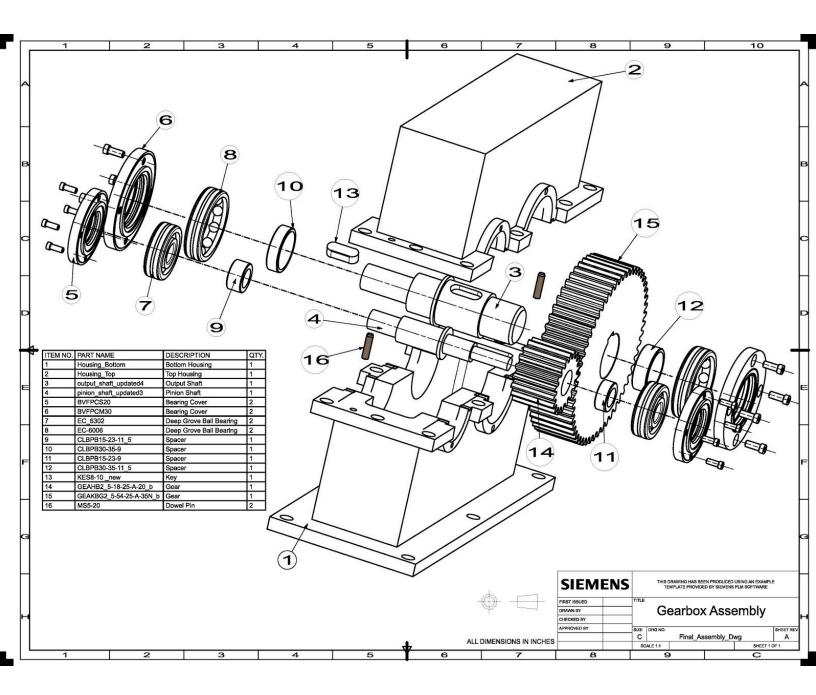
FINAL REPORT Group 20 Gilberto Guadiana, Bryan Horn, Kevin Kim, Abdallah Khawaldh

Project Description

We designed a transmission gearbox. Its function is to make speed and torque conversions from the input rotating power source connected to the input shaft and output it via the output shaft. The shafts are connected to a gear, two ball bearings, and two spacers. The output shaft has a key connecting the shaft to its gear. The bearings have bearing covers with oil seals. The housing is made up of two components and are connected via dowel pins.

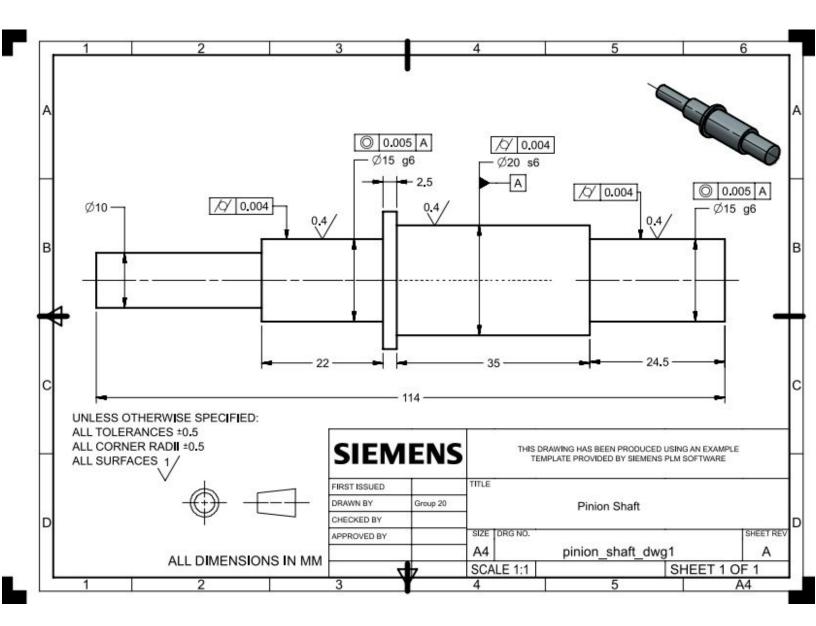
Exploded View with BOM



Pinion Shaft Data

	ITEM	DESCRIPTION SYMBOL/VALUE	RATIONALE/JUSTIFICATION	
		Pinion Fit	A permanent fit to the shaft without possibility of disassembly	
	1	H7/s6	using a medium drive interference fit.	
		Bearing Fit	The NSK-Bearing fitting recommendations were used. Thi	
	2	N7/g6	minimizes axial displacement.	
		General Tolerance	These are values that likely would not interfere with the	
	3	+/- 0.5	function of the pinion shaft.	
TOLERANCES		General Corner Radii	These are values that likely would not interfere with the	
LIMITS & FITS	4	+/- 0.5	function of the pinion shaft.	
		Primary Datum	Pinion axis selected since it is most functionally important. This	
	1	А	axis is used as reference for the bearing surfaces.	
		Pinion and Bearing Surface Cylindricity	Provides a higher form accuracy than the limits of size.	
	2	/\$\text{0.004}	Assures a better fit between the shaft components and the shaft.	
		Bearing Surface Concentricity		
GD & T	3	0.005 A	Provides a higher form accuracy than the limits of size. Allows the bearing surface axis to line up with the pinion surface axis.	
		Pinion and Bearing Surface Roughness	The specification reflects the need for a ground surface for	
	1	0.4/	proper function of the pinion shaft. We followed the 10x ru thumb.	
		General Surface Roughness		
SURFACE ROUGHNESS	2	1/	The specification allows for a ground surface and is less restrictive than the more functionally important surfaces.	

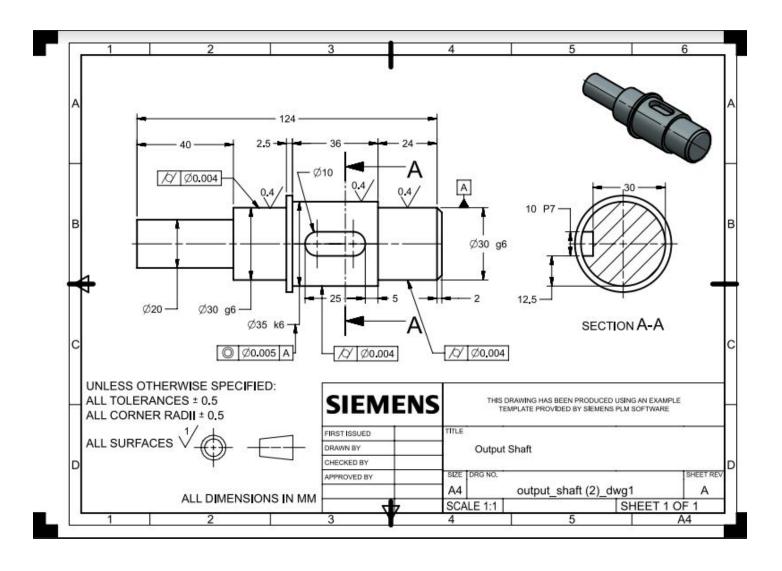
Pinion Shaft Drawing



Gear Shaft Data

	ITEM	DESCRIPTION SYMBOL/VALUE	RATIONALE/JUSTIFICATION
		Gear Fit	Provides for accurate location, a compromise between
	1	H7/k6	clearance and interference fits
		Bearing Fit	
	2	H7/g6	The NSK-Bearing fit recommendations were used
		Key Fit	This fit between key and keyway provides rigidity and
	3	P7/h6	alignment without excessive pressure requirements
TOLERANCE		General Tolerance	
S LIMITS &	4	± 0.5	This general tolerance wouldn't interfere with the key functions of the gear shaft
FITS		General Corner Radii	This general tolerance wouldn't interfere with the key functions
	5	± 0.5	of the gear shaft
		Primary Datum	The center axis of the gear shaft is very important in regards
	1	А	to the rest of the geometry of the shaft
		Gear Surface Concentricity	
	2	◎ Ø0.005 A	Important for the gear to be not oscillating and not wear quickly, so concentricity takes care of that
		Gear Surface Cylindricity	Further and use the growth mations are not and articles at the
	3	/\$\\ Ø0.004	Further ensures the gear functions properly and safely on the shaft
		Bearing Surface Cylindricity	
GD & T	4	/☆ Ø0.004	Further accuracy for the bearings to function properly with the gear
		Gear and Bearing Surfaces	
	1	0.4/	Ensures the proper ground surface for the gear and bearing to function properly
		General Surface Roughness	
SURFACE ROUGHNESS	2	1/	Ensures a proper ground surface for the other surfaces

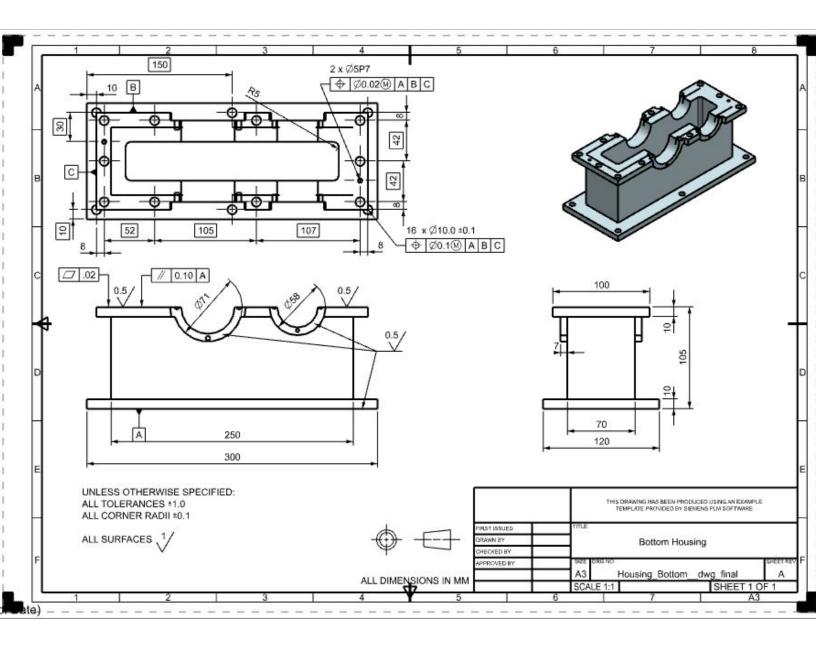
Gear Shaft Drawing



Bottom Housing Data

	ITEM	DESCRIPTION SYMBOL/VALUE	RATIONALE/JUSTIFICATION
TOLERANCES LIMITS & FITS	1	Locating Pin Fit	Locational interference fit is needed. Shaft basis is used, since the locating pin is provided.
		P7/h6	
	2	General Tolerance	This general tolerance won't restrict manufacturing processes or the function of the housing
		± 1.0	
	3	General Corner Radii	This general tolerance won't restrict manufacturing processes of
		± 0.1	the function of the housing
	1	Primary Datum	Bottom of the housing selected. The bottom can easily be put flat
		A	against a datum surface to make three-point contact.
	2	Secondary Datum	One side of the flange is selected. Important for defining the
		В	locations of holes
	3	Tertiary Datum	Other side of the flange is selected. Important for defining the
GD & T		С	locations of holes
	4	Hole Position	Controls position of hole centers relative to datums. Position
		16 x Ø10.0 ±0.1	tolerance is selected so that the holes line up reasonably close the holes on the top housing.
	5	Mating Pin Hole Position	Position tolerance must be smaller than for the general holes,
		2 x Ø5P7 ⊕ Ø0.02M A B C	since these holes are responsible for locating the top and botto housing.
	6	Mating Surface Flatness	Needed so the two mating surfaces can be closely pressed
		∠7 .02	against each other with minimal separation.
	7	Mating Surface Parallelism	
		// 0.10 A	Needed so the mating surface is oriented at the correct angle
	1	Mating Surface	Allows for a ground surface for proper function of the mating
SURFACE ROUGHNESS		0.5	components. We followed the 10x rule of thumb.
	2	Bearing Bores and Bearing Cover Mounting Surfaces	Allows for a ground surface for proper function of the Bearing Bores. We followed the 10x rule of thumb.
		0.5	

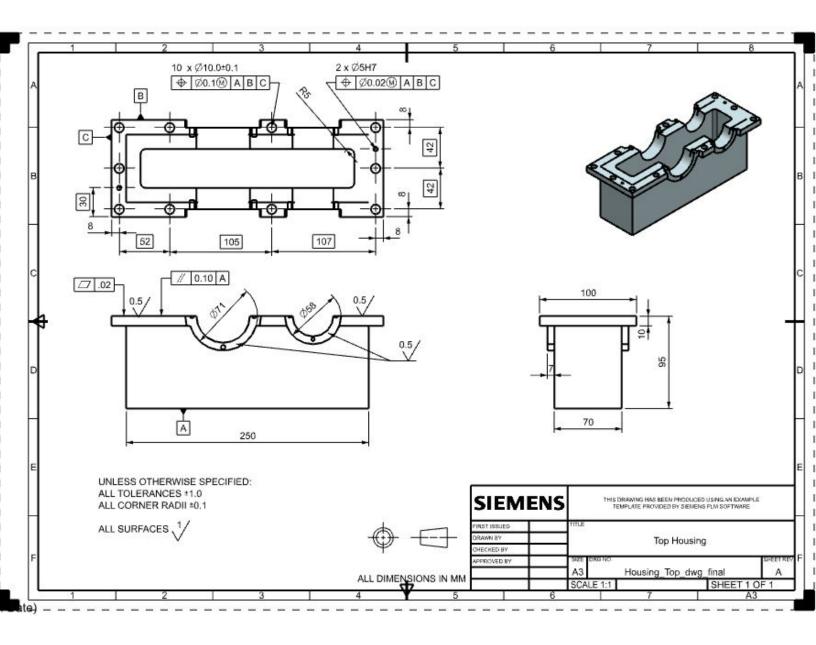
Bottom Housing Drawing



Top Housing Data

	ITEM	DESCRIPTION SYMBOL/VALUE	RATIONALE/JUSTIFICATION
TOLERANCES LIMITS & FITS	1	Locating Pin Fit	Locational clearance fit is needed. Shaft basis is used, since the locating pin is provided.
		H7/h6	
	2	General Tolerance	This general tolerance won't restrict manufacturing processes or the function of the housing
		± 1.0	
	3	General Corner Radii	This general tolerance won't restrict manufacturing
		± 0.1	processes or the function of the housing
	1	Primary Datum	Bottom of the housing selected. The bottom can easily be
		Α	put flat against a datum surface to make three-point contact.
	2	Secondary Datum	One side of the flange is selected. Important for defining the
		В	locations of holes
	3	Tertiary Datum	Other side of the flange is selected. Important for defining
		С	the locations of holes
		Hole Position	Controls position of hole centers relative to datums. Position
GD & T	4	10 x ∅10.0±0.1	tolerance is selected so that the holes line up reasonably
0D Q 1		◆ Ø0.1M A B C	close to the hole on the top housing.
	5	Mating Pin Hole Position	Position tolerance must be smaller than for the general
		2 x ∅5H7	holes, since these holes are responsible for locating the top
		◆ Ø0.02M A B C	and bottom housing.
	6	Mating Surface Flatness	Needed so the two mating surfaces can be closely pressed
		∠ √	against each other with minimal separation.
		Mating Surface Parallelism	Needed so the mating surface is oriented at the correct
	7	// 0.10 A	angle.
SURFACE ROUGHNESS	1	Mating Surface	Allows for a ground surface for proper function of the mating
		0.5	components. We followed the 10x rule of thumb.
	2	Bearing Bores and Bearing Cover Mounting Surfaces	Allows for a ground surface for proper function of the Bearing Bores. We followed the 10x rule of thumb.
		0.5	

Top Housing Drawing



Housing Assembly Data

	ITEM	DESCRIPTION SYMBOL/VALUE	RATIONALE/JUSTIFICATION
TOLERANCES LIMITS & FITS	1	Bearings Fit	The NSK-Bearing fitting recommendations were used. This fit minimizes axial displacement.
		N7/g6	
	2	General Tolerance	This general tolerance won't restrict manufacturing processes or the function of the housing
		± 1.0	
	3	General Corner Radii	This general tolerance won't restrict manufacturing
		± 0.1	processes or the function of the housing
		Primary Datum	Bottom of the housing selected. The bottom can easily be
	1	А	put flat against a datum surface to make three-point contact.
	2	Secondary Datum	Bearing hole axis selected. This axis is important for the
		В	geometry of the assembly.
		Small Bearing Hole Position	Controls position of hole center relative to datums. A
	4	Ø42N7 ⊕ Ø0.024∰ A B	diameter for tolerance zone is also specified.
	_	Large Bearing Hole Position	
	5	Ø55N7 ⊕ Ø0.024∭ A B	Controls position of hole center relative to datums. A diameter for tolerance zone is also specified.
GD & T	6	Large Bearing Cover Hole Position Tolerance	Positions of the holes must be controlled so that all four
		4 x Ø 5.0 ±0.1 ⊕ Ø0.1M A B	holes line up with the four holes on the large bearing cover.
	7	Small Bearing Cover Hole Position Tolerance	Positions of the holes must be controlled so that all four
		4 x Ø 4.0 ±0.1 ⊕ Ø0.1M A B	holes line up with the four holes on the small bearing cover
	8	perpendicularity	
		⊥ 0.1 A	Needed to ensure correct angle and orientation.
	9	Bearing Holes Cylindricity	
		2X /2/ 0.1	Provides a higher form accuracy than is given by the limits of size to ensure that the shafts and bearings can fit inside it.

SURFACE ROUGHNESS	1	General Surface Roughness	The specification allows for a ground surface and is less restrictive than the more functionally important surfaces.
		1	

Housing Assembly Drawing

