

TTK4190 Guidance and Control of Vehicles

Assignment 2, Pt. 2

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Problem 1 - Environmental Disturbances

a)

The code is modified to include 2-D irrotational ocean current in surge and sway:

```

1      %%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
2      % Part 2, 1a) Add current disturbance here
3      %%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
4      beta_vs = deg2rad(45);           % current direction in rad
5      Vc = 1;                         % Current speed m/s
6      uc = Vc*cos(beta_vs - x(6));
7      vc = Vc*sin(beta_vs - x(6));
8      nu_c = [ uc vc 0 ]';

```

b)

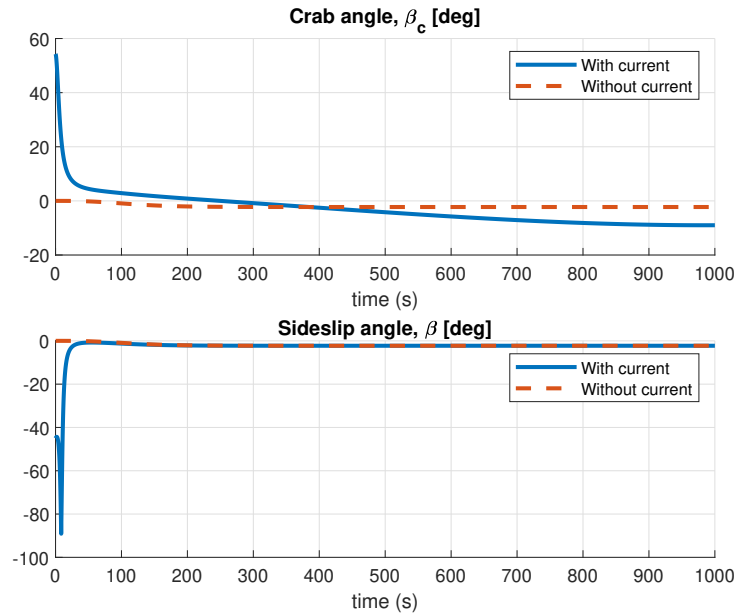


Figure 1: Comparing sideslip and crab angles when there is no current vs current

If considering our ship when there are no ocean currents, we expect the sideslip angle β and the crab angle β_c to be equal [1]:

$$\beta_c = \beta = \sin^{-1}\left(\frac{v}{U}\right) \quad (1)$$

From Figure 1 it is possible to see that the sideslip and crab angles are equal when there are no currents.

When introducing a constant irrotational ocean current, the angles are expected to be different from each other. By looking at Figure 1 it can be verified that is indeed the case.

c)

The code is updated to include wind moments Y_{wind} and N_{wind} occurring after 200 seconds:

```

1      %%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
2      % Part 2, 1c) Add wind disturbance here
3      %%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
4      Vw = 10;                        % wind speed m/s

```

```

5     beta_Vw = deg2rad(135);           % wind direction
6     gamma_w = x(6) - beta_Vw - pi;    % wind angle of attack
7
8     u_w = Vw*cos(beta_Vw - x(6));     % wind x speed
9     v_w = Vw*sin(beta_Vw - x(6));     % wind y speed
10    u_rw = x(1) - u_w;                 % relative x speed
11    v_rw = x(2) - v_w;                 % relative y speed
12    gamma_rw = atan2(v_rw,u_rw);       % relative angle of attack angle
13
14    cy = 0.95;                         % wind coefficients
15    cn = 0.15;                         % wind coefficients
16    ALw = 10*L_loa;                   % lateral projected area
17
18    C_Y = cy * sin(gamma_rw);          % wind coefficient
19    C_N = cn * sin(2*gamma_rw);        % wind coefficient
20
21    V_rw = sqrt(u_rw^2 + v_rw^2);      % relative wind speed
22    if t > 200
23        Ywind = 0.5*rho_a*V_rw^2*C_Y*ALw; %
24        Nwind = 0.5*rho_a*V_rw^2*C_N*ALw*L_loa; %
25    else
26        Ywind = 0;
27        Nwind = 0;
28    end
29    tau_wind = [0 Ywind Nwind]';

```

Problem 2 - Heading Autopilot

a)

Linearizing the nonlinear Coriolis forces $\mathbf{C}_{RB}(\boldsymbol{\nu})\boldsymbol{\nu}$ about $r = u = 0$ and $u = u_d$:

$$\begin{aligned}\mathbf{C}_{RB}\boldsymbol{\nu} &= \begin{bmatrix} 0 & -mr & -mr x_g \\ mr & 0 & 0 \\ mr x_g & 0 & 0 \end{bmatrix} \begin{bmatrix} u \\ v \\ r \end{bmatrix} \\ &= \begin{bmatrix} -vmr - mr^2 x_g \\ mr u \\ mr u x_x \end{bmatrix} := \begin{bmatrix} a \\ b \\ c \end{bmatrix}\end{aligned}$$

The linerized forces are defined as

$$\begin{aligned}\mathbf{C}_{RB}^* &= \begin{bmatrix} \frac{\partial a}{\partial u} & \frac{\partial a}{\partial v} & \frac{\partial a}{\partial r} \\ \frac{\partial b}{\partial u} & \frac{\partial b}{\partial v} & \frac{\partial b}{\partial r} \\ \frac{\partial c}{\partial u} & \frac{\partial c}{\partial v} & \frac{\partial c}{\partial r} \end{bmatrix} \big|_{u=r=0, u=u_d} \\ &= \begin{bmatrix} 0 & -mr & -vm - 2mr x_g \\ mr & mr u & mu \\ mr x_g & 0 & mu x_g \end{bmatrix} \big|_{u=r=0, u=u_d} \\ &= \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & mu_d \\ 0 & 0 & mu_d x_g \end{bmatrix}\end{aligned}\tag{2}$$

Linearizing the nonlinear Coriolis forces $\mathbf{C}_A(\boldsymbol{\nu})\boldsymbol{\nu}$ about $r = u = 0$ and $u = u_d$:

$$\begin{aligned}\mathbf{C}_A(\boldsymbol{\nu})\boldsymbol{\nu} &= \begin{bmatrix} 0 & 0 & Y_{\dot{v}}v + Y_{\dot{r}}r \\ 0 & 0 & -X_{\dot{u}}u \\ -Y_{\dot{v}}v - Y_{\dot{r}}r & X_{\dot{u}}u & 0 \end{bmatrix} \begin{bmatrix} u \\ v \\ r \end{bmatrix} \\ &= \begin{bmatrix} Y_{\dot{v}}vr + Y_{\dot{r}}r^2 \\ -X_{\dot{u}}ur \\ -Y_{\dot{v}}vu - Y_{\dot{r}}ru + X_{\dot{u}}uv \end{bmatrix}\end{aligned}$$

Performing the same operations as in Equation 2 gives

$$\mathbf{C}_A^* = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -X_{\dot{u}}u_d \\ 0 & -Y_{\dot{v}}u_d + X_{\dot{u}}u_d & Y_{\dot{r}}u_d \end{bmatrix}\tag{3}$$

b)

The linearized sway-yaw maneuvering model can be written as

$$\begin{aligned}M &:= \mathbf{M}_{RB} + \mathbf{M}_A \\ N &:= \mathbf{C}_{RB}^* + \mathbf{C}_A^* + \mathbf{D} \\ \dot{\boldsymbol{\nu}}_r &= \underbrace{-\mathbf{M}^{-1}\mathbf{N}}_A \boldsymbol{\nu}_r + \underbrace{\mathbf{M}^{-1}\mathbf{b}}_B \delta \\ y &= \underbrace{[0, 1]}_C \boldsymbol{\nu}_r + \underbrace{0}_D \delta\end{aligned}$$

which again can be transformed into a transfer function using matlab as:

```
1 %% Problem 2 b
2 % Maa hente ut sway og yaw
3 Minv = Minv(2:3,2:3);
4 N = C(2:3,2:3) + D(2:3,2:3);
```

```

5 b = 2*ud*[-Y_Δ;-N_Δ];
6 % Make the state space
7 A = Minv*(-N);
8 B = Minv*b;
9 C = [0 1];
10 D = 0;
11
12 [num,den] = ss2tf(A,B,C,D)

```

giving us the numeric transfer function

$$\frac{r}{\delta} = \frac{8.683e - 5s + 0.615e - 5}{s^2 + 0.1506s + 0.0008} \quad (4)$$

c)

The second-order Laplace transformed Nomoto model yields

$$\frac{r}{\delta}(s) = \frac{K(T_3s + 1)}{(T_1s + 1)(T_2s + 1)} \quad (5)$$

The numerical values for the time constant T_1, T_2, T_3 and the gain K can be found by comparing the numerator and denominator in 4 with 5. By defining the equivalent time constant as $T := T_1 + T_2 - T_3$, the second-order Nomoto model can be approximated as a first-order Nomoto model:

$$\frac{r}{\delta}(s) = \frac{K}{Ts + 1} \quad (6)$$

```

1 %% Problem 2c
2 poles = roots(den);
3
4 T1 = -1/poles(1);
5 T2 = -1/poles(2);
6
7 K = num(3) * T1 * T2;
8
9 T3 = num(2) * (T1 * T2) / K;
10
11 T = T1 + T2 - T3;

```

The numerical values for T and K corresponding to a first-order Nomoto model are $T = 169.54$ and $K = 0.0075$.

d)

The PID controller for the heading autopilot can be found using Algorithm 15.1 in [1]. A third order reference model to generate the desired yaw angle and yaw rate is also used. The derivation is done in the code below:

```

1 %%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
2 % Part 2, 2d) Add the heading controller here
3 e_psi      = ssa(x(6) - psi_d);          % error heading
4 e_r        = x(3) - r_d;                 % error yaw rate
5 e_int      = e_int + h * e_psi;          % integral of heading error
6 Δ_c        = PID.heading(e_psi,e_r,e_int); % rudder angle command (rad)
7 % Δ_c = 0.1;
8 %%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%

```

```

1 function Δ_c = PID.heading(e_psi,e_r,e_int)

```

```

2
3 w_b      = 0.06;           % bandwidth
4 zeta     = 1;              % relative damping ratio
5
6 K        = 0.0075;         % NOMOTO gain
7 T        = 169.549327910636; % NOMOTO time constant
8
9 m        = T/K;            %
10 d       = 1/K;            %
11 k       = 0;              %
12
13 % Compute the natural frequency
14 w_n     = 1/sqrt(1 - 2*zeta^2 + sqrt(4*zeta^4 - 4*zeta^2 + 2)) * w_b;
15
16 Kp      = m*w_n^2-k;       % Compute the P gain
17 Kd      = 2*zeta*w_n*m - d; % Compute the D gain;
18 Ki      = w_n/10 * Kp;     % Compute the I gain;
19
20 % PID control law
21 Δ_c = -( Kp*e_psi + Kd * e_r + Ki * e_int );
22 end

```

```

1 function xd_dot = ref_model(xd,psi_ref)
2 w_ref      = 0.03;
3 zeta      = 1;
4
5
6 a1 = w_ref + 2*zeta*w_ref;
7 a2 = 2*zeta*w_ref^2 + w_ref^2;
8 a3 = w_ref^3;
9 b3 = a3;
10 A = [ 0 1 0; 0 0 1; -a3 -a2 -a1];
11 B = [0 0 b3]';
12
13 xd_dot = A*xd + B*psi_ref;
14 end

```

The performance of the PID controller with regards to the environmental disturbances is shown in Figure 3. Since we have made a heading autopilot, the yaw angle is controlled to 0, handling both the ocean current and the wind disturbances. This is the expected behaviour of the PID controller. The North-East-plot shows the distance traveled, however. In this plot it is possible to see the ship moving constantly to the North-East with the $\psi = 0$, $\beta \neq 0$ and $\beta_c \neq 0$. It is because of β and β_c we get behaviour of Figure 2a. If we want the ship to follow a desired course, a course autopilot needs to be implemented.

To summarize, the PID controller for the heading autopilot does indeed compensate for environmental disturbances.

e)

Based on the simulation results from Figure 3c, it does not seem like integrator windup is a problem for the heading angle. The actual and the commanded yaw angles are close to identical when the controller changes the value of ψ_c .

```

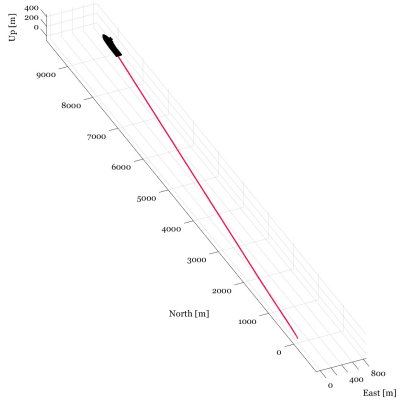
1 psi_ref = [10 * pi/180; - 20 * pi/180]; % desired yaw angle (rad)

```

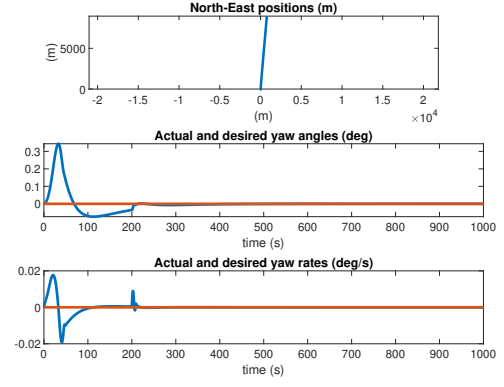
```

1 if t > 200
2     xd_dot = ref_model(xd,psi_ref(2));
3 else
4     xd_dot = ref_model(xd,psi_ref(1));
5 end

```

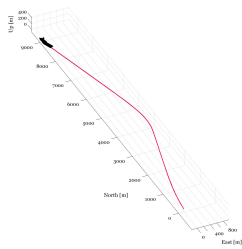


(a) The course χ of the Boat

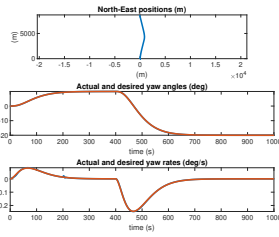


(b) The heading ψ of the Boat

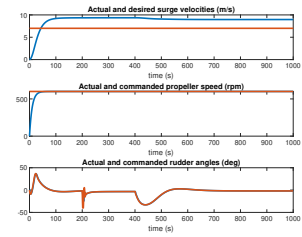
Figure 2: The performance of the PID heading controller



(a) The course χ of the Boat



(b) The heading ψ of the Boat



(c) The speed of the Boat

Figure 3: Controller performance for a 10° heading setpoint followed by a -20° heading setpoint

References

- [1] T. Fossen, *Handbook of Marine Craft Hydrodynamics and Motion Control*. John Wiley & Sons, 2011.