

Wed 12 Feb§2.5 2nd-ORDER MODELING

(ROYCE 3.7)

AKA MECHANICAL & ELECTRICAL VIBRATIONS

In this section we examine two real-world applications of ~~many~~ modeling with 2nd-order linear DEs with constant coefficients, namely vibrations & oscillations in mechanical and electrical systems.

2.5.1 Mechanical Vibrations - Derivation

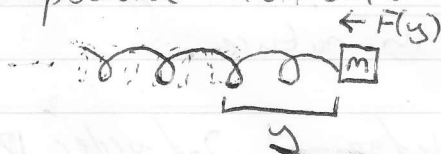
Here we are looking at the motion of an object about some equilibrium point when there is both friction present and some ~~known~~ external ^{force} ~~system~~ acting on the system.

The simplest such setup is to look at an object with mass m attached to a (massless) spring.

Let y be the displacement of the mass from its position of rest ^{as a function of t} . Recall Hooke's Law: the force pulling on the mass is proportional to its displacement. In other words,

$$F_s(y) = -ky$$

for some positive constant k



Since $F = my''$, we thus have a 2nd-order DE $my'' = -ky$, or $my'' + ky = 0$.

Next, we'll look at where the mass experiences a frictional force proportional to its velocity (recall that friction is more closely proportional to velocity squared, but this assumption is valid for a large range of velocities, and makes the mathematical analysis a ~~lot~~ easier). In other words,

$$F_f(y) = -\gamma y'$$

for some positive constant γ (we need the -ve sign, since friction always acts opposite to the direction of motion). In this case $F = my'' = F_s(y) + F_f(y)$, or

$$my'' + \gamma y' + ky = 0.$$

Finally, we'll examine what there is a third known force $g(t)$ acting on the mass (for example, if the object we're studying is a violin string being played, then $g(t)$ might be the force imparted on the string by the violinist's bow). In this case we then have that $F = my'' = F_s(y) + F_f(y) + g(t)$, or
 or $my'' + \gamma y' + ky = g(t).$ ——— ①

This is a 2nd-order linear non-homogeneous DE with constant coefficients in front of the y terms - exactly what we've been studying in the past few lectures!

* ——— Warning: This is only an approximation: springs have finite length etc.

Define 2.5.2 In the above equation we give certain names to the various parameters:

- m is the mass of the object, always > 0
- γ is the damping constant, always > 0
- k is the spring constant, always > 0
- $g(t)$ is the forcing function, which could be anything, but is usually periodic in nature.

We'll now use our knowledge of 2nd-order linear DEs to investigate what the solutions to the equation $my'' + \gamma y' + ky = g(t)$ look like for varying values of m, γ, k & $g(t)$.

2.5.3 Frictionless System with no applied Force

The simplest case is when there's no friction ($\gamma = 0$) and no external applied force on the system ($g(t) = 0$). In that case we have the DE $my'' + ky = 0$, $m, k > 0$.

This has the characteristic equation $mr^2 + k = 0$, which has the complex roots $r = \pm i\sqrt{\frac{k}{m}}$.

Going back over our notes on 2nd-order constant-coefficient ODEs where the CE has complex roots, we see that the general solution to $my'' + ky = 0$ is given by

$$y = A \cos(\omega_0 t) + B \sin(\omega_0 t), \quad (2)$$

where $\omega_0 = \sqrt{\frac{k}{m}}$ is called the natural frequency of the system.

Interpretation: heavier objects vibrate more slowly, stiffer springs \Rightarrow faster vibration.

It is often more useful to write the solution in the form

$$y = R \cos(\omega_0 t - \delta), \quad (3)$$

(this can't always be done),

where R is called the amplitude of the solution, and

δ is the phase shift of the solution.

(usually have $-\pi < \delta \leq \pi$, sometimes $0 \leq \delta < 2\pi$)

To see how to get from one form to the other, note that we can use the compound angle formula

$$\cos(P - Q) = \sin P \sin Q + \cos P \cos Q$$

to then rewrite $R \cos(\omega_0 t - \delta) = R \cos(\omega_0 t) \cos(\delta) + R \sin(\omega_0 t) \sin(\delta)$

Comparing this with the equation $A \cos(\omega_0 t) + B \sin(\omega_0 t)$, we see that

$$A = R \cos(\delta) \quad \& \quad B = R \sin(\delta)$$

So

$$R^2 = A^2 + B^2, \quad \tan(\delta) = \frac{B}{A}.$$

However, care must be taken when solving for δ in the final equation above: we must choose δ in the correct quadrant according to the signs of $\cos(\delta)$ & $\sin(\delta)$, as $\delta = \arctan(\frac{B}{A})$ only ever returns a value between $-\frac{\pi}{2}$ & $\frac{\pi}{2}$.

Example 2.5.4 A 100g ^{placed on a frictionless flat surface and} ~~iron~~ ^{to the right} weight is attached to a (sideways-acting) spring. If we pull the weight 20cm ^{to the right} from its position of rest, we measure ^{a leftward} ~~an inward~~ pulling force of 2.88 N on the block. Suppose ^{at $t=0$} we release the block ~~to the~~ left of its rest position with a velocity of ~~ms~~ ^{ms} 150 cm s^{-1} to the right. Find the position of the block at time t .

in meters

Solution We let $y(t)$ be the (horizontal) position of the block at time t . We know:

- $y(0) = -0.1$
- $y'(0) = 1.5$
- $my'' + ky = 0$, $m = 0.1$, $k = ?$

By Hooke's law $F_s(y) = -ky$.

we have $F_s(0.2) = -2.88$.

So $-2.88 = -k \cdot 0.2$

$\Rightarrow k = 14.4$

Hence the DE for the system is $0.1y'' + 14.4y = 0$

Solution (general): $y = A \cos(\omega_0 t) + B \sin(\omega_0 t)$, here

$\omega_0 = \sqrt{\frac{k}{m}} = \sqrt{\frac{14.4}{0.1}} = \sqrt{144} = 12$.

$\Rightarrow y = A \cos(12t) + B \sin(12t)$.

Apply ICs: $y(0) = -0.1$

$\Rightarrow -0.1 = A \quad (= -\frac{1}{10})$

And $y' = -12A \sin(12t) + 12B \cos(12t)$

So $y'(0) = 1.5 \Rightarrow 1.5 = 12B$

$\Rightarrow B = 0.125 \quad (= \frac{1}{8})$.

So solution is $y(t) = -\frac{1}{10} \cos(12t) + \frac{1}{8} \sin(12t)$.

[Insert previous page's stuff on $y = R \cos(\omega_0 t - \delta)$ here]

Example 2.5.5 Write the solution to the previous example

in the form $y = R \cos(\omega_0 t - \delta)$, and in so doing

find • the maximum displacement of the block from its rest position

• the first positive t when the block crosses its rest position

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Solution we have $y = A \cos(\omega_0 t) + B \sin(\omega_0 t)$ with

$$A = -\frac{1}{10}, \quad B = \frac{1}{8}, \quad \omega_0 = 12.$$

$$\text{RM } A = R \cos(\delta), \quad B = R \sin(\delta)$$

$$\text{so } R^2 = A^2 + B^2, \quad \tan(\delta) = \frac{B}{A}$$

$$R = \sqrt{\left(\frac{1}{10}\right)^2 + \left(\frac{1}{8}\right)^2}$$

$$= \frac{1}{2} \sqrt{\left(\frac{1}{5}\right)^2 + \left(\frac{1}{4}\right)^2}$$

$$= \frac{1}{2} \sqrt{\frac{1}{25} + \frac{1}{16}}$$

$$= \frac{1}{2} \sqrt{\frac{16+25}{400}}$$

$$= \frac{1}{2} \sqrt{\frac{41}{400}}$$

$$= \frac{\sqrt{41}}{40} \approx 0.16008..$$

$$\tan(\delta) = \frac{\frac{1}{8}}{-\frac{1}{10}} = -\frac{5}{4}$$

$$\text{so } \delta = \arctan^* \left(-\frac{5}{4}\right)$$

$$= -0.89606 + 1 \cdot \pi$$

But $\cos \delta < 0$, $\sin \delta > 0 \Rightarrow \delta$ 2nd Quadrant

$$\Rightarrow \delta = \pi - 0.89606 = 2.24553.$$

$$\frac{\pi}{2} < \delta < \pi$$

Thus, to 5 d.p., we have

$$y = 0.16008 \cos(12t - 2.24553)$$

We see then that the amplitude of the oscillation is ≈ 16.008 cm, and the first time when the block crosses its rest position is when $12t - 2.24553 = -\frac{\pi}{2}$
 $\Rightarrow t = \frac{1}{12} \left(2.24553 + \frac{\pi}{2} \right)$
 $= 0.05623$ s.

