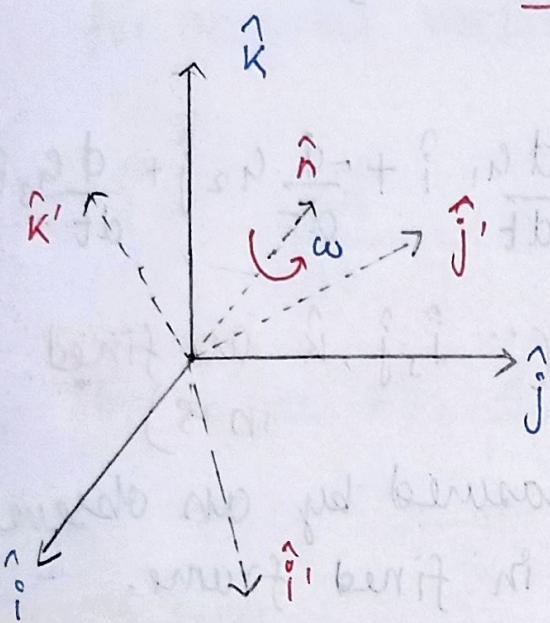


## A) Rotating frames of Reference :-

consider a fixed frame 'S' with  $(\hat{i}, \hat{j}, \hat{k})$  unit vectors.

→ Frame  $S'$  originally coincident with  $S$  is rotating about the common origin with velocity  $\vec{\omega} = \hat{n} \omega$



Now, consider some vector  $\vec{G}$ .

physically, a change in coordinate system should not change anything.

i.e

$$\vec{G}_{\text{fixed}} = \vec{G}_{\text{rot}}$$

(but individual components in  $S, S'$  might differ)

$$\vec{G}_{\text{fixed}} = G_1 \hat{i} + G_2 \hat{j} + G_3 \hat{k}$$

$$\vec{G}_{\text{rot}} = G'_1 \hat{i} + G'_2 \hat{j} + G'_3 \hat{k}$$

→ Similarly time derivatives of  $\vec{G}_{\text{fixed}}$  &  $\vec{G}_{\text{rot}}$   
should be the same.

$$\text{i.e } \frac{d}{dt} [\vec{G}_{\text{fixed}}] = \frac{d}{dt} [\vec{G}_{\text{rot}}]$$

$$\text{LHS} = \left[ \frac{d\vec{G}}{dt} \right]_{\text{fixed}} = \frac{d\vec{a}_1}{dt} \hat{i} + \frac{d\vec{a}_2}{dt} \hat{j} + \frac{d\vec{a}_3}{dt} \hat{k}$$

( ∵  $\hat{i}, \hat{j}, \hat{k}$  are fixed  
in s)

∴ measured by an observer  
in fixed frame.

RHS

$$= \frac{d}{dt} [\vec{G}_{\text{rot}}]$$

$$= \frac{d}{dt} [a'_1 \hat{i}' + a'_2 \hat{j}' + a'_3 \hat{k}']$$

$$= \left( \frac{da'_1}{dt} \hat{i}' + \frac{da'_2}{dt} \hat{j}' + \frac{da'_3}{dt} \hat{k}' \right) +$$

$$a'_1 \frac{d}{dt} \hat{i}' + a'_2 \frac{d}{dt} \hat{j}' + a'_3 \frac{d}{dt} \hat{k}'$$

$$= \left( \frac{d\vec{G}}{dt} \right)_{\text{rot}} + G_1' \frac{d\hat{i}'}{dt} + G_2' \frac{d\hat{j}'}{dt} + G_3' \frac{d\hat{k}'}{dt}$$

Recall,

for an orbit vector  $\vec{A}$ :  $\delta\vec{A} = (\delta\theta \hat{n}) \times \vec{A}$

OR:  $\frac{\Delta\vec{A}}{\Delta t} = \hat{n} \frac{\Delta\theta}{\Delta t} \times \vec{A}$

so, as  $\Delta t \rightarrow 0$ ,  $\frac{d}{dt} \vec{A} = \vec{\omega} \times \vec{A}$

$$\vec{\omega} = \hat{n} \frac{d\theta}{dt}$$

general form:

$$\frac{d(\ )}{dt} = \vec{\omega} \times (\ )$$

Example  
like

$$\frac{d\hat{i}'}{dt} = \vec{\omega} \times \hat{i}'$$

$$\therefore \left( \frac{d\vec{G}}{dt} \right)_{\text{fixed}} = \left( \frac{d\vec{G}}{dt} \right)_{\text{rot}} + \vec{\omega} \times (G_1' \hat{i}' + G_2' \hat{j}' + G_3' \hat{k}')$$

$$\hat{G}' = \vec{G}$$

$$= \left( \frac{d\vec{G}}{dt} \right)_{\text{rot}} + \vec{\omega} \times \vec{h}$$

If  $\vec{r} = \vec{s}$ , we have:

$$\left( \frac{d\vec{r}}{dt} \right)_{\text{fixed}} = \left( \frac{d\vec{s}}{dt} \right)_{\text{rot}} + \vec{\omega} \times \vec{s}$$

OR  $\vec{v}_{\text{fixed}} = \vec{v}_{\text{rot}} + \vec{\omega} \times \vec{s}$

$$\vec{v}_o = \vec{v}$$

$$\frac{d}{dt} \vec{v}_o = \left( \frac{d}{dt} \vec{v}_o \right)_{\text{rot}} + (\vec{\omega} \times \vec{v}_o)$$

$$= \frac{d}{dt} (\vec{v} + \vec{\omega} \times \vec{s}) + \vec{\omega} \times (\vec{v} + \vec{\omega} \times \vec{s})$$

$$\rightarrow \frac{d\vec{v}_o}{dt} = \frac{d}{dt} (\vec{v} + \vec{\omega} \times \vec{s}) + \vec{\omega} \times (\vec{v} + \vec{\omega} \times \vec{s})$$

$$= \frac{d\vec{v}}{dt} + \vec{\omega} \times \vec{s} + 2(\vec{\omega} \times \vec{s}) + \vec{\omega} \times (\vec{\omega} \times \vec{s})$$

measured  
in s

So, for a particle of mass  $m$  undergoing motion:

$$m \frac{d\vec{v}}{dt} = m \frac{d\vec{v}_0}{dt} + m(\vec{r} \times \vec{\omega}) + 2m(\vec{v} \times \vec{\omega}) + m(\vec{\omega} \times \vec{r}) \times \vec{\omega}$$

Measured in  $\cancel{S'}$

True Inertial force

Euler force

Coriolis force

fictitious force or Pseudo force

Centrifugal force

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Alternatively, Lagrangian formalism

L to be evaluated in an inertial frame.

$$L = \frac{1}{2} m |\vec{v}_0|^2 - V$$

$$\frac{\partial}{\partial \vec{v}} (m\vec{v} \cdot \vec{v}) = m$$

$$L = \frac{1}{2} m [(\vec{v} + (\vec{\omega} \times \vec{r}))]^2 - V$$

$$L = \frac{1}{2} m \vec{v}^2 + \frac{1}{2} m \vec{v} \cdot (\vec{\omega} \times \vec{r}) + \frac{1}{2} m (\vec{\omega} \times \vec{r}) \cdot (\vec{\omega} \times \vec{r}) - V$$

$$\frac{\partial L}{\partial \vec{v}} = m \vec{v} + m(\vec{\omega} \times \vec{r}) = m(\vec{v} + \vec{\omega} \times \vec{r})$$

$$= m \vec{v}_0$$

$$= \vec{p}$$

(momentum corresponding to  $\vec{r}$ )

This is true momentum & not  $m \vec{v}$

Foucault Pendulum: proof that the earth is a rotary frame.

$$L = \frac{1}{2} m |(\vec{v} + \vec{\omega} \times \vec{r})|^2 - V$$

$$= \frac{1}{2} m v^2 + m \vec{v} \cdot (\vec{\omega} \times \vec{r}) + \frac{1}{2} m (\vec{\omega} \times \vec{r}) \cdot (\vec{\omega} \times \vec{r}) - V$$

$$\frac{\partial L}{\partial \vec{r}} = m(\vec{v} \times \vec{\omega}) - m(\vec{\omega} \times \vec{r}) \times \vec{\omega} - \vec{\nabla} V$$

E-L equations

$$\frac{d}{dt} \left[ m \vec{v} \right] + m(\vec{\omega} \times \vec{r}) + m(\vec{\omega} \times \vec{v}) - m(\vec{v} \times \vec{\omega}) + m(\vec{\omega} \times \vec{r}) \times \vec{\omega} - \vec{\nabla} V = 0$$

OR

$$\left[ m \frac{d\vec{v}}{dt} = -\vec{\nabla} V + m(\vec{r} \times \dot{\vec{\omega}}) + 2m(\vec{v} \times \vec{\omega}) + m(\vec{\omega} \times \vec{r}) \times \vec{\omega} \right]$$

$$\begin{aligned} & \text{Now } (\vec{\omega} \times \vec{r}) \cdot (\vec{\omega} \times \vec{r}) \rightarrow (\vec{a} \times \vec{b}) \cdot (\vec{b} \times \vec{c}) \\ & = \vec{a} \cdot (\vec{b} \times \vec{c}) \cdot \vec{b} \end{aligned}$$

$$= (\vec{a} \times \vec{b}) \cdot \vec{c}$$

If you write down E-L eqns - the equations are identical to what was obtained earlier.

④ Energy in rotating frame (related to that in a stationary frame)

recall

$$\frac{\partial L}{\partial \dot{q}_i} \dot{q}_i - L \equiv \text{Jacobi Integral} \quad \left. \right\} = E$$

i.e.  $E = \vec{p} \cdot \vec{v} - L$

$$= m \vec{v}_0 (\vec{v}_0 - \vec{\omega} \times \vec{r}) - L$$

$$E = m \vec{v}_0 (\vec{v}_0 - \vec{\omega} \times \vec{r}) - \frac{1}{2} m v_0^2 + V$$

$$E = \underbrace{\frac{1}{2} m v_0^2}_{E_0} + \underbrace{V - m v_0 (\vec{\omega} \times \vec{r})}_{\vec{p} \cdot (\vec{\omega} \times \vec{r})}$$

$$E = E_0 - \vec{\omega} \cdot (\vec{r} \times \vec{p}) = E_0 - \vec{\omega} \cdot \vec{l}$$

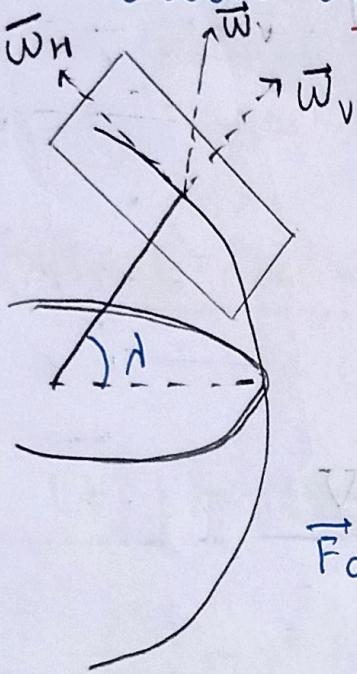
$$E = E_0 - \vec{\omega} \cdot \vec{l}$$

angular momentum

## ① Coriolis Force Effects: on river flow

Geographical latitude ' $\lambda$ '

→ choose direction of flow  $\hat{i}$  direction



$$\vec{v} = v \hat{i} \quad (\text{choosing direction of flow to be along } \hat{i})$$

$$\omega_v = \omega \cos\left(\frac{\pi}{2} - \lambda\right) = \omega \sin \lambda$$

$$\omega_u = \omega \sin\left(\frac{\pi}{2} - \lambda\right) = \omega \cos \lambda$$

$$\begin{aligned}\vec{F}_C &= 2m((\vec{v} \times \vec{\omega}_H) + (\vec{v} \times \vec{\omega}_V)) \\ &= 2m(v \omega \cos \lambda (\hat{i} \times (-\hat{j}))) + \\ &\quad (v \omega \sin \lambda (\hat{i} \times \hat{k})) \\ &= -2mv \omega (\sin \lambda \hat{j} + \cos \lambda \hat{k})\end{aligned}$$

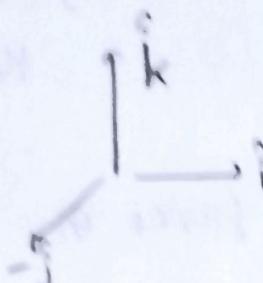
→ along  $\hat{j}$  direction there is also a component of the Coriolis force ie along  $\hat{j}$ :  $-2v \omega \sin \lambda$

is the Coriolis acceleration

latitude dependence

- If  $\lambda > 0$  - i.e in the northern hemisphere acc<sup>n</sup> is  $-2v \omega \sin \lambda$  (-ve)
- If  $\lambda < 0$  - i.e in the southern hemisphere acc<sup>n</sup> is  $2v \omega \sin \lambda$  (+ve)

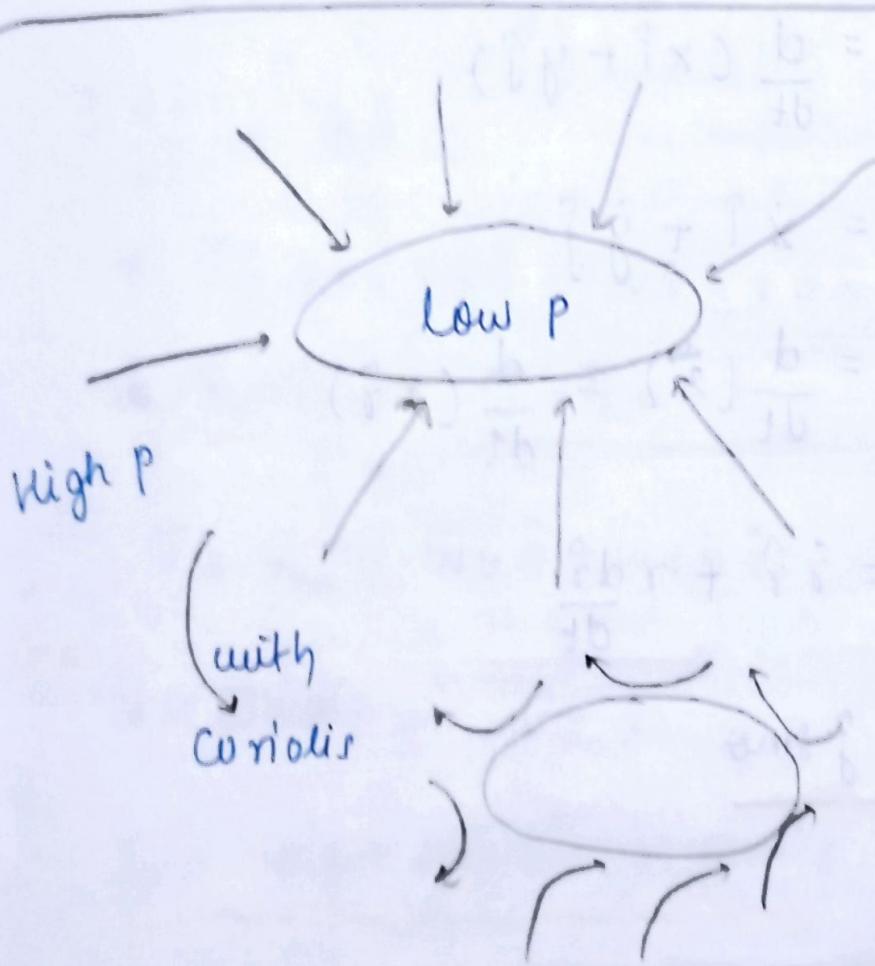
- Assuming flow in  $\vec{v}$ ,
- In Northern Hemisphere - force is along  $\vec{j}$



to the right  
of the flow

- Southern Hemisphere - force is along  $\vec{j}$

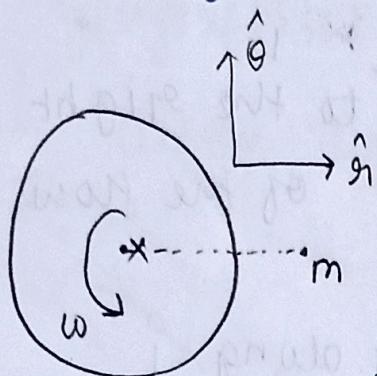
to the  
left of the  
flow



→ Acw in Northern Hemisphere

→ cw in southern hemisphere

→ Deflection of a falling stone dropped from a height  $h$  on the equator  $(r, \theta)$  - fixed to the earth



Apparent force on it is

$$\vec{F} = -mg\hat{j} - 2m(\vec{\omega} \times \vec{v}_{rot})$$

$$-2m\vec{\omega} \times (\vec{\omega} \times \hat{r})$$

Velocity,

$$\vec{v} = \frac{d}{dt}(x\hat{i} + y\hat{j})$$

$$= \dot{x}\hat{i} + \dot{y}\hat{j}$$

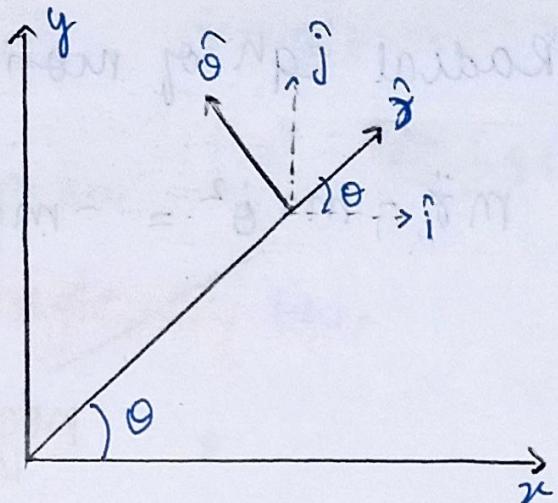
$$= \frac{d}{dt}(\hat{r}) = \frac{d}{dt}(r\hat{r})$$

$$= \dot{r}\hat{r} + r\frac{d\hat{r}}{dt}$$

$$\hat{r} = \underbrace{i\omega_{so} + j\sin\theta}_{\text{rotation of } \omega_{so}}$$

~~if  $\hat{r}$  is constant:~~

$$\hat{v} = \dot{\gamma} \hat{r} + \gamma \frac{d\hat{r}}{dt}$$



$$\hat{r} = \hat{i} \cos\theta + \hat{j} \sin\theta$$

$$\frac{d\hat{r}}{dt} = (-\hat{i} \sin\theta + \hat{j} \cos\theta) \dot{\theta}$$

$$\hat{\theta} = -\hat{i} \sin\theta + \hat{j} \cos\theta$$

$$\therefore \frac{d\hat{r}}{dt} = \hat{\theta} \hat{\theta}$$

$$\Rightarrow \vec{v} = \dot{\gamma} \hat{r} + \gamma \dot{\theta} \hat{\theta} = v \hat{r} + \gamma \dot{\theta} \hat{\theta}$$

$$\underline{\vec{\omega} \times \hat{r} = \hat{\theta}} \quad \text{OR} \quad \underline{\vec{\omega} \times \hat{\theta} = -\hat{r}}$$

$$\text{so } \vec{\omega} \times \vec{v}_{\text{tot}} = \omega v \hat{\theta} - \omega r \dot{\theta} \hat{r}$$

and

$$\vec{\omega} \times (\vec{\omega} \times \vec{r}) = -\omega^2 \vec{r}$$

$$\therefore F_r = -mg + 2m\omega \dot{\theta} \gamma + m\omega^2 \gamma$$

and

$$\underline{F_\theta = -2m\dot{\gamma}\omega}$$

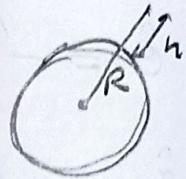
$\therefore$  Radial eqn of motion is

$$m\ddot{r} - mr\dot{\theta}^2 = -mg + 2mr\dot{\theta}\omega + mw^2r$$

neglecting this

$mw^2r$  is the dominant term - neglect  
 $\dot{\theta} \ll \omega$

$$\ddot{r} \approx -g + w^2r$$



Tangential eqn of motion is

$$mr\ddot{\theta} + 2mr\dot{\theta}\dot{\theta} = -2mr\dot{\theta}\omega$$

Now assume that  $r$  changes only slightly from  $R_\oplus + h$  to  $R_\oplus$  ( $R_\oplus + h \approx R_\oplus$ )

$\therefore$  Let  $r \approx R_\oplus$  &  $g \approx \text{const}$

$$\ddot{r} \approx -g + w^2R_\oplus = g' \Rightarrow \ddot{r} = -g't^2$$

$$\text{or } r = r_0 - \frac{1}{2}gt^2$$

Tangential eqn of motion:

$$\dot{r} = -g' t$$

$$mr\ddot{\theta} + 2m\dot{r}\dot{\theta} = -2m\dot{r}\omega \quad \text{neglecting } \dot{r}$$

$$\tau\ddot{\theta} = -2\dot{r}\omega \quad \text{neglecting this}$$

$$= -2(-g't)\omega$$

$$\text{or } \ddot{\theta} = +\frac{2g'wt}{R} \quad \text{or } \dot{\theta} = \frac{g'wt^2}{R_\oplus}$$

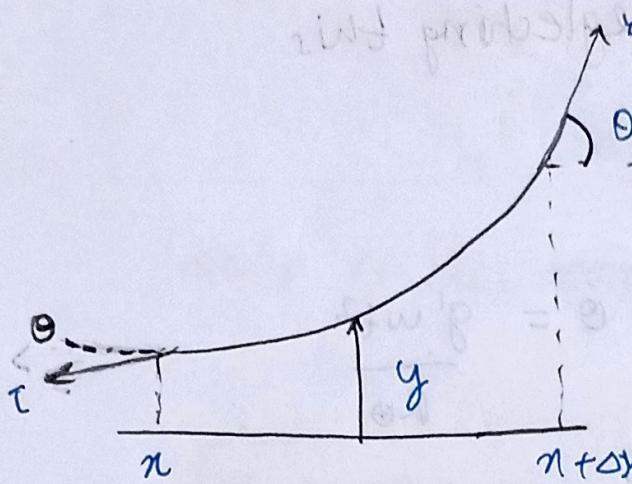
$$\text{or } \theta = \frac{g'wt^3}{3R_\oplus}$$

Deflection,

$$y \approx R_\oplus \theta$$

$$= \frac{g'wt^3}{3}$$

## A) Free vibrations of a stretched string



- String of length  $L$  held fixed at  $x=0$  and  $x=L$ . Uniform linear density " $\mu$ " stretched with tension  $T$ .

$$F_y = T \sin(\theta + \Delta\theta) - T \sin\theta$$

$$F_x = T \cos(\theta + \Delta\theta) - T \cos\theta$$

→ Assume transverse displacement is small,

$$\therefore F_y \approx T \Delta\theta \quad F_x \approx 0$$

$\theta, \theta + \Delta\theta$   
one small angle

- Transverse motion of the string is governed by  $F_y$

$$T \Delta\theta \approx (\mu \omega^2) a_y$$

$$\rightarrow T \Delta\theta = (\mu \alpha n) a_y$$

$$\text{Now } \tan \theta = \frac{\partial y}{\partial n} \Rightarrow \sec^2 \theta = \frac{\partial^2 y}{\partial n^2} \text{ on}$$

• For a small angle  $\sec \theta \approx 1$

$$\text{So, } \Delta \theta \approx \frac{\partial^2 y}{\partial n^2} \text{ on}$$

$$\text{Now Acceleration } a_y = \frac{\partial^2 y}{\partial n^2}$$

$$T \Delta \theta = (\mu \alpha n) a_y = \mu \alpha n \cdot \frac{\partial^2 y}{\partial n^2}$$

OR

$$T \Delta \theta = T \frac{\partial^2 y}{\partial n^2} \text{ on} = \mu \alpha n \frac{\partial^2 y}{\partial t^2}$$

$$v = \sqrt{T/\mu}$$

$$\frac{\partial^2 y}{\partial n^2} = \frac{\mu}{T} \frac{\partial^2 y}{\partial t^2}$$

$$\boxed{\frac{\partial^2 y}{\partial n^2} = \frac{1}{V^2} \frac{\partial^2 y}{\partial t^2}}$$

wave equation

$$\frac{\partial^2 y}{\partial n^2} = \frac{1}{v^2} \frac{\partial^2 y}{\partial t^2} \quad v = \sqrt{T/\mu} = \text{Velocity}$$

→ since each part of the string is moving sinusoidally with time and amplitude of motion is a function of distance 'x' of the point along the string.

let us assume.

$$y(n, t) = f(n) \cos(\omega t)$$

$$\begin{aligned} \therefore \frac{\partial^2 y}{\partial t^2} &= -\omega^2 f(n) \cos(\omega t) \\ \text{&} \boxed{\frac{\partial^2 y}{\partial n^2} = \frac{\partial^2 f}{\partial x^2} \cos(\omega t)} \end{aligned} \quad \Rightarrow \frac{\partial^2 f}{\partial x^2} = -\frac{\omega^2}{v^2} f$$

$$f(n) = A \sin\left(\frac{\omega n}{v}\right)$$

NOW B.C. requires  $y=0$  at  $n=0$  and  
at  $n=L$

$$\Rightarrow \frac{\omega L}{v} = n\pi \rightarrow \boxed{\frac{\omega}{v} = \frac{n\pi}{L}}$$

$$\frac{\omega}{v} = \frac{n\pi}{L}$$

$$f(n) = A \sin\left(\frac{\omega n}{v}\right) = A \sin\left(\frac{n\pi}{L} x\right)$$

$\nu \equiv \frac{\omega}{2\pi} \equiv \underline{\text{no. of cycles per unit time.}}$

$\therefore \nu = \frac{\omega}{2\pi} = \frac{n\nu}{2L} = \frac{n}{2L} \int_{\mu}^T \nu \}$  frequency velocity

$\therefore \frac{2L}{n}$  defines a wavelength

$$\lambda_n \equiv \underline{\frac{2L}{n}}$$

associated with the  $n^{\text{th}}$  node.

→ "normal mode" of vibration

$$\therefore \frac{\omega}{\nu} = \frac{n\pi}{L} = \frac{2\pi}{\lambda_n} = K_n$$
 also called the  
'Wave number'

$\therefore$  shape of the string in node  $n$  is given by  $f_n(n) = A_n \sin\left(\frac{2\pi n}{\lambda_n}\right) = A_n \sin\left(\frac{n\pi x}{L}\right)$

→ The complete description of motion is given by

$$y(n, t) = f(n) \cos(\omega t)$$

$$\text{i.e } y_n(n, t) = A_n \sin\left(\frac{2\pi n}{\lambda_n}\right) \cos(\omega t)$$

$$= A_n \sin(K_n n) \cos(\omega t)$$

$$\left( \text{with } \omega_n = \frac{n\pi}{L} v = \frac{n\pi}{L} \sqrt{\frac{T}{\mu}} = n\omega_1 \right)$$

$$\omega_1 = \frac{\pi}{L} \sqrt{\frac{T}{\mu}}$$

"Fundamental node mode"

so, given the length of the string, its diameter and density, you will know its frequencies of vibrations.

For a prism-dispersion - wave speed varies with wavelength

$$v_g = \text{group velocity} = \frac{d\omega}{dk}$$

energy transport

$$\frac{\omega}{k} = v_p = \text{phase velocity}$$

$$\begin{aligned} \Rightarrow y_n(x, t) &= A_n \sin\left(\frac{2\pi}{\lambda_n} x\right) \cos(\omega_n t) \\ &= A_n \sin(k_n x) \cos(\omega_n t) \end{aligned}$$

This can be rewritten as

$$y_n = \frac{A_n}{2} (\sin(k_n x - \omega_n t) + \sin(k_n x + \omega_n t))$$

→ Since  $\nu_n = \frac{2\pi}{\lambda} = \frac{n\pi}{L}$  &  $\omega_n = \frac{n\pi}{L} \sqrt{\frac{T}{\mu}}$ ,  
we can write:

$$y_n(n, t) = \frac{A_n}{2} \sin\left(\frac{n\pi}{L}(n - \sqrt{\frac{T}{\mu}} t)\right) +$$

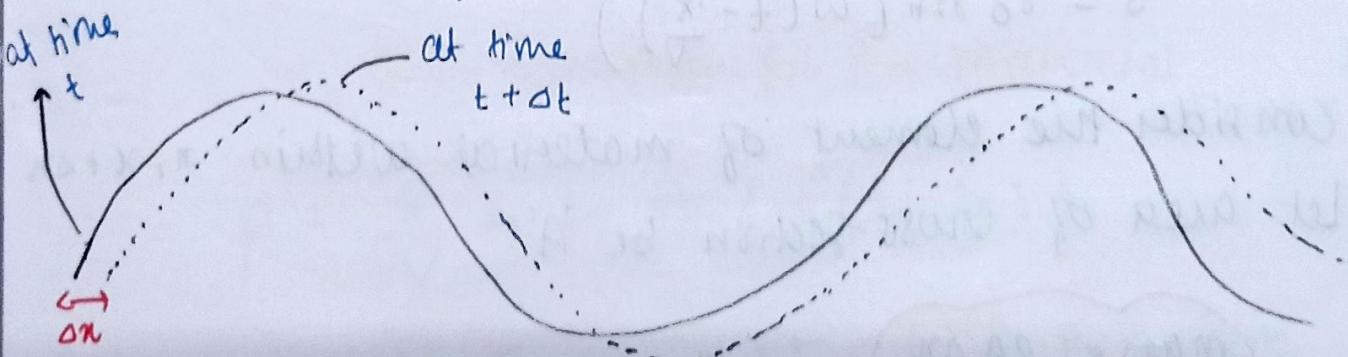
$$\frac{A_n}{2} \sin\left(\frac{n\pi}{L}(n + \sqrt{\frac{T}{\mu}} t)\right)$$

→ This is actually the eqn. of 2 travelling waves going in opposite directions, whose superposition has given us a **STANDING WAVE**.

- Consider the 1<sup>st</sup> part of the wave:

$$y(n, t) = A \sin\left(\frac{2\pi}{\lambda}(n - vt)\right)$$

This is a disturbance - at some instant of time, consider a value of  $y$  and see where we find it after  $(t + \Delta t)$  - say at  $(n + \Delta n)$



$$y(n, t) = y(n + \Delta n, t + \Delta t)$$

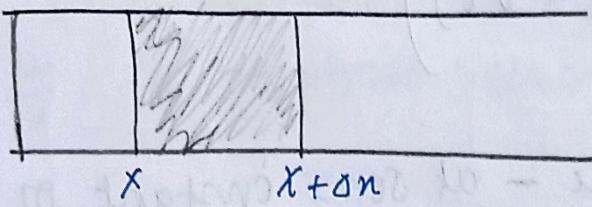
i.e.  $\sin\left(\frac{2\pi}{\lambda}(n - vt)\right) = \sin\left(\frac{2\pi}{\lambda}(n + \Delta n - v(t + \Delta t))\right)$

$$\Rightarrow \Delta n - v\Delta t = 0$$

or  $\frac{\Delta n}{\Delta t} = v$

→ the wave is moving in the tve  $x$  direction with velocity  $v$

## ① Displacement wave, Pressure wave



consider the motion of a wave going in the  $n$ -direction in a fluid with displacement ' $s$ ' in the  $n$ -direction

$$s = s_0 \sin(\omega(t - \frac{x}{v}))$$

→ consider the element of material within  $x, x + \Delta n$   
let area of cross-section be ' $A'$ .

mass =  $\rho A \Delta n$

∴ Increase in volume as wave passes is

$$\boxed{\Delta V = A \Delta s}$$

$$\therefore \Delta V = A S_0 \left( -\frac{w}{V} \right) \cos \left( w(t - \frac{x}{V}) \right) \Delta n$$

i.e there is a volume strain

$$\begin{aligned} \frac{\Delta V}{V} &= \frac{-A S_0 \frac{w}{V} \cos \left( w(t - \frac{x}{V}) \right) \Delta n}{A \Delta n} \\ &= -\frac{S_0}{V} w \cos \left( w(t - \frac{x}{V}) \right) \end{aligned}$$

$$\frac{\Delta V}{V} = -\frac{S_0}{V} w \cos \left( w(t - \frac{x}{V}) \right)$$

The stress corresponding to this

= excess pressure developed in the element under consideration is

$$P = B \left( -\frac{\Delta V}{V} \right)$$

Bulk modulus for the material

$$\therefore P = \frac{B}{V} S_0 w \cos \left( w(t - \frac{x}{V}) \right)$$

$$\text{or } P = p_0 \cos \left( w(t - \frac{x}{V}) \right) \quad \text{with } p_0 = \frac{BS_0w}{V}$$
$$= BS_0 K$$

i.e. the pressure wave differs in phase by  $\pi/2$  from the displacement wave.

Ex:

Sound wave.  $\lambda = 40 \text{ cm}$  travels in air  
 $P_{\text{max}} - P_{\text{min}} = 10^{-3} \text{ N/m}^2$  at some point

What is the amplitude of vibration of particle of the medium?

$$B = 1.4 \times 10^5 \text{ N/m}^2$$

$$\left(\frac{\rho}{V}\right) \frac{100 \omega_0^2}{V} = \frac{V_0}{V}$$

$$\left(\frac{V_0}{V}\right)^2 = 9$$

$$F = m\ddot{x} = -kx - b\dot{x}$$

spring  
force

frictional  
force  
(viscous)

$$\ddot{x} + \frac{k}{m}x + \frac{b}{m}\dot{x} = 0$$

$$\gamma \equiv \frac{b}{m} \quad \omega_0^2 \equiv \frac{k}{m}$$

$$\ddot{x} + \omega_0^2 x + \gamma \dot{x} = 0$$

$$\ddot{y} + \omega_0^2 y + \gamma \dot{y} = 0$$

$$z \equiv x + iy$$

$$\ddot{z} + \omega_0^2 z + \gamma \dot{z} = 0$$

→ if  $z = z_0 e^{\alpha t}$  is a solution then,

$$(\alpha^2 + \alpha \gamma + \omega_0^2) z_0 e^{\alpha t} = 0$$

or

$$\alpha = \left( -\frac{\gamma}{2} \pm \sqrt{\frac{\gamma^2}{4} - \omega_0^2} \right)$$

so there are 2 roots  $\alpha_1, \alpha_2$

so that

$$z = z_1 e^{\alpha_1 t} + z_2 e^{\alpha_2 t}$$

is the solution

$z_1, z_2$  ~~constants~~  
constants

$$\alpha = -\frac{\gamma}{2} \pm \sqrt{\frac{\gamma^2}{4} - \omega_0^2}$$

There are 3 distinct possibilities

- $\gamma^2 \ll 4\omega_0^2$

light damping

$\sqrt{C}$  is complex

$$\alpha = -\frac{\gamma}{2} \pm i \sqrt{\omega_0^2 - \frac{\gamma^2}{4}}$$

$$= -\frac{\gamma}{2} \pm i\omega_1 \quad \text{where } \omega_1 = \sqrt{\omega_0^2 - \frac{\gamma^2}{4}}$$

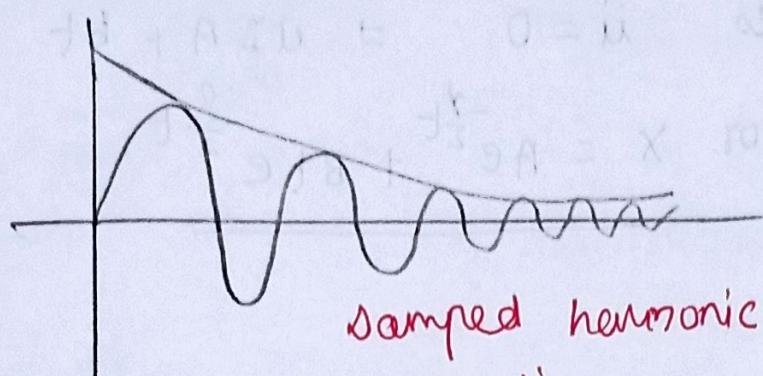
$$z = x + iy = e^{-\frac{\gamma}{2}t} ((x_1 + iy_1)(\cos \omega_1 t + i \sin \omega_1 t))$$

$$+ (x_2 + iy_2)(\cos \omega_1 t - i \sin \omega_1 t))$$

$$x = e^{-\frac{\gamma}{2}t} [B \cos \omega_1 t + C \sin \omega_1 t]$$

(taking only the real terms on R.H.S)

$$= A e^{-\frac{\gamma}{2}t} \cos(\omega_0 t + \phi)$$



$\omega^2 = 4\omega_0^2$

critical damping

so only 1 solution

$$x = A e^{-\frac{\gamma}{2}t}$$

is the incomplete soln

(a 2nd order ODE - 2 constants needed to specify both initial position and initial velocity)

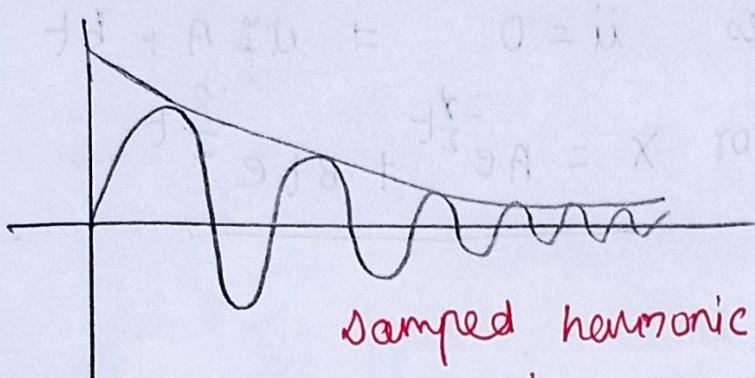
extra info

$$\ddot{y} = ax \rightarrow \text{Rewrite as a 1st order ODE}$$

$$\dot{y} = q$$

$$\dot{q} = ax$$

$$= A e^{-\frac{\gamma}{2}t} \cos(\omega t + \phi)$$



$$\omega^2 = 4\omega_0^2$$

Critical damping

so only 1 solution

$$x = A e^{-\frac{\gamma}{2}t}$$

is the incomplete sol<sup>n</sup>

(a 2<sup>nd</sup> order ODE - 2 constants needed to specify both initial push and initial velocity)

extra info

$\ddot{y} = ax \rightarrow$  Rewrite as 2 1<sup>st</sup> order ODE

$$\dot{y} = q$$

$$\dot{q} = ax$$

$$\text{Let } x = u(t) e^{-\frac{\gamma}{2}t}$$

using this gives  $\ddot{x} = 0 \Rightarrow u = A + Bt$

$$\text{or } x = A e^{-\frac{\gamma}{2}t} + Bt e^{-\frac{\gamma}{2}t}$$

•  $\gamma^2 > 4w_0^2$  *sine wave begins to damp*

### Heavy damping

$$\alpha = -\frac{\gamma}{2} \pm \frac{\gamma}{2} \sqrt{1 - \frac{4w_0^2}{\gamma^2}}$$

Both roots are negative

$$\rightarrow z = z_1 e^{-|\alpha_1|t} + z_2 e^{-|\alpha_2|t}$$

look for the real parts

$$x = A e^{-|\alpha_1|t} + B e^{-|\alpha_2|t}$$

No oscillatory behaviour here.

exponentially decaying

Overshadowed case

swin

→ Energy at some time  $t$  —  $E(t)$

$$E(t) = E(0) + \text{work done due to friction}$$

$$W_{\text{friction}} = \int_{x_0}^{x(t)} f dx = \int_0^t f v dt = \int_0^t -b v^2 dt < 0$$

•  $E(t)$  decreases with time

for light damping:

$$x(t) = A e^{-\frac{\gamma}{2}t} \cos(\omega_1 t + \phi)$$

$$\text{or } v = A e^{-\frac{\gamma}{2}t} (\omega_1 \sin(\omega_1 t + \phi) + \frac{\gamma}{2} \cos(\omega_1 t + \phi))$$

for very light damping:

$$\frac{\gamma}{\omega_1} \ll 1$$

$$\Rightarrow v \approx \omega_1 A e^{-\frac{\gamma}{2}t} \sin(\omega_1 t + \phi)$$

$$= KE = \frac{1}{2} m v^2 \approx \frac{1}{2} m \omega_1^2 A^2 e^{-\gamma t} \sin^2(\omega_1 t + \phi)$$

$$= PE = \frac{1}{2} K x^2 = \frac{1}{2} K A^2 e^{-\gamma t} \cos^2(\omega_1 t + \phi)$$

Now,

$$\omega_0^2 = \omega_1^2 + \frac{\gamma^2}{4}$$

For very light damping

$$\underline{\omega_0^2 \approx \omega_1^2}$$

$$\therefore E(t) = KE + PE \quad \left[ \because \frac{K}{m} \approx \omega_0^2 \approx \omega_1^2 \right]$$

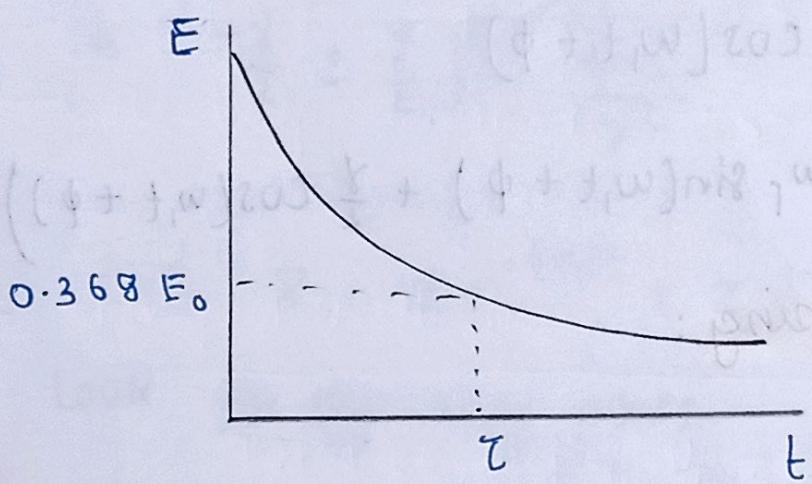
$$= \frac{1}{2} KA^2 e^{-\gamma t}$$

At  $t = 0$ ,  $E(0) = E_0 = \frac{1}{2} KA^2$

$$E(t) = E_0 e^{-\gamma t}$$

— decay characterized by time  $\tau$

require for  $E$  to drop by  $e^{-1}$



$$E(\tau) = E_0 e^{-\gamma \tau} = e^{-1} E_0$$

$$= 0.368 E_0$$

$\tau = \frac{1}{\gamma} = \frac{m}{b} = \text{"Damping Time"}$

or Time Constant

or Characteristic time for the system

In the limit of very light damping

$$\gamma \rightarrow 0$$

$$T \rightarrow \infty$$

and E is effectively constant

→ Degree of damping is also quantified by the quality factor Q.

Q = energy stored in oscillator

{ energy dissipated per radian

i.e. energy lost during the time the system oscillates through 1 radian

$$E(t) = E_0 e^{-\gamma t}$$

$$\frac{dE}{dt} = -\gamma E_0 e^{-\gamma t} = -\gamma E(t)$$

∴ Energy dissipated  $\Delta E \approx \left| \frac{dE}{dt} \right| \Delta t = \gamma E_0 \Delta t$

1 radian of oscillation

$$\Rightarrow \Delta t = \frac{1}{\omega_1}$$

$$\therefore \text{Energy lost is } \frac{\gamma E}{\omega_1} \Rightarrow \varphi = \frac{E}{\gamma E / \omega_1}$$

$$= \frac{\omega_1}{2} \approx \frac{\omega_0}{2}$$

- Undamped oscillation :  $\gamma = 0$ ,  $\varphi \rightarrow \infty$
- Lightly damped :  $\gamma < \omega_1$ ,  $\varphi \gg 1$
- Heavily damped :  $\gamma \gg \omega_1$ ,  $\varphi$  — very small

## # Damped oscillations in an LCR circuit :

$$\text{Total Energy} = U_E + U_S$$

$$= \frac{\varphi^2}{2C} + \frac{1}{2} L I^2$$

First consider,

an LCR circuit with current  $I(t)$

- pdf across inductor is  $L\dot{I}$
- pdf across capacitor  $V = Q/C$

$$LI + \frac{Q}{C} = 0 \quad \text{or} \quad L\ddot{I} + \frac{I}{C} = 0$$

$$\text{i.e } \ddot{I} + \omega^2 I = 0, \quad \omega = \underbrace{\frac{1}{\sqrt{LC}}}_{\text{Ansatz}}$$

This is a simple harmonic motion eqn  
without damping.

$$\rightarrow I(t) = I_0 \cos(\omega t - \phi) \quad \text{Ansatz}$$

Now  ~~$V = -LI$  across inductor~~

$$= L I_0 \omega \sin(\omega t - \phi)$$

$$= \int \frac{L}{C} I_0 \sin(\omega t - \phi) dt$$

$$= \int \frac{L}{C} I_0 \cos\left(\omega t - \phi - \frac{\pi}{2}\right) dt$$

i.e b/w V & I - there is a phase shift of  $\frac{\pi}{2}$

i.e V is max when I is minimum

and vice-a-versa

$$\frac{1}{\omega} = \frac{1}{\sqrt{LC}} = \phi$$

## LCR Circuit:

$$L\ddot{I} + RI + \frac{\Phi}{C} = 0 \quad \omega_0 = \frac{1}{\sqrt{LC}}$$

$$\text{or } \underline{\ddot{I} + \gamma I + \omega_0^2 I = 0} \quad \gamma = R/L$$

(damped)

harmonic  
oscillation

$b \rightarrow R$   
 $L \rightarrow m$

$$\frac{dE}{dt} = -RI^2$$

$$E = \frac{1}{2}LI^2 + \frac{1}{2}\frac{\Phi^2}{C}$$

Recall  $\phi$  factor for damped harmonic oscillation

$$\text{was } \frac{\omega_0}{\gamma}$$

$$\underline{\ddot{x} + \gamma \dot{x} + \omega_0^2 x = 0}$$

$$\omega_0^2 = \omega^2 + \frac{\gamma^2}{4}$$

$$\text{For light damping } \phi = \frac{\omega_0}{\gamma} = \frac{1}{\sqrt{LC}} \frac{1}{R} = \frac{1}{R} \sqrt{\frac{L}{C}}$$

$$\phi = \frac{1}{R} \sqrt{\frac{L}{C}}$$