

Ch 9 (cont'd)

Work and energy of a rigid body

Recall that kinetic energy separated for a collection of particles:

$$T_O = T_{G/O} + T_G$$

Using the definition of a rigid body (i.e. no relative motion of the internal particles), the transport theorem, and the definition of moment of inertia, T_G can be rewritten as,

$$T_G = \frac{1}{2} I_G (\overset{I}{\vec{\omega}}^B \cdot \overset{I}{\vec{\omega}}^B) = \frac{1}{2} I_G \|\overset{I}{\vec{\omega}}^B\|^2$$

We can also substitute the angular momentum expression to obtain:

$$T_G = \frac{1}{2} \overset{I}{h}_G \cdot \overset{I}{\vec{\omega}}^B$$

For the work-energy formulas, we begin with the multiparticle formulas and impose "no internal relative motion" constraints.

↳ There is no internal work within a rigid body

The total-work-kinetic energy formula still holds for a rigid body:

$$T_G(t_2) = T_G(t_1) + W^{(\text{tot})} \quad (\#1 \text{ of } 3)$$

Since there is no internal work we can just write $W = W^{(\text{tot})}$

The work W separates, just as kinetic energy does

$$W = \underbrace{W_{G/0}}_{\int_{t_1}^{t_2} \vec{F}_G \cdot \overset{I}{\vec{V}_{G/0}} dt} + \underbrace{W_G}_{\int_{t_1}^{t_2} \vec{M}_G \cdot \overset{I}{\vec{\omega}}^B dt}$$

rotational work about com.

It can be shown that

$$\begin{aligned} W_{G/0} &= \Delta T_{G/0} & \text{and} & \quad W_G = \Delta T_G \\ &= T_{G/0}(t_2) - T_{G/0}(t_1) & &= T_G(t_2) - T_G(t_1) \end{aligned}$$

- Notes:
- If the total external force is zero, the translational kinetic energy of G , is **conserved**
 - If the total external moment about G is zero, the **rotational kinetic energy is conserved**

The potential energy for a conservative force acting on the rigid body is,

$$U_o(t) = \int \mathbf{U}_{\text{dyno}}(\vec{r}_{\text{dyno}}) dm$$

Then

$$U_o(t_2) = U_o(t_1) - W_{(t_1, t_2)}^{(c, \text{ext})} \quad (\#2 \text{ of } 3)$$

Note that there is no internal potential energy change and no internal conservative work

Lastly, define total energy,

$$E_o(t) \triangleq T_{G/o}(t) + T_G(t) + U_o(t)$$

Then,

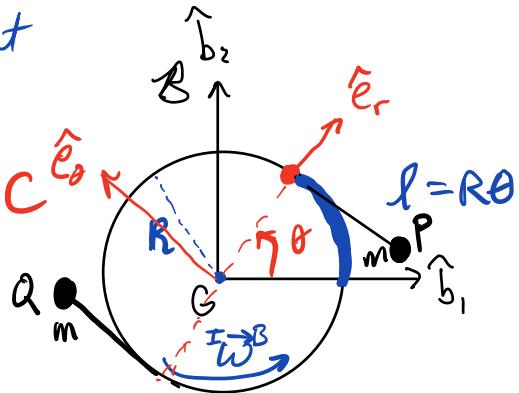
$$E_o(t_2) = E_o(t_1) + W_{G/o}^{(nc)} + W_G^{(nc)}$$

(#3 of 3)

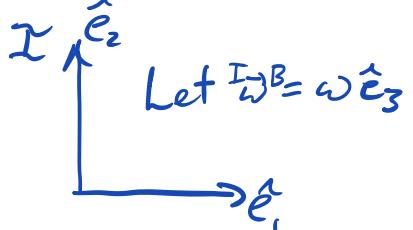
Ex. Yo-Yo De-spin

B: body frame that rotates

C: polar frame that tracks to contact point



$$I \vec{\omega}_{P/G}^{(sys)} = ?$$



Objective is to slow down the spin of a satellite.

No external forces or moments \Rightarrow Total angular momentum is conserved.

$$\vec{P}_{P/G} = R \hat{e}_r - R\theta \hat{e}_\theta$$

$$I \vec{V}_{P/G} = R(\vec{\omega}_C^C \times \hat{e}_r) - R\dot{\theta} \hat{e}_\theta - R\theta (\vec{\omega}_C^C \times \hat{e}_\theta)$$

$$\vec{\omega}_C^C = \vec{\omega}_B^B + \vec{\omega}_B^C = (\omega + \dot{\theta}) \hat{e}_3 \quad (\text{addition of angular velocity})$$

$$I \vec{V}_{P/G} = R\theta(\omega + \dot{\theta}) \hat{e}_r + R \vec{\omega}_B^B \hat{e}_\theta$$

Similarly for Q,

$${}^I \vec{V}_{Q/G} = -R\theta (\omega + \dot{\theta}) \hat{e}_r - R\omega \hat{e}_\theta$$

Total angular momentum is, $\vec{h}_G = {}^{(sys)} \vec{h}_G = {}^{(disk)} \vec{h}_G + {}^{(particle)} \vec{h}_G$

$$\vec{h}_G = I_G \omega \hat{e}_3$$

$\sum_i \vec{r}_{i/G} \times m_i \vec{v}_{i/G}$

$$\begin{aligned} {}^{(sys)} \vec{h}_G &= I_G \omega \hat{e}_3 + (\vec{r}_{P/G} \times m \vec{v}_{P/G}) + (\vec{r}_{Q/G} \times m \vec{v}_{Q/G}) \\ &= (I_G \omega + 2mR^2(\omega + \theta^2(\omega + \dot{\theta}))) \hat{e}_3 \end{aligned}$$

ω and $\dot{\theta}$ are both unknowns. We can use conservation of total energy to get another equation. This leads to a set of equations (differential in $\dot{\theta}$) that can be solved for $\theta(t)$ and $\omega(t)$.

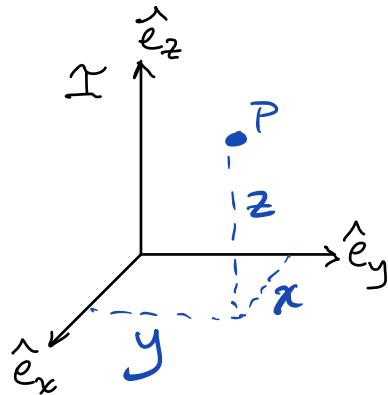
Chapter 10 - Particle Kinematics & Kinetics in 3D

3D coordinate systems

Cartesian $(x, y, z)_I$

$$\vec{r}_{P0} = x \hat{e}_x + y \hat{e}_y + z \hat{e}_z$$

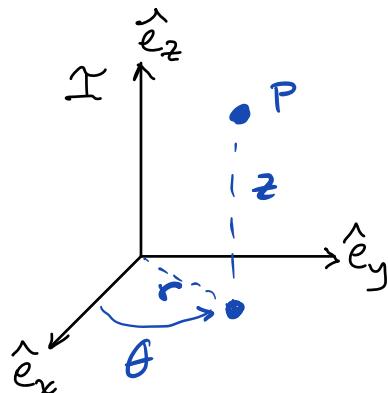
: simple time derivatives



Cylindrical $(r, \theta, z)_I$

$$\vec{r}_{P0} = r \cos \theta \hat{e}_x + r \sin \theta \hat{e}_y + z \hat{e}_z$$

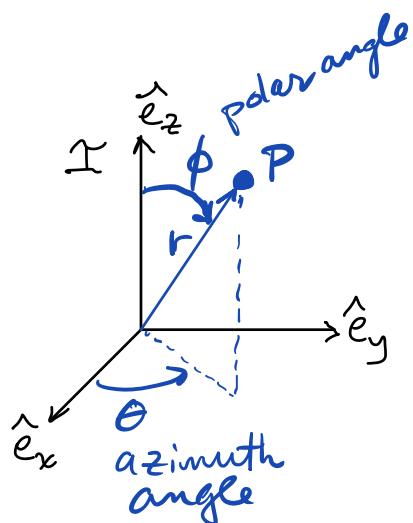
: messy time derivatives



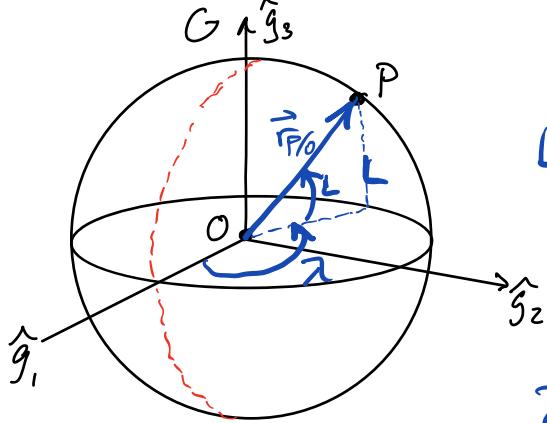
Spherical $(r, \theta, \phi)_I$

$$\begin{aligned} \vec{r}_{P0} = & r \cos \theta \sin \phi \hat{e}_x + r \sin \theta \sin \phi \hat{e}_y \\ & + r \cos \phi \hat{e}_z \end{aligned}$$

: Extremely messy
: time derivatives!



Example: Latitude and Longitude



Latitude is a complement of ϕ

$$L = \frac{\pi}{2} - \phi$$

λ : longitude

Frame $G = (O, \hat{g}_1, \hat{g}_2, \hat{g}_3)$

Note: not inertial

$$\overset{I}{\vec{\omega}}^G = \mathcal{R}_E \hat{g}_3$$

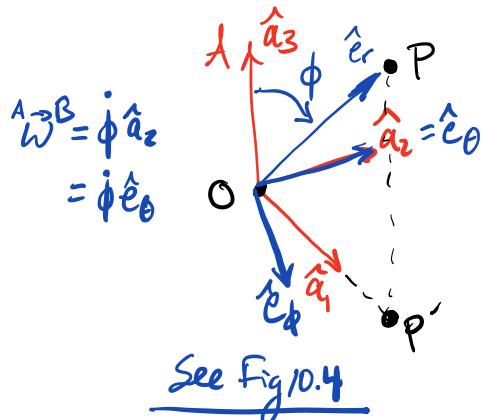
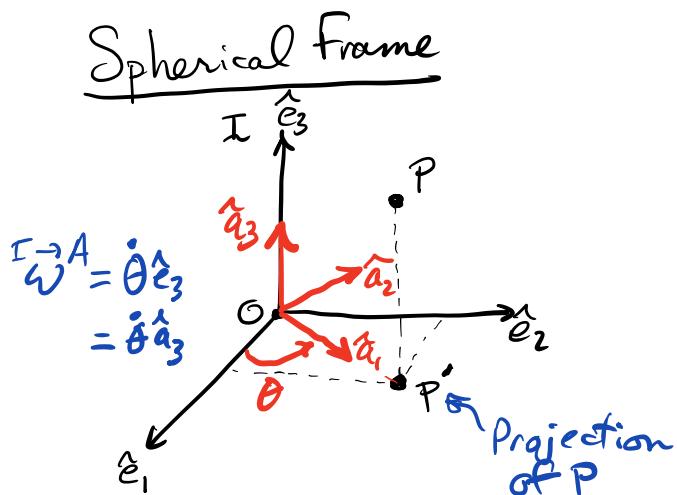
Geographic ref. frame with \hat{g}_1 pointing to the great circle which goes through Greenwich, England.

Cylindrical and Spherical Reference Frames

- These are generalizations of the 2D polar frame
- Make kinematic and dynamic calculations easier.
- angular velocity vector is crucial to kinematic derivations since it is used to differentiate unit vectors of a non-inertial frame.

Cylindrical frame - Just a polar frame that tracks the projection of \vec{r}_{PG} in (\hat{e}_x, \hat{e}_y) plane, in combination with an \hat{e}_z direction vector to complete the 3D reference frame

Spherical frame - A frame that tracks particle P that is constructed using intermediate frames that are related by elementary rotations.



$$A = (O, \hat{a}_1, \hat{a}_2, \hat{a}_3) \quad \text{"Intermediate frame"} \quad {}^I C^A$$

	\hat{a}_1	\hat{a}_2	\hat{a}_3
\hat{e}_1	$c\theta$	$-s\theta$	0
\hat{e}_2	$s\theta$	$c\theta$	0
\hat{e}_3	0	0	1

$$B = (O, \hat{e}_\phi, \hat{e}_\theta, \hat{e}_r)$$

	\hat{e}_ϕ	\hat{e}_θ	\hat{e}_r
\hat{a}_1	$c\phi$	0	$s\phi$
\hat{a}_2	0	1	0
\hat{a}_3	$-s\phi$	0	$c\phi$

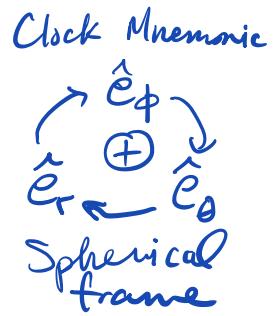
intermediate frame
spherical frame here

${}^A C^B$

Overall, our transformation can be obtained by using both elementary transformations:

$${}^I C^B = ({}^I C^A)({}^A C^B) \quad [{}^I r_{P/O}] = {}^I C^B [{}^B r_{P/O}]_B$$

$${}^I C^B = \begin{bmatrix} \cos\theta & -s\phi & c\theta s\phi \\ s\cos\phi & c\theta & s\theta s\phi \\ -s\phi & 0 & c\phi \end{bmatrix}$$



What are our kinematics in a spherical frame?

$$\begin{aligned} \vec{r}_{Pb} &= r \hat{e}_r \\ {}^I \vec{v}_{P/I_0} &= \dot{r} \hat{e}_r + r \underbrace{\frac{d}{dt}(\hat{e}_r)}_{\text{Angular velocity}} \end{aligned}$$

→ We know how to differentiate unit vectors using the angular velocity,

$$\frac{d}{dt}(\hat{e}_r) = \underbrace{{}^I \vec{\omega}^B}_{\text{Angular velocity}} \times \hat{e}_r$$

→ Addition property

$$\begin{aligned} {}^I \vec{\omega}^B &= {}^I \vec{\omega}^A + {}^A \vec{\omega}^B \\ &= \dot{\theta} \hat{e}_z + \dot{\phi} \hat{e}_\theta \\ &= \dot{\theta} (-s\phi \hat{e}_\phi + c\phi \hat{e}_r) + \dot{\phi} \hat{e}_\theta \end{aligned}$$

$$\Rightarrow \frac{d}{dt}(\hat{e}_r) = \dot{\theta} s\phi \hat{e}_\theta + \dot{\phi} \hat{e}_\phi$$

Similarly, we could find,

$$\frac{d}{dt}(\hat{e}_\theta) = \vec{\omega}^B \times \hat{e}_\theta = -\dot{\phi} s\phi \hat{e}_r - \dot{\theta} c\phi \hat{e}_\phi$$

$$\frac{d}{dt}(\hat{e}_\phi) = \vec{\omega}^B \times \hat{e}_\phi = -\dot{\phi} \hat{e}_r + \dot{\theta} c\phi \hat{e}_\theta$$

Returning to complete kinematics:

$$\vec{V}_{P/0} = \dot{r} \hat{e}_r + r \dot{\theta} s\phi \hat{e}_\theta + r \dot{\phi} \hat{e}_\phi$$

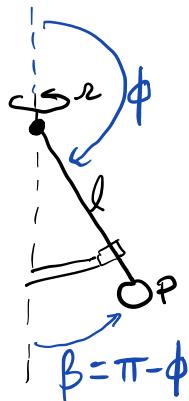
$$\begin{aligned} \vec{a}_{P/0} = & (2\dot{r}\phi + r\ddot{\phi} - r\dot{\theta}^2 c\phi s\phi) \hat{e}_\phi \\ & + (2\dot{r}\dot{\theta} s\phi + 2r\dot{\theta}\dot{\phi}c\phi + r\ddot{\theta}s\phi) \hat{e}_\theta \\ & + (\ddot{r} - r\dot{\phi}^2 - r\dot{\theta}^2 \sin^2\phi) \hat{e}_r \end{aligned}$$

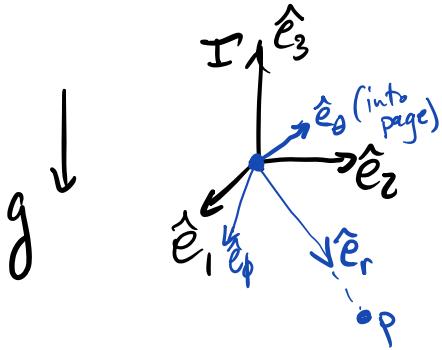
Much more manageable than the expressions for $\vec{a}_{P/0}$ expressed in the inertial frame using spherical coordinates.

Ex. "Simplified" Flyball Governor No linkages
 r is constant

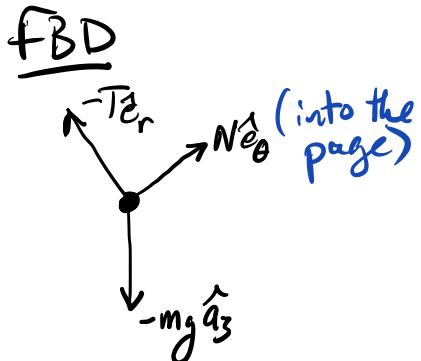
See full Flyball Governor example in Tutorial 10.2 to see a slightly different approach/model.

↳ Uses angular momentum





$$\begin{aligned}
 \text{kinematics} \\
 {}^I\vec{\omega}^B &= R \hat{e}_3 + \dot{\phi} \hat{e}_\theta \\
 &= R(-s\phi \hat{e}_\phi + c\phi \hat{e}_r) + \dot{\phi} \hat{e}_\theta \\
 {}^I\vec{r}_{P/0} &= l \hat{e}_r \\
 {}^I\vec{v}_{P/0} &= l \vec{\omega}^B \times \hat{e}_r = l(n s\phi \hat{e}_\theta + \dot{\phi} \hat{e}_\phi) \\
 {}^I\vec{a}_{P/0} &= l R \dot{\phi} c\phi \hat{e}_\theta + l R s\phi \frac{d}{dt}(\hat{e}_\theta) + l \ddot{\phi} \hat{e}_\phi \\
 &\quad + l \dot{\phi} \frac{d}{dt}(\hat{e}_\phi) \\
 &\vdots \\
 &= (-l R^2 s\phi c\phi + l \ddot{\phi}) \hat{e}_\phi + (2l R c\phi \dot{\phi}) \hat{e}_\theta \\
 &\quad + (-l R^2 s^2 \phi - l \dot{\phi}^2) \hat{e}_r
 \end{aligned}$$



$$\vec{F}_P = m_p {}^I\vec{a}_{P/0} \Rightarrow -T \hat{e}_r + N \hat{e}_\theta - mg(-s\phi \hat{e}_\phi + c\phi \hat{e}_r) = m_p (\dots)$$

$$\hat{e}_r : -T - m_p g c\phi = m_p (-l R^2 s^2 \phi - l \dot{\phi}^2)$$

$$\hat{e}_\theta : N = 2m_p l R c\phi \dot{\phi}$$

$$\hat{e}_\phi : m_p s\phi = m_p (-l R^2 s\phi c\phi + l \ddot{\phi})$$

Unknown's : $\ddot{\phi}, T, N$ Note: θ doesn't appear here
 $\theta(t) = \theta_0 + \theta(t)$

$$\text{EOM: } \ddot{\phi} = \frac{2}{l} s\phi + R^2 s\phi c\phi$$

But, in terms of β for convenience,
 $\beta = \pi - \phi$

$$\ddot{\beta} = -\frac{g}{l} s\beta + R^2 s\beta c\beta$$

Equil points

$$0 = \left(-\frac{g}{l} + R^2 c\beta^* \right) s\beta^*$$

$$-\frac{g}{l} + R^2 c\beta^* = 0 \quad \rightarrow \sin \beta^* = 0$$

$$\beta^* = 0, \pi$$

$$\beta^* = \pm \cos^{-1} \left(\frac{g}{lR^2} \right)$$

$$\beta^* = 0, \pi, \pm \cos^{-1} \left(\frac{g}{lR^2} \right)$$

only occurs for $R \geq \sqrt{\frac{g}{l}}$

Bifurcation Diagram

