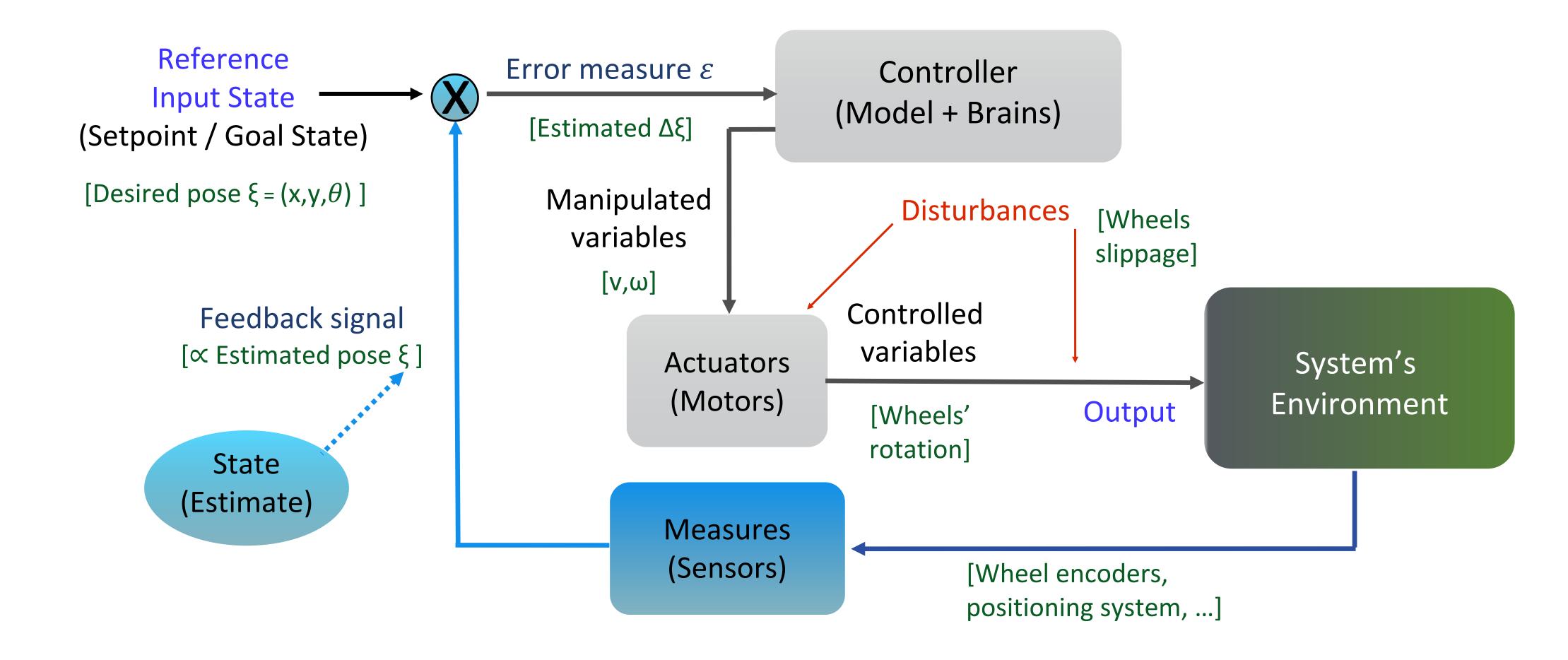


Closed-loop vs. Open-loop control



Closed-loop: status information is fed back to the controller to evaluate the difference between the desired setpoint (goal) and the actual output, and to implement corrective actions, if needed

Open-loop: feedback information is not used to implement corrective actions, the assumption is that, given the inputs, the desired results / goals will be achieved

Types of goal states

A robot is a *goal-driven* physical entity



Achievement goals (typical of AI):

States the system tries to reach and once reached, the job is done

Exit from a maze, reach a specific location or pose, complete a construction Maintenance goals (typical of Control):

Require a continual active work/tracking

Keep balance for a bipedal robot, keep following a wall, keep tracking a moving target

External goal states

Internal goal states

Get to the kitchen
Balance a pole
Find a treasure (!)

Keep battery levels in some range Avoid excessive torque on the effectors

Types of feedback-based controllers

The goal of any control system is to minimize the Error: the difference between the current (as measured) state and the desired goal state

The adopted representation of the error

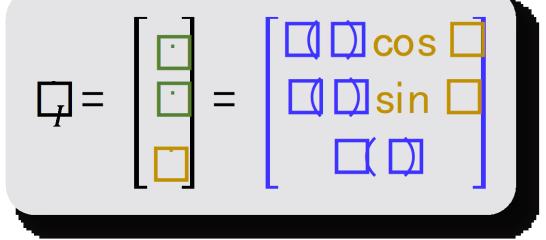
- has a magnitude
- has a direction
- has a scale, interval
- can be quantitative: continuous, discrete, binary
- can be ordinal: ranking, binary

The different ways the error is represented and is treated give raise to a number of different frameworks for control, we will focus on the case of <u>quantitative</u> errors in the context of <u>PID</u> controllers

Dynamical systems and controllers

ightharpoonup Dynamical system: a system whose state descriptor $x \in \mathbb{R}^n$ changes *continuously* (almost always) according to a law

$$\frac{d\mathbf{x}}{dt} = \dot{\mathbf{x}} = \mathbf{F}(\mathbf{x})$$



A controller is defined to change the coupled robot and environment system into a dynamical system showing the desired behavior

$$\dot{\mathbf{x}} = F(\mathbf{x}, \mathbf{u})$$
 Dynamical system coupled with controls

$$\dot{\mathbf{x}} = F(\mathbf{x}, H_i(G(\mathbf{x})))$$

$$y = G(x)$$
 Observed values of the state

$$\dot{\mathbf{x}} = \Phi(\mathbf{x})$$

$$\mathbf{u} = H_i(\mathbf{y})$$

Controls, depending on observed values of the state

Dynamical systems and Types of controllers

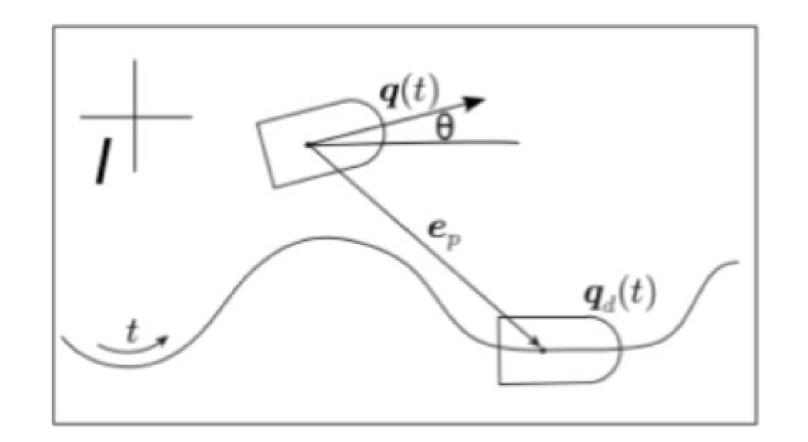
$$\dot{\mathbf{x}} = F(\mathbf{x}, \mathbf{u})$$

$$\mathbf{y} = G(\mathbf{x})$$

$$\mathbf{u} = H_i(\mathbf{y})$$

- Open-loop control: No sensing
- Feedback control (closed-loop): Sense error, determine control response.
- Feedforward control (closed-loop): Sense disturbance, predict resulting error, respond to predicted error before it happens.
- Model-predictive control (closed-loop): Plan trajectory to reach goal; Take first step; Repeat.

Example: trajectory tracking



The tracking error vector e is conveniently expressed in terms of its projections on the rotated reference frame of the robot wrt to the inertial frame. In this way the positional part of the error is the Cartesian component of the error expressed in a reference frame aligned with the current orientation of the robot:

$$e = \begin{bmatrix} e_1 \\ e_2 \\ e_3 \end{bmatrix} = \begin{bmatrix} \cos(\theta) & \sin(\theta) & 0 \\ -\sin(\theta) & \cos(\theta) & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} x_d - x \\ y_d - y \\ \theta_d - \theta \end{bmatrix} = \begin{bmatrix} (x_d - x)\cos(\theta) + (y_d - y)\sin(\theta) \\ -(x_d - x)\sin(\theta) + (y_d - y)\cos(\theta) \\ \theta_d - \theta \end{bmatrix}$$

Differentiating wrt time and using kinematic equations for expressing x(t), y(t), $\theta(t)$, $x_d(t)$, $y_d(t)$, $\theta_d(t)$, the error dynamics becomes:

$$\dot{e}_1 = v_d \cos(e_3) - v + e_2 \omega$$

$$\dot{e}_2 = v_d \sin(e_3) - e_1 \omega$$

$$\dot{e}_3 = \omega_d - \omega$$

Find v(t), $\omega(t)$ control laws that take the error steadily to zero over the entire trajectory

Controlling a simple system

- Consider a simple system: $\dot{x} = F(x, u)$
 - Scalar variables x and u, not vectors \mathbf{x} and \mathbf{u} .
 - Assume x is observable: y = G(x) = x
 - Assume effect of motor command u: $\frac{\partial F}{\partial u} > 0$
- The setpoint x_{set} is the desired value.
 - The controller responds to error: $e = x x_{set}$
- The goal is to set u to reach e = 0.

- Use action u to push back toward error e = 0
 - error e depends on state x (via sensors y)

- What does pushing back do?
 - Depends on the structure of the system
 - Velocity versus acceleration control



- How much should we push back?
 - What does the magnitude of *u* depend on?

Velocity or acceleration control?

• If error reflects \mathbf{x} , does \mathbf{u} affect \mathbf{x}' or \mathbf{x}'' ?

• Velocity control: $\mathbf{u} \to \mathbf{x}'$ (valve fills tank)

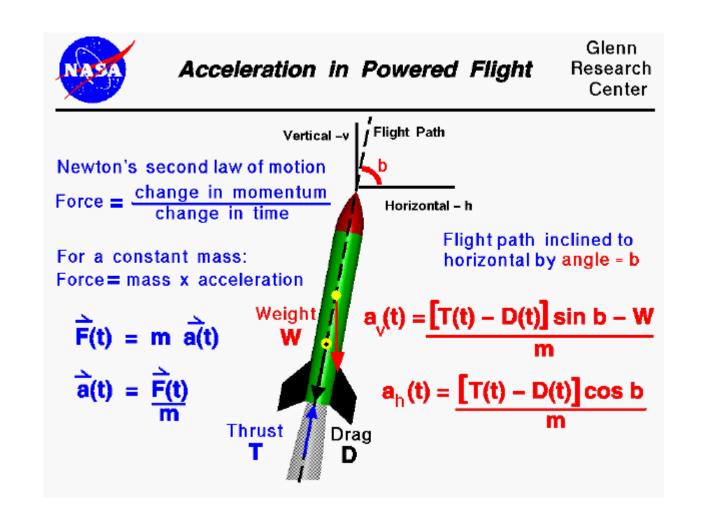


• Acceleration control: $\mathbf{u} \rightarrow \mathbf{x}''$ (rocket)

$$- \text{ let } \mathbf{x} = (x \ v)^T$$

$$\dot{\mathbf{x}} = \begin{pmatrix} \dot{x} \\ \dot{v} \end{pmatrix} = F(\mathbf{x}, \mathbf{u}) = \begin{pmatrix} v \\ u \end{pmatrix}$$

$$\dot{v} = \ddot{x} = u$$





Bang-Bang control

- Push back, against the *direction* of the error
 with constant action u
- Error is $e = x x_{set}$ $e < 0 \implies u = on \implies \dot{x} = F(x, on) > 0$ $e > 0 \implies u = off \implies \dot{x} = F(x, off) < 0$

 For implementing heating up & cooling down, the on/off switch must be replaced by a fixed signal of opposite sign/effect (e.g., amount of electrical power G used to heat up / cool down) It can be as basic an on / off switch for control:

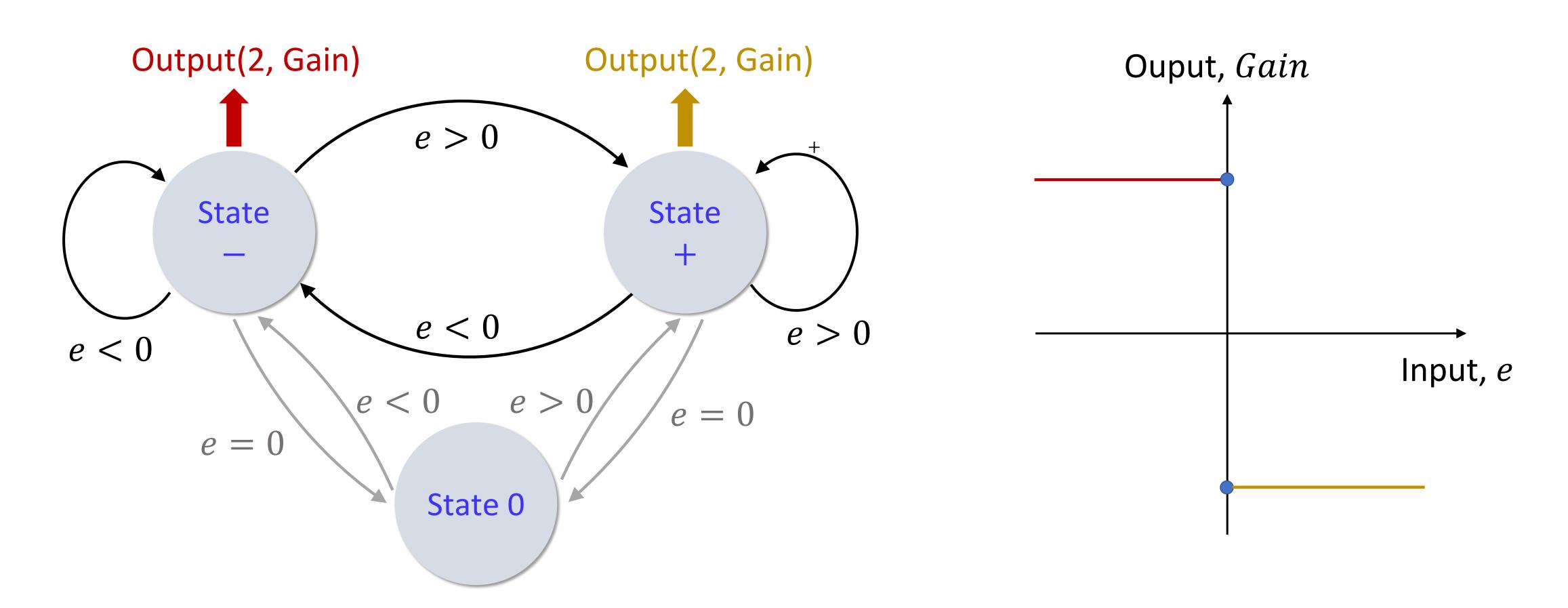
- Send a fixed control signal G when e < 0,
- Don't do any control actions when $e \ge 0$
- Or vice versa
- E.g., a thermostat in wintertime: heat up when temperature is below the setpoint, do nothing when temperature is above the setpoint

Emko ESM-3710-N Bangbang Temperature controller PTC -50 up to 130 °C 16 A relay (L x W x H) 65 x 76 x 35 mm

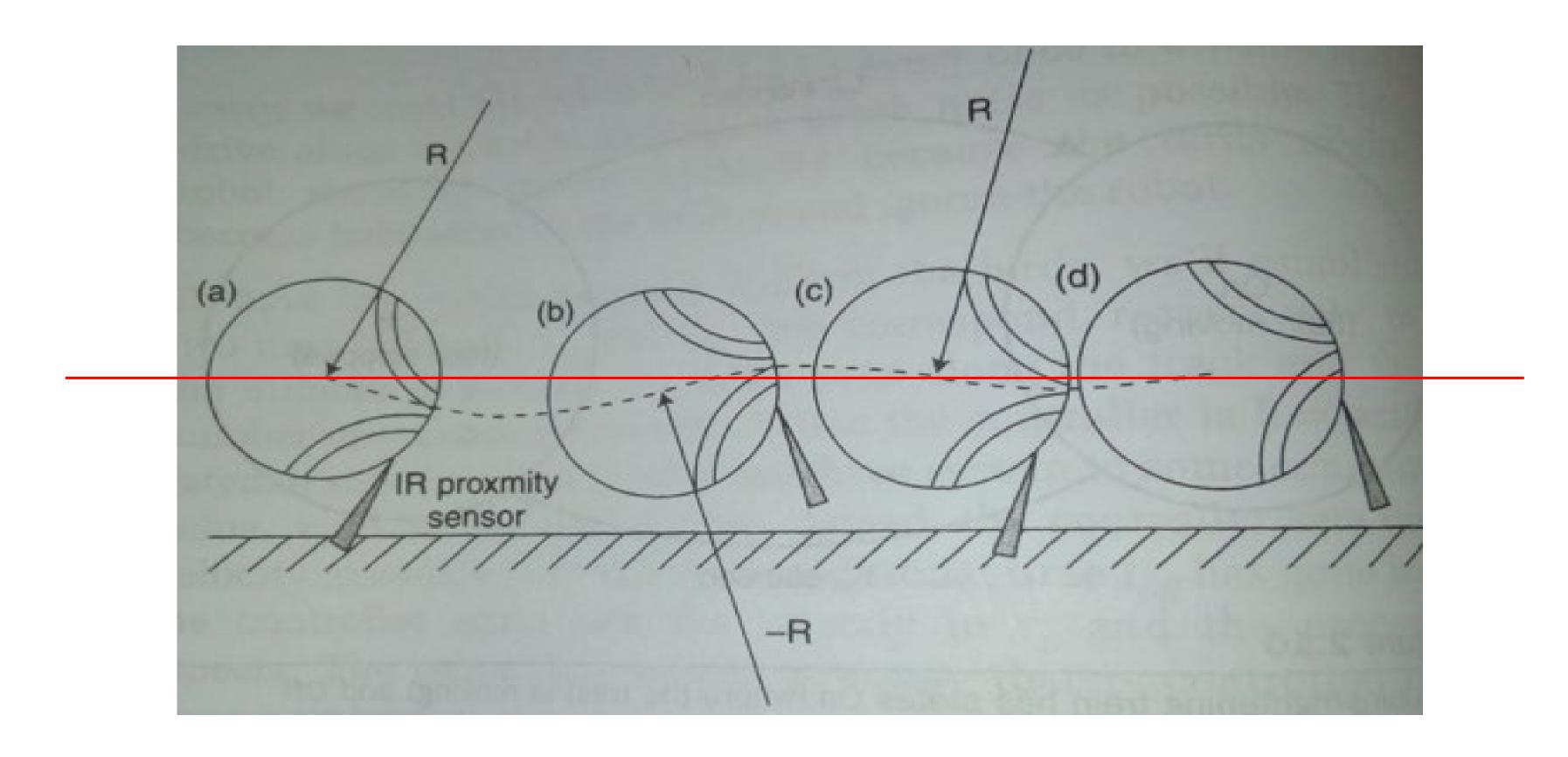


Bang-Bang control

- More in general, a bang-bang controller is equivalent to a two-state system or, more precisely, to a three-state system, where one of the states is neutral (no controls)
- State transitions happen when the error changes direction or passes from zero to non-zero and vice versa
- Action at each state depends on a fixed gain parameter G



Bang-Bang control for wall following



Setpoint: distance from wall

Bang-Bang control with Hysteresis

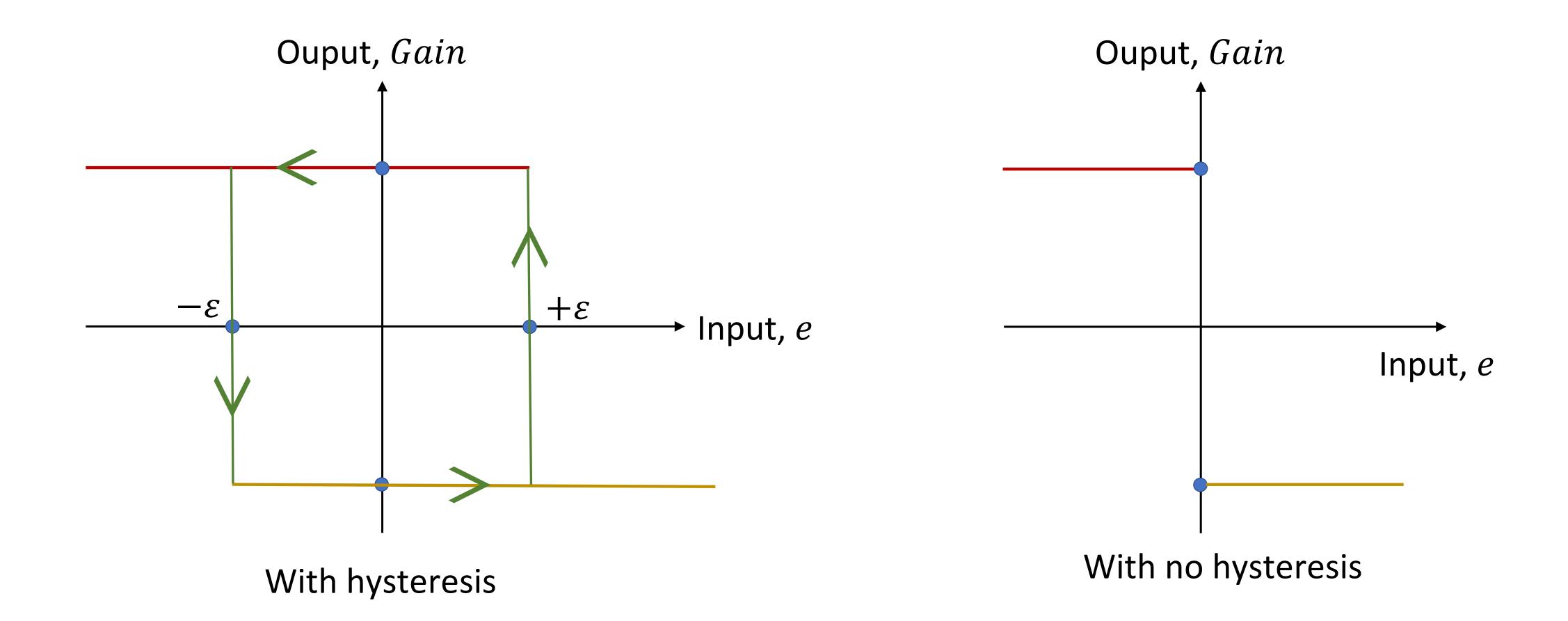
• To prevent chatter around e = 0,

$$e < -\varepsilon \implies u := +Gain (on)$$

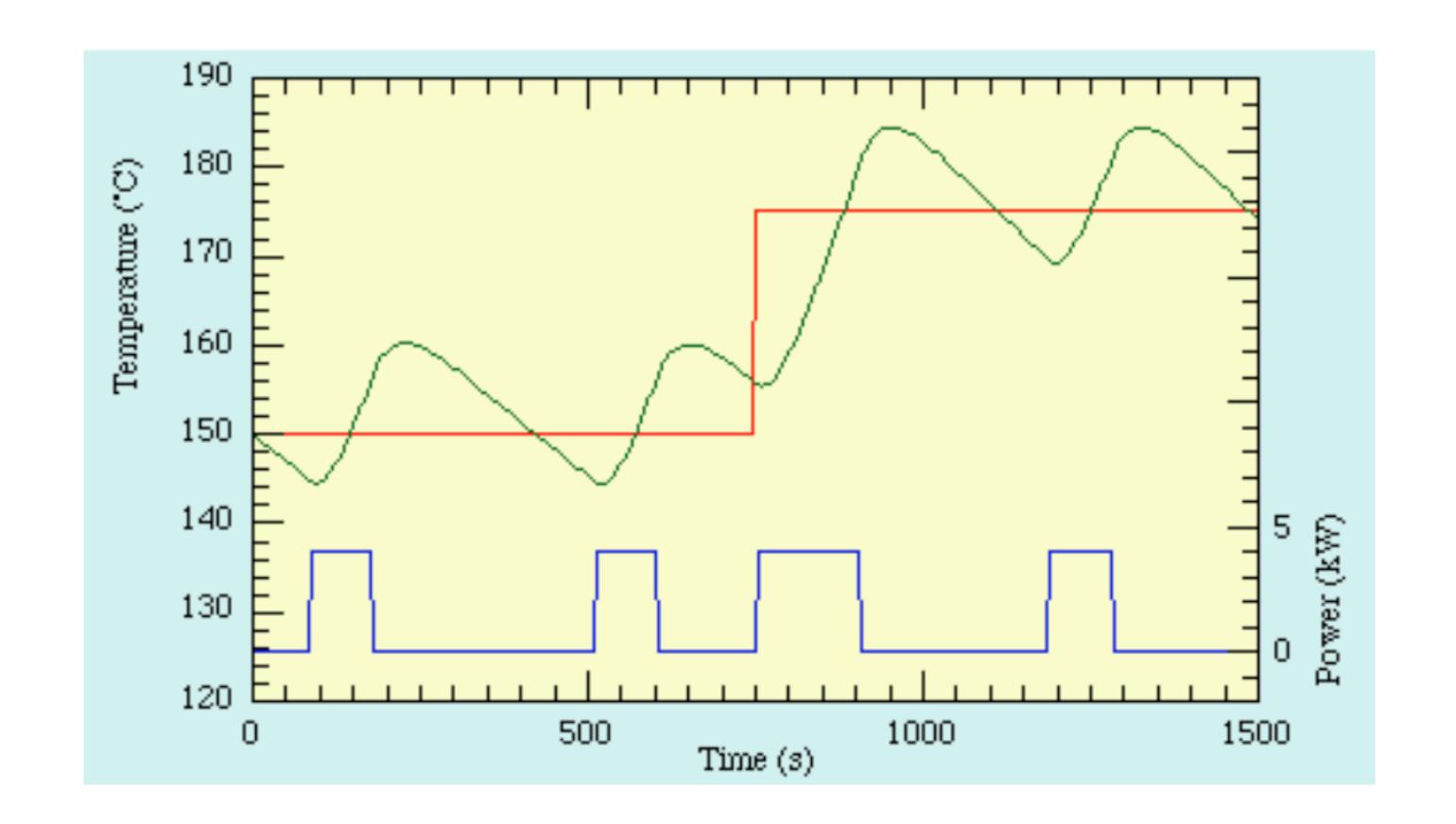
$$e > +\varepsilon \implies u := -Gain (off)$$

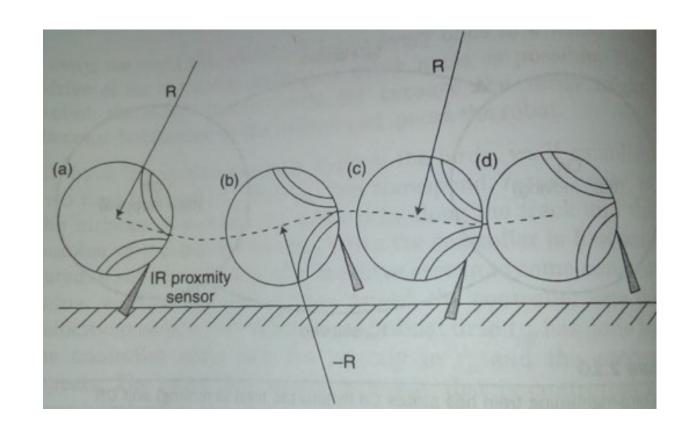
Bang-bang controls with hysteresis provide optimal controls in some cases, although they are often implemented just because of their simplicity or when binary behaviors are required

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Bang-Bang control at work: household thermostat





What to expect in wall-following?

Big oscillations around the desired state!

- Optimal for reaching the setpoint
- Not very good for staying near it

Proportional control (P)

• Push back, proportional to the error.

$$u = -ke + u_b$$

$$- set u_b so that $\dot{x} = F(x_{set}, u_b) = 0$$$

• For a linear system, we get exponential convergence.

$$x(t) = Ce^{-\alpha t} + x_{set}$$

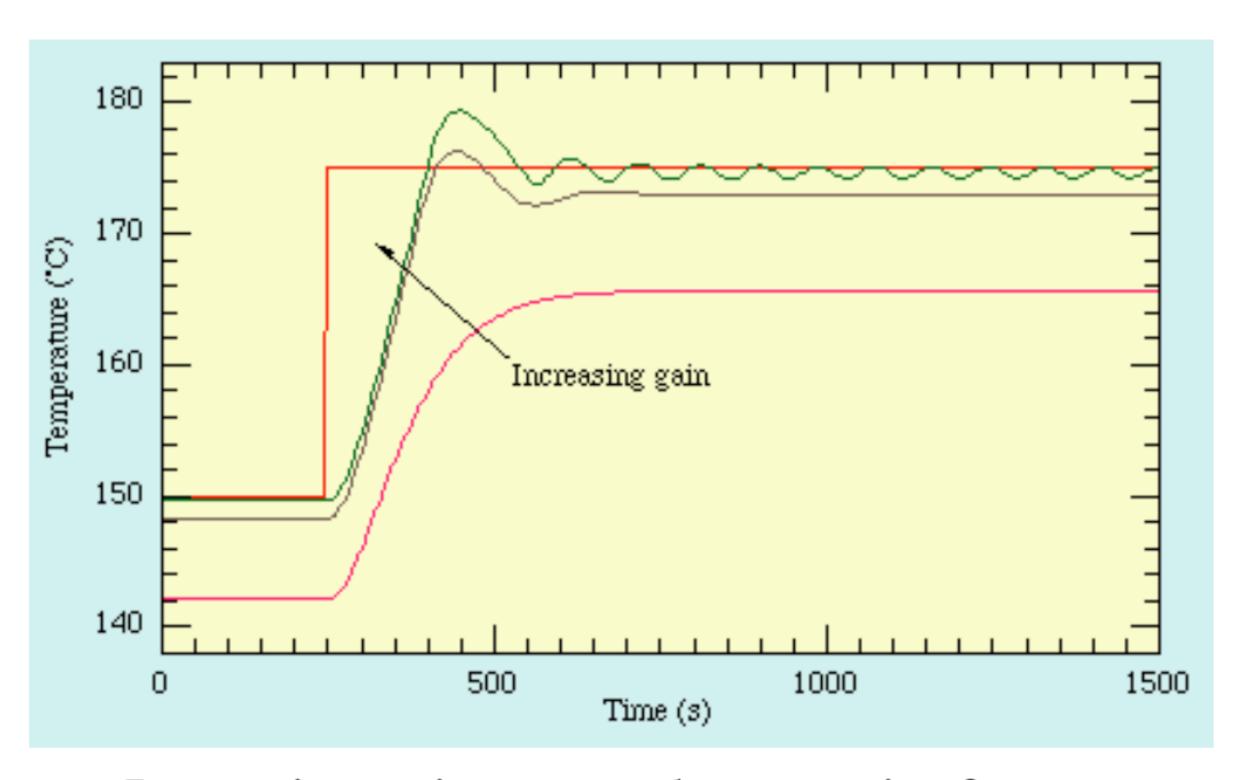
• The controller gain k determines how quickly the system responds to error.

- You want to drive your car at velocity v_{set} .
- You issue the motor command $u = pos_{accel}$
- You observe velocity v_{obs} .
- Define a first-order controller:

$$u = -k(v_{obs} - v_{set}) + u_b$$

k is the controller Gain

Proportional control for the thermostat



- Increasing gain approaches setpoint faster
- Can leads to overshoot, and even instability
- Steady-state offset

Steady offset

• Suppose we have continuing disturbances:

$$\dot{x} = F(x, u) + d$$

- The P-controller cannot stabilize at e = 0.
 - if u_b is defined so $F(x_{set}, u_b) = 0$
 - then $F(x_{set}, u_b) + d \neq 0$, so the system changes
- Must adapt u_b to different disturbances d.

And we don't know how to model such disturbances, i.e., how to include them in *F*

Adaptive control

- Sometimes one controller isn't enough.
- We need controllers at different time scales.

$$u = -k_P e + u_b$$

 $\dot{u}_b = -k_I e$ where $k_I << k_P$

• This can eliminate steady-state offset.

– Because the slower controller adapts u_b .

Proportional-Integral (PI) controller

• The adaptive controller $\dot{u}_b = -k_I$ means, by integrating to obtain a point value in time:

•
$$\boldsymbol{u}_b = -k_I \int_{t_0}^t \boldsymbol{e}(t) dt + \boldsymbol{u}_b$$

• Additively keep memory of all the errors so far within a certain time window $[t_0, t]$, ideally $t_0 = 0$

$$\mathbf{u}(t) = -k_P \mathbf{e}(t) - k_I \int_{t_0}^t \mathbf{e}(t) dt + \mathbf{u}_b$$

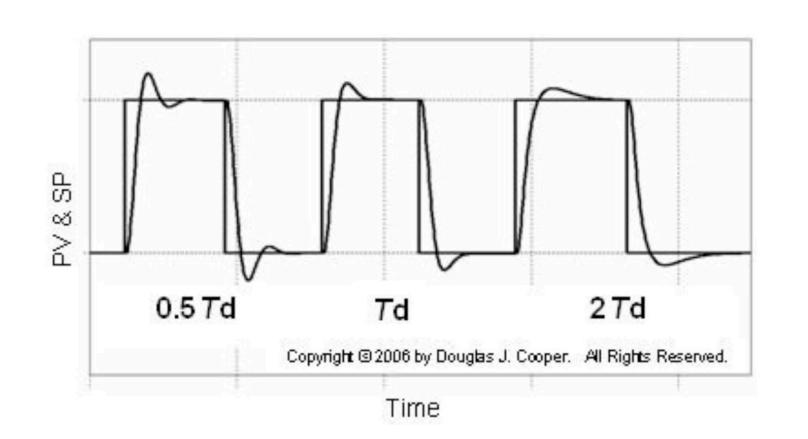
Proportional-Integral (PI) controller

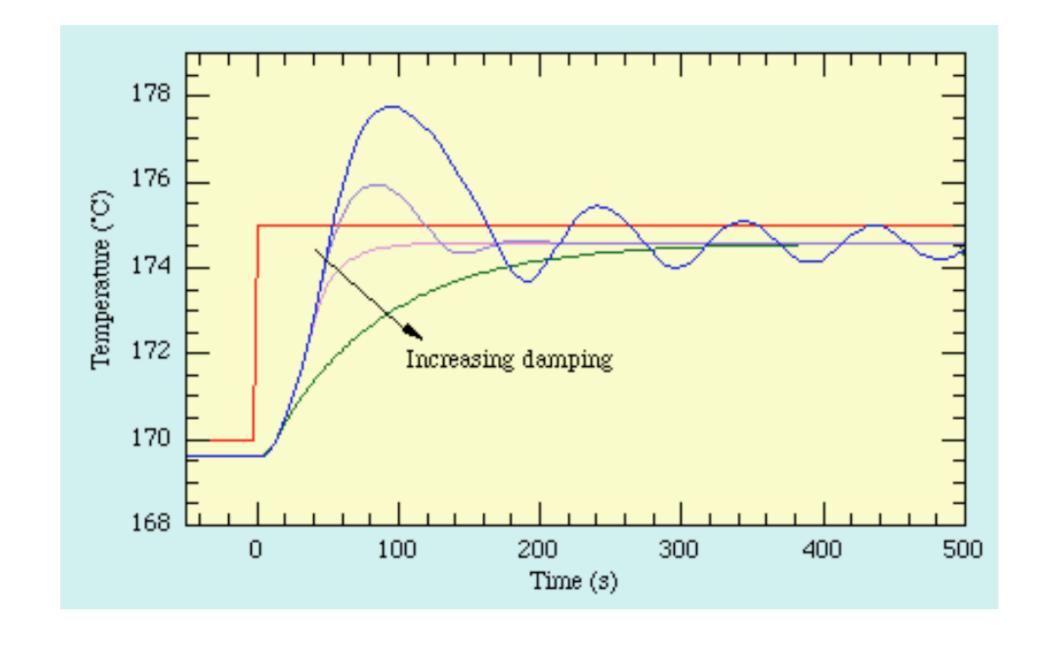
Derivative control

- Damping friction is a force opposing motion, proportional to velocity.
- Try to prevent overshoot by damping controller response.

$$u = -k_P e - k_D \dot{e}$$

• Estimating a derivative from measurements is fragile, and amplifies noise.





- Damping fights oscillation and overshoot
- But it's vulnerable to noise

Derivative \dot{e} must be numerically estimated from relative local measures close in time \rightarrow Subject to imprecisions and approximations, not very reliable in general

Different amounts of damping (without noise)

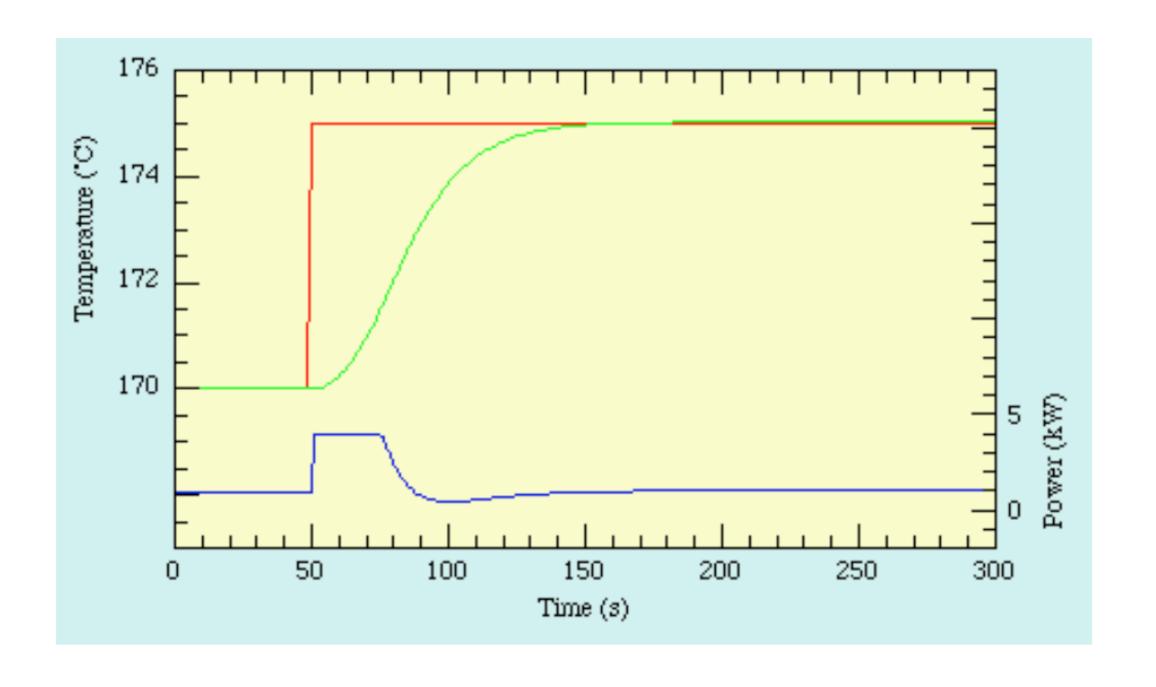
PID Control

• A weighted combination of Proportional, Integral, and Derivative terms.

$$u(t) = -k_P e(t) - k_I \int_0^t e \, dt - k_D \dot{e}(t)$$

• The PID controller is the workhorse of the control industry. Tuning is non-trivial.

To be continued ...



- But, good behavior depends on good tuning!