



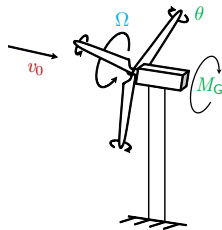
Baseline Generator Torque Controller

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Lecture #2 of Course
"Controller Design for Wind
Turbines and Wind Farms"

Baseline Generator Torque Controller



Motivation

- ▶ Generator torque is one of the two main control inputs.
 - ▶ Generator torque controller is enabled most of the time.
 - ▶ Can be implemented in a quite simple form (baseline).
- Good point to get started with our controller!

Main questions

- ▶ How can we design a baseline generator torque controller?
- ▶ How can we and how should we evaluate a controller?

Schedule

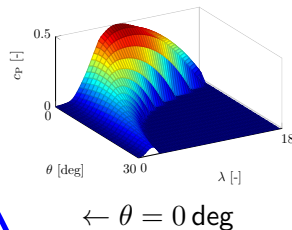
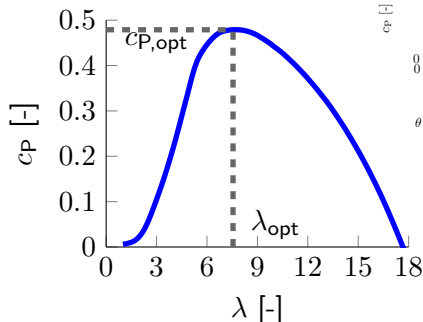
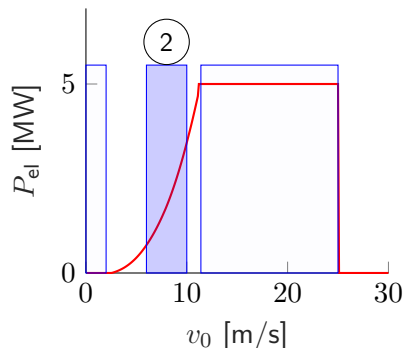
- 19.09. 1 Controller Design Objectives and Modeling
- 26.09. 2 Baseline Generator Torque Controller (online)
- 10.10. 3 Collective Pitch Controller (online)
- 17.10. 4 Filter Design (online)
- 24.10. 5 Tower Damper
- 07.11. 6 Advanced Torque Controller
- 14.11. 7 Wind Field Generation
- 21.11. 8 Steady State Calculations
- 28.11. 10 Lidar-Assisted Control I
- 05.12. 11 Lidar-Assisted Control II
- 12.12. 12 Wind Farm Effects
- 19.12. 13 Wind Farm Control
- 09.01. 14 Floating Wind Control ???

Content

1. Baseline Torque Controller Design
2. Evaluation of a Controller
3. Conclusion and Learning Objectives

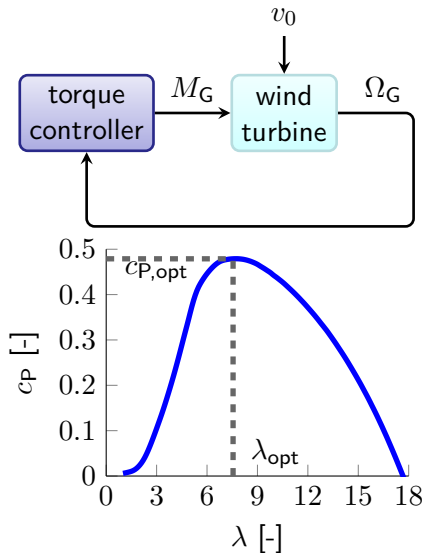


Strategy Baseline Torque Controller



- ▶ Torque controller in region 2 aims to maximize $P_{el} = \frac{1}{2} \rho \pi R^2 c_P(\lambda, \theta) v_0^3 \eta_{el}$.
- ▶ Optimum $c_{P,opt}$ is reached for $\lambda = \lambda_{opt}$ and $\theta = 0$ deg.
- ▶ Rotor-effective wind speed v_0 is not measurable.
- ▶ Indirect Speed Control: Feedback $M_G(\Omega_G)$ can be designed such that $\lambda \rightarrow \lambda_{opt}$.

Optimal Torque Control



Optimal feedback

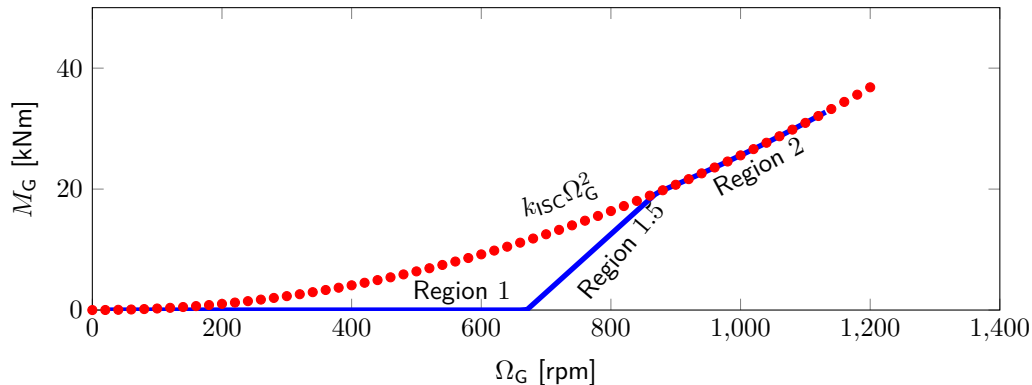
$$J\dot{\Omega} = M_a - \frac{M_G}{i_{GB}} \stackrel{!}{=} 0$$

Diagram illustrating the optimal feedback control law. The equation shows the mechanical torque M_a (represented by $\frac{1}{2}\rho\pi R^2 c_{P,opt} v_0^3$) minus the electromagnetic torque $\frac{M_G}{i_{GB}}$ (represented by $\frac{\Omega R}{\lambda_{opt}}$) set to zero. The optimal tip speed ratio λ_{opt} is shown as a function of the generator speed Ω_G and the gearbox ratio i_{GB} .

$$\rightarrow M_G = \underbrace{\frac{1}{2}\rho\pi R^5 \frac{c_{P,opt}}{\lambda_{opt}^3}}_{k_{ISC}} i_{GB}^3 \Omega_G^2$$

Proof of convergence and stability, see [1].

Nonlinear State Feedback I



Region 1:

$$\Omega_G < \Omega_{1to1.5}$$

$$M_G = 0$$

Region 1.5:

$$\Omega_{1to1.5} \leq \Omega_G < \Omega_{1.5to2}$$

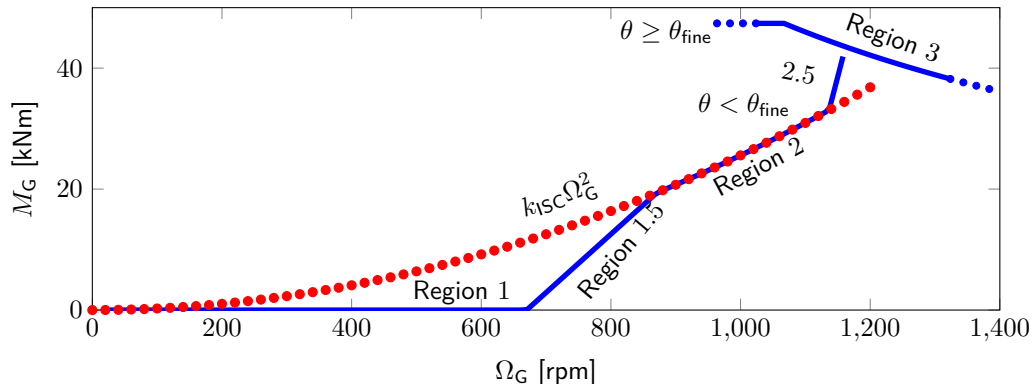
$$M_G = a_{1.5} \Omega_G + b_{1.5}$$

Region 2:

$$\Omega_{1.5to2} \leq \Omega_G < \Omega_{2to2.5}$$

$$M_G = k_{ISC} \Omega_G^2$$

Nonlinear State Feedback II



Region 2.5:

$$\Omega_{2to2.5} \leq \Omega_G \text{ and } \theta < \theta_{fine}$$

$$M_G = a_{2.5} \Omega_G + b_{2.5}$$

Region 3:

$$\theta \geq \theta_{fine}$$

$$M_G = \min(P_{a, rated} / \Omega_G, M_{G, max}) \text{ or } M_G = M_{G, rated}$$

Transition Regions

Region 1.5

Necessary to reach region 2 from region 1. Usually $\Omega_{1\text{to}1.5}$ and $\Omega_{1.5\text{to}2}$ are given:

$$\begin{aligned}0 &= a_{1.5} \Omega_{1\text{to}1.5} + b_{1.5} \\ k_{\text{ISC}} \Omega_{1.5\text{to}2}^2 &= a_{1.5} \Omega_{1.5\text{to}2} + b_{1.5}\end{aligned}$$

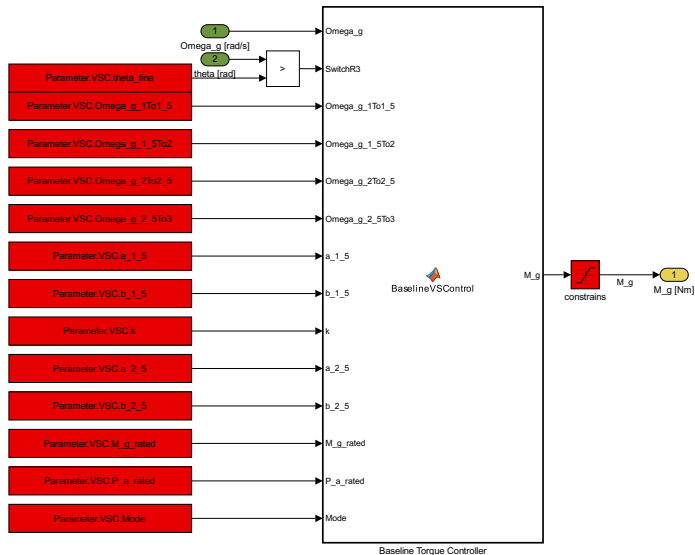
Region 2.5

Necessary to reach region 3, if rated generator speed is reached before rated torque is reached. Otherwise torque is held constant. If $\Omega_{2\text{to}2.5}$ and $\Omega_{G,\text{rated}}$ are given:

$$\begin{aligned}M_{G,\text{rated}} &= a_{2.5} \Omega_{G,\text{rated}} + b_{2.5} \\ k_{\text{ISC}} \Omega_{2\text{to}2.5}^2 &= a_{2.5} \Omega_{2\text{to}2.5} + b_{2.5}\end{aligned}$$

Alternatively, $\Omega_{2.5\text{to}3}$ slightly below $\Omega_{G,\text{rated}}$ and a generator-slip percentage can be given [2].

Implementation in Matlab Simulink I



Implementation in Matlab Simulink II

```
function M_g = BaselineVSControl(Omega_g,SwitchR3,... % signals
    Omega_g_1To1_5,Omega_g_1_5To2,...           % region limits
    Omega_g_2To2_5,Omega_g_2_5To3,...           % Region 1.5 parameters
    a_1_5,b_1_5,...                             % Region 2 parameters
    k,...                                         % Region 2.5 parameters
    a_2_5,b_2_5,...                             % Region 3 parameters
    M_g_rated,P_a_rated,Mode)

if Mode == 1                                     % Power constant
    M_g_3 = P_a_rated/Omega_g;
else                                             % Torque constant
    M_g_3 = M_g_rated;
end

if      Omega_g_2_5To3 < Omega_g                % Region 3
    M_g      = M_g_3;

elseif Omega_g_2To2_5 < Omega_g                % Region 2.5
    M_g_2_5 = a_2_5 * Omega_g + b_2_5;
    M_g      = SwitchR3*M_g_3+(1-SwitchR3)*M_g_2_5;

elseif Omega_g_1_5To2 < Omega_g                % Region 2
    M_g_2    = k * Omega_g^2;
    M_g      = SwitchR3*M_g_3+(1-SwitchR3)*M_g_2;

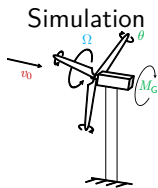
elseif Omega_g_1To1_5 < Omega_g                % Region 1.5
    M_g = a_1_5 * Omega_g + b_1_5;

else                                           % Region 1
    M_g = 0;
end
```

Evaluation Steps and Purpose

1. Step response of controller design model (e.g. Simulink):
Does the controller exactly what it should or is there an implementation error?
 2. Step response of aero-elastic tool (DLL: Dynamic Link Library):
How does model uncertainties impact the behavior?
 3. Load Simulations following standards (DLL):
How does the controller perform over a wide range of conditions?
 4. Software/Hardware-in-the-loop tests (PLC: Programmable Logic Controller):
How does the controller perform under realistic conditions?
 5. Field Testing (PLC):
How does the controller perform in reality?
-
- ▶ Controller should be initialized with current states to avoid long settling times (simulations) and bump-free switching (reality).
 - ▶ If possible use one source code for all steps, e.g. libraries to avoid translation errors.

Why do we need load simulations?



Reality



Load simulations help during all design phases:

- ▶ Which tower height should be used? → Conceptual design
- ▶ Which loads affect the components? → Design
- ▶ Are the loads within design limits? → Certification
- ▶ How can the system be optimized? → Optimization

Benefits over real tests

- ▶ simulations have low costs
- ▶ simulations are often fast
- ▶ simulations allow us to study easily many configuration
- ▶ simulations are reproducible
- ▶ however: models and tools need to be validated

Simulation Process in General

Pre-Processing

- ▶ specification of all model, controller and simulation parameters
- ▶ specification of all external input signals (wind and sea states)
- ▶ performing calculations, which are needed as input for simulations



Processing

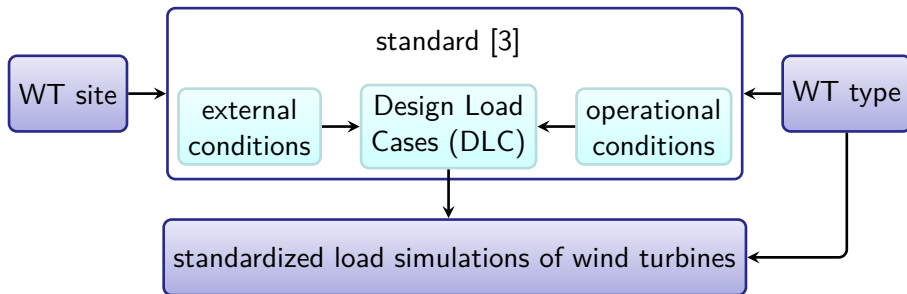
performing simulations using the defined algorithms, parameters and input signals



Post-Processing

- ▶ calculation of statistics and fatigue loads
- ▶ calculation of spectra
- ▶ plotting results etc.

Standardized load simulations for wind turbines



External conditions

- ▶ wind conditions
- ▶ other environmental inputs
- ▶ electrical power network

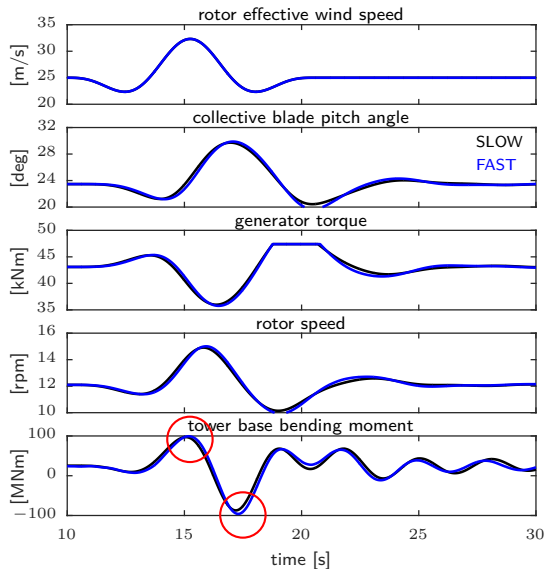
Operational conditions

- ▶ power production (+ fault)
- ▶ start up, shut down
- ▶ parked, transport

Type of analysis

- ▶ fatigue load analysis
- ▶ ultimate load analysis

Example: ultimate load simulation



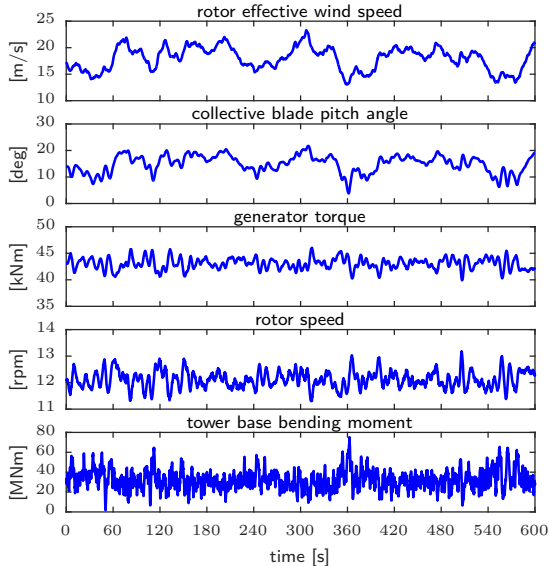
Extreme Operating Gust (EOG)

- ▶ coherent at cut-out wind speed
- ▶ often used for controller testing
- ▶ here NREL 5 MW reference wind turbine
- ▶ over speed and tower motion
- ▶ absolute values count

SLOW to FAST comparison

- ▶ main dynamic well captured
- ▶ differences mainly due to unmodeled blades and dynamic inflow / stall

Example: fatigue load simulation 1/2



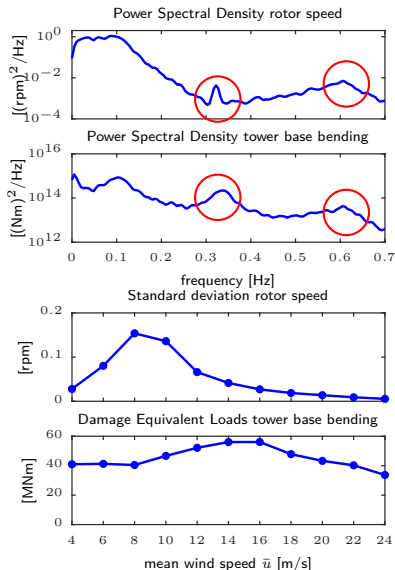
Normal Operation DLC 1.2

- ▶ 6×10 min per mean wind speed
- ▶ from cut-in to cut-out, 2 m/s steps
- ▶ turbulence according to turbine class

Damage Equivalent Loads (DEL)

DEL is an equivalent periodic load with constant amplitude and a predefined number of cycles which causes the same damage as stochastic loads.

Example: fatigue load simulation 2/2



Power Spectral Density (PSD)

- ▶ distribution of energy content over frequency
- ▶ tower resonance and 3P excitation visible
- ▶ useful to identify vibration issues at certain frequencies

Lifetime weighted load distribution

- ▶ weighting of values from each wind bin according to their relative frequency of occurrence over the lifetime
- ▶ useful to identify issues at certain operation points

Conclusion

Main questions

- ▶ How can we design a baseline generator torque controller?
- ▶ How can we and how should we evaluate a controller?

Nonlinear-state feedback controller $M_G = f(\Omega_G, \theta)$

- ▶ Region 2: $k_{ISC} \Omega_G^2$ brings turbine to chosen power coefficient
 - ▶ Region 1.5 and 2.5: linear functions for transition
 - ▶ Region 3: onshore usually constant power
 - ▶ pitch limit θ_{fine} to determine, whether Region 3 or 2.5 should be applied
-
- ▶ Various steps to focus on specific questions starting with nominal model.
 - ▶ Use same source code and initialize controller.

Quick check on learning objectives

After this lectures you should be able to...

- ▶ to describe the main tasks of a generator torque controller
- ▶ to design a baseline torque controller for all control regions
- ▶ to describe how the interaction with the pitch controller works
- ▶ to name useful steps to evaluate a controller
- ▶ to name conditions and type of analyses for standardized load simulations

References

- [1] K. Johnson, L. Pao, M. Balas, and L. Fingersh. “Control of Variable-Speed Wind Turbines”. In: IEEE Control Systems Magazine 06 (2006), pp. 70–81.
- [2] J. Jonkman, S. Butterfield, W. Musial, and G. Scott. Definition of a 5-MW Reference Wind Turbine for Offshore System Development. Tech. rep. TP-500-38060. NREL, 2009. DOI: 10.2172/947422.
- [3] IEC 61400-1. Wind turbines - Part 1: Design requirements. International Electrotechnical Commission, 2005.

Please let me know if you have further questions!

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