

Local Kinematic Analysis of Lower Pair Linkages

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Summer School *Singularities of Mechanism and Robotic Manipulator*
September 18-22, 2017, Linz, Austria – robotsingularities.org

- Finite DOF, full cycle mobility
- Instantaneous DOF
- Overconstrained mechanisms
- Paradoxical mechanisms
- Mechanism Geometry
- Screw systems
- Polynomial systems,
algebraic geometry

- Kinematic topology
- Classes/Families of mechanisms
- Chebychev-Kutzbach-Grübler (CKG) formula
- Planar/spherical/spatial mechanisms

- Generic DOF
- Stable mappings
- Combinatorial algorithms
- Pebble game

1. Phenomenology

- Mobility
- Constraint-, C-Space-, Input-, Output-Singularities
- A Model for the Mechanism Kinematics

2. Kinematics Modeling

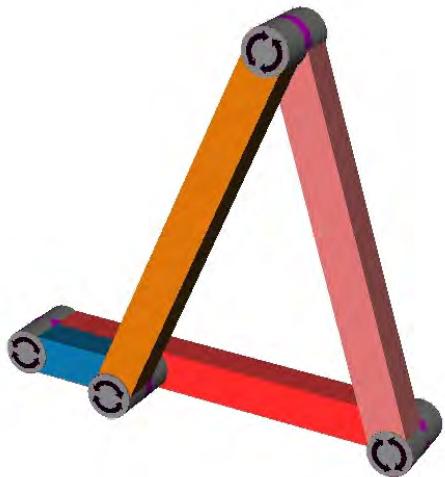
- Screws and the Product-of-Exponentials
- Geometric Jacobian
- Higher-Order Kinematics

3. Local analysis of C-Space and ‘singular set’

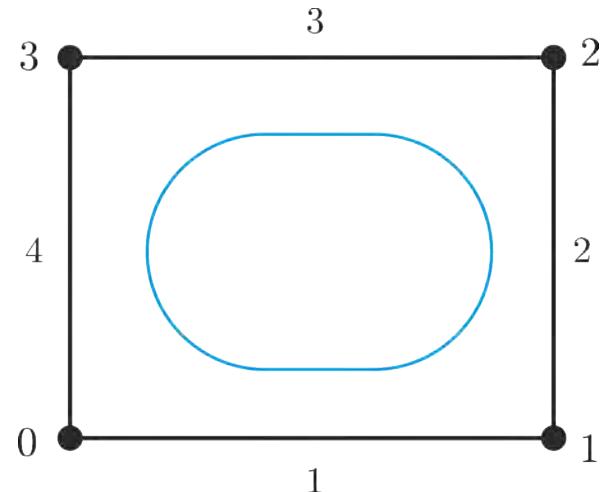
- Tangent space, Tangent cone
- ‘Kinematic tangent cone’

4. Further Topics

- Singularities of Non-Holonomic Systems
- Escapement from singularities
- Combinatorial algorithm: Pebble Game



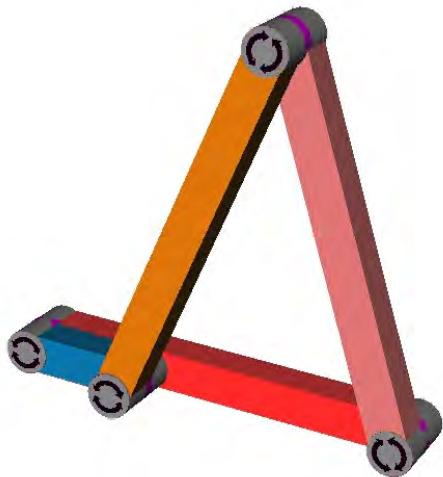
- One kinematic loop
(topology)
- Parallel joint axes
(geometry)
- Rigid links
(physical properties)



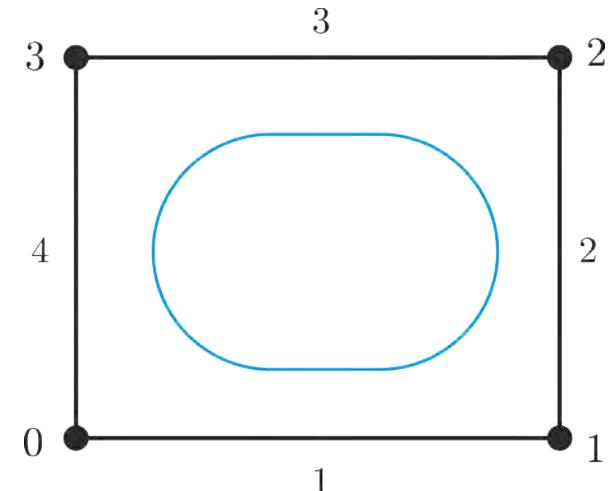
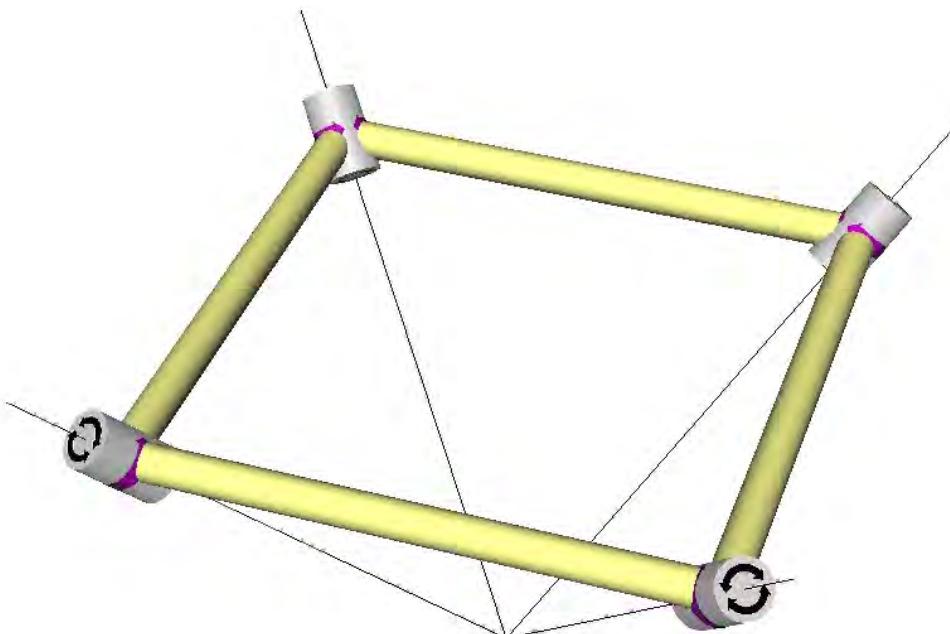
Loop constraints: $h(\mathbf{q}) = \mathbf{0}$

Coordinates \mathbf{q} :

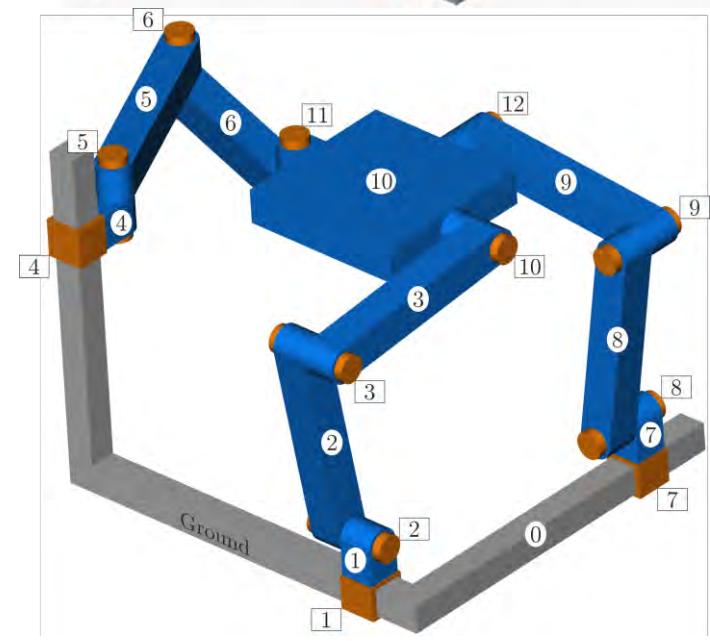
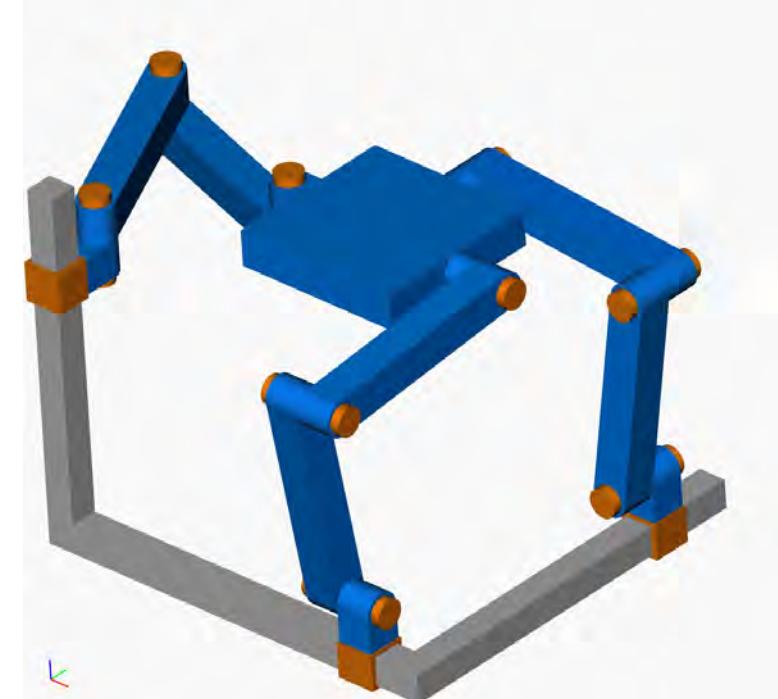
- Joint variables (angles, translations)
- ‘Absolute coordinates’



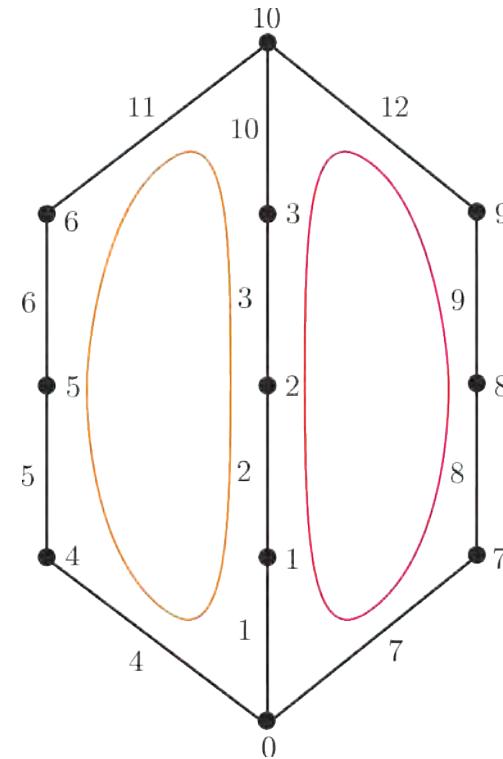
- One kinematic loop
(topology)
- Parallel joint axes
(geometry)
- Rigid links
(physical properties)

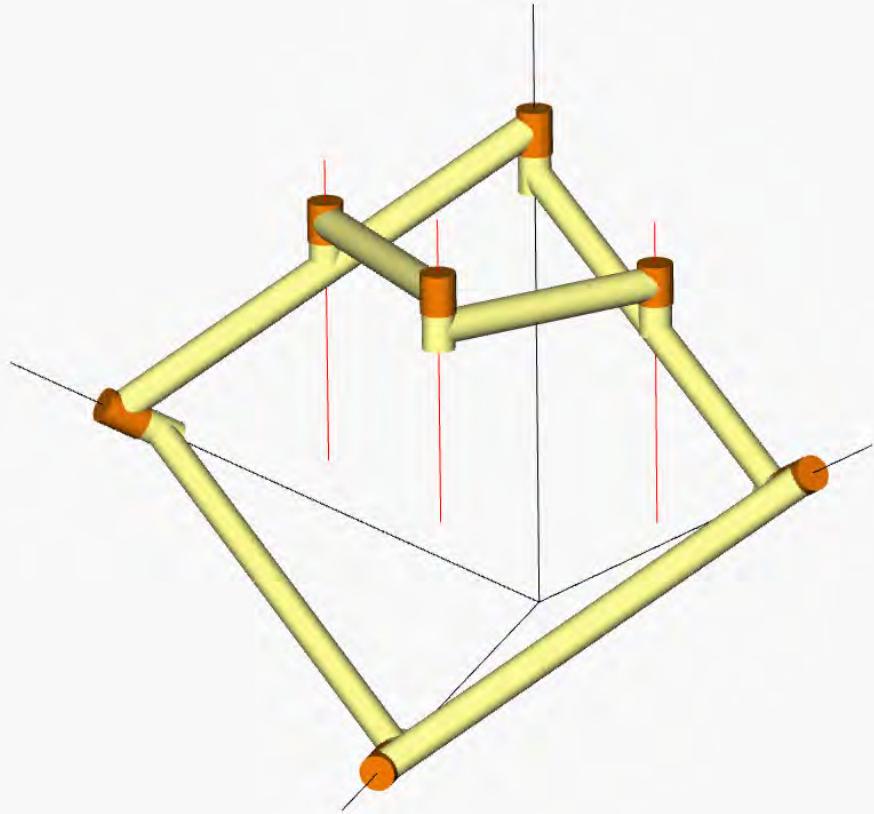


- One kinematic loop
(topology)
- Axes intersect at one point
(geometry)
- Rigid links
(physical properties)

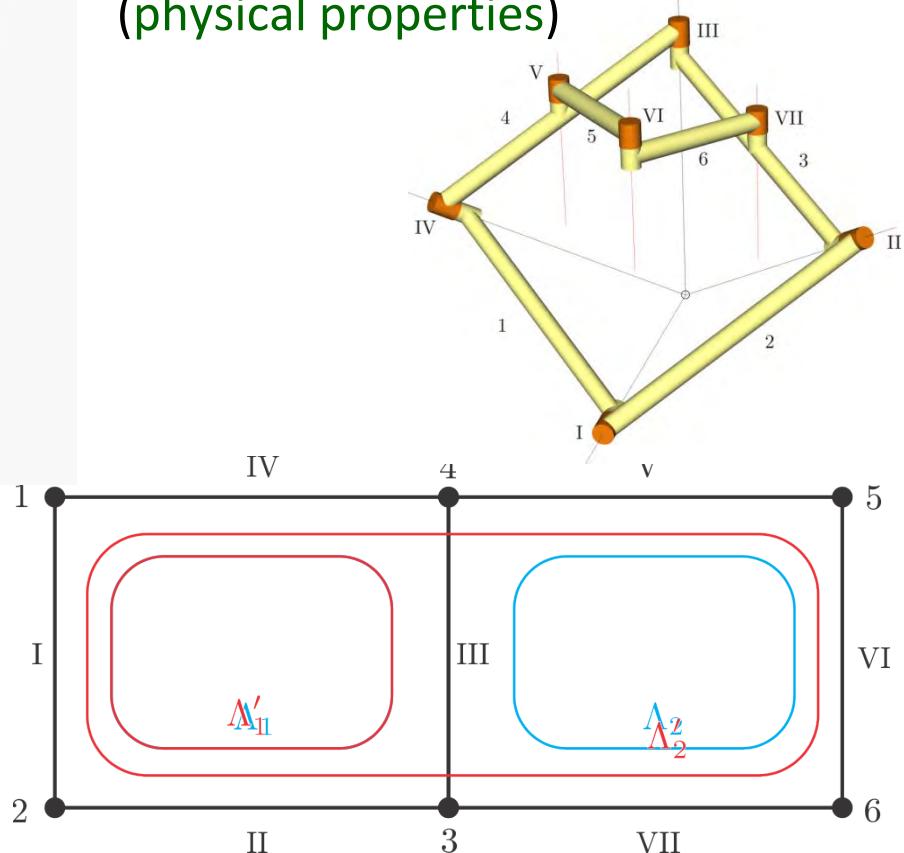


- Multiloop kinematics
(**topology**)
- Intersection of planar motions
(**geometry**)
- Rigid links
(**physical properties**)





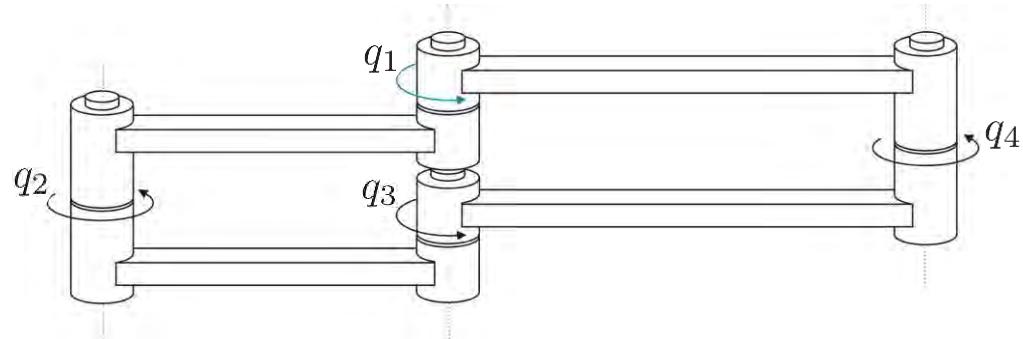
- Multiloop kinematics
(topology)
- Spherical + planar motion
(geometry)
- Rigid links
(physical properties)



- Singularity = ‘Critical situation’



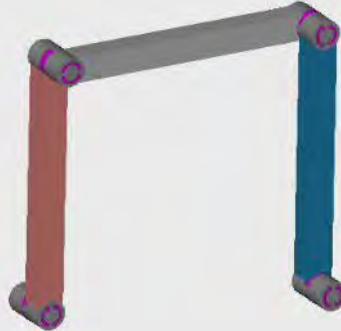
- Joint variables: $\mathbf{q} \in \mathbb{V}^n$
- Constraints: $h(\mathbf{q}) = \mathbf{0}$
 $\mathbf{J}(\mathbf{q})\dot{\mathbf{q}} = \mathbf{0}$
- Configuration space: $V := \{\mathbf{q} \in \mathbb{V}^n | h(\mathbf{q}) = \mathbf{0}\}$
(Parameter space!)
- Differential DOF: $\delta_{\text{diff}}(\mathbf{q}) := n - \text{rank } \mathbf{J}(\mathbf{q})$
(Instantaneous DOF)
- Local (finite) DOF: $\delta_{\text{loc}}(\mathbf{q}) := \dim_{\mathbf{q}} V$



$$\mathbf{q} = (q_1, q_2, q_3, q_4)^T \in \mathbb{V}^4, \quad n = 4$$

- Joint variables: $\mathbf{q} \in \mathbb{V}^n$
- Configuration space: $V := \{\mathbf{q} \in \mathbb{V}^n | h(\mathbf{q}) = \mathbf{0}\}$
(Parameter space!)
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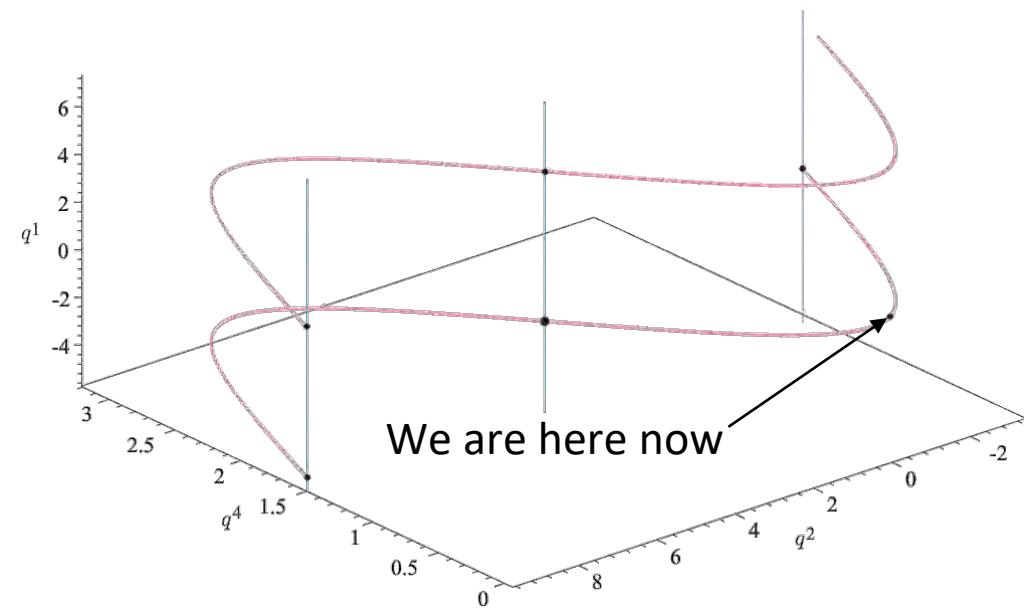
Rank drops at bifurcation point



$$\delta_{\text{diff}} = 1, 2$$

$$\delta_{\text{loc}} = 1$$

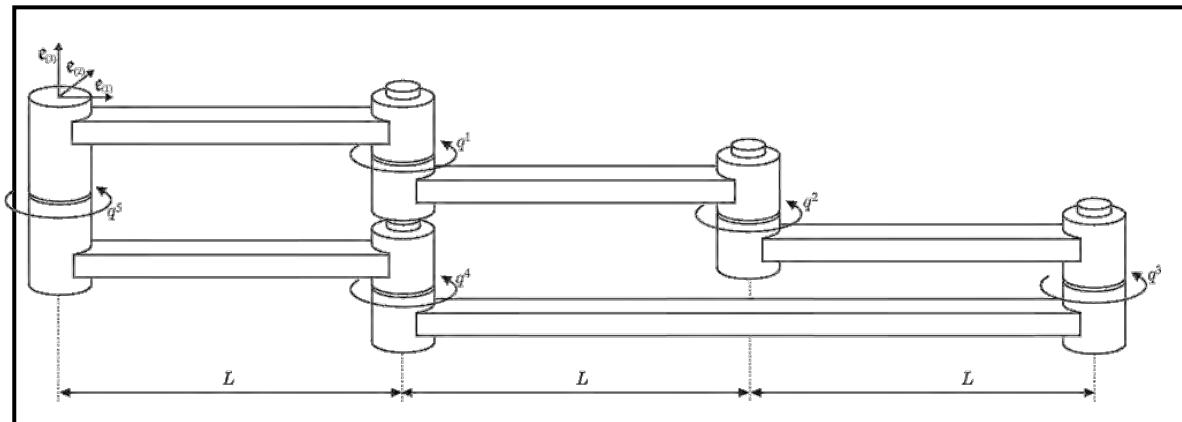
C-space is **non-smooth** at bifurcation point



Some examples: 5-bar linkage

- Joint variables: $\mathbf{q} \in \mathbb{V}^n$
- Constraints: $h(\mathbf{q}) = \mathbf{0}$
 $\mathbf{J}(\mathbf{q})\dot{\mathbf{q}} = \mathbf{0}$
- Configuration space: $V := \{\mathbf{q} \in \mathbb{V}^n | h(\mathbf{q}) = \mathbf{0}\}$
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- Differential DOF: $\delta_{\text{diff}}(\mathbf{q}) := n - \text{rank } \mathbf{J}(\mathbf{q})$
(Instantaneous DOF)
- Local (finite) DOF: $\delta_{\text{loc}}(\mathbf{q}) := \dim_{\mathbf{q}} V$

Rank drops at bifurcation point



$$\delta_{\text{diff}} = 2, 3$$

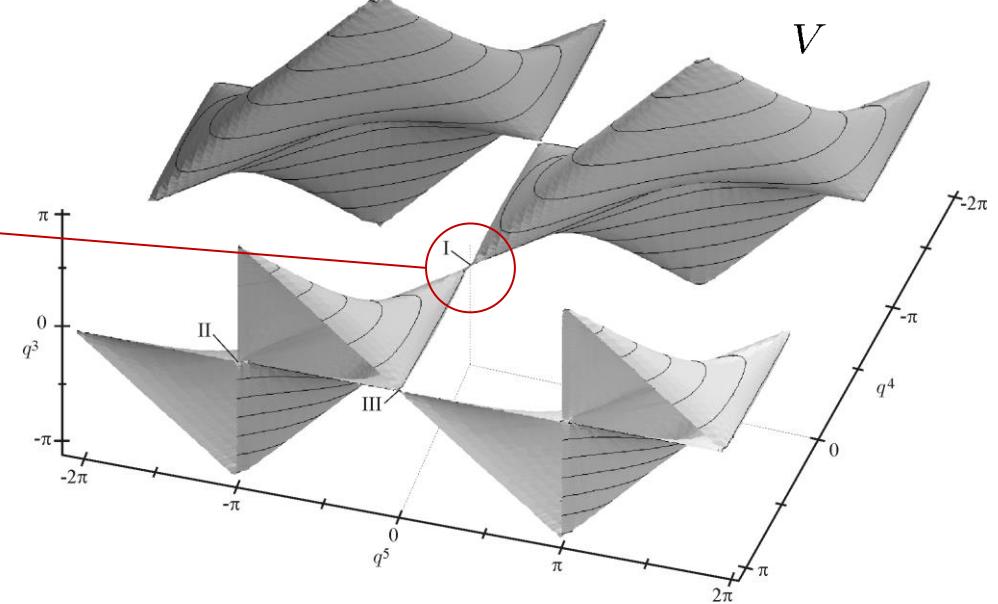
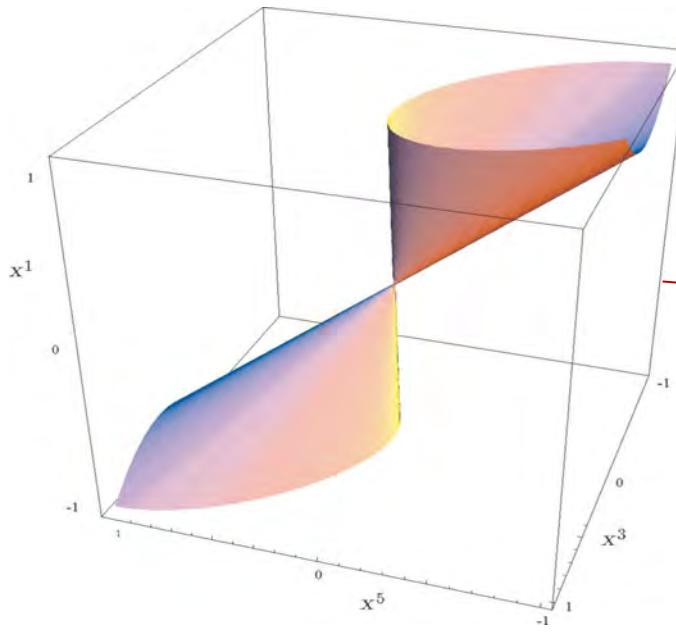
$$\delta_{\text{loc}} = 2$$

$$\mathbf{q} = (q_1, q_2, q_3, q_4, q_5) \in \mathbb{V}^5$$

$$n = 5$$

- Joint variables: $\mathbf{q} \in \mathbb{V}^n$
- Constraints: $h(\mathbf{q}) = \mathbf{0}$
 $\mathbf{J}(\mathbf{q})\dot{\mathbf{q}} = \mathbf{0}$
- Configuration space: $V := \{\mathbf{q} \in \mathbb{V}^n | h(\mathbf{q}) = \mathbf{0}\}$
 (Parameter space!)
- Differential DOF: $\delta_{\text{diff}}(\mathbf{q}) := n - \text{rank } \mathbf{J}(\mathbf{q})$
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- Local (finite) DOF: $\delta_{\text{loc}}(\mathbf{q}) := \dim_{\mathbf{q}} V$

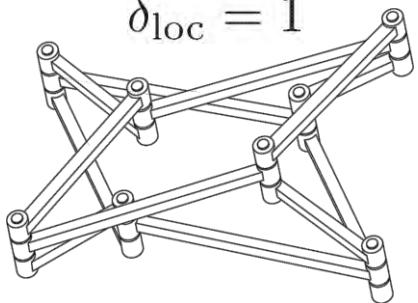
C-space is **non-smooth** at bifurcation point



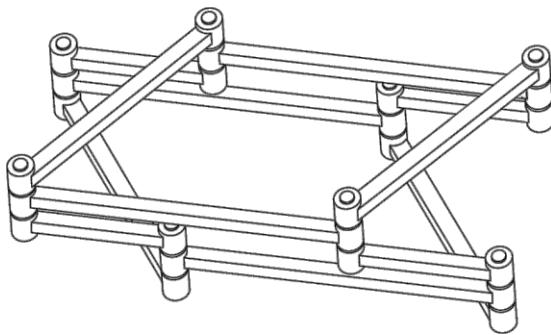
Some examples: Wunderlich's kinematotropic linkage

$$\delta_{\text{diff}} = 1$$

$$\delta_{\text{loc}} = 1$$

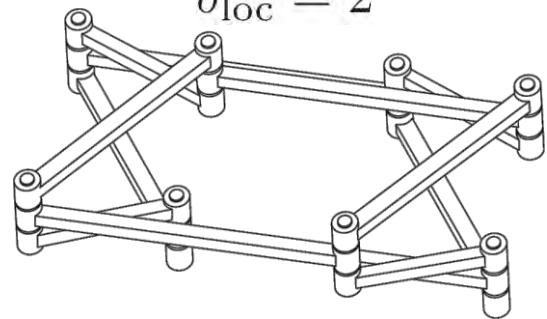


$$\delta_{\text{diff}} = 3$$

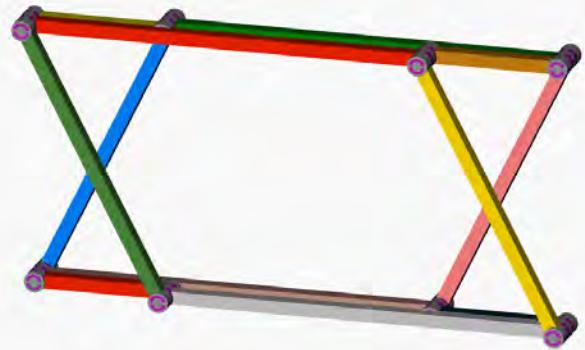
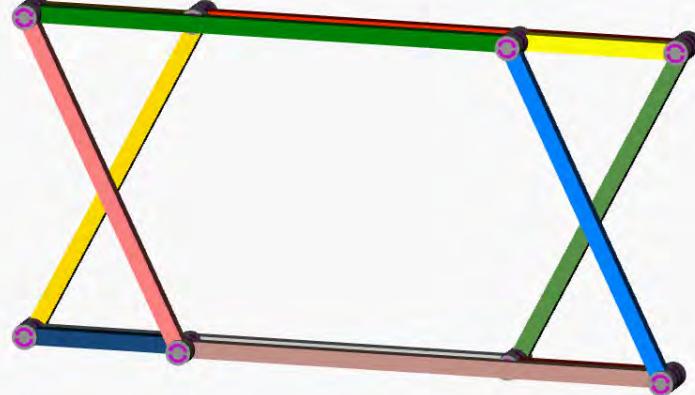


$$\delta_{\text{diff}} = 2$$

$$\delta_{\text{loc}} = 2$$

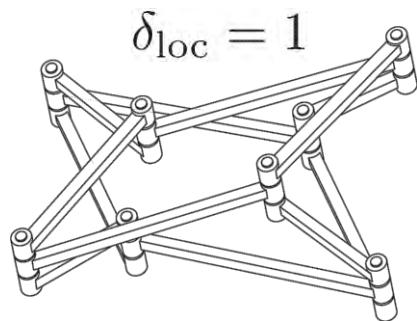


Rank drops at bifurcation point

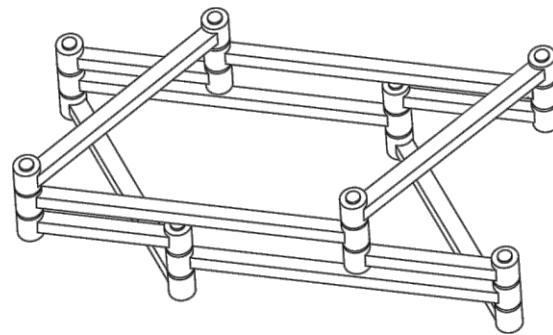


Some examples: Wunderlich's kinematotropic linkage

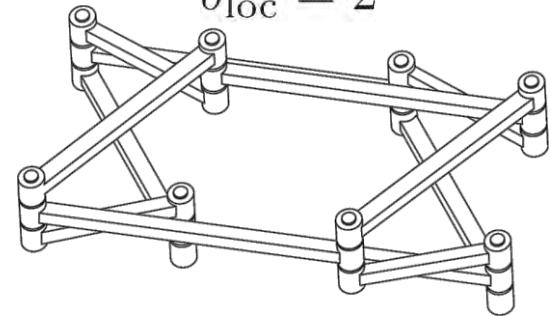
$$\delta_{\text{diff}} = 1$$



$$\delta_{\text{diff}} = 3$$

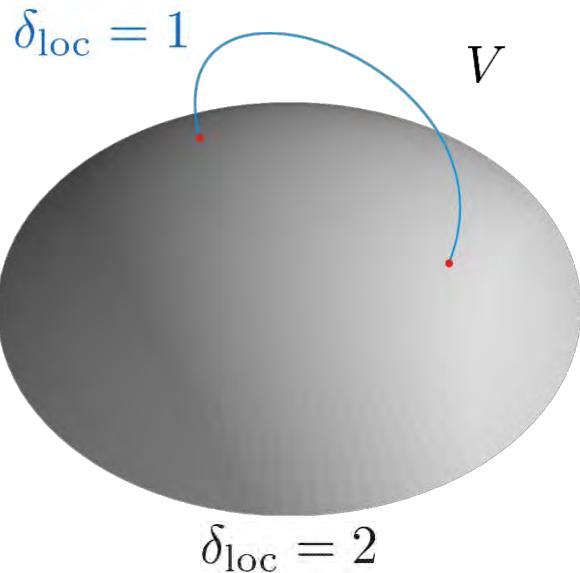


$$\delta_{\text{diff}} = 2$$



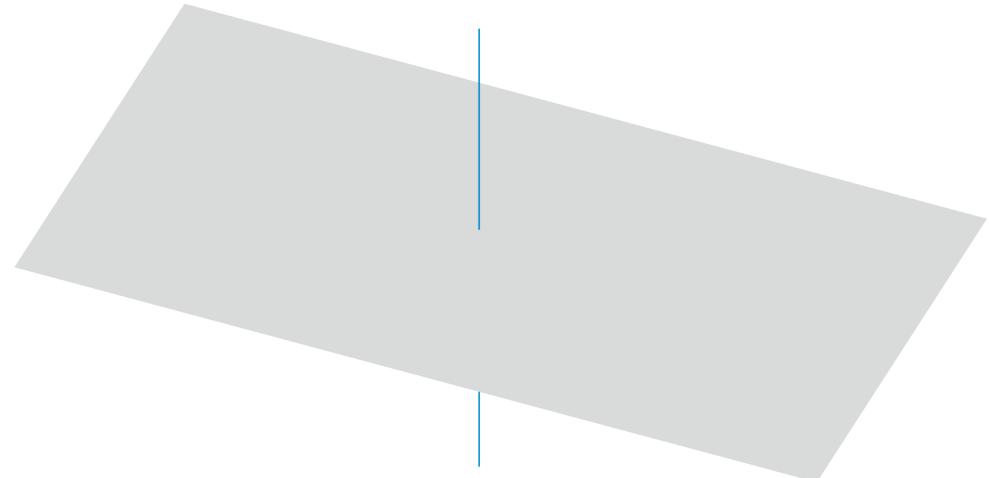
$$\delta_{\text{loc}} = 1$$

Rank **drops** at bifurcation point



$$\delta_{\text{loc}} = 2$$

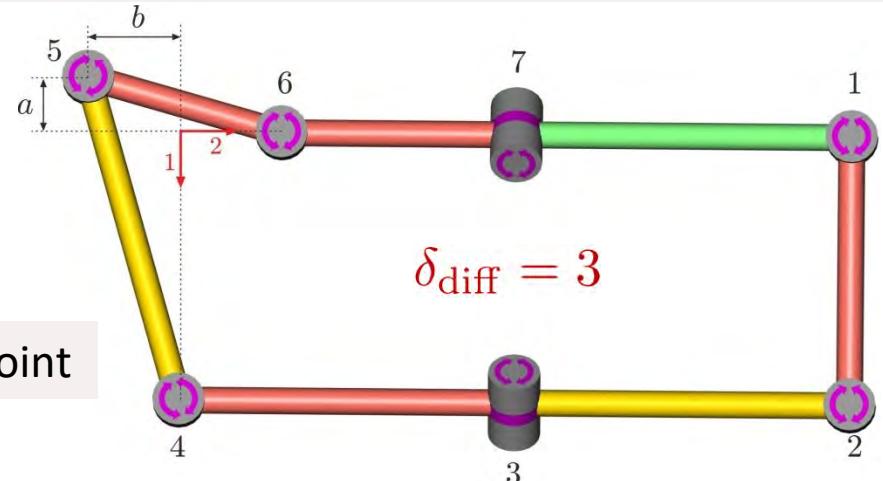
C-space is **non-smooth** at bifurcation point



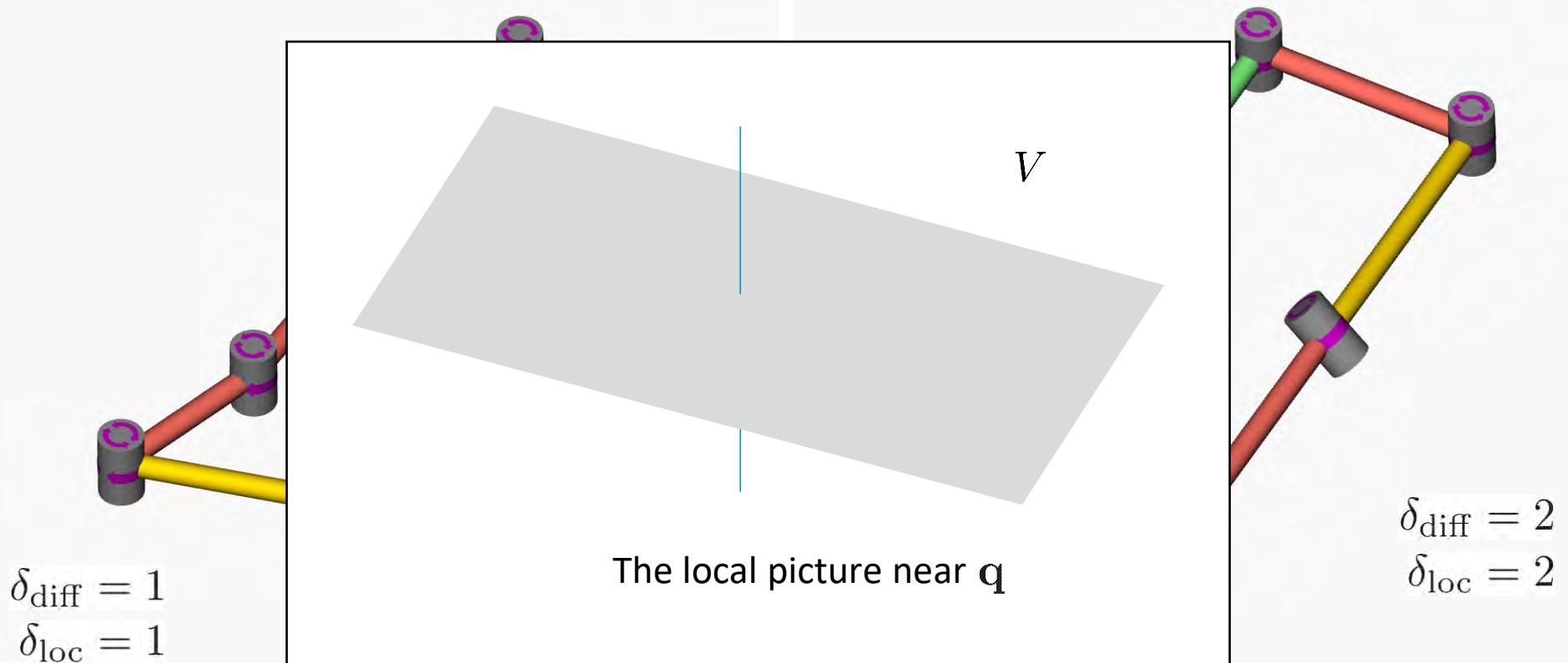
The local picture near q

Some examples: 7R kinematotropic linkage

X. Kong, M. Pfurner,
Mech. Mach. Theory, 2015



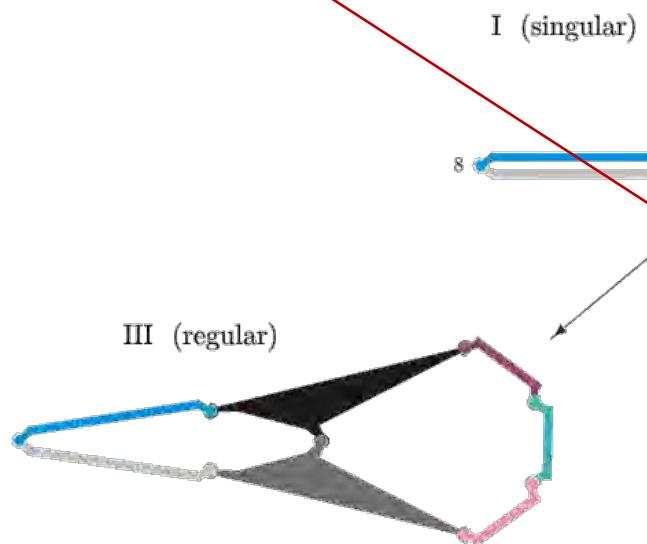
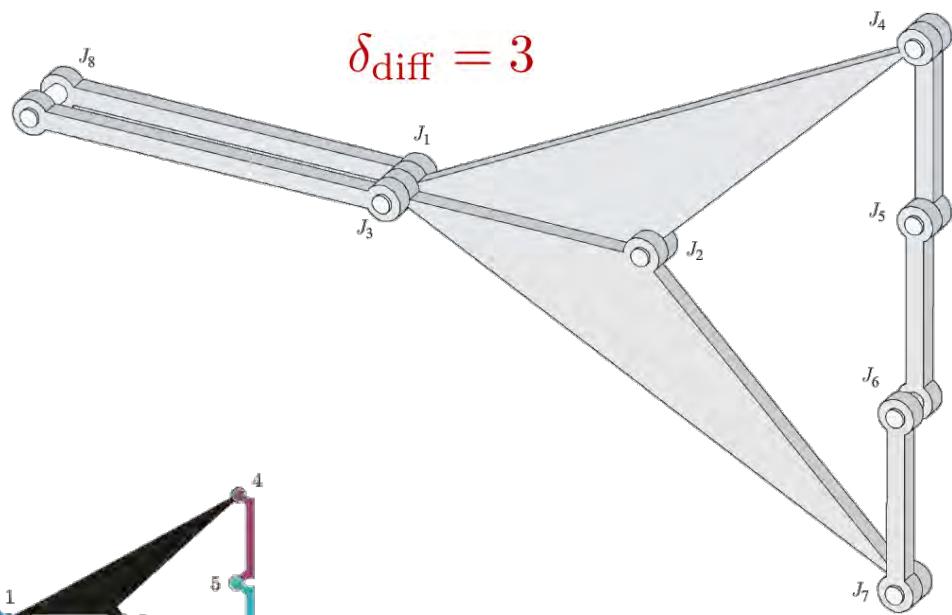
Rank drops at bifurcation point



Some examples: Underconstrained kinematotropic linkage

Rank **drops** at bifurcation point
and stays constant

$$\delta_{\text{diff}}(\mathbf{q}) = n - \text{rank } \mathbf{J}(\mathbf{q}) \\ = \dim \ker \mathbf{J}(\mathbf{q})$$



I (singular)

II (regular)

III (regular)

$$\delta_{\text{diff}} = 2$$

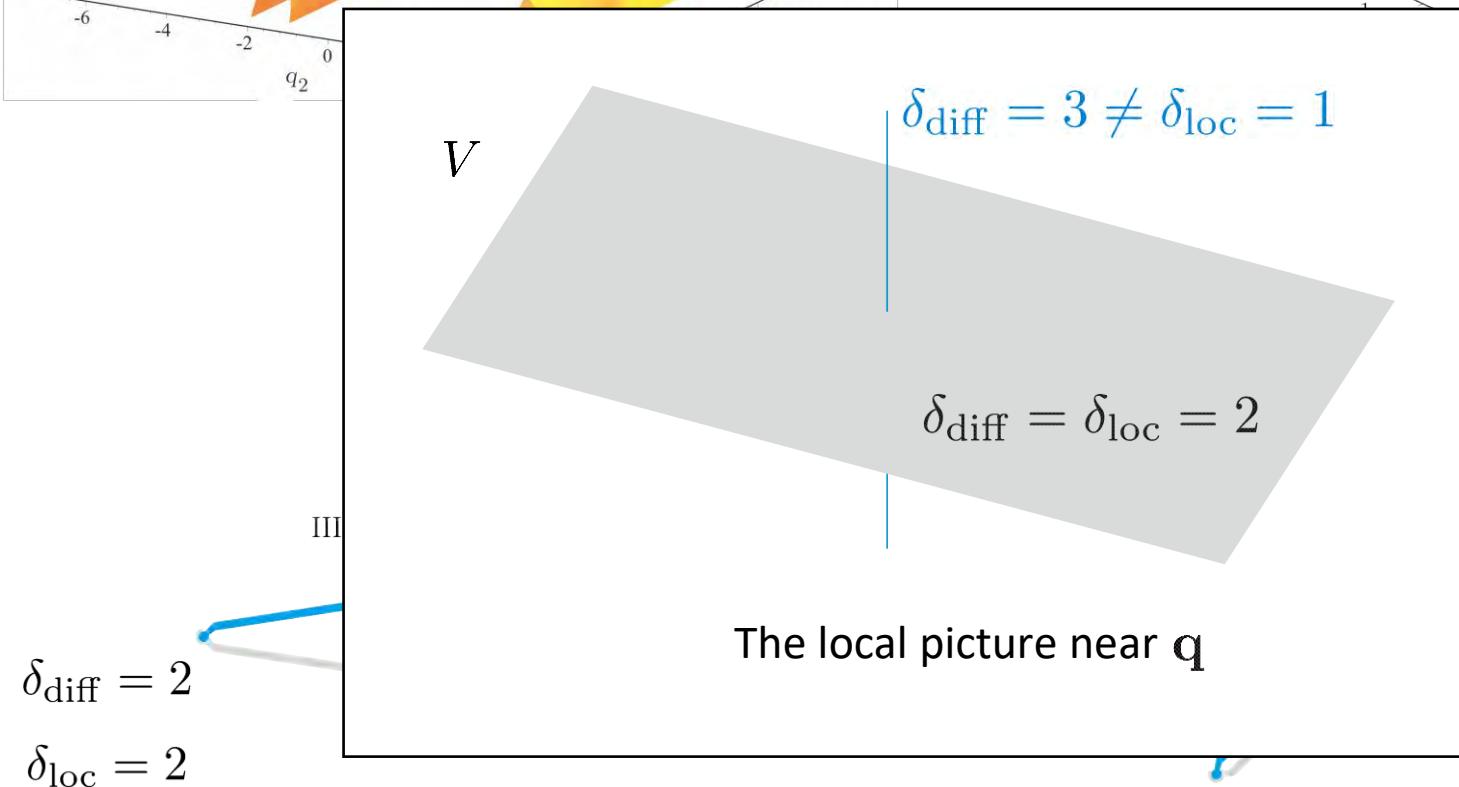
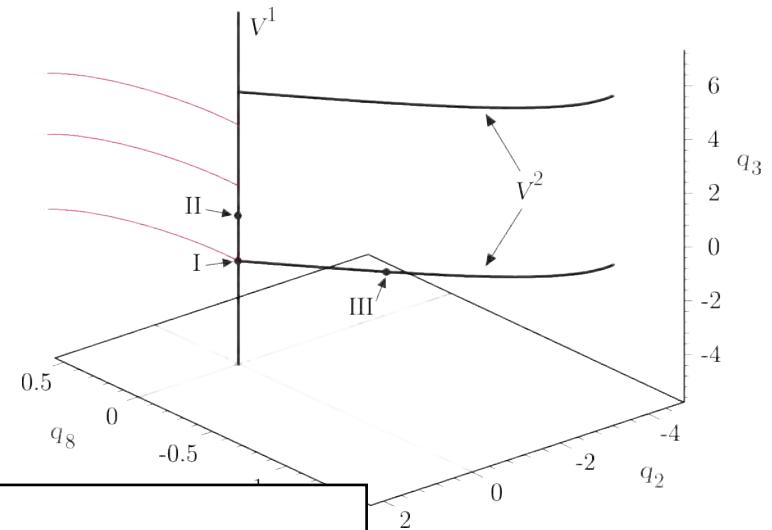
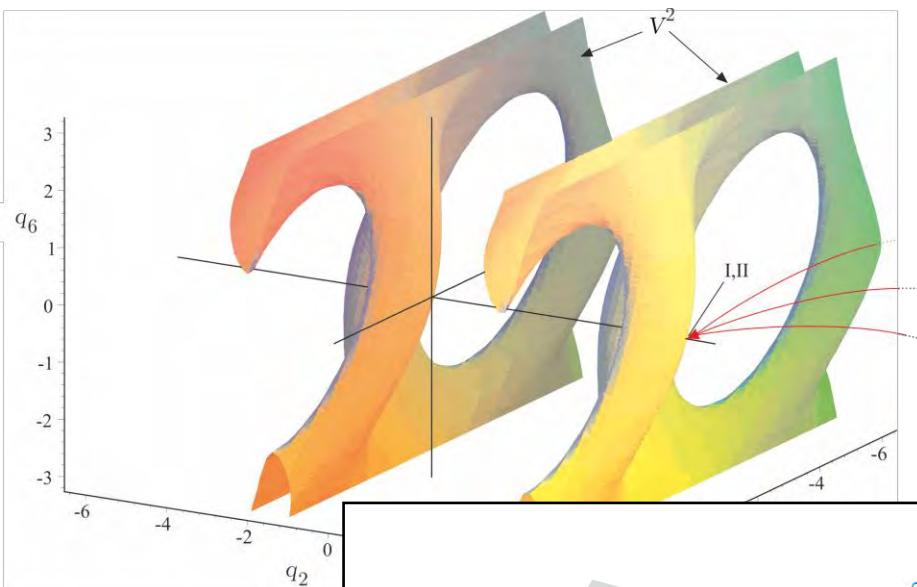
$$\delta_{\text{loc}} = 2 \quad T_{\mathbf{q}} V = \ker \mathbf{J}(\mathbf{q})$$

$$\delta_{\text{diff}} = 3$$

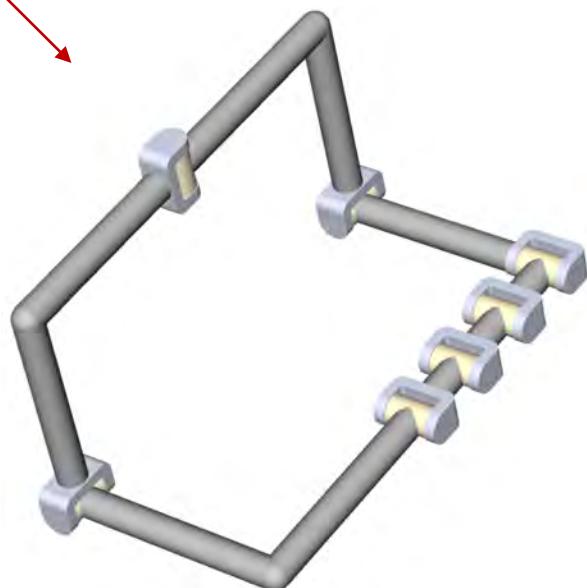
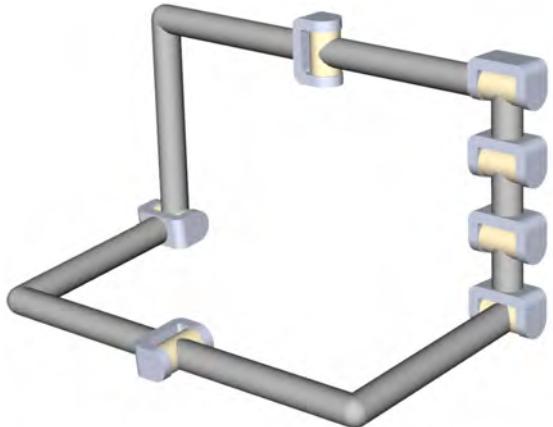
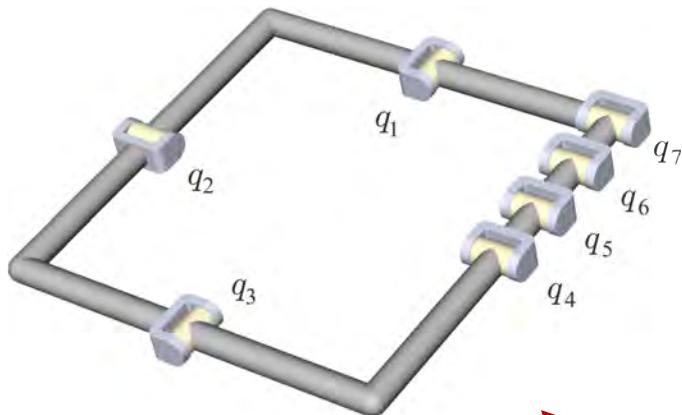
$$\delta_{\text{loc}} = 1$$

$$T_{\mathbf{q}} V \neq \ker \mathbf{J}(\mathbf{q})$$

Some examples: Underconstrained kinematotropic linkage



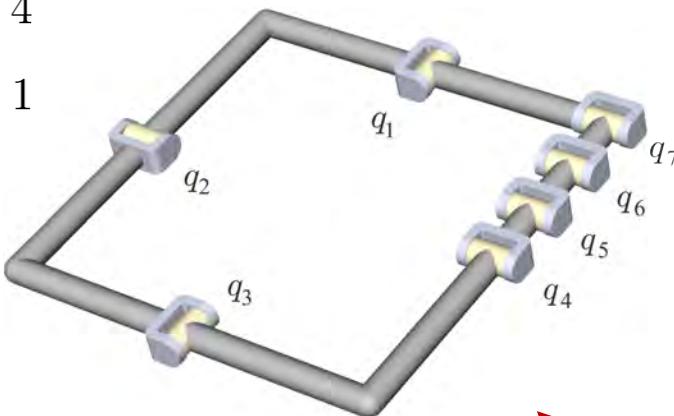
Some examples: Underconstrained linkage



Some examples: Underconstrained (shaky) linkage

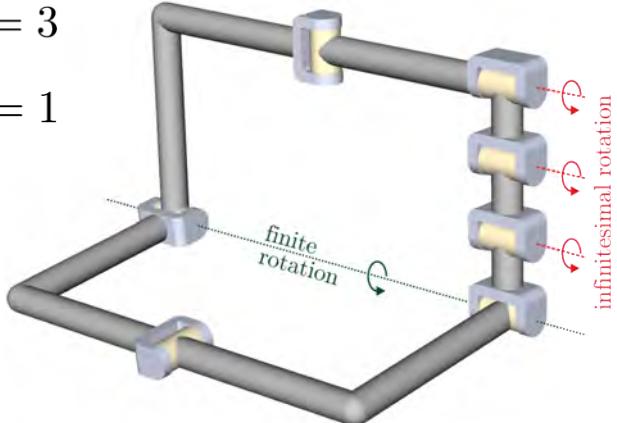
$$\delta_{\text{diff}} = 4$$

$$\delta_{\text{loc}} = 1$$



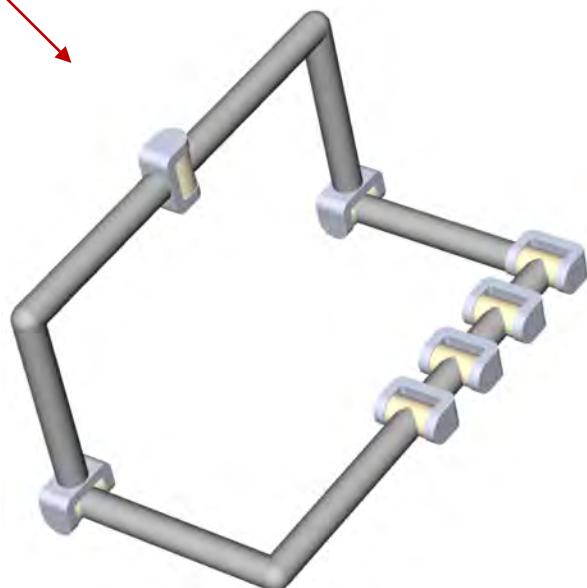
$$\delta_{\text{diff}} = 3$$

$$\delta_{\text{loc}} = 1$$



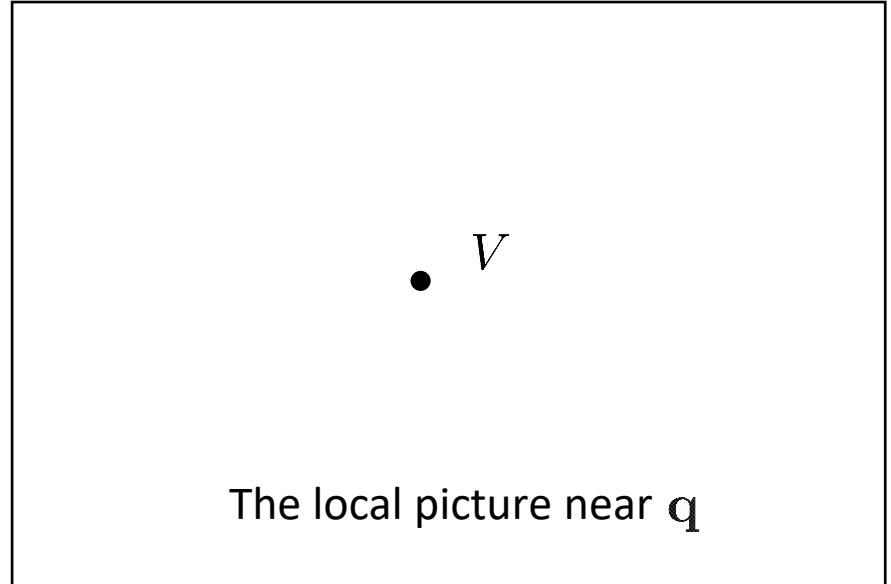
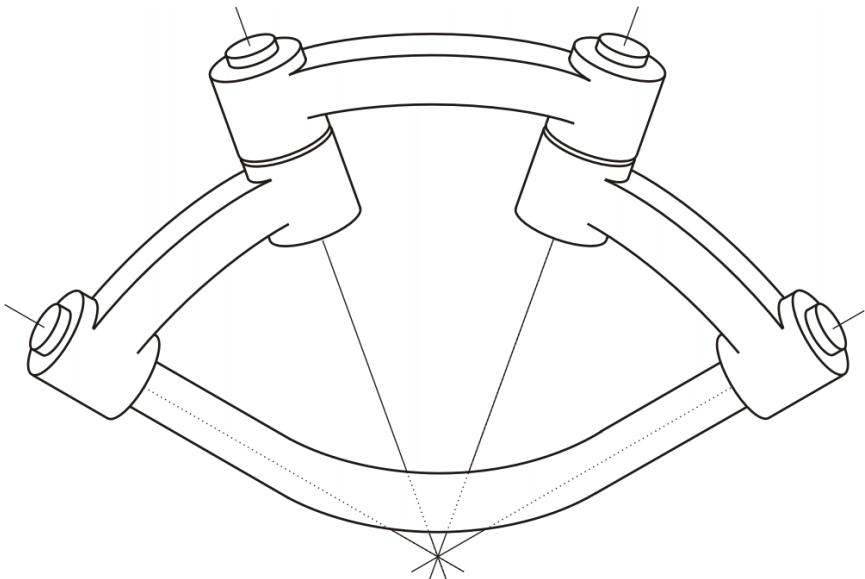
$$\delta_{\text{diff}} = 3$$

$$\delta_{\text{loc}} = 1$$



Some examples: Immobile & shaky 4R linkages

All joint axes are in a common plane

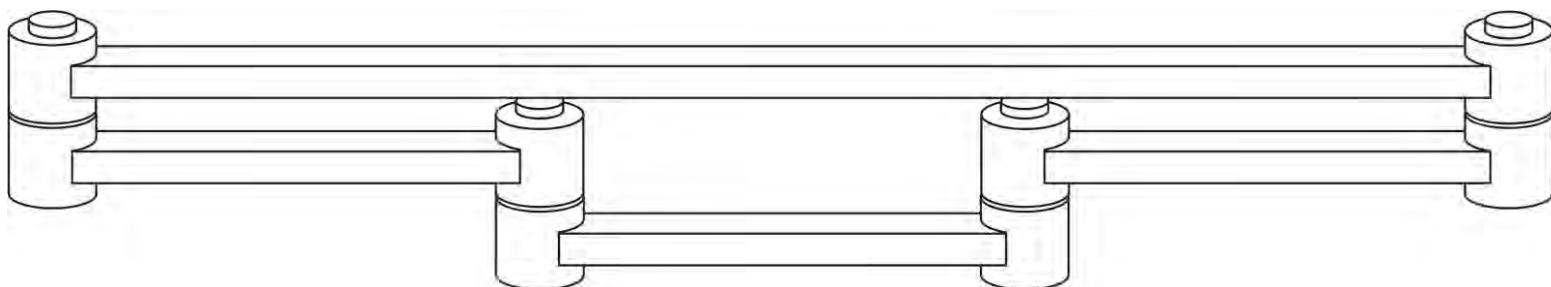


The local picture near q

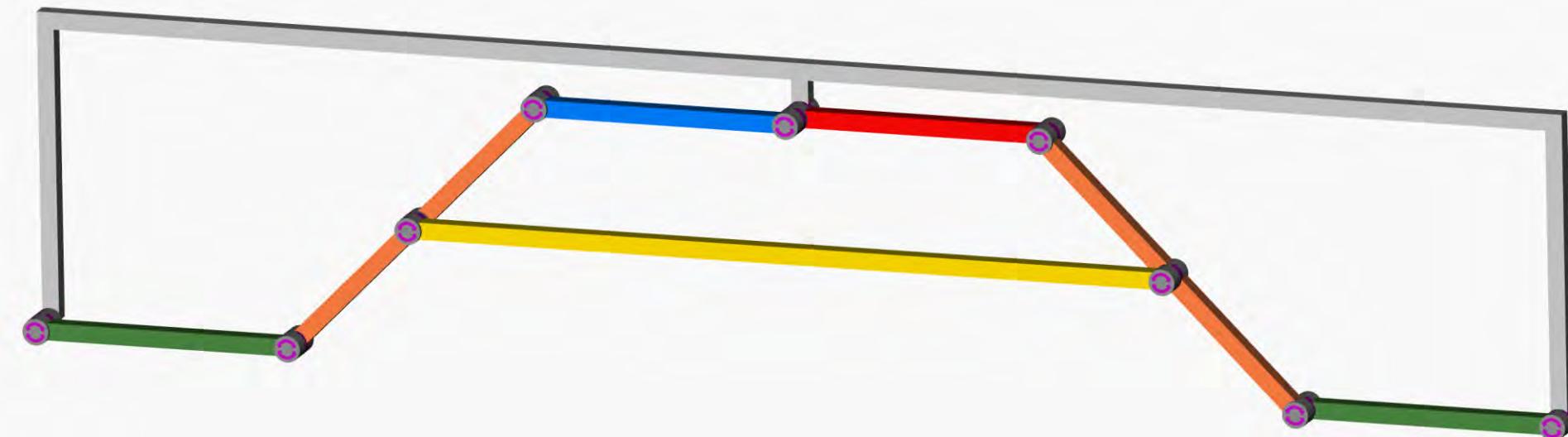
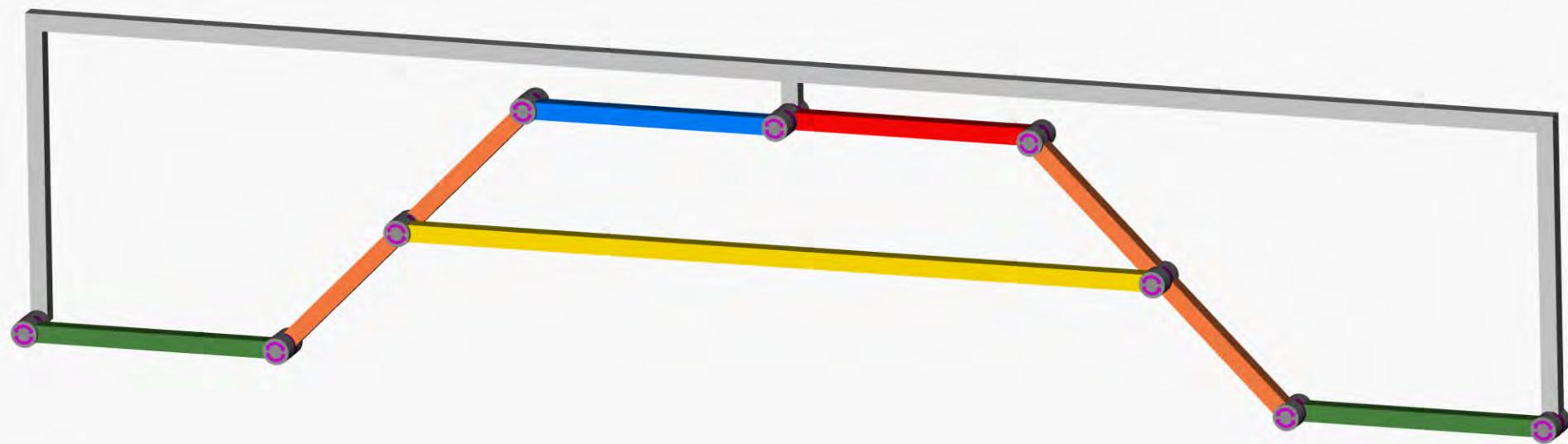
$$\delta_{\text{diff}} = 2$$

$$\delta_{\text{loc}} = 0$$

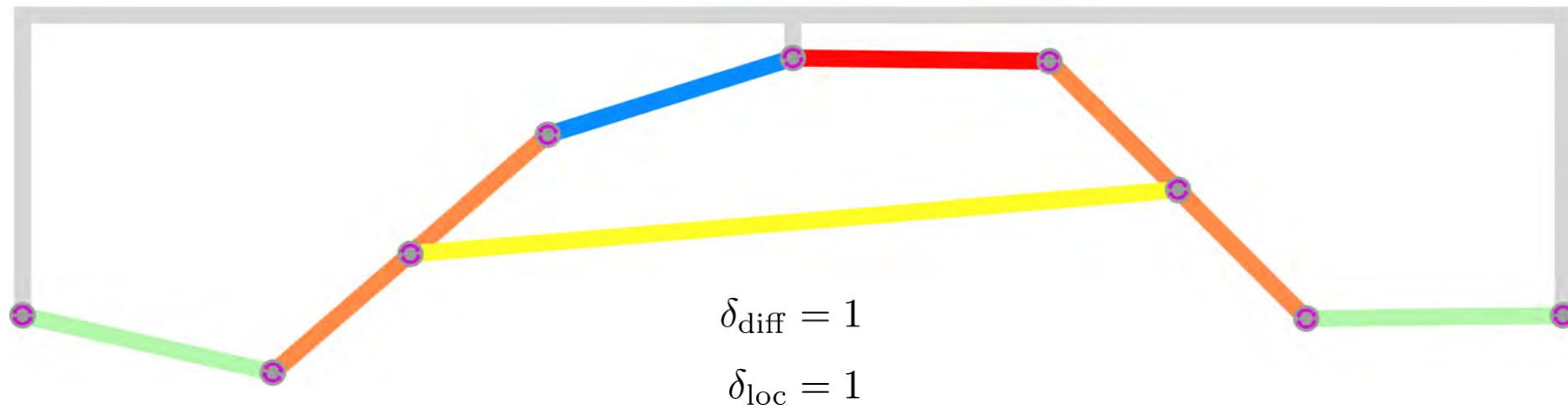
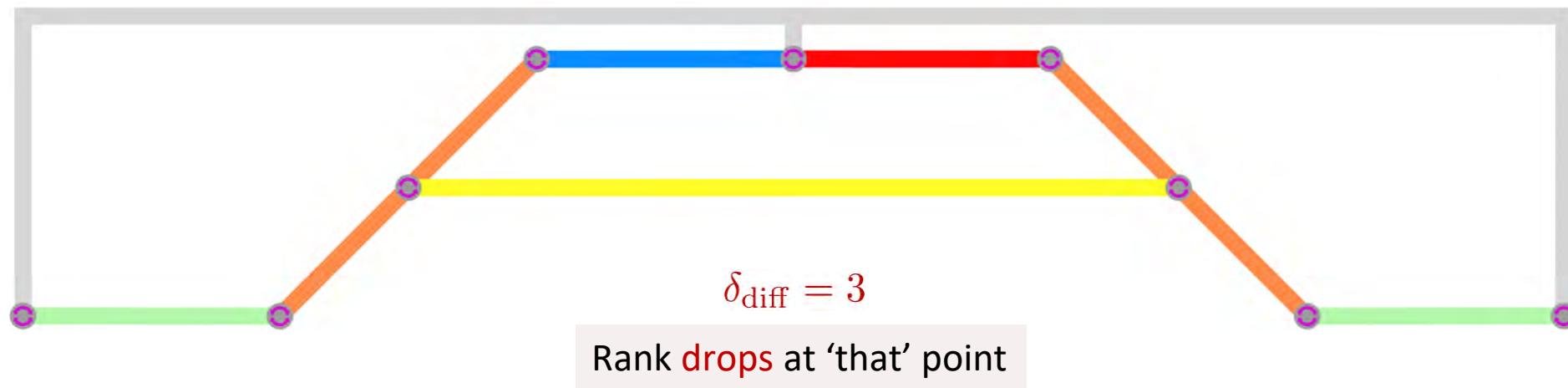
'Shaky' linkages



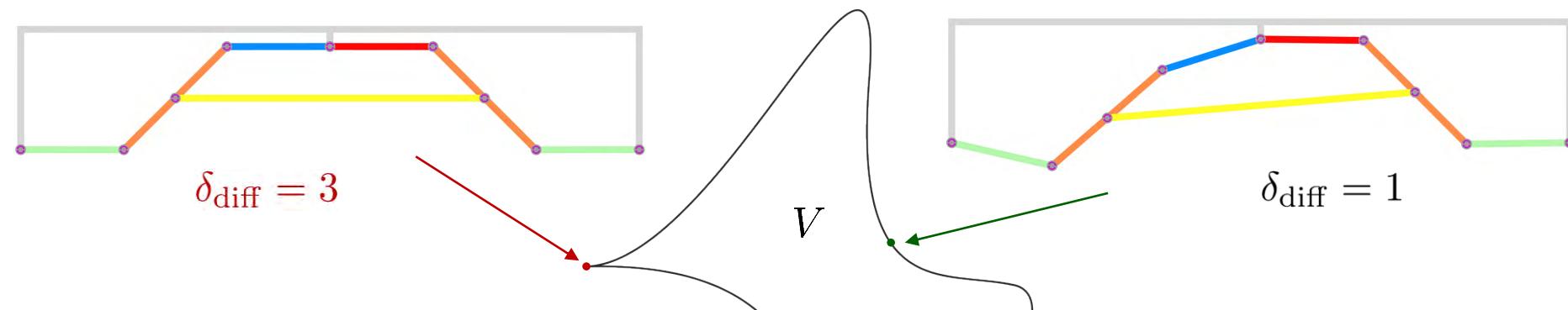
Some examples: 'Double Watt' Linkage



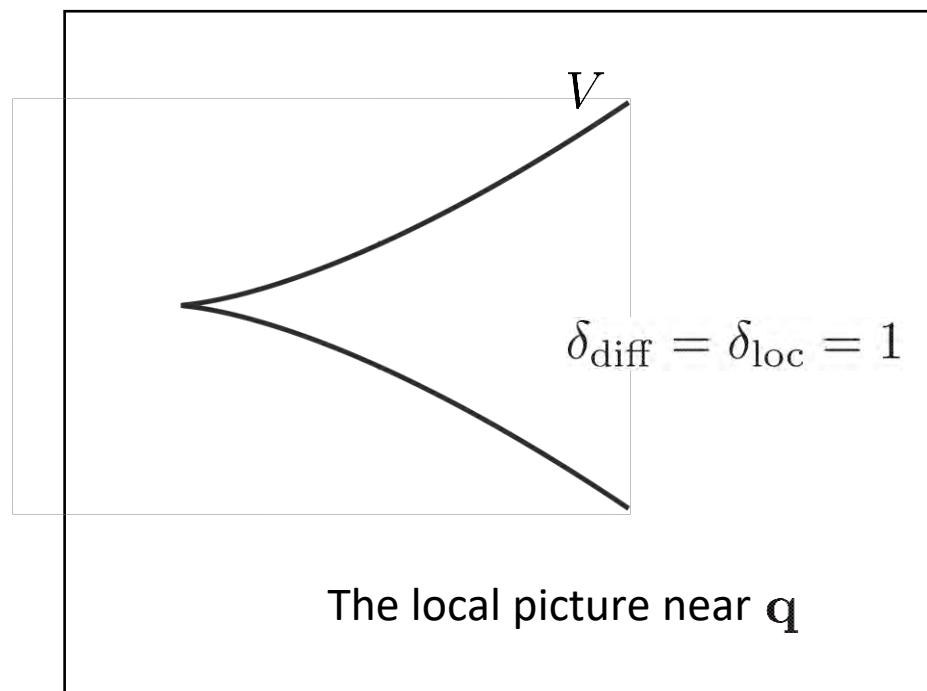
Some examples: 'Double Watt' Linkage



Some examples: 'Double Watt' Linkage

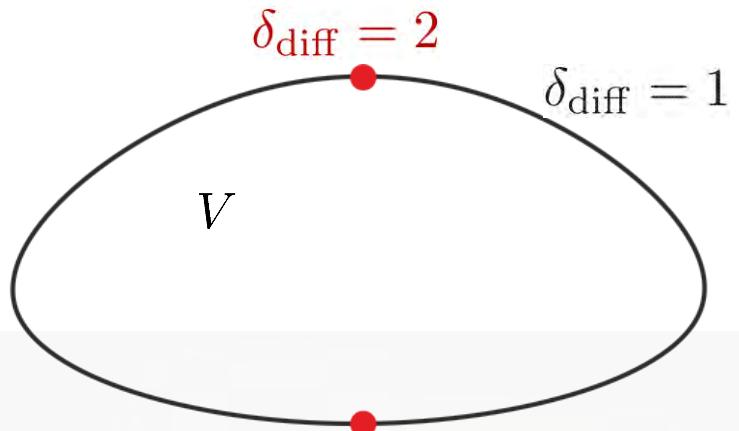


Cusp singularity:



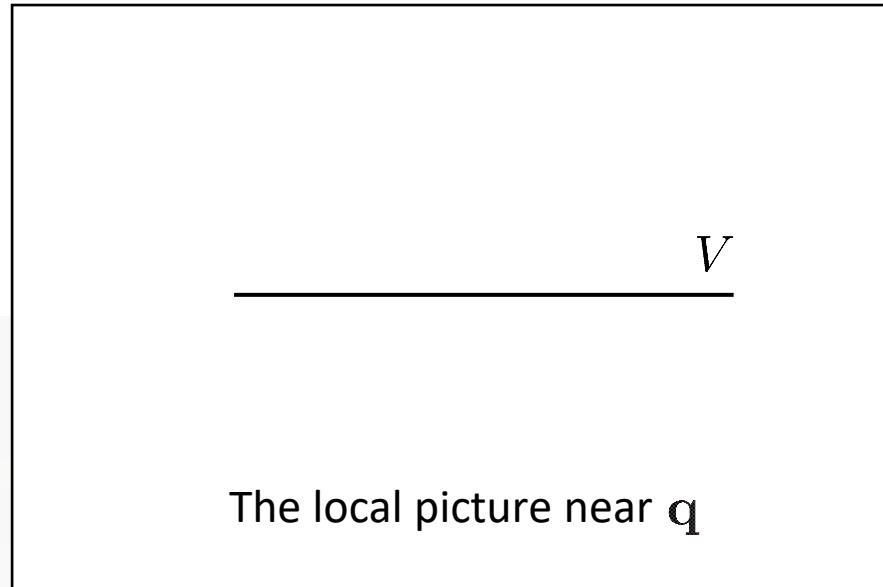
The local picture near q

Some examples: Goldberg 6R linkage

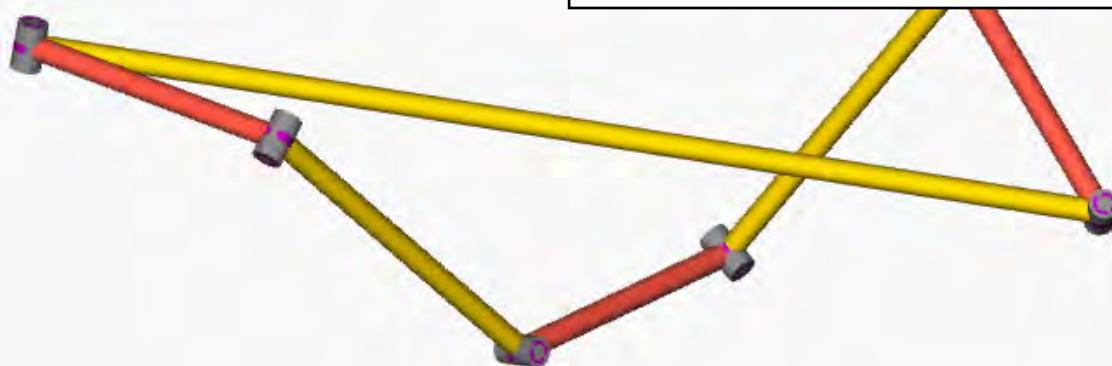


$$\delta_{\text{diff}} = 2$$

$$\delta_{\text{diff}} = 1$$



The local picture near q



$$\delta_{\text{diff}} = 1, 2$$

$$\delta_{\text{loc}} = 1$$

Configuration space is a **smooth 1-dim manifold**

Increased instantaneous mobility but **c-space is smooth**

- Singularity = ‘Critical situation’

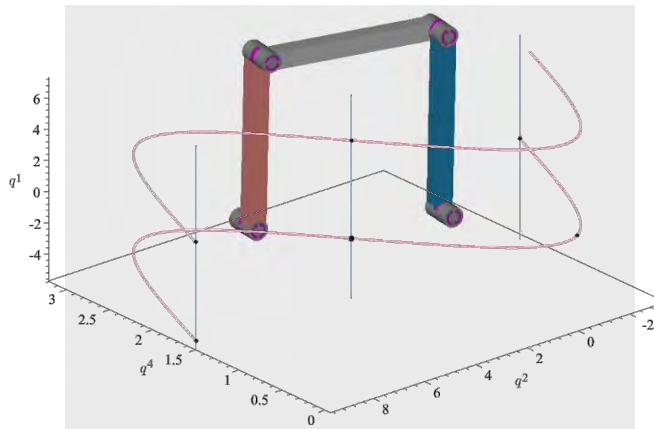


Definition 1:

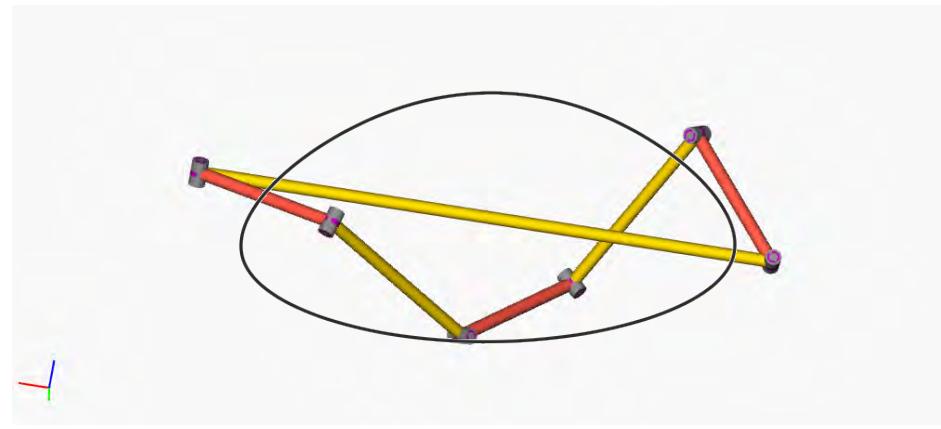
A **kinematic singularity** is a configuration $\mathbf{q} \in V$ where the differential (instantaneous) DOF is **not constant** in any neighborhood $U(\mathbf{q})$ of \mathbf{q} .

Simply: $\mathbf{q} \in V$ is singular if rank \mathbf{J} **changes** at $\mathbf{q} \in V$

V has bifurcation points



V is smooth

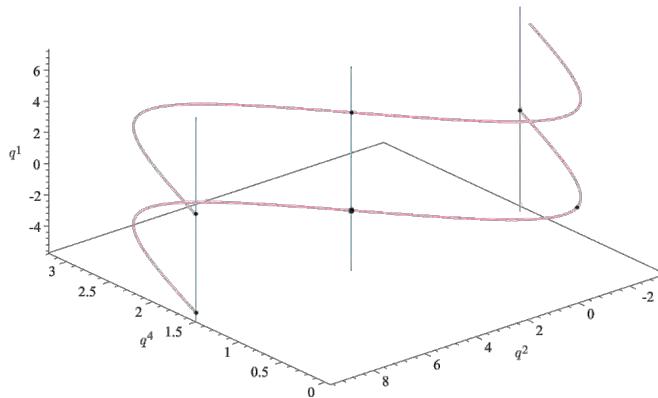


Definition 2:

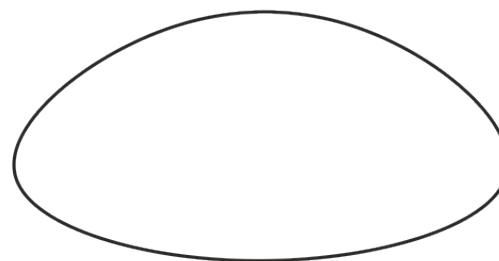
A **C-space singularity** is a configuration $\mathbf{q} \in V$ where V is **not** a smooth manifold.

- This is a topological property of V .
- C-space singularity → **kinematic singularity**

V has bifurcation points

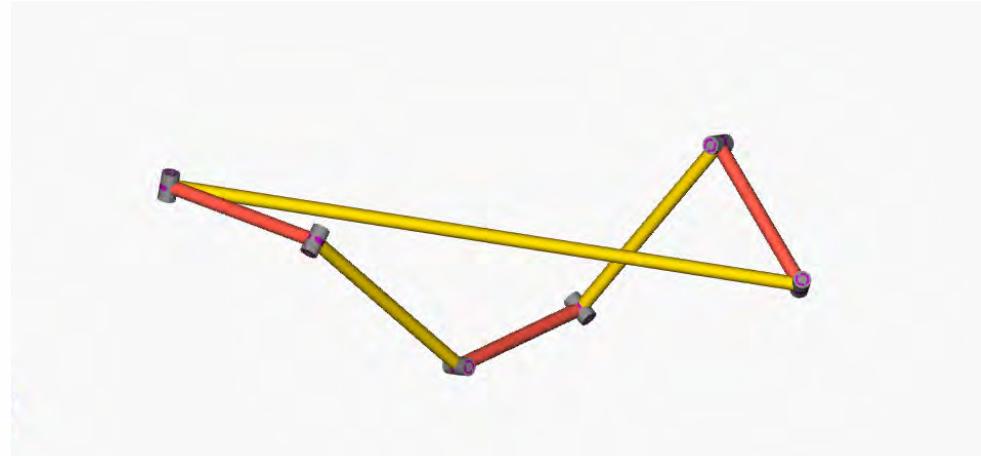
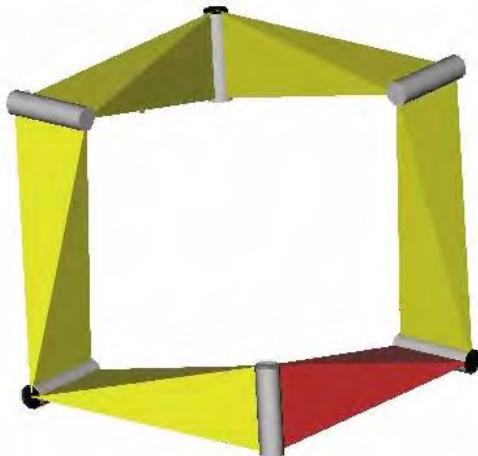


V is smooth

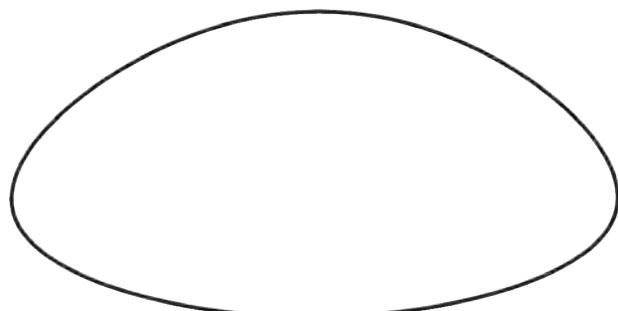


What are Singularities?

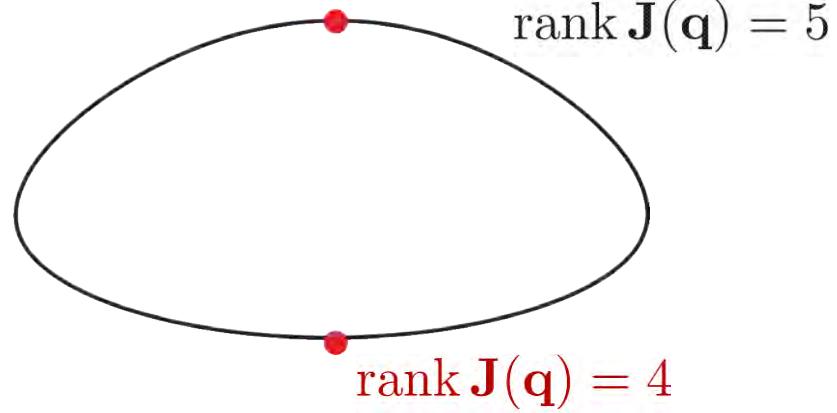
- What if \mathbf{J} is singular all the time?
- Then we have an **overconstrained** linkage.

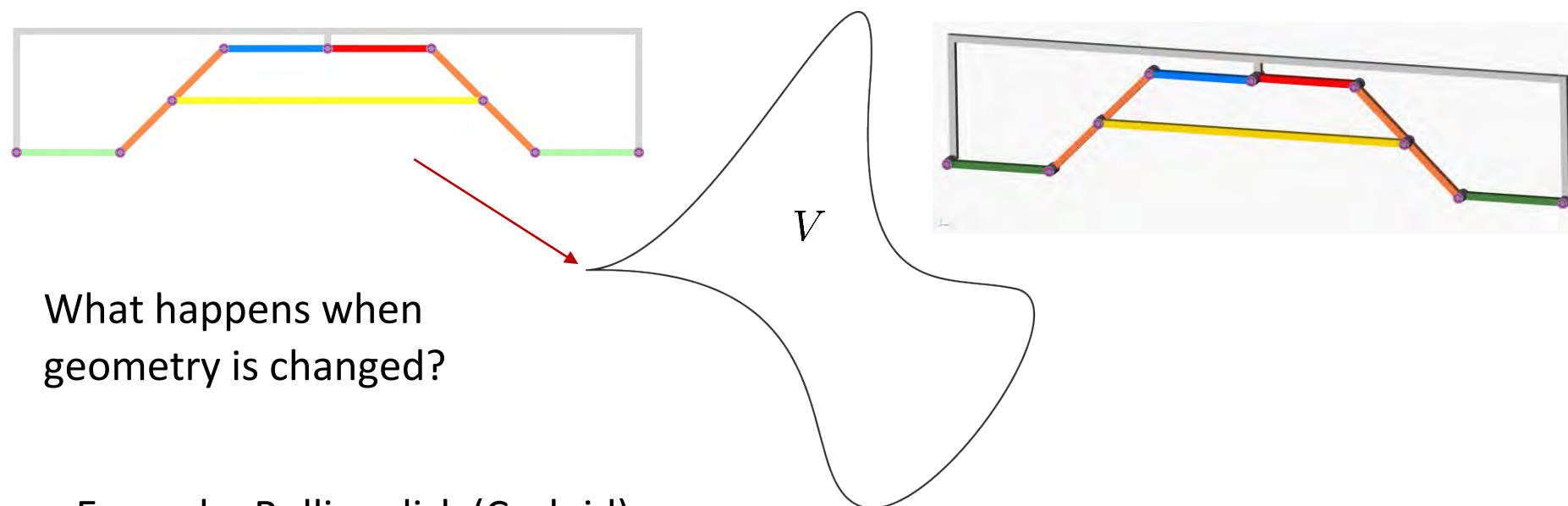


↙



$\text{rank } \mathbf{J}(\mathbf{q}) = 5 \quad \forall \mathbf{q} \in V$





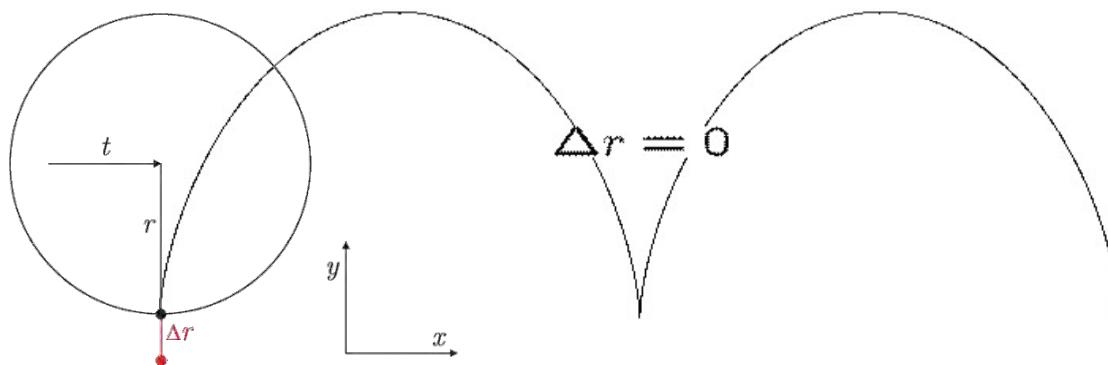
What happens when
geometry is changed?

- Example: Rolling disk (Cycloid)

$$f : \mathbb{R} \rightarrow \mathbb{R}^2 \\ t \mapsto (x, y)$$

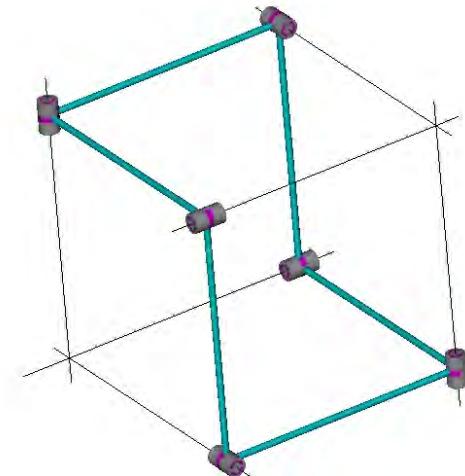
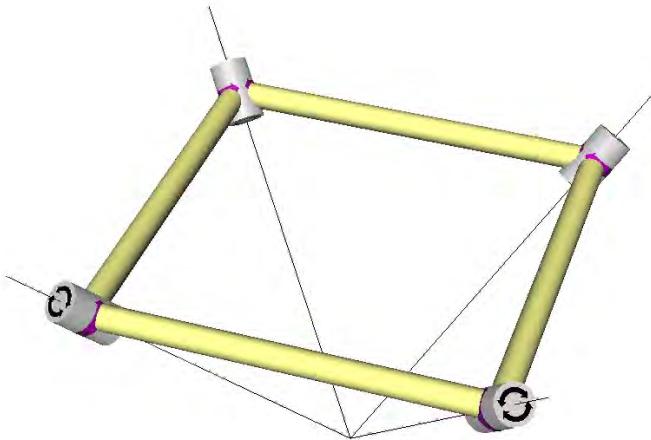
$$F_f : \mathbb{R} \times \mathbb{R} \rightarrow \mathbb{R}^2 \\ (t, \Delta r) \mapsto (x, y)$$

Deformation of the mapping
(homotopy)

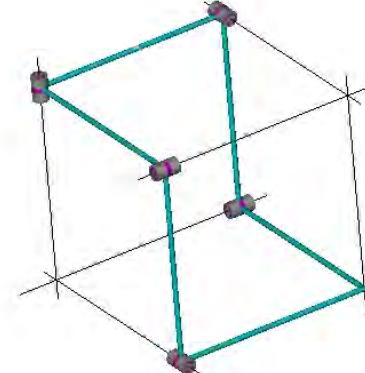
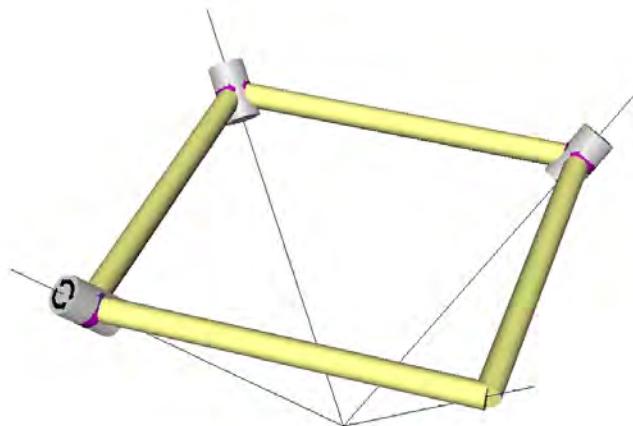


- Cusp is not ‘stable’
- Bifurcation is ‘stable’

How likely are overconstrained linkages?



✓



✗

Tasks:

- Determine the finite local DOF and mobility
 - Determine feasible finite motions
 - Approximate local c-space geometry
- Determine kinematic singularities
 - Check whether diff. DOF is constant in a neighborhood
 - Stratification of V by (co)rank

Methods:

- Screws, Lie-groups
- Higher-order analysis
- Tangent cones
- Distributions

1. Phenomenology

- Mobility
- Constraint-, C-Space-, Input-, Output-Singularities
- A Model for the Mechanism Kinematics

2. Kinematics Modeling

- Screws and the Product-of-Exponentials
- Geometric Jacobian
- Higher-Order Kinematics

3. Local analysis of C-Space and ‘singular set’

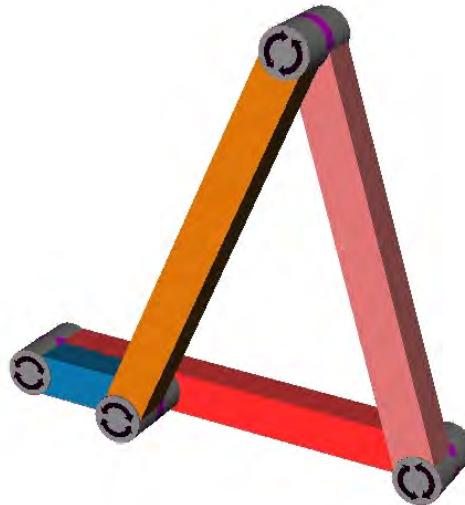
- Tangent space, Tangent cone
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4. Further Topics

- Singularities of Non-Holonomic Systems
- Escapement from singularities
- Combinatorial algorithm: Pebble Game

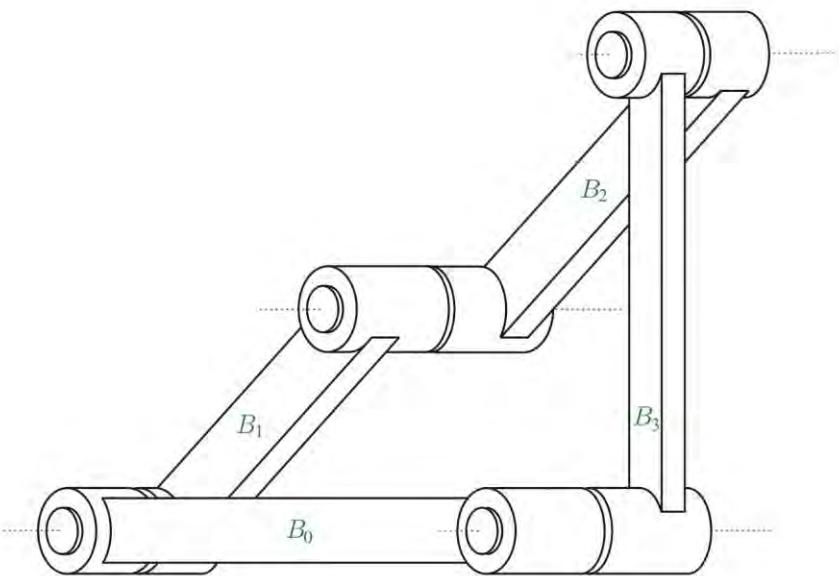
Definition [IFToMM]:

A *Mechanism* is a **system of bodies** designed to convert **motions of one or several bodies** into constrained **motions of other bodies**.



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1. System of constrained bodies

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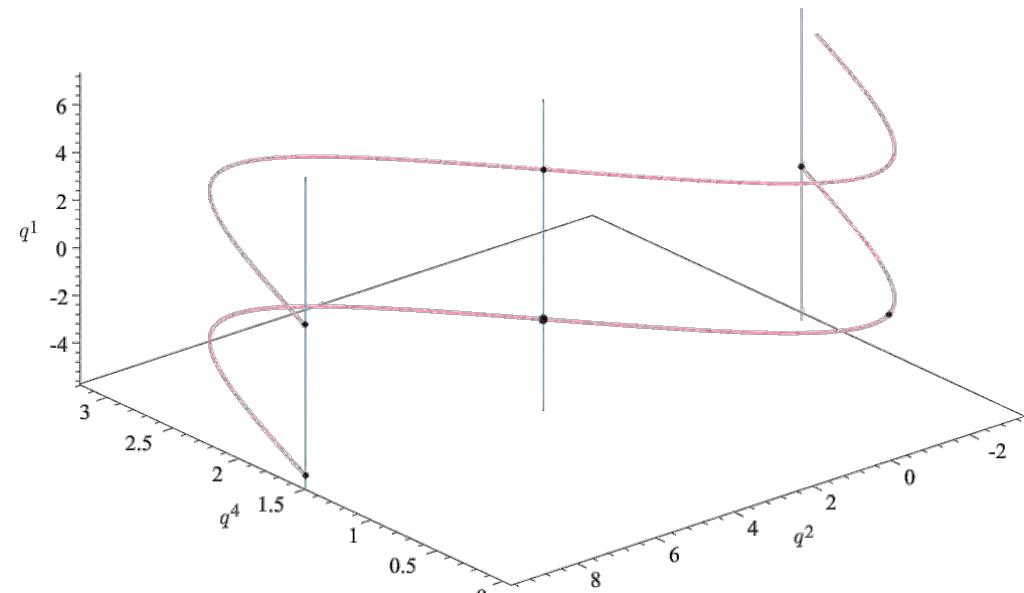
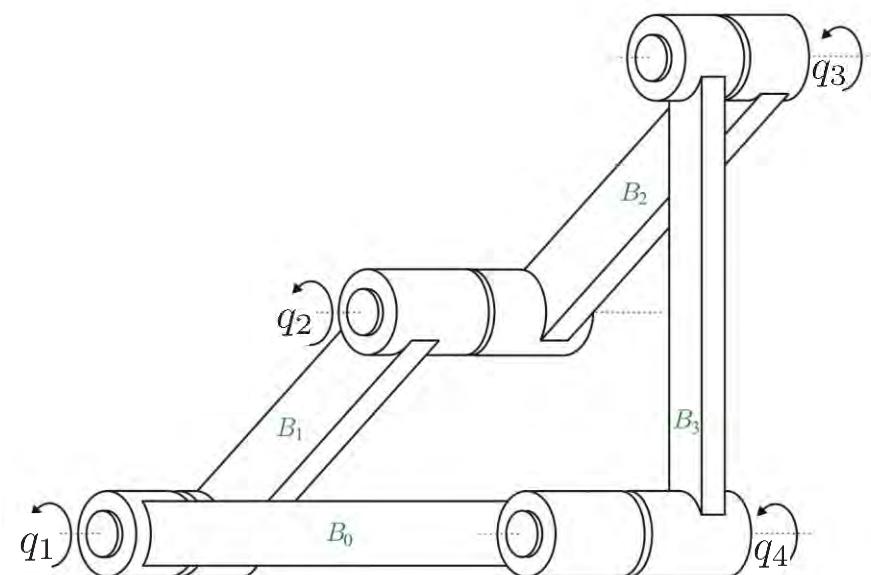
A *Mechanism* is a system of bodies designed to convert *motions* of one or several bodies into constrained *motions of other bodies*.

1. System of constrained bodies

- Joint variables: $\mathbf{q} \in \mathbb{V}^n$
- Constraints: $h(\mathbf{q}) = \mathbf{0}$
- Constraint mapping: $h : \mathbb{V}^n \rightarrow \mathbb{R}^r$

Configuration space:

$$V := \{\mathbf{q} \in \mathbb{V}^n \mid h(\mathbf{q}) = \mathbf{0}\}$$

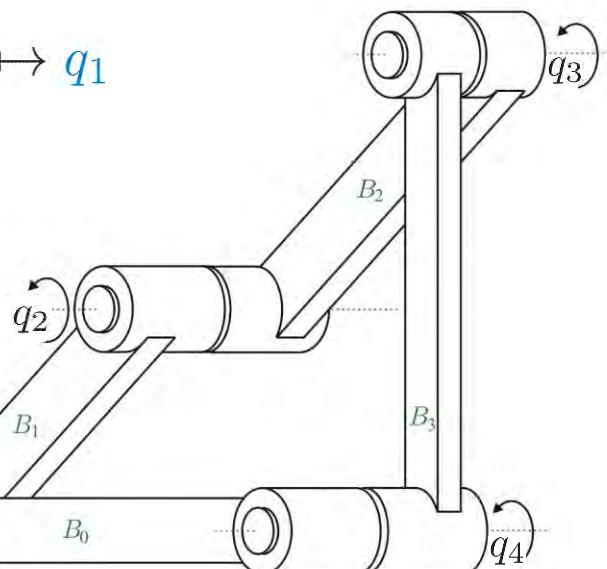


Definition [IFToMM]:

A *Mechanism* is a **system of bodies** designed to convert motions of one or several bodies into constrained **motions of other bodies**.

1. System of constrained bodies**2. Input motion**

$$f_I : \mathbf{q} \mapsto q_1$$



Configuration space:

$$V := \{\mathbf{q} \in \mathbb{V}^n | h(\mathbf{q}) = \mathbf{0}\}$$

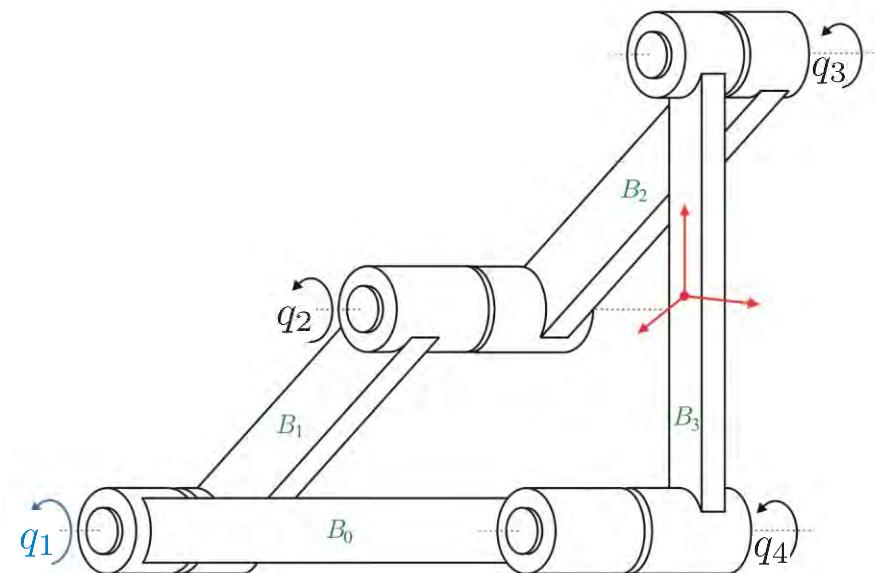
Input mapping:

$$f_I : V \rightarrow \mathbb{R}^m$$

Definition [IFToMM]:

A *Mechanism* is a **system of bodies** designed to convert **motions** of one or several bodies into constrained **motions of other bodies**.

1. System of constrained bodies
2. Input motion
3. Output motion



Configuration space:

$$V := \{\mathbf{q} \in \mathbb{V}^n | h(\mathbf{q}) = \mathbf{0}\}$$

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$$f_I : V \rightarrow \mathbb{R}^m$$

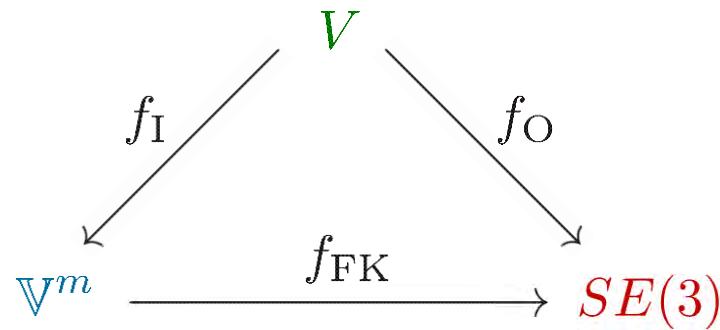
Output mapping:

$$f_O : V \rightarrow SE(3)$$

Configuration space

$$V := h^{-1}(\mathbf{0})$$

- admissible configurations
- analytic variety
- possessing singularities
- intrinsic to the kinematics

Input space

$$\mathcal{I} := f_I(V) \subset \mathbb{V}^m$$

- admissible inputs
- not necessarily a subspace of \mathbb{V}^n

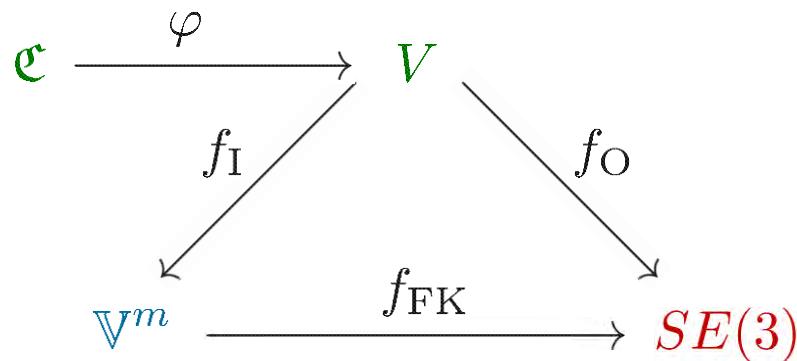
Workspace

$$\mathcal{W} := f_O(V) \subset SE(3)$$

- admissible outputs
- possible not connected

Configuration space

- Abstract representation of configuration
- Coordinate free



Parameter space

Configuration space

$$V := h^{-1}(\mathbf{0})$$

- admissible configurations
- analytic variety
- possessing singularities
- intrinsic to the kinematics

Input space

$$\mathcal{I} := f_I(V) \subset \mathbb{V}^m$$

- admissible inputs
- not necessarily a subspace of \mathbb{V}^n

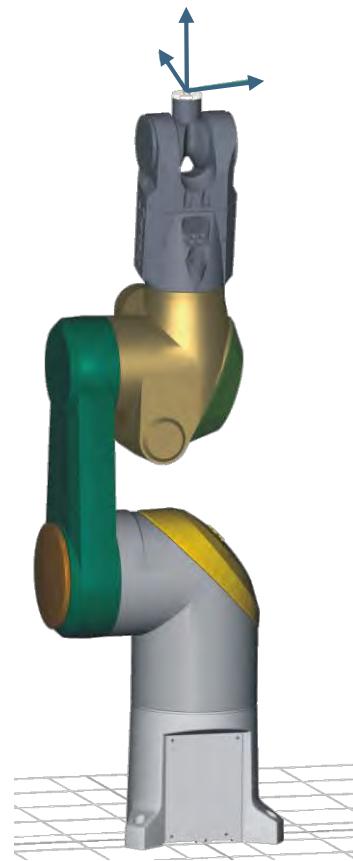
Workspace

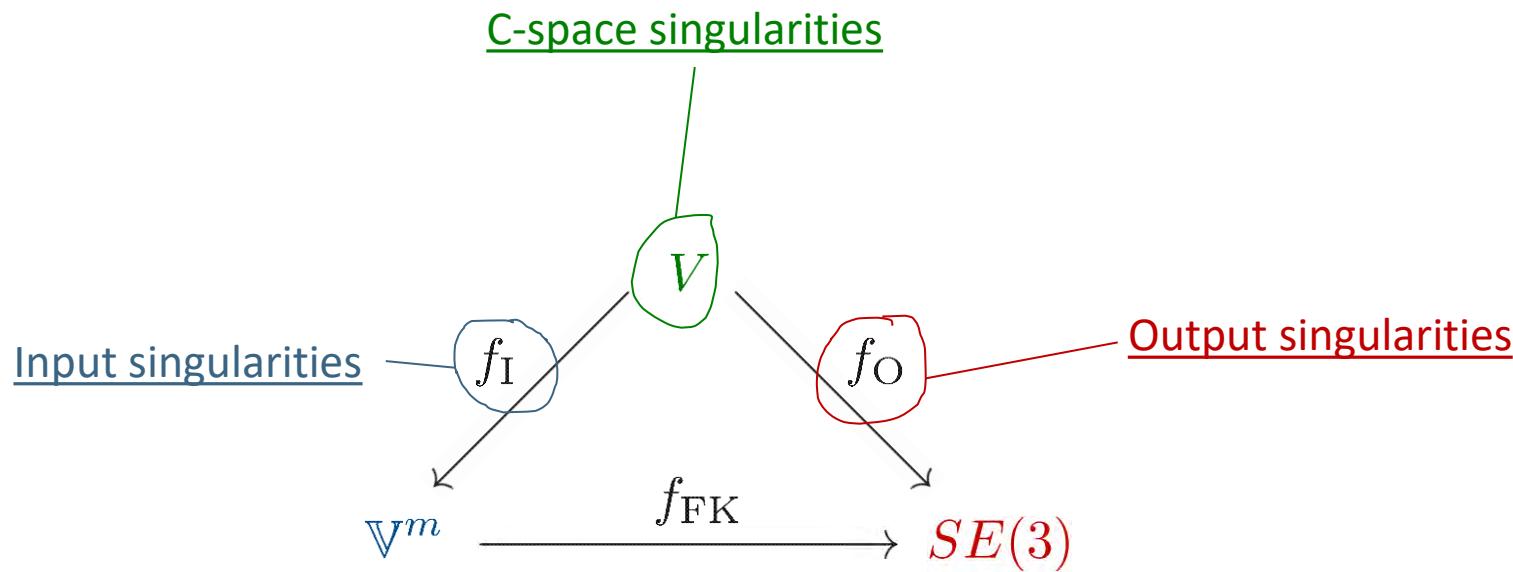
$$\mathcal{W} := f_O(V) \subset SE(3)$$

- admissible outputs
- possible not connected

Special case: serial manipulator

$$\mathcal{I} = V \xrightarrow{f_O = f_{FK}} SE(3)$$





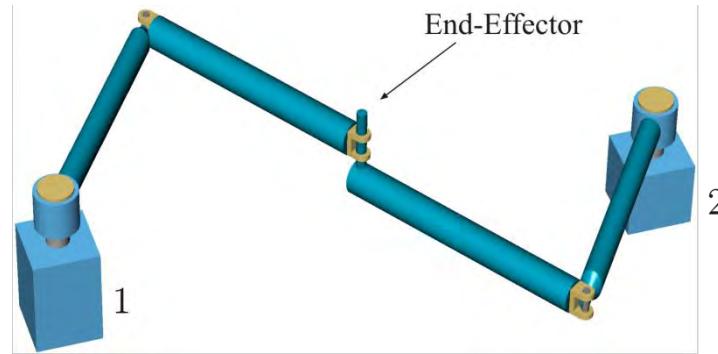
$$\mathbf{0} = h(\mathbf{q}) \quad \Rightarrow \quad V = h^{-1}(\mathbf{0}) \quad (\text{analytic, algebraic variety})$$

$$\boldsymbol{\theta} = f_I(\mathbf{q})$$

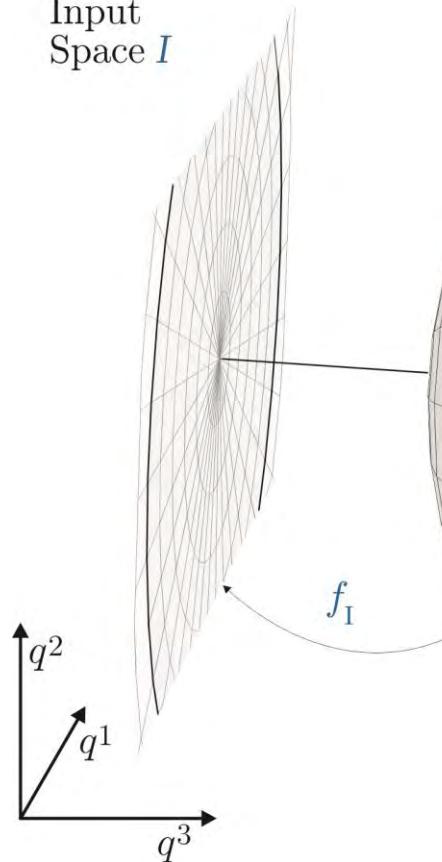
$$\mathbf{C} = \underbrace{f_O(\mathbf{q})}_{\text{analytic mappings}}$$

Sard's theorem:

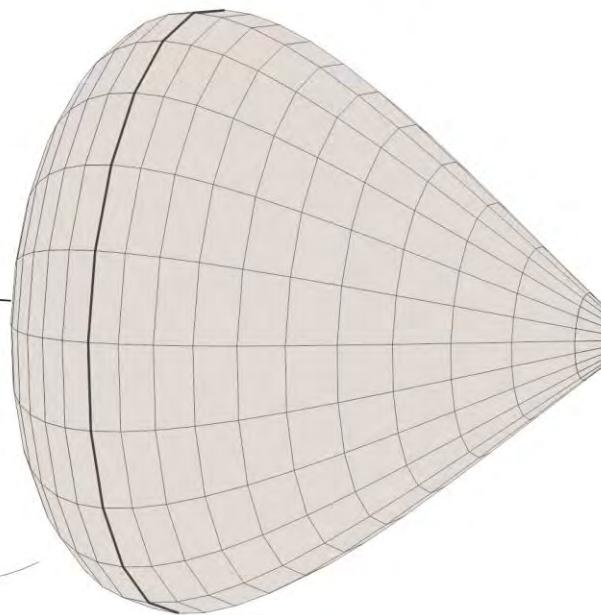
Let $f : M \rightarrow N$ be a smooth map of manifolds M and N , where only N has boundary. Almost all points of N are regular values of $f : M \rightarrow N$ and $\partial f : \partial M \rightarrow N$. That is, the set of critical values of f has measure zero in N .



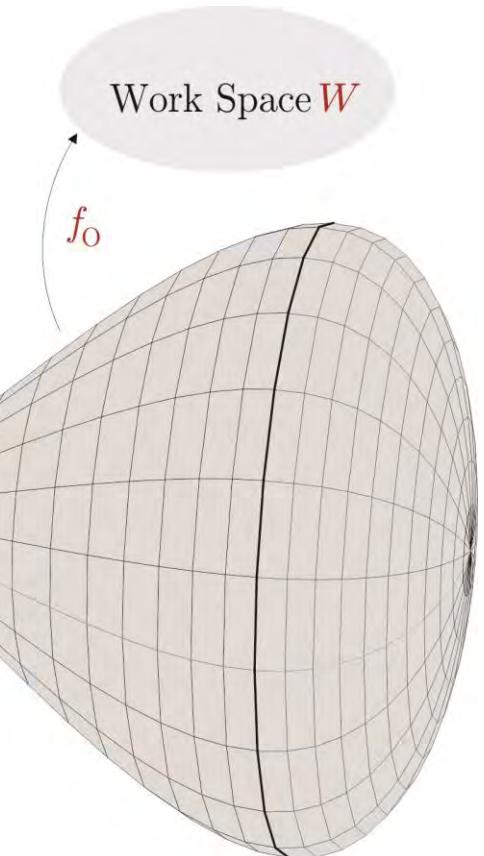
Input Space I



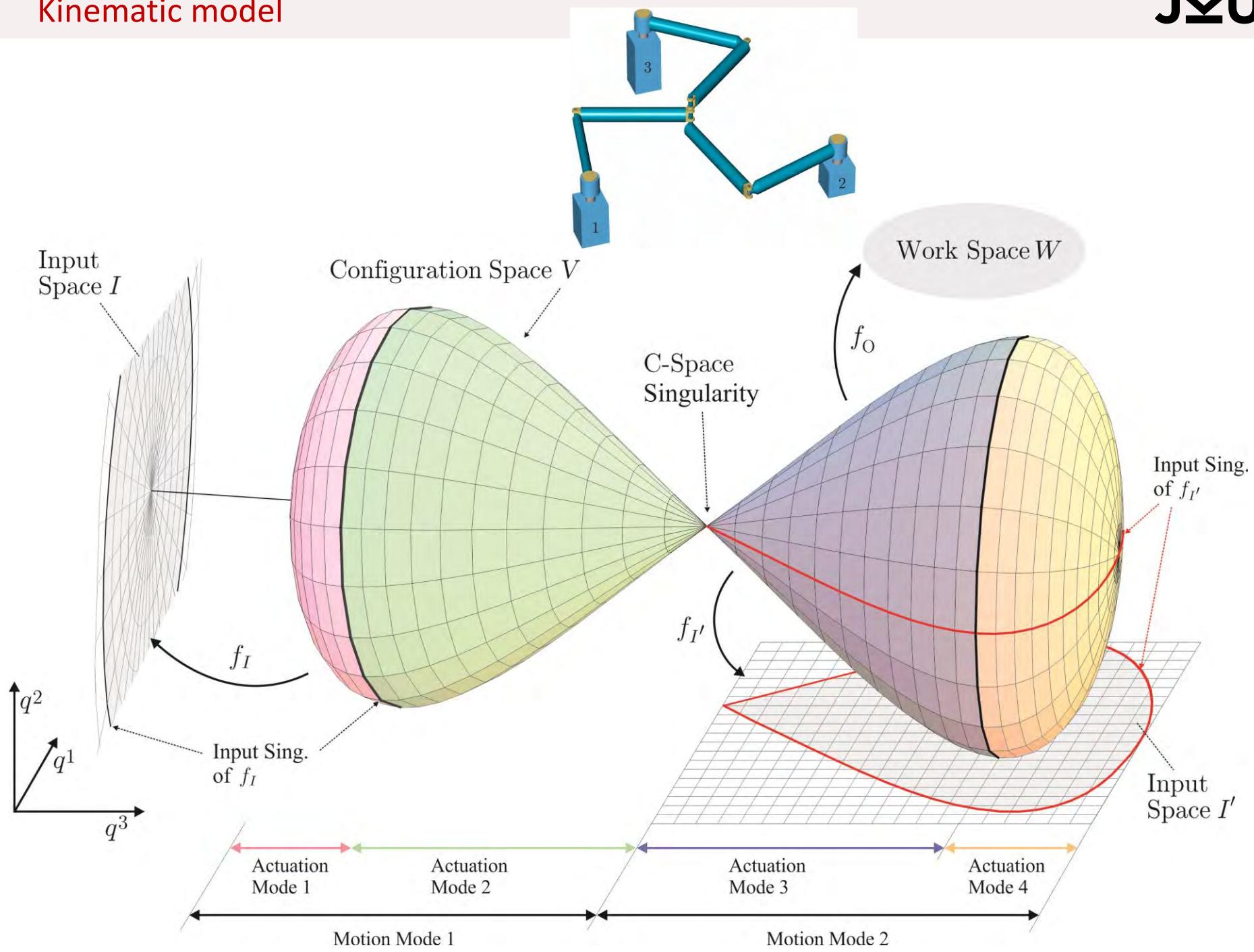
Configuration Space V

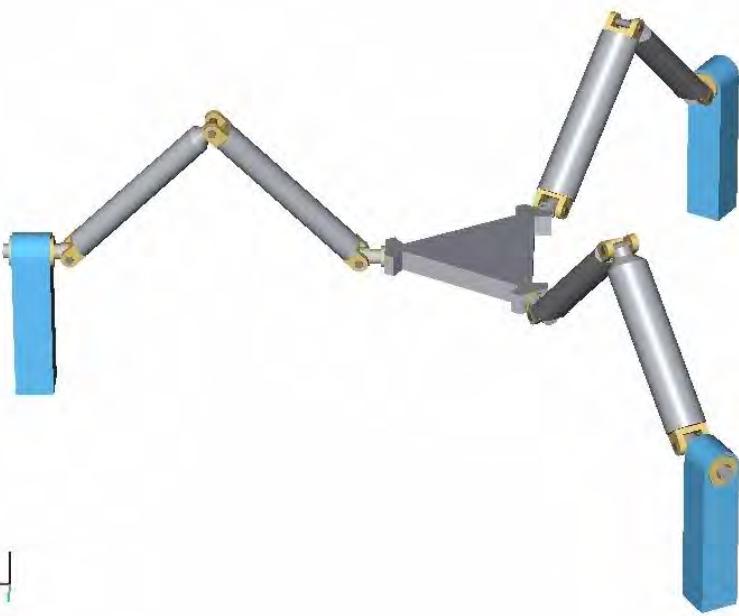


Work Space W



Kinematic model





Lock-Up mode

Planar mode

1. Phenomenology

- Mobility
- Constraint-, C-Space-, Input-, Output-Singularities
- A Model for the Mechanism Kinematics

2. Kinematics Modeling

- Screws and the Product-of-Exponentials
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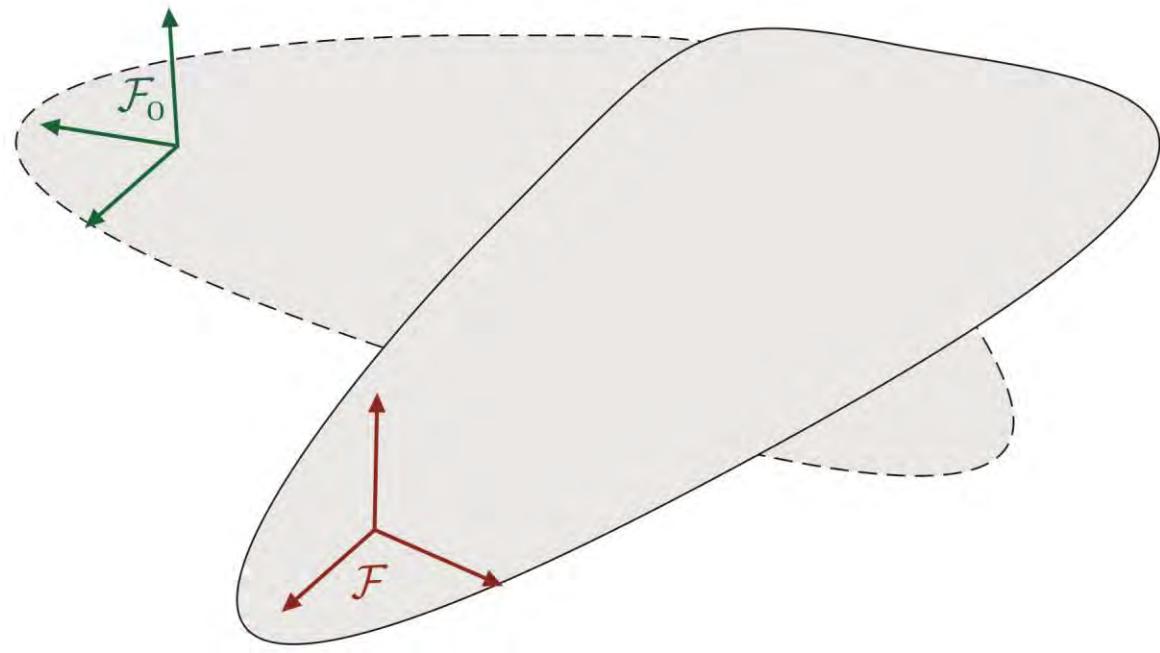
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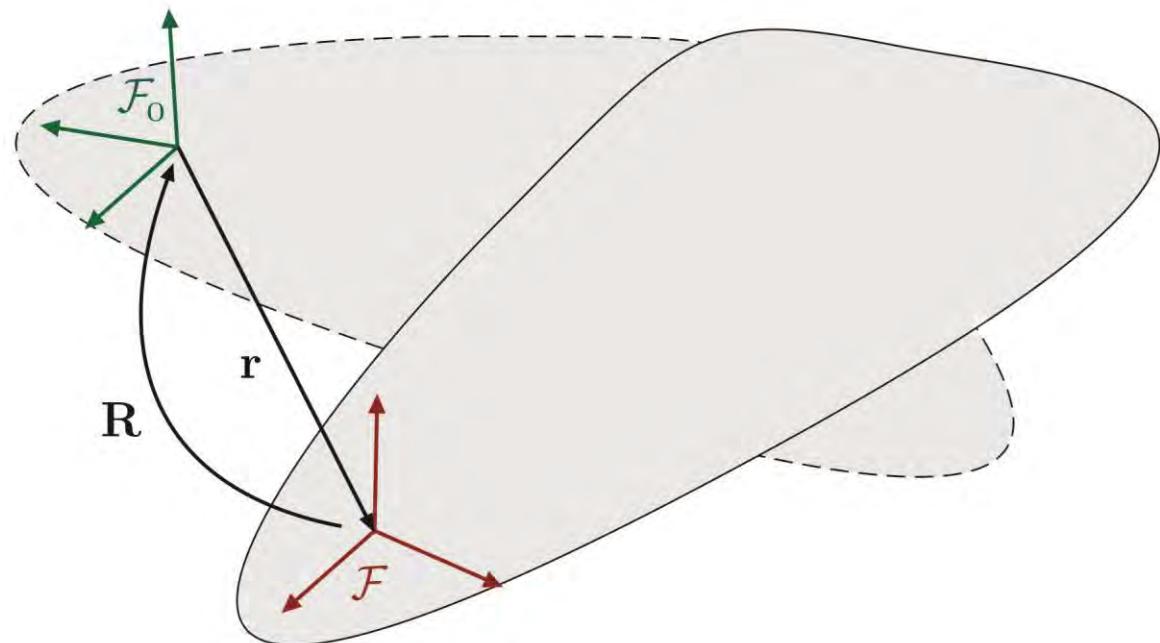
- Frame transformation $\mathcal{F} \rightarrow \mathcal{F}_0$



- Frame transformation $\mathcal{F} \rightarrow \mathcal{F}_0$

$\mathbf{R} \in SO(3)$ – rotation

$\mathbf{r} \in \mathbb{R}^3$ – translation

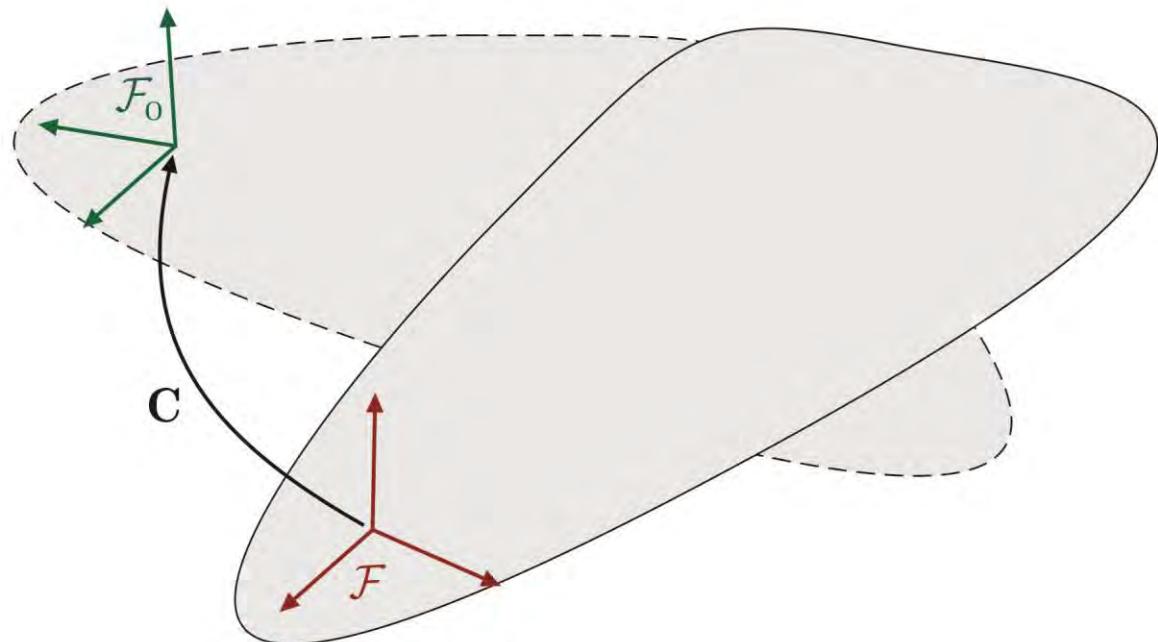


- Frame transformation $\mathcal{F} \rightarrow \mathcal{F}_0$

Configuration of \mathcal{F} : $\mathbf{C} = \begin{pmatrix} \mathbf{R} & \mathbf{r} \\ \mathbf{0} & 1 \end{pmatrix} \in SE(3)$

'Homogenous matrix'

$$SE(3) = SO(3) \ltimes \mathbb{R}^3$$



Chasles' theorem:

Any **rigid body displacement** can be achieved by a **screw motion**, i.e. a rotation about a constant axis together with a translation along this axis.

Mozzi-Cauchy theorem:

For any **rigid body motion** there exists an instantaneous screw axis.

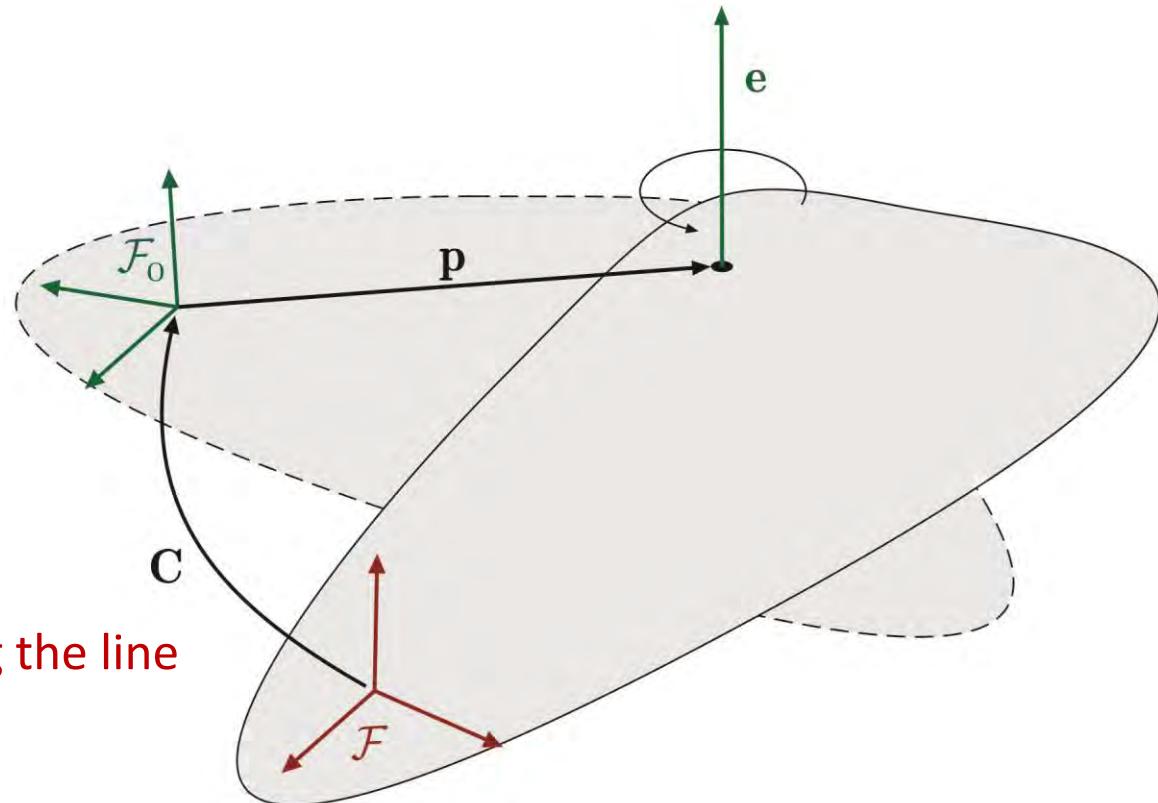
Screw coordinates:

$$\mathbf{X} = \begin{pmatrix} \mathbf{e} \\ \mathbf{p} \times \mathbf{e} + h\mathbf{e} \end{pmatrix}$$

$$= \begin{pmatrix} \mathbf{e} \\ \mathbf{p} \times \mathbf{e} \end{pmatrix} + \begin{pmatrix} 0 \\ h\mathbf{e} \end{pmatrix}$$

Translation along the line

Plücker line coordinates



- Rigid body motion in terms of screw coordinates: $\mathbf{C} = \exp(\varphi \mathbf{X})$

$$\exp(\varphi \mathbf{X}) = \begin{pmatrix} \mathbf{R} & \mathbf{r} \\ \mathbf{0} & 1 \end{pmatrix}$$

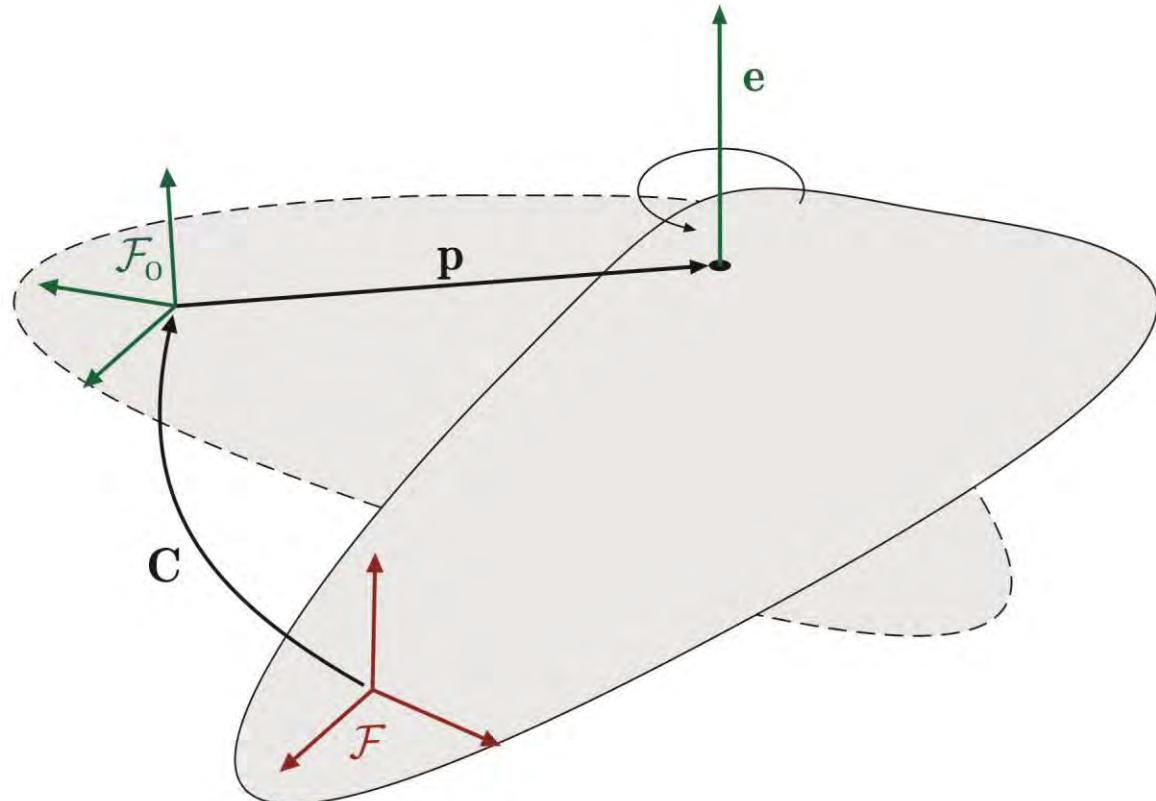
$$\mathbf{R} = \exp(\varphi \tilde{\mathbf{e}})$$

$$\mathbf{r} = (\mathbf{I} - \mathbf{R})\mathbf{p} + \varphi h \mathbf{e}$$

Screw coordinates:

$$\mathbf{X} = \begin{pmatrix} \mathbf{e} \\ \mathbf{p} \times \mathbf{e} + h \mathbf{e} \end{pmatrix}$$

- Unit vector along screw axis: \mathbf{e}
- Position vector to any point on the axis: \mathbf{p}
- Pitch: h



- Rigid body motion in terms of screw coordinates: $\mathbf{C} = \exp(\varphi \mathbf{X})$

$$\exp(\varphi \mathbf{X}) = \begin{pmatrix} \mathbf{R} & \mathbf{r} \\ \mathbf{0} & 1 \end{pmatrix}$$

$$\mathbf{R} = \exp(\varphi \tilde{\mathbf{e}})$$

$$\mathbf{r} = (\mathbf{I} - \mathbf{R})\mathbf{p} + \varphi h \mathbf{e}$$

Screw coordinates:

$$\mathbf{X} = \begin{pmatrix} \mathbf{e} \\ \mathbf{p} \times \mathbf{e} + h \mathbf{e} \end{pmatrix}$$

$$\exp(\varphi \tilde{\mathbf{e}}) = \mathbf{I} + \sin \varphi \tilde{\mathbf{e}} + (1 - \cos \varphi) \tilde{\mathbf{e}}^2$$

- Unit vector along screw axis: \mathbf{e}

Euler-Rodrigues formula

- Position vector to any point on the axis: \mathbf{p}
- Pitch: h

- Rigid body motion in terms of screw coordinates: $\mathbf{C} = \exp(\varphi \mathbf{X})$

$$\exp(\varphi \mathbf{X}) = \begin{pmatrix} \mathbf{R} & \mathbf{r} \\ \mathbf{0} & 1 \end{pmatrix} \quad \mathbf{R} = \exp(\varphi \tilde{\mathbf{e}}) \quad \mathbf{r} = (\mathbf{I} - \mathbf{R})\mathbf{p} + \varphi h \mathbf{e}$$

Why exp mapping?

$$\mathbf{e} := \begin{pmatrix} e_1 \\ e_2 \\ e_3 \end{pmatrix} \in \mathbb{R}^3 \quad \xrightarrow{} \quad \tilde{\mathbf{e}} := \begin{pmatrix} 0 & -e_3 & e_2 \\ e_3 & 0 & -e_1 \\ -e_2 & e_1 & 0 \end{pmatrix} \in so(3)$$

$$\begin{aligned} \exp(\varphi \tilde{\mathbf{e}}) &= \mathbf{I} + \tilde{\mathbf{e}}\varphi + \frac{1}{2}\tilde{\mathbf{e}}^2\varphi^2 + \frac{1}{3!}\tilde{\mathbf{e}}^3\varphi^3 + \frac{1}{4!}\tilde{\mathbf{e}}^4\varphi^4 + \dots \\ &= \mathbf{I} + \sin \varphi \tilde{\mathbf{e}} + (1 - \cos \varphi) \tilde{\mathbf{e}}^2 \end{aligned}$$

- Remember the nice property of the exp mapping!

- Rigid body motion in terms of screw coordinates: $\mathbf{C} = \exp(\varphi \mathbf{X})$

$$\exp(\varphi \mathbf{X}) = \begin{pmatrix} \mathbf{R} & \mathbf{r} \\ \mathbf{0} & 1 \end{pmatrix}$$

$$\mathbf{R} = \exp(\varphi \tilde{\mathbf{e}})$$

$$\mathbf{r} = (\mathbf{I} - \mathbf{R})\mathbf{p} + \varphi h \mathbf{e}$$

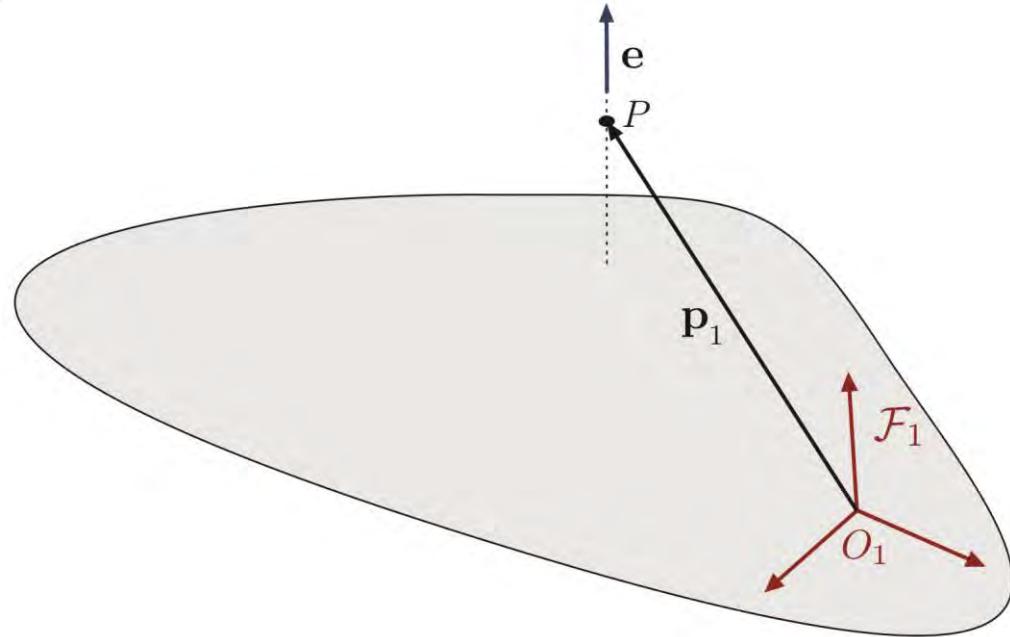
Why exp mapping?

$$\mathbf{X} = \begin{pmatrix} \mathbf{e} \\ \mathbf{p} \times \mathbf{e} + h \mathbf{e} \end{pmatrix} =: \begin{pmatrix} \boldsymbol{\xi} \\ \boldsymbol{\eta} \end{pmatrix} \in \mathbb{R}^6 \quad \Rightarrow \quad \widehat{\mathbf{X}} := \begin{pmatrix} \tilde{\boldsymbol{\xi}} & \boldsymbol{\eta} \\ \mathbf{0} & 0 \end{pmatrix} \in se(3)$$

$$\exp(\varphi \mathbf{X}) = \mathbf{I} + \widehat{\mathbf{X}}\varphi + \frac{1}{2}\widehat{\mathbf{X}}^2\varphi^2 + \frac{1}{3!}\widehat{\mathbf{X}}^3\varphi^3 + \frac{1}{4!}\widehat{\mathbf{X}}^4\varphi^4 + \dots$$

- Screw coordinates for motion of \mathcal{F}_1

$${}^1\mathbf{X} = \left({}^1\mathbf{p}_1 \times {}^1\mathbf{e} + {}^1\mathbf{e}h \right)$$



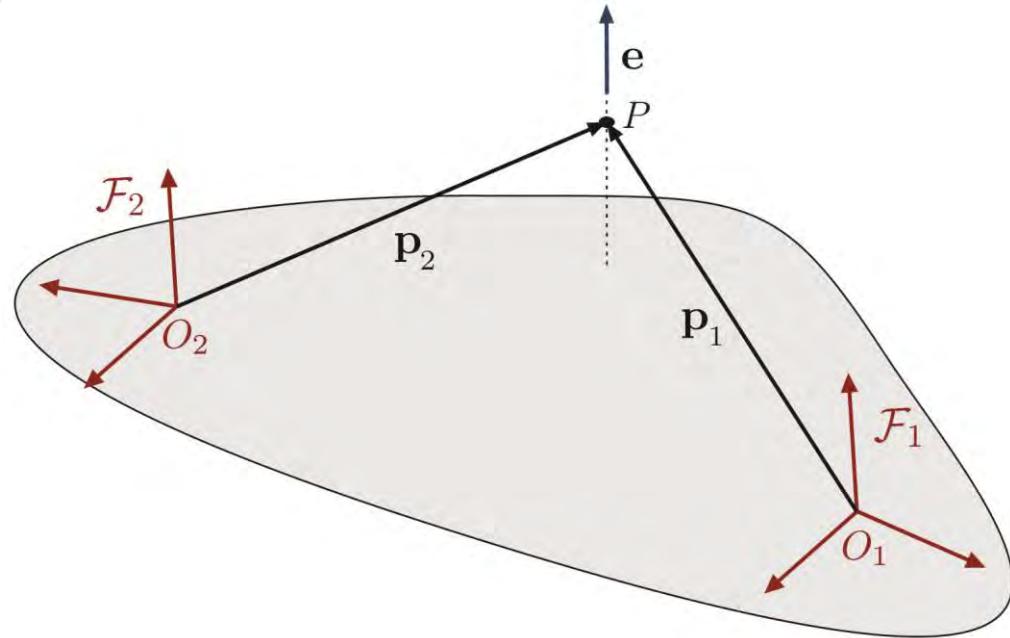
${}^1\mathbf{e}, {}^1\mathbf{p}_1$ – Coordinate vectors
resolved in frame \mathcal{F}_1

- Screw coordinates for motion of \mathcal{F}_1

$${}^1\mathbf{X} = \left({}^1\mathbf{p}_1 \times {}^1\mathbf{e} + {}^1\mathbf{e}h \right)$$

- Screw coordinates for motion of \mathcal{F}_2

$${}^2\mathbf{X} = \left({}^2\mathbf{p}_2 \times {}^2\mathbf{e} + {}^2\mathbf{e}h \right)$$



${}^1\mathbf{e}, {}^1\mathbf{p}_1$ – Coordinate vectors resolved in frame \mathcal{F}_1

${}^2\mathbf{e}, {}^2\mathbf{p}_2$ – Coordinate vectors resolved in frame \mathcal{F}_2

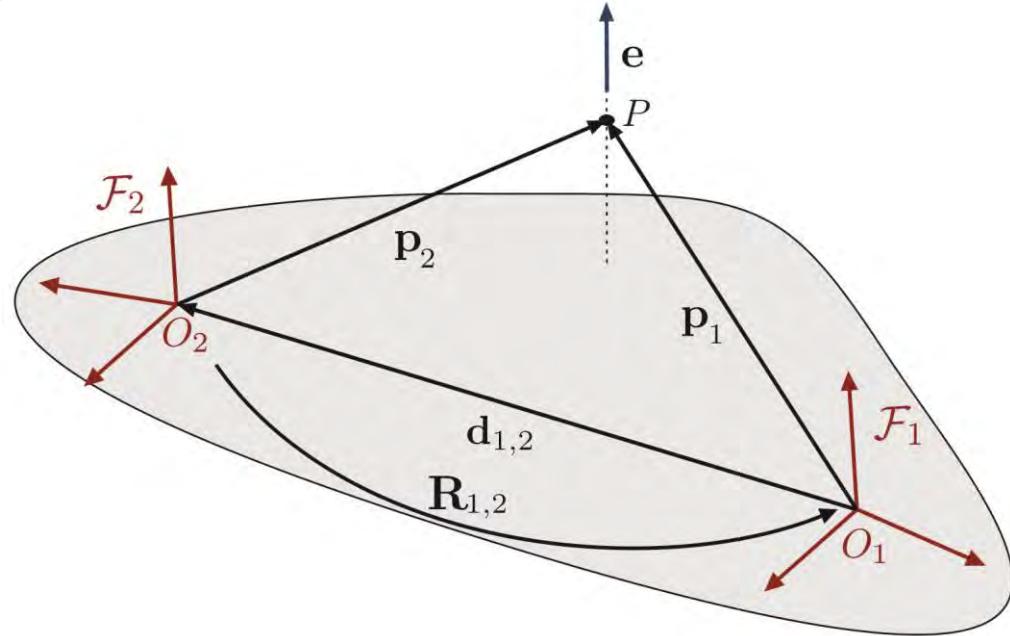
- Screw coordinates for motion of \mathcal{F}_1

$${}^1\mathbf{X} = \left({}^1\mathbf{p}_1 \times {}^1\mathbf{e} + {}^1\mathbf{e}h \right)$$

- Screw coordinates for motion of \mathcal{F}_2

$${}^2\mathbf{X} = \left({}^2\mathbf{p}_2 \times {}^2\mathbf{e} + {}^2\mathbf{e}h \right)$$

- Transformation $\mathcal{F}_2 \rightarrow \mathcal{F}_1$



$${}^1\mathbf{X} = \mathbf{Ad}_{\mathbf{S}_{1,2}} {}^2\mathbf{X} \quad \text{with } \mathbf{S}_{1,2} = \begin{pmatrix} \mathbf{R}_{1,2} & \mathbf{r}_{1,2} \\ \mathbf{0} & 1 \end{pmatrix}$$

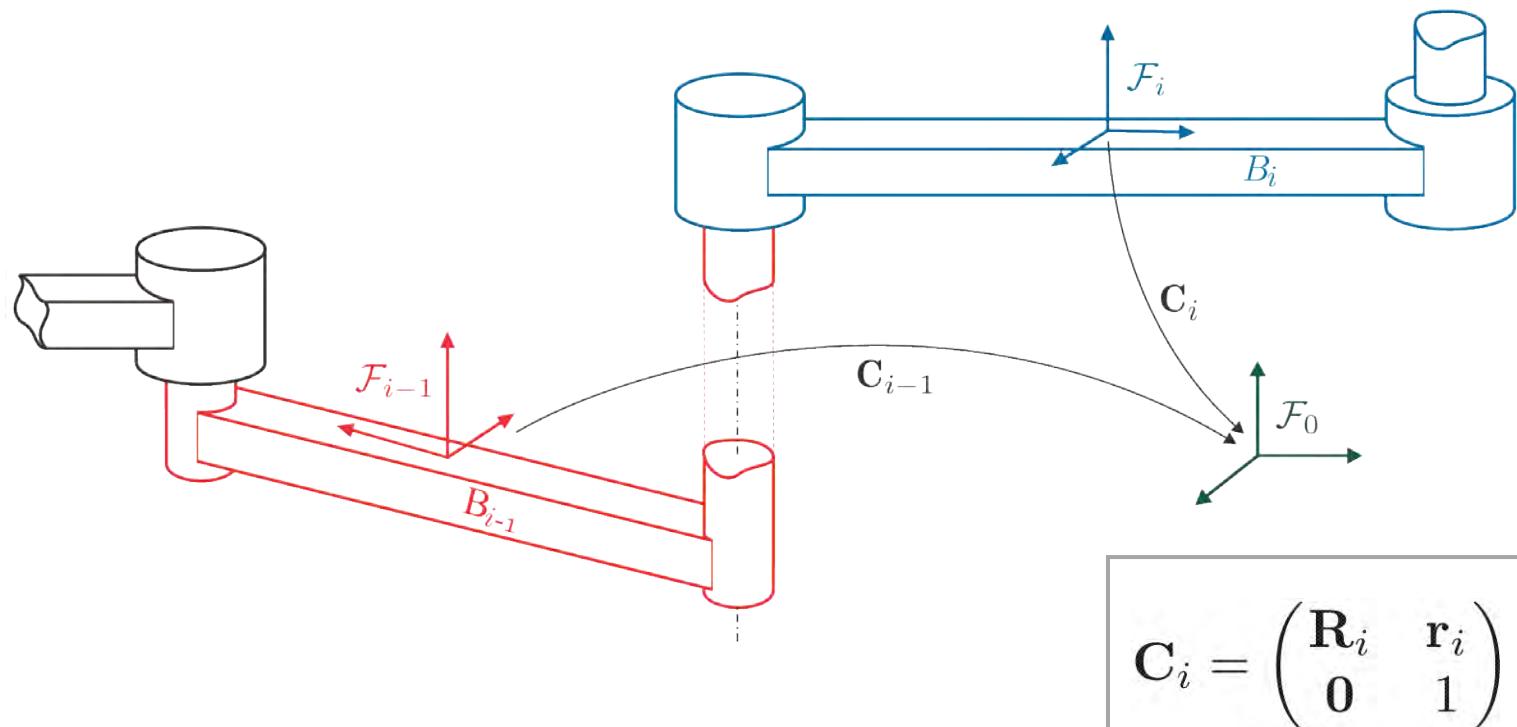
Adjoined mapping on $SE(3)$: $\mathbf{Ad}_{\mathbf{C}} = \begin{pmatrix} \mathbf{R} & \mathbf{0} \\ \tilde{\mathbf{r}}\mathbf{R} & \mathbf{R} \end{pmatrix}$ for $\mathbf{C} = \begin{pmatrix} \mathbf{R} & \mathbf{r} \\ \mathbf{0} & 1 \end{pmatrix}$

'Motion tensor'

Transforms between different representations of a screw

- Body-fixed reference frame \mathcal{F}_i at body i
- Body-fixed reference frame $\mathcal{F}_{i-1,i}$ at body i

→ Relative configuration of body i w.r.t. body $i-1$: $\mathbf{C}_{i-1,i} := \mathbf{C}_{i-1}^{-1} \mathbf{C}_i$

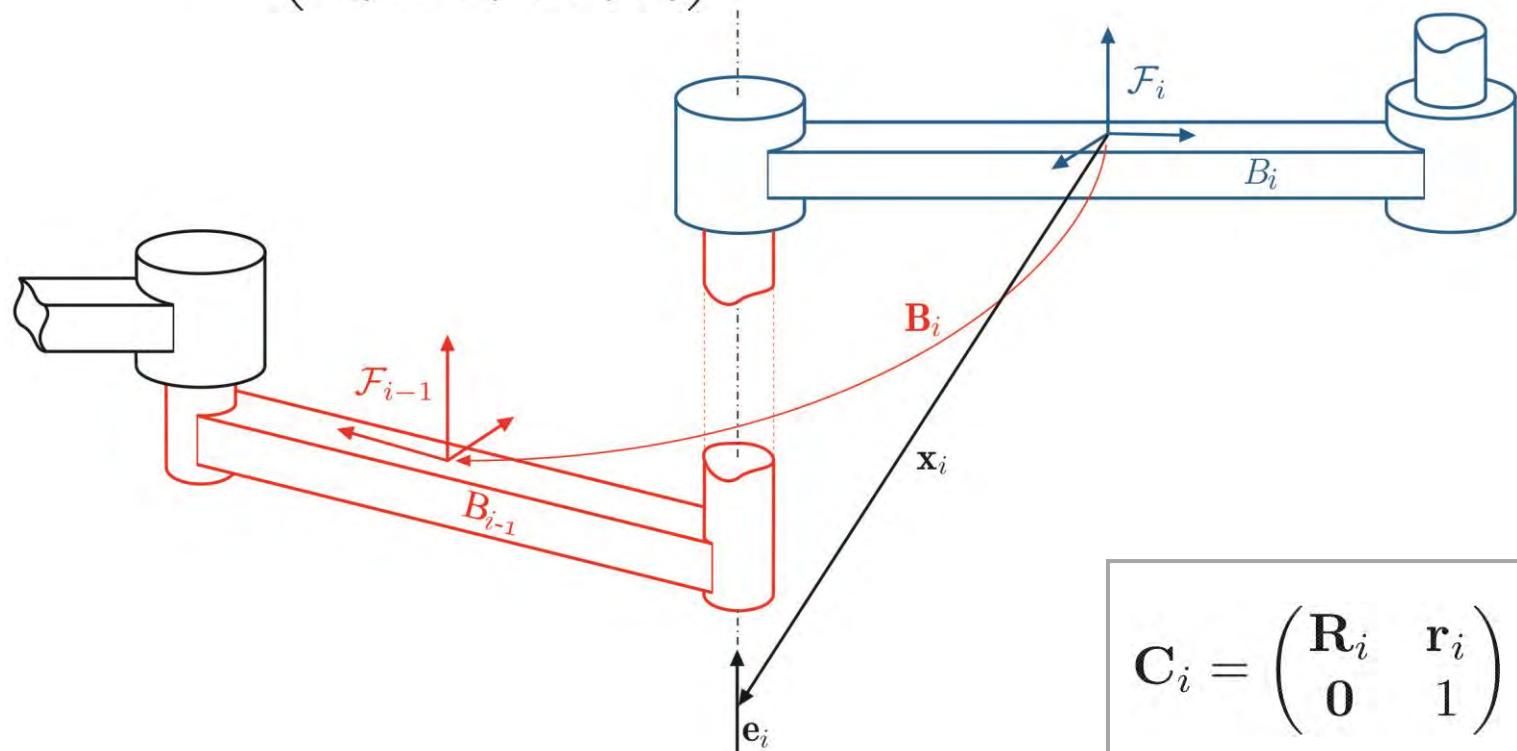


$$\mathbf{C}_i = \begin{pmatrix} \mathbf{R}_i & \mathbf{r}_i \\ \mathbf{0} & 1 \end{pmatrix}$$

$$\mathbf{C}_{i-1,i} = \mathbf{B}_i \exp({}^i\mathbf{X}_i q_i)$$

- Relative **reference** configuration for $q_i = 0$: $\mathbf{B}_i = \mathbf{C}_{i-1,i}(0)$
- Screw coordinate vector of joint i represented in \mathcal{F}_i :

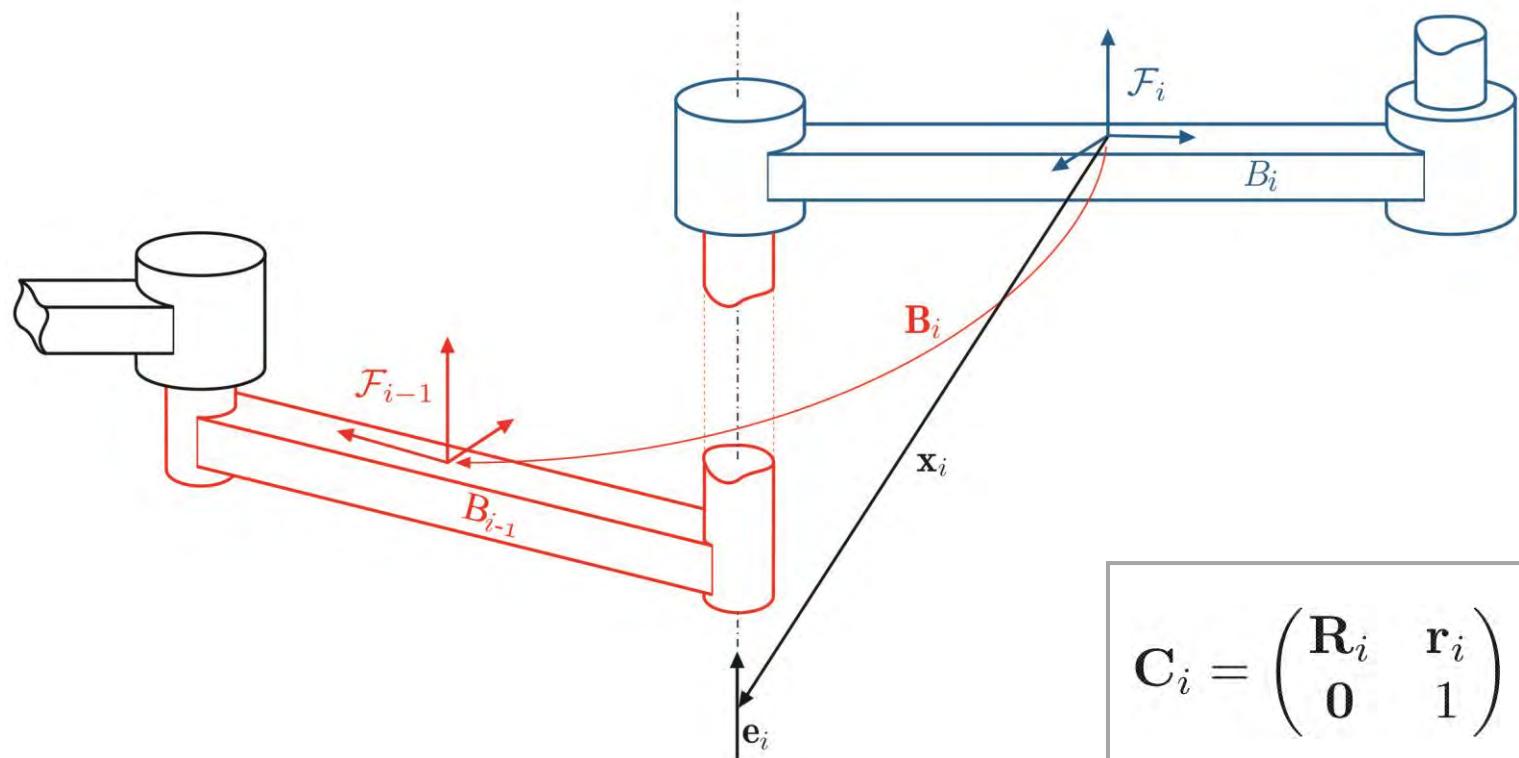
$${}^i\mathbf{X}_i = \begin{pmatrix} {}^i\mathbf{e}_i \\ {}^i\mathbf{x}_i \times {}^i\mathbf{e}_i + h_i {}^i\mathbf{e}_i \end{pmatrix}$$



$$\mathbf{C}_i = \begin{pmatrix} \mathbf{R}_i & \mathbf{r}_i \\ \mathbf{0} & 1 \end{pmatrix}$$

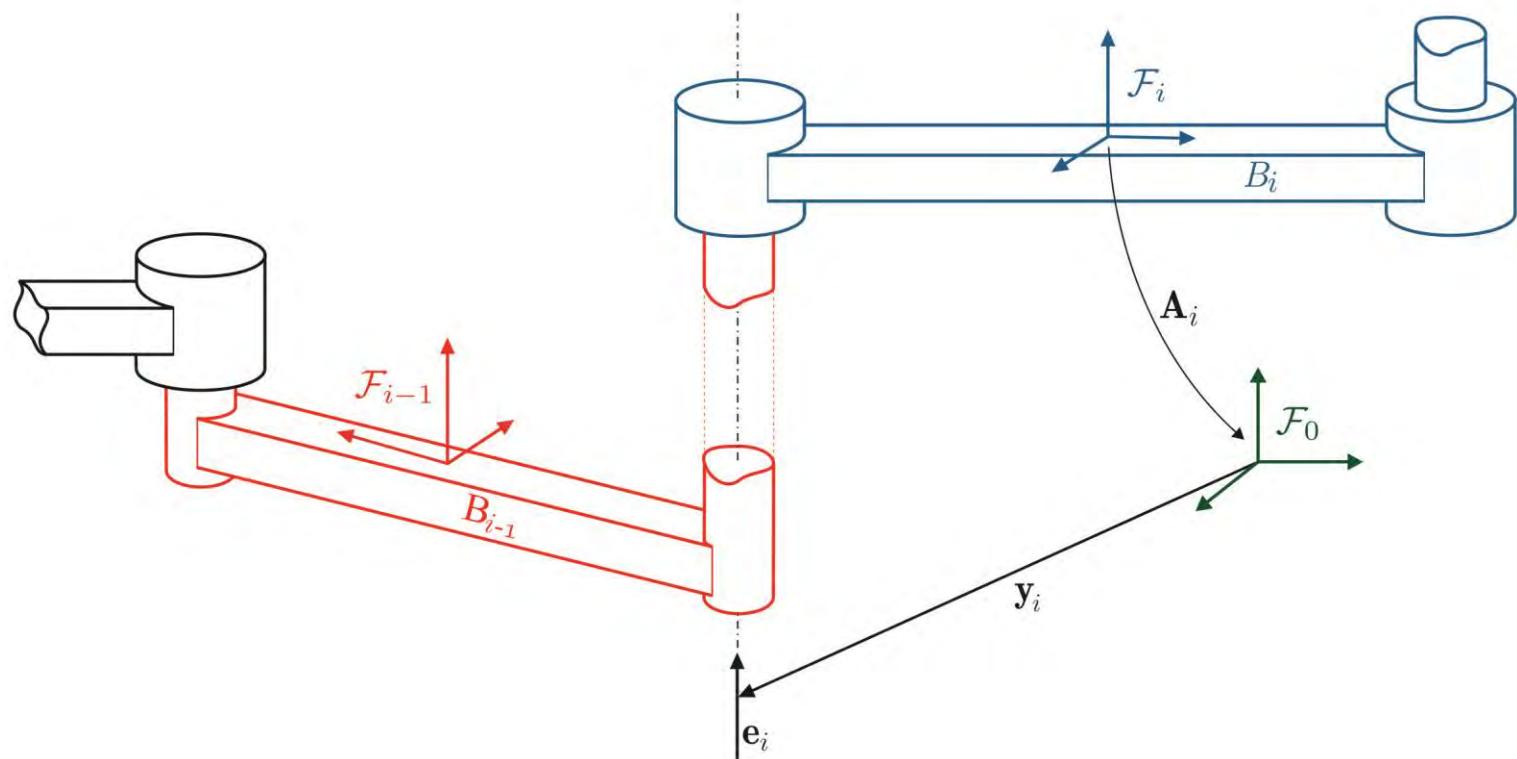
- Configuration of body i

$$\mathbf{C}_i(\mathbf{q}) = \mathbf{B}_1 \exp({}^1\mathbf{X}_1 q_1) \cdot \mathbf{B}_2 \exp({}^2\mathbf{X}_2 q_2) \cdot \dots \cdot \mathbf{B}_i \exp({}^i\mathbf{X}_i q_i)$$



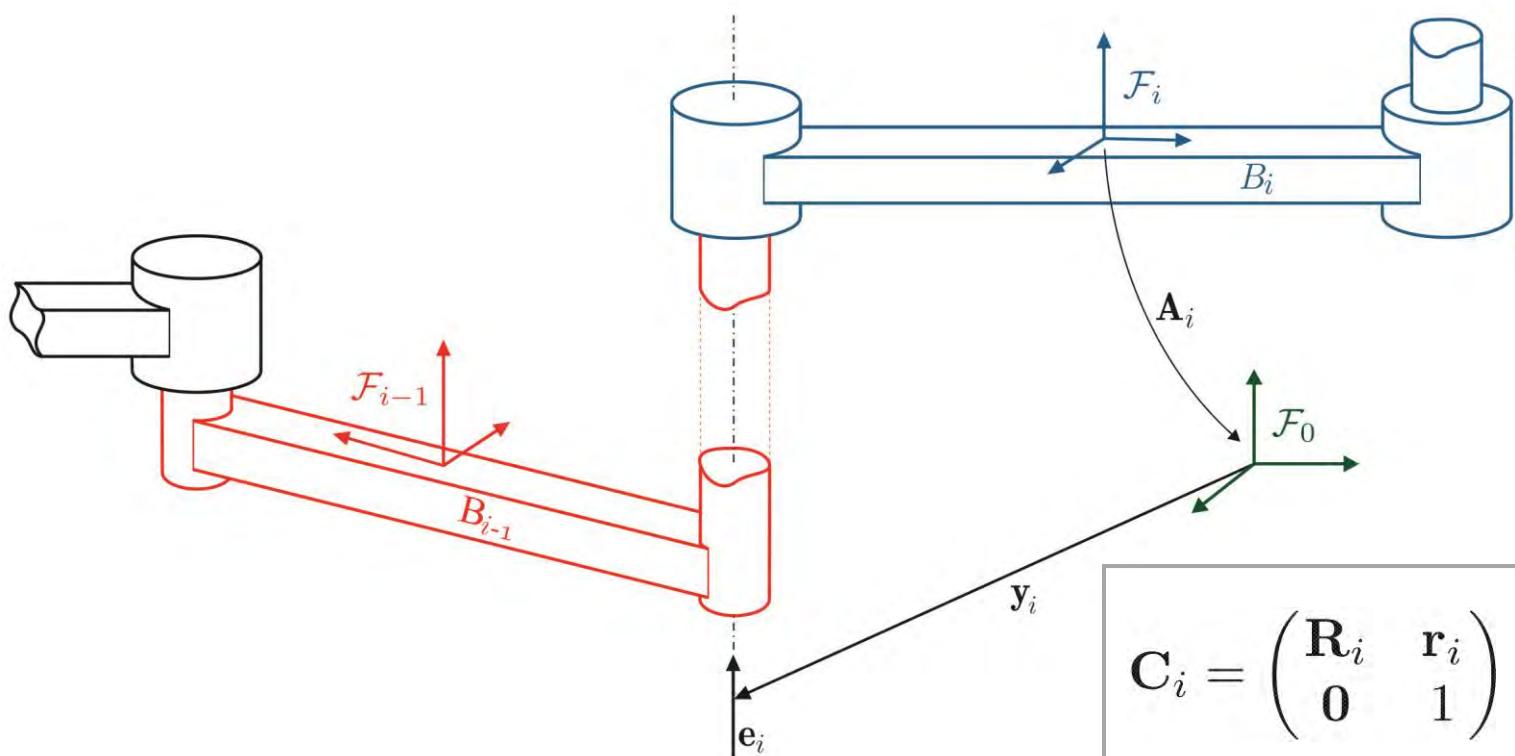
- Absolute reference configuration $\mathbf{A}_i = \mathbf{C}_i(\mathbf{0})$,
i.e. the reference configuration of body i w.r.t. the IFR \mathcal{F}_0 for $\mathbf{q} = \mathbf{0}$
- Joint screw coordinates in spatial representations (measured + resolved in IFR)

$$\mathbf{Y}_j = \begin{pmatrix} \mathbf{e}_j \\ \mathbf{y}_j \times \mathbf{e}_j + h_j \mathbf{e}_j \end{pmatrix} = \mathbf{Ad}_{\mathbf{A}_j}{}^j \mathbf{X}_j$$



- Configuration of body i

$$\mathbf{C}_i(\mathbf{q}) = \exp(\mathbf{Y}_1 q_1) \cdot \exp(\mathbf{Y}_2 q_2) \cdot \dots \cdot \exp(\mathbf{Y}_i q_i) \mathbf{A}_i$$



- Configuration of body i

$$\begin{aligned}\mathbf{C}_i(\mathbf{q}) &= \mathbf{B}_1 \exp({}^1\mathbf{X}_1 q_1) \cdot \mathbf{B}_2 \exp({}^2\mathbf{X}_2 q_2) \cdot \dots \cdot \mathbf{B}_i \exp({}^i\mathbf{X}_i q_i) \\ &= \exp(\mathbf{Y}_1 q_1) \cdot \exp(\mathbf{Y}_2 q_2) \cdot \dots \cdot \exp(\mathbf{Y}_i q_i) \mathbf{A}_i\end{aligned}$$

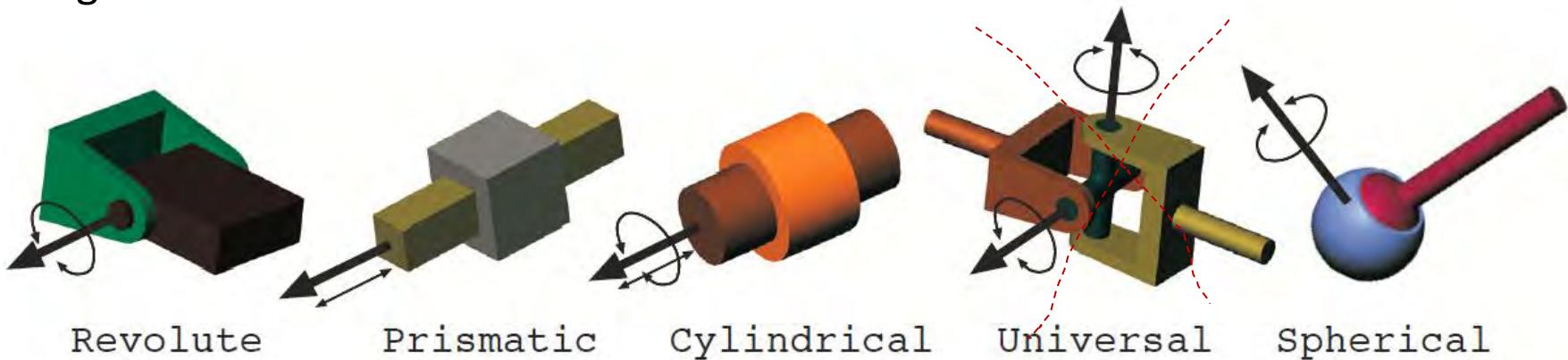
- Motion of lower pair (Reuleaux pair) is a screw motion
- Configuration of kinematic chain expressed in terms of joint screw coordinates
- Frame invariance allows for various representations of screws
 - No need for body-fixed joint frames
No modeling conventions, e.g. Denavit-Hartenberg

Definition:

A *kinematic joint* is a coincidence relation between features of two adjacent bodies.

Definition:

A *holonomic kinematic joint* imposes geometric constraints on the relative configuration of the features of the two bodies.

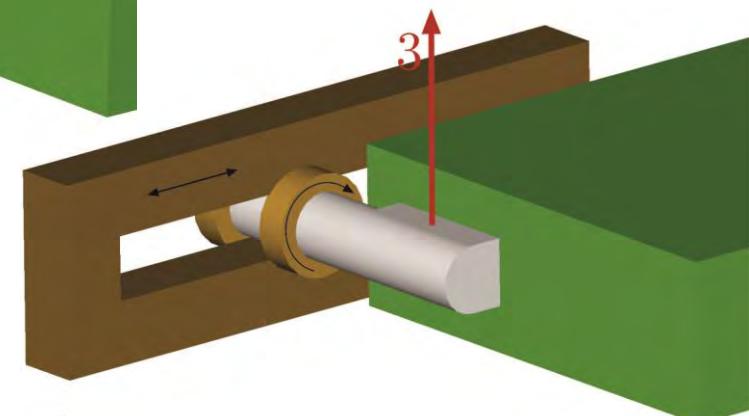
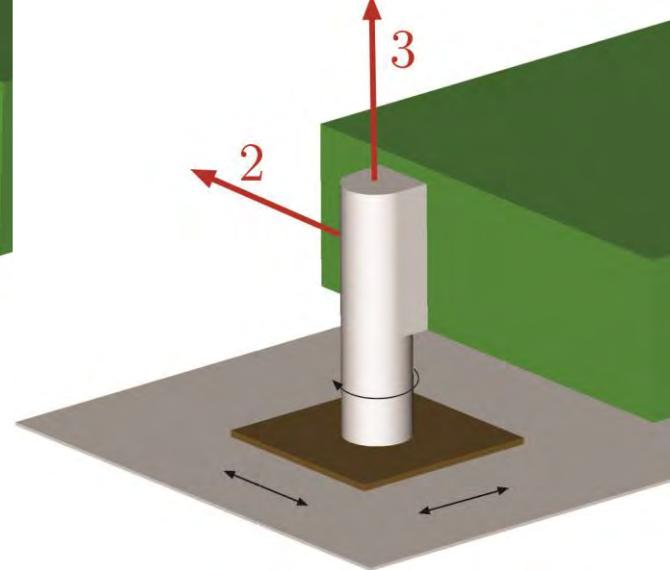
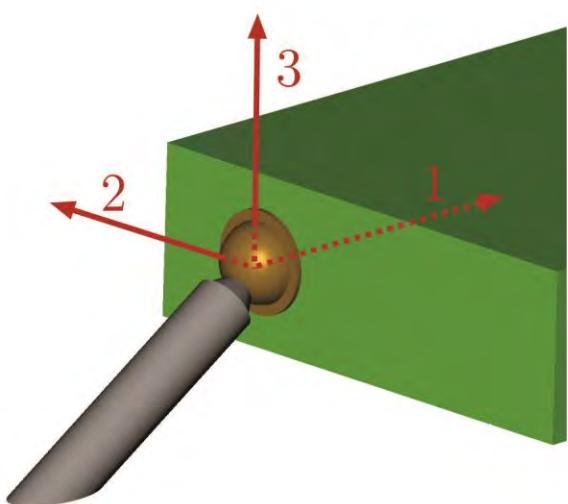
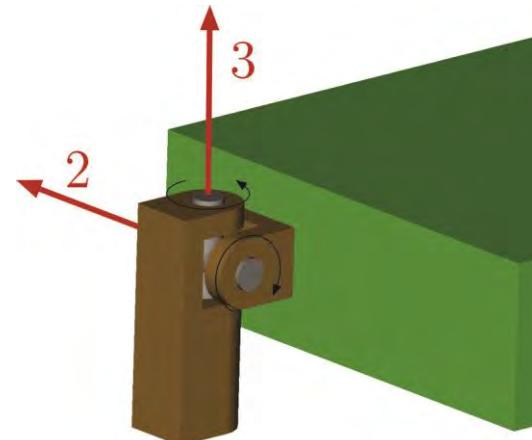
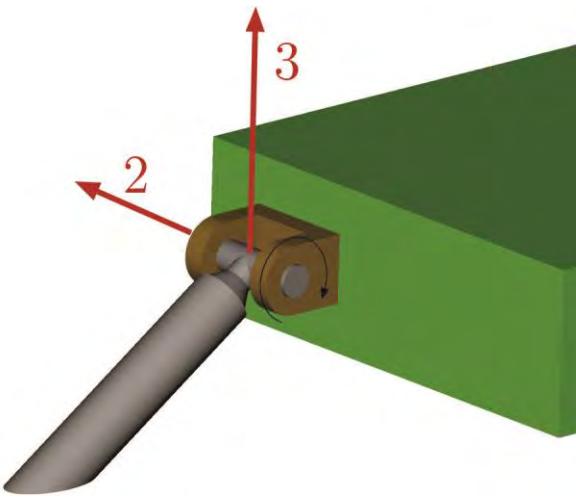
**Definition:**

A *lower pair (Reuleaux pair)* is a kinematic pair constraining two bodies so that they move relative to each other while remaining in surface contact.

F. Reuleaux: Theoretische Kinematic: Grundzüge einer Theorie des Maschinenwesens, Vieweg, Braunschweig, 1875.

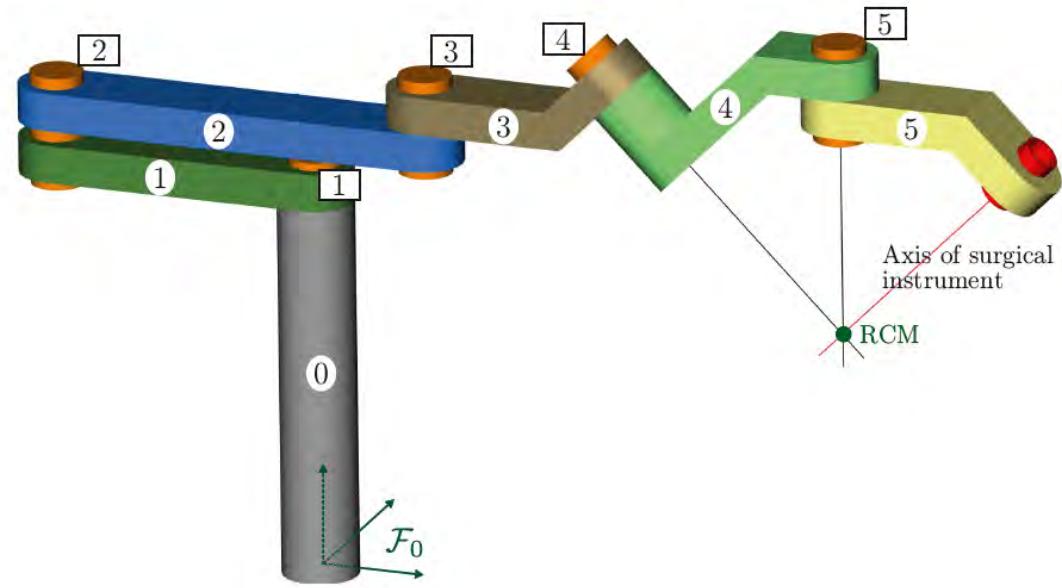
Translation: A.B.W. Kennedy as "The Kinematics of Machinery", Macmillan, London, 1876.
Reprinted, Dover, New York, 1963

Lower-Pairs vs. Higher Pairs



no exp mapping for joint motion of higher pairs

$$\mathbf{C}_i(\mathbf{q}) = \exp(\mathbf{Y}_1 q_1) \cdot \exp(\mathbf{Y}_2 q_2) \cdot \dots \cdot \exp(\mathbf{Y}_i q_i) \mathbf{A}_i$$



$$\mathbf{Y}_j = \begin{pmatrix} \mathbf{e}_j \\ \mathbf{y}_j \times \mathbf{e}_j + h_j \mathbf{e}_j \end{pmatrix}$$

Readily available (e.g. CAD)!

Example: Remote Center of Motion (RCM) mechanism – spatial screws

$$\mathbf{C}_i(\mathbf{q}) = \exp(\mathbf{Y}_1 q_1) \cdot \exp(\mathbf{Y}_2 q_2) \cdot \dots \cdot \exp(\mathbf{Y}_i q_i) \mathbf{A}_i$$

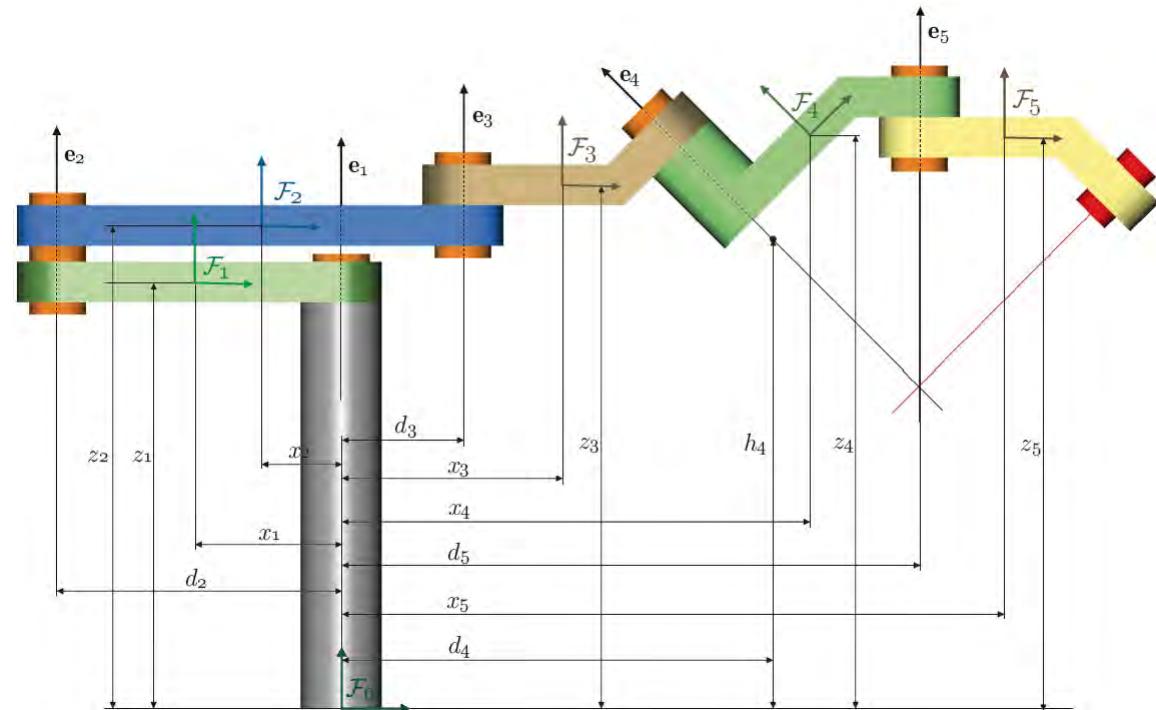
$$\mathbf{y}_1 = (0, 0, 0)^T, \mathbf{y}_2 = (-d_2, 0, 0)^T$$

$$\mathbf{y}_3 = (d_3, 0, 0)^T, \mathbf{y}_4 = (d_4, 0, h_4)^T$$

$$\mathbf{y}_5 = (d_5, 0, 0)^T$$

$$\mathbf{e}_1 = \mathbf{e}_2 = \mathbf{e}_3 = \mathbf{e}_5 = (0, 0, 1)^T$$

$$\mathbf{e}_4 = (-(1/\sqrt{2}, 0, 1/\sqrt{2})^T$$



$$\mathbf{Y}_1 = (0, 0, 1, 0, 0, 0)^T, \mathbf{Y}_2 = (0, 0, 1, 0, d_2, 0)^T, \mathbf{Y}_3 = (0, 0, 1, 0, -d_3, 0)^T$$

$$\mathbf{Y}_4 = (-1\sqrt{2}, 0, 1/\sqrt{2}, 0, -d_4/\sqrt{2} - h_4/\sqrt{2}, 0)^T, \mathbf{Y}_5 = (0, 0, 1, 0, -d_5, 0)^T$$

Example: 6R-Manipulator – body-fixed screws

$$\mathbf{C}_i(\mathbf{q}) = \mathbf{B}_1 \exp({}^1\mathbf{X}_1 q_1) \cdot \mathbf{B}_2 \exp({}^2\mathbf{X}_2 q_2) \cdot \dots \cdot \mathbf{B}_i \exp({}^i\mathbf{X}_i q_i)$$

$${}^1\mathbf{e}_1 = {}^4\mathbf{e}_4 = {}^6\mathbf{e}_6 = (0, 0, 1)^T$$

$${}^2\mathbf{e}_2 = {}^3\mathbf{e}_3 = {}^5\mathbf{e}_5 = (0, 1, 0)^T$$

$${}^i\mathbf{x}_i = \mathbf{0}, i = 1, \dots, 6$$

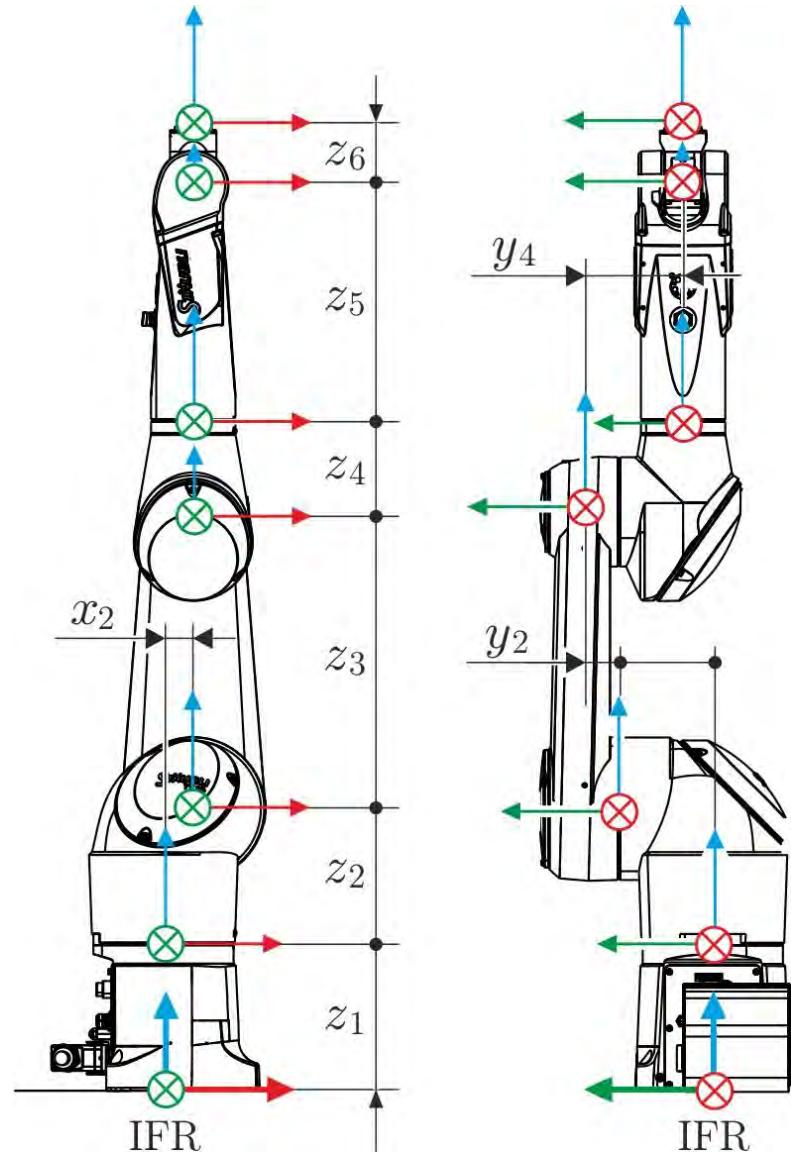
$$\mathbf{r}_1 = (0, 0, z_1)^T, \mathbf{r}_2 = (x_2, y_2, z_2)^T$$

$$\mathbf{r}_3 = (0, y_3, z_3)^T, \mathbf{r}_4 = (0, -y_4, z_4)^T$$

$$\mathbf{r}_5 = (0, 0, z_5)^T, \mathbf{r}_6 = (0, 0, z_6)^T$$

→ ${}^i\mathbf{X}_i = ({}^i\mathbf{e}_i, \mathbf{0})^T$

$$\mathbf{B}_i = \begin{pmatrix} \mathbf{I} & \mathbf{r}_i \\ \mathbf{0} & 1 \end{pmatrix}$$



Palm is a spherical mechanism

- Joint screws in IFR at center of rotation:

$$\mathbf{Y}_1 = \begin{pmatrix} \mathbf{e}_A \\ \mathbf{0} \end{pmatrix} \quad \mathbf{Y}_2 = \begin{pmatrix} \mathbf{e}_B \\ \mathbf{0} \end{pmatrix}$$

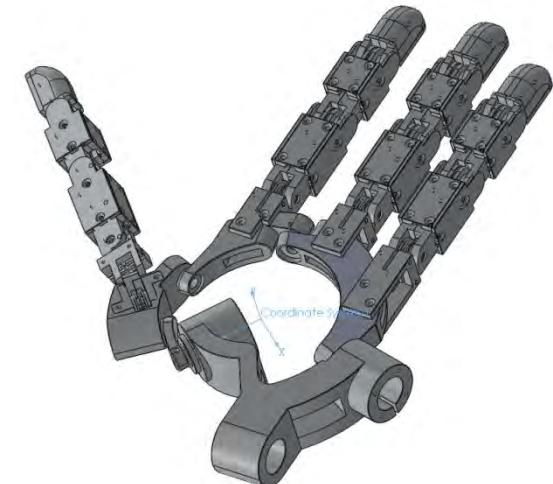
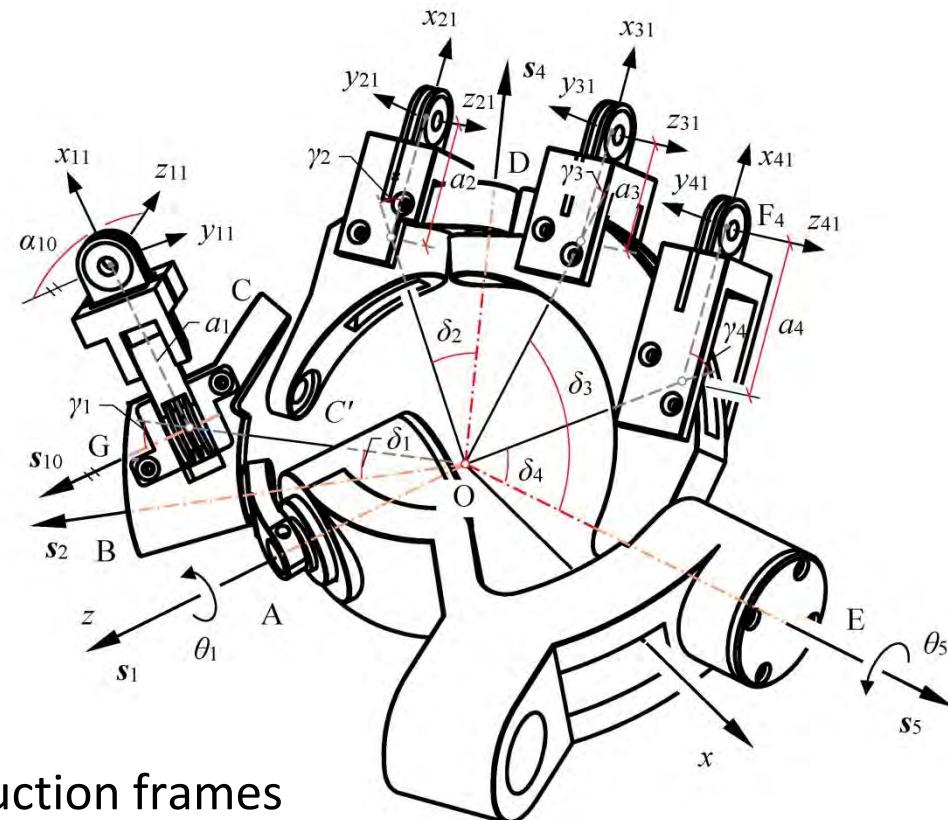
$$\mathbf{Y}_3 = \begin{pmatrix} \mathbf{e}_C \\ \mathbf{0} \end{pmatrix} \quad \mathbf{Y}_4 = \begin{pmatrix} \mathbf{e}_D \\ \mathbf{0} \end{pmatrix}$$

$$\mathbf{Y}_5 = \begin{pmatrix} \mathbf{e}_E \\ \mathbf{0} \end{pmatrix}$$

- It is most natural to locate the construction frames at center of curvature

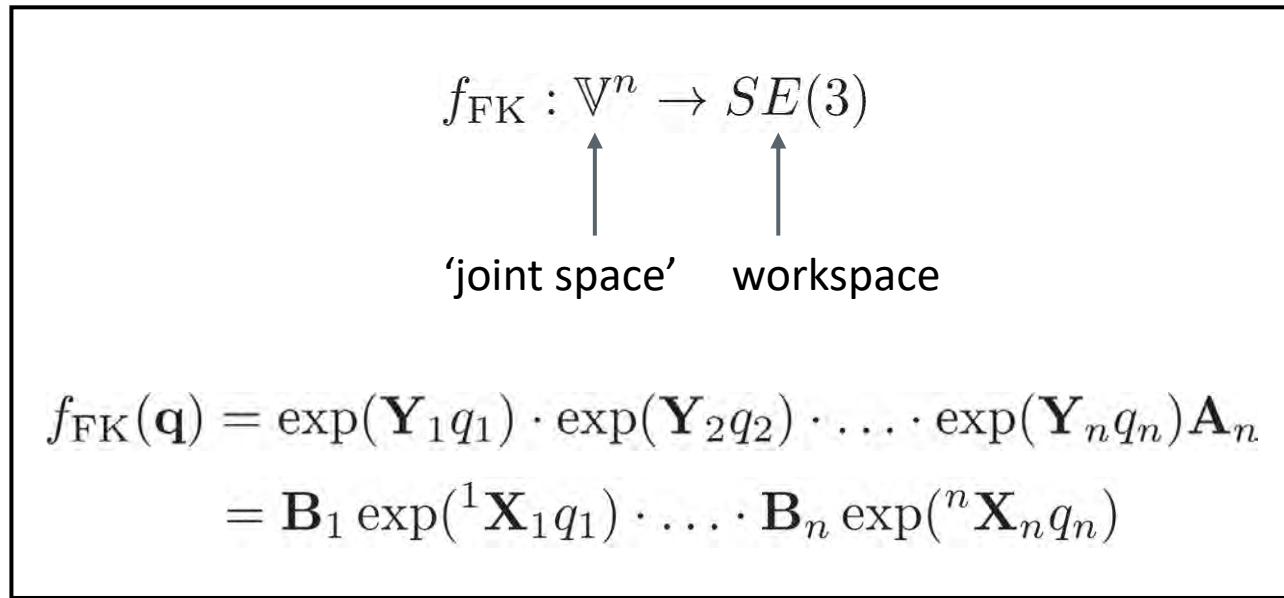
$$\mathbf{C}_1(\mathbf{q}) = \exp(\mathbf{Y}_1 q^1) = \begin{pmatrix} \mathbf{R}_1 & \mathbf{0} \\ \mathbf{0} & 1 \end{pmatrix}$$

$$\mathbf{C}_2(\mathbf{q}) = \exp(\mathbf{Y}_1 q^1) \exp(\mathbf{Y}_2 q^2) = \begin{pmatrix} \mathbf{R}_2 & \mathbf{0} \\ \mathbf{0} & 1 \end{pmatrix}$$

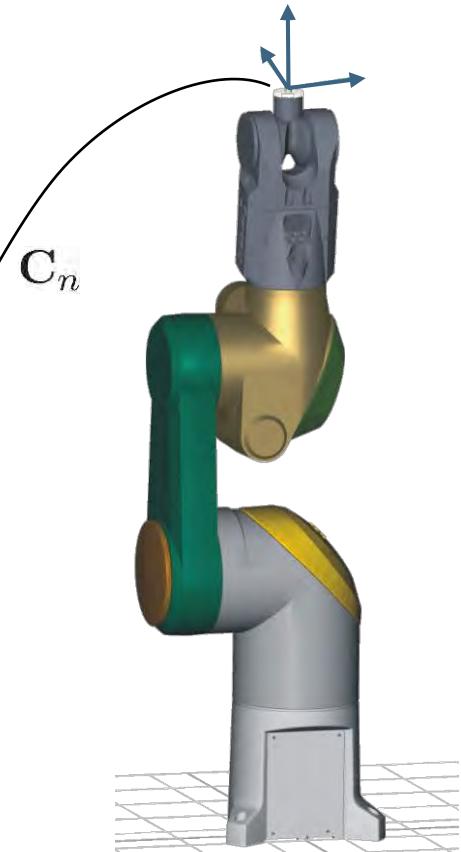
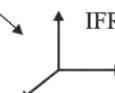
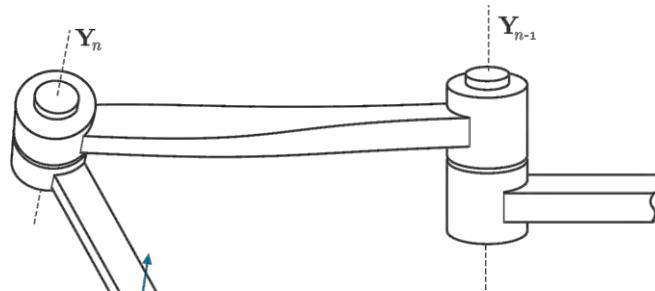


...

- Serial manipulator = Kinematic chain comprising n joints/link
- Forward kinematic mapping:



What happens for changing geometry?



Geometric loop constraints

- Kinematic loop comprising n 1-DOF lower-pair joints
- Geometric **loop constraints**:

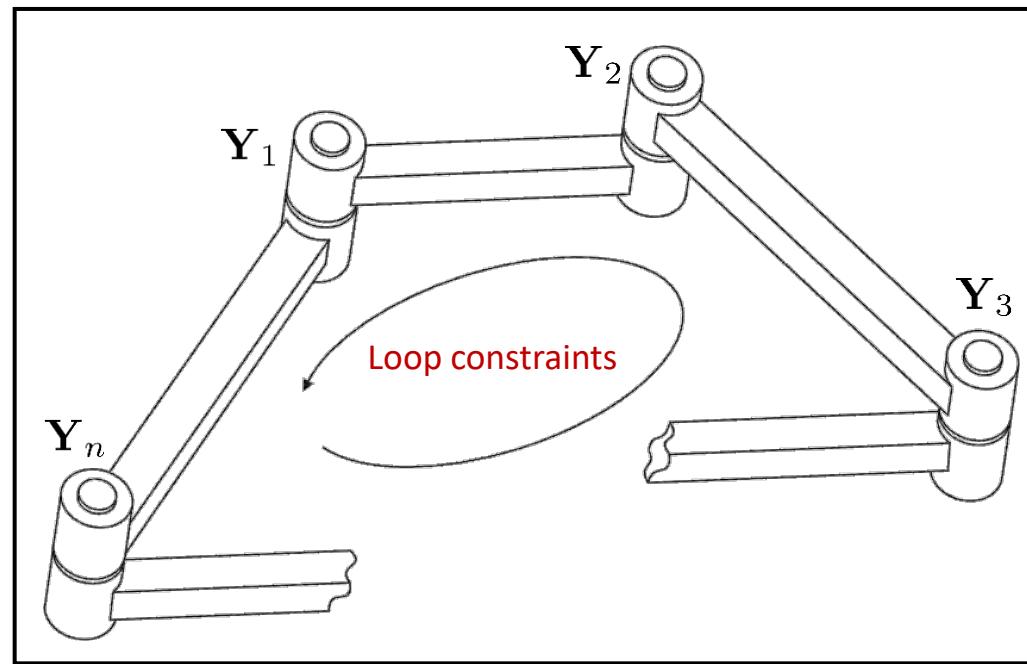
$$f(\mathbf{q}) = \mathbf{I}$$

with **constraint mapping** $f : \mathbb{V}^n \rightarrow SE(3)$

$$f(\mathbf{q}) = \exp(\mathbf{Y}_1 q_1) \exp(\mathbf{Y}_2 q_2) \cdot \dots \cdot \exp(\mathbf{Y}_n q_n)$$

Screw coordinate vector of joint j
in spatial representation:

$$\mathbf{Y}_j = \begin{pmatrix} \mathbf{e}_j \\ \mathbf{y}_j \times \mathbf{e}_j + h_j \mathbf{e}_j \end{pmatrix}$$



Geometric loop constraints

- Kinematic loop comprising n 1-DOF lower-pair joints
- Geometric **loop constraints**:

$$f(\mathbf{q}) = \mathbf{I}$$

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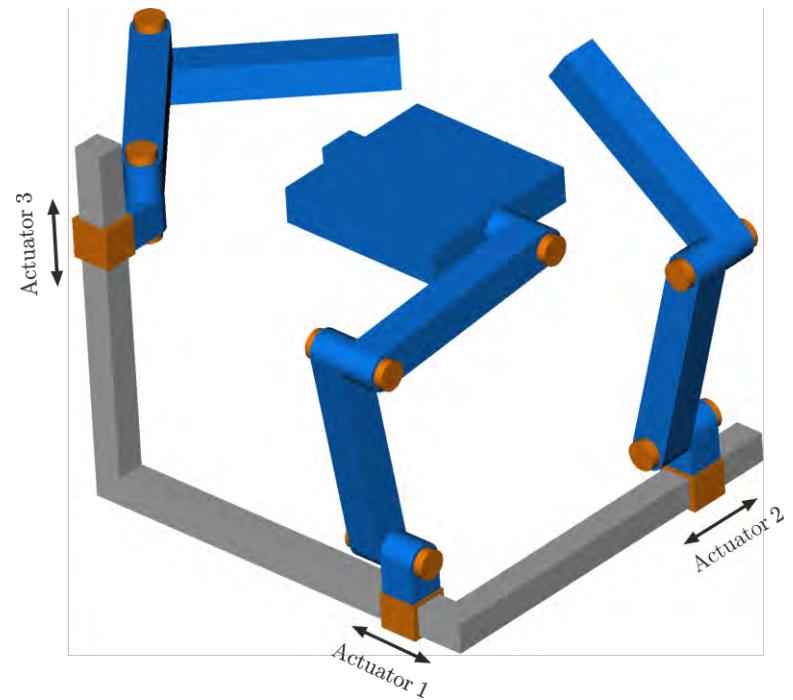
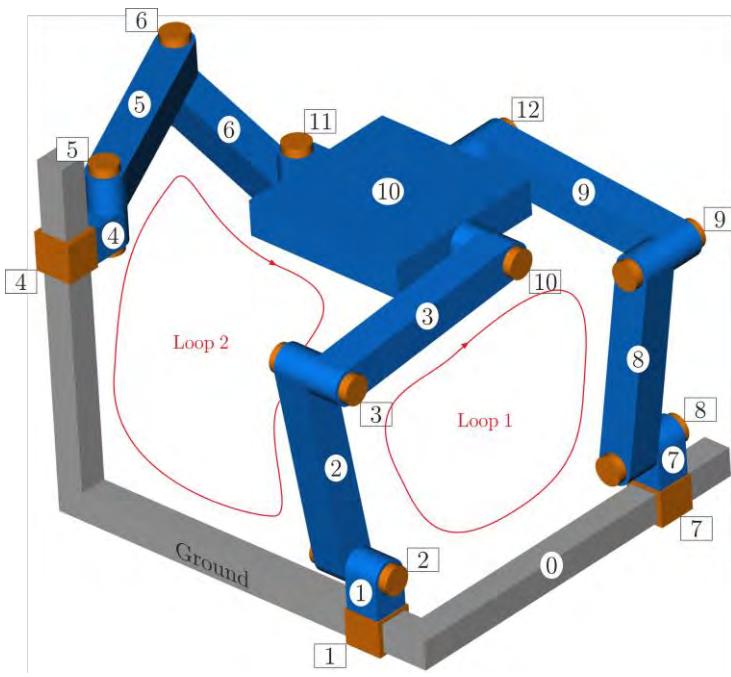
- Configuration space is the analytic variety:

$$V := \{\mathbf{q} \in \mathbb{V}^n | f(\mathbf{q}) = \mathbf{I}\}$$

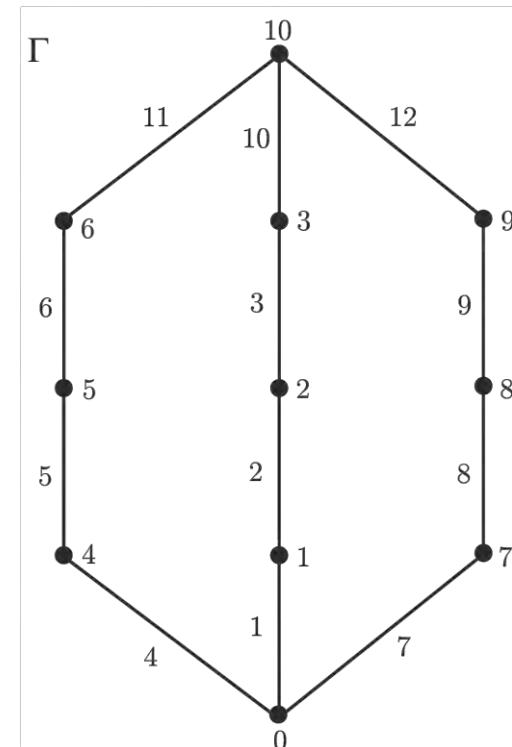
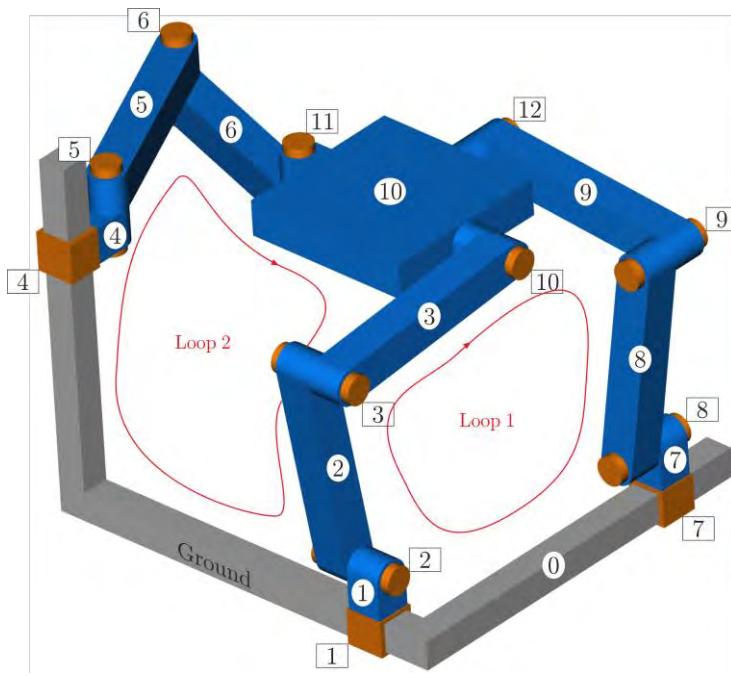
- The **local DOF** at $\mathbf{q} \in V$:

$$\delta_{\text{loc}}(\mathbf{q}) := \dim_{\mathbf{q}} V$$

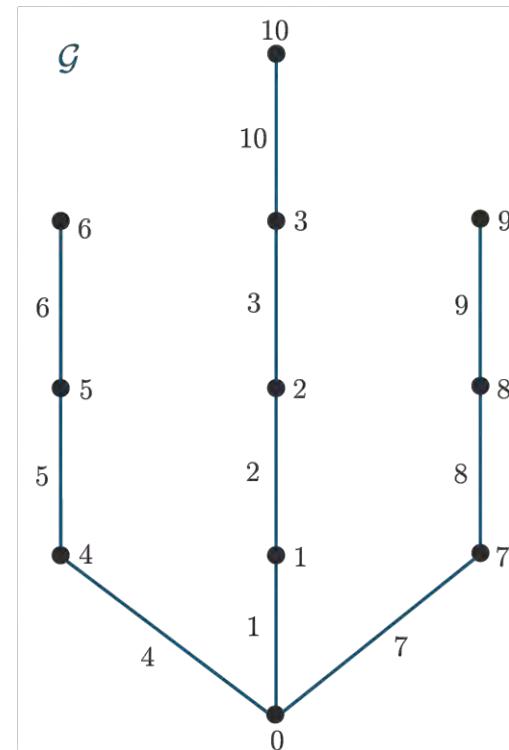
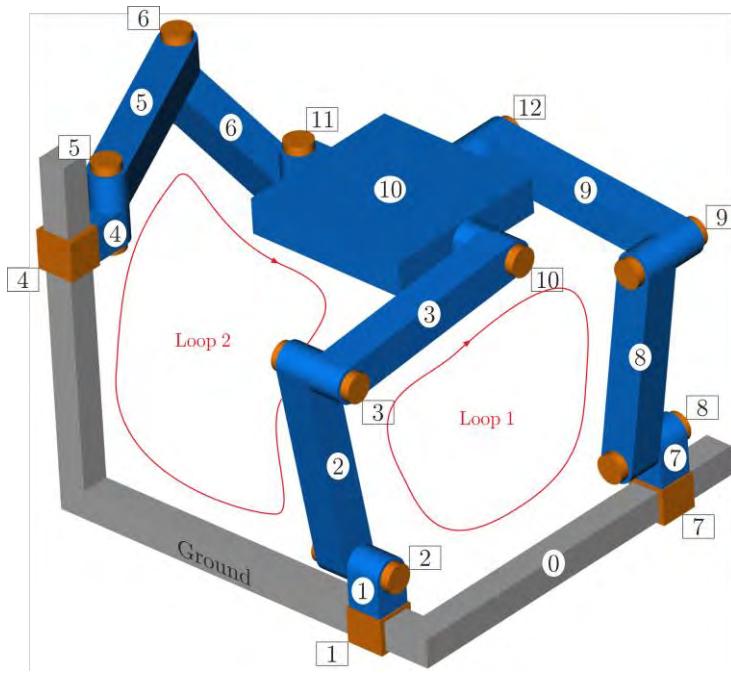
- Topologically independent loops: **Fundamental Cycles (FC)**



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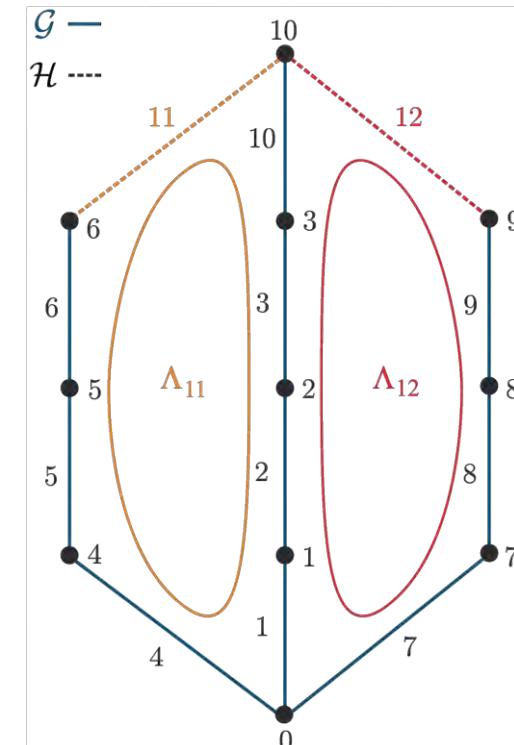
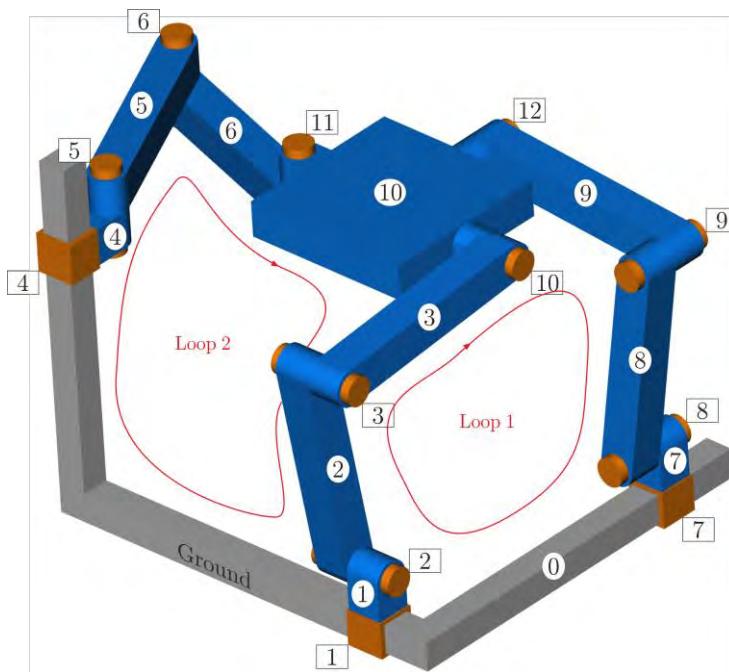


- Topologically independent loops: **Fundamental Cycles (FC)** $l = 1, \dots, \gamma$
- Geometric loop constraints:

$$f_l(\mathbf{q}) = \mathbf{I}, l = 1, \dots, \gamma$$

with constraint mapping of FC l :

$$f_l(\mathbf{q}) := \exp(\sigma_{l,\bar{l}} q_{\bar{l}} \mathbf{Y}_{\bar{l}}) \cdots \exp(\sigma_{l,l-1} q_{l-1} \mathbf{Y}_{l-1}) \exp(\mathbf{Y}_l q_l)$$



A rigid body motion is an *isometric and orientation preserving transformation*.
It can be considered as a frame transformation

Definition:

Special Euclidean Group (in 3 dimensions)

$$SE(3) := \left\{ \begin{pmatrix} \mathbf{R} & \mathbf{r} \\ \mathbf{0} & 1 \end{pmatrix} \mid \mathbf{R} \in SO(3), \mathbf{r} \in \mathbb{R}^3 \right\}$$

- Inverse of $\mathbf{C} \in SE(3)$: $\mathbf{C}^{-1} = \begin{pmatrix} \mathbf{R}^T & -\mathbf{R}^T \mathbf{r} \\ \mathbf{0} & 1 \end{pmatrix}$
- Identity: \mathbf{I}_4
- Group multiplication: $\mathbf{C}_1 \cdot \mathbf{C}_2$

Lie algebra of $SE(3)$:

$$se(3) := \left\{ \begin{pmatrix} \tilde{\boldsymbol{\xi}} & \boldsymbol{\eta} \\ \mathbf{0} & 0 \end{pmatrix} \mid \tilde{\boldsymbol{\xi}} \in so(3), \boldsymbol{\eta} \in \mathbb{R}^3 \right\}$$

- Velocity **constraints** at $\mathbf{q} \in V$:

$$\mathbf{0} = \frac{d}{dt} f(\mathbf{q}) = f(\mathbf{q})^{-1} \frac{d}{dt} f(\mathbf{q}) \quad f(\mathbf{q}) = \mathbf{I}, \mathbf{q} \in V$$

$$\mathbf{0} = \mathbf{J}(\mathbf{q})\dot{\mathbf{q}} = \mathbf{S}_1(\mathbf{q})\dot{q}_1 + \cdots + \mathbf{S}_n(\mathbf{q})\dot{q}_n$$

Geometric Jacobian: $\mathbf{J}(\mathbf{q}) = (\mathbf{S}_1(\mathbf{q}), \mathbf{S}_2(\mathbf{q}), \dots, \mathbf{S}_n(\mathbf{q}))$

Instantaneous joint screw coordinates: $\mathbf{S}_j(\mathbf{q}) := \mathbf{Ad}_{g_j(\mathbf{q})} \mathbf{Y}_j$

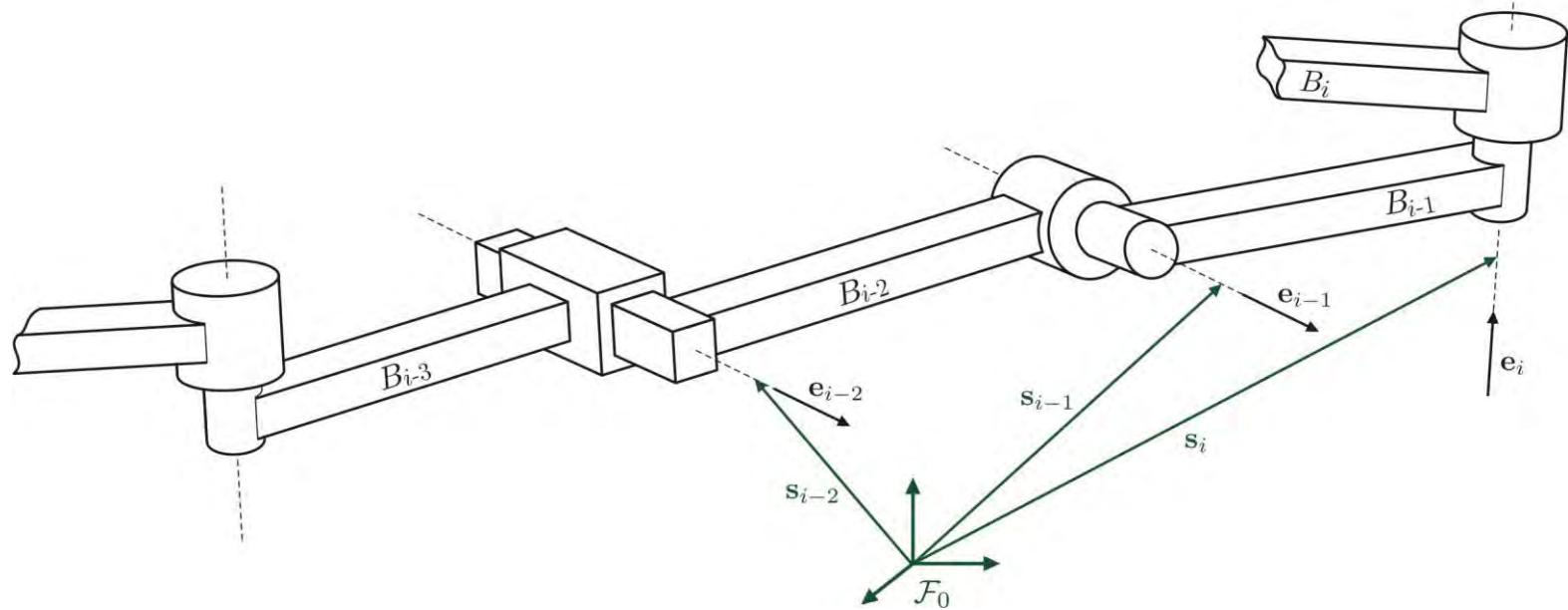
with $g_j(\mathbf{q}) := \exp(\mathbf{Y}_1 q_1) \cdot \dots \cdot \exp(\mathbf{Y}_j q_j)$

- Velocity **constraints** at $\mathbf{q} \in V$:

$$\mathbf{0} = \frac{d}{dt} f(\mathbf{q}) = f(\mathbf{q})^{-1} \frac{d}{dt} f(\mathbf{q}) \quad f(\mathbf{q}) = \mathbf{I}, \mathbf{q} \in V$$

$$\mathbf{0} = \mathbf{J}(\mathbf{q})\dot{\mathbf{q}} = \mathbf{S}_1(\mathbf{q})\dot{q}_1 + \cdots + \mathbf{S}_n(\mathbf{q})\dot{q}_n$$

$$\mathbf{0} = \dot{q}_1 \left(\mathbf{s}_1 \times \mathbf{e}_1 + h_1 \mathbf{e}_1 \right) + \dot{q}_2 \left(\mathbf{s}_2 \times \mathbf{e}_2 + h_2 \mathbf{e}_2 \right) + \dots + \dot{q}_n \left(\mathbf{s}_n \times \mathbf{e}_n + h_n \mathbf{e}_n \right)$$



- Velocity **constraints** at $\mathbf{q} \in V$:

$$\mathbf{0} = \frac{d}{dt} f(\mathbf{q}) = f(\mathbf{q})^{-1} \frac{d}{dt} f(\mathbf{q}) \quad f(\mathbf{q}) = \mathbf{I}, \mathbf{q} \in V$$

$$\mathbf{0} = \mathbf{J}(\mathbf{q})\dot{\mathbf{q}} = \mathbf{S}_1(\mathbf{q})\dot{q}_1 + \cdots + \mathbf{S}_n(\mathbf{q})\dot{q}_n$$

- The **differential DOF** at $\mathbf{q} \in V$:

$$\delta_{\text{diff}}(\mathbf{q}) := n - \text{rank } \mathbf{J}(\mathbf{q}) \quad (\text{instantaneous mobility})$$

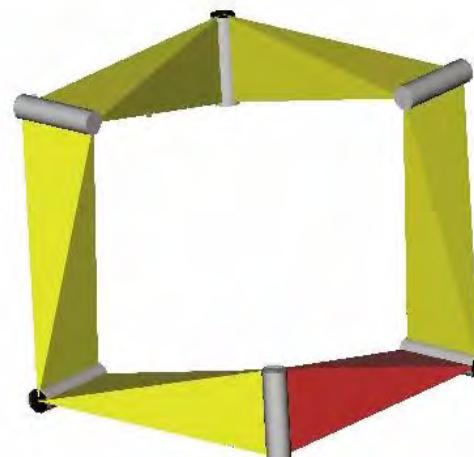
- If $\text{rank } \mathbf{J}(\mathbf{q})$ is not maximal, then $\mathbf{q} \in \mathbb{V}^n$ is a **critical point** of $f : \mathbb{V}^n \rightarrow SE(3)$
That is, the constraints become dependent

Definition:

A **constraint singularity** of a kinematic chain is a configuration $\mathbf{q} \in V$ which is a critical point of the constraint mapping, i.e. $\mathbf{0} = \mathbf{J}(\mathbf{q})$ is not full rank

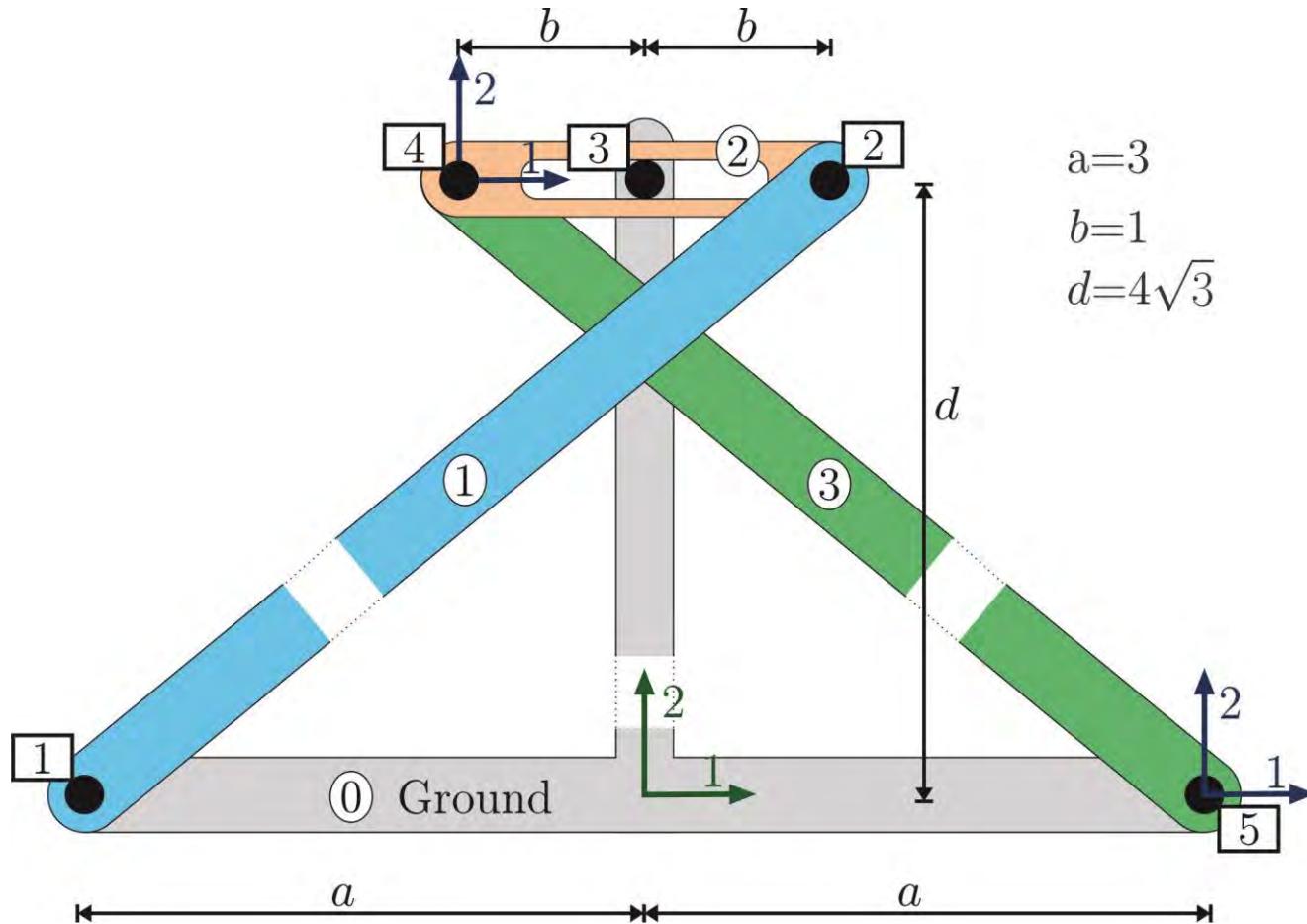
The c-space of **overconstrained** mechanisms consists of manifolds of constraint singularities

It depends on the constraints used to describe the mechanism



$$\text{rank } \mathbf{J}(\mathbf{q}) = 5$$

What is the DOF of this two-loop mechanism?



1. Phenomenology

- Mobility
- Constraint-, C-Space-, Input-, Output-Singularities
- A Model for the Mechanism Kinematics

2. Kinematics Modeling

- Screws and the Product-of-Exponentials
- Geometric Jacobian
- Higher-Order Kinematics

3. Local analysis of C-Space and ‘singular set’

- Tangent space, Tangent cone
- ‘Kinematic tangent cone’

4. Further Topics

- Singularities of Non-Holonomic Systems
- Escapement from singularities
- Combinatorial algorithm: Pebble Game

Definition 1:

A **kinematic singularity** is a configuration $q \in V$ where the differential (instantaneous) DOF is **not constant** in any neighborhood $U(q)$ of q .

- Property of the functions defining V – the generating ideal

→ Investigate set of points of certain rank

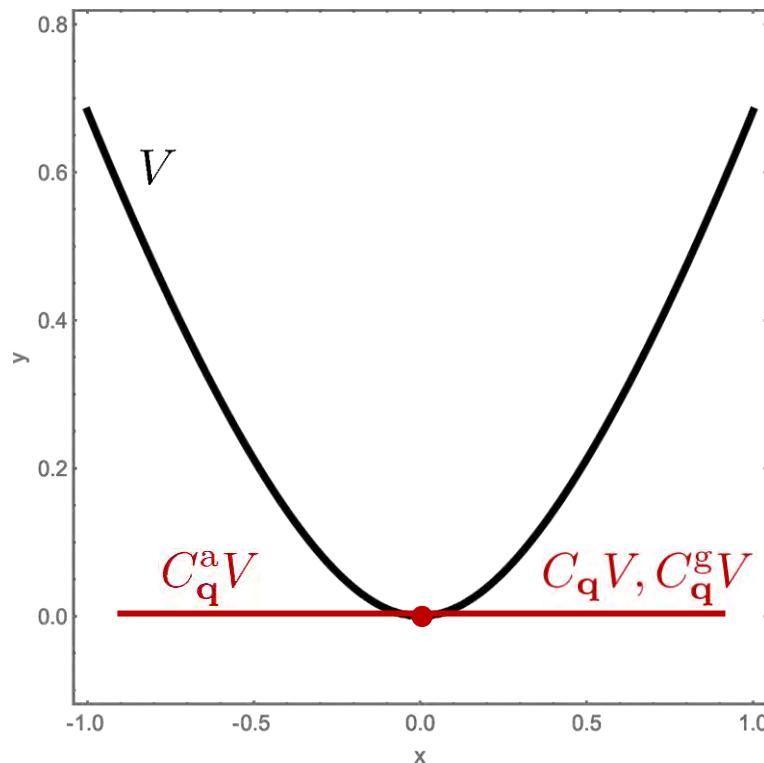
Definition 2:

A **C-space singularity** is a configuration $q \in V$ where V is **not a smooth manifold**.

- Topological property of the point set V

→ Investigate local geometry of V

- Geometry vs. Kinematics:



$$V = \mathbf{V}(y^3 + yx^2 - x^4) \subset \mathbb{R}^2$$

$$\mathbf{J}(\mathbf{q}) = \begin{pmatrix} -4x^3 + 2xy & x^2 + 3y^2 \end{pmatrix}$$

V is a smooth manifold

$$\text{but } \text{rank } \mathbf{J}(\mathbf{q}) = \begin{cases} 0, & \mathbf{q} = \mathbf{0} \\ 1, & \mathbf{q} \neq \mathbf{0} \end{cases}$$

$\mathbf{q} = \mathbf{0}$ is a kinematic singularity

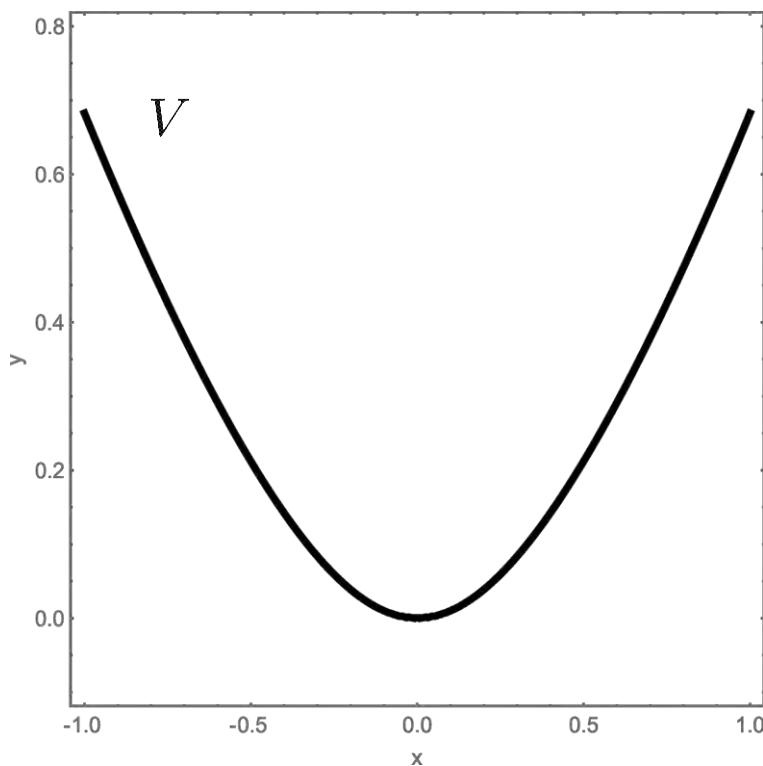
$$T_{\mathbf{0}}V = \ker \mathbf{J}(\mathbf{0}) = \mathbb{R}^2 ?$$

$$C_q^a V = \mathbf{V}(y^3 + yx^2)$$

$$= \{(x, y) | y = 0\}$$

- Let V be a real point set in \mathbb{V}^n . V is an **analytic variety** if and only if it is closed and for each $\mathbf{q} \in \mathbb{V}^n$ there is a neighborhood $U(\mathbf{q})$ in \mathbb{V}^n and analytic functions f_1, \dots, f_m such that $V \cap U(\mathbf{q})$ is the set of common zeros of f_i in $U(\mathbf{q})$.

Point set in \mathbb{R}^2 :



Function:

$$f(x, y) = y^3 + 2x^2y - x^4$$

Variety:

$$V = \mathbf{V}(y^3 + 2x^2y - x^4) \subset \mathbb{R}^2$$

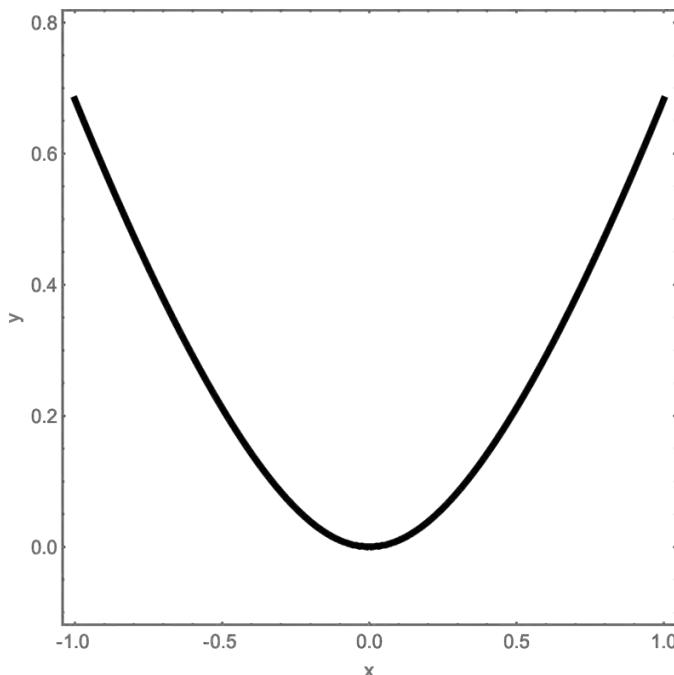
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- The **tangent space** $T_{\mathbf{q}}V$ to the **variety** V at \mathbf{q} is the vector space

$$T_{\mathbf{q}}V = \{\mathbf{x} \in \mathbb{R}^n \mid df(\mathbf{q}) \mathbf{x} = 0, \forall f \in I(V, \mathbf{q})\}$$

where $I(V, \mathbf{q})$ is the ideal of analytic germs vanishing over V at \mathbf{q}
(Think of all function that are zero at \mathbf{q} and locally look identical)

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$$\text{rank } \mathbf{J}(\mathbf{q}) = \begin{cases} 0, & \mathbf{q} = \mathbf{0} \\ 1, & \mathbf{q} \neq \mathbf{0} \end{cases}$$

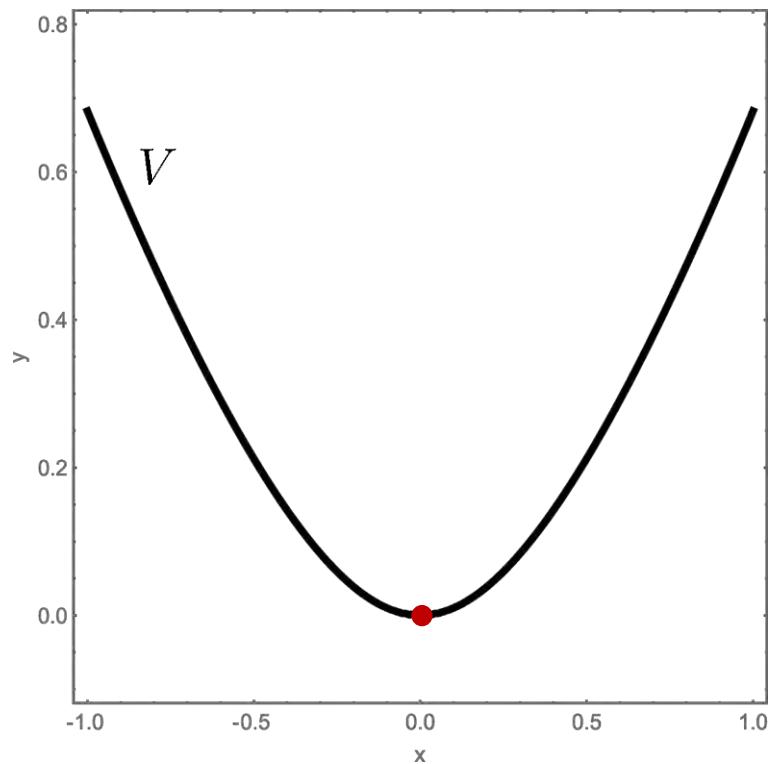
$$T_{\mathbf{0}}V = \ker \mathbf{J}(\mathbf{0}) = \mathbb{R}^2$$

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- The **tangent space** $T_{\mathbf{q}}V$ to the variety V at \mathbf{q} is the vector space

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where $I(V, \mathbf{q})$ is the ideal of analytic germs vanishing over V at \mathbf{q}
(Think of all function that are zero at \mathbf{q} and locally look identical)

- \mathbf{q} is a **regular point** of the variety V iff there is a neighborhood $U(\mathbf{q})$ and m analytic functions such that the differentials df_1, \dots, df_m are linearly independent and $V \cap U(\mathbf{q})$ is the set of common zeros of f_i in $U(\mathbf{q})$.
- \mathbf{q} is a regular point of the variety iff the dimension of the tangent space is constant in some neighborhood



$\mathbf{q} = \mathbf{0}$ is a singularity of the variety

$$\text{rank } \mathbf{J}(\mathbf{q}) = \begin{cases} 0, & \mathbf{q} = \mathbf{0} \\ 1, & \mathbf{q} \neq \mathbf{0} \end{cases}$$

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→ \mathbf{q} is a regular point of the variety iff the dimension of the tangent space is constant in some neighborhood

- The set V is a **manifold** at $\mathbf{q} \in \mathbb{V}^n$ if \mathbf{q} is a regular point of the variety V .
- It is important to distinguish between the topological set V and the variety.
- From a kinematic perspective the set V (i.e. the c-space) seems to be of primary interest.
This is not always sufficient for capturing kinematic singularities, however.
- The tangent space to the c-space V can in general not be defined as the kernel of \mathbf{J}

- Joint variables: $\mathbf{q} \in \mathbb{V}^n$

- Constraints: $h(\mathbf{q}) = \mathbf{0}$

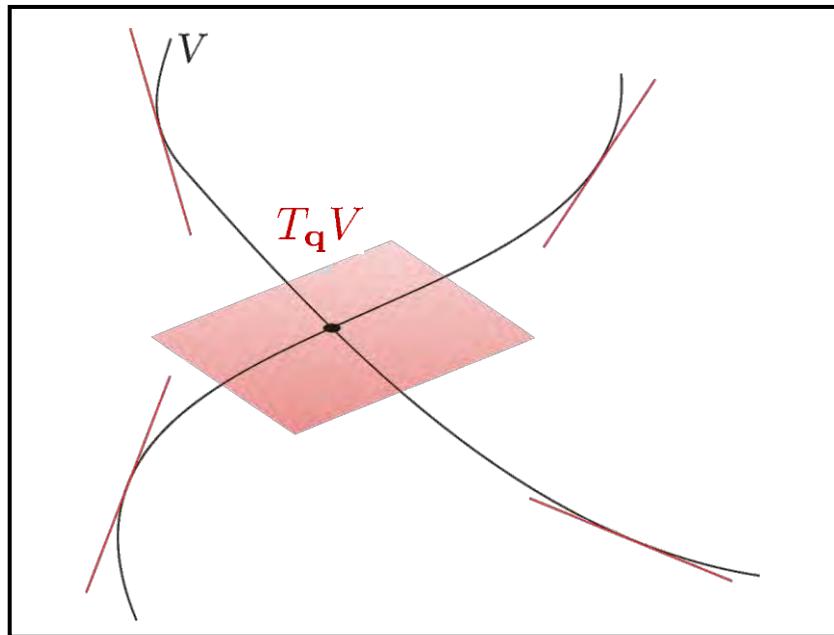
$$\mathbf{J}(\mathbf{q})\dot{\mathbf{q}} = \mathbf{0}$$

- Differential DOF: $\delta_{\text{diff}}(\mathbf{q}) := n - \text{rank } \mathbf{J}(\mathbf{q})$

- Configuration space: $V := \{\mathbf{q} \in \mathbb{V}^n | h(\mathbf{q}) = \mathbf{0}\}$

- Local (finite) DOF: $\delta_{\text{loc}}(\mathbf{q}) := \dim_{\mathbf{q}} V$

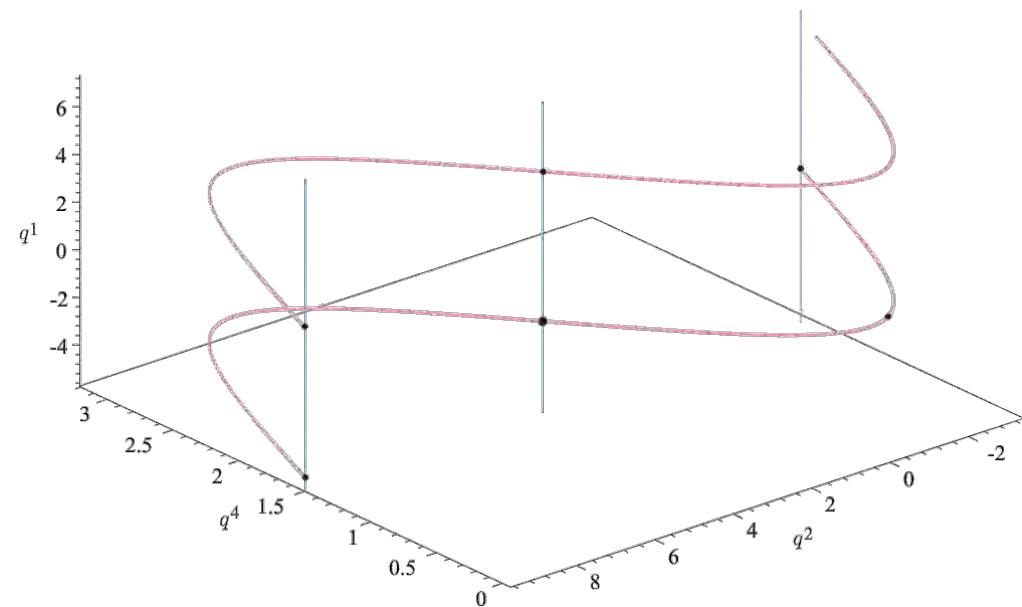
Rank drops at bifurcation point



$$\delta_{\text{diff}} = 1, 2$$

$$\delta_{\text{loc}} = 1$$

C-space is **non-smooth** at bifurcation point



- Joint variables: $\mathbf{q} \in \mathbb{V}^n$

- Constraints: $h(\mathbf{q}) = \mathbf{0}$

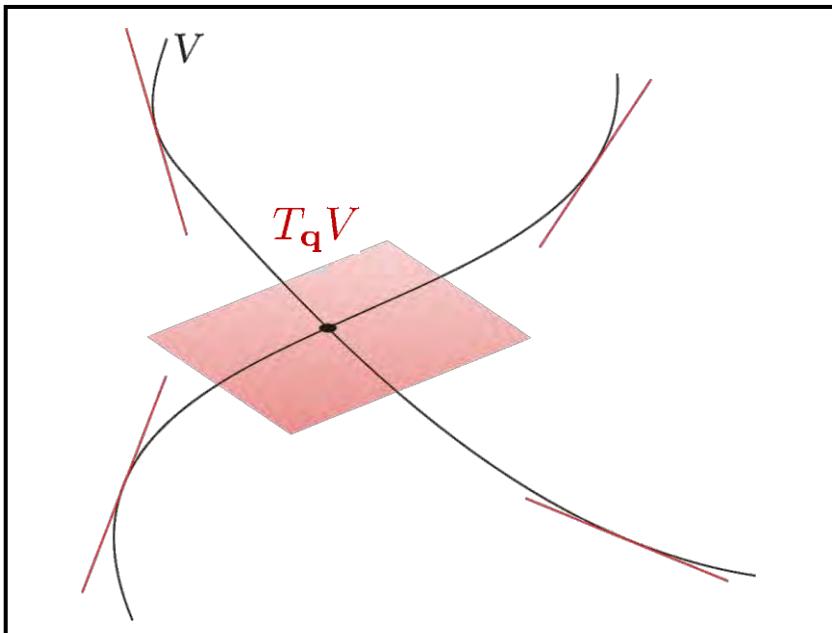
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Rank drops at bifurcation point



$$\delta_{\text{diff}} = 1, 2$$

$$\delta_{\text{loc}} = 1$$

C-space is **non-smooth** at bifurcation point

- Tangent space at $\mathbf{q} \in V$:

$T_{\mathbf{q}} V =$ Vector space of tangents to V at \mathbf{q}
 $= \ker \mathbf{J}(\mathbf{q})$ (for this linkage)

- Joint variables: $\mathbf{q} \in \mathbb{V}^n$

- Constraints: $h(\mathbf{q}) = \mathbf{0}$

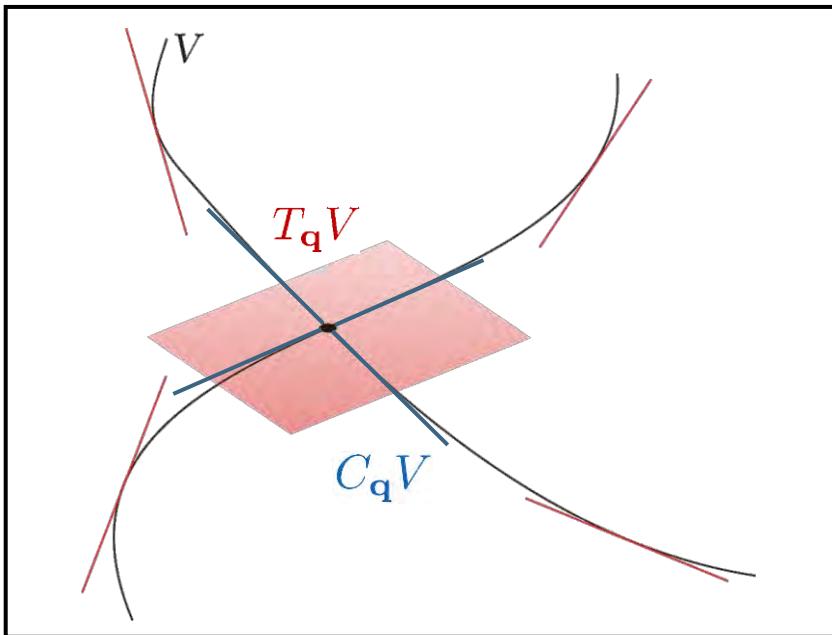
$$\mathbf{J}(\mathbf{q})\dot{\mathbf{q}} = \mathbf{0}$$

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Rank drops at bifurcation point



$$\delta_{\text{diff}} = 1, 2$$

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C-space is **non-smooth** at bifurcation point

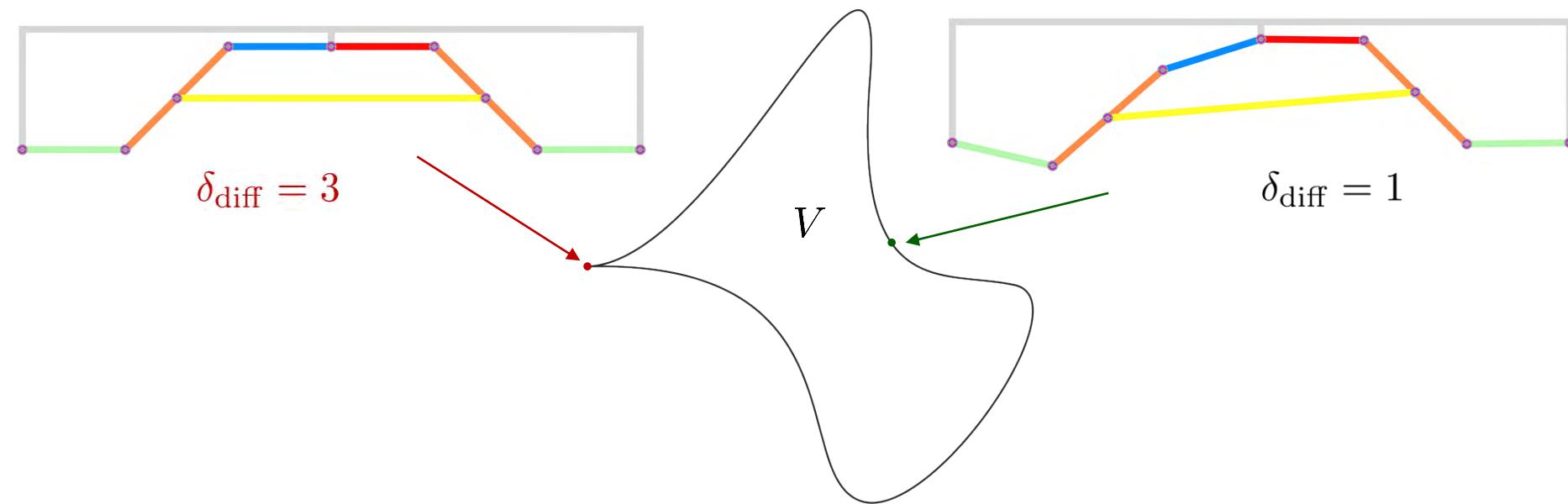
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$T_{\mathbf{q}} V =$ Vector space of tangents to V at \mathbf{q}
 $= \ker \mathbf{J}(\mathbf{q})$ (for this linkage)

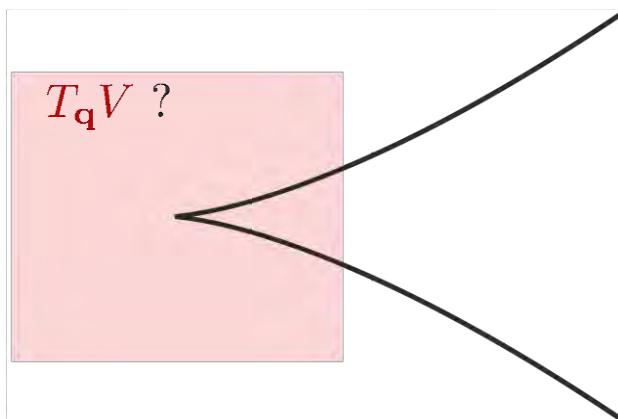
- 'Tangent cone' at $\mathbf{q} \in V$: [Whitney, 1965]

$C_{\mathbf{q}} V =$ Limit of tangent spaces near \mathbf{q}

Some examples: 'Double Watt' Linkage

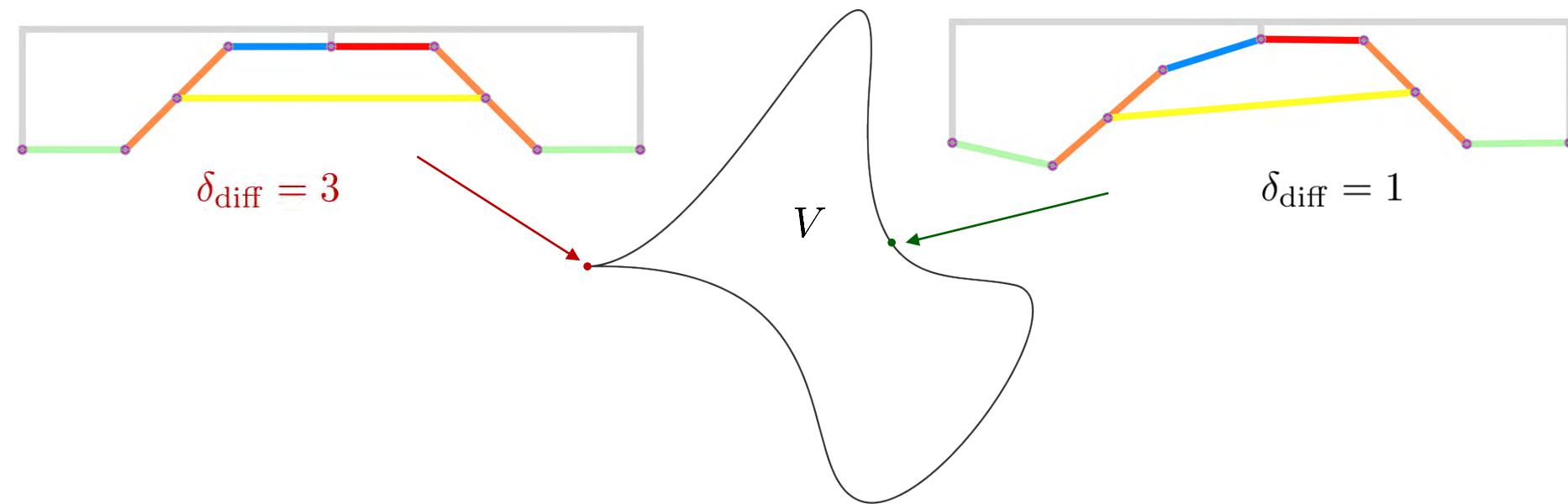


Tangent space?

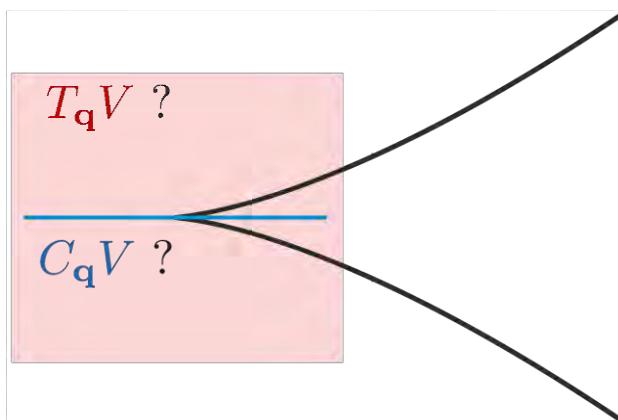


- Definition (simple version):
Tangent space is the set of **tangents** to V at q
- There are **no tangents at cusps!**

Some examples: 'Double Watt' Linkage



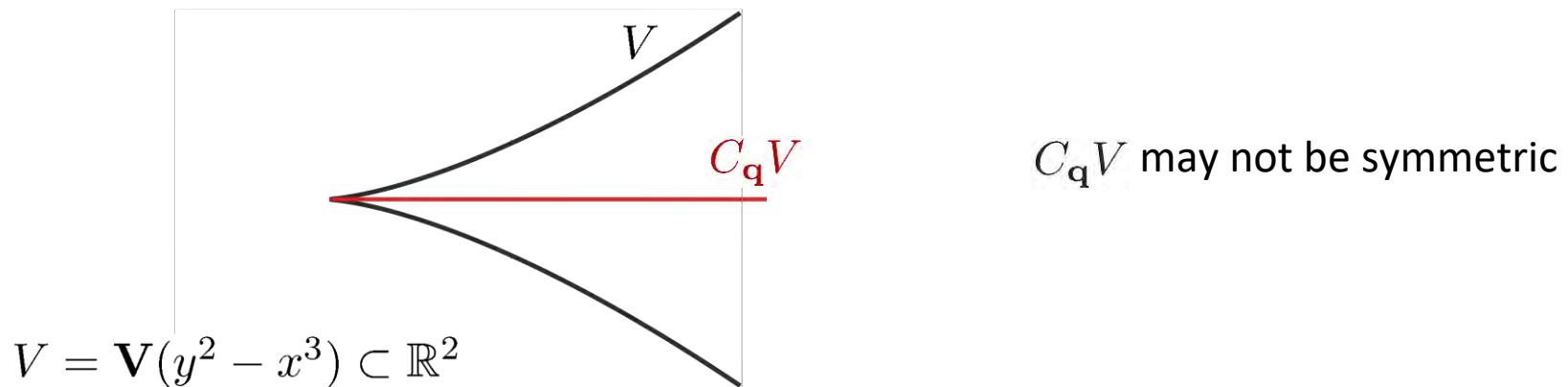
Tangent cone?



- Definition (simple version):
Tangent space is the set of **tangents** to V at q
- There are **no tangents at cusps!**

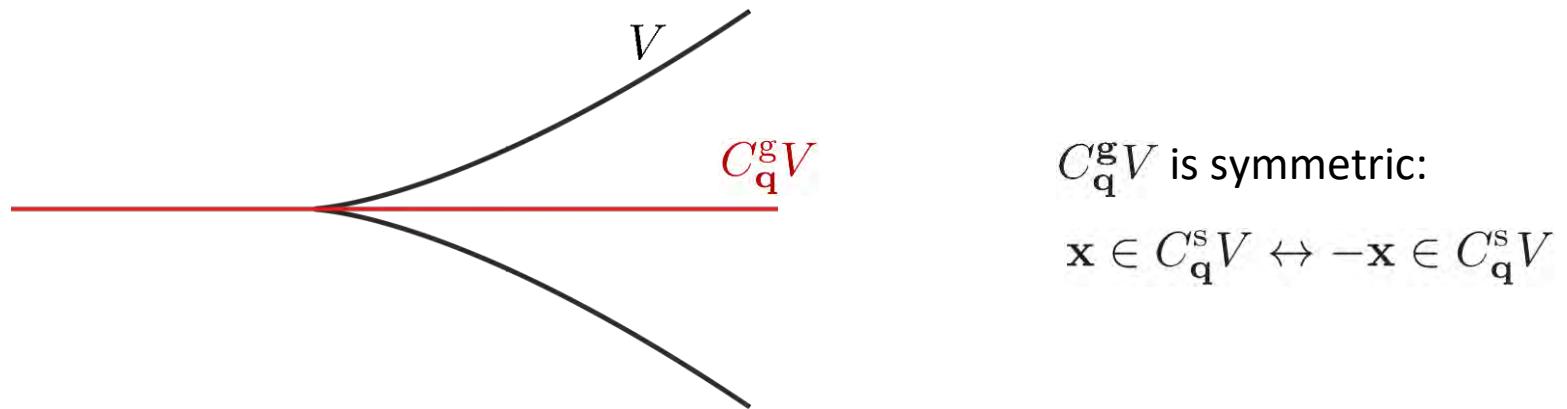
- Definition 1:

Tangent semi-cone $C_q V$ consists of the limits to **secant rays** originating from q



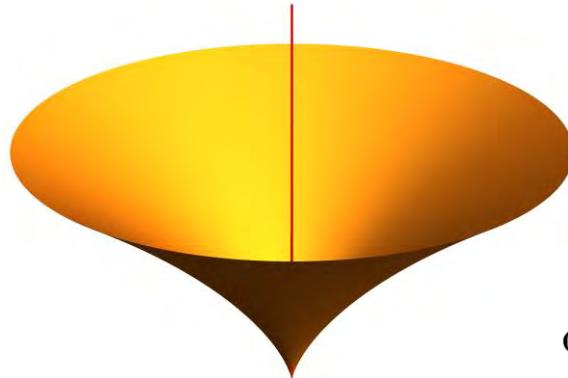
- Definition 2:

Geometric tangent cone $C_q^g V$ consists of the limits to **secant lines** originating from q

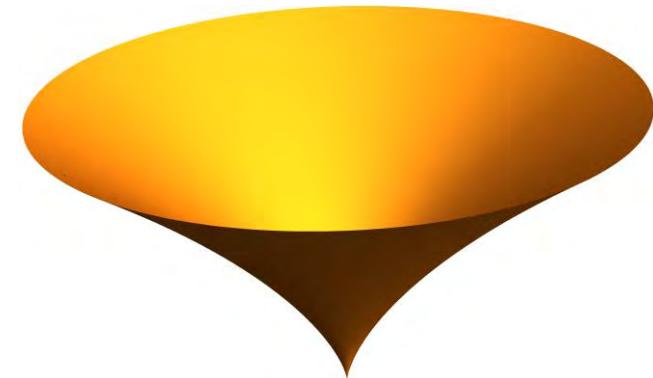


■ Definition 1:

Tangent semi-cone $C_q^s V$ consists of the limits to **secant rays** originating from q



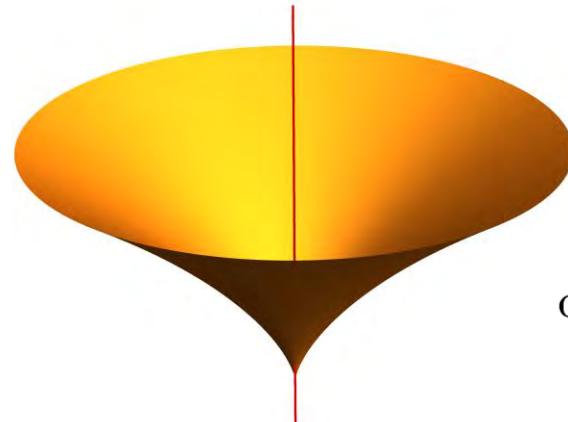
$$\dim C_q^s V \neq \dim V$$



$$V = \mathbf{V}(x^2 + y^2 - z^3) \subset \mathbb{R}^3$$

■ Definition 2:

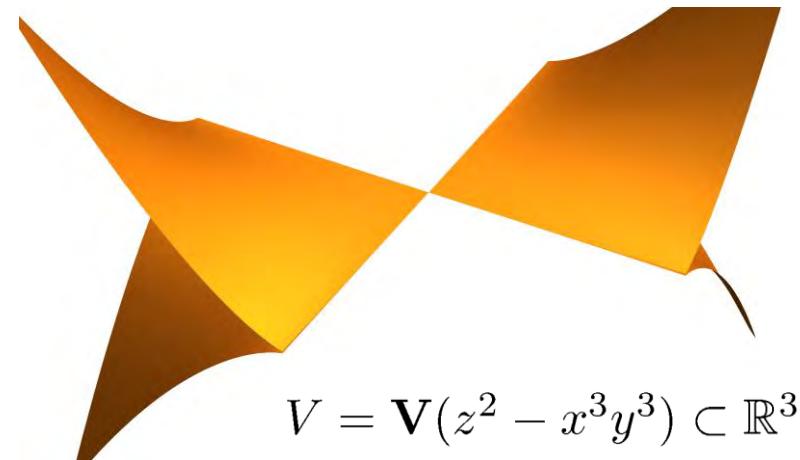
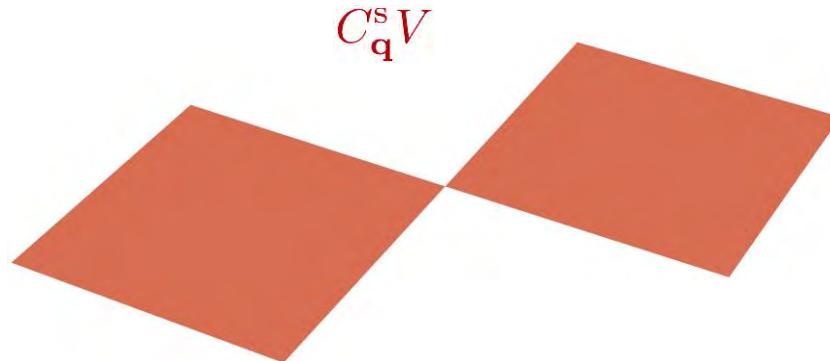
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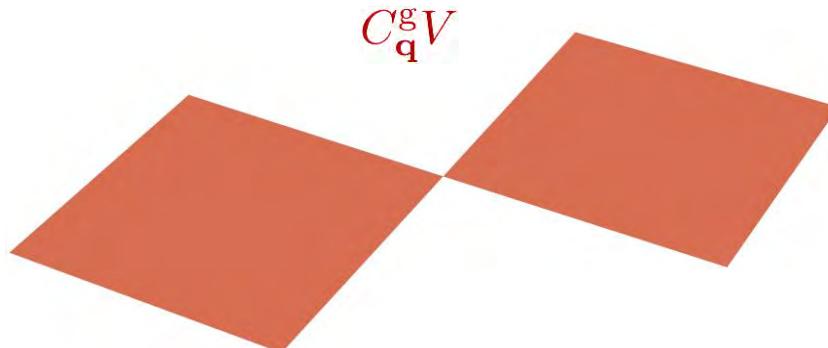
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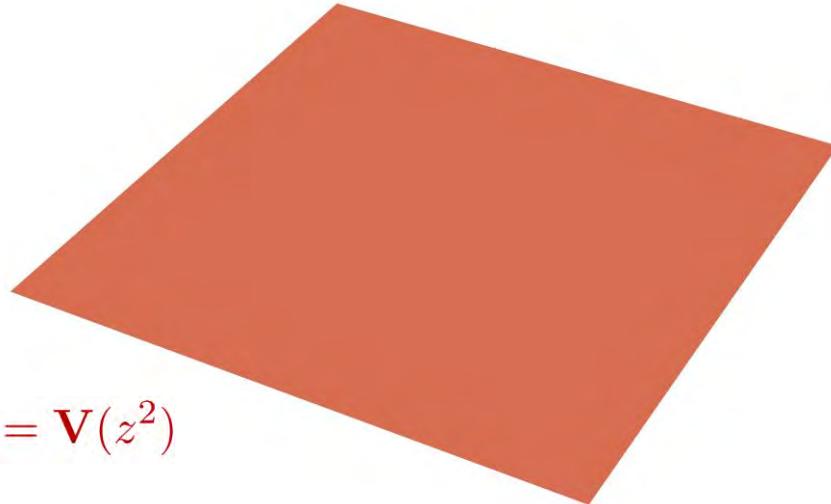
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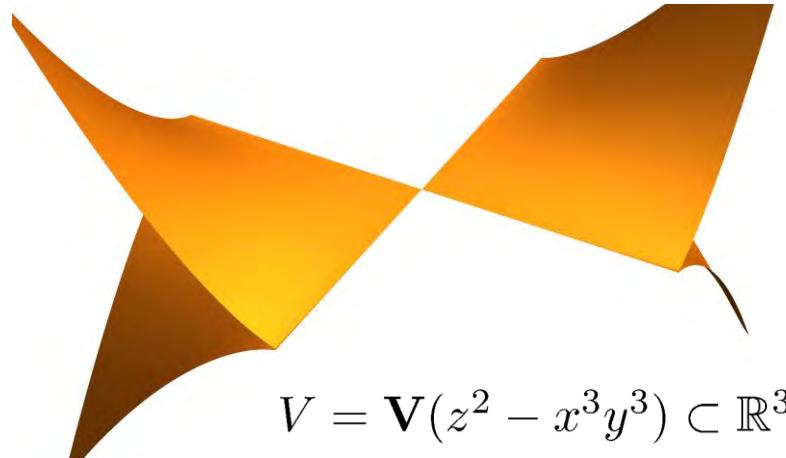


- Definition 3:

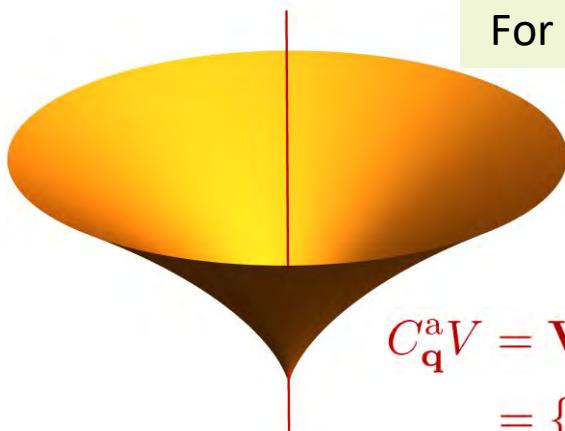
Algebraic tangent cone $C_q^a V$ is defined by the vanishing of initial polynomials



$$\begin{aligned} C_q^a V &= \mathbf{V}(z^2) \\ &= \{(x, y, z) | z = 0\} \end{aligned}$$

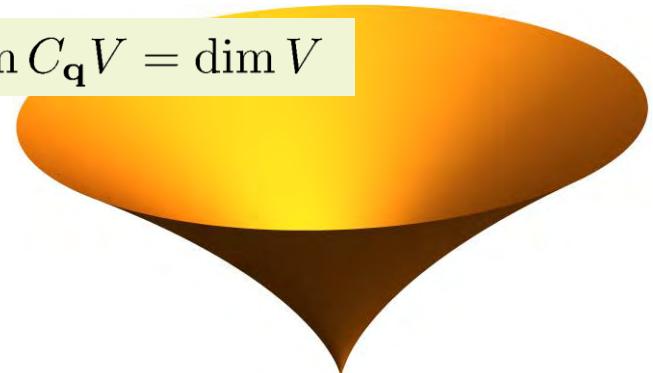


$$V = \mathbf{V}(z^2 - x^3y^3) \subset \mathbb{R}^3$$



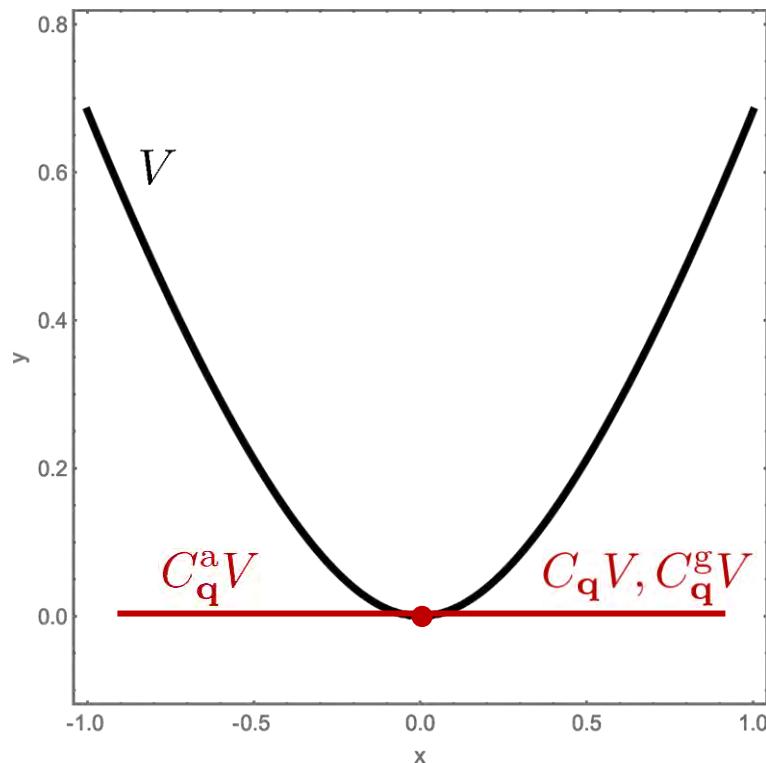
$$\begin{aligned} C_q^a V &= \mathbf{V}(x^2 + y^2) \\ &= \{(x, y, z) | x = y = 0\} \end{aligned}$$

For complex varieties $\dim C_q V = \dim V$



$$V = \mathbf{V}(x^2 + y^2 - z^3) \subset \mathbb{R}^3$$

- Geometry vs. Kinematics:



$$V = \mathbf{V}(y^3 + yx^2 - x^4) \subset \mathbb{R}^2$$

$$\mathbf{J}(\mathbf{q}) = \begin{pmatrix} -4x^3 + 2xy & x^2 + 3y^2 \end{pmatrix}$$

V is a smooth manifold

$$\text{but } \text{rank } \mathbf{J}(\mathbf{q}) = \begin{cases} 0, & \mathbf{q} = \mathbf{0} \\ 1, & \mathbf{q} \neq \mathbf{0} \end{cases}$$

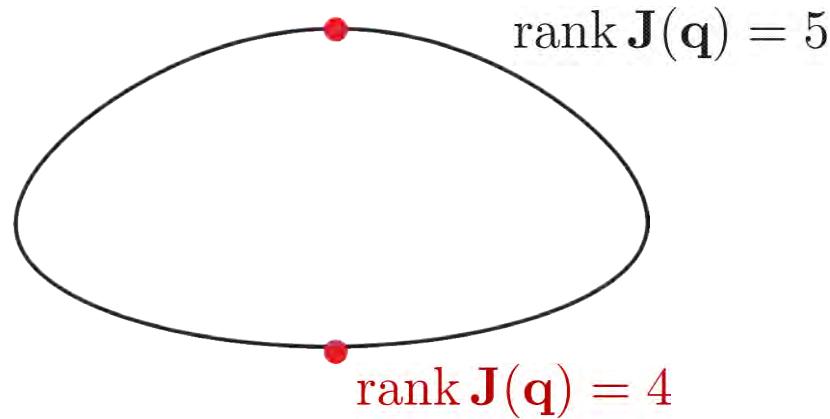
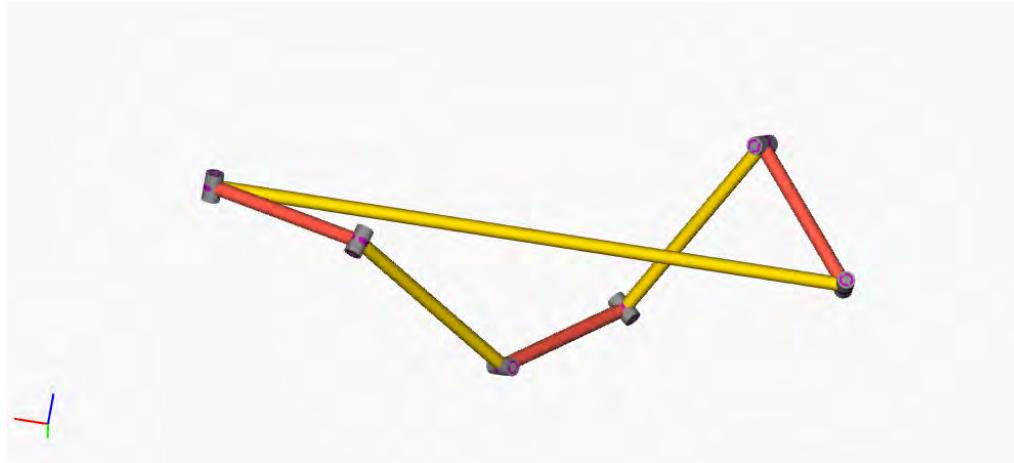
$\mathbf{q} = \mathbf{0}$ is a kinematic singularity

$$T_{\mathbf{0}}V = \ker \mathbf{J}(\mathbf{0}) = \mathbb{R}^2 ?$$

$$C_q^a V = \mathbf{V}(y^3 + yx^2) = \mathbf{V}(y(y^2 + x^2))$$

$$= \{(x, y) | y = 0\} \cup \{(x, y) | x = y = 0\}$$

- Geometry vs. Kinematics:



- Velocity constraints at $\mathbf{q} \in V$:

$$\mathbf{0} = \mathbf{J}(\mathbf{q})\dot{\mathbf{q}} = \sum_{i \leq n} \mathbf{S}_i(\mathbf{q})\dot{q}_i$$

- Acceleration constraints at $\mathbf{q} \in V$:

$$\begin{aligned}\mathbf{0} &= \mathbf{J}(\mathbf{q})\ddot{\mathbf{q}} + \dot{\mathbf{J}}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}} \\ &= \sum_{i \leq n} \mathbf{S}_i \ddot{q}_i + \sum_{i \leq n} \sum_{j < i} [\mathbf{S}_j, \mathbf{S}_i] \dot{q}_j \dot{q}_i\end{aligned}$$

$$\frac{\partial \mathbf{S}_i}{\partial q_j} = [\mathbf{S}_j, \mathbf{S}_i], j < i$$

- Jerk constraints...

Remember: for $\mathbf{q} = \mathbf{0}$ we have $\mathbf{S}_i(\mathbf{0}) = \mathbf{Y}_i$

Key Relations

- Partial derivative:

$$\frac{\partial \mathbf{S}_i}{\partial q_j} = [\mathbf{S}_j, \mathbf{S}_i], j < i$$

with screw product $[\mathbf{X}, \mathbf{Y}] = (\boldsymbol{\xi}_X \times \boldsymbol{\xi}_Y, \boldsymbol{\xi}_X \times \boldsymbol{\eta}_Y - \boldsymbol{\xi}_Y \times \boldsymbol{\eta}_X)$

$$= \text{ad}_{\mathbf{X}} \mathbf{Y}$$

$$\text{ad}_{\mathbf{Y}} = \begin{pmatrix} \tilde{\boldsymbol{\xi}}_Y & 0 \\ \tilde{\boldsymbol{\eta}}_Y & \tilde{\boldsymbol{\xi}}_Y \end{pmatrix}$$

- Bilinearity of Lie bracket:

$$\begin{aligned} [a_1 \mathbf{X}_1 + a_2 \mathbf{X}_2, b_1 \mathbf{Y}_1 + b_2 \mathbf{Y}_2] &= a_1 b_1 [\mathbf{X}_1, \mathbf{Y}_1] + a_1 b_2 [\mathbf{X}_1, \mathbf{Y}_2] + \\ &\quad a_2 b_1 [\mathbf{X}_2, \mathbf{Y}_1] + a_2 b_2 [\mathbf{X}_2, \mathbf{Y}_2] \end{aligned}$$

- Skew symmetry of Lie bracket:

$$[\mathbf{X}, \mathbf{Y}] = -[\mathbf{Y}, \mathbf{X}]$$

- Jacobi identity:

$$[\mathbf{Y}, [\mathbf{Z}, \mathbf{X}]] + [\mathbf{Z}, [\mathbf{X}, \mathbf{Y}]] + [\mathbf{X}, [\mathbf{Y}, \mathbf{Z}]] = 0$$

▪ **Higher-order** derivatives

$$\frac{\partial \mathbf{S}_i}{\partial q_j} = [\mathbf{S}_j, \mathbf{S}_i], j < i \quad \rightarrow \quad \frac{\partial^2 \mathbf{S}_i}{\partial q_l \partial q_j} = \left[\frac{\partial \mathbf{S}_j}{\partial q_l}, \mathbf{S}_i \right] + [\mathbf{S}_j, \frac{\partial \mathbf{S}_i}{\partial q_l}], \quad j < i$$

$$\frac{\partial^2 \mathbf{S}_i}{\partial q_l \partial q_j} = \begin{cases} [\mathbf{S}_l, [\mathbf{S}_j, \mathbf{S}_i]] & , l < j < i \\ [\mathbf{S}_j, [\mathbf{S}_l, \mathbf{S}_i]] & , j \leq l < i \end{cases}$$

$$\frac{\partial^3 \mathbf{S}_i}{\partial q_m \partial q_l \partial q_j} = \begin{cases} [\mathbf{S}_l, [\mathbf{S}_m, [\mathbf{S}_j, \mathbf{S}_i]]] , & l < m < j < i \\ [\mathbf{S}_m, [\mathbf{S}_l, [\mathbf{S}_j, \mathbf{S}_i]]] , & m \leq l < j < i \\ [\mathbf{S}_l, [\mathbf{S}_j, [\mathbf{S}_m, \mathbf{S}_i]]] , & l < j \leq m < i \\ [\mathbf{S}_j, [\mathbf{S}_l, [\mathbf{S}_m, \mathbf{S}_i]]] , & j \leq l < m < i \\ [\mathbf{S}_m, [\mathbf{S}_j, [\mathbf{S}_l, \mathbf{S}_i]]] , & m < j \leq l < i \\ [\mathbf{S}_j, [\mathbf{S}_m, [\mathbf{S}_l, \mathbf{S}_i]]] , & j \leq m \leq l < i \end{cases}$$

The non-zero terms of the ν -th derivative of \mathbf{S}_i are

$$\begin{aligned}\frac{\partial^\nu \mathbf{S}_i}{\partial q_{\alpha_1} \partial q_{\alpha_2} \cdots \partial q_{\alpha_\nu}} &= [\mathbf{S}_{\beta_\nu}, [\mathbf{S}_{\beta_{\nu-1}}, [\mathbf{S}_{\beta_{\nu-2}}, \dots [[\mathbf{S}_{\beta_1}, \mathbf{S}_i] \dots]]]] \\ &= \text{ad}_{\mathbf{S}_{\beta_\nu}} \text{ad}_{\mathbf{S}_{\beta_{\nu-1}}} \text{ad}_{\mathbf{S}_{\beta_{\nu-2}}} \cdots \text{ad}_{\mathbf{S}_{\beta_1}} \mathbf{S}_i\end{aligned}$$

where $\beta_\nu \leq \beta_{\nu-1} \leq \cdots \leq \beta_1 < i$ is the ordered sequence of the indices $\alpha_1, \dots, \alpha_\nu$

There are $\nu!$ different non-vanishing terms!

Introduce $S_i^\nu(\mathbf{q}, \mathbf{q}^{(\nu)}) := \sum_{j \leq i} \mathbf{S}_j(\mathbf{q}) \frac{d^\nu}{dt^\nu} q_j$

$$\rightarrow \dot{S}_i^\nu = \sum_{j < i} [\mathbf{S}_j^1, \mathbf{S}_j] \frac{d^\nu}{dt^\nu} q_j + S_i^{\nu+1}, \quad \dot{\mathbf{S}}_i = [\mathbf{S}_i^1, \mathbf{S}_i]$$

- Arbitrary k -th order constraints:

Velocity constraints:

$$\mathbf{0} = S_n^1(\mathbf{q}, \dot{\mathbf{q}})$$

Acceleration constraints:

$$\begin{aligned} \mathbf{0} &= \dot{S}_n^1(\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}}) \\ &= \sum_{j < i} [\mathbf{S}_j^1, \mathbf{S}_j] \dot{q}_j + S_n^2 \end{aligned}$$

Jerk constraints:

$$\begin{aligned} \mathbf{0} &= \ddot{S}_n^1(\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}}, \dddot{\mathbf{q}}) \\ &= \sum_{j < n} ([\dot{S}_j^1, \mathbf{S}_j] \dot{q}_j + [S_j^1, \dot{\mathbf{S}}_j] \dot{q}_j + [S_j^1, \mathbf{S}_j] \ddot{q}_j) + \dot{S}_n^2 \\ &= \sum_{j < n} ([[S_j^1, \mathbf{S}_j], S_n^1] \dot{q}_j + 2[S_j^1, [S_j^1, \mathbf{S}_n]] \dot{q}_j + [S_j^2, \mathbf{S}_j] \dot{q}_j + 2[S_j^1, \mathbf{S}_j] \ddot{q}_j) + S_n^3 \end{aligned}$$

Introduce $S_i^\nu(\mathbf{q}, \mathbf{q}^{(\nu)}) := \sum_{j \leq i} \mathbf{S}_j(\mathbf{q}) \frac{d^\nu}{dt^\nu} q_j$

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- Arbitrary k -th order constraints:

$$D^{(k)} \mathbf{S}_i = \sum_{r=0}^{k-1} \binom{k-1}{r} [D^{(r)} \mathbf{S}_i^1, D^{(k-r-1)} \mathbf{S}_i]$$

$$D^{(k)} := \frac{d^k}{dt^k}$$

$$D^{(k)} S_i^\nu = \sum_{j \leq i} \sum_{r=0}^{k-1} \binom{k-1}{r} D^{(k-r)} \mathbf{S}_j q_j^{(k+r)} + D^{(k-1)} S_i^{\nu+1}$$

- A finite trajectory $\mathbf{q}(t)$ of a mechanism satisfies constraints of any order:

$$\mathbf{0} = S_n^1(\mathbf{q}, \dot{\mathbf{q}})$$

$$\mathbf{0} = \dot{S}_n^1(\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}})$$

$$\mathbf{0} = \ddot{S}_n^1(\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}}, \dddot{\mathbf{q}})$$

$$\vdots$$

$$\mathbf{0} = \frac{d^\nu}{dt^\nu} S_n^1(\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}}, \mathbf{q}^{(\nu+1)})$$

$$\vdots$$

- i -th order Cone: $K_{\mathbf{q}}^i := \{\mathbf{x} | \exists \mathbf{y}, \mathbf{z}, \dots \in \mathbb{R}^n : H^{(1)}(\mathbf{q}, \mathbf{x}) = \mathbf{0}, H^{(2)}(\mathbf{q}, \mathbf{x}, \mathbf{y}) = \mathbf{0}, H^{(3)}(\mathbf{q}, \mathbf{x}, \mathbf{y}, \mathbf{z}) = \mathbf{0}, \dots, H^{(i)}(\mathbf{q}, \mathbf{x}, \mathbf{y}, \mathbf{z}, \dots) = \mathbf{0}\}$

- with i -th order constraint Mappings:

$$\begin{aligned} H^{(1)}(\mathbf{q}, \dot{\mathbf{q}}) &:= S_n^1(\mathbf{q}, \dot{\mathbf{q}}) \\ H^{(2)}(\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}}) &:= \frac{d}{dt} S_n^1(\mathbf{q}, \dot{\mathbf{q}}) \\ &\dots \\ H^{(i)}(\mathbf{q}, \dot{\mathbf{q}}, \dots, \mathbf{q}^{(i)}) &:= D^{(i-1)} S_n^1(\mathbf{q}, \dot{\mathbf{q}}). \end{aligned}$$



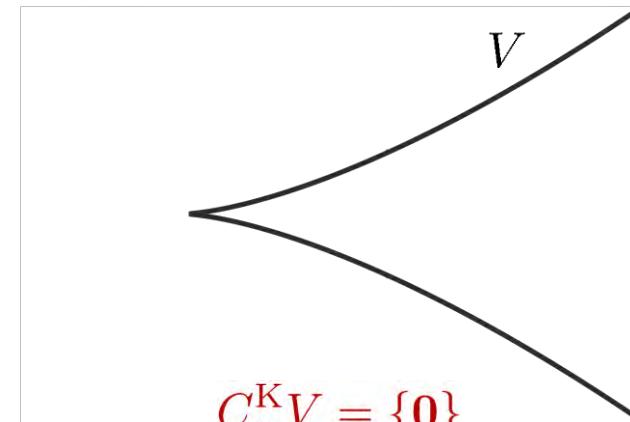
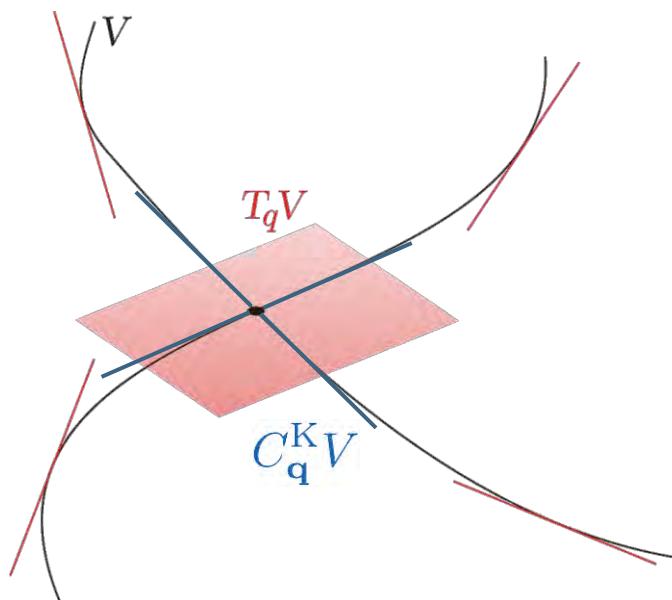
‘Kinematic’ Tangent Cone:

$$C_{\mathbf{q}}^K V = K_{\mathbf{q}}^\kappa \subset \dots \subset K_{\mathbf{q}}^3 \subset K_{\mathbf{q}}^2 \subset K_{\mathbf{q}}^1$$

Definition:

The **kinematic tangent cone** to V at q is the set $C_q^K V$ of tangents to smooth curves in V through q

- $x \in C_q^K V$ can be thought of as velocities
- $C_q^K V$ reveals the finite motions *through* q
- May not describe the local geometry of V



Definition:

The **kinematic tangent cone** to V at \mathbf{q} is the set $C_{\mathbf{q}}^K V$ of tangents to smooth curves in V *through* \mathbf{q}

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- May not describe the local geometry of V

$$\delta_{\text{loc}}(\mathbf{q}) = \dim_{\mathbf{q}} V \geq \dim C_{\mathbf{q}} V$$

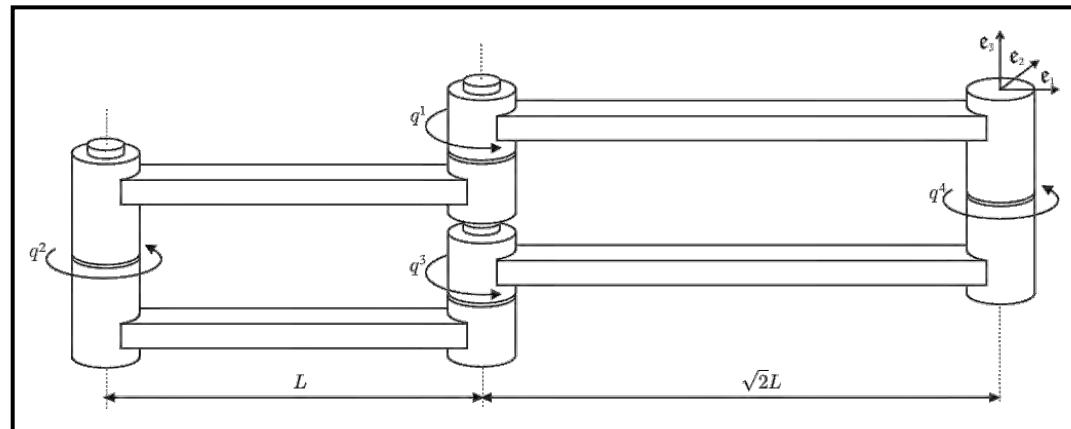
Example: planar 4-bar linkage

- Screw coordinates at $\mathbf{q}_0 = \mathbf{0}$

$$\mathbf{Y}_1 = \mathbf{Y}_3 = (0, 0, 1, 0, -L, 0)^T$$

$$\mathbf{Y}_2 = (0, 0, 1, 0, -2L, 0)^T$$

$$\mathbf{Y}_4 = (0, 0, 1, 0, 0, 0)^T$$



- Tangent cone: $K_{\mathbf{0}}^1 = \mathbf{V}(x_1 + x_3 + 2x_4, x_2 - x_4) \subset \mathbb{R}^4$

$$K_{\mathbf{0}}^2 = K_{\mathbf{0}}^{2,1} \cup K_{\mathbf{0}}^{2,2} \quad \text{with} \quad K_{\mathbf{0}}^{2,1} = \mathbf{V}(x_1 + x_3, x_2, x_4)$$

$$K_{\mathbf{0}}^{2,2} = \mathbf{V}(x_1 + x_2, x_3 + x_2, x_4 - x_2)$$

$$K_{\mathbf{0}}^3 = K_{\mathbf{0}}^2 = C_{\mathbf{0}}V.$$

- Mobility: $\delta_{\text{diff}}(\mathbf{0}) = \dim K_{\mathbf{0}}^1 = 2$

$$\delta_{\text{loc}}(\mathbf{0}) = \dim C_{\mathbf{0}}V = 1$$

- Singularity?:

$$K_{\mathbf{0}}^1 = T_{\mathbf{0}}V \equiv \text{span } C_{\mathbf{q}}V \neq C_{\mathbf{q}}V \quad \rightarrow \quad \mathbf{q}_0 = \mathbf{0} \text{ is a c-space singularity}$$

$$\rightarrow \mathbf{q}_0 = \mathbf{0} \text{ is a kinematic singularity}$$

Example: planar 5-bar linkage

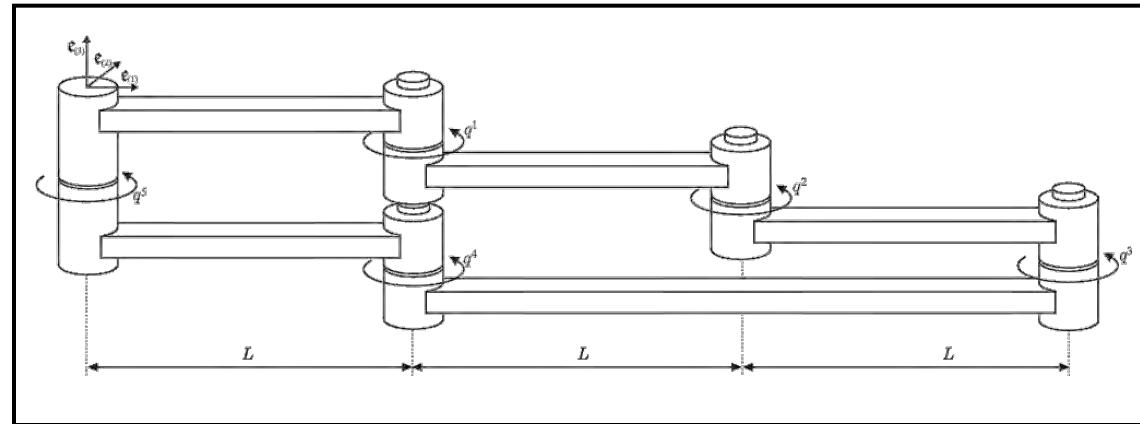
- Screw coordinates at $\mathbf{q}_0 = \mathbf{0}$

$$\mathbf{Y}_1 = (0, 0, 1, 0, 0, 0)^T$$

$$\mathbf{Y}_2 = \mathbf{Y}_5 = (0, 0, 1, 0, -1, 0)^T$$

$$\mathbf{Y}_3 = (0, 0, 1, 0, -2, 0)^T$$

$$\mathbf{Y}_4 = (0, 0, 1, 0, -3, 0)^T$$



- Tangent cone: $K_{\mathbf{0}}^1 = \mathbf{V}(x_1 - x_3 - 2x_4, x_2 + 2x_3 + 3x_4 + x_5) \subset \mathbb{R}^5$

$$K_{\mathbf{0}}^2 = \mathbf{V}\left(x_2(x_3 + 2x_4) + (x_3 + x_4)(x_3 + 3x_4), x_1 - x_3 - 2x_4, x_4^2 + (x_3 + 2x_4)(x_3 + x_4 + x_5)\right)$$

$$K_{\mathbf{0}}^3 = K_{\mathbf{0}}^2 = C_{\mathbf{0}}V.$$

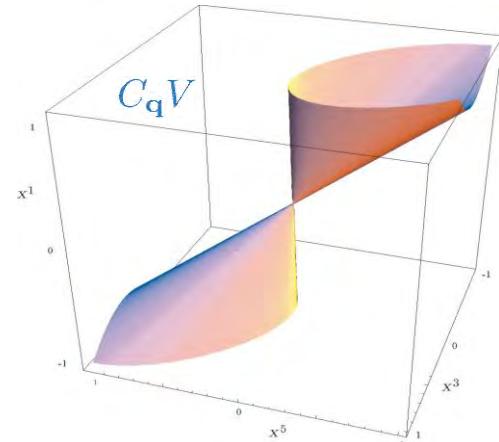
- Mobility: $\delta_{\text{diff}}(\mathbf{0}) = \dim K_{\mathbf{0}}^1 = 3$

$$\delta_{\text{loc}}(\mathbf{0}) = \dim C_{\mathbf{0}}V = 2.$$

- Singularity?:

$K_{\mathbf{0}}^1 = T_{\mathbf{0}}V \equiv \text{span } C_{\mathbf{q}}V \neq C_{\mathbf{q}}V \rightarrow \mathbf{q}_0 = \mathbf{0}$ is a c-space singularity

$\rightarrow \mathbf{q}_0 = \mathbf{0}$ is a kinematic singularity



Example: pathological spherical 4-bar linkage

- Screw coordinates at $\mathbf{q}_0 = \mathbf{0}$

$$\mathbf{Y}_1 = (\sqrt{3}/2, 0, 1/2, 0, 0, 0)^T$$

$$\mathbf{Y}_2 = (1/2, 0, \sqrt{3}/2, 0, 0, 0)^T$$

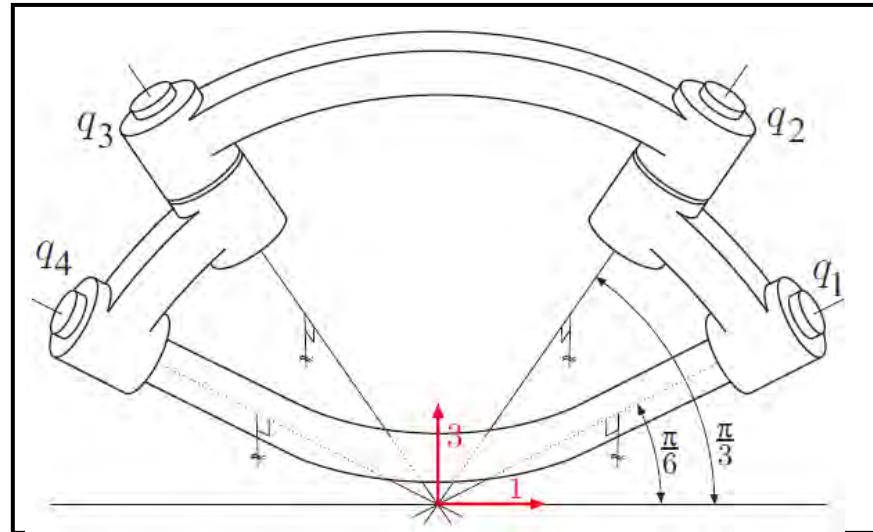
$$\mathbf{Y}_3 = (-1/2, 0, \sqrt{3}/2, 0, 0, 0)^T$$

$$\mathbf{Y}_4 = (-\sqrt{3}/2, 0, 1/2, 0, 0, 0)^T$$

- Tangent cone:

$$K_{\mathbf{0}}^1 = \mathbf{V}(x_1 - \sqrt{3}x_3 - 2x_4, x_2 - 2x_3 + \sqrt{3}x_4) \subset \mathbb{R}^4$$

$$K_{\mathbf{0}}^2 = \{\mathbf{0}\}, K_{\mathbf{0}}^3 = K_{\mathbf{0}}^2 = C_{\mathbf{0}}V.$$



- Mobility: $\delta_{\text{diff}}(\mathbf{0}) = \dim K_{\mathbf{0}}^1 = 2$

$$\delta_{\text{loc}}(\mathbf{0}) = \dim C_{\mathbf{0}}V = (\delta_{\text{loc}}(\mathbf{0}) = \dim C_{\mathbf{0}}V = 0)$$

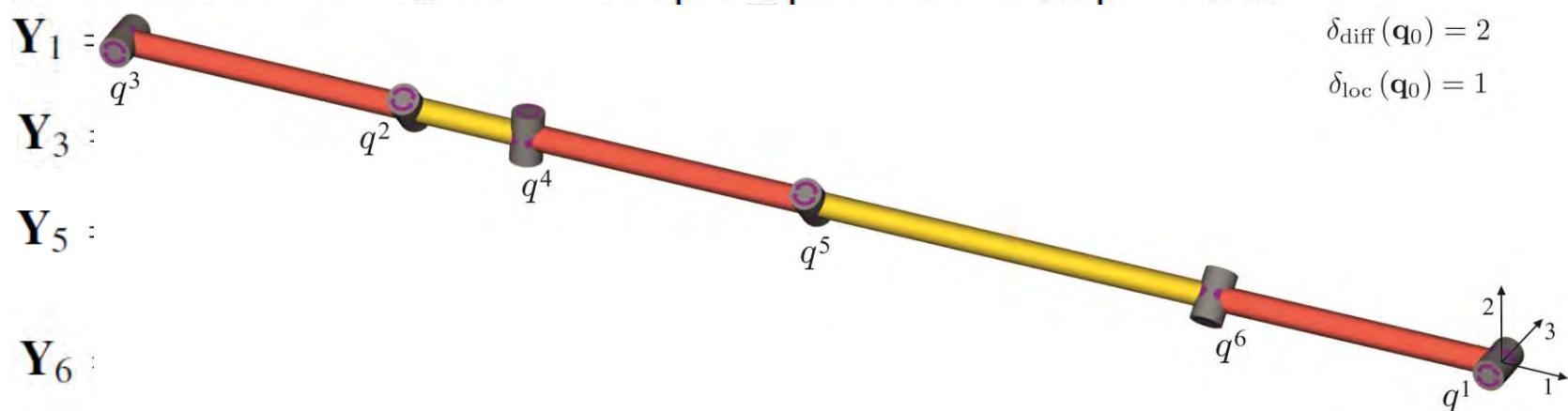
- Singularity?: $T_{\mathbf{0}}V \equiv \text{span } C_{\mathbf{q}}V = C_{\mathbf{q}}V \rightarrow \mathbf{q}_0 = \mathbf{0}$ is a regular point of V

$$\delta_{\text{loc}}(\mathbf{q}) = 0 \rightarrow \mathbf{q}_0 = \mathbf{0}$$
 is not a kinematic singularity

$$\delta_{\text{diff}}(\mathbf{0}) \neq \delta_{\text{loc}}(\mathbf{0}) \rightarrow \text{Linkage is underconstrained}$$

Example: pathological spherical 4-bar linkage

- Screw coordinates at $\mathbf{q}_0 = \mathbf{0}$



- Tangent cone: $K_0^1 = \mathbf{V}(2x_1 + x_5 + (\sqrt{2} - 1)x_6, 2x_3 - x_5 + (\sqrt{2} - 1)x_6, x_2 + (1 + \sqrt{2})x_5 - x_6, x_4 - x_5 + \sqrt{2}x_6) \subset \mathbb{R}^6$

$$K_0^2 = \mathbf{V}(x_1 + x_5, x_2, x_3, x_4 + (1 + \sqrt{2})x_5, x_6 - (1 + \sqrt{2})x_5)$$

$$K_0^3 = K_0^2 = C_0 V.$$

- Mobility: $\delta_{\text{diff}}(\mathbf{0}) = \dim K_0^1 = 2$
 $\delta_{\text{loc}}(\mathbf{0}) = \dim C_0 V = 1$

- Singularity?: $T_0 V \equiv \text{span } C_{\mathbf{q}} V = C_{\mathbf{q}} V \rightarrow \mathbf{q}_0 = \mathbf{0}$ is a regular point of V

But $\mathbf{q}_0 = \mathbf{0}$ is a kinematic singularity! But how do we know that?

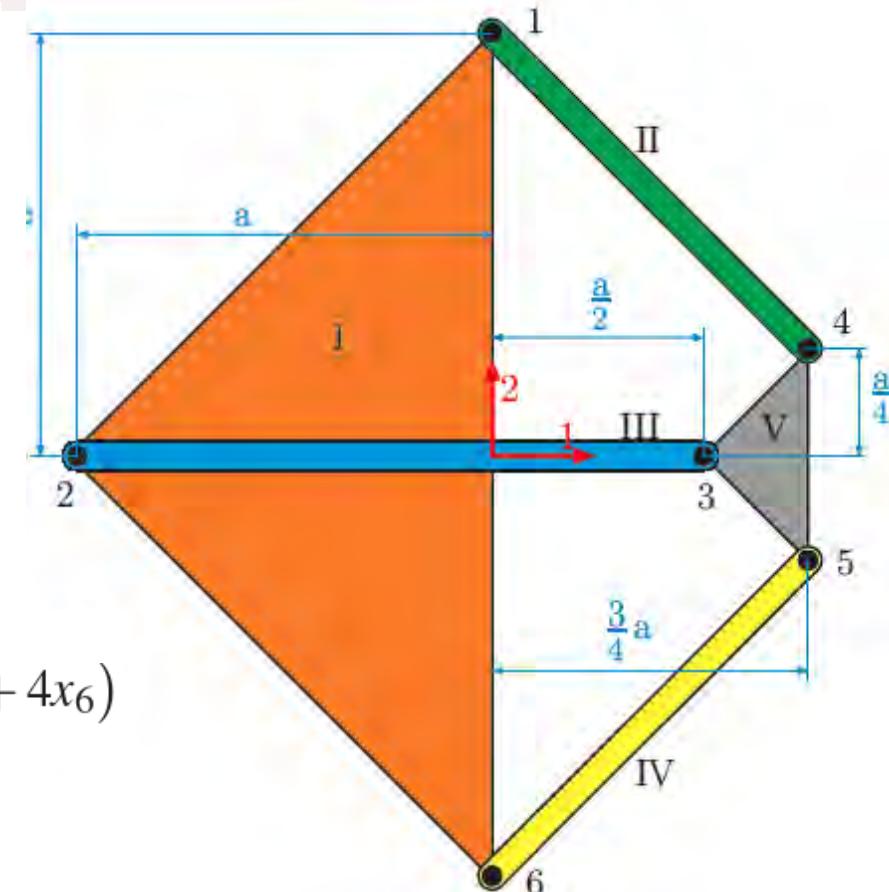
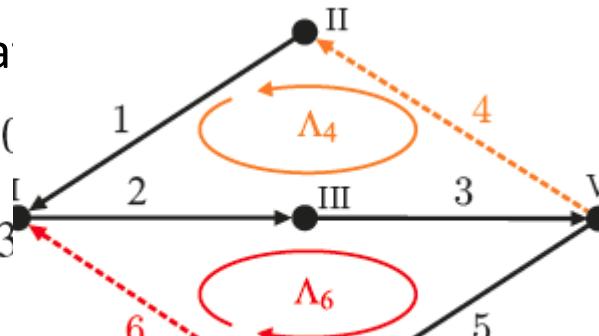
Example: Shaky 2-Loop Linkage (Assur graph)

- Screw coordinates:

$$\mathbf{Y}_1 = (0, 0, 1, 6, 0, 0)$$

$$\mathbf{Y}_3 = (0, 0, 1, 0, -3, 0)$$

$$\mathbf{Y}_5 = (0, 0, 1, -\frac{3}{2}, 0, 0)$$



- Tangent cone:

$$K_0^1 = K_0^2 = K_0^3$$

$$= \mathbf{V}(x_1 - x_6, x_2 + x_6, x_3 - 4x_6, x_4 + 4x_6, x_5 + 4x_6)$$

$$K_0^4 = C_0 V = \{\mathbf{0}\}$$

- Mobility: $\delta_{\text{diff}}(\mathbf{0}) = \dim K_0^1 = 1$

$$\delta_{\text{loc}}(\mathbf{0}) = \dim C_0 V = 0$$

- Singularity?: $T_0 V \equiv \text{span } C_q V = C_q V \rightarrow q_0 = \mathbf{0}$ is a regular point of V
 $\delta_{\text{loc}}(q) = 0 \rightarrow q_0 = \mathbf{0}$ is not a kinematic singularity

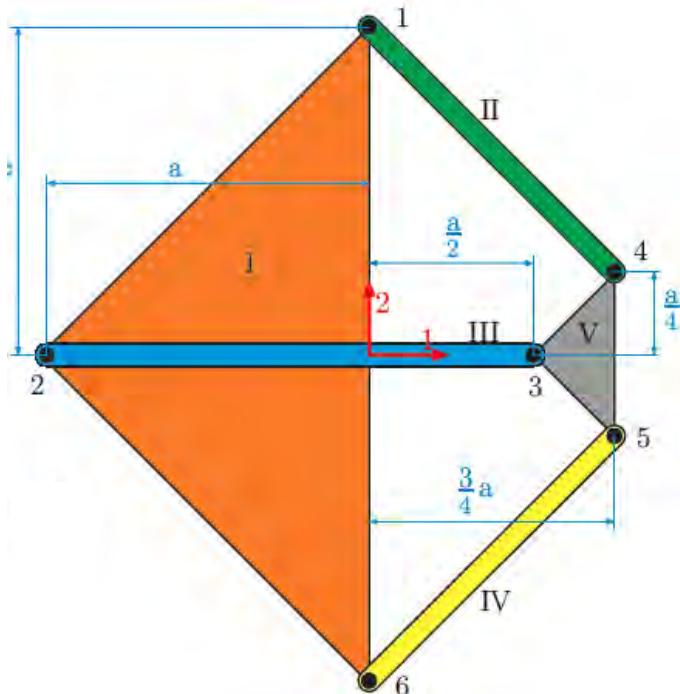
$\delta_{\text{diff}}(\mathbf{0}) \neq \delta_{\text{loc}}(\mathbf{0}) \rightarrow$ Linkage is underconstrained

Example: Shaky 2-Loop Linkage (Assur graph)

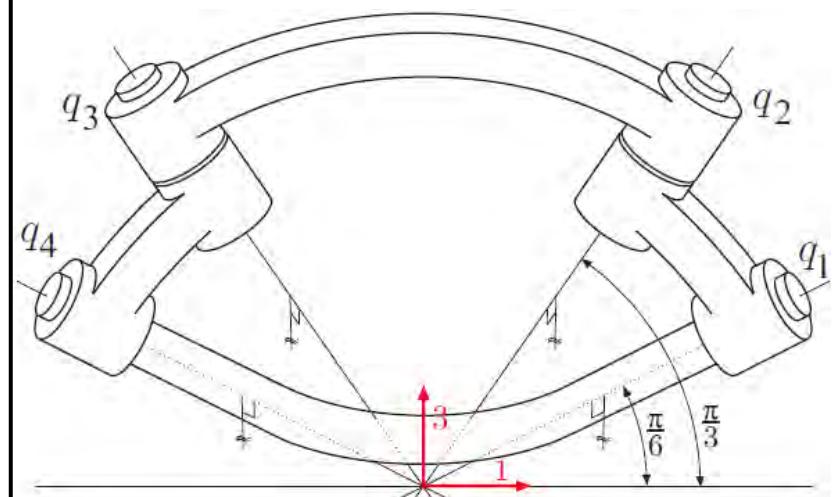
- Definition:

If $C_{\mathbf{q}}V = K_{\mathbf{q}}^{\kappa} \subset K_{\mathbf{q}}^{\kappa-1}$ for $\kappa > 1$, the linkage is **shaky of order** $i = \kappa - 1$ and the configuration \mathbf{q} is called **shaky**. The dimension $\dim K_{\mathbf{q}}^i$ is referred to as the ***i*th-order DOF** at \mathbf{q} . If there is a regular shaky configuration, the linkage is **underconstrained**.

Shaky of order $i = 1, 2, 3$



Shaky of order $i = 1, 2$
Underconstrained



Example: Shaky 2-Loop Linkage (Assur graph)

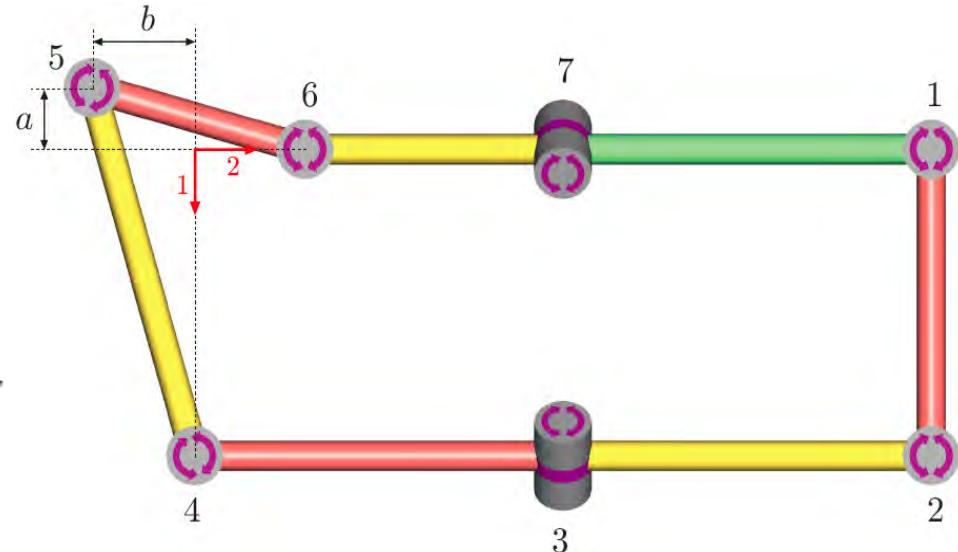
- Screw coordinates:

$$\mathbf{Y}_1 = (0, 0, 1, 110, 0, 0)^T, \mathbf{Y}_2 = (0, 0, 1, 110, -40, 0)^T$$

$$\mathbf{Y}_3 = \left(-\frac{1}{2}, 0, \frac{1}{2}\sqrt{3}, 30\sqrt{3}, -20\sqrt{3}, 30\right)^T$$

$$\mathbf{Y}_4 = (0, 0, 1, 0, -40, 0)^T, \mathbf{Y}_5 = (0, 0, 1, -b, a, 0)^T$$

$$\mathbf{Y}_6 = (0, 0, 1, 25, 0, 0)^T, \mathbf{Y}_7 = \left(-\frac{1}{2}, 0, \frac{1}{2}\sqrt{3}, 30\sqrt{3}, 0, 30\right)^T$$



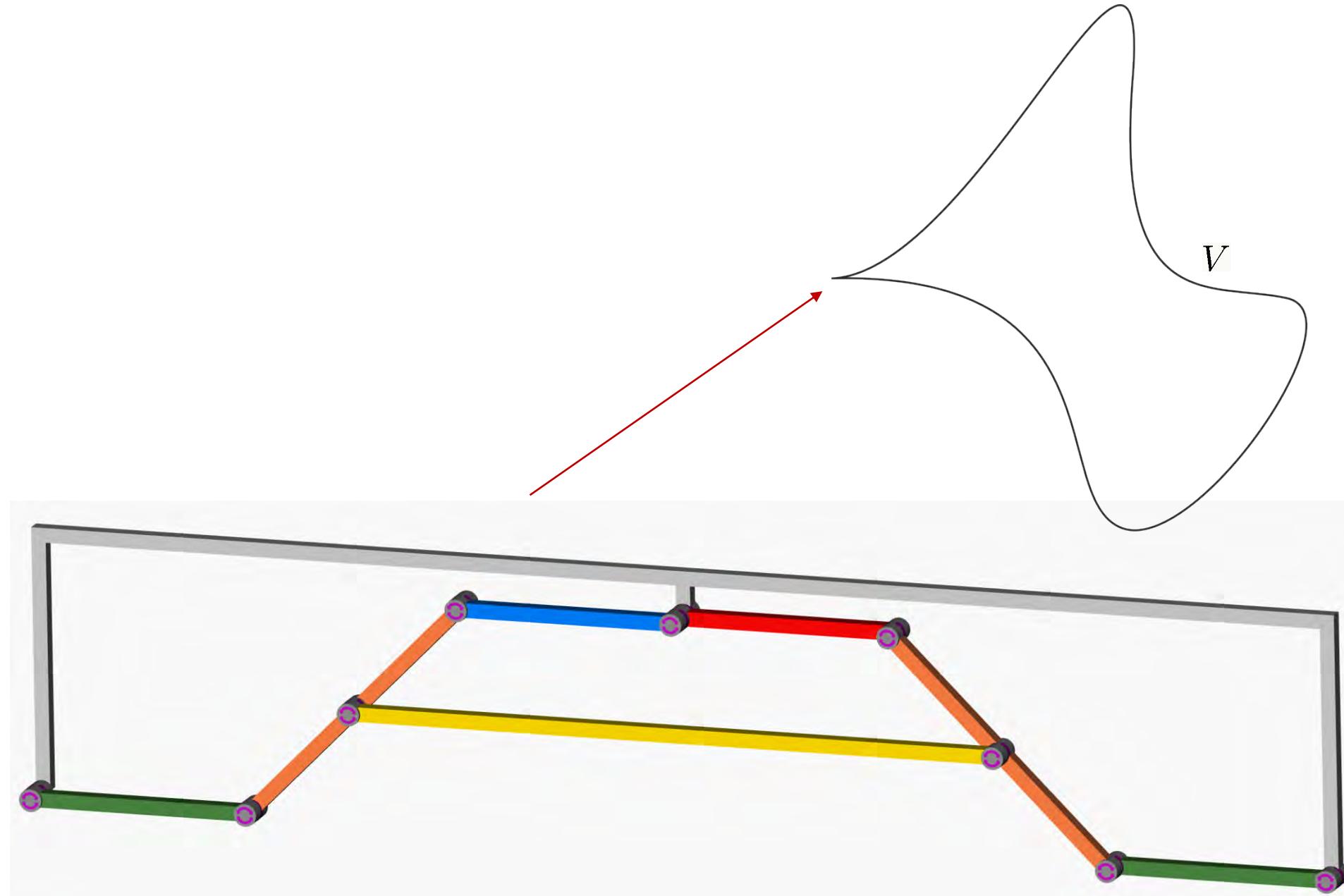
X. Kong, M. Pfurner,
Mech. Mach. Theory, Vol. 85, 2015

$$K_0^1 = \mathbf{V}(x_3 + x_7, 40x_2 + 20\sqrt{3}x_3 + 40x_4 - ax_5, \\ x_1 + x_2 + \frac{1}{2}\sqrt{3}x_3 + x_4 + x_5 + x_6 + \frac{1}{2}\sqrt{3}x_7, \\ 110(x_1 + x_2) + 30\sqrt{3}(x_3 + x_7) - bx_5 + 25x_6) \in \mathbb{R}^7$$

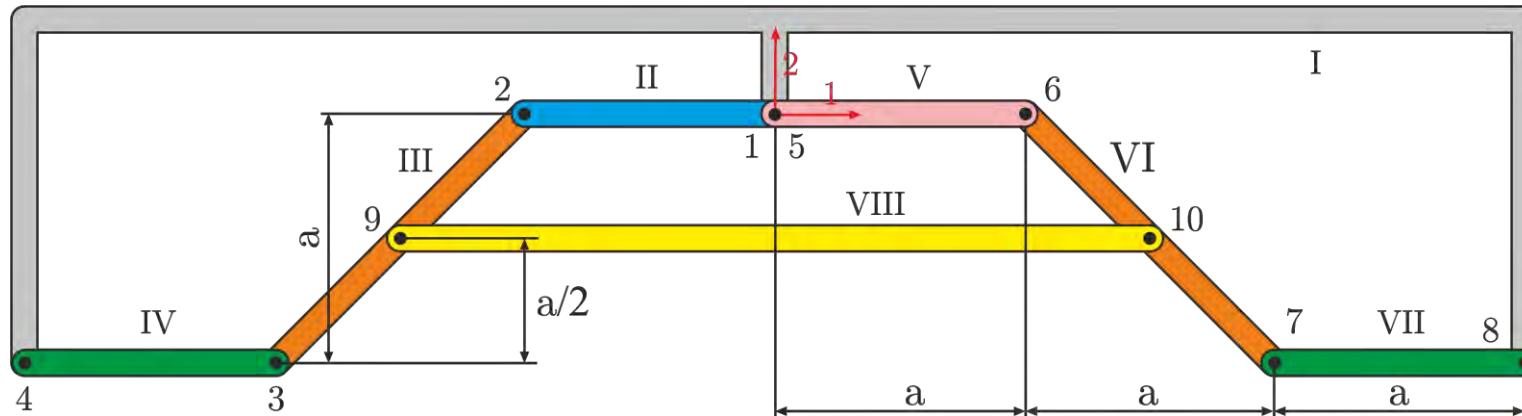
$$K_0^2 = K_0^{2,1} \cup K_0^{2,2} \quad \text{with} \quad K_0^{2,1} = \mathbf{V}(200x_1 - (200 + 5a + 8b)x_5, 200x_2 + (200 + 5a + 8b)x_5, \\ 25x_4 + 25x_5 + bx_5, 150x_7 - \sqrt{3}(200 + 5a + 8b)x_5x_7, \\ x_6 - b/25x_5, \sqrt{3}(200 + 5a + 8b)x_5 - 150x_3x_7). \\ K_0^{2,2} = \mathbf{V}(680x_1 + 880x_2 - (200 + 5a + 8b)x_5, 40x_4 + 40x_2 - ax_5, \\ 340x_6 + (440 + 11a + 4b)x_5 - 440x_2, x_3, x_7)$$

$$\dim K_0^1 = 3 \quad \boxed{\dim K_0^{2,1} = 1, \dim K_0^{2,2} = 2}$$

Examples: Double-Watt linkage



Examples: Double-Watt linkage



- Fundamental cycles: $\gamma = 3$

- Joint screw coordinates:

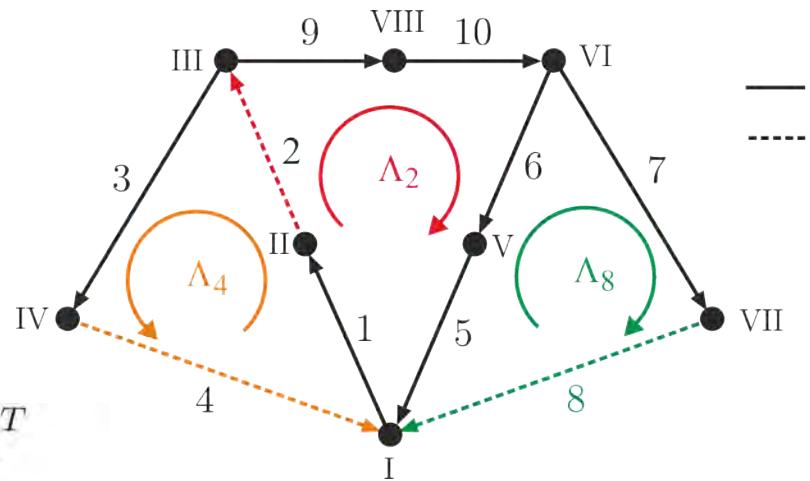
$$\mathbf{Y}_1 = \mathbf{Y}_5 = (0, 0, 1, 0, 0, 0)^T, \mathbf{Y}_2 = (0, 0, 1, 0, 1, 0)^T$$

$$\mathbf{Y}_3 = (0, 0, 1, -1, 2, 0)^T, \mathbf{Y}_4 = (0, 0, 1, -1, 3, 0)^T$$

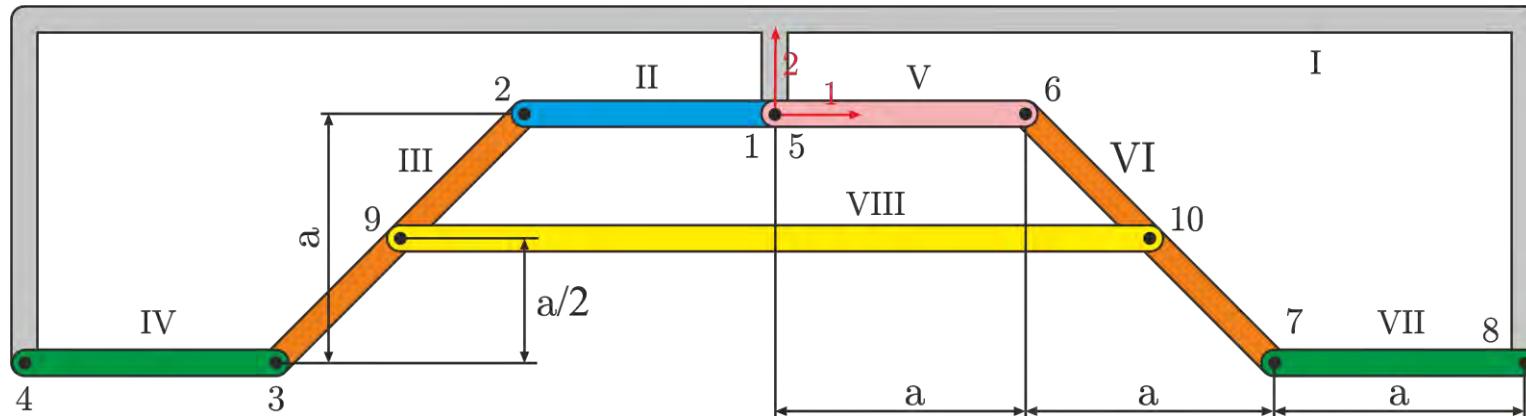
$$\mathbf{Y}_6 = (0, 0, 1, 0, -1, 0)^T, \mathbf{Y}_7 = (0, 0, 1, -1, -2, 0)^T$$

$$\mathbf{Y}_8 = (0, 0, 1, -1, -3, 0)^T, \mathbf{Y}_9 = (0, 0, 1, -1/2, 3/2, 0)^T$$

$$\mathbf{Y}_{10} = (0, 0, 1, -1/2, -3/2, 0)^T$$



Examples: Double-Watt linkage



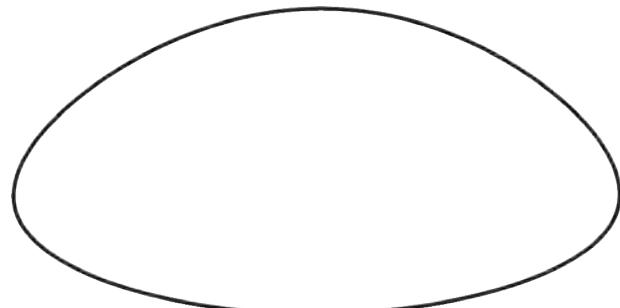
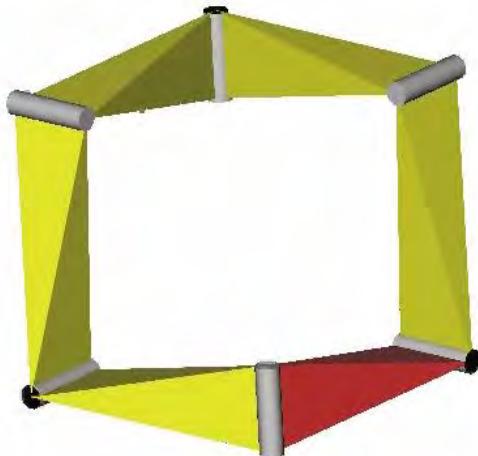
- Cones: $K_{\mathbf{q}_0}^1 = \mathbf{V}(x_1 + x_2, x_1 + x_3, x_1 - x_4, x_5 + x_6, x_5 - x_7, x_5 + x_8, x_1 - x_5 - 3x_9, x_1 + 3x_{10} - x_5)$

$$K_{\mathbf{q}_0}^2 = \mathbf{V}(x_1 + x_2, x_1 + x_3, x_4 - x_1, x_5 - x_1, x_1 + x_6, x_7 - x_1, x_1 + x_8, x_9, x_{10})$$

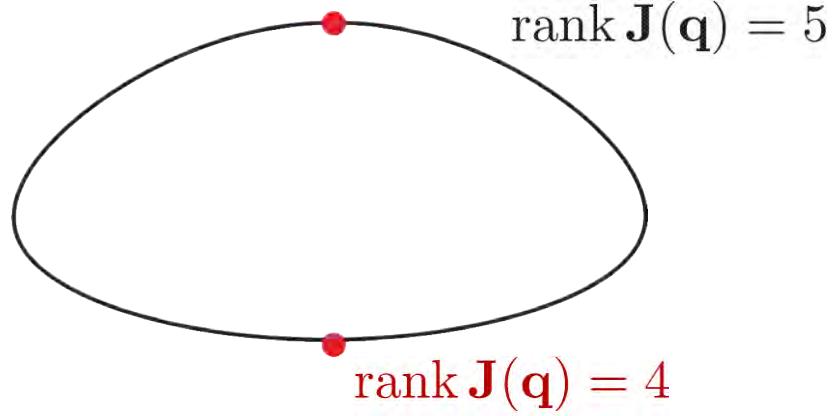
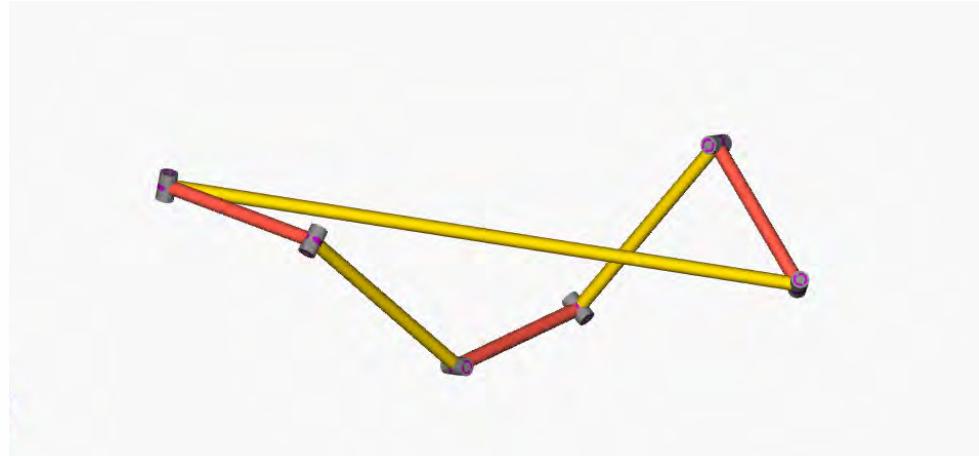
$$K_{\mathbf{q}_0}^3 = \{\mathbf{0}\}$$

$$\rightarrow C_{\mathbf{q}_0}^K V = \{\mathbf{0}\}$$

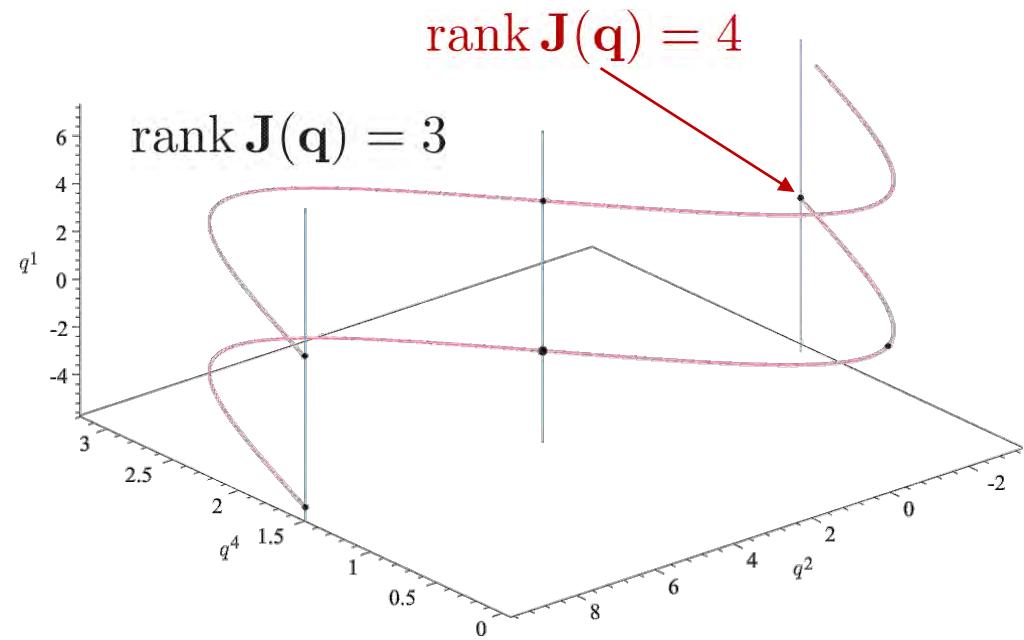
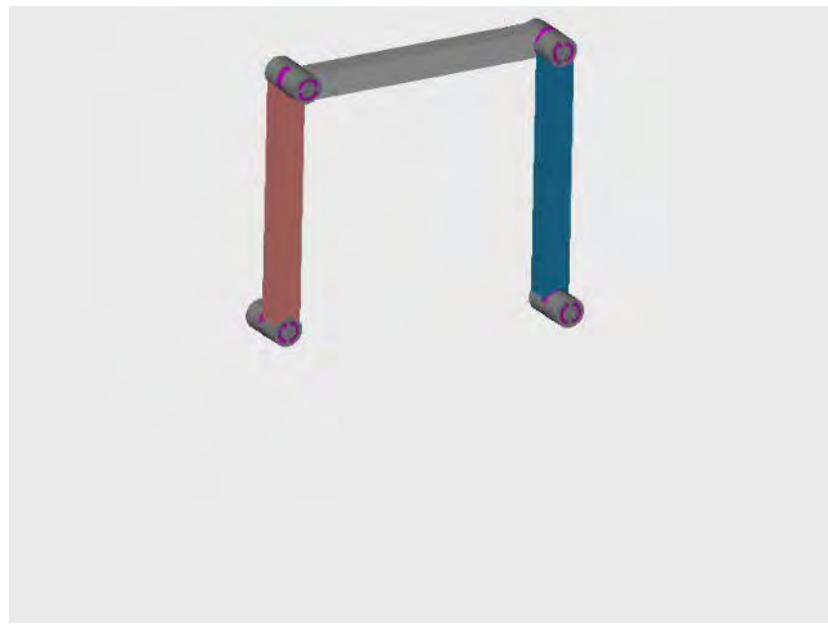
- Identifying kinematic singularities boils down to determine the varieties of points with certain rank



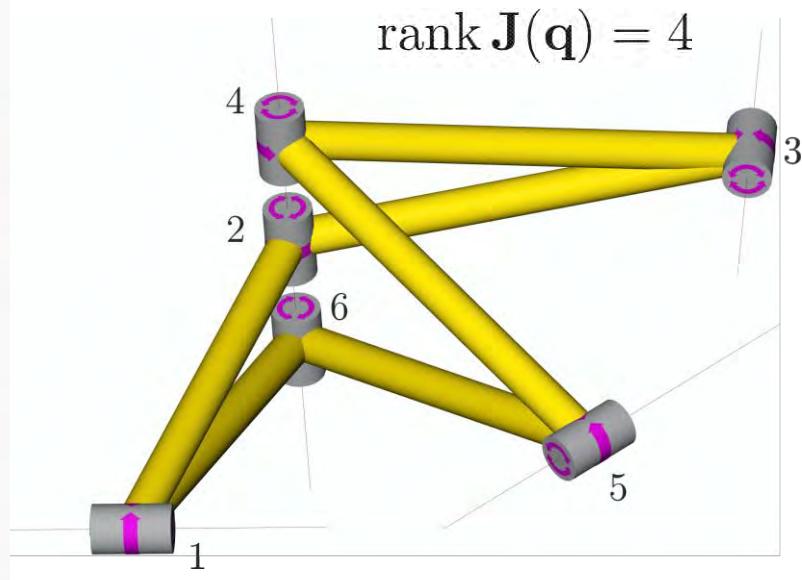
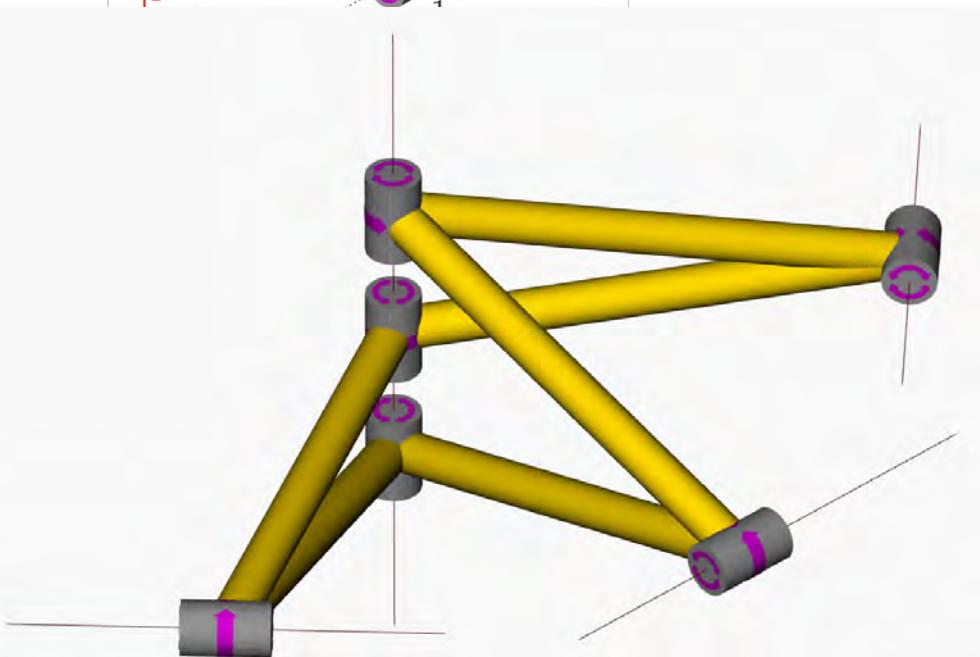
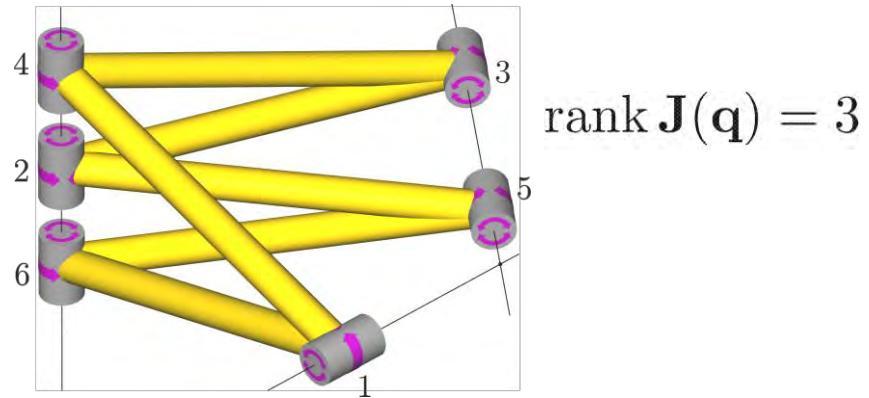
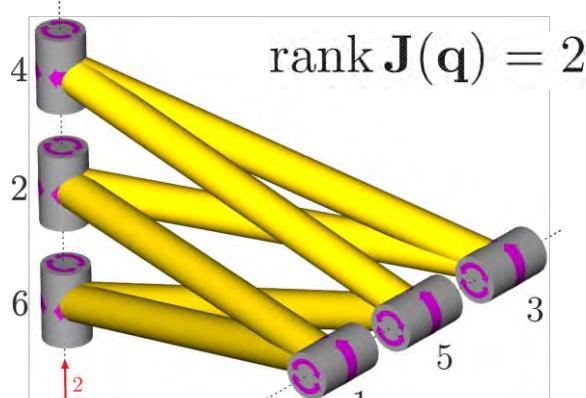
$\text{rank } \mathbf{J}(\mathbf{q}) = 5 \quad \forall \mathbf{q} \in V$



- Identifying kinematic singularities boils down to determine the varieties of points with certain rank



- Identifying kinematic singularities boils down to determine the varieties of points with certain rank



- Set of critical points with rank less than k :

$$\begin{aligned}
 L_k &:= \{\mathbf{q} \in V \mid \text{rank } \mathbf{J}(\mathbf{q}) < k\} \\
 &= \{\mathbf{q} \in \mathbb{V}^n \mid f(\mathbf{q}) = \mathbf{I}, m_{\mathbf{ab}}(\mathbf{q}) = 0 \\
 &\quad \forall \mathbf{a} \subseteq \{1, \dots, 6\}, \mathbf{b} \subseteq \{1, \dots, n\}, |\mathbf{a}| = |\mathbf{b}| = k\}
 \end{aligned}$$

where $m_{\mathbf{ab}}(\mathbf{q}) := \det \mathbf{J}_{\mathbf{ab}}(\mathbf{q})$

$\mathbf{J}_{\mathbf{ab}}$ is the \mathbf{ab} -minor of \mathbf{J} of order k

- $C_{\mathbf{q}}^K L_k$ consists of tangents to finite curves (motions) through \mathbf{q} with $\text{rank } \mathbf{J} < k$
- Computation...

$$\mathbf{J} = (\mathbf{S}_1 \cdots \mathbf{S}_k) \quad \mathbf{S}_i(\mathbf{q}) \text{ – screw coordinates of joint } i$$

We need $\frac{d^i}{dt^i} m_{\mathbf{ab}}(\mathbf{q}(t)) = \frac{d^i}{dt^i} \det \mathbf{J}_{\mathbf{ab}}(\mathbf{q}(t))$

- Geometric loop constraints:

$$f(\mathbf{q}) = \mathbf{I}$$

with $f(\mathbf{q}) := \exp(\mathbf{Y}_1 q_1) \exp(\mathbf{Y}_2 q_2) \cdot \dots \cdot \exp(\mathbf{Y}_n q_n)$

- Velocity constraints

$$\mathbf{J}(\mathbf{q}) \dot{\mathbf{q}} = \mathbf{0}$$

- Derivative of determinant

$$\begin{aligned} \frac{d^i}{dt^i} m_{\mathbf{ab}}(\mathbf{q}(t)) &= \frac{d^i}{dt^i} \det \mathbf{J}_{\mathbf{ab}}(\mathbf{q}(t)) \\ &= \sum_{\substack{\mu_1 \neq \dots \neq \mu_p \in \mathbf{b} \\ \pi \in P(i)}} \left| \mathbf{S}_{\mathbf{a}b_1} \cdots \mathbf{S}_{\mathbf{a}\mu_1}^{(\pi_1)} \cdots \mathbf{S}_{\mathbf{a}\mu_p}^{(\pi_p)} \cdots \mathbf{S}_{\mathbf{a}b_k} \right| \frac{i!}{\pi_1! \cdots \pi_p! n_1! \cdots n_i!} \end{aligned}$$

$$\mathbf{S}_{\mathbf{a}j}^{(k)} := D^{(k)} \mathbf{S}_{\mathbf{a}j} \quad D^{(k)} \mathbf{S}_i = \sum_{r=0}^{k-1} \binom{k-1}{r} [D^{(r)} \mathbf{S}_i^1, D^{(k-r-1)} \mathbf{S}_i] \quad D^{(k)} := \frac{d^k}{dt^k}$$

Examples

- $C_{\mathbf{q}}^K V = \mathbf{V}(x_1 + x_3 + x_5, x_2, x_4, x_6) \cup \mathbf{V}(x_1 + x_3, x_4 + x_6, x_2, x_5)$
 $\cup \mathbf{V}(x_2 + x_6, x_3 + x_5, x_1, x_4) \cup \mathbf{V}(x_2 + x_4 + x_6, x_1, x_3, x_5)$
 $\cup \mathbf{V}(x_1 + x_5, x_2 + x_4, x_3, x_6)$

$$\delta_{\text{loc}}(\mathbf{q}) = \dim C_{\mathbf{q}}^K V = 2$$

- $C_{\mathbf{q}}^K L_6 = C_{\mathbf{q}}^K L_5 = C_{\mathbf{q}}^K V$

→ all points have (locally) maximal rank 4

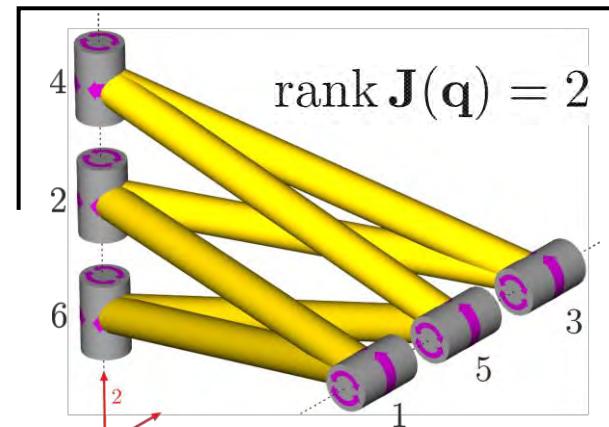
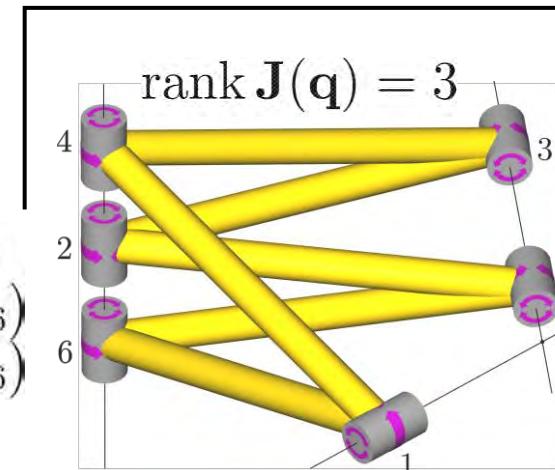
- $C_{\mathbf{q}_0}^K L_4 = \mathbf{V}(x_1, x_2, x_3, x_4 + x_6, x_5) \cup \mathbf{V}(x_1, x_2, x_3 + x_5, x_4, x_6)$
 $\cup \mathbf{V}(x_1, x_2 + x_6, x_3, x_4, x_5) \cup \mathbf{V}(x_1, x_2 + x_4, x_3, x_5, x_6)$
 $\cup \mathbf{V}(x_1 + x_3, x_2, x_4, x_5, x_6) \cup \mathbf{V}(x_1 + x_5, x_2, x_3, x_4, x_6)$

$$\dim C_{\mathbf{q}}^K L_4 = 1$$

→ points with $\text{rank } \mathbf{J}(\mathbf{q}) \leq 4$ (locally) 1-dim manifold

- $C_{\mathbf{q}}^K L_3 = \{\mathbf{0}\}$

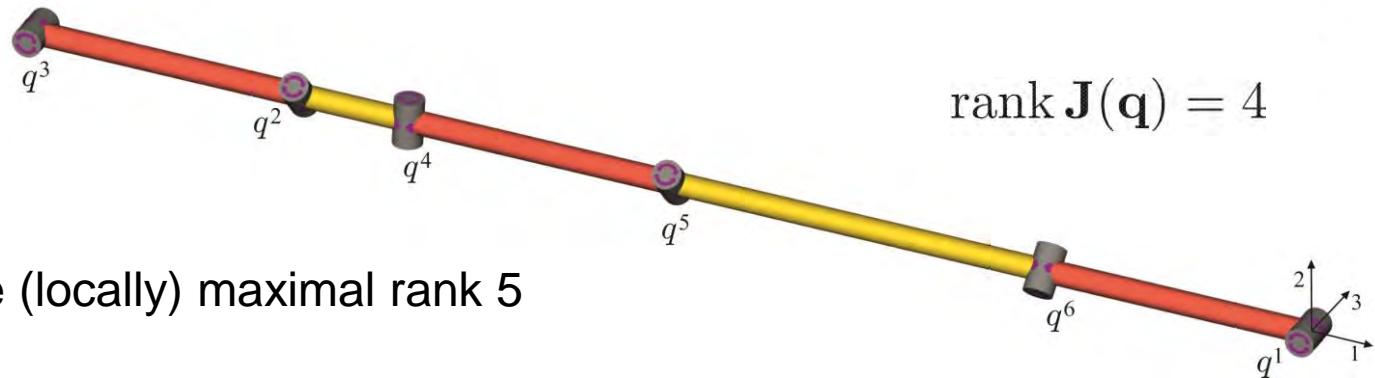
$$\dim C_{\mathbf{q}}^K L_3 = 0$$



Examples

- $C_{\mathbf{q}}^K V = \mathbf{V}(x_1 + x_5, x_2, x_3, x_4 + (1 + \sqrt{2})x_5, x_6 - (1 + \sqrt{2})x_5)$

$$\delta_{\text{loc}}(\mathbf{q}) = \dim C_{\mathbf{q}}^K V = 1$$



$$\text{rank } \mathbf{J}(\mathbf{q}) = 4$$

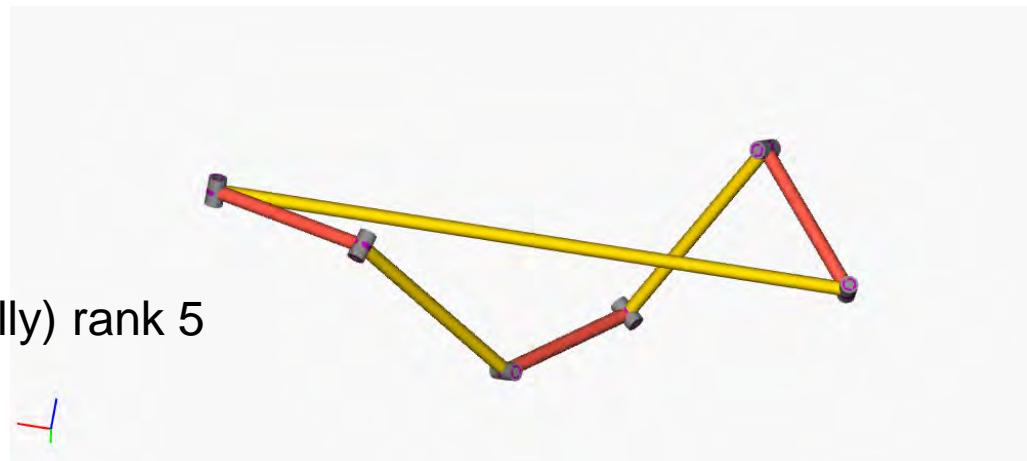
- $C_{\mathbf{q}}^K L_6 = C_{\mathbf{q}}^K V$

→ all points have (locally) maximal rank 5

- $C_{\mathbf{q}}^K L_5 = \{\mathbf{0}\}$

$$\dim C_{\mathbf{q}}^K L_5 = 0$$

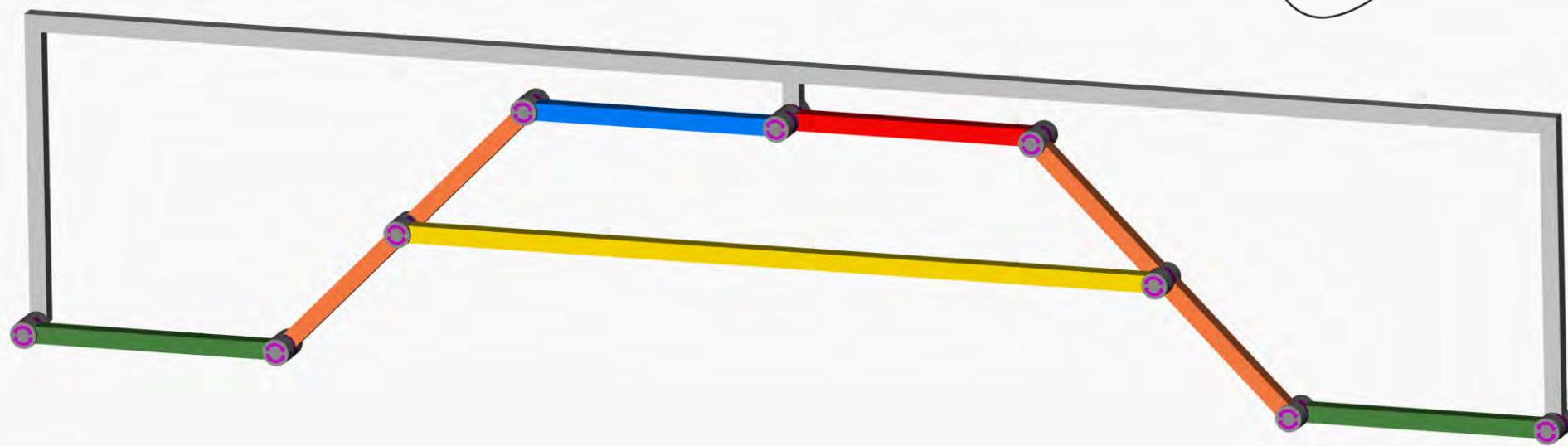
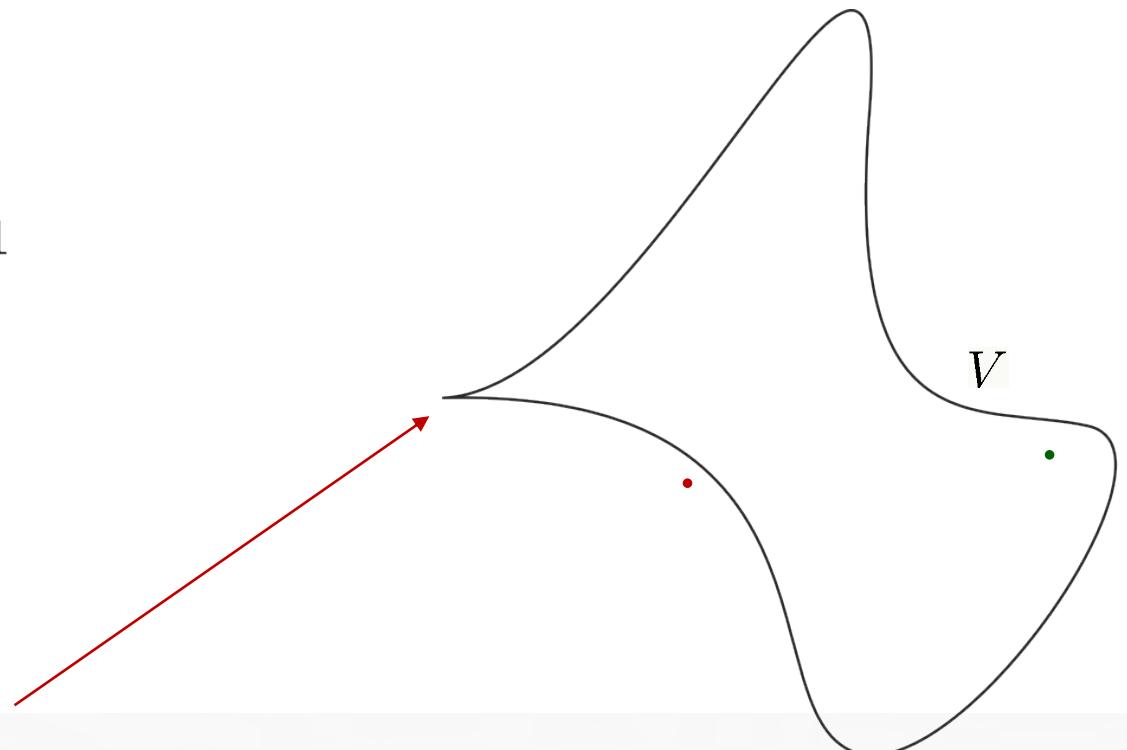
→ all points, except \mathbf{q} have (locally) rank 5



Examples

- $C_{\mathbf{q}}^K V = C_{\mathbf{q}}^K L_5 = \{\mathbf{0}\}$

$$\dim C_{\mathbf{q}}^K V = 0 \neq \delta_{\text{loc}}(\mathbf{q}) = 1$$



- Understanding of kinematic singularities of general linkages is still open
- No generally applicable math framework/algorithm for (local) analysis
- We have a method to investigate finite motions with certain rank

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