## **Chapter 1**

## Demo problem: Buckling of a clamped cylindrical shell under pressure loading

In this document, we discuss the solution of the buckling of a cylindrical shell using oomph-lib's Kirchhoff $\leftarrow$  LoveShell elements.

[No documentation yet: Here's the driver code.]

```
//LIC// This file forms part of oomph-lib, the object-oriented,
//LIC// multi-physics finite-element library, available
//LIC// at http://www.oomph-lib.org.
             Version 1.0; svn revision $LastChangedRevision$
//LIC//
//LIC// $LastChangedDate$
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//LIC// The authors may be contacted at oomph-lib@maths.man.ac.uk.
//LIC//
//Driver function for a simple test shell problem:
//Calculate the deformation of an elastic tube approximated
//using Kirchoff--Love shell theory
//Standard system includes
#include <iostream>
#include <fstream>
#include <typeinfo>
#include <algorithm>
#include <cstdio>
//Include files from the finite-element library
#include "generic.h"
#include "shell.h"
#include "meshes/rectangular_quadmesh.h"
using namespace std:
using namespace oomph;
```

```
/// Global variables that represent physical properties
namespace Global_Physical_Variables
 /// Prescribed position of control point
double Prescribed_y = 1.0;
 /// \ short Pointer to pressure load (stored in Data so it can
 /// become an unknown in the problem when displacement control is used
Data* Pext data pt;
 /// Perturbation pressure
double Pcos=1.0;
 /// \short Return a reference to the external pressure
 /// load on the elastic tube.
 double external_pressure()
  {return (*Pext_data_pt->value_pt(0))*pow(0.05,3)/12.0;}
 /// Load function, normal pressure loading
 void press_load(const Vector<double> &xi,
                  const Vector<double> &x,
                  const Vector<double> &N,
                  Vector<double>& load)
  //std::cout << N[0] << " " << N[1] << " " << N[2] << std::endl; //std::cout << xi[0] << " " << xi[1] << std::endl;
  for(unsigned i=0;i<3;i++)</pre>
    load[i] = (external_pressure() -
                Pcos*pow(0.05,3)/12.0*cos(2.0*xi[1]))*N[i];
   }
/// A 2D Mesh class. The tube wall is represented by two Lagrangian
/// coordinates that correspond to z and theta in cylindrical polars.
/// The required mesh is therefore a 2D mesh and is therefore inherited
/// from the generic RectangularQuadMesh
template <class ELEMENT>
class ShellMesh : public virtual RectangularQuadMesh<ELEMENT>,
                  public virtual SolidMesh
public:
 ///Constructor for the mesh
ShellMesh(const unsigned &nx, const unsigned &ny,
           const double &lx, const double &ly);
 /// \short In all elastic problems, the nodes must be assigned an undeformed,
 /// or reference, position, corresponding to the stress-free state
 \ensuremath{///} of the elastic body. This function assigns the undeformed position
 /// for the nodes on the elastic tube
void assign_undeformed_positions(GeomObject* const &undeformed_midplane_pt);
};
/// Mesh constructor
/// Argument list:
/// nx : number of elements in the axial direction /// ny : number of elements in the azimuthal direction
/// lx : length in the axial direction
/// ly : length in theta direction
template <class ELEMENT>
ShellMesh<ELEMENT>::ShellMesh(const unsigned &nx,
                                const unsigned &ny,
                                const double &lx,
                                const double &ly) :
 RectangularQuadMesh<ELEMENT>(nx,ny,lx,ly)
 //Find out how many nodes there are
 unsigned n_node = nnode();
```

```
//Now in this case it is the Lagrangian coordinates that we want to set,
 //so we have to loop over all nodes and set them to the Eulerian
 //coordinates that are set by the generic mesh generator
 for(unsigned i=0;i<n_node;i++)</pre>
   node_pt(i)->xi(0) = node_pt(i)->x(0);
node_pt(i)->xi(1) = node_pt(i)->x(1);
 //Assign gradients, etc for the Lagrangian coordinates of
 //hermite-type elements
 //Read out number of position dofs
 unsigned n_position_type = finite_element_pt(0) -> nnodal_position_type();
 //If this is greater than 1 set the slopes, which are the distances between //nodes. If the spacing were non-uniform, this part would be more difficult
 if(n_position_type > 1)
   double xstep = (this->Xmax - this->Xmin)/((this->Np-1)*this->Nx);
double ystep = (this->Ymax - this->Ymin)/((this->Np-1)*this->Ny);
   for(unsigned n=0;n<n_node;n++)</pre>
      //The factor 0.5 is because our reference element has length 2.0
     node_pt(n) \rightarrow xi_gen(1,0) = 0.5 \times xstep;
      node_pt(n) \rightarrow xi_gen(2,1) = 0.5*ystep;
}
/// Set the undeformed coordinates of the nodes
//=====
template <class ELEMENT>
void ShellMesh<ELEMENT>::assign_undeformed_positions(
 GeomObject* const &undeformed_midplane_pt)
{
 //Find out how many nodes there are
 unsigned n_node = nnode();
 //Loop over all the nodes
 for (unsigned n=0;n<n_node;n++)</pre>
   //Get the Lagrangian coordinates
   Vector<double> xi(2);
   xi[0] = node_pt(n) -> xi(0);
   xi[1] = node_pt(n) -> xi(1);
    //Assign memory for values of derivatives, etc
   Vector<double> R(3);
   DenseMatrix<double> a(2,3);
   RankThreeTensor<double> dadxi(2,2,3);
    //Get the geometrical information from the geometric object
   undeformed_midplane_pt->d2position(xi,R,a,dadxi);
    //Loop over coordinate directions
   for(unsigned i=0;i<3;i++)</pre>
    {
     //Set the position
     node_pt(n) \rightarrow x_gen(0,i) = R[i];
      //Set the derivative wrt Lagrangian coordinates
      //Note that we need to scale by the length of each element here!!
     \label{eq:node_pt} \mbox{node\_pt(n)->x\_gen(1,i) = 0.5*a(0,i)*((this->Xmax - this->Xmin)/this->Nx);}
     node_pt(n) ->x_gen(2,i) = 0.5*a(1,i)*((this->Ymax - this->Ymin)/this->Ny);
      //Set the mixed derivative
     //(symmetric so doesn't matter which one we use) node_pt(n)->x_gen(3,i) = 0.25*dadxi(0,1,i);
}
//Problem class to solve the deformation of an elastic tube
template<class ELEMENT>
class ShellProblem : public Problem
public:
 /// Constructor
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ShellProblem(const unsigned &nx, const unsigned &ny,
              const double &lx, const double &ly);
 /// Overload Access function for the mesh
ShellMesh<ELEMENT>* mesh_pt()
  {return dynamic_cast<ShellMesh<ELEMENT>*>(Problem::mesh_pt());}
 /// Actions after solve empty
 void actions_after_newton_solve() {}
 /// Actions before solve empty
void actions_before_newton_solve() {}
 //A self_test function
 void solve();
private:
 /// Pointer to GeomObject that specifies the undeformed midplane
GeomObject* Undeformed_midplane_pt;
 /// First trace node
Node* Trace_node_pt;
 /// Second trace node
Node* Trace_node2_pt;
template<class ELEMENT>
ShellProblem<ELEMENT>::ShellProblem(const unsigned &nx, const unsigned &
     ny,
                                    const double &lx, const double &ly)
 //Create the undeformed midplane object
Undeformed_midplane_pt = new EllipticalTube(1.0,1.0);
 //Now create the mesh
 Problem::mesh_pt() = new ShellMesh<ELEMENT>(nx,ny,lx,ly);
 //Set the undeformed positions in the mesh
mesh_pt()->assign_undeformed_positions(Undeformed_midplane_pt);
 //Reorder the elements, since I know what's best for them....
mesh pt()->element reorder();
 //Apply boundary conditions to the ends of the tube
 unsigned n_ends = mesh_pt()->nboundary_node(1);
 //Loop over the node
 for(unsigned i=0;i<n_ends;i++)</pre>
   //Pin in the axial direction (prevents rigid body motions)
   mesh_pt()->boundary_node_pt(1,i)->pin_position(2);
   mesh_pt()->boundary_node_pt(3,i)->pin_position(2);
   //Derived conditions
   mesh_pt()->boundary_node_pt(1,i)->pin_position(2,2);
   mesh_pt()->boundary_node_pt(3,i)->pin_position(2,2);
            -----CLAMPING CONDITIONS-----
   //----Pin positions in the transverse directions-----
   // Comment these out to get the ring case
   mesh_pt()->boundary_node_pt(1,i)->pin_position(0);
   mesh_pt()->boundary_node_pt(3,i)->pin_position(0);
   //Derived conditions
   mesh_pt()->boundary_node_pt(1,i)->pin_position(2,0);
   mesh_pt()->boundary_node_pt(3,i)->pin_position(2,0);
   mesh_pt()->boundary_node_pt(1,i)->pin_position(1);
   mesh_pt()->boundary_node_pt(3,i)->pin_position(1);
   //Derived conditions
   mesh_pt()->boundary_node_pt(1,i)->pin_position(2,1);
   mesh_pt()->boundary_node_pt(3,i)->pin_position(2,1);
   // Set the axial gradients of the transverse coordinates to be
   // zero --- need to be enforced for ring or tube buckling
   //Pin dx/dz and dy/dz
   mesh_pt()->boundary_node_pt(1,i)->pin_position(1,0);
   mesh_pt()->boundary_node_pt(1,i)->pin_position(1,1);
   mesh_pt()->boundary_node_pt(3,i)->pin_position(1,0);
   mesh_pt()->boundary_node_pt(3,i)->pin_position(1,1);
   //Derived conditions
```

```
mesh_pt()->boundary_node_pt(1,i)->pin_position(3,0);
  mesh_pt()->boundary_node_pt(1,i)->pin_position(3,1);
  mesh_pt()->boundary_node_pt(3,i)->pin_position(3,0);
  mesh_pt()->boundary_node_pt(3,i)->pin_position(3,1);
 //Now loop over the sides and apply symmetry conditions
 unsigned n_side = mesh_pt()->nboundary_node(0);
 for(unsigned i=0;i<n_side;i++)</pre>
  //At the side where theta is 0, pin in the \ensuremath{\mathbf{y}} direction
  mesh_pt()->boundary_node_pt(0,i)->pin_position(1);
   //Derived condition
  mesh_pt()->boundary_node_pt(0,i)->pin_position(1,1);
   //Pin dx/dtheta and dz/dtheta
   mesh_pt()->boundary_node_pt(0,i)->pin_position(2,0);
  mesh_pt()->boundary_node_pt(0,i)->pin_position(2,2);
   //Pin the mixed derivative
  mesh_pt()->boundary_node_pt(0,i)->pin_position(3,0);
  mesh_pt()->boundary_node_pt(0,i)->pin_position(3,2);
   //At the side when theta is 0.5\mathrm{pi} pin in the x direction
  mesh_pt()->boundary_node_pt(2,i)->pin_position(0);
   //Derived condition
   mesh_pt()->boundary_node_pt(2,i)->pin_position(1,0);
   //Pin dy/dtheta and dz/dtheta
   mesh_pt()->boundary_node_pt(2,i)->pin_position(2,1);
   mesh_pt()->boundary_node_pt(2,i)->pin_position(2,2);
   //Pin the mixed derivative
  mesh_pt()->boundary_node_pt(2,i)->pin_position(3,1);
  mesh_pt()->boundary_node_pt(2,i)->pin_position(3,2);
      //Set an initial kick to make sure that we hop onto the
      //non-axisymmetric branch
      if((i>1) && (i<n_side-1))
       mesh_pt() \rightarrow boundary_node_pt(0,i) \rightarrow x(0) += 0.05;
       mesh_pt()->boundary_node_pt(2,i)->x(1) -= 0.1;
// Setup displacement control
// //Setup displacement control
  //Fix the displacement at the mid-point of the tube in the "vertical"
   //(y) direction.
    //Set the displacement control element (located halfway along the tube)
// Disp_ctl_element_pt = dynamic_cast<ELEMENT*>(mesh_pt()->element_pt(3*Ny-1));
   //{\mbox{The midpoint}} of the tube is located exactly half-way along the element
// Vector<double> s(2); s[0] = 1.0; s[1] = 0.0; //s[1] = 0.5
   //{	t Fix} the displacement at this point in the y (1) direction
  Disp_ctl_element_pt->fix_displacement_for_displacement_control(s,1);
   //Set the pointer to the prescribed position
// Disp_ctl_element_pt->prescribed_position_pt() = &Prescribed_y;
// Choose element in which displacement control is applied: This
 // one is located about halfway along the tube -- remember that
 // we've renumbered the elements!
 unsigned nel_ctrl=0;
Vector<double> s_displ_control(2);
 // Even/odd number of elements in axial direction
if (nx%2==1)
  nel\_ctrl=unsigned(floor(0.5*double(nx))+1.0)*ny-1;
   s_displ_control[0]=0.0;
  s_displ_control[1]=1.0;
else
  nel_ctrl=unsigned(floor(0.5*double(nx))+1.0)*ny-1;
  s_displ_control[0]=-1.0;
  s_displ_control[1]=1.0;
 // Controlled element
 SolidFiniteElement* controlled_element_pt=
  dynamic_cast<ELEMENT*>(mesh_pt()->element_pt(nel_ctrl));
// Fix the displacement in the v (1) direction...
```

```
unsigned controlled_direction=1;
// Pointer to displacement control element
DisplacementControlElement* displ_control_el_pt;
// Build displacement control element
displ_control_el_pt=
 new DisplacementControlElement(controlled_element_pt,
                                 s_displ_control,
                                 controlled_direction,
                                 &Global_Physical_Variables::Prescribed_y
     );
// Doc control point
Vector<double> xi(2);
Vector<double> x(3);
controlled_element_pt->interpolated_xi(s_displ_control,xi);
controlled_element_pt->interpolated_x(s_displ_control,x);
std::cout << std::endl;
std::cout << "Controlled element: " << nel_ctrl << std::endl;</pre>
// The constructor of the DisplacementControlElement has created
// a new Data object whose one-and-only value contains the
// adjustable load: Use this Data object in the load function:
Global_Physical_Variables::Pext_data_pt=displ_control_el_pt->
 displacement control load pt();
// Add the displacement-control element to the mesh
mesh_pt()->add_element_pt(displ_control_el_pt);
// Complete build of shell elements
//Find number of shell elements in mesh
unsigned n element = nx*nv;
//Explicit pointer to first element in the mesh
ELEMENT* first_el_pt = dynamic_cast<ELEMENT*>(mesh_pt()->element_pt(0));
//Loop over the elements
for(unsigned e=0;e<n_element;e++)</pre>
  //Cast to a shell element
  ELEMENT *el_pt = dynamic_cast<ELEMENT*>(mesh_pt()->element_pt(e));
  //Set the load function
  el_pt->load_vector_fct_pt() = & Global_Physical_Variables::press_load
  //Set the undeformed surface
  el_pt->undeformed_midplane_pt() = Undeformed_midplane_pt;
  //The external pressure is external data for all elements
  el_pt->add_external_data(Global_Physical_Variables::Pext_data_pt)
  //Pre-compute the second derivatives wrt Lagrangian coordinates
  //for the first element only
  if(e==0)
   {
    el_pt->pre_compute_d2shape_lagrangian_at_knots();
  //{\rm Otherwise} set the values to be the same as those in the first element
  //this is OK because the Lagrangian mesh is uniform.
  else
   {
    el_pt->set_dshape_lagrangian_stored_from_element(first_el_pt);
   }
//Set pointers to two trace nodes, used for output
Trace_node_pt = mesh_pt()->finite_element_pt(2*ny-1)->node_pt(3);
Trace_node2_pt = mesh_pt()->finite_element_pt(ny)->node_pt(1);
// Do equation numbering
cout << std::endl;
cout << "# of dofs " << assign_eqn_numbers() << std::endl;</pre>
cout << std::endl;
```

```
}
// /Define the solve function, disp ctl and then continuation
template<class ELEMENT>
void ShellProblem<ELEMENT>::solve()
   //Increase the maximum number of Newton iterations.
   //Finding the first buckled solution requires a large(ish) number
    //of Newton steps -- shells are just a bit twitchy
   Max_newton_iterations = 40;
   Max_residuals=1.0e6;
   //Open an output trace file
   ofstream trace("trace.dat");
    //Gradually compress the tube by decreasing the value of the prescribed
    //position
    for (unsigned i=1; i<11; i++)</pre>
           Global_Physical_Variables::Prescribed_y -= 0.05;
            cout << std::endl << "Increasing displacement: Prescribed_y is "</pre>
                                << Global_Physical_Variables::Prescribed_y << std::endl;
            // Solve
            newton_solve();
            //Output the pressure (on the bending scale)
            trace << Global_Physical_Variables::external_pressure() / (pow</pre>
                        (0.05,3)/12.0)
                                    << " "
                                   //Position of first trace node
<< Trace_node_pt->x(0) << " " << Trace_node_pt->x(1) << " "
//Position of second trace node
<< Trace_node2_pt->x(0) << " " << Trace_node2_pt->x(1) << std::endl;</pre>
            // Reset perturbation
          Global_Physical_Variables::Pcos=0.0;
   //Close the trace file
   trace.close();
   //Output the tube shape in the most strongly collapsed configuration % \left( 1\right) =\left( 1\right) +\left( 1\right
   ofstream file("final_shape.dat");
   mesh_pt()->output(file,5);
   file.close();
}
/// Driver
int main()
   //Length of domain
   double L = 10.0;
   double L_phi=0.5*MathematicalConstants::Pi;
    //Set up the problem
   ShellProblem<StorableShapeSolidElement<DiagHermiteShellElement>
      problem(5,3,L,L_phi);
    //Solve the problem
   problem.solve();
```

## 1.1 PDF file

A pdf version of this document is available.