# Nuclear Reactor Thermal-Hydraulics

NUGN520 - Homework

By

GUILLAUME L'HER



Department of Nuclear Engineering COLORADO SCHOOL OF MINES

Homework submitted for the Nuclear Reactor Thermal-Hydraulics class at the Colorado School of Mines.

**SPRING 2017** 

# TABLE OF CONTENTS

													]	Page
1	Rea	ctor H	eat Genera	tion										1
	1.1	[6-20]	- Variable h	eat gene	ration	 	 	 	 	 				. 1
		1.1.1	Problem .			 	 	 	 	 				. 1
		1.1.2	Solution .			 	 	 	 	 				. 1
	1.2	7-1 - C	uter square	corner .		 	 	 	 	 				. 2
		1.2.1	Problem .			 	 	 	 	 				. 2
		1.2.2	Solution .			 	 	 	 	 				. 2
	1.3	[G1] -	Fine mesh c	ruciform	l	 	 	 	 	 				. 3
		1.3.1	Problem .			 	 	 	 	 				. 3
		1.3.2	Solution .			 	 	 	 	 				. 3
Bi	bliog	raphy												9

CHAPTER

# REACTOR HEAT GENERATION

everal exercises from the book written by M. M. El Wakil [1] are tackled in this homework. The problems in this section relate to the sixth and seventh chapter of the book, covering the subject of heat conduction in reactor elements.

# 1.1 [6-20] - Variable heat generation

#### 1.1.1 Problem

A cylindrical fuel elment, 0.06 ft in diameter using  $UO_2$  as fuel material, generates  $5 \times 10^6$  Btu.h<sup>-1</sup>.ft<sup>-3</sup> at its surface when that surface is at 2000 °F. The thermal conductivity may be considered constant at 1.25 Btu.h<sup>-1</sup>.ft<sup>-1</sup>.°F<sup>-1</sup>. How much hotter would the center of this element operate if the heat generation increases by 1 percent for each 1000 °F temperature rise above surface conditions instead of being uniform at the above value?

#### 1.1.2 Solution

In the uniform cylinder, we know Equation 1.1.

(1.1) 
$$T_m = T_s + \frac{q'''R^2}{4k_f}$$

Consequently,  $T_m = 2900 \, {}^{\circ}F$ .

Now, if the heat generation depends on the temperature following the given relation, then we can write Equation 1.2.

(1.2) 
$$q''' = q_s''' + 10^{-5} q_s''' \Delta T$$

For each 1000 °F temperature increase, q" will gain 1 percent. Hence, we can now solve the Poisson equation for  $T' = q_s''' + 10^{-5} q_s''' \Delta T$ . The solution is given by Equation 1.3, following the book.

(1.3) 
$$T' = AJ_0\left(\sqrt{\frac{10^{-5}q_s'''}{k_f}}r\right) + BY_0\left(\sqrt{\frac{10^{-5}q_s'''}{k_f}}r\right)$$

We know two boundary conditions. First and foremost, the temperature needs to be finite at the center of the cylinder, r=0. Thus, considering that  $Y_0(0) \to -\infty$ , B=0. Next, we know that at the boundary,  $q'''=5\times 10^6~Btu.h^{-1}.ft^{-3}$ . Consequently, since we have T'=q''', we have Equation 1.4 at R=0.03~ft.

(1.4) 
$$AJ_0\left(\sqrt{\frac{10^{-5}q_s'''}{k_f}}R\right) = 5 \times 10^6$$

Solving for A, we obtain  $A = 5.045 \times 10^6$ .

So, we now have Equation 1.5.

(1.5) 
$$5.045 \times 10^{6} * J_{0} \left( \sqrt{\frac{10^{-5} q_{s}^{\prime\prime\prime}}{k_{f}}} r \right) = q_{s}^{\prime\prime\prime} + 10^{-5} q_{s}^{\prime\prime\prime} \Delta T$$

At r = 0, since  $J_0(0) = 1$ , we can write Equation 1.6.

(1.6) 
$$5.045 \times 10^6 = 5 \times 10^6 + 10^{-5} * 5 \times 10^6 * \Delta T$$

Finally, we obtain  $\Delta T = T_m - T_s = 906.2 \,^{\circ}F$ , hence  $T_m = 2906.2 \,^{\circ}F$ . If we take into account the given change in heat generation, we obtain a maximum temperature 6.2  $^{\circ}F$  hotter.

### 1.2 7-1 - Outer square corner

#### 1.2.1 Problem

Derive the finite difference equation of temperature for a nodal point at the outer square corner of a body with heat generation, having thermal conductivity k, and cooled by a fluid at  $T_f$  with a uniform heat transfer coefficient h.

#### 1.2.2 Solution

We can write the heat balance for a node n on a corner,  $q_g + q_{xo} + q_{xi} + q_{yo} + q_{yi}$ . In our case,  $q_{xo} = h(\Delta x)(T_f - T_n)$  and  $q_{yo} = h(\Delta y)(T_f - T_n)$ .

So, we can write Equation 1.7, considering  $\Delta x = \Delta y$ .

(1.7) 
$$\frac{1}{2}q'''(\Delta x)^2 + k[(T_{n-\Delta x} - T_n) + (T_{n-\Delta y} - T_n)] + 2h\Delta x(T_f - T_n) = 0$$

So, we can divide by k and reorganize.

(1.8) 
$$\frac{1}{2k}q'''(\Delta x)^2 + [(T_{n-\Delta x} - T_n) + (T_{n-\Delta y} - T_n)] + 2\frac{h\Delta x}{k}(T_f - T_n) = 0$$

We can recognize the Biot number,  $Bi_{\Delta x} = \frac{h\Delta x}{k}$ , and  $\Delta T_g = \frac{q'''(\Delta x)^2}{k}$ .

(1.9) 
$$\frac{\Delta T_g}{2} + T_{n-\Delta x} + T_{n-\Delta y} - 2T_n + 2Bi_{\Delta x}T_f - 2Bi_{\Delta x}T_n = 0$$

And so:

(1.10) 
$$T_n = \frac{\Delta T_g}{4Bi_{\Delta x} + 4} + \frac{T_{n-\Delta x} + T_{n-\Delta y} + 2Bi_{\Delta x}T_f}{2Bi_{\Delta x} + 2}$$

## 1.3 [G1] - Fine mesh cruciform

## 1.3.1 Problem

Calculate the temperature in a cruciform with a mesh  $\Delta x = \Delta y = 0.1$ .

#### 1.3.2 Solution

We can define the origin, taken at the center of the cruciform, to be T(0,0). By symmetry, we know that T(i,j) = T(-i,j) and T(i,j) = T(i,-j). Consequently, the solution will be complete when T(i,j),  $i \in [0,11]$ ,  $j \in [0,1]$  are computed. By symmetry, we also know that T(0,1) = T(1,0), so we can get rid of T(0,1) in our computation.

We can approximate the temperature at these discrete points by using Equation 1.11, knowing that  $\Delta x = \Delta y$ .

$$T_n = \frac{T_{n+\Delta x} + T_{n-\Delta x} + T_{n+\Delta y} + T_{n-\Delta y}}{4} + \frac{\Delta T_g}{2}$$

Where, in the case of uniform heat generation:

$$\Delta T_g = \frac{(\Delta x)^2 q'''}{2k}$$

In our case, we consider  $q'''=1\times 10^7$  and k=19.84. Consequently,  $\Delta T_g=17.5~^{\circ}F$ . We also consider a surface temperature of 600  $^{\circ}F$ .

Our goal is to solve the matrix Equation 1.12 for x.

$$(1.12) A.x = b$$

We can calculate a few values in order to construct the matrice A and b, using Equation 1.11. We can obtain:

(1.13) 
$$T(0,0) = T(1,0) + \frac{\Delta T_g}{2} \implies T(0,0) - T(1,0) = \frac{\Delta T_g}{2}$$

$$(1.14) \quad T(1,0) = \frac{T(0,0) + T(2,0) + 2T(1,1)}{4} + \frac{\Delta T_g}{2} \implies T(1,0) - \frac{T(0,0)}{4} - \frac{T(2,0)}{4} - \frac{T(1,1)}{2} = \frac{\Delta T_g}{2}$$

$$(1.15) \hspace{1cm} T(1,1) = \frac{2T(1,0) + 2T(2,1)}{4} + \frac{\Delta T_g}{2} \implies T(1,1) - \frac{T(1,0)}{2} - \frac{T(2,1)}{2} = \frac{\Delta T_g}{2}$$

$$(1.16) \quad T(2,0) = \frac{T(1,0) + T(3,0) + 2T(2,1)}{4} + \frac{\Delta T_g}{2} \implies T(2,0) - \frac{T(1,0)}{4} - \frac{T(3,0)}{4} - \frac{T(2,1)}{2} = \frac{\Delta T_g}{2}$$

$$T(2,1) = \frac{T(1,1) + T(3,1) + T_S + T(2,0)}{4} + \frac{\Delta T_g}{2}$$

$$\implies T(2,1) - \frac{T(1,1)}{4} - \frac{T(3,1)}{4} - \frac{T(2,0)}{4} = \frac{T_S}{4} + \frac{\Delta T_g}{2}$$

T(i,0) and T(i,1) for  $i \in [2,10]$  will be identical, given the geometry of the problem.

$$(1.18) \quad T(11,0) = \frac{T(10,0) + 2T(11,1) + T_S}{4} + \frac{\Delta T_g}{2} \implies T(11,0) - \frac{T(11,1)}{2} - \frac{T(10,0)}{4} = \frac{T_S}{4} + \frac{\Delta T_g}{2}$$

$$(1.19) \quad T(11,1) = \frac{T(11,0) + T(10,1) + 2T_S}{4} + \frac{\Delta T_g}{2} \implies T(11,1) - \frac{T(11,0)}{4} - \frac{T(10,1)}{4} = \frac{T_S + \Delta T_g}{2}$$

Consequently, we can write A:

And we can also write *b*:

$$\begin{bmatrix} 8.75 \\ 8.75 \\ 8.75 \\ 8.75 \\ 8.75 \\ 8.75 \\ 158.75 \\ 8.75 \\$$

We can now solve Equation 1.12, using Python. The python script is given in 1.3.2.1. We obtain the following results (Equation 1.20):

$$T(0,0) = 723.8$$

$$T(1,0) = 715.1$$

$$T(1,1) = 703.2$$

$$T(2,0) = 695.1$$

$$T(2,1) = 673.8$$

$$T(3,0) = 682.5$$

$$T(3,1) = 662.0$$

$$T(4,0) = 675.9$$

$$T(4,1) = 656.8$$

$$T(5,0) = 672.5$$

$$T(5,1) = 654.3$$

$$T(6,0) = 670.6$$

$$T(6,1) = 652.9$$

$$T(7,0) = 668.9$$

$$T(7,1) = 651.8$$

$$T(8,0) = 666.7$$

$$T(8,1) = 650.2$$

$$T(9,0) = 662.5$$

$$T(9,1) = 647.2$$

$$T(10,0) = 654.0$$

$$T(10,1) = 641.1$$

$$T(11,0) = 636.3$$

$$T(11,1) = 628.1$$

(1.20)

## 1.3.2.1 Python script

```
import numpy as np
m = 23
dx = 0.1
deltax, k, qtriple = (dx/12), 19.84, 1.e7
ts = 600.
dtg = (0.5*deltax**2)*qtriple/k
amat = np.zeros((m,m))
b = np.zeros(m)
amat[0,0] = 1.
amat[0,1] = -1.
b[0] = dtg/2.
amat[1,0] = -0.25
amat[1,1] = 1.
amat[1,2] = -0.5
amat[1,3] = -0.25
b[1] = dtg/2.
amat[2,1] = -0.5
amat[2,2] = 1.
amat[2,4] = -0.5
b[2] = dtg/2.
for i in range(3, m, 2):
    try:
        \mathtt{amat[i,i-2]} = -0.25
        amat[i,i] = 1.
        \mathtt{amat[i,i+1]} = -0.5
        \mathtt{amat[i,i+2]} = -0.25
        b[i] = dtg/2.
    except IndexError:
        break
for i in range(4, m, 2):
   try:
        amat[i,i-2] = -0.25
        amat[i,i-1] = -0.25
        amat[i,i] = 1.
        \mathtt{amat[i,i+2]} = -0.25
        b[i] = ts/4. + dtg/2.
    except IndexError:
        break
amat[m-1,m-1] = 1.
amat[m-1,m-2] = -0.25
amat[m-1,m-3] = -0.25
b[m-1] = ts/2. + dtg/2.
```

```
amat[m-2,m-2] = 1.
amat[m-2,m-1] = -0.5
amat[m-2,m-4] = -0.25
b[m-2] = ts/4. + dtg/2.
ysol = np.linalg.solve(amat,b)
print('Temperature solution:')
temp = []
for i in range(12):
  for j in range(2):
       if i == 0 and j == 1:
           continue
       temp.append("T(%s,%s)" % (i, j))
for i,t in enumerate(temp):
    print("%s = %.1f" % (t, ysol[i]))
# check of solution:
if not np.allclose(np.dot(amat, ysol), b):
    print("Solution does not match!")
```

# **BIBLIOGRAPHY**

[1] M. M. EL-WAKIL, Nuclear Heat Transport, American Nuclear Society, 1993.