Harmonie Oscillations

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-ntroduction

Harmonic oscillators are usually described in three variations. In the simplest case, there is no external force and no friction. Secondly, there can be still no external force but a friction that is proportional to re-locity. In the third and most complicated case, there is both an external (harmonic) torce and friction.

The basic harmonic ossillator only involves a repolling force that is directly proportional to displacement (but in opposite direction)

F= $m\bar{x} = -Dx$ which is easily solved with the ansatz $x(t) = e^{i\omega t}$ as $x(t) = A_0 \cos(\omega_0 t) + A_1 \sin(\omega_0 t)$ with the angular frequency $\omega_0 = \sqrt{\frac{D}{m}}$

When we take friction into account, we get one more tem in the differential equation:

mx + kx + Dx =0

Again, we can use the same ansatz x(t) = xivot

 $\Rightarrow -\omega^2 m \times + i \omega k \times + D \times = 0$

$$\left(\omega - \frac{ik}{2m}\right)^2 = \frac{D}{m} - \frac{k^2}{4m^2}$$

$$\Rightarrow \omega_{n,2} = i \frac{k}{2m} \pm \sqrt{\frac{D}{m} - \frac{k^2}{4m^2}}$$

so is again the angular frequency of the resulting

When we insert this back into the ansatz, we will find that there is a a harmonic ostillation only if we has no imaginary part, i.e. $\omega^2 > \frac{k^2}{4m^2}$. In any other case, the oscillator will return to its position of rest andy. If the friction is small enough, however, and allows oscillation, the movement can be described as

 $x(t) = e^{-St} \left[A_1 \cos(\omega_1 t) + A_2 \sin(\omega_2 t) \right] \quad \text{with}$ $A_1 = x_0 \quad \text{and} \quad A_2 = \frac{Sx_0 + \dot{x}}{\dot{\omega}_0} \quad \text{or, more conveniently}$ $x(t) = A \cdot e^{-St} \cos(\omega_1 t + \beta)$

When we add an external driving force, be expect the system to adapt the driving frequency (forced oscillation). Mathematically, we have to solve an inhomogeness lifterential equation

 $m\ddot{x} + k\ddot{x} + Dx = f_0 \cos(\Omega t)$

To solve it, we must find one particular solution, which we then add to the homogeneous solution.

As an ansatz, we choose \times (t) = A_5 (os (Ω t+d),

Ex(t) = As e (2t+6)

When we insert this ansatz, we find

$$A_{S} = \frac{F_{o}/m}{\sqrt{(\omega_{o}^{2} - \Omega)^{2} + 4S^{2}\Omega}}$$

$$\phi = Arctan \left(\frac{-2S\Omega}{\omega^{2} - \Omega^{2}}\right)$$

After a sufficiently long time, this is the only remaining part of the solution, as the homogeneous part decreases exponentially

During the initial phase, when this term still has a visible effect, we have to consider the full solution

 $x(t) = 4 \cdot e^{-8t} \cos(\omega_0 t + \beta) + A_5(\Omega) \cos(\Omega t + \phi)$

The resulting escillation can be very complicated.

With certain initial conditions we can find a simply solution

If 8440, $\Omega = 40$, $x_0 = 0$, $x_0 = 0$ $\Rightarrow x(t) = A_5 \left[\cos(\Omega t + \phi) - e^{-8t} \cos(\omega t + \beta) \right]$ This results in beads with a frequency of $\frac{|\Omega - \omega|}{2}$ If the two cosine terms have are equal with apposite sign, the maximum initial amplitude is reached $\Omega = 400$, we find that $x(t) = A_5 (1 - e^{-8t}) \sin \omega_0 t$

As we have seen As is dependent on Q. It has a peak close to wo, the width of the curve is determined by the friction coefficient

If this coefficient is small, the amplitude can reach very high values at the resonance frequency, and will jump quickly to its peak

If we are and by -Dl=10 we can write to

If w= and lw; -21=12 we can write As in a simplified way as

$$\Delta_{S}(\Delta\Omega) = \frac{E/m}{2S\omega\sqrt{1+\left(\frac{\Delta\Omega}{S}\right)^{2}}}$$

The amplitude reaches its maximum at $\Omega = \omega_0$ then and has a value $\frac{A_{max}}{127}$ at $\Omega = \omega_0 \pm 8$, which allows to easily measure S

Assignments

- 1) Examination of harmonic oscillation with friction, bu without external force. Maisurement of displacement in dependency of time.

 (alculation of they eigen beginning and the system's friction coefficient
- 2) Examination of forced oscillations. Treasurement of displacement in dependency of frequency Calculation of eigen frequency and briefion coefficient.
- 3) Qualitative examination of the phase shift between the external force and the oscillator, in dependency of the external force's trequency
- 4) Examination of the initial oscillation in the case of resonance and in the vicinity

Experiment

21.03 05

Start 945 and 1300

Michael Goerz Into Maase

Tutor: Angolica Zacarias

4. signment 1 : See printont

damping levels 1,2,3 the "undamped" oscillator has a constant friction!

Assignment ? : see printont

frequency (V)	am pli Ande	incurrence of damping level 1		
Ť	0,0\$ 57 ± 0,0070	trequercy is gi	ven as th	e velturge
8	0,0162 ±0,0020	at the driving	notr	
9	0,0280	the error is from	m the non	- Constant
10	0,000	amplitude		
11	0,2800 ±0,0100			
12	0,0570 ±0,0010		1	
13	0,0320 + 0,0010	frequency	amplitud	
۷۵'5	0,6980 ± 0,0026	11,3	0,1000	£0,0100
10,4	0,1300 ± 0,0100	$\Lambda \Lambda_1 \mathbf{k}$	0,0930	±0,0036
10,6	0, 1700 ± 0,0100	V1')	0,0610	
110,8	Ů, 4100	10,5	0,1500	± 0,0100
10,7	0,3000	9,5	0,0370	t0,0030
10,9	0,5300	M, 5	0,685	
11,1	0,1800			
14,Z	0,1100			

Conversion Voltage > Heats

11,5 V -> 7,8 Sec for 5 oscillations

10,9 V -> 8,5 Sec for 5 "

10,6 V -> 8,7 Sec "

11,1 V -> S,1 Sac

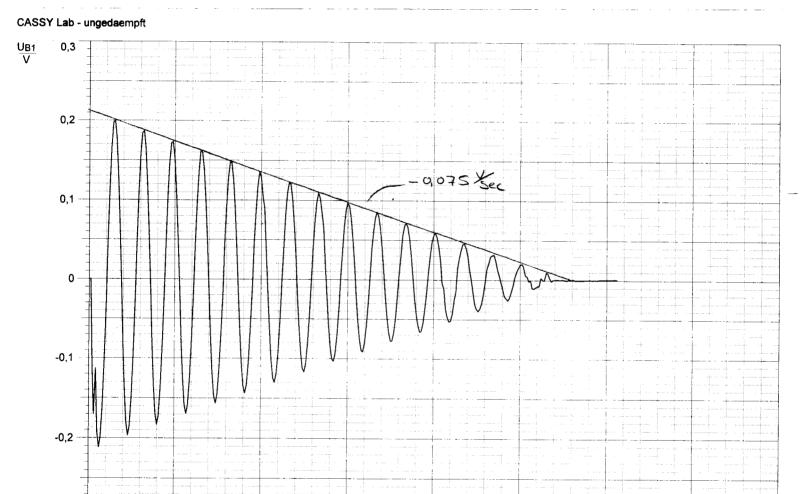
Assignment 3: at low Ω ($\Omega_v = 7V$) oscillators tollows external torce at resonance ($\Omega_v = 10.9V$) = 90° phase shift at high Ω ($\Omega_v = 13V$) $\approx 180°$ phase shift

Assignment 4: See printent

Analysis

"Undamped" oscillation:

As a controll measurement, we tried the indamped and undriven oscillator. It appears that there is a constant friction reducing the amplitude. This might have a further effect on other measurements A linear approximation of that friction shows that there is about 0,075 be lost in the amplitude



In the following measurement of the driven oscillator, however, that would not be a valid calculation because

of the constantly acting external force. Instead, the friction will show as a systematic error, the measured amplitudes will be slightly lower than their real value. The exact error would be hard to determine, but it probably is in the order of several percent.

Assignment A:

Also, in the experiment with the damped oscillator, this linear correction cannot be applied without destroying the exponential decrease, as we find by trial. We are not completely sure how to correctly include the friction and will ignore it for now , bearing in mind that it should at least give a systematic error.

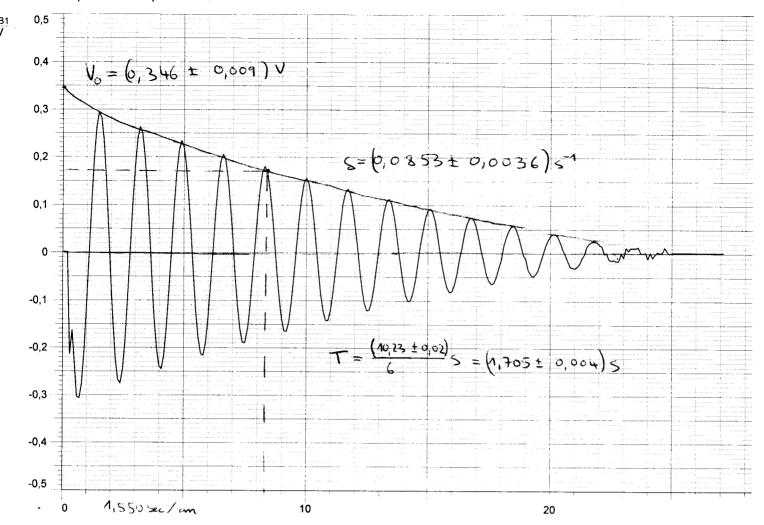
Assignment 1:

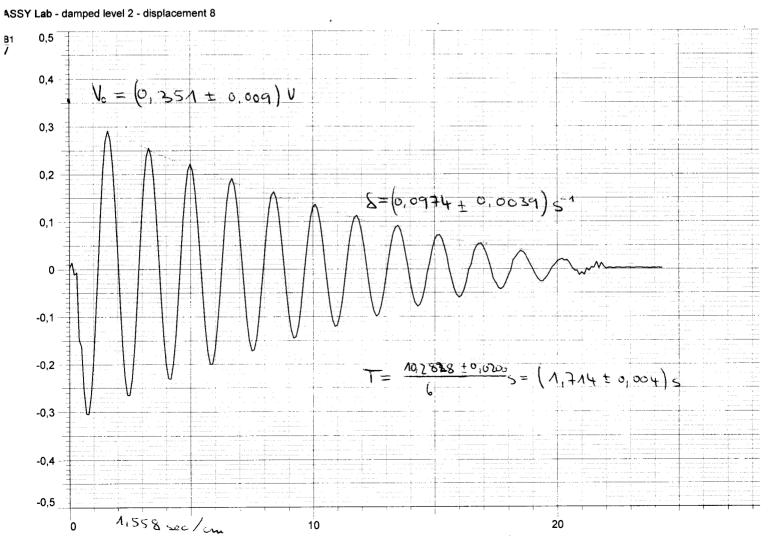
In this assignment, we worked with three different damping levels for each measurement, we can read S, the damping of decays, from the logicale plot and the time of one period from the normal graph.

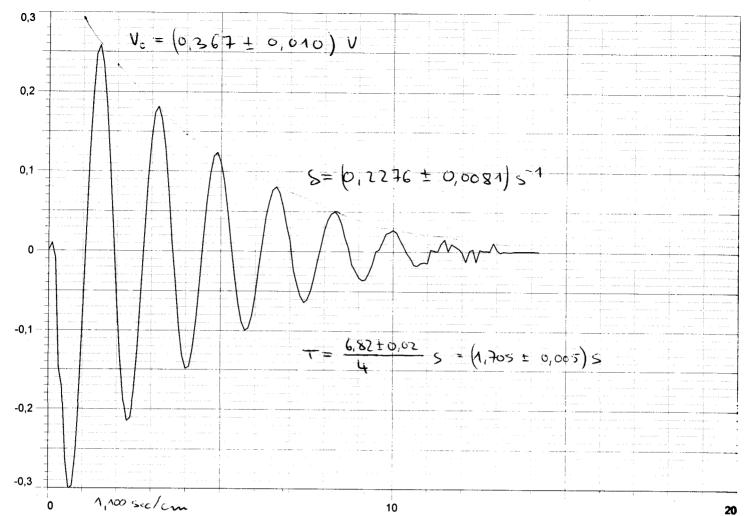
From this information, is can calculate we (frequency of the undamped oscillator) as follows:

$$\omega_1 = \sqrt{\frac{D}{m} - \frac{k^2}{4m^2}} \iff \omega_0^2 = \omega_1^2 + S^2$$

Theoretically, we could calculate $k = 8.2 \, \text{m}$, but since we have neither m nor D we have to use 8 as a description for k.







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For the eigen frequencies, we get:

level 1: w = 3,686 Hz

level ? . Wo = 3,667 Hz

level 3 Wc = 3,692 Hz

the error on these values is < 1%

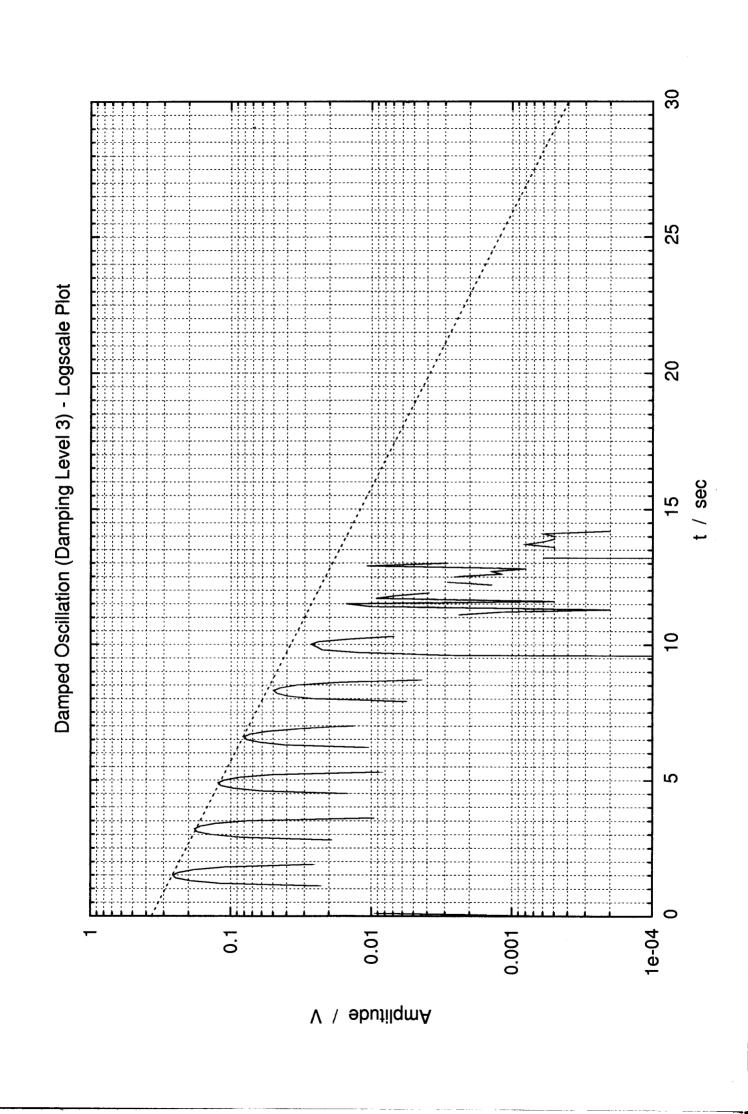
In theory, the values should be identical, the discrepancy is probably due to systematic errors (like the linear friction influence)

For comparison, the value read from the undamped oscillation is up = 3,689 Hz

Both the exponential decay of your frequency depends of your damping level.

30 25 Damped Oscillation (Damping Level 1) - Logscale Plot 20 10 Ŋ 0.001 0.1 0.01 V \ əbutilqmA

30 25 Damped Oscillation (Damping Level 2) - Logscale Plot 20 10 Ŋ 0.1 0.01 V \ əbutilqmA



```
************************
Tue Mar 22 17:28:16 2005
        data read from "damped1_orig.csv" every :::1::1
        #datapoints = 11
        residuals are weighted equally (unit weight)
After 6 iterations the fit converged.
final sum of squares of residuals: 0.000731234
rel. change during last iteration: -4.37473e-07
degrees of freedom (ndf) : 9
rms of residuals
                   (stdfit) = sqrt(WSSR/ndf)
                                                 : 0.00901378
variance of residuals (reduced chisquare) = WSSR/ndf : 8.12483e-05
Final set of parameters
                                 Asymptotic Standard Error
               = 0.3462
                                 +/- 0.00877
                                                 (2.533%)
                                 +/- 0.003532 (4.137%)
               = 0.085362
k 1
correlation matrix of the fit parameters:
              A1
               1.000
A 1
k1
               0.801 1.000
data read from "damped2_orig.csv" every :::1::1
        #datapoints = 10
       residuals are weighted equally (unit weight)
After 6 iterations the fit converged. final sum of squares of residuals: 0.000528404
rel. change during last iteration : -5.82171e-07
degrees of freedom (ndf) : 8
rms of residuals (stdfit) = sqrt(WSSR/ndf) : 0.00812715
variance of residuals (reduced chisquare) = WSSR/ndf : 6.60505e-05
Final set of parameters
                                 Asymptotic Standard Error
                                +/- 0.00874 (2.491%)
+/- 0.003813 (3.915%)
A2
               = 0.350893
k2
               = 0.0973979
correlation matrix of the fit parameters:
              A2
A2
              1.000
               0.809 1.000
data read from "damped3_orig.csv" every :::1::1
FIT:
       #datapoints = 5
       residuals are weighted equally (unit weight)
After 7 iterations the fit converged.
final sum of squares of residuals: 7.16525e-05
rel. change during last iteration: -1.29311e-07
degrees of freedom (ndf) : 3
rms of residuals
                   (stdfit) = sqrt(WSSR/ndf)
                                               : 0.00488714
variance of residuals (reduced chisquare) = WSSR/ndf : 2.38842e-05
Final set of parameters
                                Asymptotic Standard Error
                                +/- 0.009612
+/- 0.008009
АЗ
               = 0.366875
                                                (2.62%)
k3
               = 0.227618
                                                (3.519%)
correlation matrix of the fit parameters:
              АЗ
              A3 k3
1.000
А3
k3
              0.848 1.000
```

In comparison, the three damped oscillations behave as exspected. The amplitude decreases exponentially, a higher decay constant.

Assignment 2:

We can get the eigen frequency directly from the graph of the displacement measured at resonance. We get up = 3,738 Hz without any significant reading error 'See next page)

Theoretically, the resonance curse (amplitude in dependency of external trequincy R) Should also provide a measurement of us, but various problems make this data muscle. Firstly, the resolution wound the puch is very low for exact values and secondly, the attempt to convert the voltage at the motor to a a frequency R was unsuccessful. The data collected for this purpose does not fit any linear law. If we try to do this anyway, we can only say that $R = (0.345 \pm 0.169) \cdot V$ (values calculated with graphet)

We can than read from the graph $\omega_0 = (3,754 \pm 1,186)$ Hz

Liberise, S should be readcible from the graph, The same problems apply. Also, the curve is not symmetric around to at the levels of $\frac{1}{12}$, where we could read of S.

The average value is $S = 0.0328 \pm 0.0053$, which is still not anywhere near the exspected value. In addition to the mentioned problems, there are also systematic errors.

Qualitatively, however, the resonance curve is exactly as we expect, with the sharp peak at the resonance frequency and the asymmetric development for bett and right from that frequency

Assignment 3.

The observations tollow our expectations. At low chrising frequences, there is enough time for the spring to follow the motor's movement. With increasing frequency, the oscillator's inertia will create a phase shift. At resonance, the phase shift will be exactly 90°, which makes sense, as we can add a maximum of energy when exerting force at the moment when the pendulum goes through its rest position (just like on a swing)

Finally, at the very high frequencies, the pendulum's inertia is too high to follow the motor, the phase shift becames 180°

Assignment 4:

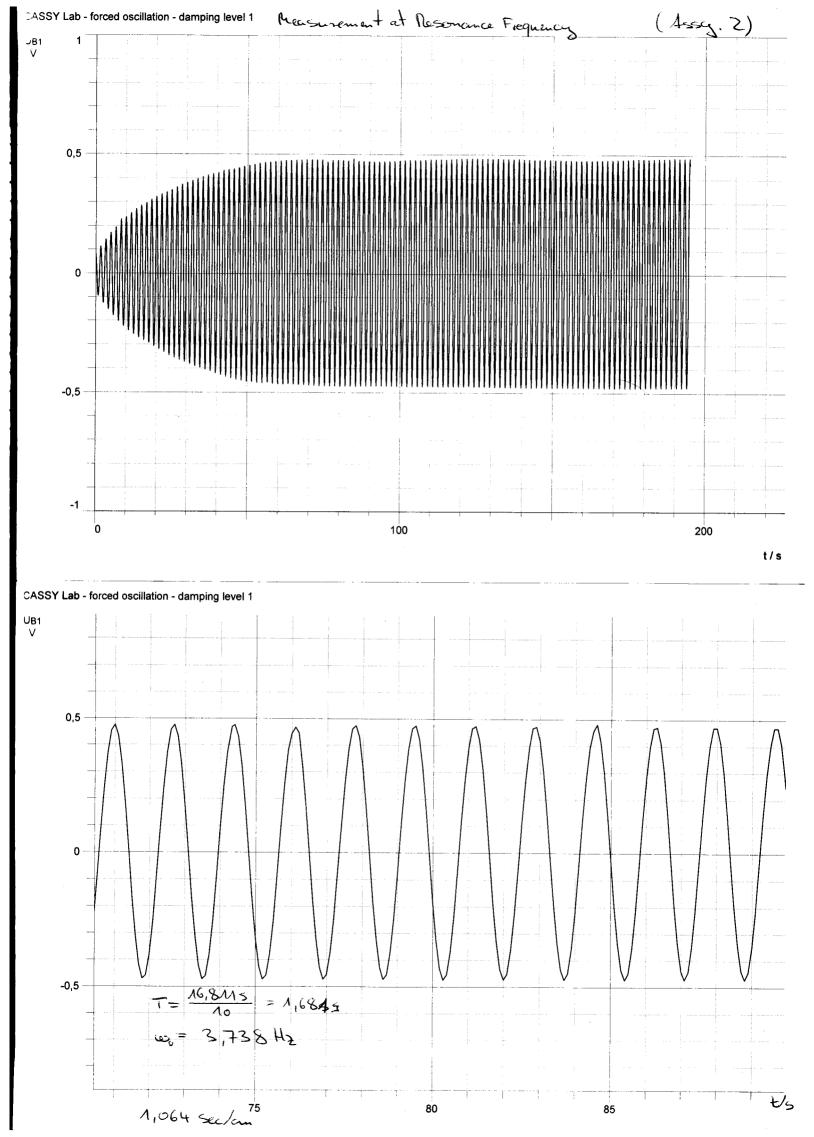
The initial phases behave like exspected. The first graph (10,8V) is barely below the resonance frequency. We can see that the amplifude reaches a higher level at first, and then stabilizes on the lever level. The terminal phase, when the motor is switched off, behaves like

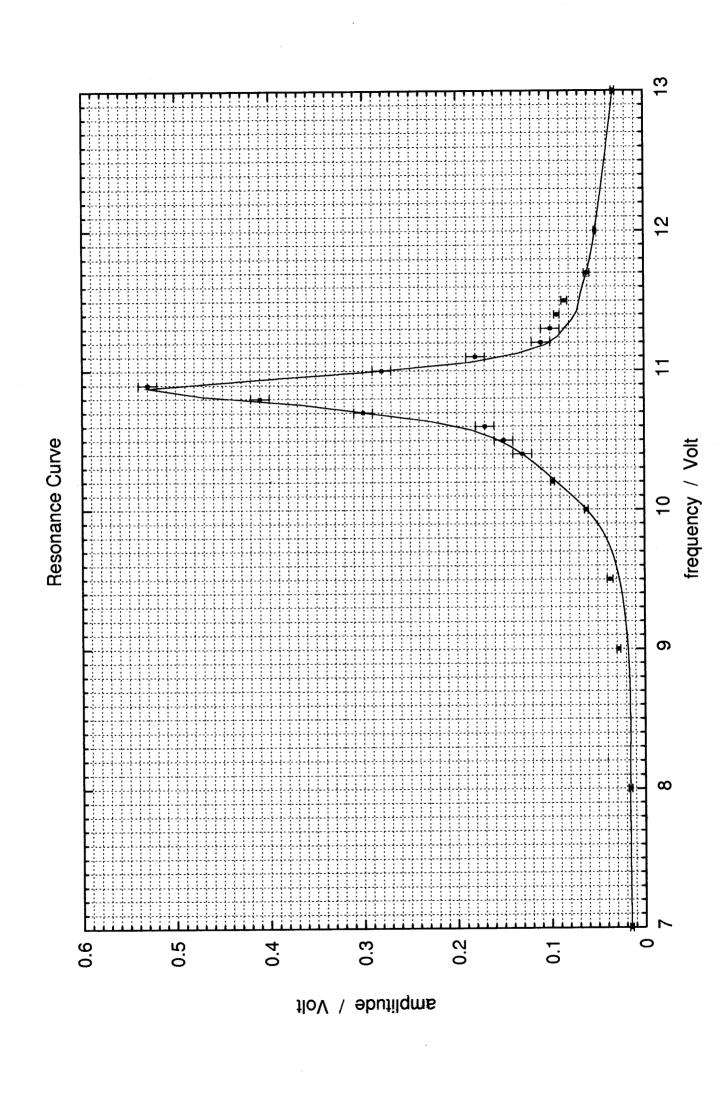
(Asses. 4, cont.)

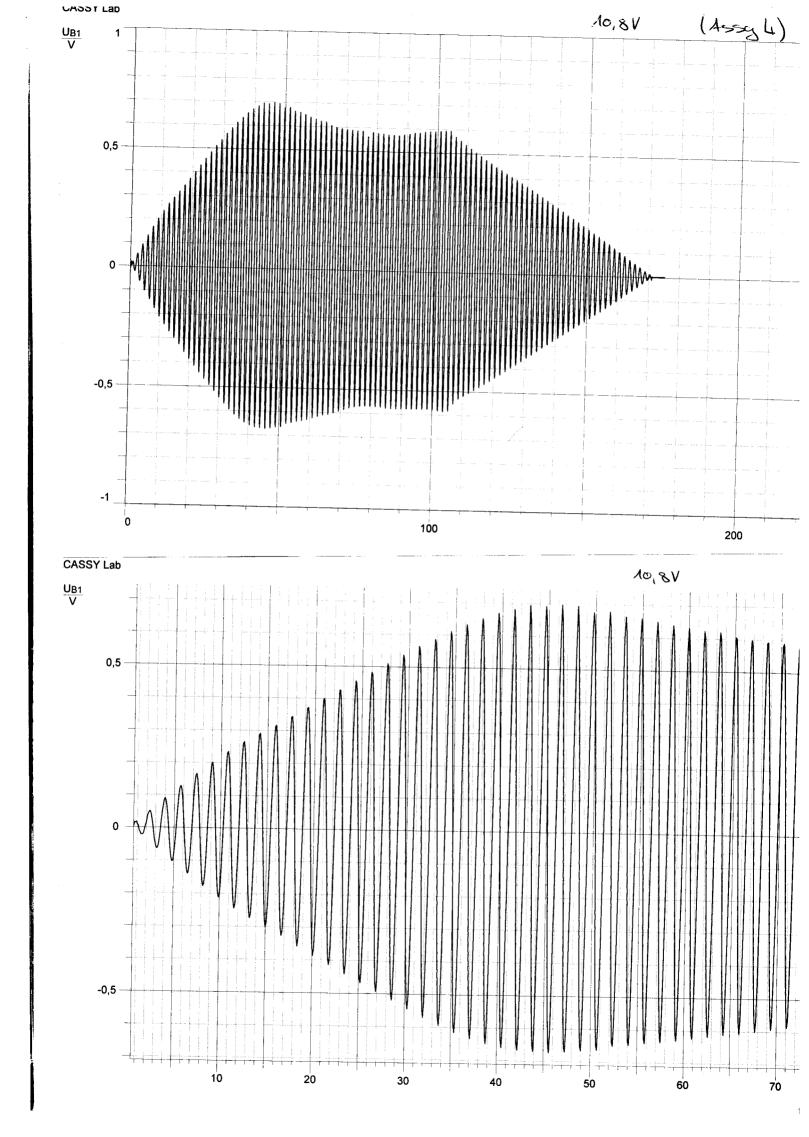
The imdamped oscillator at the bacginning (in the first experiment)

At the resonance frequency, the amplitude directly goes to its maximum and stays there (almost without reaching a higher volume first)

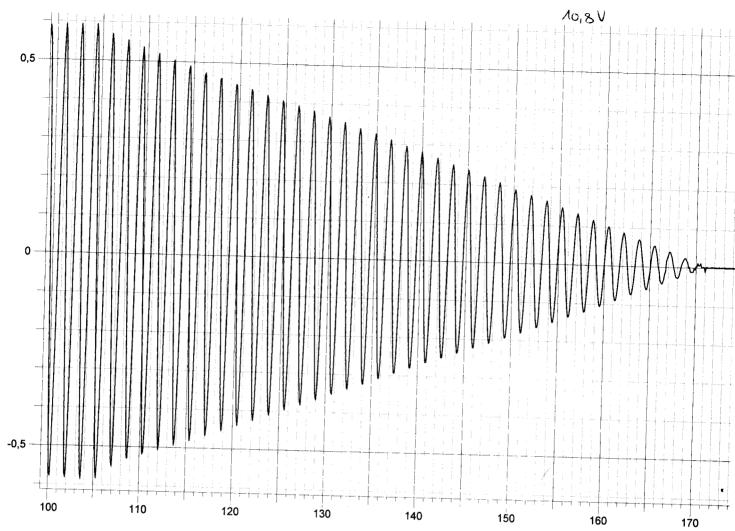
At a frequency above resonance, we clearly see the characteristic boots. The amplitude also is much lower, as we are further away from the resonance



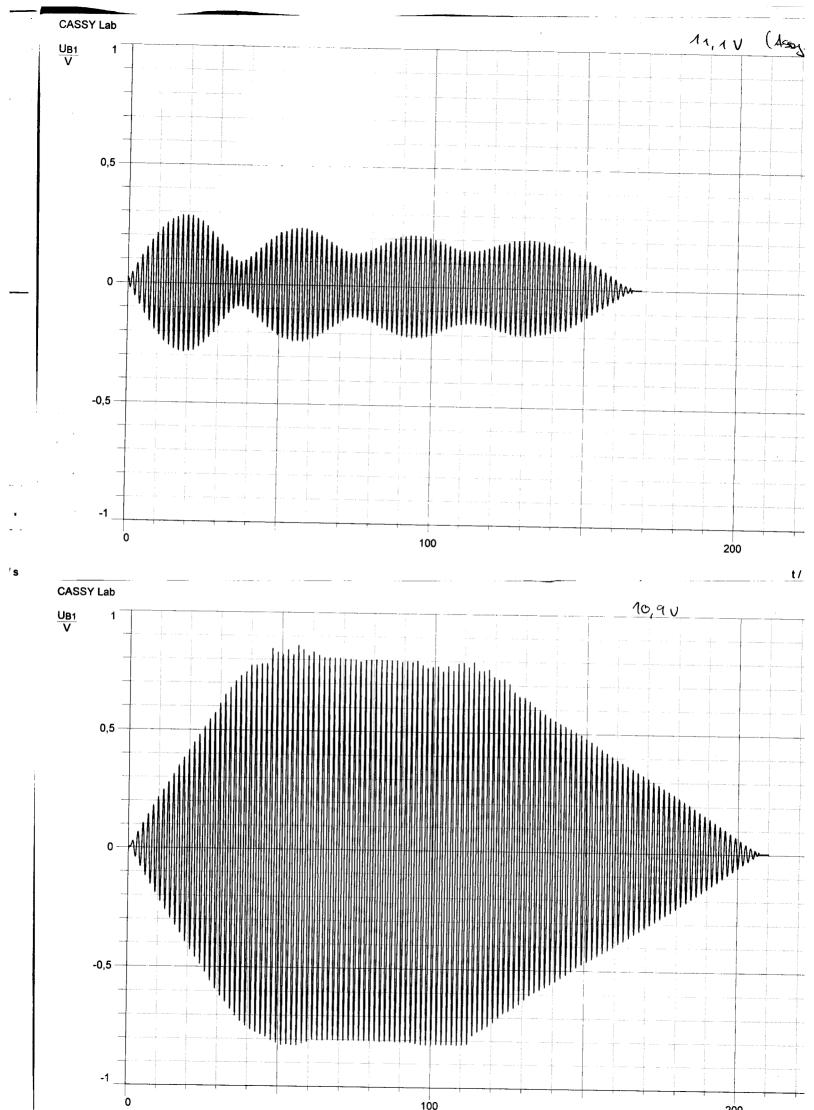








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Con clusion

Assignment 1 allowed to measure the eigen frequency and the decay constant tother accurately. There are no literature values for comparison, but the measured values seem reasonable and are in agreement with each other. The exponential decay of the amplitude that we expected was clearly visible.

Assignment 2 gave a resonance curve as exspected, but was a failure for quantitative values, as we already discussed in the analysis. The vaccuracy of the motor and the impossibility to set the frequency directly did not allow any use tul measurement beyond qualitative examination

Assignment 3 vas in dull correspondance vit our exspectation.

Assignment & also qualitatively behaved according to the Heory. Different effect during the initial phase were clearly visible

6,00 28.5.4;