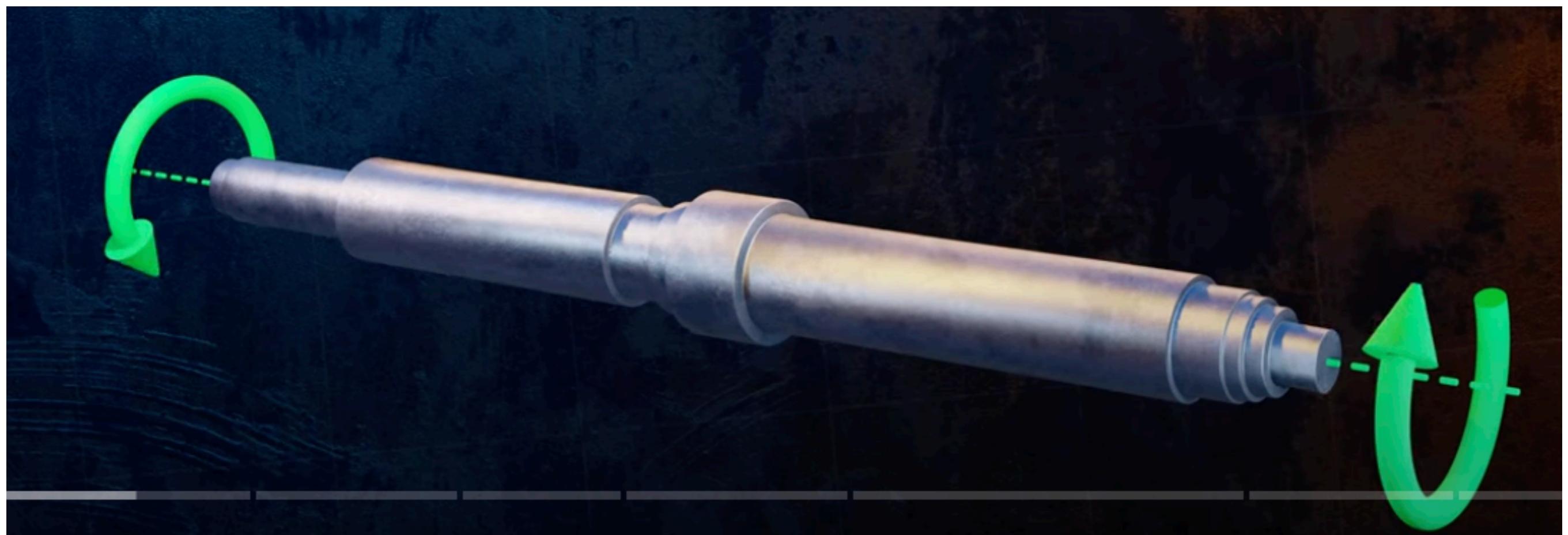
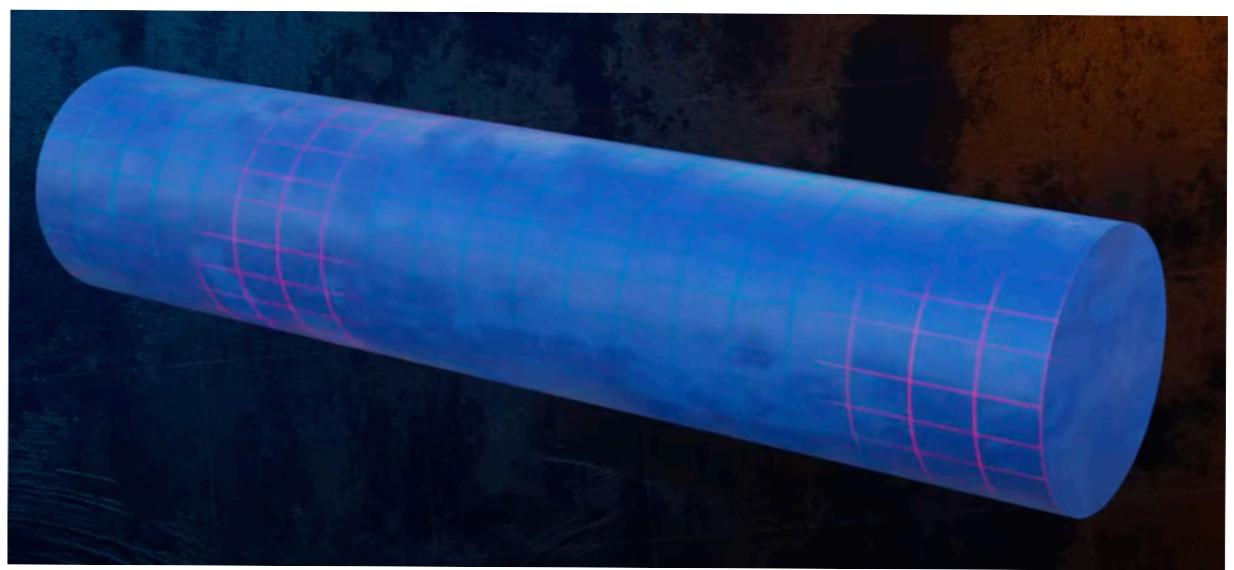


DEFORMAZIONI  
TORSIONALI

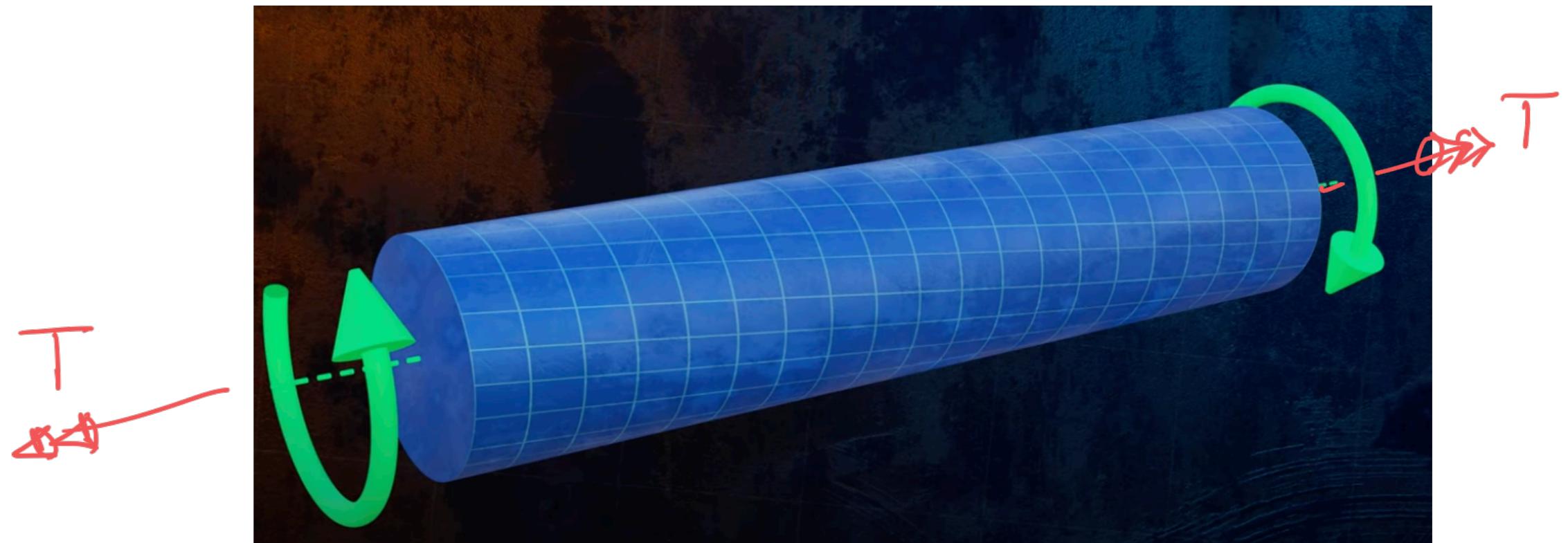
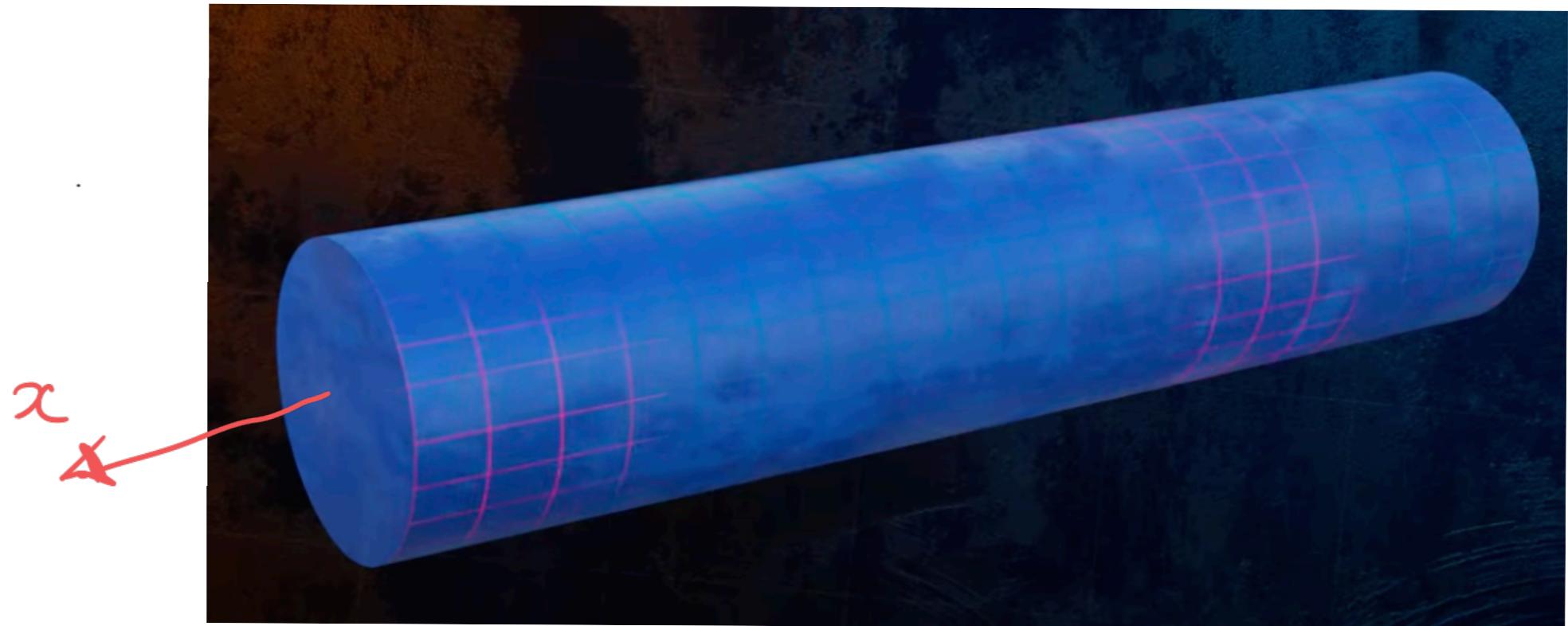
TORSIONE DI UN ALBERGO MOTORE

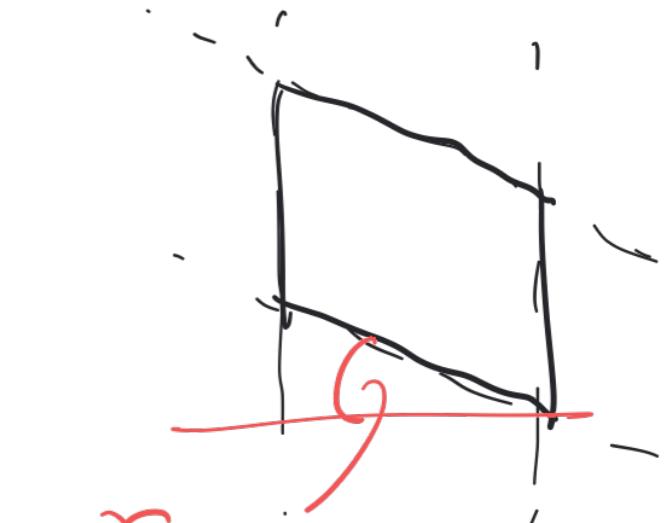
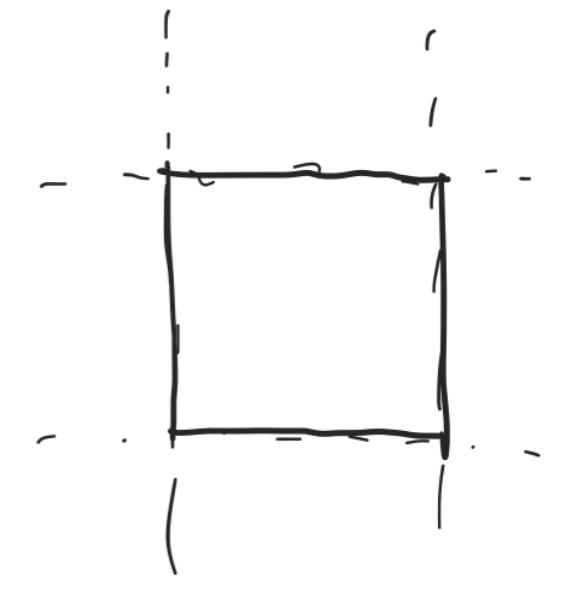
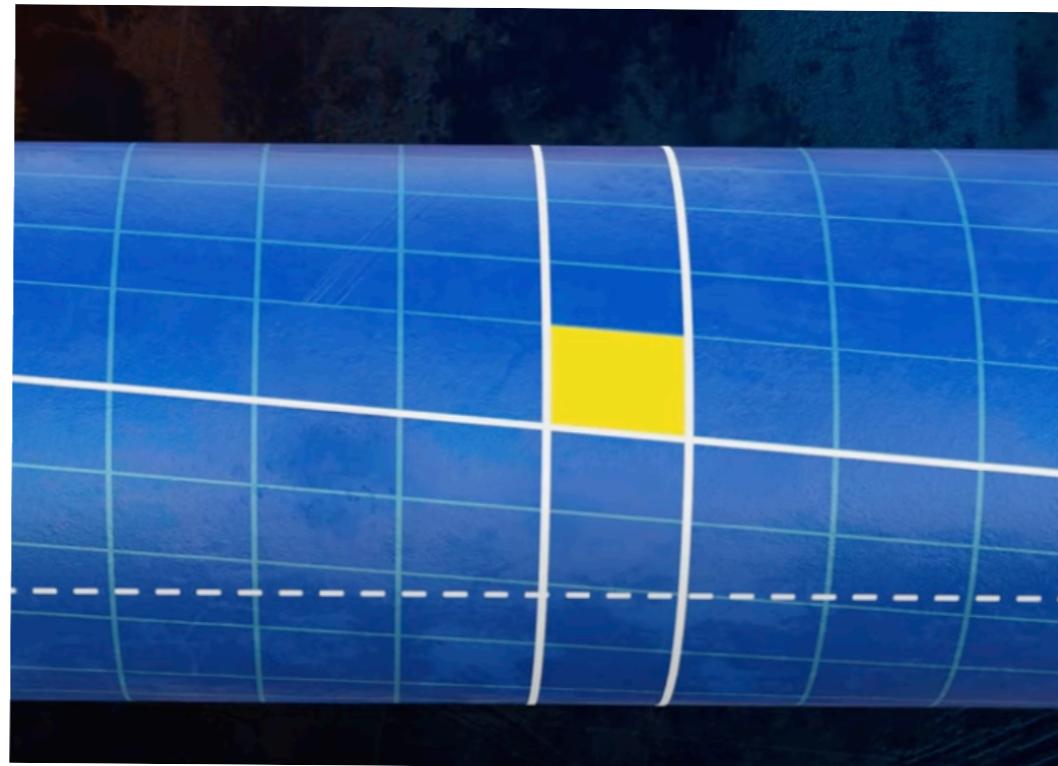
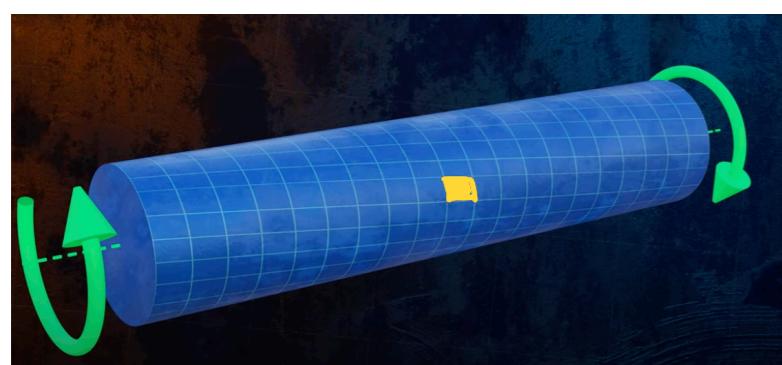
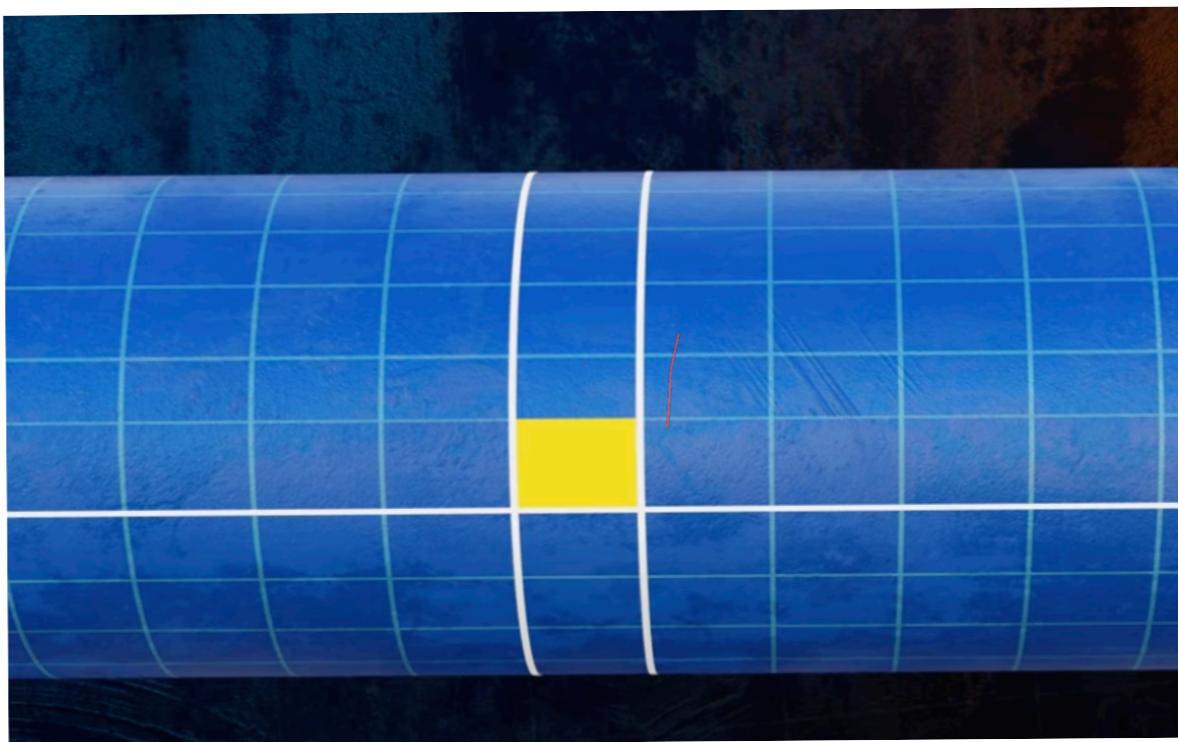
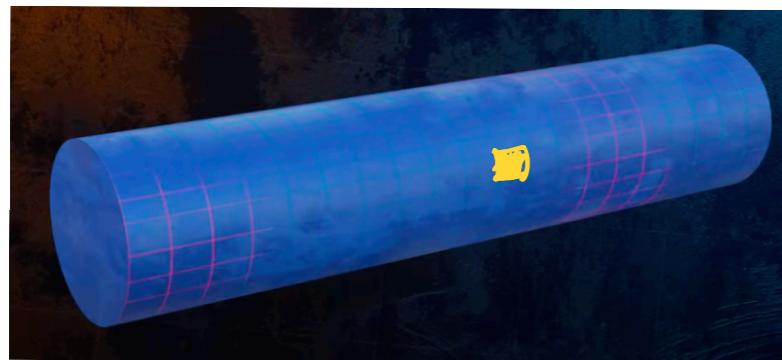




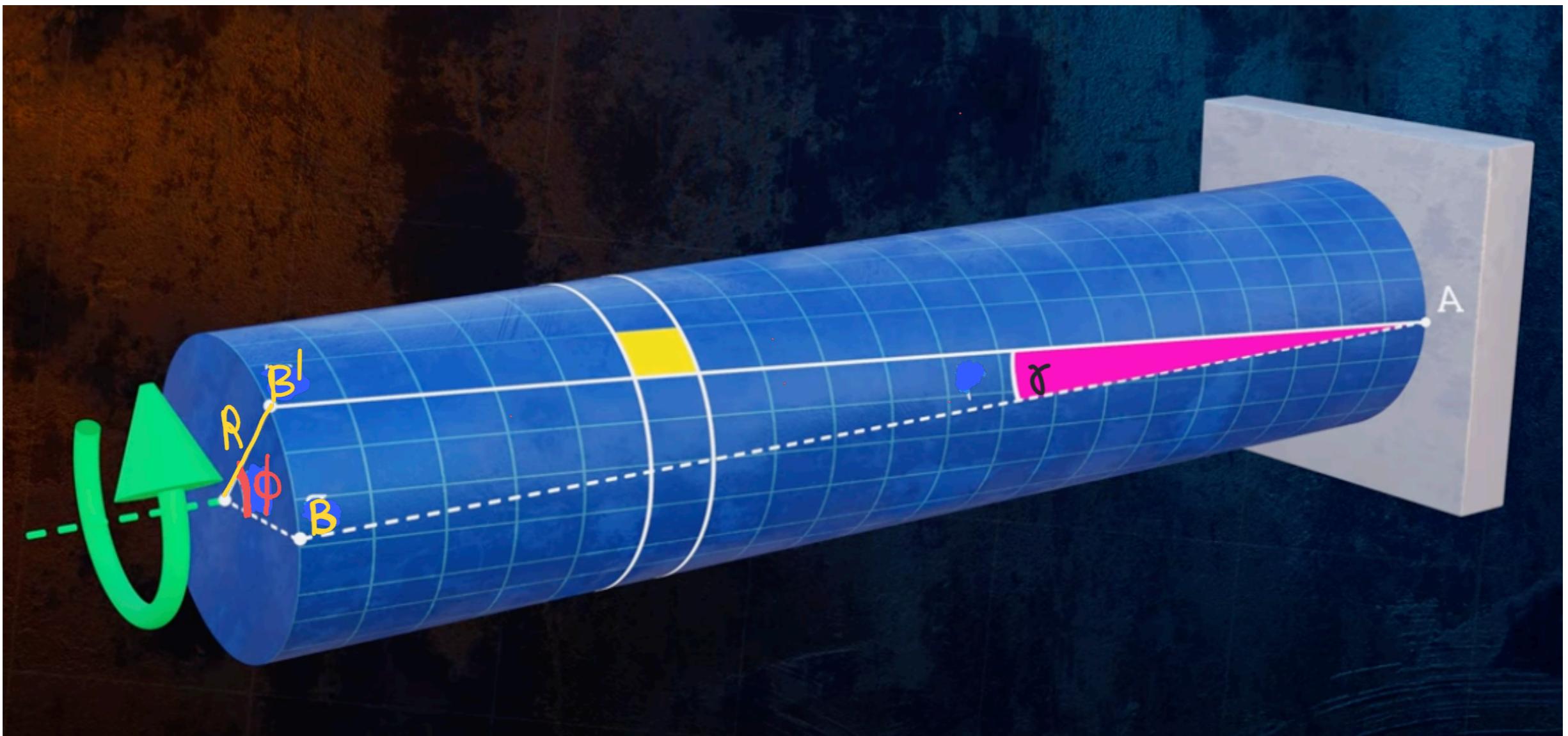


# PARTE 1: TORSIONE UNIFORME





$\delta$   
scorrimento



$$|BB'| \approx AB \tan \gamma \approx |AB|\gamma \quad |BB'| = R\phi \quad \Rightarrow \quad \gamma = \frac{R\phi}{|AB|}$$

Lo scorrimento sulla superficie del cilindro è  $\boxed{\gamma = \frac{R}{L} \phi}$ , dove

$R$  è il raggio della sezione,  
 $L$  è la lunghezza della trave

$\phi$  è l'angolo di torsione.

Punto generico

$$\gamma = \rho \frac{\phi}{L}$$

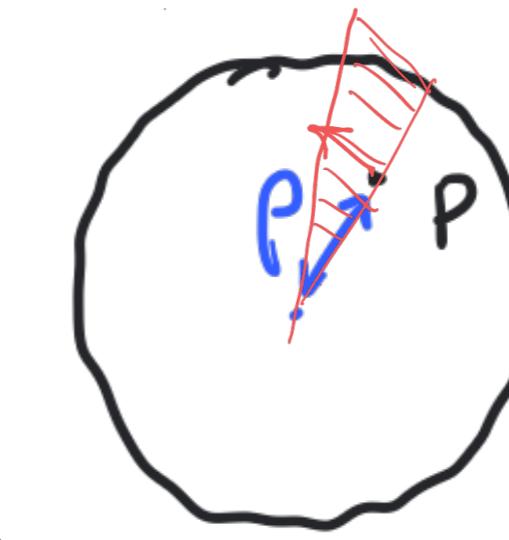
Tensione tangenziale

$$\tau = G \gamma$$

$$\tau = G_p \frac{\phi}{L}$$

$$T = 2\pi \int_0^R \tau(\rho) \rho d\rho$$

$$\delta = \frac{\phi}{L}$$



angolo umano del  
torrone

$$T = G J \delta$$

$$\tau = \rho \frac{T}{J}$$

$$T = GJ \frac{\phi}{L}$$

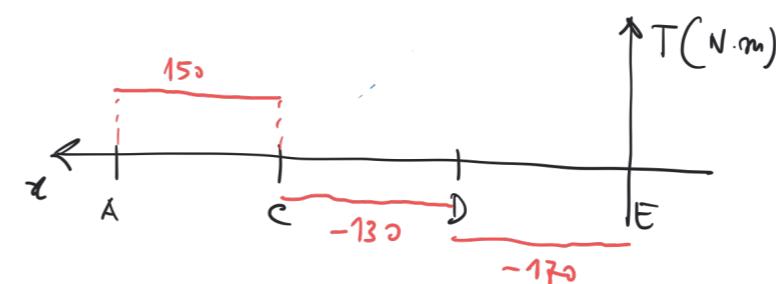
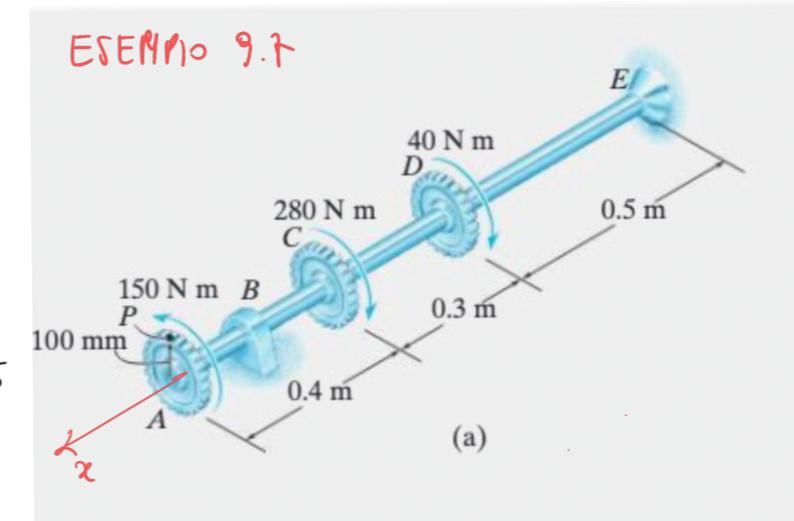
$$\tau(\rho) = \frac{\rho T}{J}$$

Calcolare lo spostamento  
del dente P.

$$G = 80 \text{ GPa}$$

d = 14 mm diametro albero.

Diagramma di T



$$T = GJ \frac{\phi}{L} \quad \leftarrow$$

$$\varphi(\rho) = \frac{\rho T}{J}$$

Calcolare lo spostamento del rullo P.

$$G = 80 \text{ GPa}$$

d = 14 mm diametro albero.

$$\phi_{ED} = \frac{T_{ED} L_{ED}}{G J}$$

$$\phi_{DC} = //$$

$$\phi_{CA} = //$$

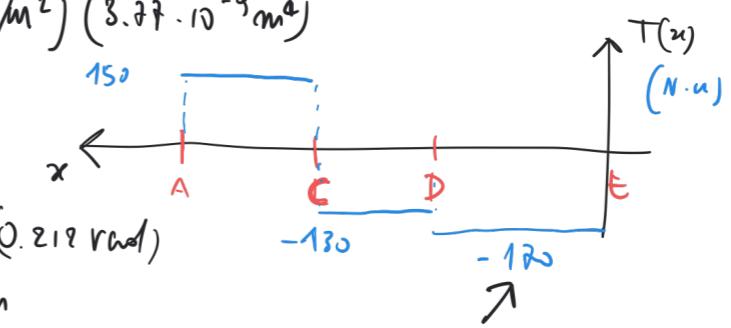
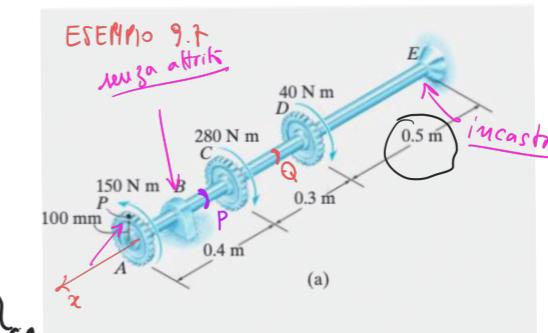
$$J = \frac{\pi}{2} R^4 = \frac{\pi}{2} (7 \text{ mm})^4 = \\ = 3.77 \cdot 10^{-9} \text{ m}^4$$

$$\phi_A = \phi_{EA} = \phi_{ED} + \phi_{DC} + \phi_{CA} \\ = \frac{T_{ED} L_{ED} + T_{DC} L_{DC} + T_{CA} L_{CA}}{G J}$$

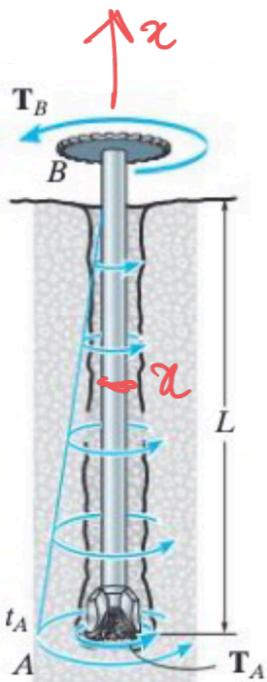
$$= \frac{(-120 \text{ Nm})(0.5 \text{ m}) + (-130 \text{ Nm})(0.3 \text{ m}) + (150 \text{ Nm})(0.6 \text{ m})}{(80 \cdot 10^9 \text{ N/m}^2)(3.77 \cdot 10^{-9} \text{ m}^4)}$$

$$= -0.212 \text{ rad.}$$

$$S_P = \phi_A = (100 \text{ mm})(0.212 \text{ rad}) \\ = 21.2 \text{ mm}$$



Parte 2: torsione non uniforme



Problema 9.12

$$t(x) = t_A \left(1 - \frac{x}{L}\right)$$

$$T(x) = T_A + \int_0^x t_A \left(1 - \frac{x}{L}\right) dx = T_A + t_A \left(x - \frac{1}{2} \frac{x^2}{L}\right)$$

$\phi(x)$  rotazione relativa tra sezione  $x$  e seg-A

$$\phi(0) = 0 \quad \phi_{AB} = \phi(L) = \int_0^L \frac{d\phi}{dx} dx = \int_0^L \frac{1}{GJ} \left(T_A + t_A x \left(1 - \frac{1}{2} \frac{x}{L}\right)\right) dx$$

$$= \frac{1}{GJ} \left(T_A L + t_A \frac{L^2}{3}\right)$$

~~$$\phi = \frac{T}{GJ} L$$~~

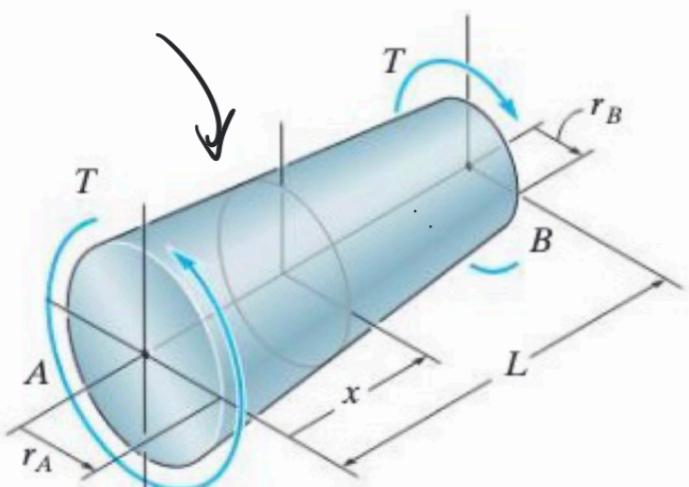
$$d\phi = \frac{T}{GJ} dx$$

$$\frac{d\phi}{dx} = \frac{T}{GJ}$$

$$\phi = \int_0^L \frac{T}{GJ} dx = \frac{T}{G} \int_0^L \frac{1}{J(x)} dx$$

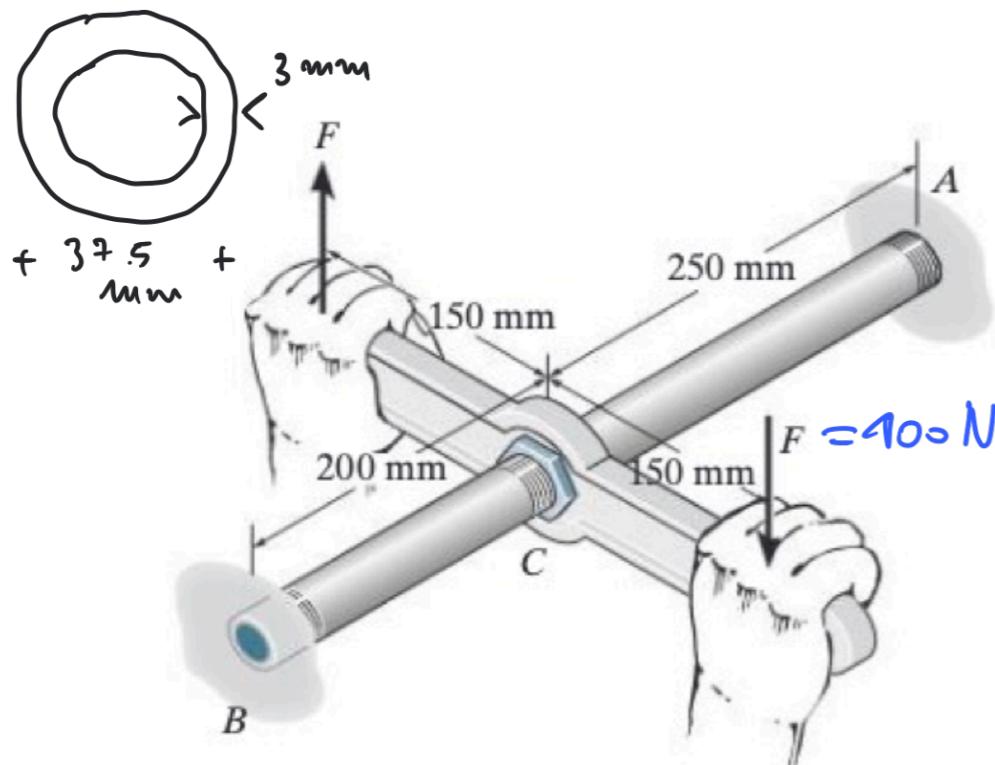
La torsione può essere non uniforme anche a causa di caratteristiche della sezione variabili.

$$J = J(x)$$



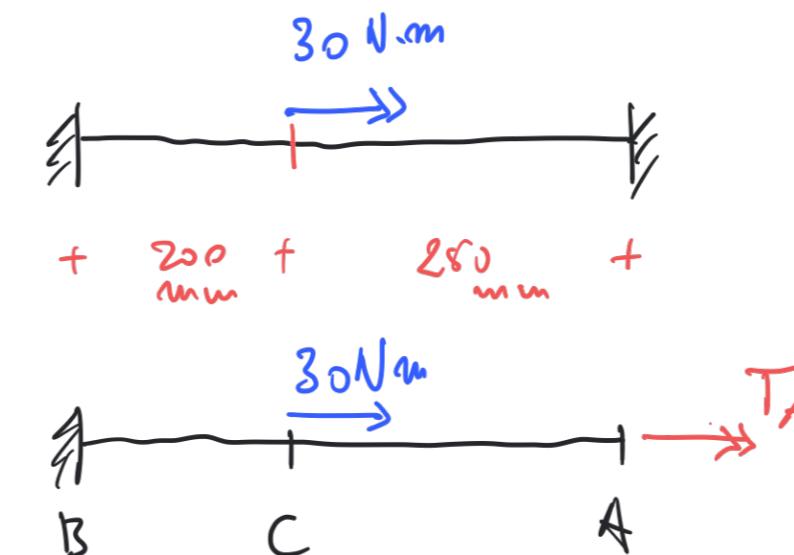
Problema 9.11

Torsione și traxi staticalemente indeterminate.



Problema 9.28

Determinare la maximă  
tensiune tangențială



$$T_{BC} = T_A + 30 \text{ N}\cdot\text{m}$$

$$T_{CA} = T_A$$

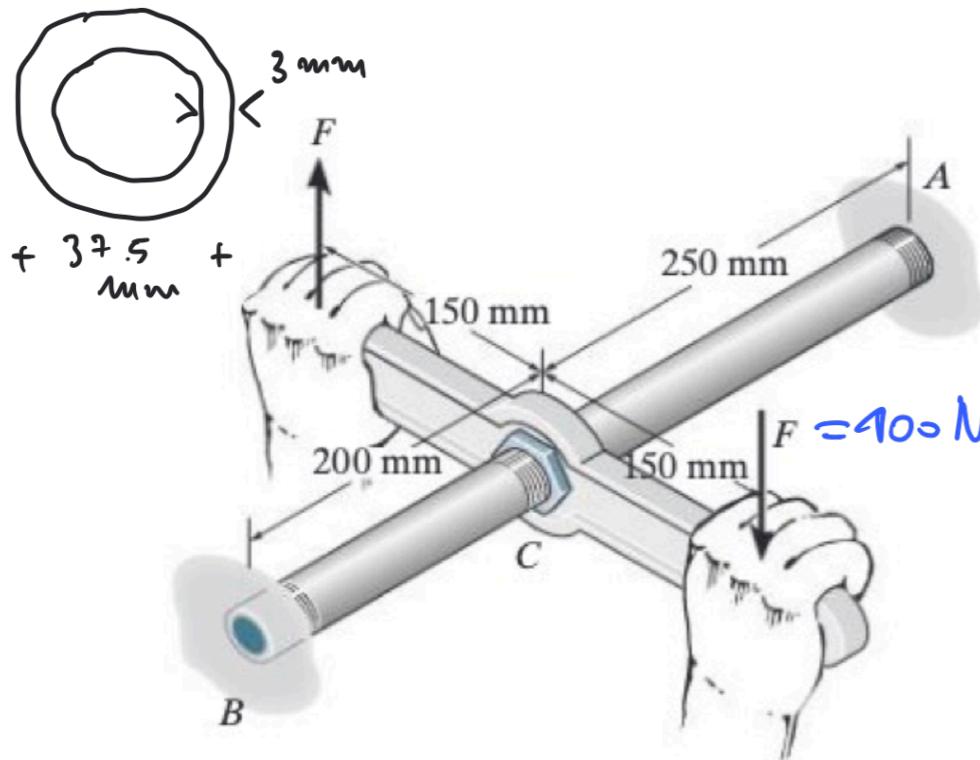
$$d\phi = \frac{T}{GJ} dx$$

$$\theta = \phi = \int_0^L \frac{T}{GJ} dx = \frac{1}{GJ} (T_{BC} L_{BC} + T_{CA} L_{CA})$$

$$\Rightarrow T_{BC} L_{BC} + T_{CA} L_{CA} = 0$$

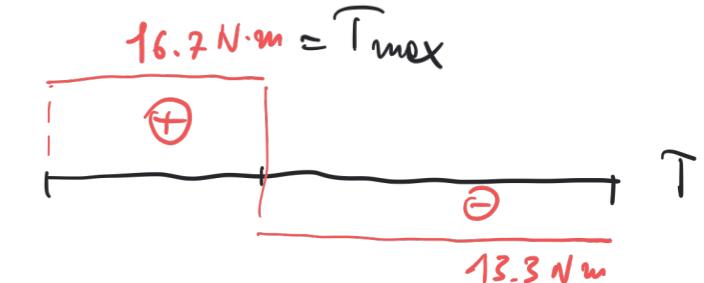
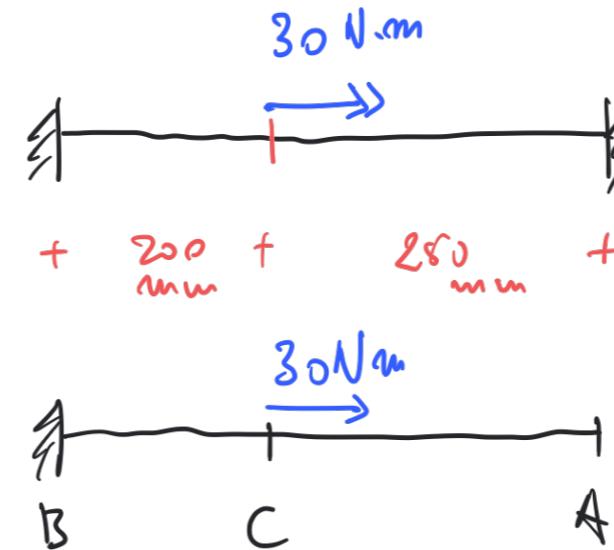
$$T_A \cdot 450 \text{ mm} + 30 \text{ N}\cdot\text{m} \cdot 200 \text{ mm} = 0 \Rightarrow T_A = -\frac{200}{450} 30 \text{ N}\cdot\text{m} = -13.3 \text{ N}\cdot\text{m}$$

Torsione obișnuite și teoreme indeterminate.



Problema 9.28

Determinare la maximă tensiune tangențială



$$J = \frac{\pi}{2} (R_e^4 - R_i^4)$$

$$\sigma_{max} = R_e \frac{T_{max}}{J}$$

$$T_{BC} = T_A + 30 \text{ N}\cdot\text{m}$$

$$d\phi = \frac{T}{GJ} dx$$

$$T_{CA} = T_A$$

$$\theta = \phi = \int_0^L \frac{T}{GJ} dx = \frac{1}{GJ} (T_{BC} L_{BC} + T_{CA} L_{CA})$$

$$\Rightarrow T_{BC} L_{BC} + T_{CA} L_{CA} = 0$$

$$T_A \cdot 450 \text{ mm} + 30 \text{ N}\cdot\text{m} \cdot 200 \text{ mm} \Rightarrow T_A = -\frac{200}{450} 30 \text{ N}\cdot\text{m} = -13.3 \text{ N}\cdot\text{m}$$