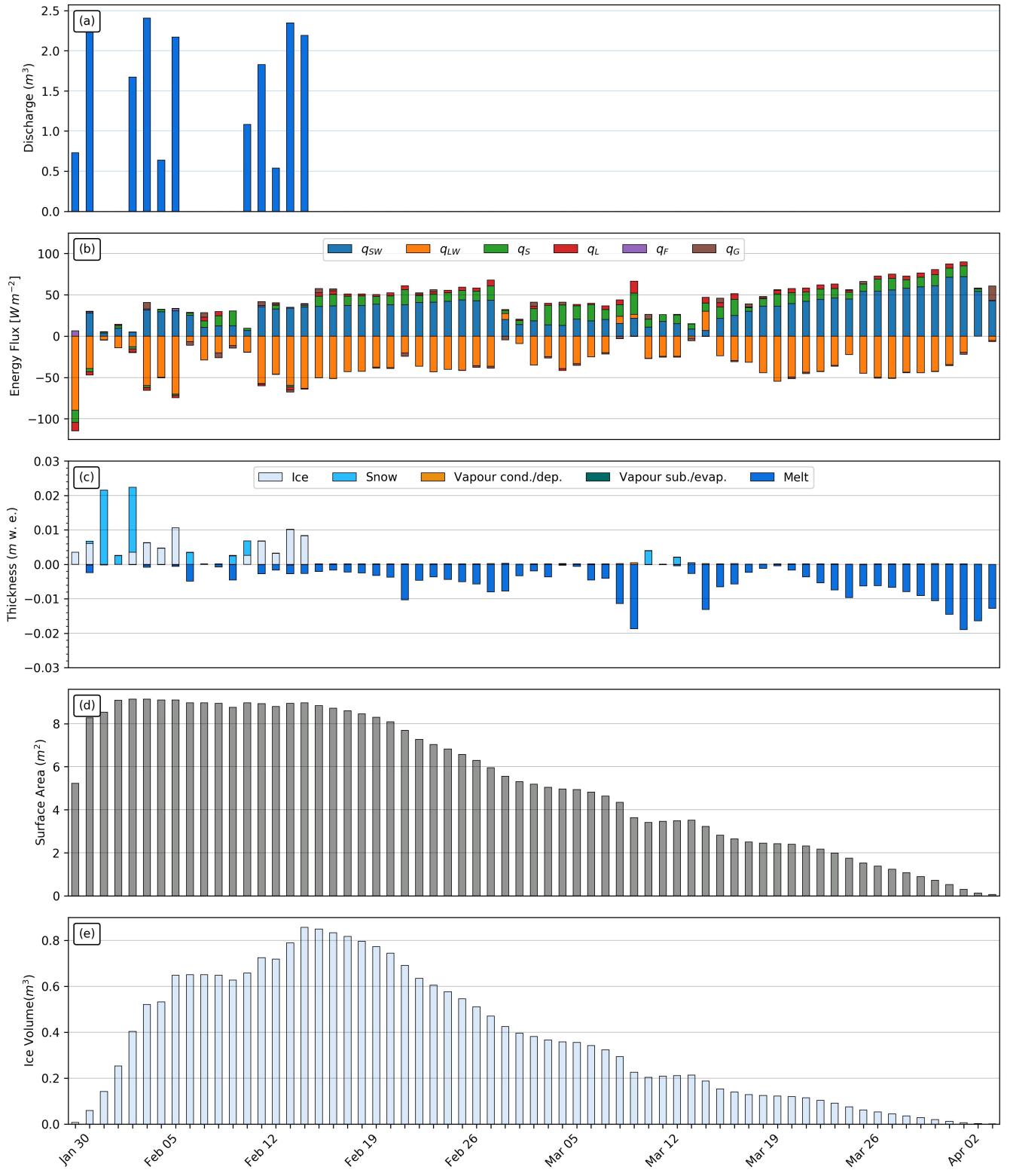


Figure 6. (a) Fountain discharge (b) energy flux components, (c) mass flux components (d) surface area and (e) volume of the Icestupa in daily time steps. q_{SW} is the net shortwave radiation; q_{LW} is the net longwave radiation; q_L and q_S are the turbulent latent and sensible heat fluxes. q_F represents the interactions of the ice-water boundary during fountain on time steps. q_G quantifies the heat conduction process between the Icestupa surface layer and the ice body.

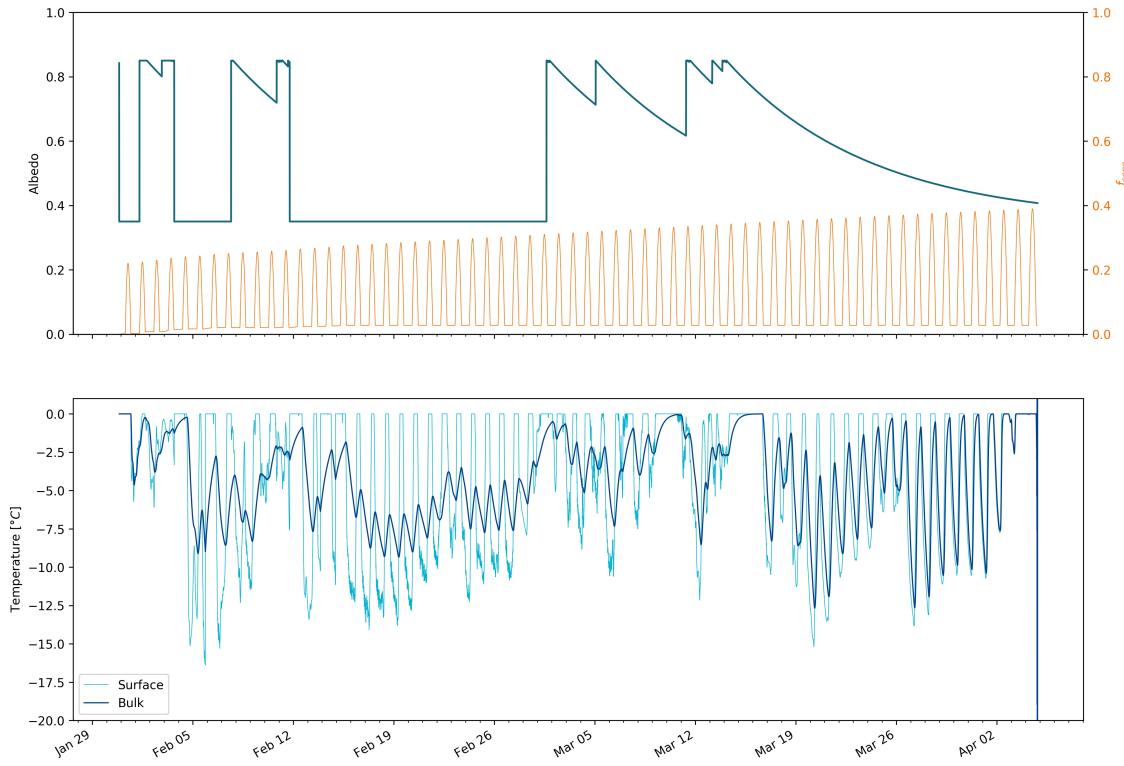


Figure 7. Some derived parameters of the model, namely, albedo and f_{cone} (a), Surface and bulk temperature (b). In (a), the green curve shows how snow and fountain-on events reset albedo between ice albedo and snow albedo. The decay of the snow albedo to ice albedo can also be observed. The orange curve shows how the solar radiation area fraction varied diurnally with variations in the solar elevation angle. In (b), the surface temperature (light blue curve) was forced to be 0 °C during fountain activity. The corresponding bulk temperature is shown with the dark blue curve.

285 Snowfall is the atmospheric variable connected most closely and proportionally to albedo. Higher and/or
 286 more frequent snowfall thus decreases the energy available for melt due to the corresponding increase in α .
 287 q_{LW} was predominantly negative indicating that this energy balance component drove the freezing of
 288 the ice structure. Daily values of q_{LW} ranged from -89 to 21 Wm^{-2} . Turbulent sensible heat flux q_S
 289 contributed mostly to the melt of the ice structure. Daily values of q_S ranged from -14 to 26 Wm^{-2} .
 290 Deposition/condensation was favored over evaporation/sublimation during 95 % of the model runtime.
 291 Daily values of the turbulent latent heat flux q_L ranged from -10 to 15 Wm^{-2} . Since the mean of q_L was
 292 positive, the Icestupa gained mass cumulatively from the atmosphere due to the condensation process.
 293 Fountain water heat flux q_F had a mean of zero as it was only nonzero during 1002 time steps or around
 294 100 hours. Daily values of q_F ranged from -1 to 6 Wm^{-2} . The contribution of heat flux by conduction
 295 q_G was also minimal as it only varied between -6 to 12 Wm^{-2} with a mean of 0 Wm^{-2} . The energy
 296 contributing to surface temperature changes (q_T) was insignificant in comparison to the energy spent on
 297 freezing and melting (q_{melt}). The resulting bulk temperature and the surface temperature are shown in Fig.
 298 7 (b). For the total considered period, q_{LW} accounted for 33.8% of overall energy turnover. The energy
 299 turnover is calculated as the sum of energy fluxes in absolute values. q_{SW} accounted for 31.3%, followed
 300 by q_{melt} (19%), q_S (9.6%), q_L (3.6%), q_G (2.4%), q_F (0.2%) and q_T (0.4%).

301 Fig. 6 (c) represents the mass fluxes associated with these energy exchanges expressed in m w.e. It
 302 shows the ice and meltwater formed due to q_{melt} , snow accumulated due to precipitation, water vapour

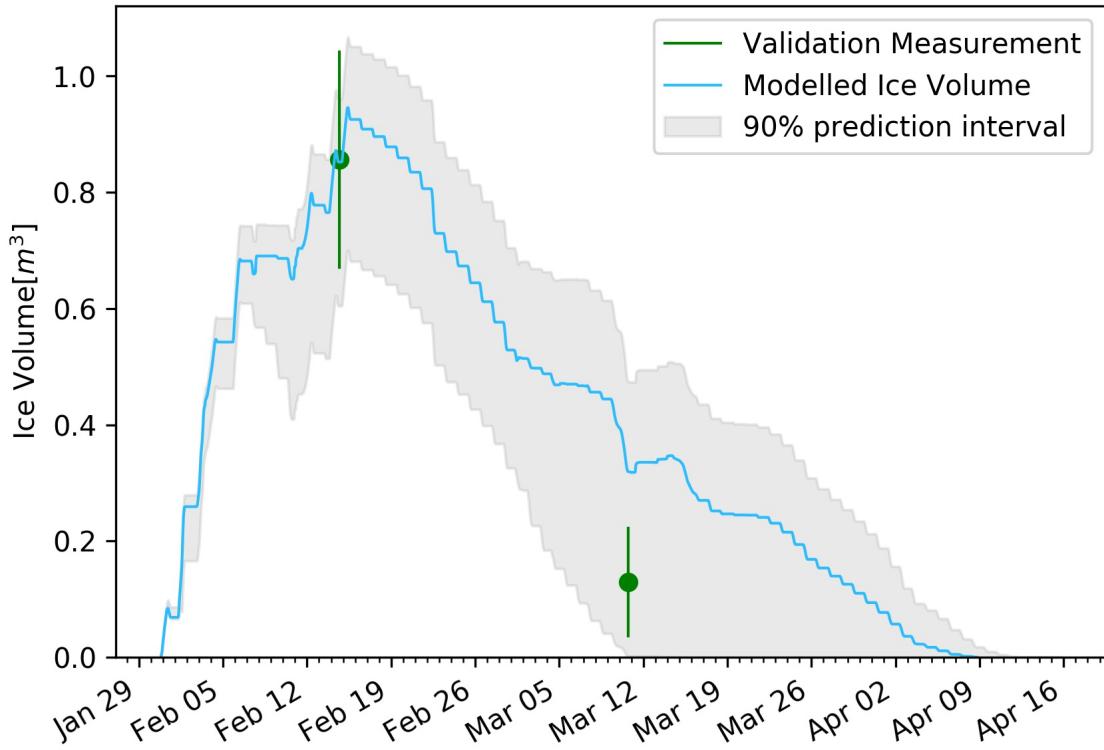


Figure 8. Modelled ice volume during the lifetime of the EP Icestupa (blue curve). Green line segments indicate the first and second validation measurements. The prediction interval is based on the ice volume uncertainty caused by the most sensitive parameters, namely, temperature threshold below which precipitation falls as snow and the ice emissivity.

303 deposition/condensation and sublimation/evaporation due to q_L . Growth rate ($\frac{\Delta M_{ice}}{\Delta t}$) shows a better
 304 correlation with net energy flux ($r^2 = 0.15$) compared to Icestupa surface area ($r^2 = 0.04$). This is because
 305 the variance in growth rate is mostly due to the variance in q_{net} as illustrated in Fig. 6. Since r_{ice} was
 306 initialised with the spray radius r_F , the surface area maintains a maximum initially until the energy flux
 307 becomes positive. This trend favours the positive over the negative thickness changes resulting in a steep
 308 increase and gradual melting of ice volume as can be seen in Fig. 8.

309 The total water used for the Icestupa development includes contributions from the fountain (97.3%),
 310 snowfall (2.5 %), condensation (0.2 %) as shown in Table 2. The maximum ice mass during the whole
 311 measurement period was 809 kg, which occurred after the last fountain run on Feb 16th 2019 in the morning.
 312 Therefore, in the case of EP we used a water input of 18,558 kg, with a resultant storage efficiency of only
 313 5.5 %.

5 MODEL SENSITIVITY AND UNCERTAINTY ANALYSIS

314 The icestupa model can be regarded as a function $f(x_1, x_2 \dots, x_n) = (y_1, y_2 \dots, y_m)$, where
 315 $(x_1, x_2 \dots, x_n)$ are the model parameters and $(y_1, y_2 \dots, y_m)$ are the model outputs. The influence of each
 316 parameter on the output variables of interest were quantified and the most important physical parameters
 317 for the subsequent uncertainty analysis were determined. The sensitivity of a parameter x_j is determined
 318 by keeping all other parameters $x_i, i \neq j$ fixed at their baseline value and varying x_j within values that are
 319 physically plausible.

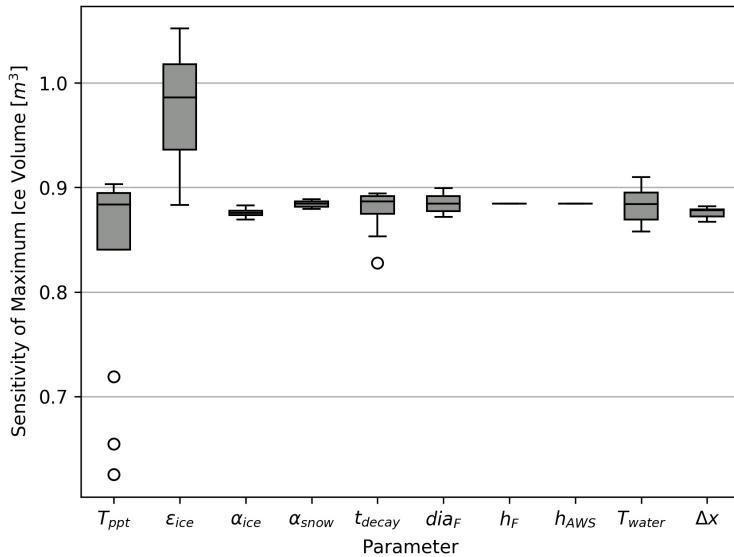


Figure 9. Sensitivities of maximum ice volume to all the uncertain and site parameters used in the model (Table 1). Outliers in the bar plot are shown as 'o'.

320 A sensitivity study on the parameters (listed in Table 1) was performed with the maximum ice volume
 321 as the target variable. All the parameters were assumed to be independent of each other with a uniform
 322 distribution. This assumption ignores the auto-correlation present among the parameters associated with
 323 the albedo parameterisation. The range of uncertain parameters were set based on available literature values
 324 or varied $\pm 5\%$ from the base value if no such reference was available. The uncertainty of all the site
 325 parameters were caused due to parallax errors during manual measurement. This was quantified with a
 326 range of $\pm 1\%$ from the base value. However, it must be kept in mind that, even though intended to be as
 327 objective as possible, the selection of a parameter range has a subjective part that influences the results and
 328 conclusions obtained in this analysis. The variation of the model outputs y_k is evaluated to quantify the
 329 local sensitivities $s_{j,k}$ that are defined here as the 95% range of the simulated outputs.

330 To perform the uncertainty analysis, we included only parameters that influence the maximum ice volume
 331 by at least $0.1 m^3$. All other parameters were fixed at their baseline value. Fig. 9 shows all the variance
 332 produced by these uncertain parameters in maximum ice volume calculation. It shows that ϵ_{ice} and T_{ppt} are
 333 the only parameters with a maximal sensitivity of more than $0.1 m^3$ for the maximum ice volume estimate.
 334 Consequently, all other parameters were excluded from the subsequent uncertainty analysis.

335 The temperature threshold below which precipitation falls as snow (T_{ppt}) was found to be the most
 336 sensitive parameter. It is used in the model to reset the albedo to snow albedo and determine snow
 337 precipitation events. The lower T_{ppt} parameter the higher the albedo (as the Icestupa surface has a lower
 338 albedo when ice-covered than when snow-covered). The variation of T_{ppt} by $\pm 1^\circ C$ caused maximum ice
 339 volume variation of $0.84 \pm 0.2 m^3$.

340 Ice emissivity was also found to be a sensitive parameter. The higher the ice emissivity the larger the
 341 maximum ice volume as the emitted longwave radiation increases with ice emissivity. Variation of ϵ_{ice} by
 342 5% caused a maximum ice volume range from $0.98 \pm 0.1 m^3$.

343 In total, the sensitivity analysis required 120 simulations, and the uncertainty analysis a total of 32
 344 simulations.

6 DISCUSSION

345 6.1 Model validation quality

346 We first evaluate the model against the validation measurements at the EP site. The uncalibrated model
 347 is able to capture both the freezing and the melting process sufficiently well as the modelled ice volume
 348 lies within the uncertainty of both validation measurements. Furthermore, the validation measurements fit
 349 well within the estimated model uncertainty. However, since this validation is based on only two points, it
 350 does limit the confidence in the model results. Even though the model estimates validate well with the ice
 351 volume, the same is not the case for the surface area. The surface area estimated from the first validation
 352 measurement is just around $4 m^2$, roughly half the model estimated surface area. However, the validation
 353 surface area estimate again underestimates the surface area as the actual surface area in contact with the
 354 atmosphere could have been amplified by the inherent roughness of the ice surface. Another major cause of
 355 this discrepancy was the conical shape assumption, as in reality, the Icestupa shape ranged between a cone
 356 and a cylinder (Fig. 2). The sensitivity of the model results to these errors was further amplified due to the
 357 relatively small volume of the EP Icestupa. In summary, better and frequent validation measurements on a
 358 more voluminous Icestupa would have increased confidence on the model results.

359 6.2 Important assumptions

360 In the sensitivity and uncertainty analysis presented above, we did not account for several general
 361 assumptions and parametrisation choices that may cause model errors. Some assumptions and their
 362 potential to cause errors are discussed below.

- 363 • Turbulent Sensible and Latent Heat Fluxes: The method used to calculate the turbulent heat fluxes
 364 by Garratt (1992) assumes that the turbulent heat fluxes are acting over a uniform planar surface to
 365 determine the roughness length. Since our application is on a conical surface, the distance to the ice
 366 surface is not uniform and well defined. Hence, z_{ice} has no real physical significance here.
- 367 • Droplet flight time loss: Water losses during the flight time of fountain droplets were neglected making
 368 all the fountain spray available for freezing. For the EP experiment, inclusion of this parameter does
 369 not influence results since it is already accounted for in the runoff water discharge rate which was at
 370 least $3 l min^{-1}$.
- 371 • Nucleation of droplets: Corresponding to droplet flight time, ice/snow formation is also possible before
 372 surface contact if nucleation occurs during flight time. For the EP experiment, this process will further
 373 increase the freeze rate and hence the storage efficiency. This process is neglected for model simplicity.
- 374 • Shape Effects: The suppression of heat exchange between the snow/ice surface and the air adjacent
 375 to the surface effectively slows down snow/ice ablation in spring and promotes the stagnation of the
 376 cold air within topographical depressions (Fujita et al., 2010). The quantitative contribution of the
 377 atmospheric decoupling over melting snow/ice for the total mass and energy balance is ignored in the
 378 model.

379 6.3 Icestupa construction decisions

380 There are several decisions one has to take when constructing Icestupas. These can be broadly divided
 381 into two types of decisions, namely location and fountain decisions. Both the meteorological conditions
 382 of the location and the surface area produced by the fountain significantly influence the observed growth
 383 rate. Since our validation is restricted to just one location, we restrict our discussion to the optimization
 384 possibilities of Icestupa constructions through fountain decisions.

385 Assuming a constant spray for the fountain, we can divide the fountain decisions into fountain state
 386 (on/off) and type (height and nozzle diameter). From an energy balance point of view, the fountain should

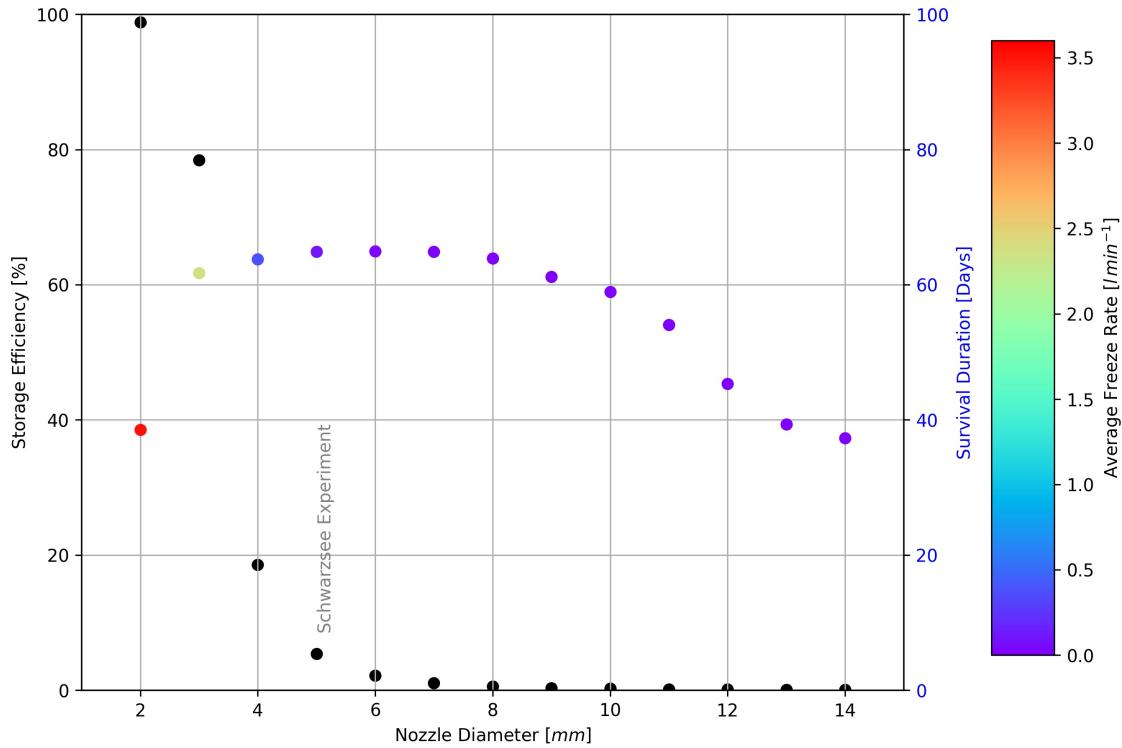


Figure 10. Variation in storage efficiency (black dots) and storage duration (coloured dots) with changes in fountain's nozzle diameter. The dot colours represent average freeze rate based on the color bar.

387 be switched on for all time intervals when $q_{net} < 0$ which would have increased the fountain discharge
 388 duration from 90 hours to 850 hours. However, in our experiment, the fountain state decision was set on the
 389 basis of prior Icestupa construction experience where a critical temperature of -5°C was recommended.
 390 But ambient temperature was a poor indicator of q_{net} ($r^2 = 0.17$). Therefore, using air temperature to
 391 determine when the fountain should be switched on was not ideal.

392 The fountain type used can be characterised by the physical structure of the fountain, namely its height and
 393 nozzle diameter. Maintaining the same spray rate and height, one can optimize the Icestupa development
 394 by identifying the minimum nozzle diameter that yields the maximum storage efficiency. Since we never
 395 changed the fountain height for the EP Icestupa, we only focus on optimization of fountain diameter below.

396 Fig. 10 shows reducing the nozzle diameter to 3 mm increases storage efficiency up to 78 % without
 397 compromising much on storage duration. The corresponding storage quantity of the 3 mm nozzle diameter
 398 was more than 23 times higher than the 5 mm fountain used in our experiment. This is because the spray
 399 radius r_F of the 3 mm fountain was much higher at 8.5 m compared to the 1.7 m spray radius of the 5
 400 mm fountain. Here, we define growth rate as freeze rate when fountain is active and melt rate otherwise.
 401 Therefore, this higher spray radius both increases the freeze rate and increases the melt rate since they are
 402 both directly proportional to the surface area. However, since the freeze rate cannot increase beyond a spray
 403 rate of 3.6 lmin^{-1} (except during precipitation or deposition/condensation events), an optimum spray
 404 radius or nozzle diameter exists, beyond which storage duration suffers due to a disproportionate increase
 405 in melt rate compared to the freeze rate. So even though 3 mm nozzle diameter had a much higher storage
 406 quantity than the 5 mm nozzle, its storage duration was around 2 days less than the 5 mm nozzle. One
 407 physical cause of this effect is the different shapes of both the ice structures. A flat sheet of ice (effectively

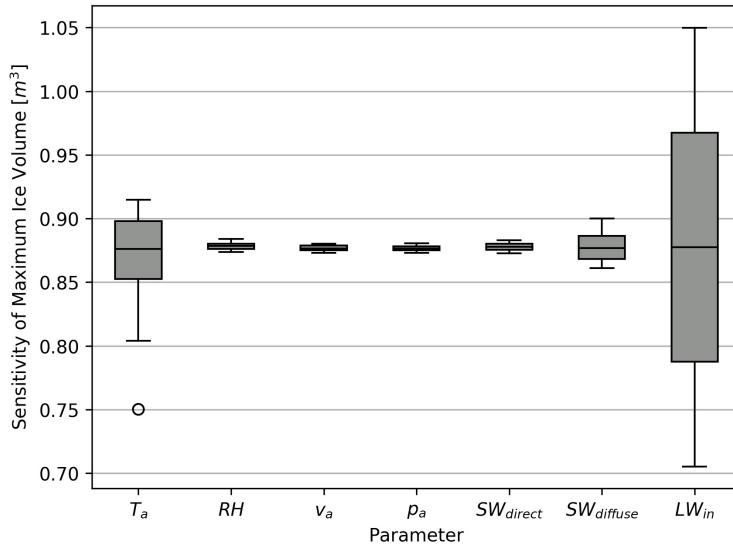


Figure 11. Sensitivities of maximum ice volume to all the meteorological input variables used in the model (Figure 3). Outliers in the bar plot are shown as 'o'.

408 a cone with a high spray radius) with higher mass might have a storage duration shorter than a conical ice
 409 structure. As the spray radius decreases with increasing nozzle diameter, the ice structure's average slope
 410 increases and so the 5 mm nozzle's ice structure is "more" conical than the 3 mm ice structure. Fig. 10
 411 shows that a nozzle diameter of 3 mm has an average freeze rate ($2.4 \text{ l min}^{-1} \text{ w.e.}$) which is large enough
 412 to increase the storage efficiency and small enough to not reduce the storage duration of the Icestupa
 413 significantly.

414 6.4 Sensitivity of meteorological variables

415 The sensitivity of maximum ice volume associated with changes in meteorological variables was
 416 quantified. The meteorological variables used in the model are T_a , RH , v_a , p_a , SW_{direct} , $SW_{diffuse}$
 417 and LW_{in} . In each perturbation scenario, a certain meteorological parameter was perturbed to a certain
 418 extent. Among those changes, T_a was absolute changes ($\pm 1^\circ\text{C}$) and other variables were relative variations
 419 ($\pm 5\%$). All perturbations were implemented uniformly in space on the modeling domain and in time
 420 through the modeling periods. To separate the effects of individual meteorological variables, only one
 421 parameter was changed in each case, while all the other variables were kept unchanged. One has to be
 422 aware that in reality many of these meteorological variables are linked and therefore, autocorrelated.

423 As can be seen from Fig. 11, the maximum ice volume is significantly influenced by variations in
 424 LW_{in} , T_a and $SW_{diffuse}$. Therefore, an optimal construction location needs to have minimal exposure to
 425 longwave radiation from nearby topographical features, a lower mean winter temperature and minimal
 426 winter cloud cover to attain a higher maximum ice volume.

427 6.5 Schwarzsee vs Leh Icestupa

428 It could be argued that the relatively small EP Icestupa cannot be compared with the much larger Icestupas
 429 in Ladakh which store millions of litres of water for several months (see Appendix 8.1). However, this is
 430 the only Icestupa dataset currently available for such a model validation.

431 Table 2 clearly shows that for our EP experiment most of the input water (94.5 %) simply drained away
 432 (M_{runoff}). This high water loss through drainage is due to the fact that the average spray rate of the fountain

433 $((\frac{\Delta M_F}{\Delta t})_{mean} = 3.6 \text{ l min}^{-1})$ far exceeded the max Icestupa growth rate ($((\frac{\Delta M_{ice}}{\Delta t})_{max} = 0.9 \text{ l min}^{-1}$ (w.e.)
 434).

435 In the city of Leh, Ladakh at an altitude of 3500 m a.s.l. the air temperature shows values down to
 436 -27.9°C in winter (Chevuturi et al., 2018) whereas EP had a minimum temperature of just -11.6°C
 437 during the study period. Moreover, subzero temperatures were only reached for 7 nights of fountain
 438 operation at the EP site compared to the 43 nights of fountain operation possible in Ladakh (see Appendix
 439 8.1). Thus, the Icestupa growth rate is expected to be much higher in Ladakh. However, water spray rates
 440 in Ladakh are also much higher (around 210 l min^{-1}). This suggests that the water losses in Ladakh could
 441 be caused by excessive fountain spray.

7 CONCLUSIONS

442 We outlined a methodology for estimating ice, liquid water, water vapour and runoff quantities produced
 443 during the construction of an Icestupa using measurements of fountain spray rate, air temperature, radiation,
 444 humidity, pressure, wind and cloudiness at the EP study site. The comparison with validation measurements
 445 at two different dates during the experiment led to satisfying results, although a more rigorous model
 446 validation was not possible due to few icestupa volume measurements.

447 According to the model, the EP Icestupa achieved a storage quantity of 1005 litres of water with a storage
 448 duration of 65 days. However, the corresponding storage efficiency of 6 % was very low for a water input of
 449 18,560 litres. These estimates were most sensitive to the temperature threshold that determined precipitation
 450 phase and ice emissivity parameters which created an uncertainty of 0.2m^3 in the maximum ice volume
 451 calculated. This is to be expected as net longwave radiation and net shortwave radiation together accounted
 452 for around 65 % of the overall energy turnover.

453 Although the location, storage quantity and duration of our experimental EP Icestupa are not representative
 454 of the much larger Icestupas of Ladakh, the model results do support the hypothesis that there could be
 455 considerable water loss during the formation of Icestupas particularly due to excessive fountain spray.
 456 Using model calculations, it was shown that a decreased fountain nozzle diameter of 3 mm can increase the
 457 storage efficiency drastically. This is because a change in the fountain nozzle diameter causes an effective
 458 change of the ice surface area over which the net energy flux can act. This result has relevance on the future
 459 design of Icestupa fountains. However, care has to be taken as our model is currently only validated by one
 460 experiment at the EP site. Further experiments at different locations with different fountains are required to
 461 better understand the influence of construction decisions on the results.

8 APPENDIX

462 8.1 Ladakh Icestupa 2014/15

463 A 20 m tall Icestupa (Wangchuk, 2015c) was built in Phyang village, Ladakh at an altitude of 3500
 464 m a.s.l. Assuming a conical shape with a diameter of 20 m, the corresponding volume of this Icestupa
 465 becomes 2093 m^3 or $1,920 \text{ m}^3$ w.e. The fountain sprayed water at a rate of 210 l min^{-1} (Wangchuk,
 466 2015e) from 21st January (Wangchuk, 2015a) to at least until 5th March 2015 (Wangchuk, 2015b) (around
 467 43 nights). Assuming fountain spray was active for 8 hours each night, we estimate water consumption to
 468 be around $4,334 \text{ m}^3$. Thus, during the construction/freezing period of the Icestupa, roughly 56 % of the
 469 water provided was wasted. The actual water loss is bound to be much higher due to further vapour losses
 470 during the melting period. This Icestupa completely melted away on 6th July 2015 (Wangchuk, 2015d).
 471 Therefore, the storage duration was 166 days or roughly 5 months.

CONFLICT OF INTEREST STATEMENT

472 The authors declare that the research was conducted in the absence of any commercial or financial
473 relationships that could be construed as a potential conflict of interest.

AUTHOR CONTRIBUTIONS

474 SB wrote the initial version of the manuscript. MH, ML, SW, JO, and FK commented on the initial
475 manuscript and helped improve it. SB developed the methodology with inputs from MH. SB performed the
476 analysis with support from MH and ML. SB and MH participated in the fieldwork.

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DATA AVAILABILITY STATEMENT

484 The data and code used to produce results and figures will be published at a later stage and can, until then,
485 be obtained from the authors upon request.

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