

## Part II of course

Methods of Analytical Mechanics  
 Goldstein, Classical Mechanics, 3rd Ed.  
 Hand & Finch, Analytical Mechanics

- Newtonian formulation can be thought of as "vectorial mechanics"
- Although Newtonian approaches sometimes look at aggregated quantities (e.g. COM), bodies are usually treated separately and linked by constraint forces, which are not always of interest.
- We will focus on two scalar properties of motion
  - ↳ kinetic and potential energies, instead of forces to analyze the motion.

### Constraint Types (Ch 1 Goldstein)

Assume that our system has  $q_1, \dots, q_{N_c}$  generalized coordinates

Holonomic	$f(q_1, \dots, q_{N_c}, t) = 0$	$f(q_1, \dots, q_{N_c}) = 0$
Non-holonomic		
	INEQUALITIES	
Semi-holonomic	$f(q_1, \dots, q_{N_c}, \dot{q}_1, \dots, \dot{q}_{N_c}, t) = 0$	$f(q_1, \dots, q_{N_c}, \dot{q}_1, \dots, \dot{q}_{N_c}) \neq 0$
Time-dependent "Rheonomous"		
Time-independent "Scleronomous"		

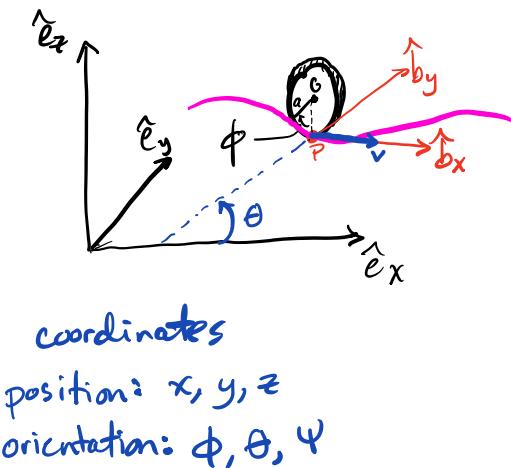
"Holonomic" constraints  $f(\vec{r}_{10}, \vec{r}_{20}, \dots, \vec{r}_{N0}, t) = 0$

Ex. Rigid body  $\|\vec{r}_{ij}\|^2 - c_{ij} = 0 \quad \forall i, j$

Other constraints are "Non-holonomic" if you cannot write simple equations connecting the #DOF with the location of all parts of the system.

Warning: Some constraints contain time rates of change and appear to be non-holonomic at first, but if they are integrable, then they are actually holonomic. Non-holonomic constraints resist integration.

Ex. Non-holonomic constraint: Rolling motion w/o slipping



# DOF

1 Rigid body

$$M = 6(1) - \frac{k_f}{2} - \frac{k_s}{2} = 2 \quad N_c = 4$$

Constraints

1) Upright coin:  $\hat{e}_z \cdot \vec{b}_y = 0 \Rightarrow \psi = 0$

2) In contact with the table:  $\vec{F}_{G0} \cdot \hat{e}_z = a \Rightarrow z = a$

3+4) Cannot move sideways instantaneously  
 $\vec{V}_{G0} \cdot \vec{b}_y = 0$

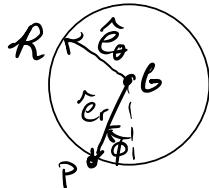
(Leads to two scalar constraints that are non-holonomic)

Aside: Revisit rolling w/o slipping

(See 8.2, 13.3 in K+P)

Rolling w/o slipping  $\overset{I}{\vec{V}}_{P/0} = 0$

$$\begin{aligned}\overset{I}{\vec{V}}_{P/0} &= \overset{I}{\vec{V}}_{G/0} + \overset{I}{\vec{V}}_{P/G} \\ &= 0 \quad \text{by assumption} \\ &= \overset{I}{\vec{V}}_{G/0} + \overset{R}{\vec{Y}}_{P/G} + \overset{I}{\vec{\omega}} \times \overset{R}{\vec{r}}_{P/G} \\ &= 0 \quad (\text{Body is rigid})\end{aligned}$$



$$\overset{O}{\vec{v}} = \overset{I}{\vec{V}}_{G/0} + (\dot{\phi} \hat{e}_3) \times (a \hat{e}_r)$$

$$\Rightarrow \overset{I}{\vec{V}}_{G/0} = -a \dot{\phi} \hat{e}_\theta$$

But  $\overset{I}{\vec{V}}_{G/0} = v \hat{b}_x$  in the  $B$  frame

At the instant  $P$  contacts the ground

$$\boxed{v = a \dot{\phi}}$$

Note: Not a constraint in this problem for the purposes of counting constraints, because  $v$  has been introduced here

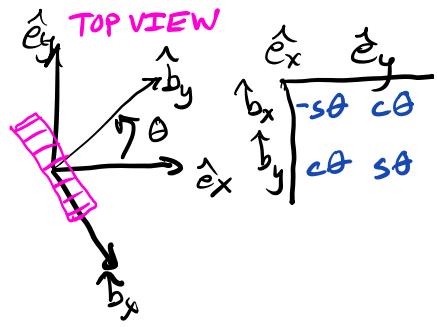
Q: How do we obtain coordinate representations of the non holonomic constraints in the 3D coin problem?

from kinematics:  $\overset{R}{\vec{r}}_{G/0} = x \hat{e}_x + y \hat{e}_y + a \hat{e}_z$

$$\overset{I}{\vec{V}}_{G/0} = \dot{x} \hat{e}_x + \dot{y} \hat{e}_y$$

In the B frame,

$$\begin{aligned}\overset{I}{\vec{v}_{B/0}} &= \dot{x}(-s\theta \hat{b}_x + c\theta \hat{b}_y) \\ &\quad + \dot{y}(c\theta \hat{b}_x + s\theta \hat{b}_y) \\ &= (-\dot{x}s\theta + \dot{y}c\theta)\hat{b}_x + (\dot{x}c\theta + \dot{y}s\theta)\hat{b}_y\end{aligned}$$



Then,

$$\overset{I}{\vec{v}_{B/0}} \cdot \hat{b}_y = 0 \quad (\text{Recall constraint of no sideways motion})$$

$$\Rightarrow \dot{x}\cos\theta + \dot{y}\sin\theta = 0 \quad \xrightarrow{\text{One form of the constraint, but let's try to isolate } \dot{x} \text{ and } \dot{y}.}$$

Instead, look at  $\overset{I}{\vec{v}_{B/0}} = v \hat{b}_x$

$$\left\{ \begin{array}{l} = -vs\theta \hat{e}_x + vc\theta \hat{e}_y \\ = \dot{x}\hat{e}_x + \dot{y}\hat{e}_y \end{array} \right.$$

$$\Rightarrow \left\{ \begin{array}{l} \dot{x} = -v\sin\theta \\ \dot{y} = v\cos\theta \end{array} \right.$$

Plug in and eliminate  $v$ :

$$\begin{aligned}\dot{x} + a\dot{\phi}\sin\theta &= 0 \\ \dot{y} - a\dot{\phi}\cos\theta &= 0\end{aligned}$$

Eliminating  $d\dot{\phi}$ ,

$$dx + ad\phi \sin\theta = 0$$

$$dy - ad\phi \cos\theta = 0$$

Neither equation can be integrated without solving the entire problem.

You can't integrate to get  $x(\theta, \phi)$  and  $y(\theta, \phi)$

In theory, if you had this, you could plug these in to eliminate  $x$  and  $y$  from the problem.

However the values of  $x$  and  $y$  depend not only on  $\theta$  and  $\phi$ , but on the history of the system  
(Hand and Finch p37)

Question: How can we determine if a constraint is non-holonomic?

Let's look at a necessary condition to be holonomic.

If  $f(x, \theta, \phi) = 0$  is holonomic, then  $df = 0$ ,  
and we can write  
$$df = \frac{\partial f}{\partial x} dx + \frac{\partial f}{\partial \theta} d\theta + \frac{\partial f}{\partial \phi} d\phi = 0$$

Compare this to  $dx + ad\phi \sin \theta = 0$ ,

$$\frac{\partial f}{\partial x} = 1 \quad \frac{\partial f}{\partial \theta} = 0 \quad \frac{\partial f}{\partial \phi} = a \sin \theta$$

Mixed partial derivatives trick:

$$\frac{\partial^2 f}{\partial x \partial \phi} = \frac{\partial^2 f}{\partial \phi \partial x}$$

$$\frac{\partial^2 f}{\partial \theta \partial \phi} = 0 \quad \text{and} \quad \frac{\partial^2 f}{\partial \phi \partial \theta} = a \cos \theta$$

$\neq$

$\Rightarrow$  Non-holonomic

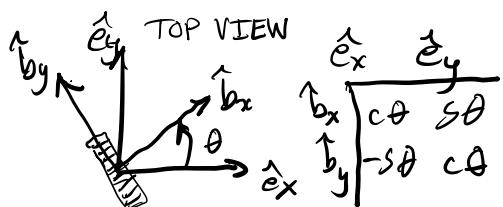
$\Rightarrow f$  that satisfies the form of an exact differential for our constraint doesn't exist.

$$\Rightarrow dx + ad\theta \sin\theta = 0$$

is a non-holonomic constraint.

Aside: The setup for this problem differs between texts and the equations vary slightly based on the reference frames used and definition of the  $\theta$  angle.

Goldstein appears to use:



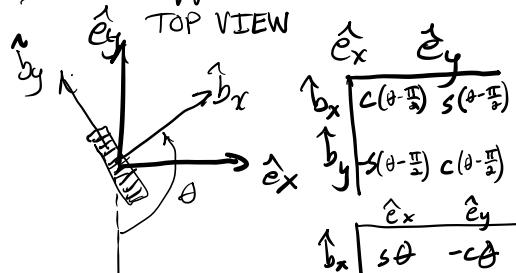
$$v_{C/b} = -v \hat{b}_y$$

$$\Rightarrow \begin{aligned} \dot{x} &= v \sin \theta \\ \dot{y} &= -v \cos \theta \end{aligned}$$

$$dx - a \sin \theta d\phi = 0$$

$$dy + a \cos \theta d\phi = 0$$

H&F appear to use:



$$v_{C/b} = -\hat{b}_y$$

$$\Rightarrow \begin{aligned} \dot{x} &= -v \cos \theta \\ \dot{y} &= -v \sin \theta \end{aligned}$$

$$dx + a \cos \theta d\phi = 0$$

$$dy + a \sin \theta d\phi = 0$$

- These reference frames are not clearly defined in the text, but these interpretations appear to match.

- The convention isn't very important, because it does not impact the non-holonomy of the kinematic constraints.

## Revisiting Degrees of freedom

In the presence of  $K_H$  holonomic constraints, we can introduce a set of  $N_c = 3N - K_H$  independent scalars for the case of particles (or  $N_c = 6N - K_H$  for rigid bodies) that are called generalized coordinates to describe the configuration of our system.

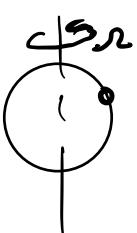
When non-holonomic constraints are present, they reduce the number of degrees of freedom, but we can't make use of them to reduce to configuration space.

$$\left. \begin{array}{l} M = \underset{\substack{\# \text{DOF} \\ \text{3D particles}}}{3N - K_H - K_S} \\ N_c = \underset{\substack{\# \text{of holonomic} \\ \text{constraints}}}{3N - K_H} \end{array} \right\} \underset{\substack{\# \text{of non-holonomic} \\ \text{constraints}}}{}$$

$$\left. \begin{array}{l} M = \underset{\substack{\# \text{DOF} \\ \text{Rigid bodies} \\ \text{in 3D}}}{6N - K_H - K_S} \\ N_c = \underset{\substack{\# \text{of holonomic} \\ \text{constraints}}}{6N - K_H} \end{array} \right\}$$

- Constraints can be further classified as time-dependent (rheonomous) or time-independent (scleronomous)

Ex. bead on a stationary hoop  $\longrightarrow$  time-independent



bead on a moving hoop  
(with prescribed motion)  $\longrightarrow$  time-dependent

bead on a moving wire  
that freely moves in  
response to the bead  $\longrightarrow$  time-independent.

## Difficulties in working with constraints

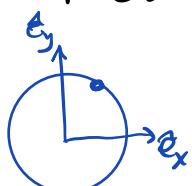
- ① Coordinates may not be independent when constraints are present.
- ② Forces that impose constraints are usually not provided "a priori" and depend upon the system's state  
 ↳ They are among the unknowns of the system.

For holonomic constraints the positions  $\vec{r}_{\text{p0}}$  are not independent, however we can choose independent generalized coordinates to express them.

$$\begin{aligned}\vec{r}_{\text{p0}} &= \vec{r}_{\text{p0}}(q_1, \dots, q_{N_c}, t) \\ &\vdots \\ \vec{r}_{N_c} &= \vec{r}_{N_c}(q_1, \dots, q_{N_c}, t)\end{aligned}$$

Then, the constraint equations are contained in our formulation implicitly.

Ex. Particle in the plane constrained to the unit circle:



$$\vec{r}_{\text{p0}} = x \hat{\mathbf{e}}_x + y \hat{\mathbf{e}}_y$$

with  $x^2 + y^2 = 1$  (holonomic constraint;  
Note that  $x, y$  are dependent)

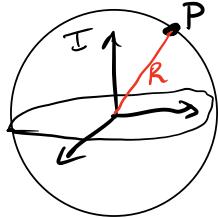
OR

you could use polar coordinate  $\theta$ :

$$\vec{r}_{\text{p0}} = \cos \theta \hat{\mathbf{e}}_x + \sin \theta \hat{\mathbf{e}}_y$$

(constraint is implicit in our choice of coordinates)

Ex Particle constrained on a sphere



$$\vec{r}_P = x \hat{i}_x + y \hat{i}_y + z \hat{i}_z$$

with  $x^2 + y^2 + z^2 = R^2$

OR You could use *latitude* and *longitude* as generalized coordinates. (Note: these angles are independent).

Notes:

- Generalized coordinates do not need to be positional or angular → they can be general.
- If constraints are nonholonomic, they do not help reduce the number of coordinates needed
- Proper choice of coordinates can cause the constraint force to disappear since the constraint will appear implicitly

## D'Alembert's Principle

We want a formulation of mechanics that doesn't require constraint forces to be calculated.

Only applied forces need to be known.

The trick: The constraint forces do no work.

Start with N2L

$$\vec{F}_i = \frac{d}{dt}(\vec{P}_{i0})$$

Rewrite and sum over all particles

$$\sum_{i=1}^N \vec{F}_i - \frac{d}{dt}(\vec{P}_{i0}) = 0$$

Let's consider an infinitesimal displacement  $\vec{s}_{i0}$  of particle  $i$ , which is called a virtual displacement, which is consistent with the instantaneous constraints and forces.



Let's look at the virtual work associated with the virtual displacement,

$$\sum_{i=1}^N \vec{F}_i \cdot \delta \vec{r}_{i/0} \quad \text{virtual work}$$

$\vec{F}_i = \vec{F}_i^{(a)} + \vec{F}_i^{(c)}$   
 ↑ "applied" or "active" forces  
 ↑ constraint forces

Dot both sides of N2L with the virtual displacement to create an expression that contains virtual work.

$$\sum_{i=1}^N \left( \vec{F}_i - \frac{d}{dt}(\vec{P}_{i/0}) \right) \cdot \delta \vec{r}_{i/0} = 0$$

Sub in the force labeling we came up with:

$$\sum_{i=1}^N \left( \vec{F}_i^{(a)} + \vec{F}_i^{(c)} - \frac{d}{dt}(\vec{P}_{i/0}) \right) \cdot \delta \vec{r}_{i/0} = 0$$

$$\sum_{i=1}^N \left( \vec{F}_i^{(a)} - \frac{d}{dt}(\vec{P}_{i/0}) \right) \cdot \delta \vec{r}_{i/0} + \boxed{\sum_{i=1}^N \vec{F}_i^{(c)} \cdot \delta \vec{r}_{i/0}} = 0$$

$= 0$

By assumption, since we designed our virtual displacements to be consistent with the constraints

Virtual displacements for which this holds are called "reversible" (Meirovitch)

$$\sum_{i=1}^n \left( \vec{F}^{(a)} - \frac{d}{dt} (\vec{r}_{i0}) \right) \cdot \delta \vec{r}_{i0} = 0$$

D'Alembert's Principle (Form #1 of 2)

The total virtual work from applied forces and "inertial forces" vanishes for a reversible virtual displacement.

~~Let's rewrite D'Alembert's principle to make it more useful for solving problems~~

Let's rewrite D'Alembert's principle using  $M$  generalized coordinates which are independent of each other.

$$\vec{r}_{i0} = \vec{r}_{i0}(q_1, \dots, q_M, t)$$

Rewrite velocity:

$$\vec{V}_{i0} = \frac{d}{dt} (\vec{r}_{i0}) = \sum_{j=1}^M$$

(Note: that we drop the I superscript on partial derivatives for convenience).

Rewrite virtual displacement:

$$\delta \vec{r}_{i0} = \sum_{j=1}^M \frac{\partial \vec{r}_{i0}}{\partial q_j} \delta q_j$$

Note: there is no variation in time in a virtual displacement

Rewriting virtual work:

$$\sum_{i=1}^N \vec{F}_i^{(a)} \cdot \delta \vec{r}_{i\delta} = \sum_{j=1}^M \sum_{i=1}^N \vec{F}_i^{(a)} \cdot \frac{\partial \vec{r}_{i\delta}}{\partial q_j} \delta q_j$$