

KEY IDEA-1

Generalized concept
of stress:

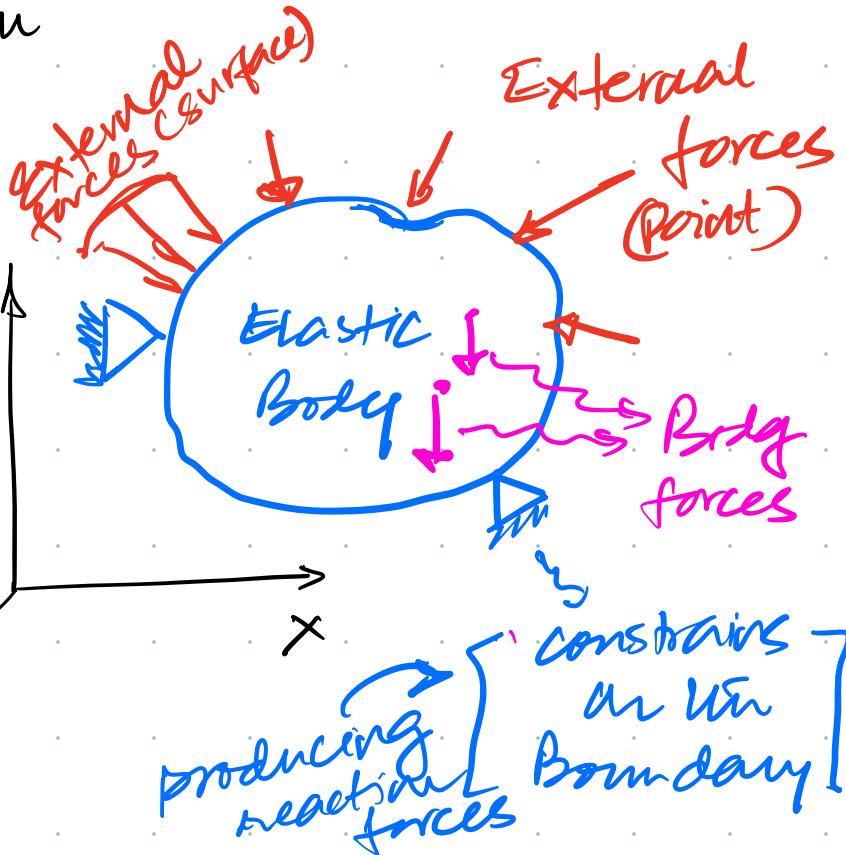
Consider a
Reference axis:
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Cartesian  
coordinate  
system  
( $x, y, z$ )

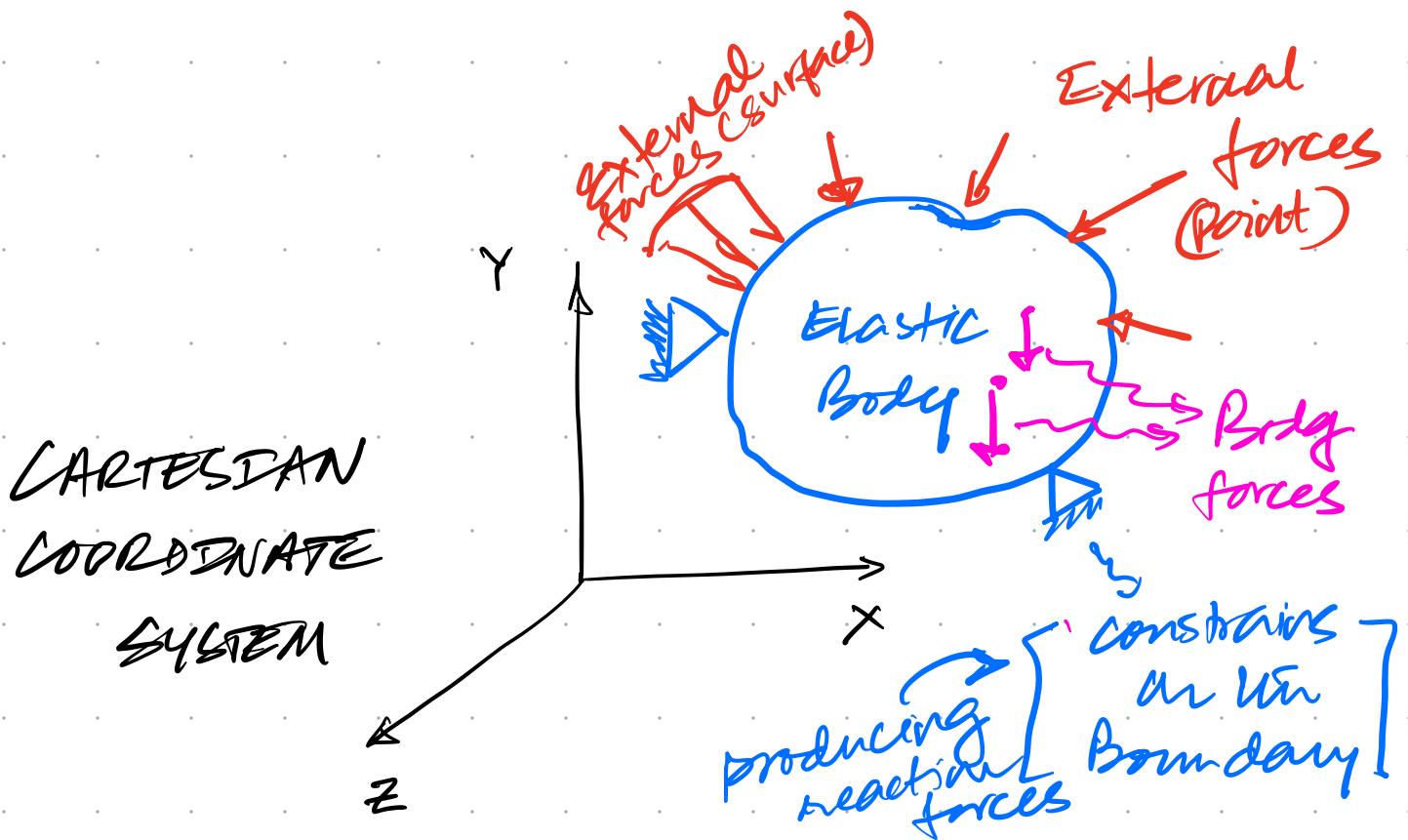
Cylindrical  
coordinate system  
( $r, \theta, z$ )

Spherical  
coordinate system  
( $r, \theta, \phi$ )

CARTESIAN  
COORDINATE  
SYSTEM



(2)



→ Stress is generated inside the body

→ Strain is caused due to the stress

All this leads to deformation.

③

STRESS

It is a mathematical construct

Force acting normal to an area divided by the value of the area ( $F/A$ )

Euler (1757)

Force acting over an oriented area at any point in a body.

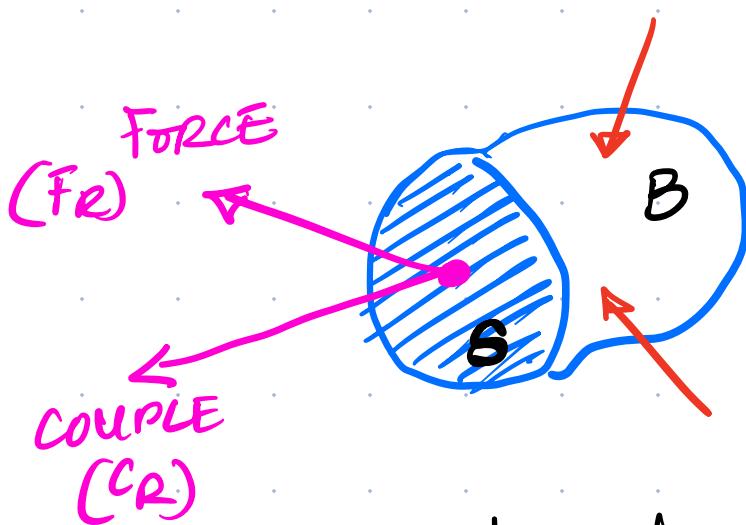
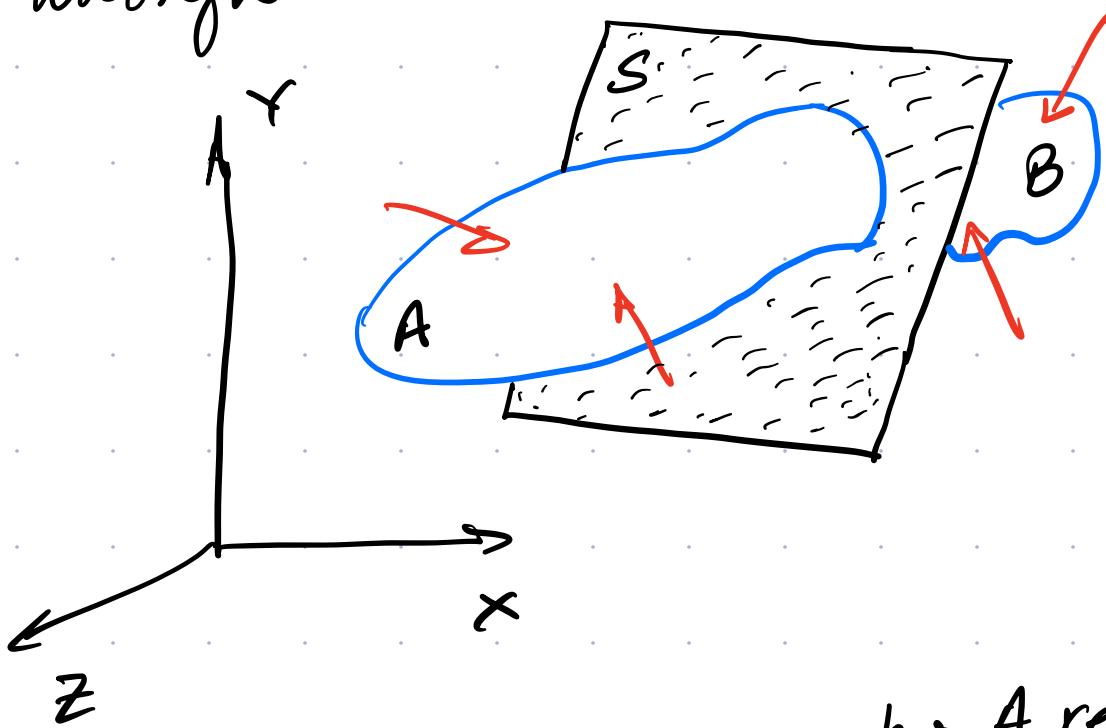
Cauchy  
(1823-1827)

This depends on the choice of the coordinate system.

There can be many forms of stresses acting at a point

(4)

↳ Consider a body in equilibrium with some hypothetical plane  $S$  passing through



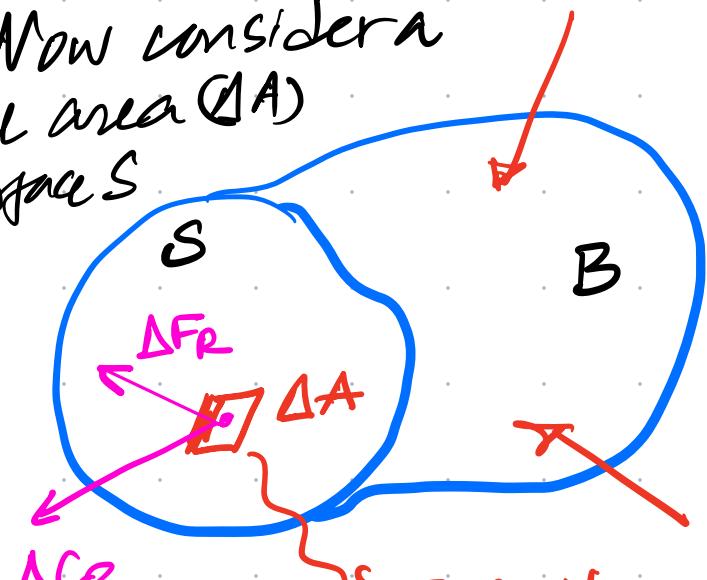
↳ A resultant force ( $F_r$ ) and moment ( $C_r$ ) are transmitted across the surface ( $S$ )

↳ We know this from rigid body mechanics of force and moment balance.

↳ Force ( $F_r$ ) is distributed across the surface ( $S$ )

(5)

Now consider a small area ( $\Delta A$ ) on surface S



↳  $(\Delta F_R)$  is the Resultant force and  
 $(\Delta C_R)$  is the Resultant moment

transmitted across  $(\Delta A)$ .

↳ Using concept of continuum:

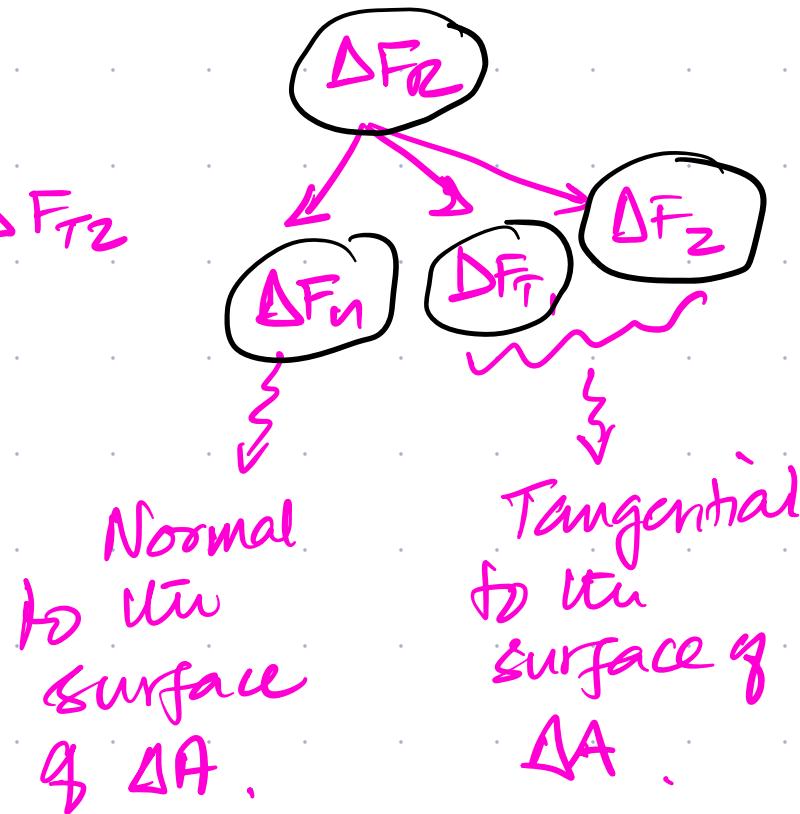
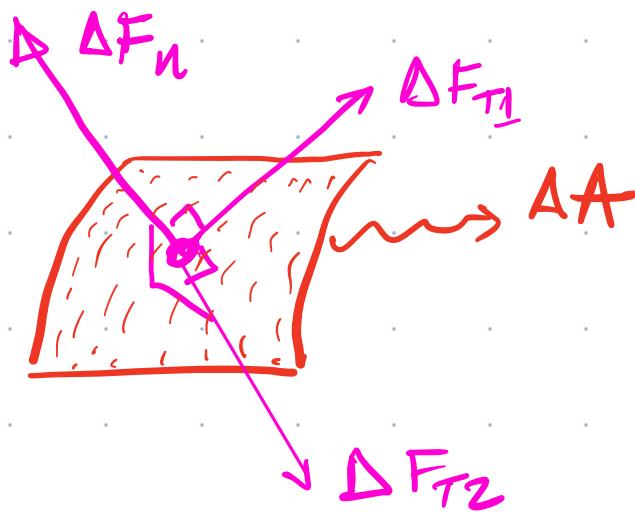
↳ We will let  $(\Delta A) \rightarrow 0$  and

↳ We will decompose  $\Delta F_R$  into three orthogonal components

↳ To generate Force intensity

↳ On a point over surface S.

(6)



We can now formally define stress:

$$\tau_n = \lim_{\Delta A \rightarrow 0} \frac{\Delta F_n}{\Delta A} = \frac{dF_n}{dA}$$

Normal Stress

$$\tau_{T1} = \lim_{\Delta A \rightarrow 0} \frac{\Delta F_{T1}}{\Delta A} = \frac{dF_{T1}}{dA}$$

Shear Stress

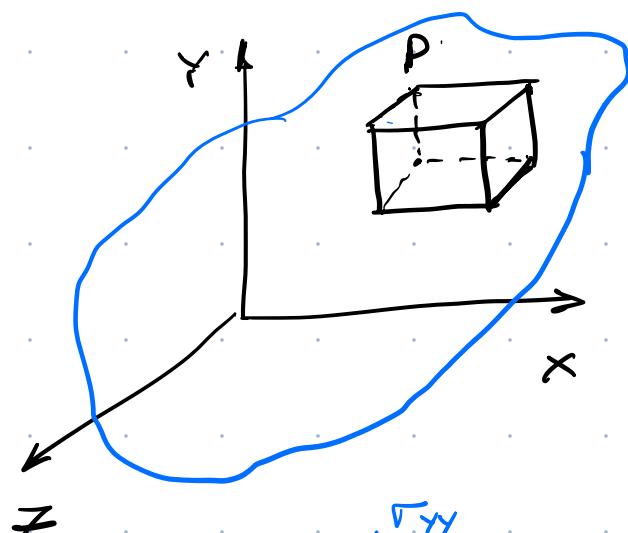
$$\tau_{T2} = \lim_{\Delta A \rightarrow 0} \frac{\Delta F_{T2}}{\Delta A} = \frac{dF_{T2}}{dA}$$

Shear Stress

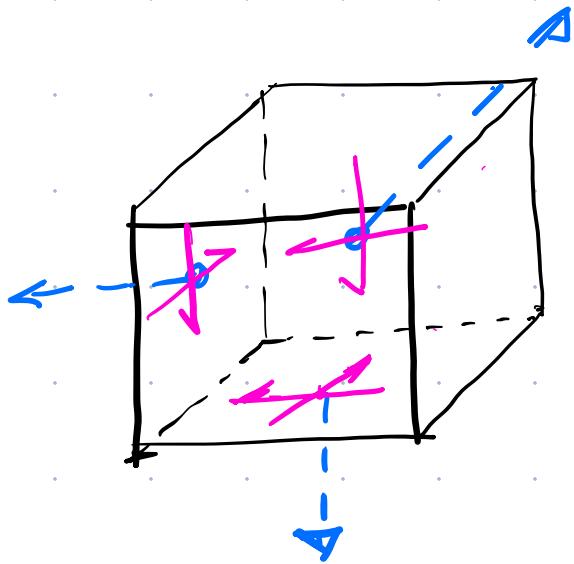
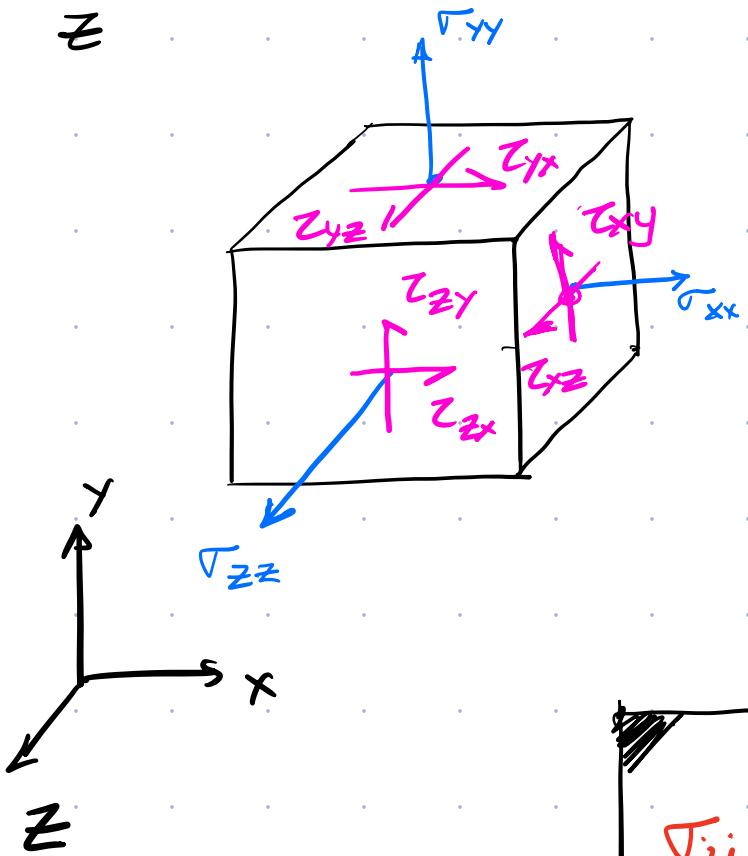
## KEY IDEA - 2

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Stress components  
in Cartesian co-ordinate  
system (X,Y,Z) :



Consider a point (P) with interface parallel to reference planes



Plane normal to  $\tau_{yy}$  axis

Direction of Reference axis

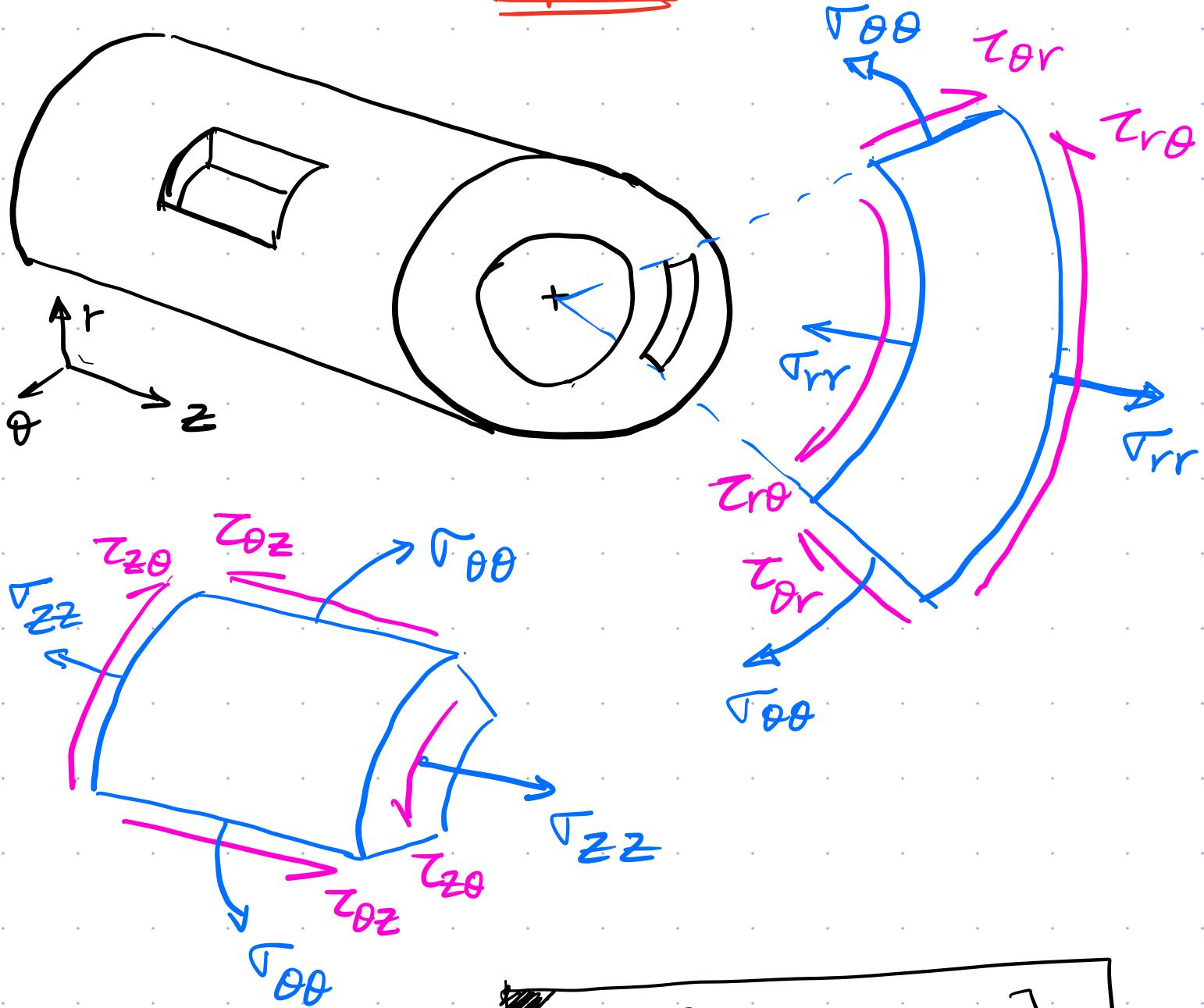
$$\sigma_{ij} = \begin{bmatrix} \sigma_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_{yy} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_{zz} \end{bmatrix}$$

STRESS TENSOR

2<sup>nd</sup> order tensor

### KEY IDEA - 3

Stress components in  
Cylindrical coordinate  
System  $(r, \theta, z)$ :



plane normal  
to  $\theta$  axis  
Direction of  
Reference axis

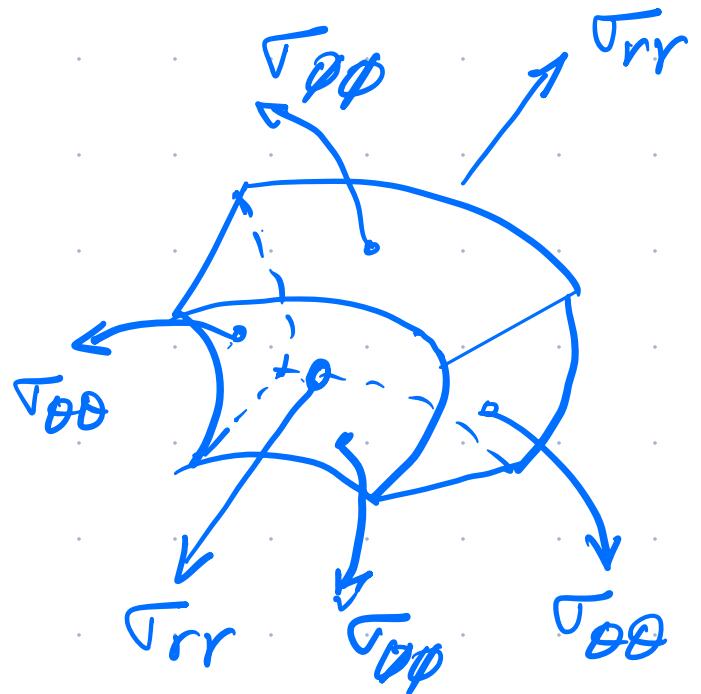
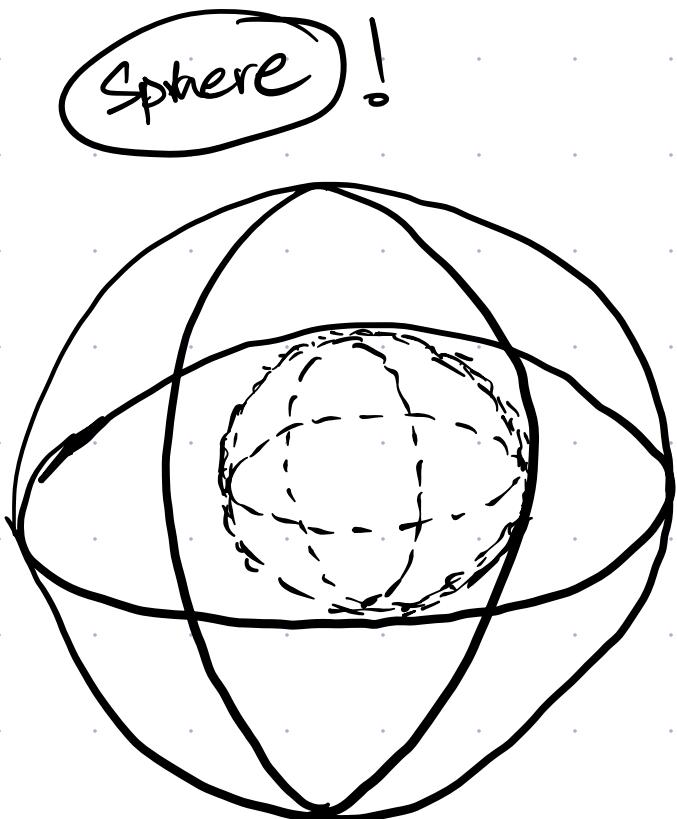
$$\sigma_{ij} = \begin{bmatrix} \sigma_{rr} & \tau_{rz} & \tau_{\theta z} \\ \tau_{\theta r} & \sigma_{\theta\theta} & \tau_{\theta z} \\ \tau_{zr} & \tau_{z\theta} & \sigma_{zz} \end{bmatrix}$$

STRESS TENSOR

2<sup>nd</sup> order tensor

### KEY IDEA - 3

Stress components in spherical coordinate system  $(r, \theta, \phi)$ :



$$\sigma_{ij} = \begin{bmatrix} \sigma_{rr} & \sigma_{r\theta} & \sigma_{r\phi} \\ \sigma_{\theta r} & \sigma_{\theta\theta} & \sigma_{\theta\phi} \\ \sigma_{\phi r} & \sigma_{\phi\theta} & \sigma_{\phi\phi} \end{bmatrix}$$

STRESS TENSOR

plane normal to  $\sigma_{rr}$  axis

Direction of reference axis

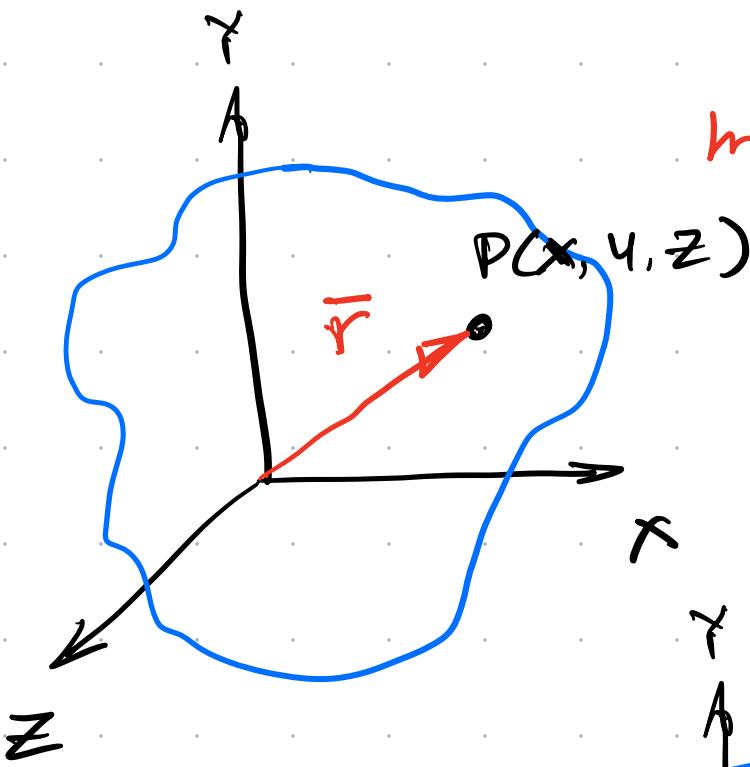
2nd order tensor

## KEY IDEA - 4

Generalizes concept  
of strain:

(9)

**STRAIN:** Quantitative measure of deformation of a body.

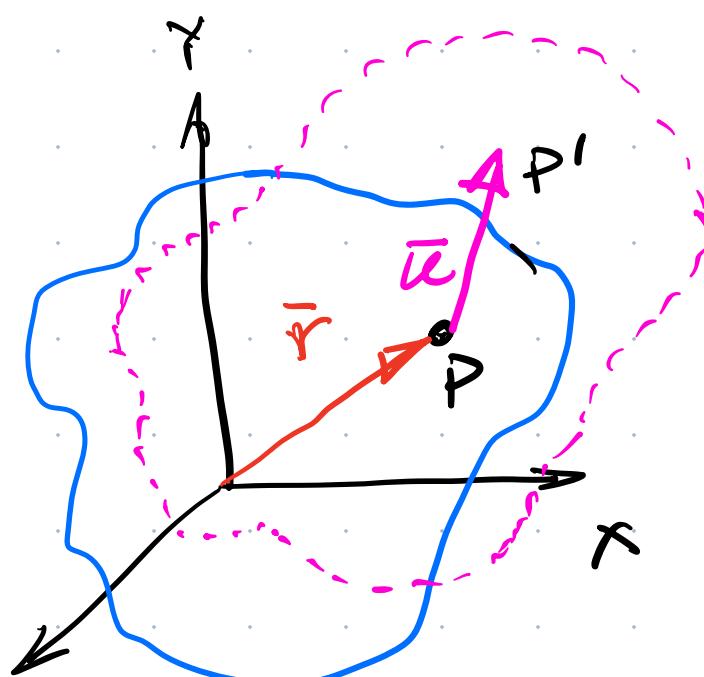


↳ Position vector ( $\vec{r}$ )  
in a body  
↳ Undeformed  
elastic body

↳ Point  $P(x, y, z)$   
moves to a new  
point  $P'$ .

↳  $P'$  is in the  
deformed geometry

↳ Displacement  
field is defined by  $z$   
 $u(x, y, z)$



## DISPLACEMENT FIELD

$\vec{u}(x, y, z)$

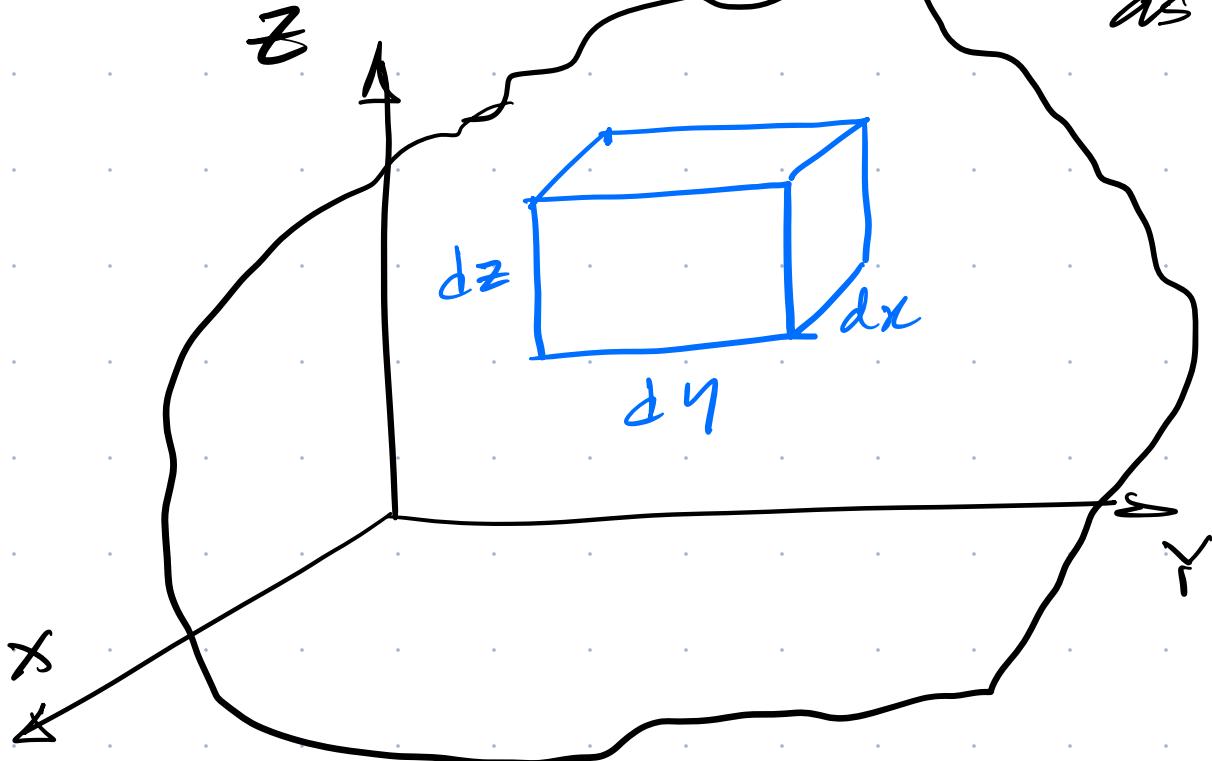
Mathematical description of how a body moves and deforms, point-by-point.  
↳ change in geometry

↳ We need to somehow relate the change in geometry to stress distributions.

↳ Recall: Stress is expressed as a stress tensor.

↳ Relative movement of adjacent points lying along 3 orthogonal axes at a point

## Strain components

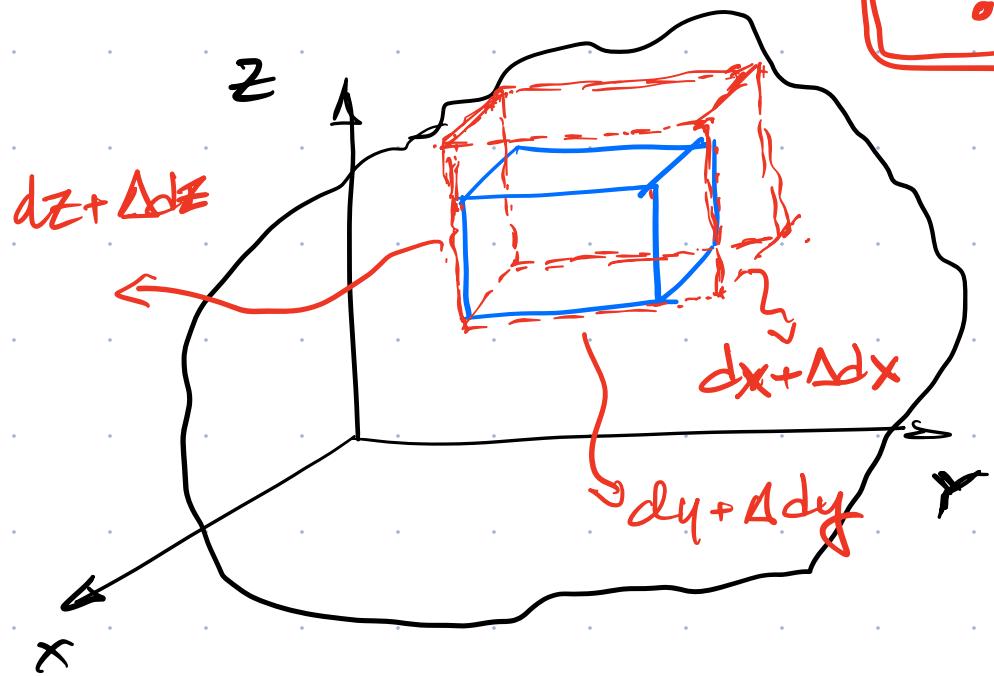


consider a parallelepiped as shown:

$\rightarrow$  It is an infinitesimal entity of an undeformed body.

Case(i): Let the parallelepiped only develop normal stresses on its faces

DILATION: • Volume changes  
• Shape does not change



Strains:

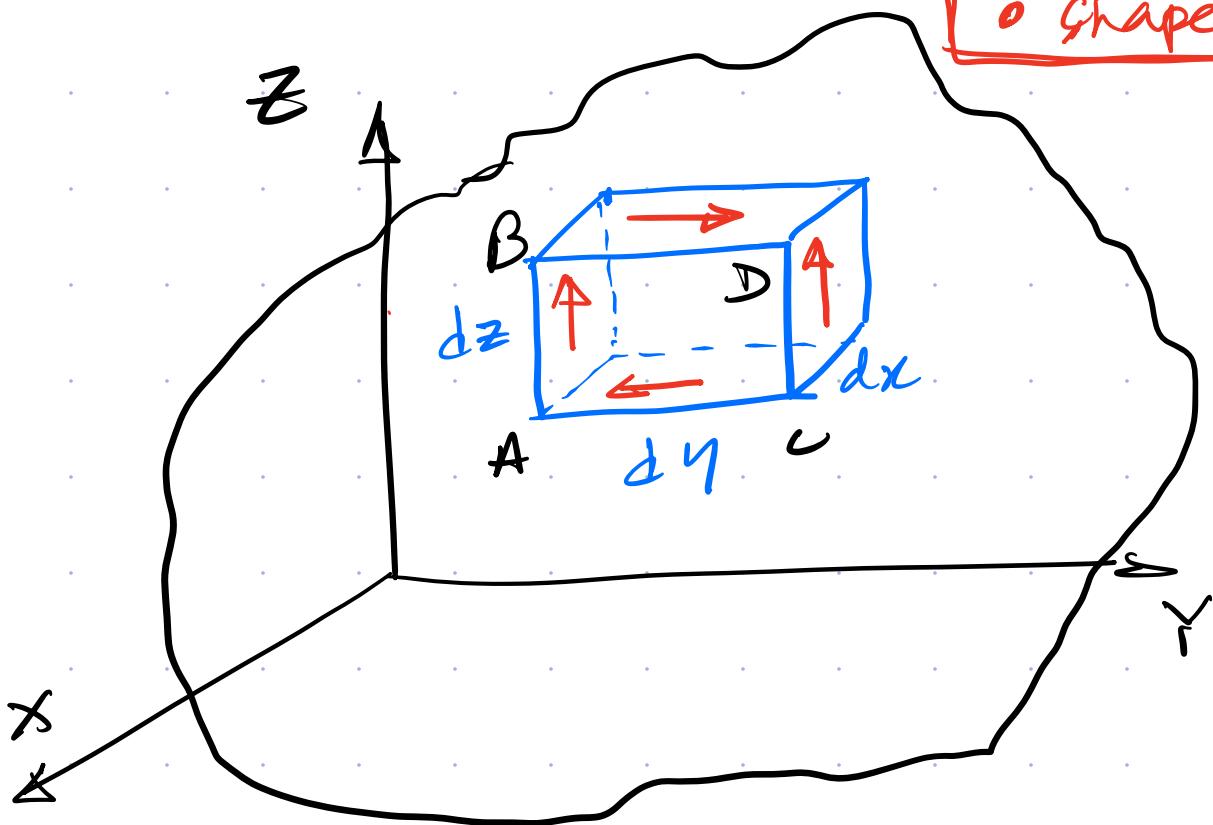
$$\epsilon_{xx} = \frac{\Delta dx}{dx}$$

$$\epsilon_{yy} = \frac{\Delta dy}{dy}$$

$$\epsilon_{zz} = \frac{\Delta dz}{dz}$$

Case (ii) Let the parallelopiped only develop shear stress on its faces. (12)

- Volume does not change
- Shape changes



$$\gamma_{yz} = \gamma_{zy} = \alpha + \beta$$

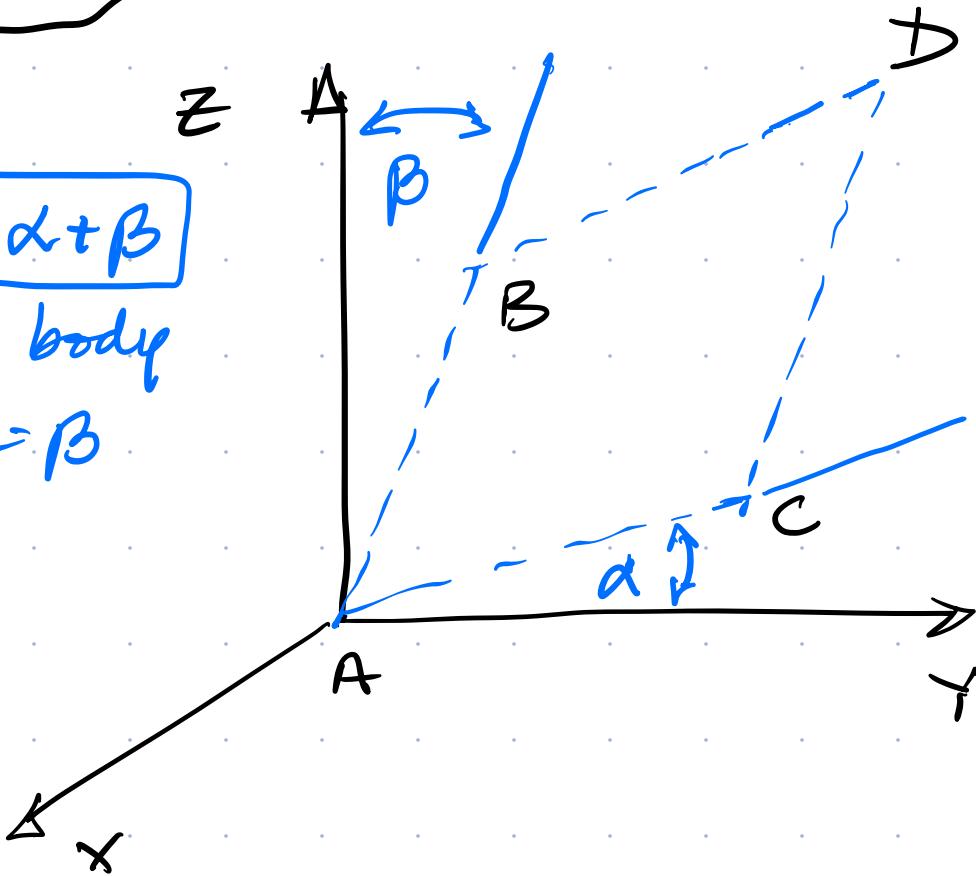
Assume: Rigid body rotation  $\Rightarrow \alpha = \beta$

Strains:

$$\epsilon_{zy} = \epsilon_{yz}$$

$$= \frac{1}{2} \gamma_{yz}$$

$$= \frac{1}{2} \gamma_{zy}$$



COMBINATION OF CASE(i) & CASE(ii)

can lead to a tensor, similar to a stress tensor!

Note: We are dealing with Cartesian co-ordinates in this discussion

$$\epsilon_{ij} = \begin{bmatrix} \epsilon_{xx} & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{yx} & \epsilon_{yy} & \epsilon_{yz} \\ \epsilon_{zx} & \epsilon_{zy} & \epsilon_{zz} \end{bmatrix}$$

Diagonal terms:

Off-diagonal terms:

↳ 6 symmetric terms

↳ Change in shape

↳ No change in volume

↳ 3 normal strains

↳ dilation effect

↳ Change in volume

↳ No change in shape

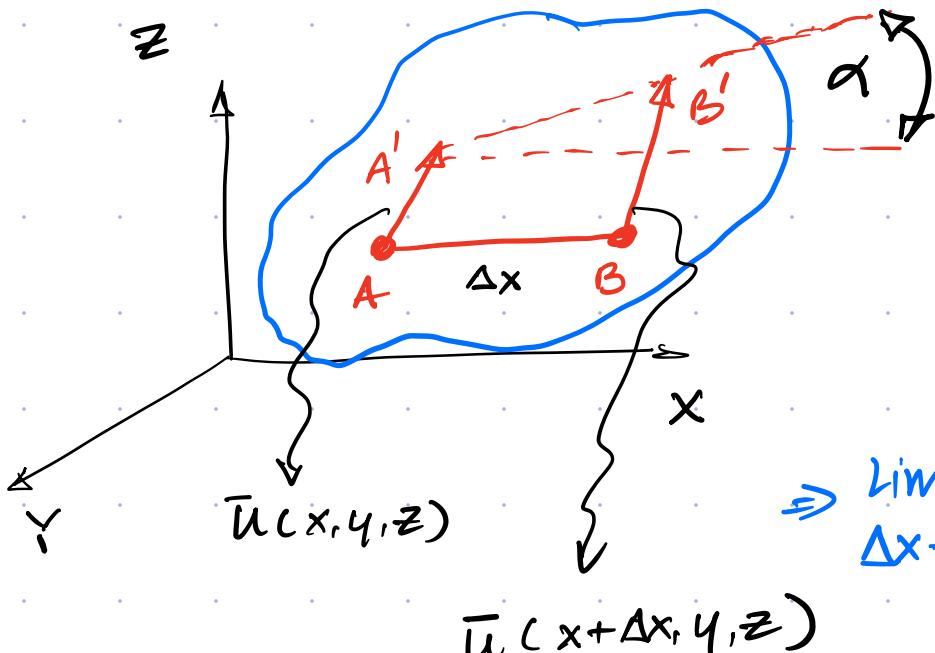
# STRAIN-DISPLACEMENT RELATIONS

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$$\epsilon_{ij} = \begin{bmatrix} \epsilon_{xx} & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{yx} & \epsilon_{yy} & \epsilon_{yz} \\ \epsilon_{zx} & \epsilon_{zy} & \epsilon_{zz} \end{bmatrix}$$

Assume: small deformations

→ Express each term in the matrix in terms of displacement field.



$$\epsilon_{xx} = \lim_{\Delta x \rightarrow 0} \frac{A'B' - AB}{\Delta x}$$

$$\Rightarrow \lim_{\Delta x \rightarrow 0} \frac{u_x(x + \Delta x, y, z) - u_x(x, y, z)}{\Delta x}$$

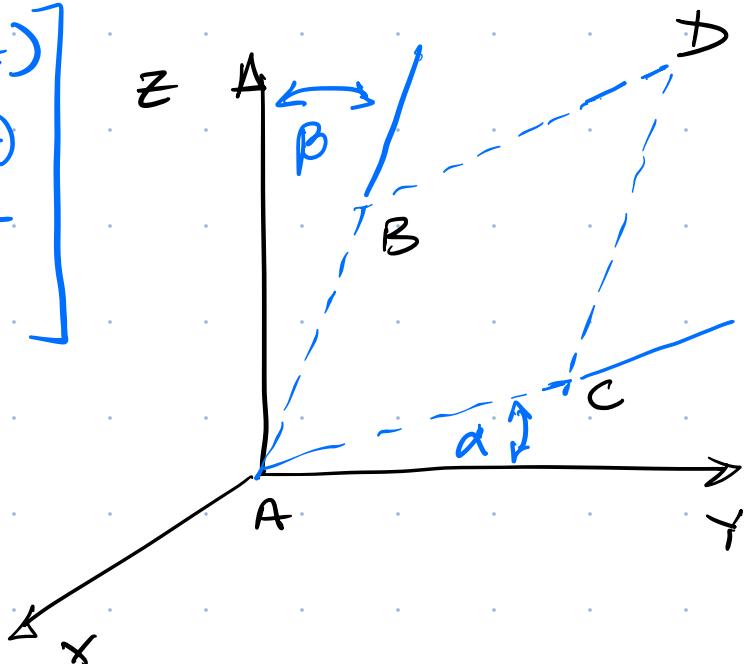
$$\Rightarrow \epsilon_{xx} = \frac{\partial u_x}{\partial x}$$

Similarly,

$$\epsilon_{yy} = \frac{\partial u_y}{\partial y}$$

$$\epsilon_{zz} = \frac{\partial u_z}{\partial z}$$

$$\alpha = \lim_{\Delta x \rightarrow 0} \left[ \frac{u_y(x + \Delta x, y, z) - u_y(x, y, z)}{\Delta x} \right]$$



$$\alpha = \frac{\partial u_y}{\partial x}$$

Similarly

$$\beta = \frac{\partial u_x}{\partial y}$$

Recall:  $E_{yz} = E_{zy}$

$$= \frac{1}{2} \gamma_{yz}$$

$$= \frac{1}{2} \gamma_{zy}$$

$$\gamma_{yz} + \gamma_{zy} = \alpha + \beta$$

$$\gamma_{xy} = \frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x}$$

$\therefore E_{xy} = E_{yx} = \frac{1}{2} \left( \frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right)$

$$E_{xz} = E_{zx} = \frac{1}{2} \left( \frac{\partial u_x}{\partial z} + \frac{\partial u_z}{\partial x} \right)$$

$$E_{yz} = E_{zy} = \frac{1}{2} \left( \frac{\partial u_y}{\partial z} + \frac{\partial u_z}{\partial y} \right)$$

Let's now fill out all terms in the matrix

(16)

$$E_{ij} = \begin{bmatrix} E_{xx} & E_{xy} & E_{xz} \\ E_{yx} & E_{yy} & E_{yz} \\ E_{zx} & E_{zy} & E_{zz} \end{bmatrix}$$

Cartesian  
co-ordinates

$$E_{xx} = \frac{\partial u_x}{\partial x}$$

$$E_{yy} = \frac{\partial u_y}{\partial y}$$

$$E_{zz} = \frac{\partial u_z}{\partial z}$$

$$E_{xy} = E_{yx} = \frac{1}{2} \left( \frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right)$$

$$E_{xz} = E_{zx} = \frac{1}{2} \left( \frac{\partial u_x}{\partial z} + \frac{\partial u_z}{\partial x} \right)$$

$$E_{yz} = E_{zy} = \frac{1}{2} \left( \frac{\partial u_y}{\partial z} + \frac{\partial u_z}{\partial y} \right)$$

$$\boldsymbol{\Gamma}_{ij} = \begin{bmatrix} \sigma_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_{yy} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_{zz} \end{bmatrix}$$

Cartesian  
co-ordinates

$$\boldsymbol{\epsilon}_{ij} = \begin{bmatrix} \epsilon_{xx} & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{yx} & \epsilon_{yy} & \epsilon_{yz} \\ \epsilon_{zx} & \epsilon_{zy} & \epsilon_{zz} \end{bmatrix}$$

Cartesian  
co-ordinates



$$\boldsymbol{\epsilon}_{ij} = \begin{bmatrix} \frac{\partial u_x}{\partial x} & \frac{1}{2} \left( \frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right) & \frac{1}{2} \left( \frac{\partial u_x}{\partial z} + \frac{\partial u_z}{\partial x} \right) \\ \frac{1}{2} \left( \frac{\partial u_y}{\partial x} + \frac{\partial u_x}{\partial y} \right) & \frac{\partial u_y}{\partial y} & \frac{1}{2} \left( \frac{\partial u_y}{\partial z} + \frac{\partial u_z}{\partial y} \right) \\ \frac{1}{2} \left( \frac{\partial u_z}{\partial x} + \frac{\partial u_x}{\partial z} \right) & \frac{1}{2} \left( \frac{\partial u_z}{\partial y} + \frac{\partial u_y}{\partial z} \right) & \frac{\partial u_z}{\partial z} \end{bmatrix}$$

$$\boldsymbol{\tau}_{ij} = \begin{bmatrix} \sigma_{rr} & \sigma_{r\theta} & \sigma_{rz} \\ \sigma_{\theta r} & \sigma_{\theta\theta} & \sigma_{\theta z} \\ \sigma_{zr} & \sigma_{z\theta} & \sigma_{zz} \end{bmatrix}$$

**Cylindrical  
co-ordinates**

$$\boldsymbol{\epsilon}_{ij} = \begin{bmatrix} \epsilon_{rr} & \epsilon_{r\theta} & \epsilon_{rz} \\ \epsilon_{\theta r} & \epsilon_{\theta\theta} & \epsilon_{\theta z} \\ \epsilon_{zr} & \epsilon_{z\theta} & \epsilon_{zz} \end{bmatrix}$$

**Cylindrical  
co-ordinates**



$$\boldsymbol{\epsilon}_{ij} = \begin{bmatrix} \frac{\partial u_r}{\partial r} & \frac{1}{r} \left( \frac{1}{r} \frac{\partial u_r}{\partial \theta} + \frac{\partial u_\theta}{\partial r} - \frac{u_\theta}{r} \right) & \frac{1}{r} \left( \frac{\partial u_r}{\partial z} + \frac{\partial u_z}{\partial r} \right) \\ \frac{1}{r} \left( \frac{1}{r} \frac{\partial u_r}{\partial \theta} + \frac{\partial u_\theta}{\partial r} - \frac{u_\theta}{r} \right) & \left( \frac{u_r}{r} + \frac{1}{r} \frac{\partial u_\theta}{\partial \theta} \right) & \frac{1}{r} \left( \frac{\partial u_\theta}{\partial z} + \frac{1}{r} \frac{\partial u_z}{\partial \theta} \right) \\ \frac{1}{r} \left( \frac{\partial u_r}{\partial z} + \frac{\partial u_z}{\partial r} \right) & \frac{1}{r} \left( \frac{\partial u_\theta}{\partial z} + \frac{1}{r} \frac{\partial u_z}{\partial \theta} \right) & \frac{\partial u_z}{\partial z} \end{bmatrix}$$

$$E_{rr} = \frac{\partial u_r}{\partial r}$$

$$E_{\theta\theta} = \left( \frac{u_r}{r} + \frac{1}{r} \frac{\partial u_\theta}{\partial \theta} \right)$$

$$E_{zz} = \frac{\partial u_z}{\partial z}$$

$$E_{r\theta} = E_{\theta r} = \frac{1}{2} \left( \frac{1}{r} \frac{\partial u_r}{\partial \theta} + \frac{\partial u_\theta}{\partial r} - \frac{u_\theta}{r^2} \right)$$

$$E_{\theta z} = E_{z\theta} = \frac{1}{2} \left( \frac{\partial u_\theta}{\partial z} + \frac{1}{r} \frac{\partial u_z}{\partial \theta} \right)$$

$$E_{rz} = E_{zr} = \frac{1}{2} \left( \frac{\partial u_r}{\partial z} + \frac{\partial u_z}{\partial r} \right)$$

$$\boldsymbol{\tau}_{ij} = \begin{bmatrix} \sigma_{rr} & \sigma_{r\theta} & \sigma_{r\phi} \\ \sigma_{\theta r} & \sigma_{\theta\theta} & \sigma_{\theta\phi} \\ \sigma_{\phi r} & \sigma_{\phi\theta} & \sigma_{\phi\phi} \end{bmatrix}$$

Spherical  
co-ordinates

$$\boldsymbol{\epsilon}_{ij} = \begin{bmatrix} \epsilon_{rr} & \epsilon_{r\theta} & \epsilon_{r\phi} \\ \epsilon_{\theta r} & \epsilon_{\theta\theta} & \epsilon_{\theta\phi} \\ \epsilon_{\phi r} & \epsilon_{\phi\theta} & \epsilon_{\phi\phi} \end{bmatrix}$$

Spherical  
co-ordinates

(20)

$$E_{rr} = \frac{\partial u_r}{\partial r}$$

$$E_{\theta\theta} = \left( \frac{u_r}{r} + \frac{1}{r} \frac{\partial u_\theta}{\partial \theta} \right)$$

$$E_{\phi\phi} = \frac{1}{rsin\theta} \frac{\partial u_\phi}{\partial \phi} + \frac{u_\theta}{r} cot\theta + \frac{u_r}{r}$$

$$E_{r\theta} = E_{\theta r} = \frac{1}{2} \left( \frac{1}{r} \frac{\partial u_r}{\partial \theta} + \frac{\partial u_\theta}{\partial r} - \frac{u_\theta}{r} \right)$$

$$E_{\theta\phi} = E_{\phi\theta} = \frac{1}{2} \left( \frac{1}{rsin\theta} \frac{\partial u_\theta}{\partial \phi} + \frac{1}{r} \frac{\partial u_\phi}{\partial \theta} - \frac{u_\phi}{r} cot\phi \right)$$

$$E_{r\phi} = E_{\phi r} = \frac{1}{2} \left( \frac{1}{rsin\theta} \frac{\partial u_r}{\partial \phi} + \frac{\partial u_\phi}{\partial r} - \frac{u_\phi}{r} \right)$$