



#### Second Seminar - PPG-EM UERJ / 2016

## Validation of Experimental Rig with Emphasis on Dynamic Phenomenon of Rotors

Paulo Roberto Farias Junior

Advisors: Francisco Soeiro

Renato Rocha

## **Experimental Rig**



Figure 1 – Experimental Rig

#### **Motivations**

- ✓ Work Experimental
- ✓ The First Work About Fenomenon in This Experimental Rig
- ✓ Interface Instrumentation/Hardware/Software
- ✓ Signal Processing
- ✓ Finite Element Analyses

## **Objective**

- ✓ Build computer models using analysis software for finite element.
- ✓ Identify the main characteristics presented by the rotor when subjected to different pre-set settings to rehearse it.
- ✓ Compare the simulations results with results of experimental tests.
- ✓ Present Results.

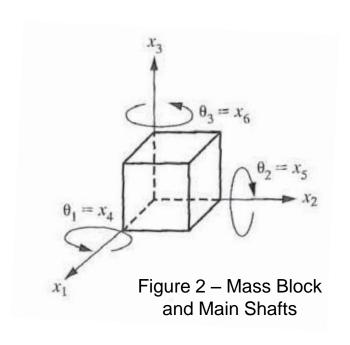
## Methodology

- ✓ Simulations (Solidworks e Rotmef)
- ✓ Experimental Tests
- ✓ Signal Processing

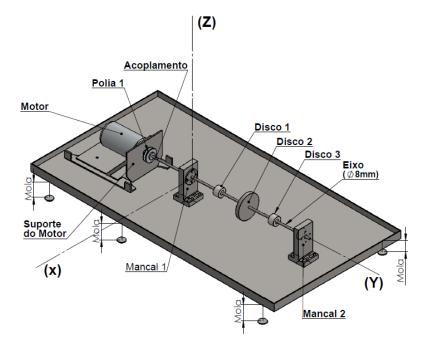
# Evaluation of the Dynamic Behavior of the Experimental Rig (Rigid Body Analysis)

- √ Frequency Sweep Test
- ✓ SolidWorks Simulation
- ✓ Experimental Instrumented Tests

## Rigid Body Analysis - Fundamentals



- ✓ Single mass element illustrating all the possible degrees of freedom.
- ✓ The six degrees of freedom of the rigid body consist of three rotational and three translational motions.



## Rotor Behavior and Phenomena Analysis

✓ Definition of rotor

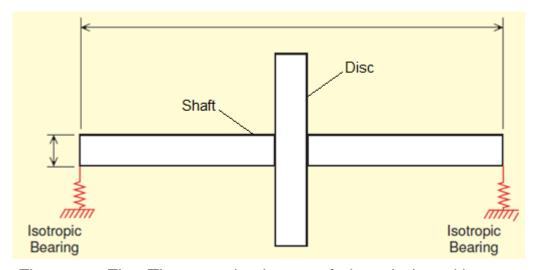


Figure 5 – First Three mode shapes of pinned-pinned beam

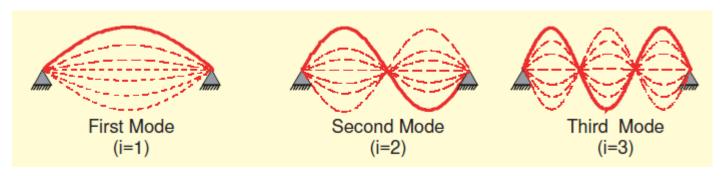
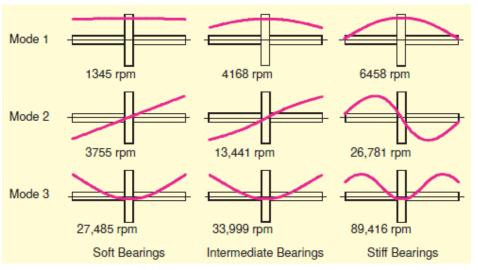


Figure 6 – Basic machine model cross section

## Rotor Behavior and Phenomena Analysis

✓ Gyroscopic effect



✓ Reverse modes

Figure 7 - Mode shapes versus bearing stiffness, shaft not rotating.

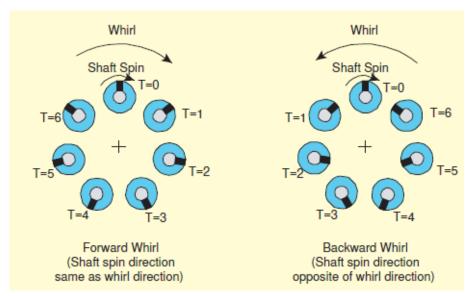


Figure 8 - Whirl sense.

## Rotor Behavior and Phenomena Analysis

#### ✓ Campbell Diagram

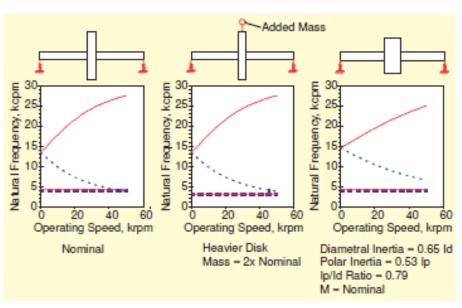


Figure 13. Comparison of different disk properties, center disk configuration.

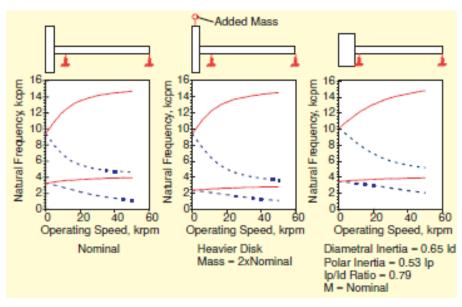


Figure 14. Comparison of different disk properties, overhung configuration

## Rigid Body Analysis Frequency Sweep Test

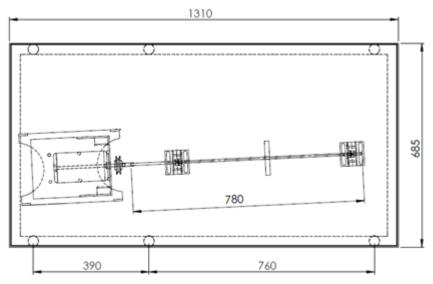
\*Unbalanced disc 2
Unbalanced mass = 60g

Table 1 – Frequency and Experimental Modes – Frequency Sweep

Frequency (rpm)	Modes
310,20	Translation em X
387,00	Translation em Z
452,00	Rotation em Z
505,00	Rotation em X
678,00	Rotation em Y

Tabela 2 - Materiais

Components	Material	Elastic Module (N/m²)	Yield strength (N/m²)	Poisson's Coefficient	Specific Mass (kg/m³)
Coupling	Borracha	6,10E+06	9,24E+11	0,49	1000
Pulley 1 e 2	Alumínium	6,90E+10	2,76E+12	0,33	2700
Gears	Nylon	1,0E+09	6,00E+07	0,30	1150
Other Components	Aço 1020	2,00E+11	3,52E+13	0,29	7900



Total Mass of Model: 105,213 Kg

Axial and Radial Stiffness of Bearings: 10<sup>7</sup> N/m

Dimensions in mm

Figure 12 – Main dimensions of Experimental Rig

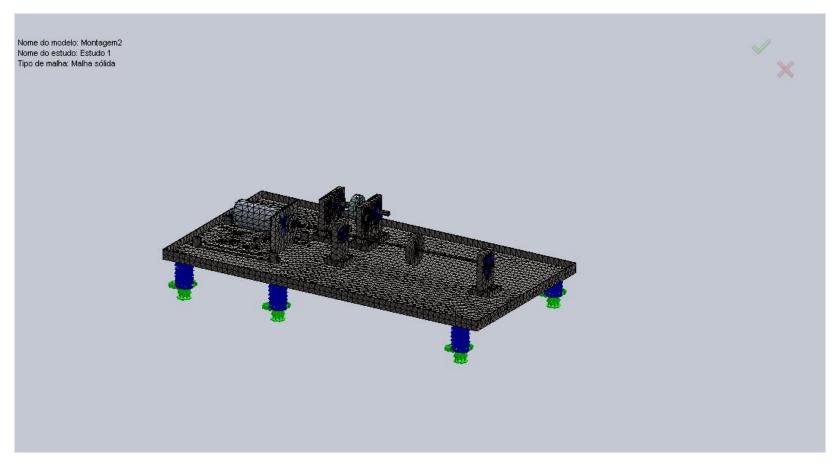


Figure 13 - Mesh

Table 4 – Mesh Configurations

Type of Mesh	Solid Mesh
Generator of Mesh:	Mesh based on curvature
Points Jacobianos	4 Points
Total of nodes	80359
Element Types	Tetraédrico 3D
Total of elements	42969
Maximum Proportion	62.584
% of elements with proportion < 3	61.8
% of elements with proportion < 10	4.16
Time for run the mesh (hh;mm;ss):	00:01:24

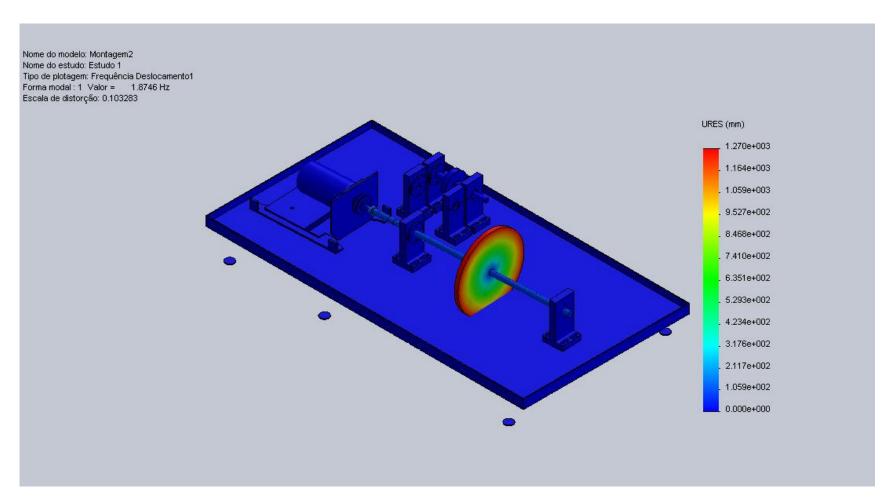


Figure 14 – Mode 1

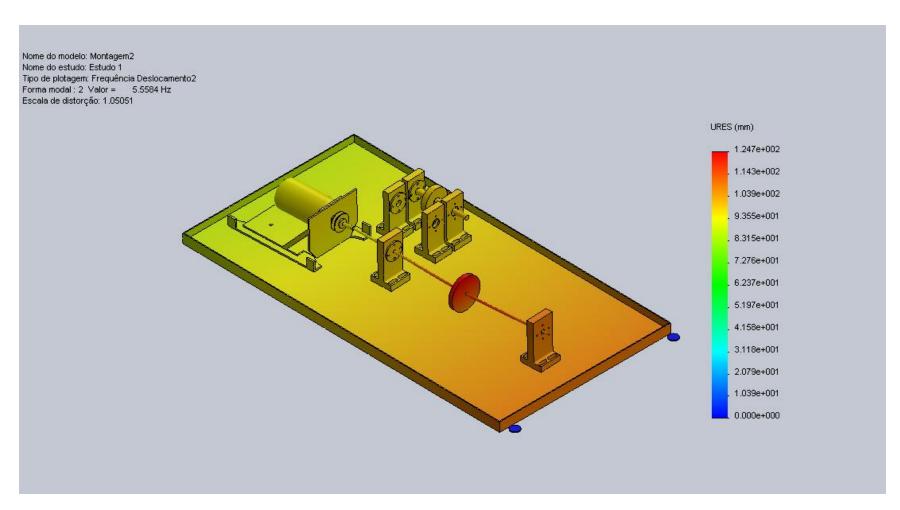


Figure 9 - Mode 2

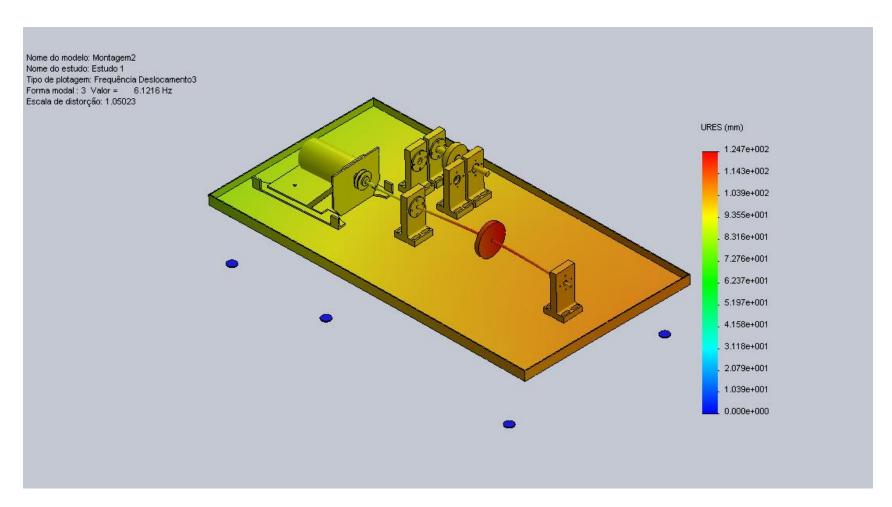


Figure 10 - Mode 3

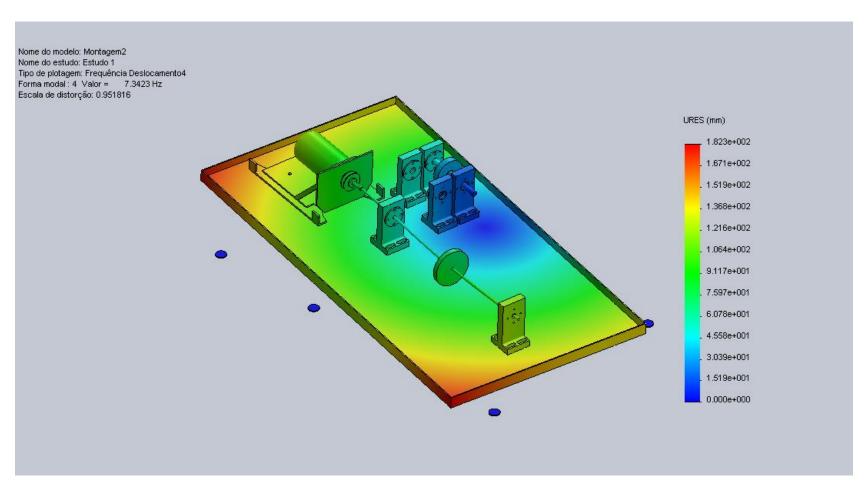


Figure 11 - Mode 4

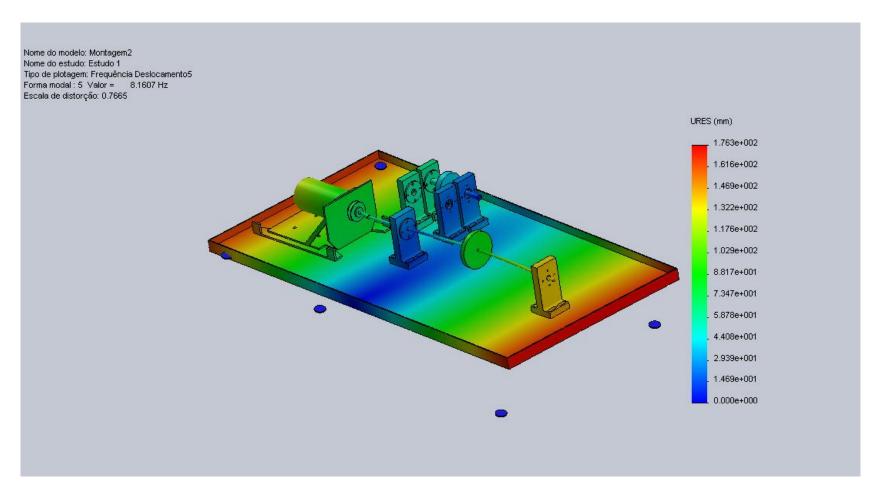


Figure 12 - Mode 5

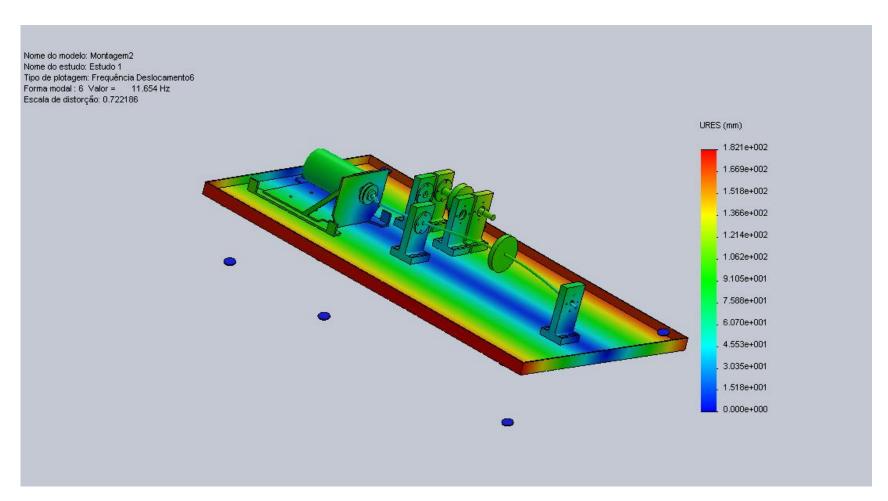


Figure 13 - Mode 6

Table 5 – Frequency and Modes (Model and Experimental)

Frequencies	Model (rpm)	Experimental-Sweep (Rpm)	Modes	Difference (%)
1	112,6	Not identified	Translation Y	
2	333,50	310,20	Translation X	6,99
3	367,30	387,00	Translation Z	-5,36
4	440,54	452,00	Rotation Z	-2,60
5	489,64	505,00	Rotation X	-3,14
6	699,24	678,00	Rotation Y	3,04

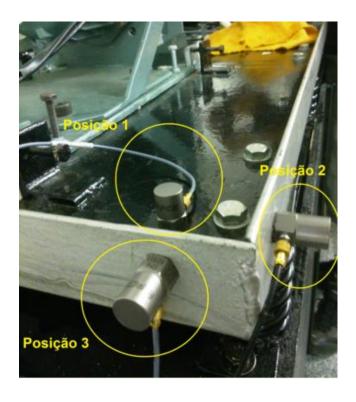


Figure 14 – Position of accelerometer

#### **Translation Y**

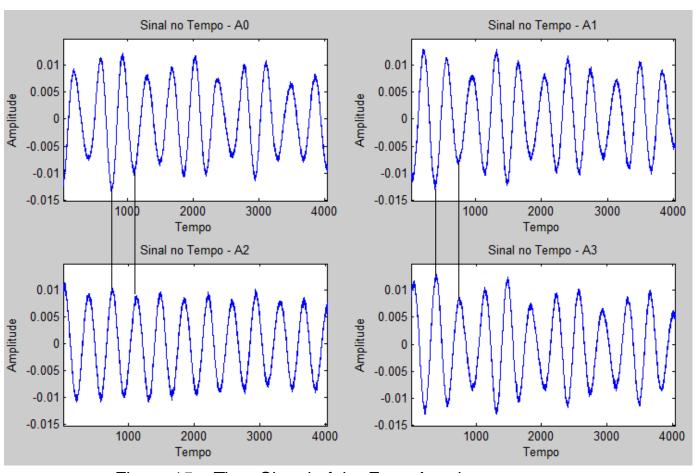


Figura 15 – Time Signal of the Four Accelerometer

#### **Translation Y**

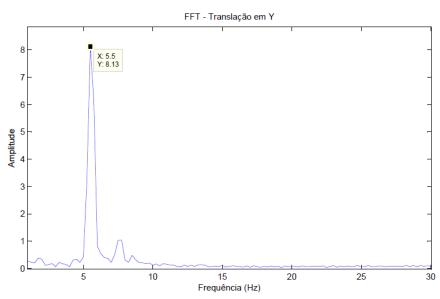


Figure 16 - FFT of Time Signal A0

Table 6 – Phases for Translation Y

Phase Analysis	A0	A1	A2	А3
A0	<b>0</b> ō	0∘	180⁰	180º
A1	0ō	0ō	180⁰	180º
A2	180⁰	180º	0₀	Oō
А3	180⁰	180⁰	0₀	Oō

#### **Translation X**

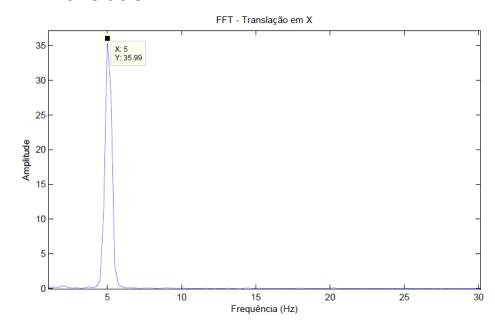


Figure 17 – FFT of Time Signal A0

Table 7 - Phases Translation X

Phase Analysis	A0	<b>A1</b>	A2	А3
A0	<b>0</b> ō	180⁰	0ō	180º
A1	180⁰	0₀	180⁰	Oō
A2	<b>0</b> ō	180⁰	0∘	180⁰
A3	180⁰	0ō	180⁰	Оō

#### Translação em Z

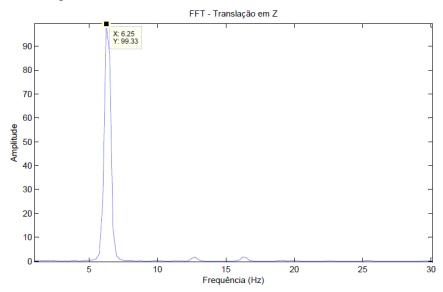


Figura 18 - FFT do sinal no tempo de A0

Tabela 8 – Fases para Translação em Z

Análise de fase	A0	A1	A2	А3
A0	Oō	<b>0</b> ō	0ō	0ō
A1	Oō	Oō	Oō	Oō
A2	Oō	05	05	0 <sub>ō</sub>
А3	Oō	05	05	Оō

#### **Rotation X**

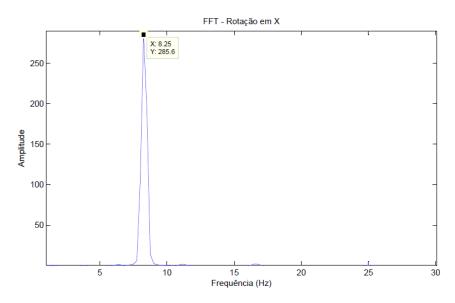


Figura 19 - FFT of Time Signal A0

Table 9 – Phases for Rotation X

Phase Analysis	Α0	A1	A2	А3
A0	0∘	05	180º	180º
A1	Oā	<b>0</b> ō	180º	180º
A2	180º	180º	<b>0</b> ō	0₀
А3	180⁰	180º	0₀	05

#### **Rotation Y**

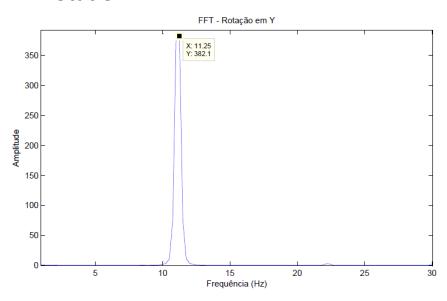


Figure 20 - FFT Time Signal A0

Table 10 – Phases for Rotation Y

Phase Analysis	Α0	A1	A2	А3
A0	<b>0</b> ō	180º	0₀	180º
A1	180º	0₀	180º	Оō
A2	O <u>a</u>	180º	0ō	180º
А3	180º	O <u>a</u>	180º	Oō

## Verificação Experimental (Acelerômetros)

#### Rotação em Z

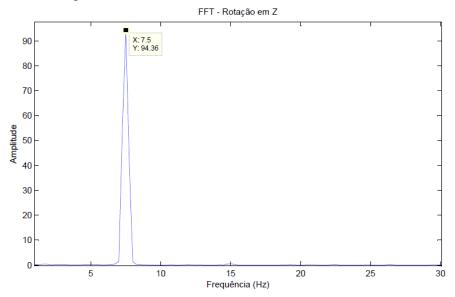


Figura 21 - FFT do sinal no tempo de A0

Tabela 11 – Fases para Rotação em Z

Phase Analysis	A0	A1	A2	А3
A0	<b>0</b> ∘	180º	<b>0</b> ∘	180º
<b>A1</b>	180º	Оō	180º	0ō
A2	О <sub>Б</sub>	180º	Оō	180º
А3	180⁰	Oā	180⁰	<b>0</b> ō

Table 12 – Comparison of Results

Frequecies	Model (rpm)	Sweep (rpm)	Experimental Test(rpm)	Modes
1	112,6	Not identified	330	Translation Y
2	333,5	310,2	300	Translation X
3	367,3	387	375	Translation Z
4	440,54	452	450	Rotation Z
5	489,64	505	495	Rotation X
6	699,24	678	675	Rotation Y

## Rotor Analysis Rotmef Simulations

Table 14 – Tests configuration

Configuration	Disc position	Distance between bearings	Critical speed(rpm)
1	Centered between bearings		950
2	136mm of bearing 1	590 mm	1180
3	190mm of bearing 1		1025
Cantilever - 130mm of bearing 2		380 mm	832
5	Cantilever - 190mm of bearing 2	360 11111	600

Tabela 15 – Speed test

Speeds (rpm)
550
1300
2000

#### Configuration 1

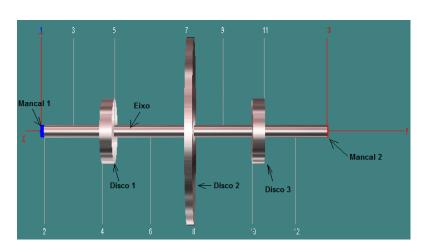


Figure 21 - Configuration 1 - Rotmef

Table 15 – Frequency of each mode acording to rotation of first configuration.

Rotation	Frequency of modes (Hz)								
(rpm)	1	2	3	4	5	6			
550	15,4	15,41	84,78	88,72	161,77	162,1			
1300	15,39	15,41	82,03	91,34	161,55	162,31			
2000	15,38	15,42	79,45	93,7	161,33	162,5			

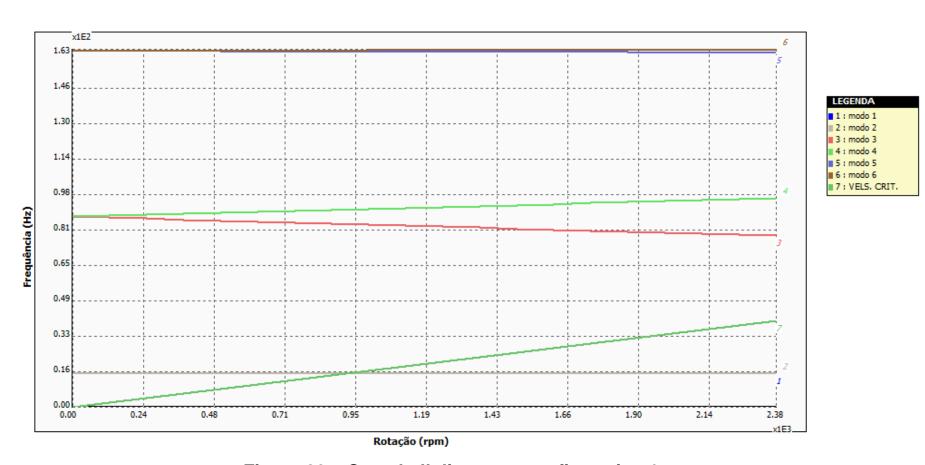
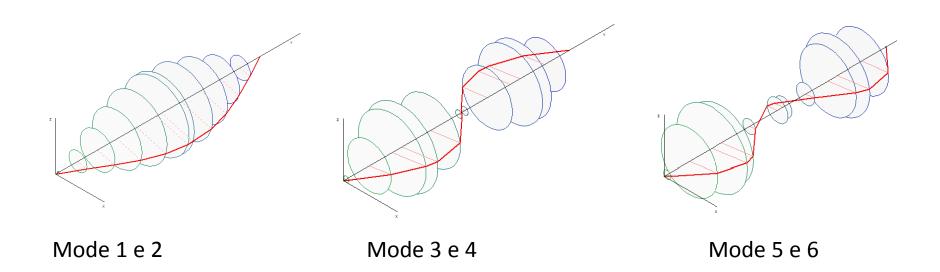


Figure 22 - Campbell diagram - configuration 1

**Critical speed = 950rpm** 



#### Configuration 2

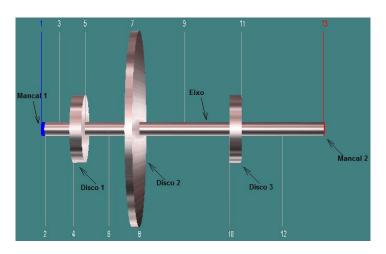
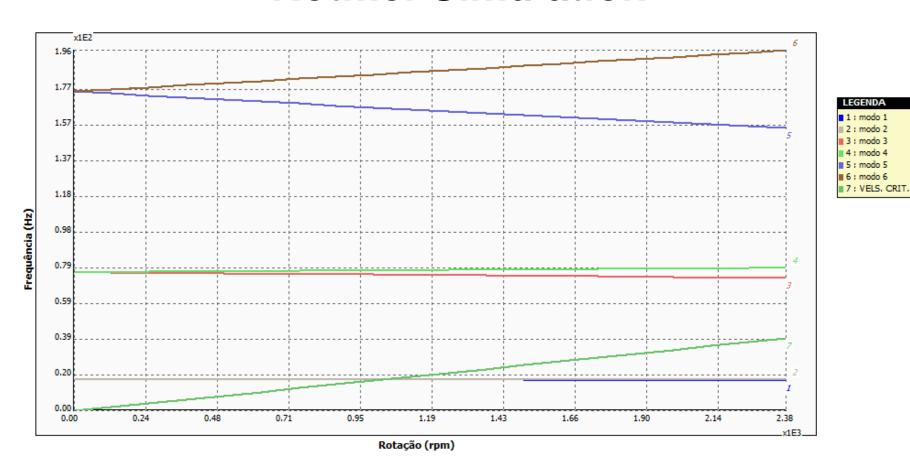


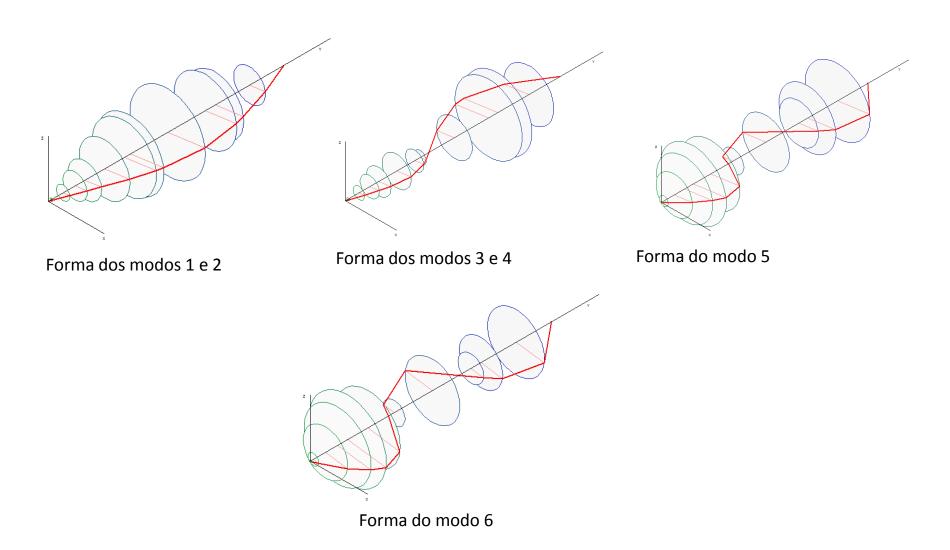
Figure 23 - Modelo da configuração 2 no Rotmef

Tabela 16 – Frequência de cada Modo de acordo com as rotações para a configuração 2

Rotações (rpm)	Frequência (Hz) dos Modos							
Kotações (Epili)	1	2	3	4	5	6		
550	19,79	20,14	72,66	72,85	196,07	208,97		
1300	19,55	20,37	72,51	72,98	187,76	218,22		
2000	19,33	20,59	72,37	73,09	180,37	227,15		



**Velocidade Crítica = 1180rpm** 



#### Configuration 3

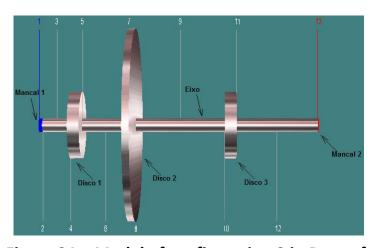
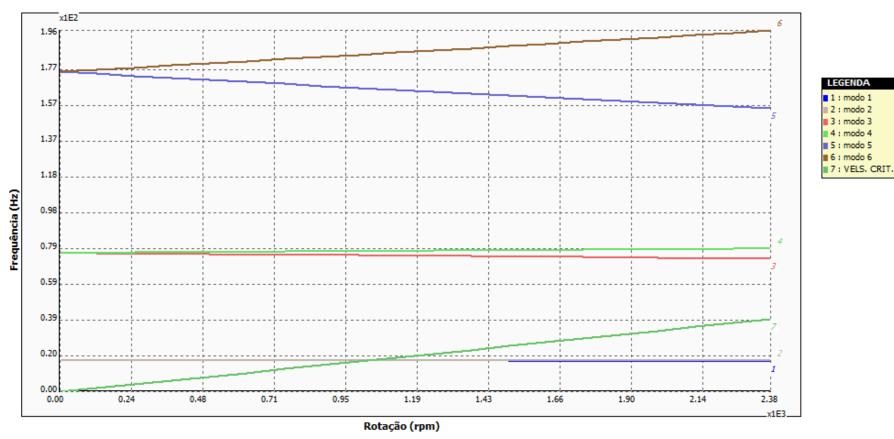


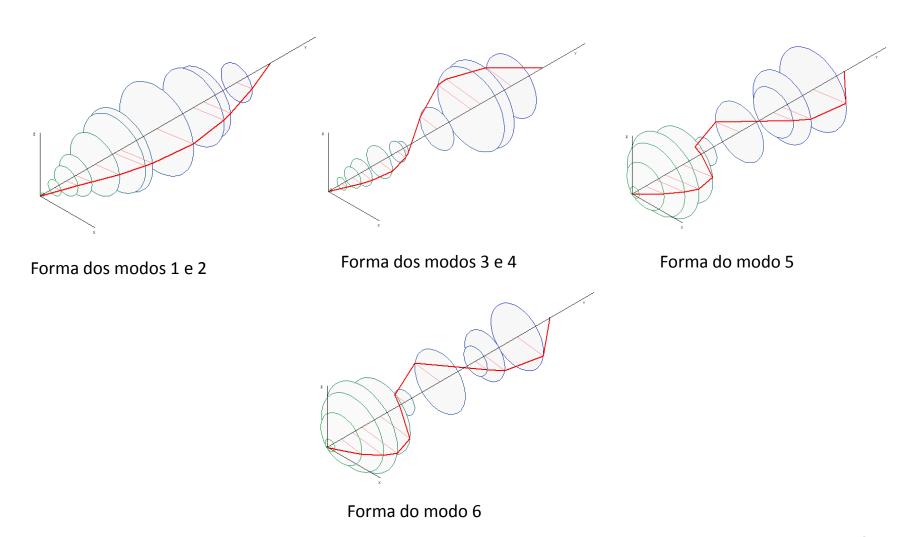
Figura 24 – Model of configuration 3 in Rotmef

Tabela 17 – Frequência de cada Modo de acordo com as rotações para a configuração 3

Botasãos (rom)	Frequência(Hz) dos Modos						
Rotações (rpm)	1	2	3	4	5	6	
550	17,07	17,2	74,81	76,08	169,22	179,1	
1300	16,98	17,28	73,87	76,86	162,85	186,1	
2000	16,9	17,36	72,92	77,55	157,21	192,75	



**Velocidade Crítica = 1025rpm** 



#### Configuração 4

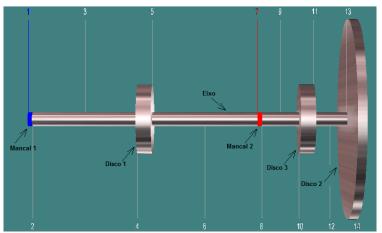
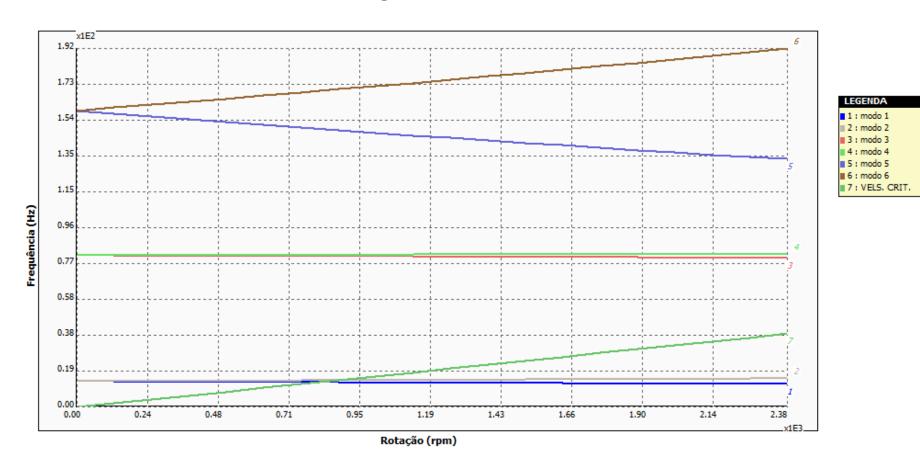


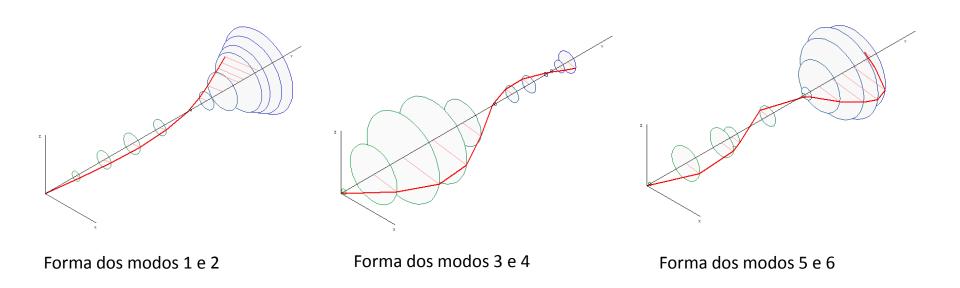
Figura 25 - Modelo da configuração 4 no Rotmef

Tabela 18 – Frequência de cada Modo de acordo com as rotações para a configuração 4

Potaçãos (rnm)	Frequência(Hz) dos Modos						
Rotações (rpm)	1	2	3	4	5	6	
550	13,56	14,26	81,24	81,73	152,65	166,41	
1300	13,1	14,73	80,85	82,01	144,17	176,66	
2000	12,67	15,18	80,43	82,24	136,93	186,82	



**Velocidade Crítica = 832rpm** 



#### Configuração 5

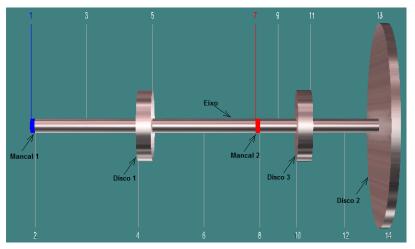
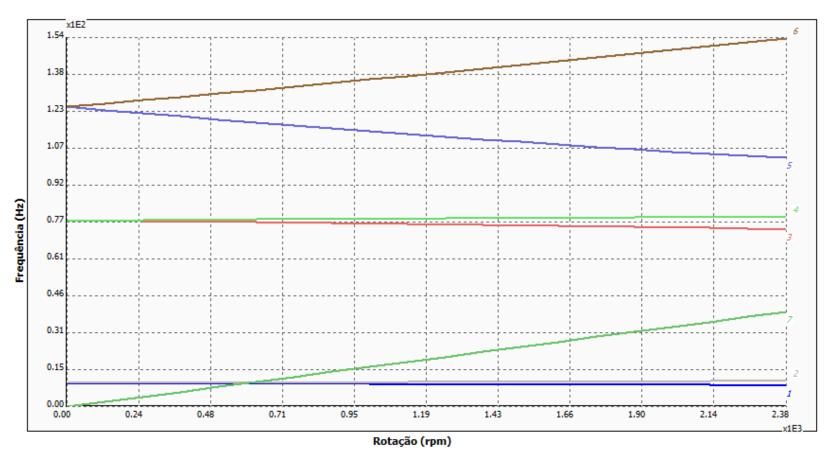


Figura 26 - Modelo da configuração 5 no Rotmef

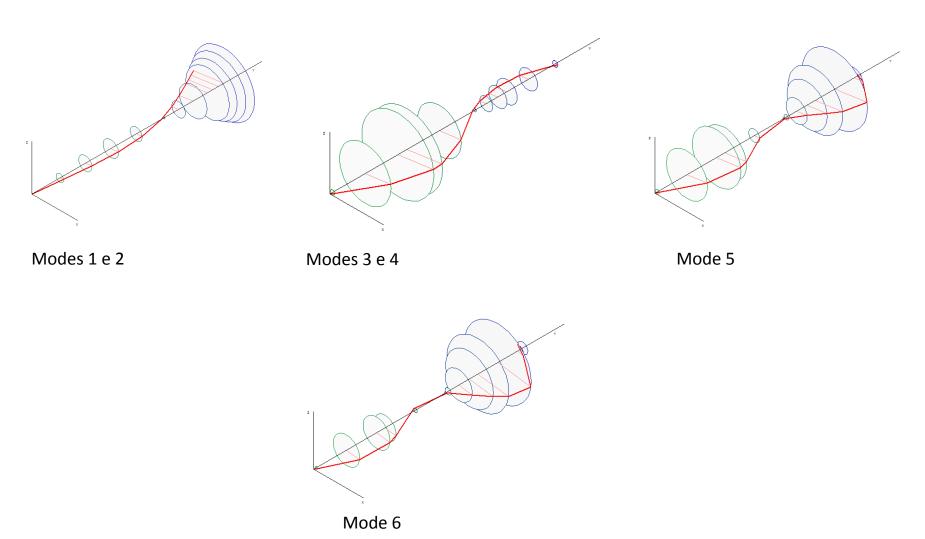
Tabela 19 – Frequência de cada Modo de acordo com as rotações para a configuração 4

Rotações	Frequência(Hz) dos Modos					
(rpm)	1	2	3	4	5	6
550	9,64	10,05	76,84	77,89	119,06	130,92
1300	9,36	10,33	75,86	78,42	112	139,85
2000	9,1	10,6	74,7	78,83	106,25	148,65

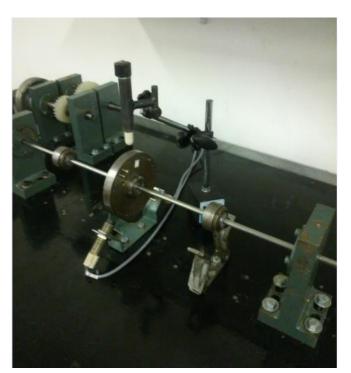


1: modo 1
2: modo 2
3: modo 3
4: modo 4
5: modo 5
6: modo 6
7: VELS. CRIT.

**Velocidade Crítica = 600rpm** 



#### **Rotor Tests**



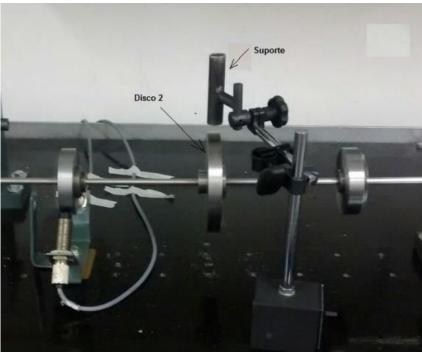


Figure 27 – Magnetic Base

#### **Rotor Tests**

#### Matrix of tests

Configurações	5
Rotações	3
Input	3
Output	3
Total de Ensaios	135

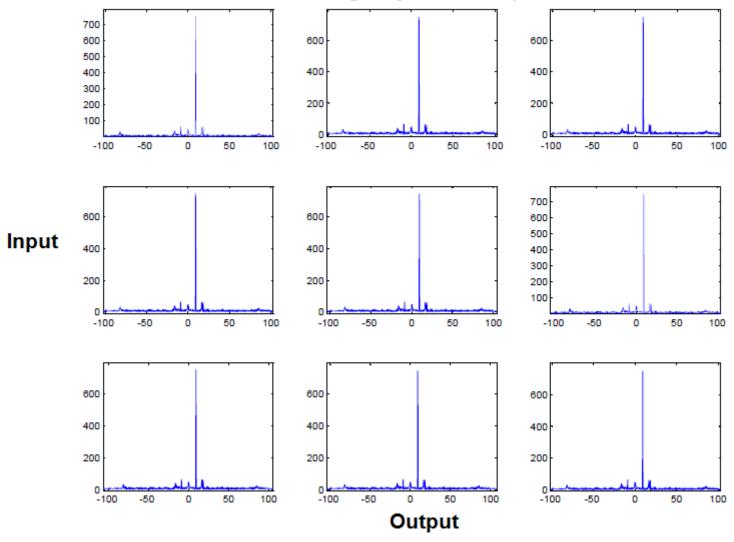
Input: Impacto com pino

Output: Leitura do sinal (Instrumentos)

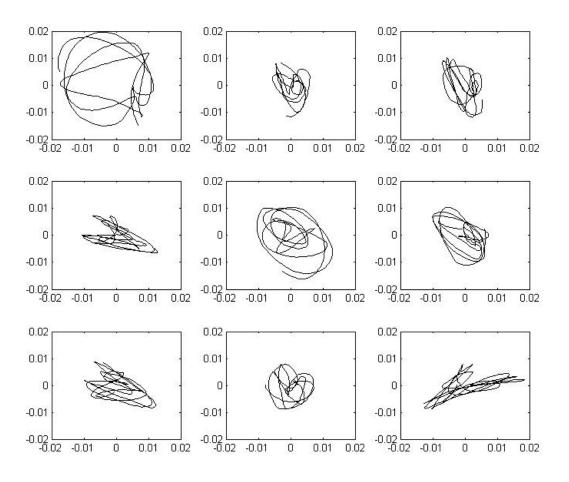
## **Rotor Tests**

#### (Matrix of Spectros)

Configuração 1 - 550rpm



# Rotor Tests (Órbitas)



Orbits – Configuration 1 – 550 Rpm

#### Conclusions

- ✓ In the evaluation of non-rotating parts of the critical frequencies were identified as was also possible to correspondingly validating the first six vibration modes.
- ✓ With respect to the horizontal rotor and the phenomena that the same features can be seen that the values are quite close to the values generated by Rotmef, then it can be concluded that the tests have expected results and about frequency analysis the gyroscopic effect can be observed and the presence of reverse modes by analyzing the full spectra.
- ✓ It is considered that the horizontal rotor of the experimental rig has similar characteristcs of a real rotating machine and can be used for research and development in this field of mechanical engineering.

#### References

- [1] Farias, M.R., "Construção de Bancada Experimental para Análise Dinâmica de Máquinas Rotativas Horizontais", Trabalho Final de Graduação, Departamento de Engenharia Mecânica, UERJ, 2014
- [2] Rocha, R.O., "Modelagem e Simulação Computacional em Dinâmica de Rotores Via Método dos Elementos Finitos", Dissertação de Mestrado, COPPE, 1992
- [3] Murta, M.S., "Projeto, Construção e Avaliação Dinâmica de um Rotor Vertical Suportado em Mancais Hidrodinâmicos, UFRJ, 2000
- [4] Muszynka, A. (2005). Rotordynamics. Nova Iorque: Taylor & Francis Group.
- [5] Eisenmann, R. C., & Eisenmann Jr., R. (2005). *Machinery Malfunction Diagnosis and Correction*. Texas: Pearson Education, Inc.
- [6] Inman, Daniel J., Engineering Vibration, Prentice-Hall Inc., New Jersey, US, 1996
- [7] Souza, Sanderson Pereira Simões de, "Estudo de Técnicas não Convencionais de Processamento de Sinais para Diagnóstico de máquinas Rotativas", Dissertação de Mestrado, COPPE, 2000