

# Experimental Validation of Reference Spreading for Robotic Manipulation of Unmodeled Objects

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## I. INTRODUCTION

Automation has historically played a crucial role in the logistics industry. Our current way of living depends on autonomous systems for global transportation and warehousing. The growing labor shortage and increasing demand for online retail motivate further developments in the logistics sector [1].

A logistical aspect where machines struggle to fully compete with humans is object manipulation. Practical examples of this include order picking or depalletizing. While robots are strong and consistent when manipulating objects, humans are versatile and swift. Robots are held back from faster performance because they must often slow down prior to making contact; establishing contact at a high velocity – an event referred to as an impact – could cause damage to the robot or its environment. On the contrary, humans intrinsically exploit impacts in the form of grabbing, bouncing and hitting.

The field of impact-aware control aims to better equip robots for making contact at high velocities. These impacts are paired with large contact forces that could damage the system. Previous work describes control using the maximum allowable impact velocity that complies with safety constraints such as limits for the contact force [2, 3]. This was combined with a compliant cover for the robot that reduces contact forces at impact, facilitating higher feasible impact velocities. Rather than using a soft cover, compliancy may also be achieved by designing a robot with low inertia and high backdrivability as was done in [4].

In addition to the large contact forces, a subject of interest is the velocity jump at the time of impact. Time misalignments between velocity jumps in the reference and in the actual system cause the velocity tracking error to peak [5], as is shown in Figure xxx. This error peak results in undesired control effort and should therefore be avoided. In [6], the robot’s velocities are projected into an impact-invariant subspace based on the expected point of impact. As a result, impact-driven peaks in the velocity tracking error are reduced significantly. It is not always possible to describe a point of impact, however. Often times, impacts occur between surfaces rather than just points. Furthermore, corners of the surface may impact at diverging intervals in uncertain order during what is called near-simultaneous impacts, shown in Figure xxx.

The impact-aware control scheme called Reference Spreading [7] addresses error peaking caused by misaligned impacts. It operates on the basis of a tracking error that switches once an impact is detected. This concept is best explained at the hand of Figure xxx. The reference is split at the nominal impact time into an ante- and post impact reference. These references are then extended. Initially, the tracking error is based on the extended ante-impact reference, but this is switched to the post-impact reference once an impact is detected. Evidently, this can reduce the error peaking.

Reference spreading can also handle simultaneous impacts. [8] (explanation)

By addressing the peaking error, reference spreading facilitates faster object manipulation, making it interesting to industry if its effectivity can be proven in practice. Reference spreading for object manipulation has already been validated in simulations [8, 9]. Experimental validations have been limited to interaction with a fixed environment, however [10, 11]. The goal of this work is therefore to **provide a real-world implementation of reference spreading for practical object manipulation tasks**. To translate the results from simulation to reality, the following contributions are made:

### 1. Motion planning for impacts without object models:

Generating a reference with velocity jumps that is coherent with the system’s dynamics is challenging. One approach maps the ante-impact velocity to the post-impact velocity based on conservation of momentum [ref for impact map]. This requires a model of the environment, which is feasible in simulations with simplified dynamics, but challenging in reality. Impact-driven velocity jumps could instead be inferred experimentally. In previous studies [12], the control gains are reduced to zero upon detection of the impact while inferring an impact map, so that the velocity jump would not result in excessive motor torques. A different model-free motion planning strategy is proposed, which not only produces velocity-reference jumps that are coherent with the system dynamics, but also leverages human intuition to generate fluid motions before and after the impact. This is achieved by introducing a human in the loop by means of teleoperation.

**2. Impact detection:** The reference spreading scheme should switch between ante-, intermediate-, and post-impact references at the appropriate time. This requires an impact detection algorithm. Approaches in literature look either at position data [13] or external force estimations [11, 14, 15] for signs that could be caused by impacts. We show that

these signs are necessary, but not sufficient conditions for an impact – only looking at position or contact force can result in false positives. To limit false detection of impacts, a novel impact detector that looks at both force and position data is proposed and evaluated.

### 3. Custom end effector:

### 4. Intermediate impact phase controller:

## II. SYSTEM OVERVIEW

Considering the goal of evaluating reference spreading in a practical usecase, this work will focus on a dual-arm robotic setup. Having two arms increases the maximum payload. Furthermore, some object manipulation tasks, such as grabbing, require contact from multiple sides. The impact tasks that are considered in this work are stamping, swiping, grabbing, and tilting.

The Franka Emika robot [16] is used for the setup as it is affordable and, more importantly, capable of torque control which is critical for impact-aware manipulation. The robot uses harmonic drives that inherently have poor backdriveability, however, torque control is still possible thanks to the torque sensors.

The system is limited in certain aspects to make it more representative of affordable setups in industry; in contrast to other object-manipulation works, we do not employ object pose estimation, environment models, or exteroceptive force/torque sensors.

### A. Robot dynamics

The robot dynamics can be modeled as

$$M(q)\ddot{q} + h(q, \dot{q}) = \tau_{cmd} + \sum_{i=1}^n J_i^T F_{c,i} \quad (1)$$

with inertia matrix  $M$ , joint accelerations  $\ddot{q}$ , centrifugal, coriolis and gravity terms  $h$ , and commanded torque  $\tau_{cmd}$ . The robot has  $n$  contacts, and for each contact  $i$  there is an external contact wrench  $F_{c,i}$ . The wrench's contribution on joint level is related through contact jacobian  $J_i$  for which it holds that  $[\omega \ \dot{p}]_i^T = J_i \dot{q}$ , with  $\omega$  the angular velocity of the contact body and  $\dot{p}$  the cartesian velocity of the contact point.

For control purposes, it is assumed that contact forces only act on the end effector body. These forces are modeled as if they act at the same control point  $p$ , which is chosen as the intersection of robot link 5 and 7. Furthermore, omitting dependency on  $q$  and  $\dot{q}$  for brevity results in

$$M\ddot{q} + h = \tau_{cmd} + J^T F_c. \quad (2)$$

### B. Base controller

For articulated robot arms, it is convenient to control the pose of the end effector, rather than controlling the joint angles. This can be accomplished with an impedance controller.

The desired impedance behaviour is

$$\Lambda \begin{bmatrix} \dot{\omega} - \dot{\omega}_t \\ \ddot{p} - \ddot{p}_t \end{bmatrix} + D \begin{bmatrix} \omega - \omega_t \\ \dot{p} - \dot{p}_t \end{bmatrix} + K \begin{bmatrix} e_{rot} \\ p - p_t \end{bmatrix} = F_c - F_{c,t} : \quad (3)$$

a mass spring damper with task-space inertia, damping and stiffness matrices  $\Lambda$ ,  $D$  and  $K$ , and rotation tracking error  $e_{rot}$  as defined in Appendix xxx. Subscript  $t$  denotes a target.

Stiffness  $K$  is typically chosen as a diagonal matrix. The damping matrix is determined following  $D = 2(\Lambda K)^{\frac{1}{2}}$  which guarantees stable behavior when  $K$  and  $\Lambda$  are symmetric. Furthermore, For the inertia matrix, two options were considered. Choosing a diagonal matrix  $\Lambda$  decouples the accelerations w.r.t. to  $e$ , resulting in better tracking in free motion. In this work however, the task-space inertia is set to match the joint-space inertia following  $\Lambda = JM^{-1}J$ , decoupling the contact force w.r.t.  $e$  for better performance during contact. Further motivation for this decision is provided in Appendix xxx.

Based on (x), the desired task-space accelerations are given by

$$\begin{bmatrix} \dot{\omega} \\ \ddot{p} \end{bmatrix} = \begin{bmatrix} \dot{\omega}_t \\ \ddot{p}_t \end{bmatrix} + \Lambda^{-1} \left( D \begin{bmatrix} \omega - \omega_t \\ \dot{p} - \dot{p}_t \end{bmatrix} + K \begin{bmatrix} e_{rot} \\ p - p_t \end{bmatrix} + F_{c,t} \right). \quad (4)$$

Note the exclusion of  $F_c$  as the exerted contact wrench is not modeled and therefore unknown. Quadratic Programming is used to find  $\tau_{cmd}$  so that the weighted squared error between the desired and actual accelerations is minimized while accounting for the system's dynamics and safety constraints such as maximum joint torques and velocities. The described impedance task only tracks 6 DoF's however, leaving one redundant DoF of the Franka Emika robot. To resolve this redundancy, one more task which describes the desired acceleration of the first robot joint is added. This so-called posture task with stiffness  $k_p$  is given by

$$\ddot{q}_1 = 2\sqrt{k_p}\dot{q}_1 + k_p(q_1 - q_{1,t}). \quad (5)$$

An overview of the used control parameters is given in Table xxx.

Parameter	Value
$q_{1,t}$	0 rad
$k_1$	50 Nm/rad
$k_2$	800 N/m
$k_p$	50 Nm/rad
$K$	$\begin{bmatrix} k_1 I & 0 \\ 0 & k_2 I \end{bmatrix}$

### C. Custom end effector

## III. TELEOPERATION

### IV. REFERENCE SPREADING CONTROLLER

Reference spreading reduces control effort by cleverly redefining the tracking target. These target definitions differ between the ante, intermediate, and post-impact phase based on separate ante- and post impact references. Impact detection is used to switch between the impact phases at the appropriate time. This section describes the three key components of reference spreading: impact detection, reference formulation, and tracking error definitions.

### A. Impact detection

### B. Reference formulation

(subscript  $r$  indicates a reference, superscripts  $a$  and  $p$  stand for ante-impact and post-impact phase respectively)

### C. Target definition

Table IV-C shows an overview of the target definitions during the three impact phases. These targets are used for the impedance controller. During the ante- and post impact phase, the target is equal to the respective ante- or post impact reference. For the intermediate phase, multiple options are considered.

Intermediate phase option 0 is equivalent to the post-impact mode, meaning that the controller effectively jumps from ante- to post impact mode directly.

In [8], it is mentioned that following the post-impact reference does not make sense during the intermediate mode where contact is not yet completed. Instead, the ante-impact reference should be targeted until contact completion, with exception of the reference velocity which is causing the error peak. Setting the target velocity to the current velocity, i.e.,  $\dot{p}_t = \dot{p}$ , causes the velocity tracking error to be zero. This aligns with intermediate option 1.

Option 2 was inspired by [11] where the velocity target was set to zero following from the observation that damping is beneficial for reducing oscillations.

Upon the transition from intermediate to the post-impact phase, intermediate option 1 and 2 will experience a jump in the targets  $p_t$  and  $F_t$  as the respective ante and post impact references are not guaranteed to coincide. To address this issue, we propose mixing of the ante and post impact reference during the intermediate phase in option 3. Mixing value  $\gamma$  equals zero at the start of the intermediate mode, and increments linearly up to 1 at the end of the intermediate phase. Furthermore, opposed to option 2 where the damping target is a velocity of zero, option 3 employs damping with respect to a target velocity equal to the post impact reference.

TABLE I  
IMPEDANCE TARGET DEFINITION

Definition of the targets  $p_t$ ,  $\dot{p}_t$ , and  $F_t$  during the different impact phases. Multiple options for the intermediate phase are considered.

	Ante	Post	Intermediate			
			0	1	2	3
$p_t$	$p_r^a$	$p_r^p$	$p_r^p$	$p_r^a$	$p_r^a$	$\gamma p_r^a + (1 - \gamma) p_r^p$
$\dot{p}_t$	$\dot{p}_r^a$	$\dot{p}_r^p$	$\dot{p}_r^p$	$\dot{p}$	0	$\gamma \dot{p}_r^a + (1 - \gamma) \dot{p}_r^p$
$F_t$	$F_r^a$	$F_r^p$	$F_r^p$	$F_r^a$	$F_r^a$	$\gamma F_r^a + (1 - \gamma) F_r^p$

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