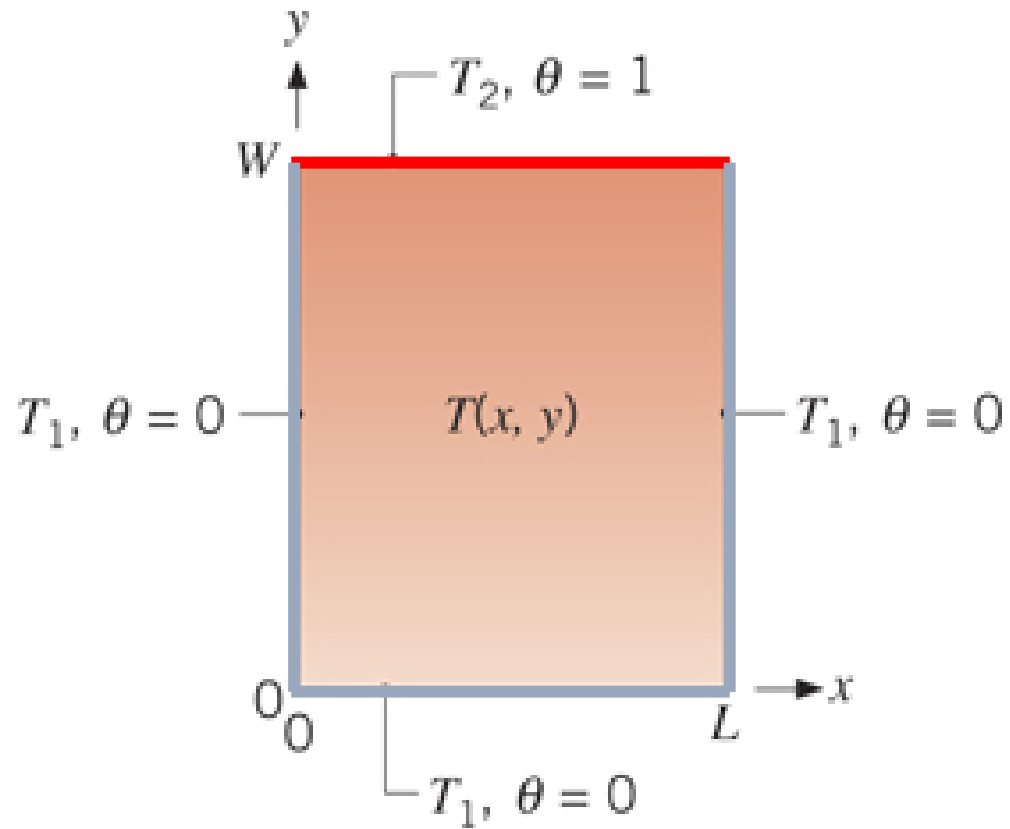


Heat transfer

Chap. 4 two-dimensional conduction





$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} = 0$$

$$\theta(x, y) = \frac{T(x, y) - T_1}{T_2 - T_1}$$

Dimensionless

$$\frac{\partial^2 \theta}{\partial x^2} + \frac{\partial^2 \theta}{\partial y^2} = 0$$

$$\frac{T(x, y) - T_1}{T_2 - T_1} = \theta(x, y) = \frac{2}{\pi} \sum_{n=1}^{\infty} \frac{(-1)^{n+1} + 1}{n} \sin \frac{n\pi x}{L} \frac{\sinh n\pi y/L}{\sinh n\pi W/L}$$

$$q''_x = -k \frac{\partial T}{\partial x}$$

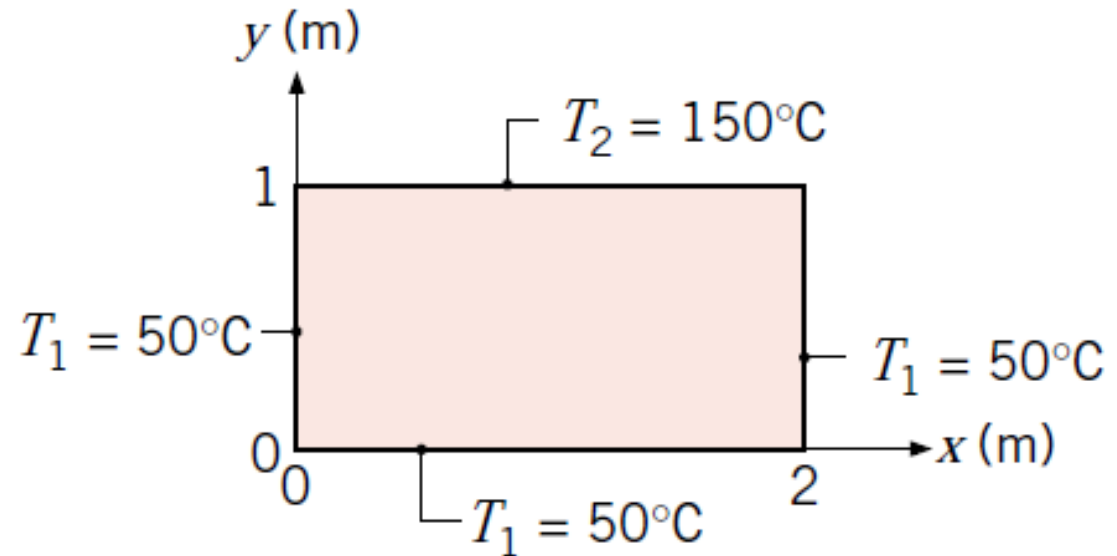
$$q''_y = -k \frac{\partial T}{\partial y}$$

$$\Rightarrow q = Sk\Delta T_{1-2}$$

S shape factor [m]

Eksempel

- A two-dimensional rectangular plate is subjected to
- prescribed boundary conditions. Using the results of
- the exact solution for the heat equation presented in
- Section 4.2, calculate the temperature at the midpoint
- $(1, 0.5)$ by considering the first five nonzero terms of
- the infinite series that must be evaluated.



Two-dimensional
steady-state conduction

Shape factor

$$q = Sk\Delta T_{1-2}$$

Resistance $R_{t,cond} = \frac{\Delta T_{1-2}}{q} = \frac{1}{Sk}$

TABLE 4.1 Conduction shape factors and dimensionless conduction heat rates for selected systems.

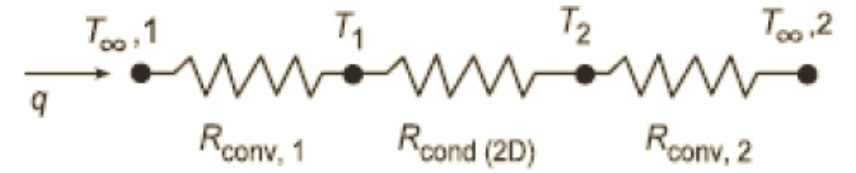
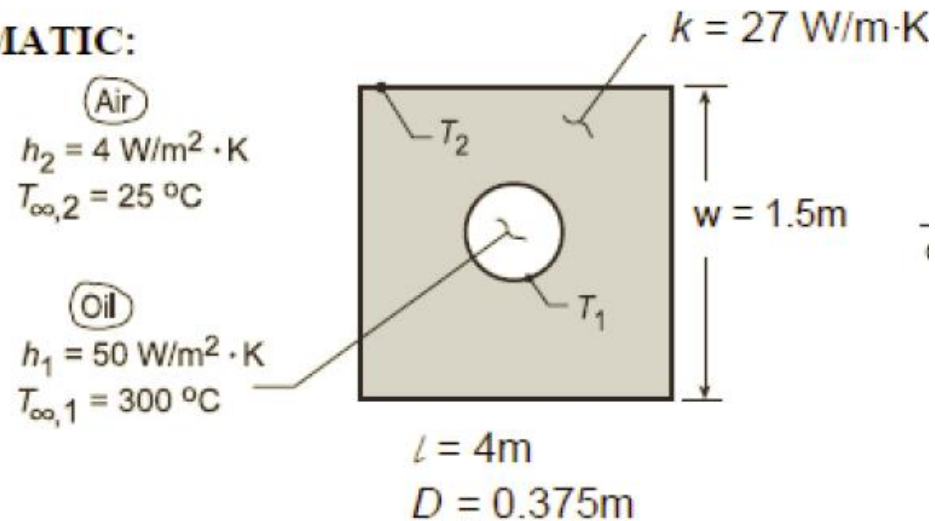
(a) Shape factors [$q = Sk(T_1 - T_2)$]

System	Schematic	Restrictions	Shape Factor
Case 1 Isothermal sphere buried in a semi-infinite medium		$z > D/2$	$\frac{2\pi D}{1 - D/4z}$
Case 2 Horizontal isothermal cylinder of length L buried in a semi-infinite medium		$L \gg D$ $L \gg D$ $z > 3D/2$	$\frac{2\pi L}{\cosh^{-1}(2z/D)}$ $\frac{2\pi L}{\ln(4z/D)}$
Case 3 Vertical cylinder in a semi-infinite medium		$L \gg D$	$\frac{2\pi L}{\ln(4L/D)}$
Case 4 Conduction between two cylinders of length L in infinite medium		$L \gg D_1, D_2$ $L \gg w$	$\frac{2\pi L}{\cosh^{-1}\left(\frac{4w^2 - D_1^2 - D_2^2}{2D_1D_2}\right)}$

TABLE 4.1 Continued

System	Schematic	Restrictions	Shape Factor
Case 5 Horizontal circular cylinder of length L midway between parallel planes of equal length and infinite width		$z \gg D/2$ $L \gg z$	$\frac{2\pi L}{\ln(8z/\pi D)}$
Case 6 Circular cylinder of length L centered in a square solid of equal length		$w > D$ $L \gg w$	$\frac{2\pi L}{\ln(1.08 w/D)}$

SCHEMATIC:



$$q = \frac{T_{\infty,1} - T_{\infty,2}}{R_{\text{conv},1} + R_{\text{cond}(2D)} + R_{\text{conv},2}}$$

$$S = \frac{2\pi L}{\ln\left(\frac{1.08w}{D}\right)} = 17.176 \text{ m}$$

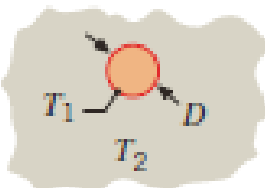
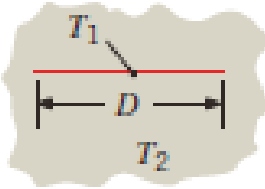
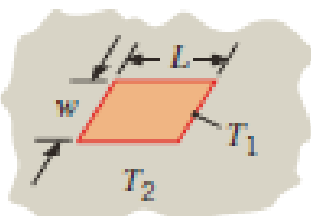
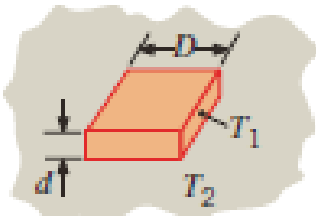
$$R_{2D} = \frac{1}{Sk} = 0.00216 \frac{\text{K}}{\text{W}}$$

Eksempel

A hole of diameter $D = 0.375$ m is drilled through the center of a solid block of square cross section with $w = 1,5$ m on a side. The hole is drilled along the length, $l = 4$ m, of the block, which has a thermal conductivity of $k = 27$ W/m . K. The outer surfaces are exposed to ambient air, with $T_2 = 25^\circ\text{C}$ and $h_2 = 4$ W/m² .K, while hot oil flowing through the hole is characterized by $T_1 = 300^\circ\text{C}$ and $h_1 = 50$ W/m² .K.

Determine the corresponding heat rate and surface temperatures.

(b) Dimensionless conduction heat rate $[q = q_{ss}^* k A_s (T_1 - T_2) / L_c; L_c = (A_s / 4\pi)^{1/2}]$

System	Schematic	Active Area, A_s	q_{ss}^*										
Case 12 Isothermal sphere of diameter D and temperature T_1 in an infinite medium of temperature T_2		πD^2	1										
Case 13 Infinitely thin, isothermal disk of diameter D and temperature T_1 in an infinite medium of temperature T_2		$\frac{\pi D^2}{2}$	$\frac{2\sqrt{2}}{\pi} = 0.900$										
Case 14 Infinitely thin rectangle of length L , width w , and temperature T_1 in an infinite medium of temperature T_2		$2wL$	0.932										
Case 15 Cuboid shape of height d with a square footprint of width D and temperature T_1 in an infinite medium of temperature T_2		$2D^2 + 4Dd$	<table><tr><th>d/D</th><th>q_{ss}^*</th></tr><tr><td>0.1</td><td>0.943</td></tr><tr><td>1.0</td><td>0.956</td></tr><tr><td>2.0</td><td>0.961</td></tr><tr><td>10</td><td>1.111</td></tr></table>	d/D	q_{ss}^*	0.1	0.943	1.0	0.956	2.0	0.961	10	1.111
d/D	q_{ss}^*												
0.1	0.943												
1.0	0.956												
2.0	0.961												
10	1.111												

Infinite medium
Uniform surface temperature

$$L_c = \sqrt{\frac{A_s}{4\pi}}$$

$$q_{ss}^* = \frac{q L_c}{k A_s (T_1 - T_2)}$$

Opgave 4.8

Radioactive wastes are temporarily stored in a spherical container, the center of which is buried a distance of 10 m below the earth's surface. The outside diameter of the container is 2 m, and 500 W of heat are released as a result of the radioactive decay process. If the soil surface temperature is 20°C , what is the outside surface temperature of the container under steady-state conditions? On a sketch of the soil–container system drawn to scale, show representative isotherms and heat flow lines in the soil.



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