

**ESO201A**  
**Lecture#29**  
**(Class Lecture)**

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By

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# Adiabatic Flame Temperature

①

Consider a given combustion process that takes place adiabatically and with no work or changes in kinetic or potential energy involved. For such a process (Fig. 1) the temperature of the products is referred to as the adiabatic flame temperature. With the assumptions of no work and no changes in kinetic or potential energy, this is the maximum temperature that can be achieved for the given reactants because any heat transfer from the reacting substances and any incomplete combustion would tend to lower the temperature of the products.

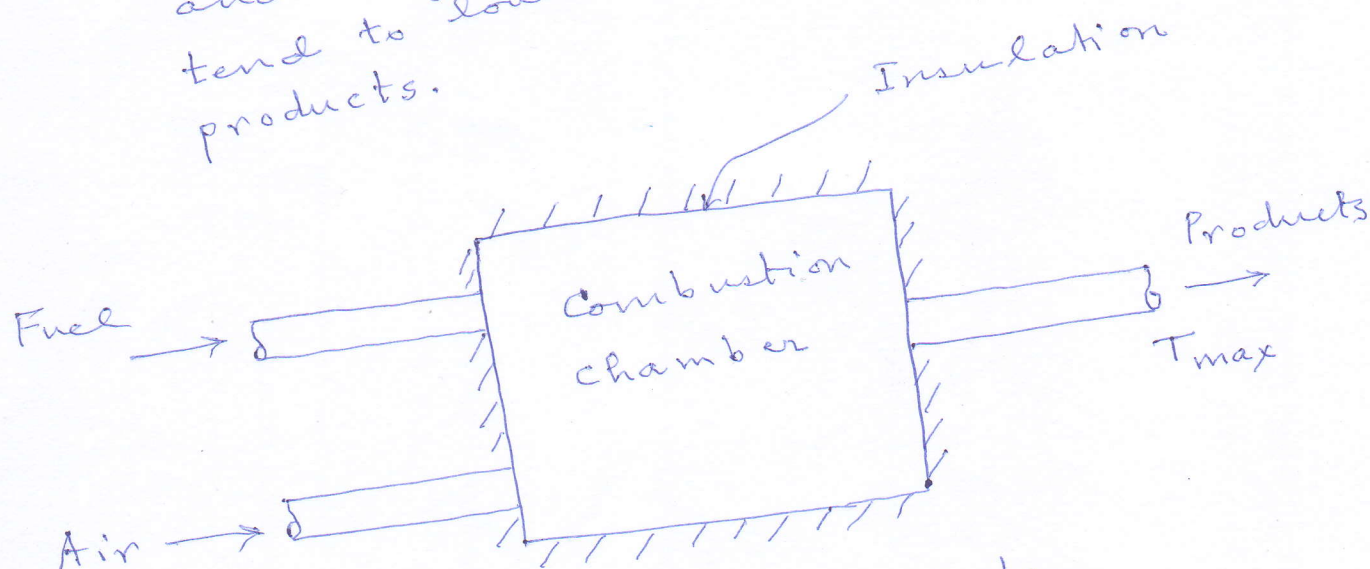


Fig. 1 The temperature of a combustion chamber becomes maximum when the combustion is complete and  $Q=0$ ,  $W=0$ ,  $\Delta KE=0$ ,  $\Delta PE=0$  and the fuel-air mixture is stoichiometric

For a given fuel and given pressure and temperature of the reactants, the maximum adiabatic flame temperature that can be achieved is with a stoichiometric mixture. The adiabatic flame temperature can be controlled by the amount of excess air that is used. This is important, for example, in gas turbines, where the maximum permissible temperature is determined by metallurgical considerations in the turbine and close control of the temperature of the products is essential.

The adiabatic flame temperature of a steady-flow combustion process is determined from the application of first law to the combustion chamber by setting

$$Q = 0, W = 0, \Delta KE = 0, \Delta PE = 0.$$

$$\dot{Q} - \dot{W} = H_{\text{products}} - H_{\text{reactants}}$$

$$\Rightarrow H_{\text{products}} = H_{\text{reactants}}$$

where

$$H_{\text{products}} = \sum_p N_e (\bar{h}_f^0 + \Delta \bar{h})_e$$
$$H_{\text{reactants}} = \sum_r N_i (\bar{h}_f^0 + \Delta \bar{h})_i$$

In other words,

$$\sum_p N_p (\bar{h}_f^\circ + \bar{h} - \bar{h}^\circ)_p = \sum_r N_r (\bar{h}_f^\circ + \bar{h} - \bar{h}^\circ)_r$$

Once the reactants and their states are specified, the enthalpy of the reactants  $H_{\text{react}}$  can be easily determined. The calculation of the enthalpy of the products  $H_{\text{prod}}$  is not so straightforward, however, because the temperature of the products is not known prior to the calculations.

Therefore, the determination of the adiabatic flame temperature requires the use of an iterative technique. A temperature is assumed for the product gases, and  $H_{\text{prod}}$  is determined for this temperature. If it is not equal to  $H_{\text{react}}$ , calculations are repeated with another temperature. The adiabatic flame temperature is then determined from these two results by interpolation.



④

When the oxidant is air, the product gases mostly consist of  $N_2$ , and a good first guess for the adiabatic flame temperature is obtained by treating the entire product gases as  $N_2$ .

### Example Problem #1

Liquid octane at standard reference state is burned with 400% theoretical air at the same state in a steady-flow process. Determine the adiabatic flame temperature.

Solution:

Control volume: Combustion chamber

Inlet state for fuel:  
0.1 MPa, 25°C

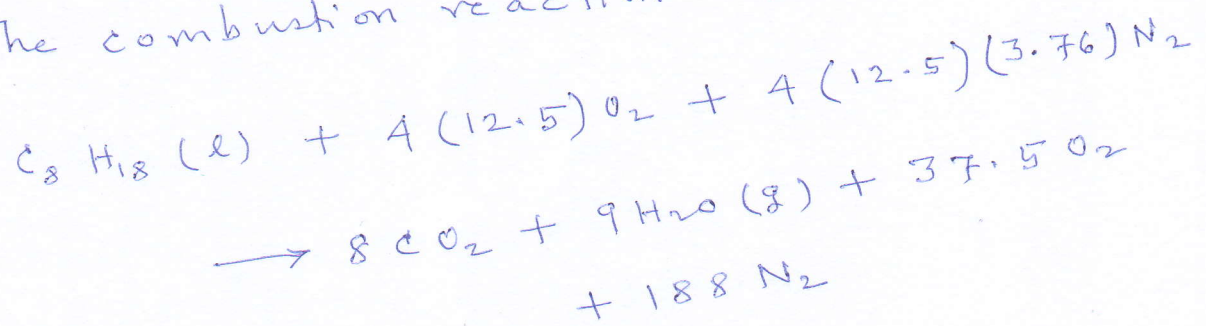
Inlet state for air:  
0.1 MPa, 25°C

Process: steady state

Assumption: All gases are ideal gases

(5)

The combustion reaction is :



$$H_{\text{react}} = \sum_r N_i (\bar{h}_f^\circ + \Delta \bar{h})_i$$

$$= (1)(\bar{h}_f^\circ)_{\text{C}_8\text{H}_{18}(\text{l})}$$

$$= \boxed{-249,950 \text{ kJ/kmol fuel}}$$

$$H_{\text{prod}} = \sum_p N_e (\bar{h}_f^\circ + \Delta \bar{h})_e$$

$$= 8(-393,520 + \Delta \bar{h}_{\text{CO}_2}) + 9(-241,820 + \Delta \bar{h}_{\text{H}_2\text{O}}) \\ + 37.5 \Delta \bar{h}_{\text{O}_2} + 188 \Delta \bar{h}_{\text{N}_2}$$

Iteration #1

Assuming  $T_p = 900 \text{ K}$ ,

$$H_{\text{prod}} = 8(-393,520 + \bar{h}_{\text{CO}_2, 900\text{K}} - \bar{h}_{\text{CO}_2, 298\text{K}}) \\ + 9(-241,820 + \bar{h}_{\text{H}_2\text{O}, 900\text{K}} - \bar{h}_{\text{H}_2\text{O}, 298\text{K}})$$

$$+ 37.5 (\bar{h}_{\text{O}_2, 900\text{K}} - \bar{h}_{\text{O}_2, 298\text{K}})$$

$$+ 188 (\bar{h}_{\text{N}_2, 900\text{K}} - \bar{h}_{\text{N}_2, 298\text{K}})$$

(6)

$$\Rightarrow H_{\text{Prod}} = 8(-393,520 + 37,405 - 9364) \\ + 9(-241,820 + 31,828 - 9904) \\ + 37.5(27,928 - 8682) \\ + 188(26,890 - 8669)$$

$$= 8(-365,479) + 9(-219,896) \\ + 37.5(19,246) \\ + 188(18,221)$$

$$= -2,923,832 + (-1,979,064) \\ + 721,725 + 3,425,548$$

$$= \boxed{-755,623 \text{ kJ/kmol fuel}}$$

$H_{\text{Prod, iter \#1}} \neq H_{\text{React}}$

Iteration # 2

$$T_{\text{Prod}} = 1000 \text{ K}$$

$$H_{\text{Prod}} = 8(-393,520 + 42,769 - 9364) \\ + 9(-241,820 + 35,882 - 9904) \\ + 37.5(31,389 - 8682) \\ + 188(30,129 - 8669)$$



(7)

$$\Rightarrow H_{\text{prod}} = 8(-360,115) \\ + 9(-215,842) \\ + 37.5(22,707) \\ + 188(21,460)$$

$$= \boxed{62,494.5 \text{ kJ/kmol fuel}}$$

$$H_{\text{prod, iter \# 2}} \neq H_{\text{react}}$$

Since  $H_{\text{prod}} = H_{\text{react}} = -249,950 \text{ kJ/kmol fuel}$ , we find by linear interpolation that the adiabatic flame temperature is  $961.79 \text{ K}$ . Because ideal-gas enthalpy is not really a linear function of temperature, the true answer will be slightly different from this value.

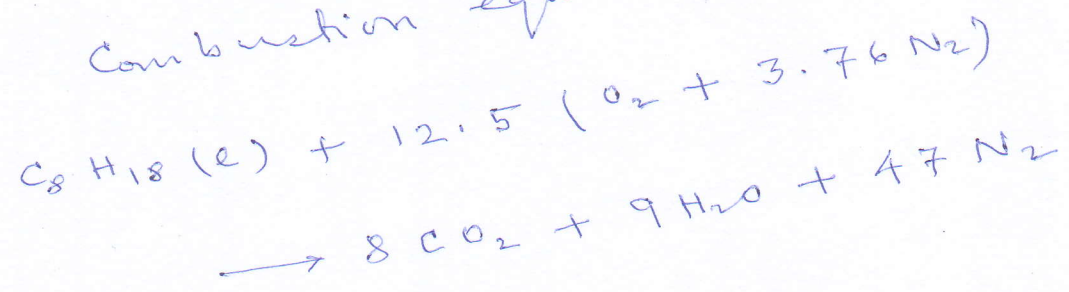


## Example Problem #2

Liquid octane ( $C_8H_{18}$ ) enters the combustion chamber of a gas turbine steadily at 1 atm and  $25^\circ C$ , and it is burned with air that enters the combustion chamber at the same state. Determine the adiabatic flame temperature for (a) complete combustion with 100% theoretical air, (b) complete combustion (but with free  $O_2$  in the products) with 400% theoretical air, and (c) incomplete combustion (some CO in the products) with 90% theoretical air.

### Solution:

(a) Combustion equation:



Since  $H_{prod} = H_{react}$

(9)

$$\sum_p N_e (\bar{h}_f^o + \bar{h} - \bar{h}^o)_e = \sum_r N_e (\bar{h}_f^o + \bar{h} - \bar{h}^o)_i$$

The above equation reduces to

$$\begin{aligned} \sum_p N_e (\bar{h}_f^o + \bar{h} - \bar{h}^o)_e &= \sum_r N_i \bar{h}_f^o \\ &= (N \bar{h}_f^o)_{C_8 H_{18}} \end{aligned}$$

since all reactants are at the standard reference state and  $\bar{h}_f^o = 0$  for  $O_2$  and  $N_2$ . The  $\bar{h}_f^o$  and  $\bar{h}$  values of various components at 298K (considering all gases ideal gases) are shown in Table 1.

Table 1

Substance	$\bar{h}_f^o$ kJ/kmol	$\bar{h}_{298K}$ kJ/kmol
$C_8 H_{18} (l)$	-249,950	8682
$O_2$	0	8669
$N_2$	0	9904
$H_2O (g)$	-241,820	9364
$CO_2$	-393,520	

Substituting we have

$$\begin{aligned}
 & (8) [(-393,520 + \bar{h}_{\text{CO}_2} - 9364)] \\
 & + (9) [(-241,820 + \bar{h}_{\text{H}_2\text{O}} - 9904)] \\
 & + 47 [(0 + \bar{h}_{\text{N}_2} - 8669)] \\
 & = (1) (-249,950)
 \end{aligned}$$

$$\begin{aligned}
 \Rightarrow 8 \bar{h}_{\text{CO}_2} + 9 \bar{h}_{\text{H}_2\text{O}} + 47 \bar{h}_{\text{N}_2} \\
 = 5,646,081 \text{ kJ}
 \end{aligned}$$

The only unknown in the above equation is the temperature of the products  $T_{\text{prod}}$  since  $h = h(T)$  for ideal gases.



A trial-and-error approach is used to determine the temperature of the products.

A first guess is obtained by dividing the right hand side of the equation by the total number of moles, which yields

$$\begin{aligned} & 5,646,081 / (8 + 9 + 47) \\ & = 88,220 \text{ kJ/kmol} \end{aligned}$$

This enthalpy value corresponds to about 2650 K for  $N_2$  (Table A-18). Noting that the majority of the moles are  $N_2$ , we see that  $T_{\text{prod}}$  should be close to 2650 K, but somewhat under it. Therefore, a good first guess is 2400 K.

At this temperature,

$$8 \bar{h}_{\text{CO}_2} + 9 \bar{h}_{\text{H}_2\text{O}} + 47 \bar{h}_{\text{N}_2}$$

$$= 8 (125, 152) + 9 (103, 508) + 47 (79, 320)$$

$$= 5, 660, 828 \text{ kJ}$$

This value is higher than 5, 646, 081 kJ. Therefore,

the actual temperature is slightly under 2400 K. Next we choose 2350 K. It yields

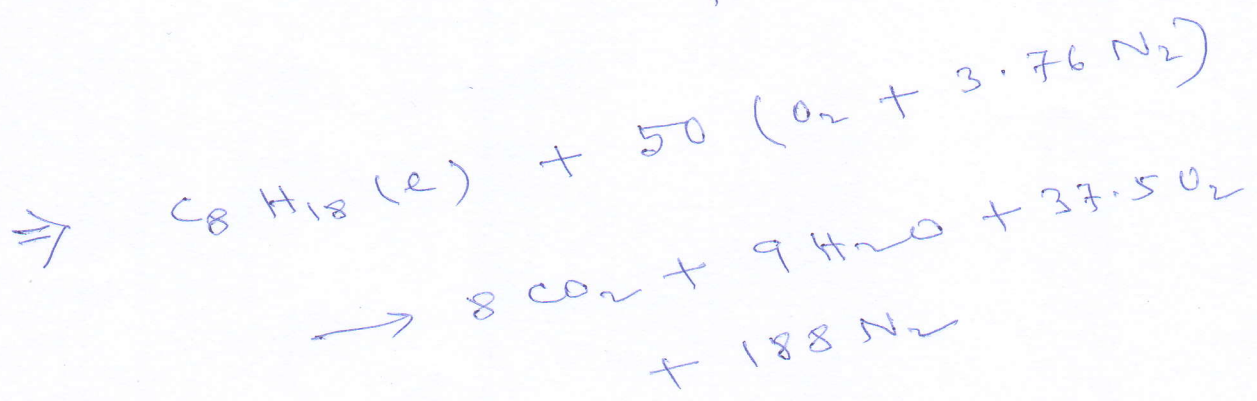
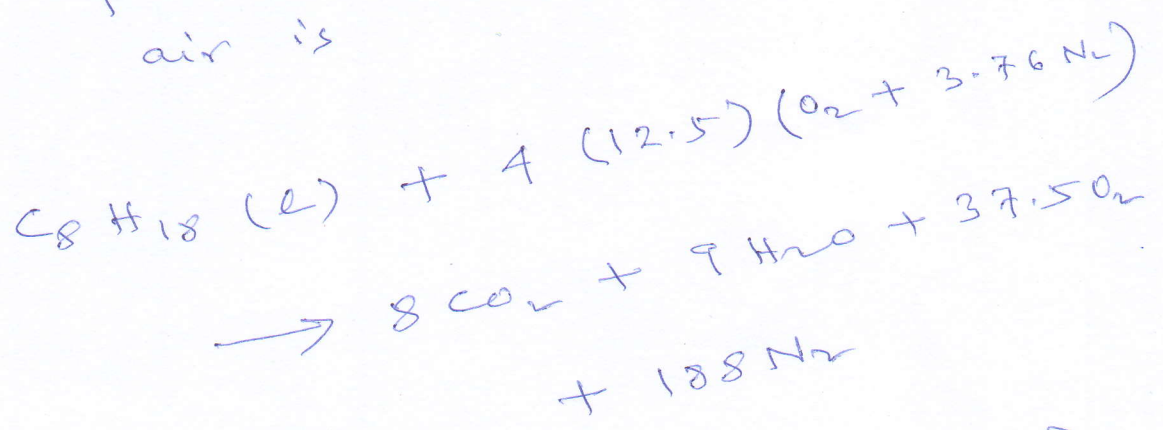
$$8 (122, 091) + 9 (100, 846) + 47 (77, 496)$$

$$= 5, 526, 654$$

which is lower than 5, 646, 081 kJ.

Therefore, the actual temperature of the products is <sup>linear</sup> between 2350 and 2400 K. By interpolation, it is found to be  $T_{\text{prod}} = 2395 \text{ K}$ .

(b) The balanced equation for the complete combustion process with 400% theoretical air is

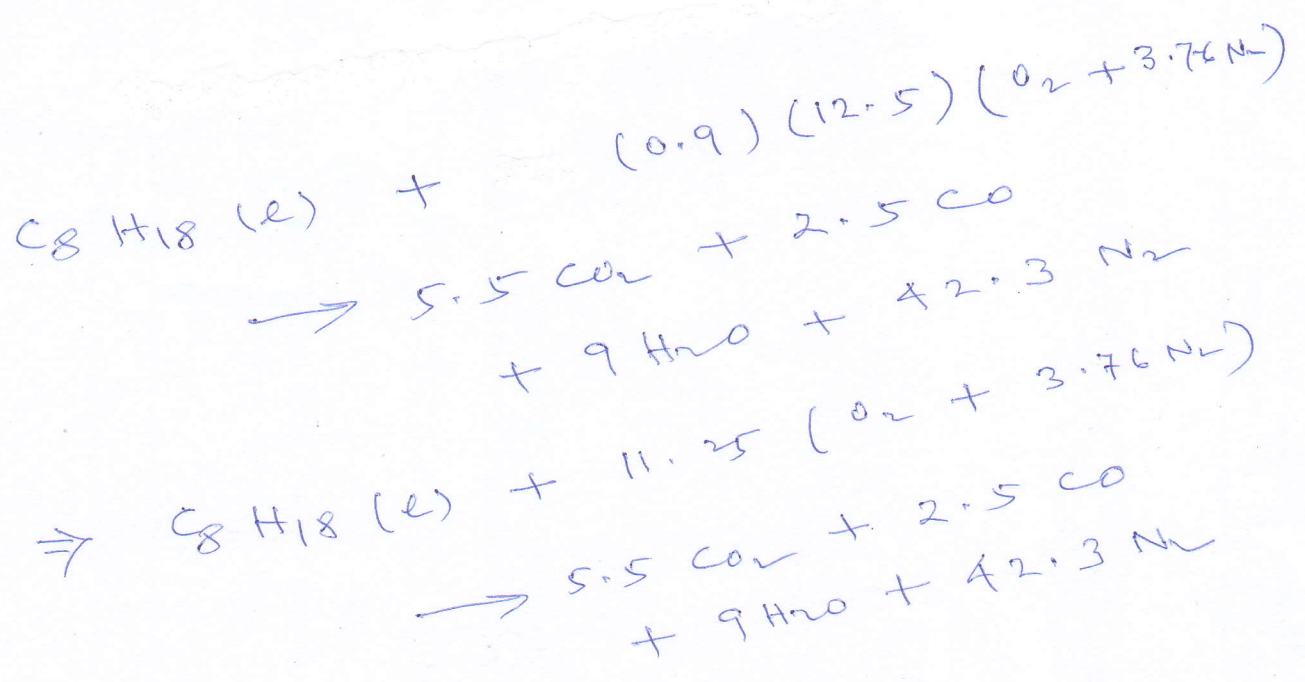




By following the procedure in (a), the adiabatic flame temperature in this case is determined to be  $T_{\text{prod}} = 962 \text{ K}$ .

Notice that the temperature of the products decreases significantly as a result of using excess air.

(c) The balanced equation for the incomplete combustion process with 90% theoretical air is



Following the procedure in (a), we find that the adiabatic flame temperature in this case is  $T_{\text{prod}} = 2236 \text{ K}$ .

### Conclusions

Notice that the adiabatic flame temperature decreases as a result of incomplete combustion or using excess air. Also, the maximum adiabatic flame temperature is achieved when complete combustion occurs with the theoretical amount of air.