

Submitted by Hasan (Prospective Ph.D. student)

Previously, I submitted the truss hand calculation (FEM) for Reaction force, deflection, and stress in each element, **deformed and undeformed plot**. This is the corresponding code for the Hand calculation. (direct stiffness method).

Four Matlab files complete the whole function:

1. local_stiffness_matrix (local nodal stiffness matrix)
2. assembly (global stiffness matrix from local stiffness matrix)
3. stress_in_each_element (calculation of the stress in each element)
4. final_matrix_solving (getting nodal deflection and reaction force, deformed vs undeformed plot)

1. local_stiffness_matrix

```
%obtaining local stiffness matrix for each element
function x = local_stiffness_matrix(E,A,L,theta)
t=theta*pi/180; %converting theta degree to radian
c=cos(t); %t is in radian angle value
s=sin(t);
x=A*E/L * [c*c   c*s   -c*c   -c*s;
           c*s   s*s   -c*s   -s*s;
           -c*c  -c*s    c*c    c*s;
           -c*s  -s*s    c*s    s*s];
```

```

1      %obtaining local stiffness matrix for each element
2      function x = local_stiffness_matrix(E,A,L,theta)
3      -
4      -   t=theta*pi/180; %converting theta degree to radian
5      -   c=cos(t); %t is in radian angle value
6      -   s=sin(t);
7      -   x=A*E/L * [c*c   c*s   -c*c   -c*s;
8      -               c*s   s*s   -c*s   -s*s;
9      -               -c*c   -c*s   c*c    c*s;
10      -              -c*s   -s*s   c*s    s*s];
11

```

Figure 1: Local_stiffness_matrix code

2. assembly

%assembling global stiffness matrix

function y = assembly(K,k,i,j)

%i and j are the connecting node number of the element

%Here capital K value is 10*10 global stiffness matrix, small k(row,column) is the local stiffness corresponding position value

% every iteration, the global stiffness matrix will be updated

$K(2*i-1, 2*i-1) = K(2*i-1, 2*i-1) + k(1,1);$ %for $k1:i=3, j=1, K(2*3-1=5, 2*3-1=5)$ $Ux3, Ux3$

$K(2*i-1, 2*i) = K(2*i-1, 2*i) + k(1,2);$ %for $k1:i=3, j=1, K(2*3-1=5, 2*3=6)$ $Uy3, Ux3$

$K(2*i-1, 2*j-1) = K(2*i-1, 2*j-1) + k(1,3);$ %for $k1:i=3, j=1, K(2*3-1=5, 2*1-1=1)$ $Ux1, Ux3$

$K(2*i-1, 2*j) = K(2*i-1, 2*j) + k(1,4);$ %for $k1:i=3, j=1, K(2*3-1=5, 2*1=2)$ $Uy1, Ux3$

$K(2*i, 2*i-1) = K(2*i, 2*i-1) + k(2,1);$ %for $k1:i=3, j=1, K(2*3=6, 2*3-1=5)$ $Ux3, Uy3$

$K(2*i, 2*i) = K(2*i, 2*i) + k(2,2);$ %for $k1:i=3, j=1, K(2*3=6, 2*3=6)$ $Uy3, Uy3$

$K(2*i, 2*j-1) = K(2*i, 2*j-1) + k(2,3);$ %for $k1:i=3, j=1, K(2*3=6, 2*1-1=1)$ $Ux1, Uy3$

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K(2*i, 2*j) = K(2*i, 2*j) + k(2,4);%for k1:i=3,j=1,K(2*3= 6, 2*1 =2) Uy1,Uy3
K(2*j-1, 2*i-1) = K(2*j-1, 2*i-1) + k(3,1);%for k1:i=3,j=1,K(2*1-1=1, 2*3-1=5) Ux3,Ux1
K(2*j-1, 2*i) = K(2*j-1, 2*i) + k(3,2);%for k1:i=3,j=1,K(2*1-1=1, 2*3 =6) Uy3,Ux1
K(2*j-1, 2*j-1) = K(2*j-1, 2*j-1) + k(3,3);%for k1:i=3,j=1,K(2*1-1=1, 2*1-1=1) Ux1,Ux1
K(2*j-1, 2*j) = K(2*j-1, 2*j) + k(3,4);%for k1:i=3,j=1,K(2*1-1=1, 2*1 =2) Uy1,Ux1
K(2*j, 2*i-1) = K(2*j, 2*i-1) + k(4,1);%for k1:i=3,j=1,K(2*1= 2, 2*3-1=5) Ux3,Uy1
K(2*j, 2*i) = K(2*j, 2*i) + k(4,2);%for k1:i=3,j=1,K(2*1= 2, 2*3 =6) Uy3,Uy1
K(2*j, 2*j-1) = K(2*j, 2*j-1) + k(4,3);%for k1:i=3,j=1,K(2*1= 2, 2*1-1=1) Ux1,Uy1
K(2*j, 2*j) = K(2*j, 2*j) + k(4,4);%for k1:i=3,j=1,K(2*1= 2, 2*1 =2) Uy1,Uy1

y= K;

```

```

1 %assembling global stiffness matrix
2 function y = assembly(K,k,i,j)
3 %i and j are the connecting node number of the element
4 %Here capital K value is 10*10 global stiffness matrix, small k(row,column)is the local stiffness corresponding position
5 % every iteration, the global stiffness matrix will be updated
6 K(2*i-1, 2*i-1) = K(2*i-1, 2*i-1) + k(1,1);%for k1:i=3,j=1,K(2*3-1=5, 2*3-1=5) Ux3,Ux3
7 K(2*i-1, 2*i) = K(2*i-1, 2*i) + k(1,2);%for k1:i=3,j=1,K(2*3-1=5, 2*3 =6) Uy3,Ux3
8 K(2*i-1, 2*j-1) = K(2*i-1, 2*j-1) + k(1,3);%for k1:i=3,j=1,K(2*3-1=5, 2*1-1=1) Ux1,Ux3
9 K(2*i-1, 2*j) = K(2*i-1, 2*j) + k(1,4);%for k1:i=3,j=1,K(2*3-1=5, 2*1 =2) Uy1,Ux3
10 K(2*i, 2*i-1) = K(2*i, 2*i-1) + k(2,1);%for k1:i=3,j=1,K(2*3= 6, 2*3-1=5) Ux3,Uy3
11 K(2*i, 2*i) = K(2*i, 2*i) + k(2,2);%for k1:i=3,j=1,K(2*3= 6, 2*3 =6) Uy3,Uy3
12 K(2*i, 2*j-1) = K(2*i, 2*j-1) + k(2,3);%for k1:i=3,j=1,K(2*3= 6, 2*1-1=1) Ux1,Uy3
13 K(2*i, 2*j) = K(2*i, 2*j) + k(2,4);%for k1:i=3,j=1,K(2*3= 6, 2*1 =2) Uy1,Uy3
14 K(2*j-1, 2*i-1) = K(2*j-1, 2*i-1) + k(3,1);%for k1:i=3,j=1,K(2*1-1=1, 2*3-1=5) Ux3,Ux1
15 K(2*j-1, 2*i) = K(2*j-1, 2*i) + k(3,2);%for k1:i=3,j=1,K(2*1-1=1, 2*3 =6) Uy3,Ux1
16 K(2*j-1, 2*j-1) = K(2*j-1, 2*j-1) + k(3,3);%for k1:i=3,j=1,K(2*1-1=1, 2*1-1=1) Ux1,Ux1
17 K(2*j-1, 2*j) = K(2*j-1, 2*j) + k(3,4);%for k1:i=3,j=1,K(2*1-1=1, 2*1 =2) Uy1,Ux1
18 K(2*j, 2*i-1) = K(2*j, 2*i-1) + k(4,1);%for k1:i=3,j=1,K(2*1= 2, 2*3-1=5) Ux3,Uy1
19 K(2*j, 2*i) = K(2*j, 2*i) + k(4,2);%for k1:i=3,j=1,K(2*1= 2, 2*3 =6) Uy3,Uy1
20 K(2*j, 2*j-1) = K(2*j, 2*j-1) + k(4,3);%for k1:i=3,j=1,K(2*1= 2, 2*1-1=1) Ux1,Uy1
21 K(2*j, 2*j) = K(2*j, 2*j) + k(4,4);%for k1:i=3,j=1,K(2*1= 2, 2*1 =2) Uy1,Uy1
22 y= K;
23

```

Figure 2:assembly of global stiffness matrix code

3.Stress in each elements

```

function z= stress_in_each_element (E,L,theta,U)
%Here U is the global displacement vector
t=theta*pi/180; %converting theta degree to radian
c=cos(t); %t is in radian angle value
s=sin(t);
z = E/L*[-c -s c s]*U;

```

```

1 function z= stress_in_each_element (E,L,theta,U)
2     %Here U is the global displacement vector
3     t=theta*pi/180; %converting theta degree to radian
4     c=cos(t); %t is in radian angle value
5     s=sin(t);
6     z = E/L*[-c -s c s]*U;

```

Figure 3: Stress in each elements code

4. Final matrix solving

```

% solving 10 degree of freedom stress reaction force at the hinged point.
clear; clc;
% declaring all data, all in SI unit
A=.02; %cross sectional area of each element
E=100000000000; %Youngs Modulus for all element
L1=2;theta1=60;%length of the first element is 2 & angle is 60 degree
L2=2;theta2=120; %length of the second element is 2 & angle is 120 degree
L3=3.01;theta3=4.67;%length of the third element is 2 & angle is 4.67 degree
L4=2.83;theta4=45;%length of the fourth element is 2.83 & angle is 45 degree
L5=2;theta5=90; %length of the fifth element is 2 & angle is 90 degree

% local_stiffness_matrix(E,A,L,theta) from individual function.
k1= local_stiffness_matrix(E,A,L1,theta1);%stiffness matrix for element 1; node 3 to 1
k2= local_stiffness_matrix(E,A,L2,theta2);%stiffness matrix for element 2; node 4 to 1
k3= local_stiffness_matrix(E,A,L3,theta3);%stiffness matrix for element 3; node 1 to 2
k4= local_stiffness_matrix(E,A,L4,theta4);%stiffness matrix for element 4; node 4 to 2
k5= local_stiffness_matrix(E,A,L5,theta5);%stiffness matrix for element 5; node 5 to 2
fprintf('Local stiffness matrix')
k1
k2
k3
k4
k5
% ending of local stiffness matrix part

% assembly process of global stiffness matrix K= assembly (K,k,i,j)
% Here uppercase K is global and lowercase k is local stiffness matrix
K=zeros(10,10); % 5 nodes * 2 = 10 degree of freedom. so 10*10 zeros matrix
K=assembly(K,k1,3,1);%connect the stiffness between node i=3 to j=1 or element 1
K=assembly(K,k2,4,1);%connect the stiffness between node i=4 to j=1 or element 2
K=assembly(K,k3,1,2);%connect the stiffness between node i=1 to j=2 or element 3

```

```

K=assembly(K,k4,4,2);%connect the stiffness between node i=4 to j=2 or element 4
K=assembly(K,k5,5,2);%connect the stiffness between node i=5 to j=2 or element 5
fprintf('Global stiffness matrix K')
K
%fprintf('10*10 global stiffness matrix is %.4f\n', K);
%eliminating 5,6,7,8,9,10 row and columns and getting
%reduced_global_stiffness matrix(M)
M=[K(1:4);K(2,1:4);K(3,1:4);K(4,1:4)];
fprintf('Boundary Condition')
f=[0;0;1000;0];%boundary conditions
u=M\f; %backslash operator used for gauss elimination technique, here u is deflection value of
node 1 and 2 (Ux1,Uy1,Ux2,uy2)
%10 deflection value of the nodes
fprintf('deflection matrix')
U=[u(1:4); 0;0;0;0;0;0;0;0;0;0];
%now for reaction force (R)of each point in x and y
%direction,R1x=R1y=R2x=R2y=0; only three hinged point have 6 reaction force.
fprintf('Reaction force')
R=K*U
%stress in each element (sigma1,-----,sigma5)
u1=[U(5);U(6);U(1);U(2)];%U3x=5,U3y=6,U1x=1,U1y=2 are the connecting node deflection
u2=[U(7);U(8);U(1);U(2)];%U4x=7,U4y=8,U1x=1,U1y=2 are the connecting node deflection
u3=[U(1);U(2);U(3);U(4)];%U1x=1,U1y=2,U2x=3,U2y=4 are the connecting node deflection
u4=[U(7);U(8);U(3);U(4)];%U4x=7,U4y=8,U2x=3,U2y=4 are the connecting node deflection
u5=[U(9);U(10);U(3);U(4)];%U5x=9,U5y=10,U2x=3,U2y=4 are the connecting node deflection
fprintf('Stress in element 1')
sigma1=stress_in_each_element(E,L1,theta1,u1)
fprintf('Stress in element 2')
sigma2=stress_in_each_element(E,L2,theta2,u2)
fprintf('Stress in element 3')
sigma3=stress_in_each_element(E,L3,theta3,u3)
fprintf('Stress in element 4')
sigma4=stress_in_each_element(E,L4,theta4,u4)
fprintf('Stress in element 5')
sigma5=stress_in_each_element(E,L5,theta5,u5)
%----- Plot drawing-----
%Plot structure
f1=figure();
Co_ordinate =[0 0; 3 0.268; -1 -1.732; 1 -1.732; 3 -1.732]; %node coordinate. node 1 is in center
point
Connection_point = [3 1; 4 1; 1 2; 4 2; 5 2]; %element connecting node
df = [1 2; 3 4; 5 6; 7 8; 9 10];%each node deflection
serial.1=Ux1;2=Uy1;3=Ux2;4=Uy2;5=Ux3;6=Uy3;7=Ux4;8=Uy4;9=Ux5;10=Uy5
NN = size(Co_ordinate,1); %no of nodes
NE = size(Connection_point,1);%no of elements
NCOORD = zeros(size(Co_ordinate));%deformed co-ordinate generation through zero matrix
scale = 100;
for n =1:NN
    NCOORD(n,1) = Co_ordinate(n,1) +scale*U(df(n,1));
    NCOORD(n,2) = Co_ordinate(n,2) +scale*U(df(n,2));
end

for k =1:NE

```

```

i=Connection_point(k,1);%first starting point of local node
j=Connection_point(k,2);%ending point of local node
x=[Co_ordinate(i,1) Co_ordinate(j,1)];
y=[Co_ordinate(i,2) Co_ordinate(j,2)];
xlim([-1.5 4]);
ylim([-2 1]);
plot(x,y,'k-');
hold on
ux=[NCOORD(i,1) NCOORD(j,1)];
uy=[NCOORD(i,2) NCOORD(j,2)];
xlim([-1.5 4]);
ylim([-2 1]);
plot(ux,uy,'r--');
hold on
end

```

Results

Local stiffness matrix

k1 =

```

1.0e+08 *

    2.5000    4.3301   -2.5000   -4.3301
    4.3301    7.5000   -4.3301   -7.5000
   -2.5000   -4.3301    2.5000    4.3301
   -4.3301   -7.5000    4.3301    7.5000

```

k2 =

```

1.0e+08 *

    2.5000   -4.3301   -2.5000    4.3301
   -4.3301    7.5000    4.3301   -7.5000
   -2.5000    4.3301    2.5000   -4.3301
    4.3301   -7.5000   -4.3301    7.5000

```

k3 =

```

1.0e+08 *

    6.6005    0.5392   -6.6005   -0.5392
    0.5392    0.0440   -0.5392   -0.0440
   -6.6005   -0.5392    6.6005    0.5392

```

-0.5392	-0.0440	0.5392	0.0440
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k4 =

1.0e+08 *

3.5336	3.5336	-3.5336	-3.5336
3.5336	3.5336	-3.5336	-3.5336
-3.5336	-3.5336	3.5336	3.5336
-3.5336	-3.5336	3.5336	3.5336

k5 =

1.0e+09 *

0.0000	0.0000	-0.0000	-0.0000
0.0000	1.0000	-0.0000	-1.0000
-0.0000	-0.0000	0.0000	0.0000
-0.0000	-1.0000	0.0000	1.0000

Global stiffness matrix K

K =

1.0e+09 *

Columns 1 through 7

1.1600	0.0539	-0.6600	-0.0539	-0.2500	-0.4330	-0.2500
0.0539	1.5044	-0.0539	-0.0044	-0.4330	-0.7500	0.4330
-0.6600	-0.0539	1.0134	0.4073	0	0	-0.3534
-0.0539	-0.0044	0.4073	1.3578	0	0	-0.3534
-0.2500	-0.4330	0	0	0.2500	0.4330	0
-0.4330	-0.7500	0	0	0.4330	0.7500	0
-0.2500	0.4330	-0.3534	-0.3534	0	0	0.6034
0.4330	-0.7500	-0.3534	-0.3534	0	0	-0.0797
0	0	-0.0000	-0.0000	0	0	0
0	0	-0.0000	-1.0000	0	0	0

Columns 8 through 10

0.4330	0	0
-0.7500	0	0
-0.3534	-0.0000	-0.0000
-0.3534	-0.0000	-1.0000
0	0	0
0	0	0
-0.0797	0	0
1.1034	0	0
0	0.0000	0.0000
0	0.0000	1.0000

Boundary Condition

f =

0
0
1000
0

deflection matrix

U =

1.0e-05 *

0.1042
0.0028
0.1877
-0.0521
0
0
0
0
0
0
0

Reaction force

R =

1.0e+03 *

-0.0000
-0.0000
1.0000 (This is not correct. This must be zero)
0.0000
-0.2728
-0.4726
-0.7272
-0.0489
0.0000
0.5214

Stress in element 1

sigma1 =

2.7285e+04

Stress in element 2

sigma2 =

-2.4827e+04

Stress in element 3

sigma3 =

2.6143e+04

Stress in element 4

sigma4 =

3.3862e+04

Stress in element 5

sigma5 =

-2.6072e+04

Deformed vs undeformed plot:

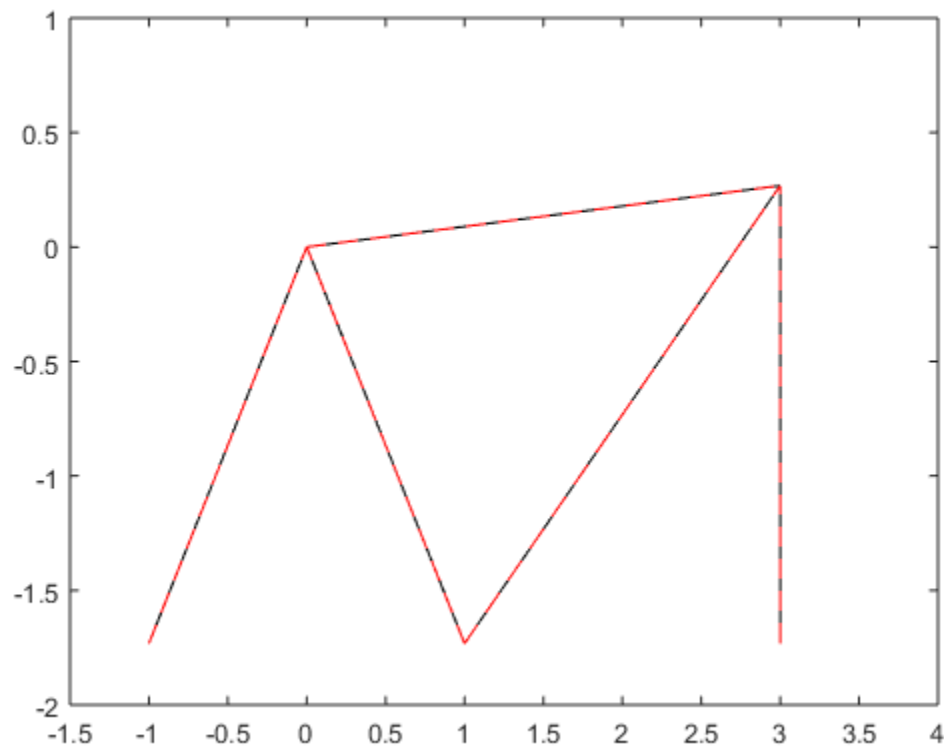


Figure 4: Main plot [Published with MATLAB® R2015a](#)

In all subsequent figures,

RED dotted line=deformed shape

Black straight line= undeformed shape

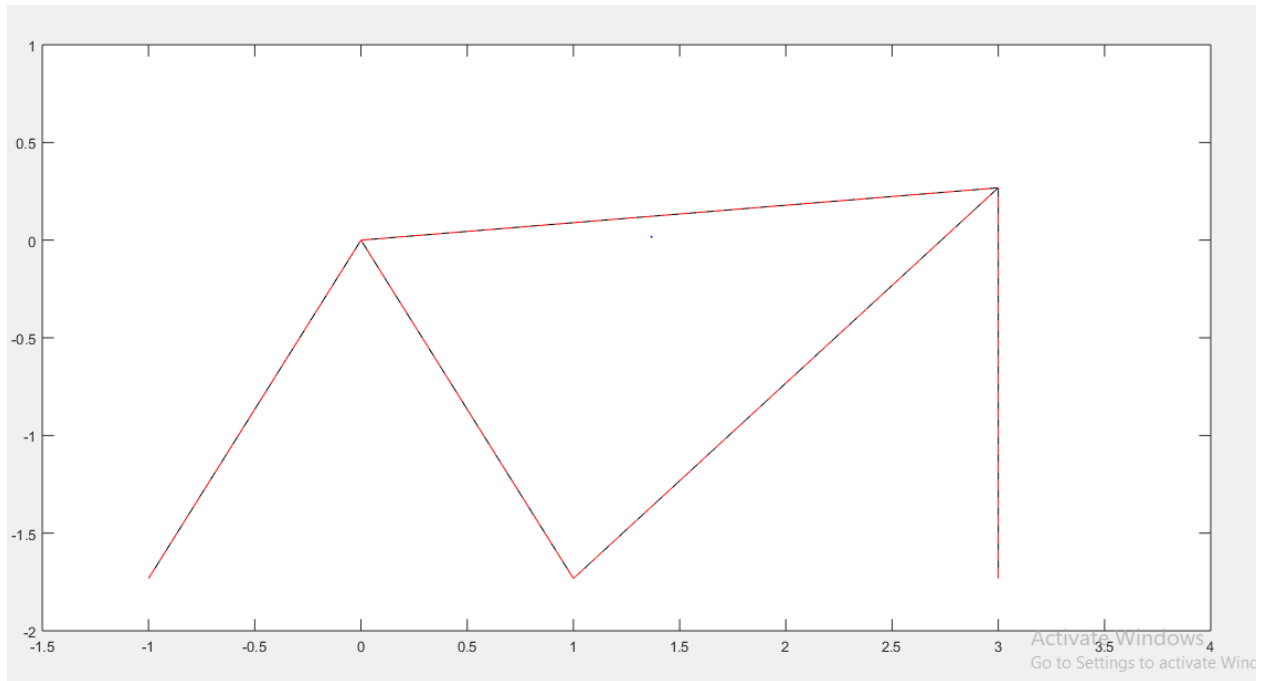


Figure 5: Deformed vs undeformed shape(1:1 ratio)

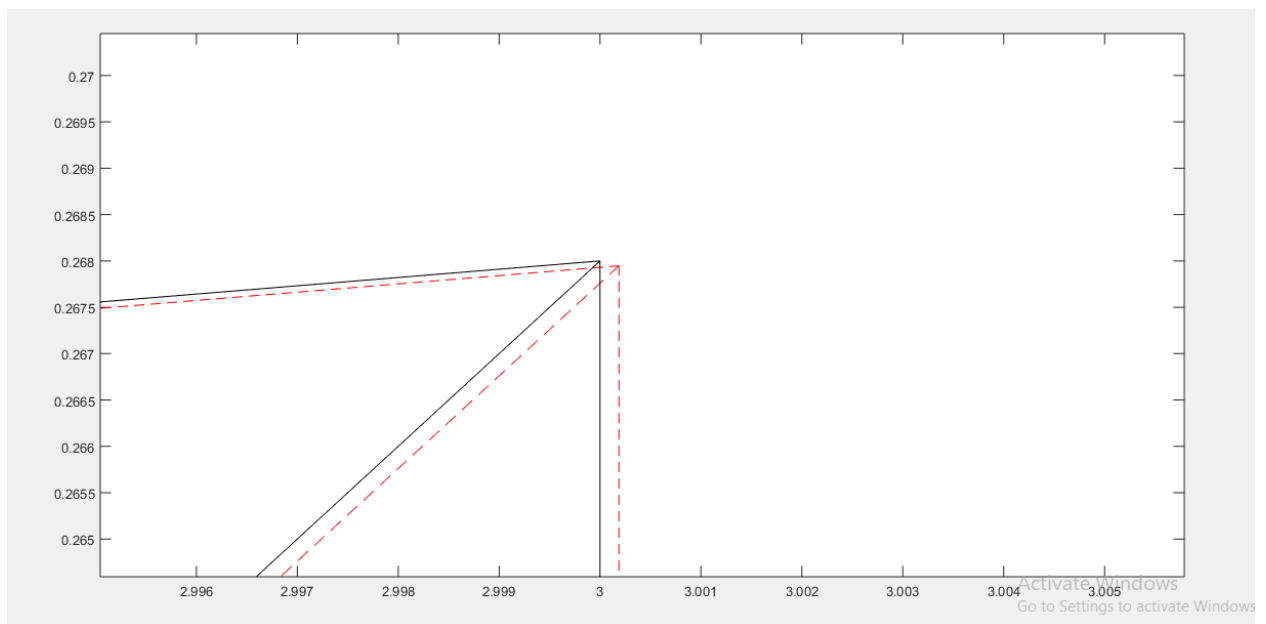


Figure 6: Node 2 deformed vs undeformed shape(external only horizontal load point)

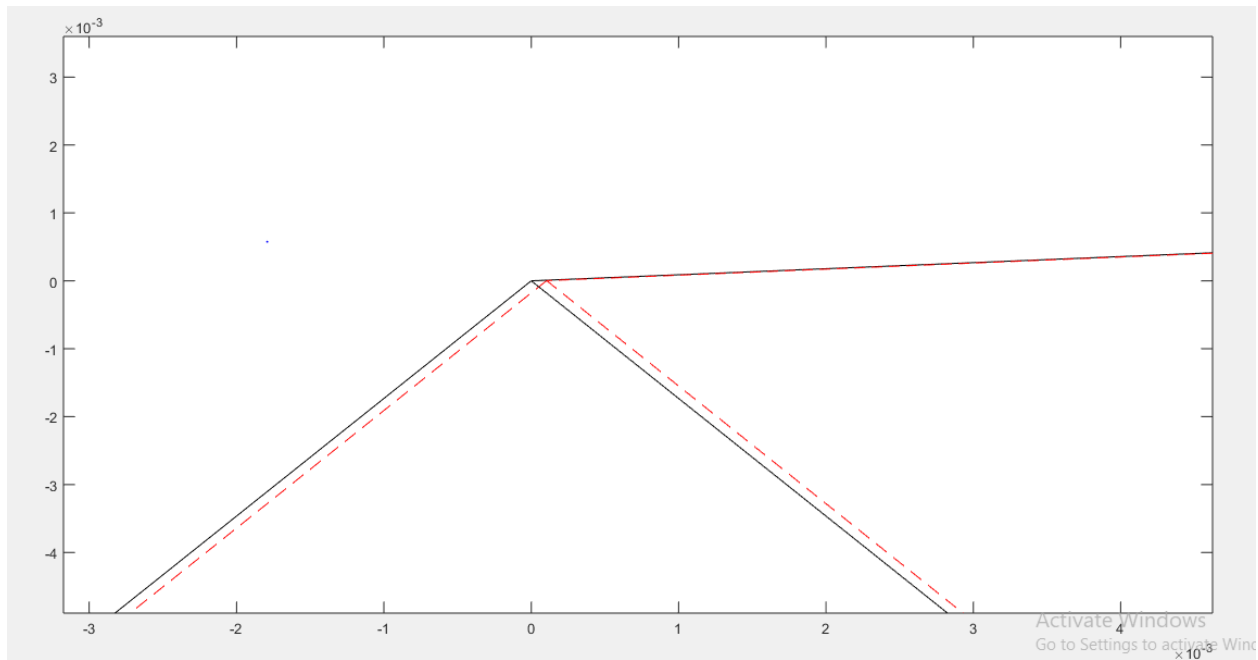


Figure 7: Node 1 deformed vs undeformed shape

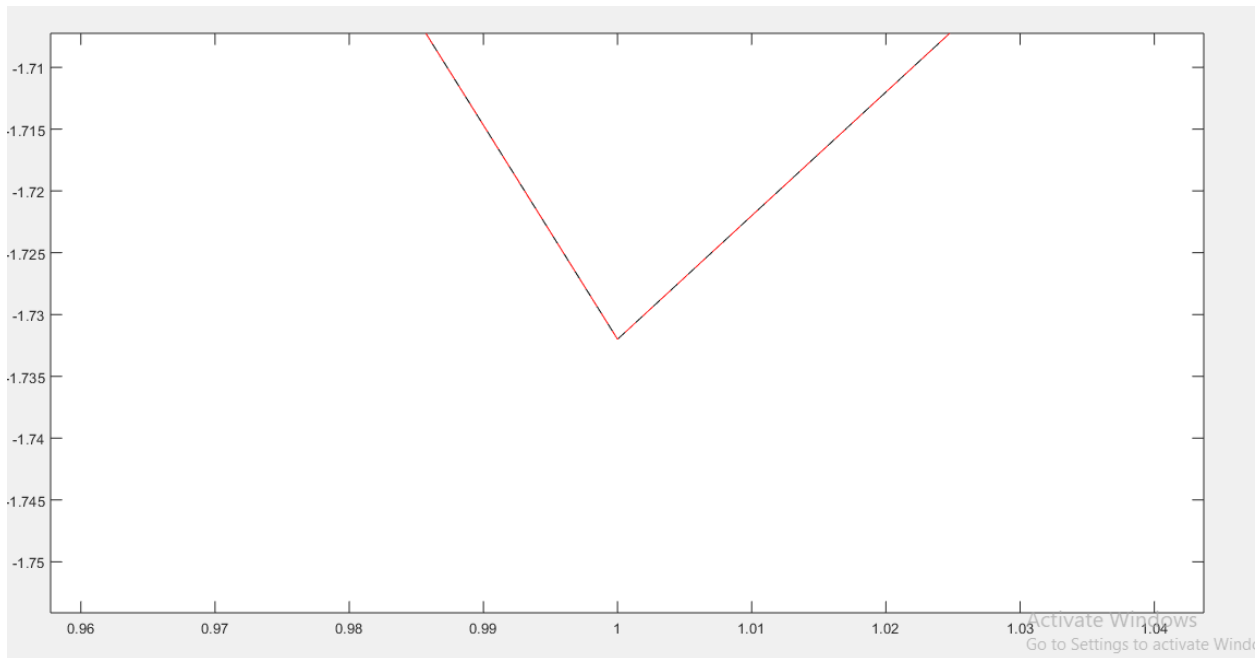


Figure 8: Node 4 deformed vs undeformed shape(hinged point)

Equilibrium Checking of the Entire System

- There is a by hand calculation error for reaction force which is also found in MATLAB code and output.
- Hand calculated global stiffness matrix first $4 \times 4 = 16$ value is almost same as code output.
- MATLAB by default took more values after the point value. So, most of the value is slightly different from hand calculation.
- Undeformed vs Undeformed plot follows the theoretical zero value in the hinged position.