

MCT-241: Modelling & Stimulation

Modeling, Simulation, and Control of a Four-Bar Mechanism with a Brushless Servo Motor

Submitted By:

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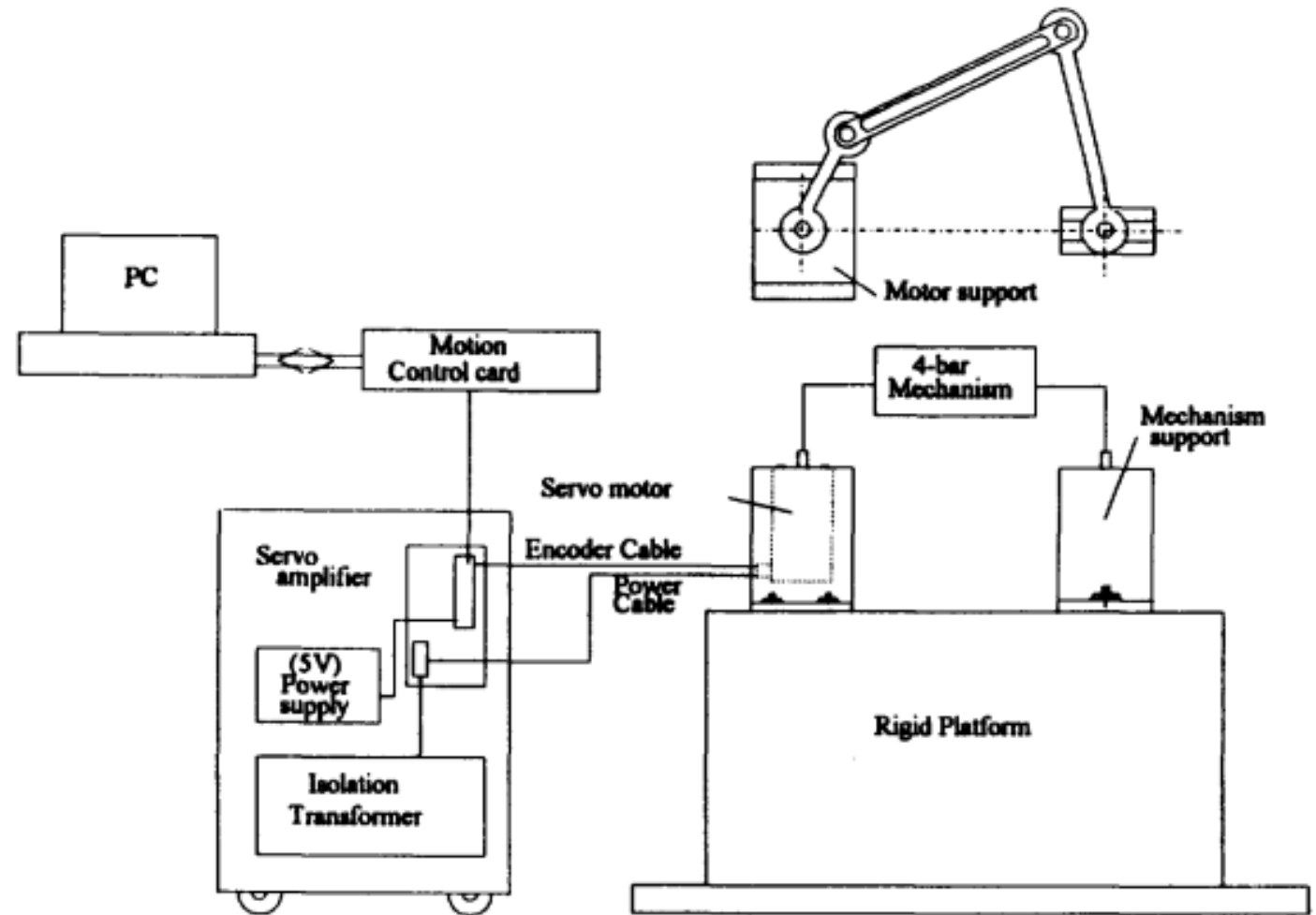
Objectives

- To obtain the mathematical model of the four-bar mechanism and motor system.
- To study the variation of system parameters and analyze their effects.
- To model and simulate the system using MATLAB/Simulink.
- To design a control strategy to achieve precise motion control.
- To evaluate the system's response under different conditions and optimize performance.

Introduction

Imagine a mechanical arm connected to a motor that allows it to move in a controlled manner. This arm setup, known as a four-bar mechanism, is common in machines that need to transfer rotational motion into a specific type of movement, like robotic arms or vehicle suspension systems. By modeling and simulating this setup on a computer, we can understand how the system behaves under different conditions and tweak the motor's performance to ensure it moves just right.

Block Diagram



Mathematical Model of Entire System

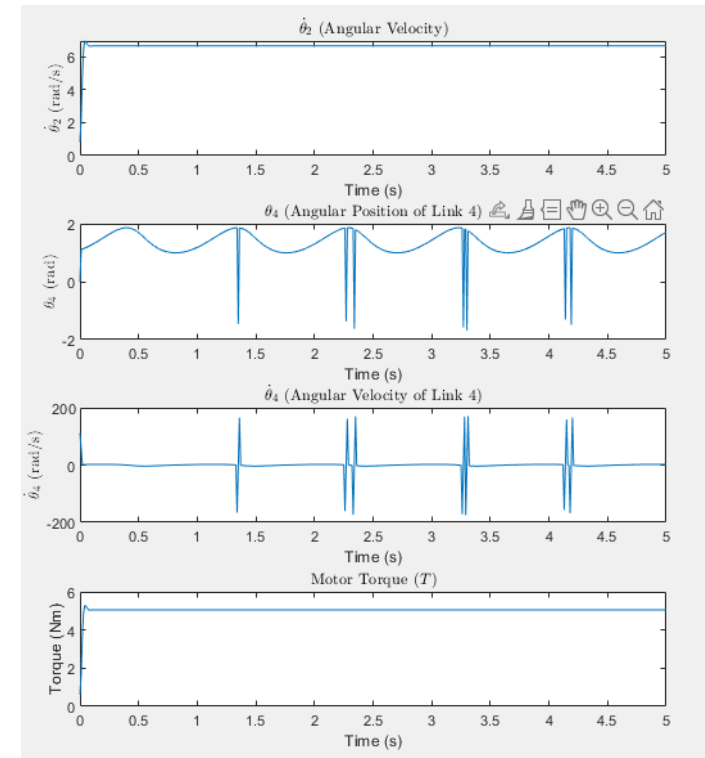
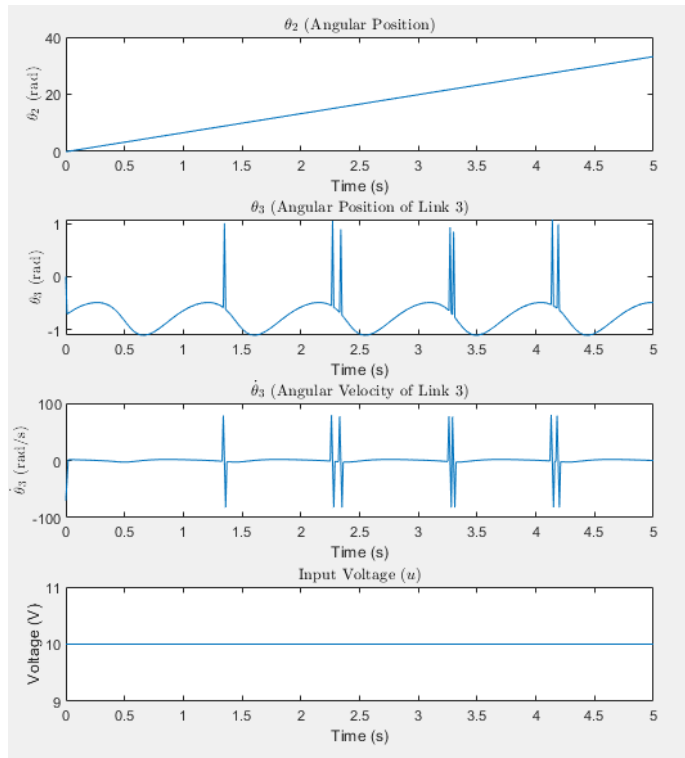
$$\begin{aligned} \ddot{\theta}_2 \left[J_m + m_2 r_2^2 + J_2 + m_3 a_2^2 + 2m_3 a_2 \left(\frac{\partial \theta_3}{\partial \theta_2} \right) (p \cos(\theta_2 - \theta_3) + q \sin(\theta_2 - \theta_3)) \right. \\ \left. + \left(\frac{\partial \theta_3}{\partial \theta_2} \right)^2 (m_3 (p^2 + q^2) + J_3) + \left(\frac{\partial \theta_4}{\partial \theta_2} \right)^2 (m_4 r_4^2 + J_4) \right] \\ + \dot{\theta}_2^2 \left[\left(\frac{\partial^2 \theta_3}{\partial \theta_2^2} \right) \left(\frac{\partial \theta_3}{\partial \theta_2} \right) (m_3 (p^2 + q^2) + J_3) + \left(\frac{\partial^2 \theta_3}{\partial \theta_2^2} \right) m_3 a_2 (p \cos(\theta_2 - \theta_3) \right. \\ \left. + q \sin(\theta_2 - \theta_3)) + \left(\frac{\partial^2 \theta_4}{\partial \theta_2^2} \right) \left(\frac{\partial \theta_4}{\partial \theta_2} \right) (m_4 r_4^2 + J_4) \right. \\ \left. + \left(\frac{\partial \theta_3}{\partial \theta_2} \right) m_3 a_2 \left(q \cos(\theta_2 - \theta_3) \left(1 - \frac{\partial \theta_3}{\partial \theta_2} \right) - p \sin(\theta_2 - \theta_3) \left(1 - \frac{\partial \theta_3}{\partial \theta_2} \right) \right) \right] = K_t I, \quad (2) \end{aligned}$$

State-Space Representation

$$\dot{\mathbf{x}} = A\mathbf{x} + B\mathbf{u}$$

$$A = \begin{bmatrix} 0 & 1 & 0 \\ 0 & -\frac{\partial C}{\partial \dot{\theta}_2} / J_{\text{eff}} & K_t / J_{\text{eff}} \\ 0 & -K_e / L & -R / L \end{bmatrix}, \quad B = \begin{bmatrix} 0 \\ 0 \\ 1 / L \end{bmatrix}$$

Stimulated Graphs



Conclusion

This project modeled and analyzed a servo motor coupled to a four-bar mechanism using linear and nonlinear approaches. While the linear model provided simplicity and efficiency for steady-state analysis, the nonlinear model captured intricate dynamics and offered high-fidelity results. Graphical outputs revealed the significant impact of nonlinear effects on performance, highlighting their necessity for realistic simulations. This work provides a foundation for advanced control strategies and system optimization.