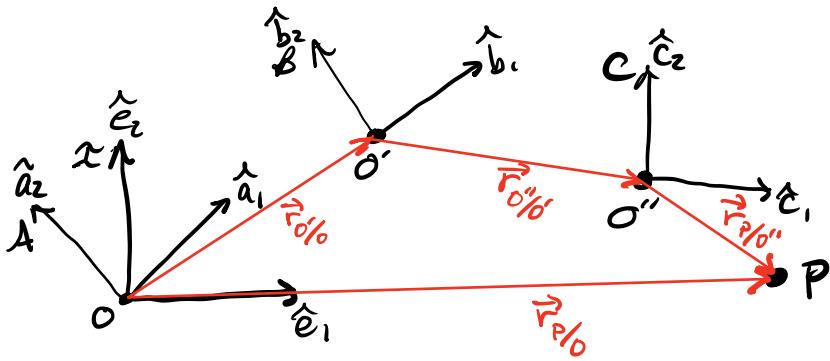


8.2.2 Addition of angular velocity vectors



Compare to Ex 2.5, which is a three-link robotic manipulator.
Note: the picture drawn here is setup more generally though.

$$\vec{v}_{P/0} = \vec{v}_{0''/0} + \vec{v}_{P/0''}$$

$$\vec{v}_{P/0} = \vec{v}_{0/0} + \vec{v}_{0''/0'} + \vec{v}_{P/0''}$$

Apply T.E. to put in terms of frames A, B, and C.

$$\underbrace{\frac{d}{dt}(\vec{v}_{P/0})}_{\vec{v}_{P/0}} = \underbrace{\frac{d}{dt}(\vec{v}_{0/0})}_{\vec{v}_{0/0}} + \overset{B}{v}_{0''/0'} + \overset{B}{\omega} \times \vec{r}_{0''/0'} + \overset{C}{v}_{P/0''} + \overset{C}{\omega} \times \vec{r}_{P/0''}$$

Apply T.E. again

$$\overset{A}{v}_{0/0} + \overset{A}{\omega} \times \vec{r}_{0/0}$$

Note that the angular velocities at all directly referenced to I.

(*)

$$\vec{v}_{P/0} = \overset{A}{v}_{0/0} + \overset{A}{\omega} \times \vec{r}_{0/0} + \overset{B}{v}_{0''/0'} + \overset{B}{\omega} \times \vec{r}_{0''/0'} + \overset{C}{v}_{P/0''} + \overset{C}{\omega} \times \vec{r}_{P/0''}$$

Now rewrite these kinematics to express the intermediate angular velocities (e.g. $\overset{A}{\omega} \times \vec{r}_{0/0}$) (See 3rd-5th eqns on Pg 308)

$$\vec{v}_{P/0} = \frac{d}{dt}(\vec{v}_{0/0}) + \frac{d}{dt}(\vec{v}_{0''/0'}) + \frac{d}{dt}(\vec{v}_{P/0''})$$

We previously applied
T.E. referenced to I for
each term

Instead, use intermediate frames for reference.

$$\frac{^I d}{dt}(\quad) = \frac{^A d}{dt}(\quad) + \overset{I \rightarrow A}{\omega} \times (\quad)$$

Then

$$\frac{^A d}{dt}(\quad) = \frac{^B d}{dt}(\quad) + \overset{A \rightarrow B}{\omega} \times (\quad)$$

Following that process leads to,

$$\begin{aligned} \overset{I \rightarrow}{V}_{P/0} &= \overset{A}{V}_{0/0} + \overset{I \rightarrow A}{\omega} \times \vec{r}_{0/0} + \overset{B}{V}_{0''/0'} + \overset{A \rightarrow B}{\omega} \times \vec{r}_{0''/0'} + \overset{I \rightarrow A}{\omega} \times \vec{r}_{0''/0'} \\ &\quad + \overset{C}{V}_{P/0''} + \overset{B \rightarrow C}{\omega} \times \vec{r}_{P/0''} + \overset{A \rightarrow B}{\omega} \times \vec{r}_{P/0''} + \overset{I \rightarrow A}{\omega} \times \vec{r}_{P/0''} \end{aligned}$$

Regrouping and using the distributive property of cross product,

$$\begin{aligned} \overset{I \rightarrow}{V}_{P/0} &= \overset{A}{V}_{0/0} + \overset{I \rightarrow A}{\omega} \times \vec{r}_{0/0} + \overset{B}{V}_{0''/0'} + (\overset{I \rightarrow A}{\omega} + \overset{A \rightarrow B}{\omega}) \times \vec{r}_{0''/0'} \\ &\quad + \overset{C}{V}_{P/0''} + (\overset{I \rightarrow A}{\omega} + \overset{A \rightarrow B}{\omega} + \overset{B \rightarrow C}{\omega}) \times \vec{r}_{P/0''} \quad (***) \end{aligned}$$

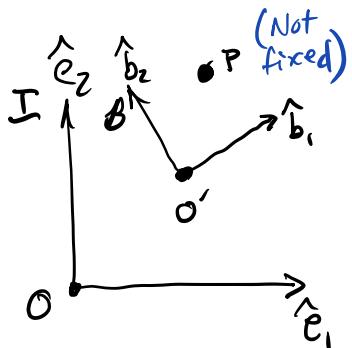
Now compare (*) and (***), equations and equate terms

to see $\overset{I \rightarrow}{\omega} B = \overset{I \rightarrow}{\omega} A + \overset{A \rightarrow}{\omega} B$

$$\overset{I \rightarrow}{\omega} C = \overset{I \rightarrow}{\omega} A + \overset{A \rightarrow}{\omega} B + \overset{B \rightarrow}{\omega} C$$

This shows that the angular velocity vectors add for frames in relative motion.

8.3 Planar Kinetics in a rotating frame



$${}^I \vec{v}_{P/0} = {}^I \vec{v}_{0/0} + {}^B \vec{v}_{P/B} + {}^I \vec{\omega} \times \vec{r}_{P/0}$$

Differentiating,

$${}^I \vec{a}_{P/0} = \frac{d}{dt}({}^I \vec{v}_{0/0}) + \frac{d}{dt}({}^B \vec{v}_{P/B} + {}^I \vec{\omega} \times \vec{r}_{P/0})$$

Using T.E.

$$= {}^I \vec{a}_{0/0} + \frac{d}{dt}({}^B \vec{v}_{P/B} + {}^I \vec{\omega} \times \vec{r}_{P/0}) + {}^I \vec{\omega} \times ({}^B \vec{v}_{P/B} + {}^I \vec{\omega} \times \vec{r}_{P/0})$$

Distribute the cross product and define

$$\text{angular acceleration } {}^I \vec{\alpha} \triangleq \frac{d}{dt}({}^I \vec{\omega}^B) = \frac{d}{dt}({}^I \vec{\omega}^B) + \underbrace{{}^I \vec{\omega}^B \times {}^I \vec{\omega}^B}_{=0}$$

Then we get the general vector expression,

$${}^I \vec{a}_{P/0} = {}^I \vec{a}_{0/0} + {}^B \vec{a}_{P/B} + \frac{d}{dt}({}^B \vec{v}_{P/B} + {}^I \vec{\omega} \times \vec{r}_{P/0}) + 2 {}^I \vec{\omega} \times {}^I \vec{v}_{P/0} + {}^I \vec{\omega} \times ({}^I \vec{\omega} \times \vec{r}_{P/0})$$

Note: You can use this formula or just follow the process of deriving the inertial acceleration in coordinates (like you have been).

$\underbrace{\quad\quad\quad}_{\text{Coriolis Acceleration}}$ $\underbrace{\quad\quad\quad}_{\text{centripetal acceleration}}$

Apparent Forces

The above frame acceleration terms give rise to the apparent force terms if you simply move them to the other side of the equation (NL).

↳ Note: we don't include Apparent Forces in FBD's.

Ex. Carousel

↳ The ball is thrown subject to no horizontal forces (ignoring drag)

$$\vec{F}_p = 0 \Rightarrow \vec{a}_{p0}^I = 0$$

⇒ The ball goes straight
(Although it doesn't look like it does in the rotating frame of the carousel)

What does the carousel rider see?

Set $\vec{a}_{p0}^B = 0$, Solve \vec{a}_{p0}^B

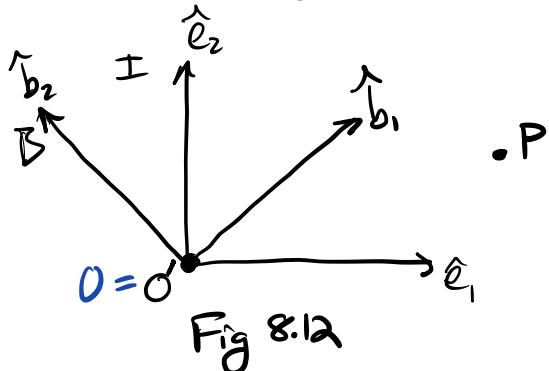
re-arranging,

$$\vec{a}_{p0}^B = -2\vec{\omega}^B \times \vec{v}_{p0}^B - \vec{\omega}^B \times (\vec{\omega}^B \times \vec{r}_{p0}^B) - \underbrace{\vec{\omega}^B \times \vec{r}_{p0}^B}_{\text{Zero in this example}} - \vec{a}_{00}^B$$

Zero in this example

Ex. Merry-go-round

What apparent force is req'd to keep a point P stationary in the frame of a rotating observer?



$$\text{We want } {}^B\vec{\alpha}_{P/I'} = 0$$

Since B is not accelerating in translation or rotation,

$${}^I\vec{a}_0 = 0 \quad {}^I\vec{\omega} = 0$$

Since P is fixed in B ,

$${}^B\vec{r}_{P/B} = {}^B\vec{\alpha}_{P/I'} = 0$$

Then, we have,

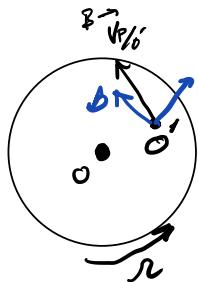
$$m_p {}^B\vec{\alpha}_{P/I'} = \vec{F}_p - m_p {}^I\vec{a}_0 - m_p {}^I\vec{\omega} \times \vec{r}_{P/B} - 2m_p {}^I\vec{\omega} \times {}^B\vec{\omega} \times \vec{r}_{P/B} - m_p {}^I\vec{\omega} \times ({}^I\vec{\omega} \times \vec{r}_{P/B})$$

Then,

$$\vec{F}_p = m_p {}^I\vec{\omega} \times ({}^I\vec{\omega} \times \vec{r}_{P/B})$$

Qualitatively examining acceleration terms

Ex. Carousel - In what direction does the ball appear to accelerate (based on Coriolis contribution)

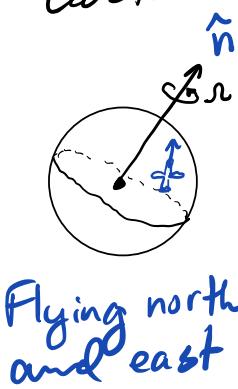


From ${}^B\vec{\alpha}_{P/B}$, the Coriolis term is,

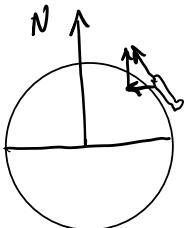
$-2 {}^I\vec{\omega} \times {}^B\vec{v}_{P/B} =$ appears to accelerate to the right in the frame of the rider.

Ex. Coriolis for an airplane for an observer on earth.

$$-2\vec{\omega} \times \vec{v}_{\text{rel}}$$



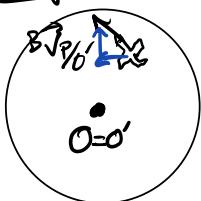
Side View



(Breaking into components)

Notice that the component of velocity parallel to N direction does not contribute to the cross product.

Top View



$$-2\vec{\omega} \times \vec{v}_{\text{rel}}$$

= to the right of the plane's velocity vector
 \vec{v}_{rel}

Ch 9

Dynamics of a planar, rigid body

Many concepts we have already covered apply or can be extended to rigid bodies in 2D or 3D.

Rigid Body Dynamics

A rigid body is a collection of particles with the requirement that they are all fixed relative to one another.

That is, $\frac{d}{dt} \|\vec{r}_{ij}\| = 0 \quad \forall (i,j) \text{ pairs}$

for all

This provides enough constraints to reduce the total degrees of freedom of the multiparticle system so that it behaves as a single object.

Planar R.B. = 3 DOF
3D R.B. = 6 DOF

Euler's Law's

$$\textcircled{1} \quad \vec{F}_G = m_G \vec{a}_G / \textcircled{1}$$

$$\textcircled{2} \quad \frac{d}{dt} (\vec{I}_{h_0}) = \vec{M}_0$$

Note: The internal forces between particles cancel due to N3L, so we don't need to use an ^{ext} superscript!

We can use the separation of angular momentum,

$$\vec{I}_{h_0} = \vec{I}_{h_{ext}} + \vec{I}_{h_G}$$

This leads to two additional expressions for Euler's 2nd law:

$$\frac{^I}{dt}(\vec{h}_{G/B}) = \vec{M}_{G/B}$$

$$\rightarrow \frac{^I}{dt}(\vec{h}_G) = \vec{M}_G$$

The last of these three is often the most useful, because working in the body frame is often easiest for analyzing angular momentum of a body and the total moment acting on the body.

To do this, apply the T.E.

$$\frac{^B}{dt}(\vec{h}_G) + ^I\omega^B \times \vec{h}_G = \vec{M}_G$$

Euler's
Equation

It would be convenient to write $\frac{^I}{dt}(\vec{h}_G)$ in terms of angular velocity since we often measure angular rates in the Body-frame.

Think of this:

translational N2L

$$\frac{^I}{dt}(\vec{P}_{G/B}) = m_G \frac{^I}{dt}(\vec{V}_{G/B})$$

inertial term

Rotational case:

$$\frac{^I}{dt}(\vec{h}_G) = (?) \frac{^I}{dt}(\vec{\omega}^B)$$

Can we do this?

Moment of Inertia

Goal: We would like to write $\vec{I}\vec{h}_G$ in terms of $\vec{I}\vec{\omega}^B$

Angular momentum of the C.O.M. of a multiparticle system,

$$\vec{I}\vec{h}_G = \sum_{i=1}^N \vec{r}_{i/G} \times m_i \vec{v}_{i/G}$$

Apply Transport equation to reveal $\vec{I}\vec{\omega}^B$

$$\vec{I}\vec{h}_G = \sum_{i=1}^N \vec{r}_{i/G} \times m_i \left(\frac{d}{dt} \vec{r}_{i/G} + \vec{I}\vec{\omega}^B \times \vec{r}_{i/G} \right)$$

Apply a vector triple product identity,

$$\vec{I}\vec{h}_G = \sum_{i=1}^N m_i \left[(\vec{r}_{i/G} \cdot \vec{r}_{i/G}) \vec{I}\vec{\omega}^B - \vec{r}_{i/G} (\vec{r}_{i/G} \cdot \vec{I}\vec{\omega}^B) \right]$$

In tensor notation

$$(\vec{r}_{i/G} \cdot \vec{r}_{i/G}) U \circ \vec{I}\vec{\omega}^B$$

↑
Unity
Tensor

Tensor
operation

$$(\vec{r}_{i/G} \otimes \vec{r}_{i/G}) \circ \vec{I}\vec{\omega}^B$$

↑
tensor
product

$$\vec{r}_{i/G} (\vec{r}_{i/G}^T \vec{I}\vec{\omega}^B) = (\vec{r}_{i/G} \vec{r}_{i/G}^T) \vec{I}\vec{\omega}^B$$

outer
product
or
Tensor
product

Factor out $\vec{I}\vec{\omega}^B$,

$$\vec{I}\vec{h}_G = \sum_{i=1}^N m_i [(\vec{r}_{i/G} \cdot \vec{r}_{i/G}) U - (\vec{r}_{i/G} \otimes \vec{r}_{i/G})] \circ \vec{I}\vec{\omega}^B$$

\triangleq Moment of Inertia Tensor $\vec{I}\vec{I}_G$

Finally,

$$\overset{\text{I}}{\vec{h}_G} = \overset{\text{I}}{\mathbb{I}_G} \circ \overset{\text{I}}{\vec{\omega}^B}$$

New compact way to write the angular momentum about G. This is a vector-tensor equation, so it is coordinate independent.

In body-frame coordinates,

$$[\overset{\text{I}}{\vec{h}_G}]_B = [\overset{\text{I}}{\mathbb{I}_G}]_B [\overset{\text{I}}{\vec{\omega}^B}]_B$$

↑
Moment of
inertia matrix

For a continuous rigid body B,

$$\overset{\text{I}}{\mathbb{I}_G} = \int_B [(\vec{r}_{dm/G} \cdot \vec{r}_{dm/G}) I - (\vec{r}_{dm/B} \otimes \vec{r}_{dm/G})] dm$$

where we have passed from

particle \bullet $\curvearrowright dm$
differential mass element

In body frame,

$$\overset{\text{I}}{\mathbb{I}_G} = \sum_{i=1}^3 \sum_{j=1}^3 I_{ij} \vec{b}_i \otimes \vec{b}_j$$

E.g.
 $\vec{b}_1 \otimes \vec{b}_2 = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$

To express in coordinates in the B frame,

$$[\mathbb{I}_G]_B = \int_B \left(\|\vec{r}_{dm/G}\|^2 I - [\vec{r}_{dm/G}]_B [\vec{r}_{dm/G}]_B^T \right) \rho dV$$

$\frac{dm}{dm}$

In cartesian coordinates,

$$I_{11} = \int_B (y^2 + z^2) dm \quad I_{12} = I_{21} = - \int_B xy dm$$

$$I_{22} = \int_B (x^2 + z^2) dm$$

$$[\mathbb{I}_G]_B = \begin{bmatrix} I_{11} & I_{12} & I_{13} \\ I_{21} & I_{22} & I_{23} \\ I_{31} & I_{32} & I_{33} \end{bmatrix}$$

with $[\mathbb{I}_G]_B = [\mathbb{I}_G]^T_B$
real symmetric matrix
"product of inertia"

Properties of Moment of Inertia Matrices

- ① All the e-values are real
- ② N linearly independent e-vectors \rightarrow Full rank (invertible)
- ③ (Normalized) e-vectors form an orthonormal basis.

For $N=3$, the normalized e-vectors represent the unit vectors of a special body frame known as the Principal Axes Frame. The principal axes are often easy to find if there is symmetry.

Suppose B is an arbitrary body frame.
Let C be the principal axes frame.

$$[\mathbb{I}_G]_C = ({}^C R^B) [\mathbb{I}_G]_B ({}^C R^B)^T \quad \text{Change of Basis}$$

$\begin{bmatrix} I & 0 & 0 \\ 0 & I_2 & 0 \\ 0 & 0 & I_3 \end{bmatrix}_C$

This matrix has been diagonalized!

- The columns of ${}^B R^C = ({}^C R^B)^T$ are the of $[\mathbb{I}_G]_B$ (principal axes)
- The diagonal entries of $[\mathbb{I}_G]_C$ are eigenvalues of $[\mathbb{I}_G]_B$ (the principal moments of inertia)