

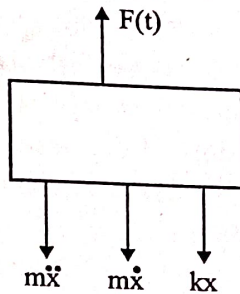
$$\omega_d > \omega_n$$

$$\frac{(1)}{(2)} = \frac{\omega_p}{\omega_d} = \frac{\sqrt{1-2\zeta^2}}{\sqrt{1-\zeta^2}} < 1$$

$$\Rightarrow \omega_p < \omega_d < \omega_n$$

50. Ans: (c)

Sol: FBD of mass



$$m\ddot{x} + c\dot{x} + kx = F(t)$$

Solution of differential equation

$$x(t) = (C.F) + (P.I)$$

Considering condition (P)

$$\text{If } c > 0 \text{ and } \omega = \sqrt{k/m}$$

For this condition the displacement (x) is given by

[Assume the system to be under damped]

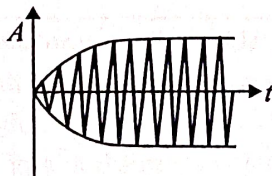
$$x(t) = X_0 e^{-\zeta \omega_n t} \cos(\omega_d t - \phi) + X \cos(\omega t - \phi)$$

Transient response + steady state response

As $t \rightarrow \infty$ the transient response decays to zero and only steady state response will remain and the amplitude will be maximum because of resonance.

$$x(t) = X \cos(\omega t - \phi)$$

For this condition the response curve will be



Considering condition (Q)

$$c < 0 \text{ and } \omega \neq 0$$

The differential equation becomes

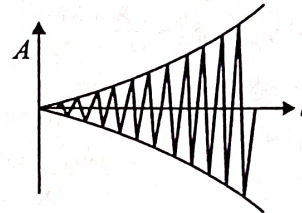
$$m\ddot{x} - c\dot{x} + kx = F(t)$$

Solution of above differential equation is

$$x(t) = X_0 e^{+\zeta \omega_n t} \cos(\omega_d t - \phi) + X \cos(\omega t - \phi)$$

As $t \rightarrow \infty$ the transient response approaches to ∞ and increases exponentially

The plot will be



Considering condition (R)

$$c = 0, \quad \omega = \sqrt{\frac{k}{m}} \quad (\text{Resonance})$$

$$\text{If } c = 0, \quad \xi = 0$$

$$x(t) = X \sin(\omega_n t + \phi) + \frac{X_{\text{static}} \cos(\omega t + \phi)}{\left[\left(1 - \left(\frac{\omega}{\omega_n} \right)^2 \right)^2 + \left(2\xi \frac{\omega}{\omega_n} \right)^2 \right]^{0.5}}$$

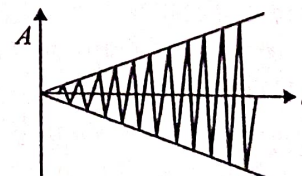
If $\omega = \omega_n$ this term will $\frac{0}{0}$.

Applying L Hospital rule,

$$\begin{aligned} x(t) &= X \sin(\omega_n t + \phi) + \frac{X_{\text{static}} \frac{d}{d\omega} (\cos(\omega t + \phi))}{\frac{d}{d\omega} \left(1 - \left(\frac{\omega}{\omega_n} \right)^2 \right)} \\ &= X \sin(\omega_n t + \phi) + \frac{X_{\text{static}} (-\sin(\omega t + \phi)) t}{\left(0 - \left(\frac{2\omega}{\omega_n^2} \right) \right)} \\ &= X \sin(\omega_n t + \phi) + \frac{X_{\text{static}} t (\sin(\omega t + \phi))}{\left(2 \times \frac{\omega_n}{\omega_n^2} \right)} \\ &= X \sin(\omega_n t + \phi) + \frac{X_{\text{static}} \omega_n t \sin(\omega t + \phi)}{2} \end{aligned}$$

Response will be linearly increasing with t.

So the correct plot will be



Considering condition (S)

$$c = 0, \quad \omega \cong \sqrt{\frac{k}{m}}, \Rightarrow c = 0, \quad \xi = 0$$

$$x(t) = X \sin(\omega_n t + \phi) + \frac{x_{static} \sin(\omega t - \phi)}{[(1 - r^2)^2 + (2\xi r)^2]^{0.5}}$$

$$x(t) = X \sin(\omega_n t + \phi) + \frac{x_{static} \sin(\omega t - \phi)}{\frac{(\omega_n^2 - \omega^2)}{\omega_n^2}}$$

$$x(t) = X \sin(\omega_n t + \phi) + \frac{x_{static} \omega_n^2 \sin(\omega t - \phi)}{\omega_n^2 - \omega^2}$$

$$x(t) = X \sin(\omega_n t + \phi) + \frac{F \sin(\omega t - \phi)}{m (\omega_n^2 - \omega^2)}$$

$$x(t) = X \sin(\omega_n t + \phi) - \frac{F \sin(\omega t - \phi)}{m (\omega^2 - \omega_n^2)}$$

If the force frequency is close to, but not exactly equal to, natural frequency of the system, a phenomenon is known as beating. In this kind of vibration the amplitude builds up and then diminishes in a regular pattern.

The correct plot will be

