

# Signals and Systems

## Lecture 11: System Identification

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# Outline

## 1 Frequency Domain System Identification

- Introduction
- Choice of signals inputs

## 2 Identifying a System based on its Impulse Response

- Impulse response, no noise
- Impulse response, with noise

### 3 Identification Using Sinusoidal Inputs

- Experimental procedure
- Closed-loop system identification
- Identifying the transfer function
  - Method
  - Example
- Weighted least squares

# Introduction

In this lecture apply tools learned to **identify an unknown system**.

**This process is known as system identification.** The key idea:

- 1 by **applying a known input** to a system
- 2 and observing the system's output,

⇒ a model of the system can be estimated.

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- **Grey-box:** we have a first-principles model, but with unknown parameters. Data is used to identify these parameters. This is covered in the IDSC lecture *System Modeling*.
- **White-box:** we have a first-principles model, with no unknown parameters, or where parameters can be measured directly (mass, for example); no data is required in this case.

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- 2 Next, **estimate the system's transfer function of the system**, based on the frequency response estimate.

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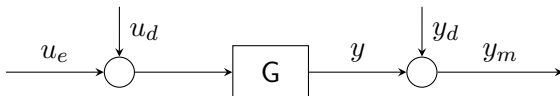
⇒ We focus on the black-box identification of causal, linear time-invariant systems.

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# Frequency Domain System Identification

We will consider the following system identification problem:



Where  $G$  is a causal, stable LTI system, and

- $u_e$  is a **known input** that we use **to excite the system**
- $u_d$  is an **unknown process noise**, assumed to be **white**
- $y_d$  is an **unknown measurement noise**, assumed to be **white**
- $y_m$ , given by  $y_m = Gu_e + y_d + Gu_d$ , is a measurement of the **system's output**, which is corrupted by process noise and measurement noise

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# Input choice for frequency domain system identification

Based on the above problem structure, the task is now to design:

- 1 a set of inputs  $u_e$ , such that by analyzing the measured output  $y_m$
- 2 and the known input  $u_e$ ,

⇒ the frequency response of the system  $H(\Omega)$  can be determined accurately, despite inherent process and measurement noise.



We will see later in the lecture how to estimate the system's transfer function based on its estimated frequency response.

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**Question:** What kind of input signal can be used for identification?

We now introduce two possible inputs:

- a unit impulse,
- and a sinusoid.



# Identifying a System based on its Impulse Response



In Lecture 2, we saw that

an LTI system is fully characterized by its impulse response.

## Conclusion

By measuring its impulse response, the system can be identified.

In the absence of process and measurement noise, we have that

$$y_m = G u_e .$$

We then have a simple way of determining freq. resp.  $H(\Omega)$ :

① Let  $\{u_e[n]\} = \{\delta[n]\}$

② Then  $\{y_m[n]\} = \{h[n]\}$  and  $H(\Omega) = \sum_{n=0}^{\infty} y_m[n] e^{-j\Omega n}$



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### In practice

since most systems have an infinite impulse response

- $\Rightarrow$  impossible to measure it entirely,
- $\Rightarrow$  we collect  $N$  pieces of data, where  $N$  is large enough such that the impulse response has decayed significantly ( = so that  $y_m[n]$  is small enough for  $n \geq N$ ).

We then take the DFT:

$$Y_m[k] = \sum_{n=0}^{N-1} y_m[n] e^{-j \frac{2\pi k}{N} n}.$$

At the discrete frequency  $\Omega_k = 2\pi k/N$  where  $k = 0, 1, \dots, N-1$ , ( $N$  DFT coefficients).

The frequency response estimate  $\hat{H}(\Omega_k)$  is defined as:

$$\hat{H}(\Omega_k) := Y_m[k] = H(\Omega_k) - \underbrace{\sum_{n=N}^{\infty} h[n] e^{-j\Omega_k n}}_{H_N(\Omega_k)},$$

where  $H_N(\Omega_k)$  is the 'estimation' error introduced by not measuring the entire duration of the impulse response.

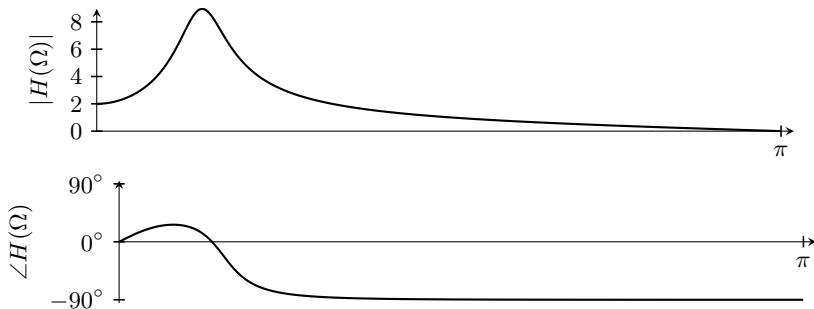
**Remark:** Note that  $H_N(\Omega) \rightarrow 0$  as  $N \rightarrow \infty$  since  $G$  is stable.

## Example : Estimating frequency response, based on impulse response, without noise

Let the system be given by the transfer function

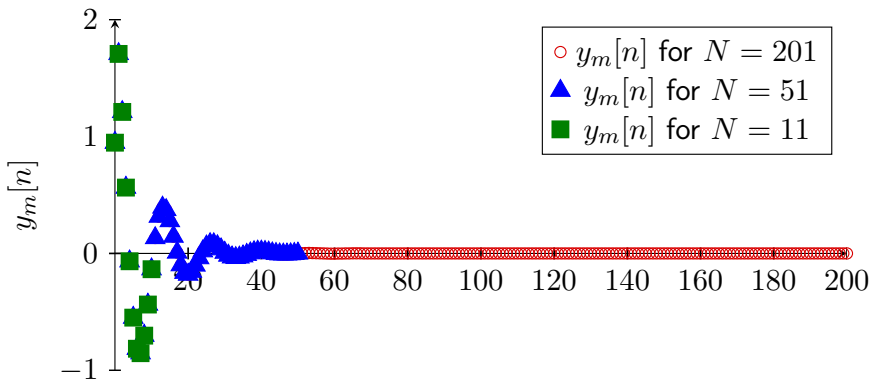
$$H(z) = \frac{0.9474 + 0.2105z^{-1} - 0.7368z^{-2}}{1 - 1.579z^{-1} + 0.7895z^{-2}},$$

which has the following frequency response:

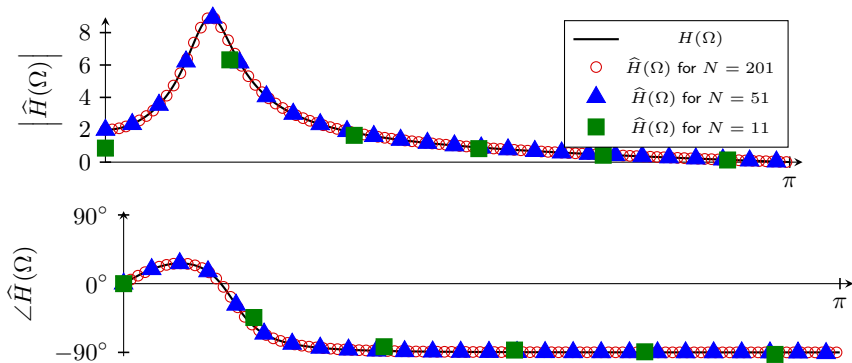




We apply the input  $u_e[n] = \delta[n]$  in the absence of noise, and record the system's output for  $N = 11$ ,  $N = 51$  and  $N = 201$  samples:



Take DFT of these measurements to estimate system's frequency response  $\hat{H}(\Omega)$ :



## Conclusion:

- 1 a larger  $N$  gives a higher frequency resolution, since we are dividing the interval  $-\pi$  to  $\pi$  into  $N$  equally spaced frequency intervals.
- 2 We also note that as  $N$  increases, estimation error decreases.

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We study effects of **measurement noise**  $y_d$  on above estimation procedure.

where  $E(y_d[n]) = 0$  and  $E(y_d[n]y_d[m]) = \sigma_y^2\delta[n - m]$ .

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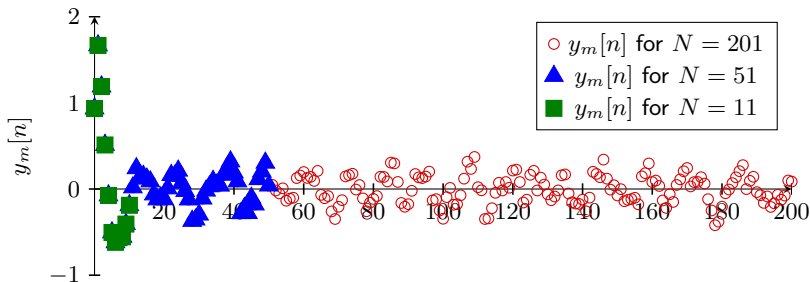
We now demonstrate the effect of zero-mean process noise  $u_d$  and measurement noise  $y_d$  with  $\sigma_y^2 = \sigma_u^2 = \frac{1}{3}$ .

Using the identical system, given by

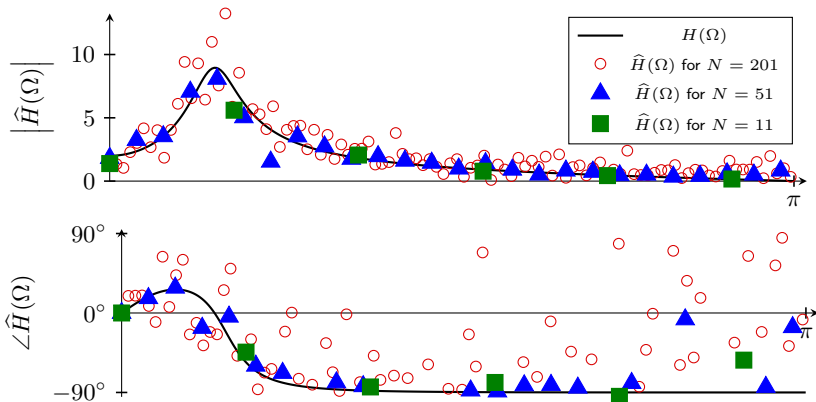
$$H(z) = \frac{0.9474 + 0.2105z^{-1} - 0.7368z^{-2}}{1 - 1.579z^{-1} + 0.7895z^{-2}},$$

we reapply the input  $u_e[n] = \delta[n]$ .

We record the system's output for  $N = 11$ ,  $N = 51$  and  $N = 201$  samples:



We then take the DFT of these measurements in order to estimate the frequency response  $\hat{H}(\Omega)$  of the system:



**Conclusion:** Here we see that the process noise and measurement noise cause large estimation errors for large  $N$ .





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## Identification Using Sinusoidal Inputs

**Goal:** To identify the frequency response of a system from its input-output behavior.

**An alternative method :** is to apply sinusoidal inputs at different frequencies .





$$y_e[n] = H(\Omega_l) u_e[n] - w[n],$$

We now take the DFT of the sequences introduced above for  $n \geq N_T$ . Let

$$Y_e[l] = \sum_{n=N_T}^{N_T+N-1} y_e[n] e^{-j \frac{2\pi}{N} l n},$$

$$U_e[l] = \sum_{n=N_T}^{N_T+N-1} u_e[n] e^{-j \frac{2\pi}{N} l n} = N,$$

$$W[l] = \sum_{n=N_T}^{N_T+N-1} w[n] e^{-j \frac{2\pi}{N} l n},$$

then  $Y_e[l] = H(\Omega_l) U_e[l] - W[l]$ . **Remark:**  $l$  is the index for the DT frequency



We can thus estimate the system's frequency response as follows. Let

$$Y_m[l] = \sum_{n=N_T}^{N_T+N-1} y_m[n] e^{-j \frac{2\pi}{N} l n}$$
$$Y_d[l] = \sum_{n=N_T}^{N_T+N-1} y_d[n] e^{-j \frac{2\pi}{N} l n},$$

then, since  $y_m = y_e + y_d$ ,

$$Y_m[l] = Y_e[l] + Y_d[l] = H(\Omega_l) U_e[l] - W[l] + Y_d[l]$$

**Frequency response estimate:**  $\hat{H}(\Omega_l)$

$$\hat{H}(\Omega_l) := \frac{Y_m[l]}{U_e[l]} = H(\Omega_l) - \frac{W[l]}{N} + \frac{Y_d[l]}{N}.$$

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We therefore have

1

$$\mathbb{E} \left( \hat{H}(\Omega_l) - H(\Omega_l) \right) = -\frac{W[l]}{N},$$

which approaches zero as  $N_T \rightarrow \infty$  or  $N \rightarrow \infty$ . Therefore, as  $N \rightarrow \infty$ , we have an unbiased estimate.

2

$$\mathbb{E} \left( \left| \hat{H}(\Omega_l) - H(\Omega_l) \right|^2 \right) = \frac{W^2[l]}{N^2} + \frac{\sigma_y^2}{N}.$$

The mean-squared error goes to 0. A similar result holds when  $u_d$  is included.

**Conclusion:** A sinusoidal input at different frequencies is a very effective input for system identification.



## Experimental procedure: summary

- Choose  $N_T$  large enough to let the transient die down (first  $N_T$  samples of the output to discard),

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- Set  $u_e[n] = A \cos(\Omega_l n)$  for  $n = 0, \dots, N_T + N - 1$ , where  $A$  scales the input.
  - Choosing a large  $A$  increases the signal to noise ratio;
  - however, if  $A$  is chosen too large, nonlinearities such as saturation may be excited, thus negatively affecting the frequency response estimate.
- Calculate

- The frequency response estimate at the frequency  $\Omega_l$  is then given by

- Repeat this for the frequencies of interest by picking the appropriate integer  $l$ .

## Example: Estimating frequency response using sinusoidal inputs, with noise

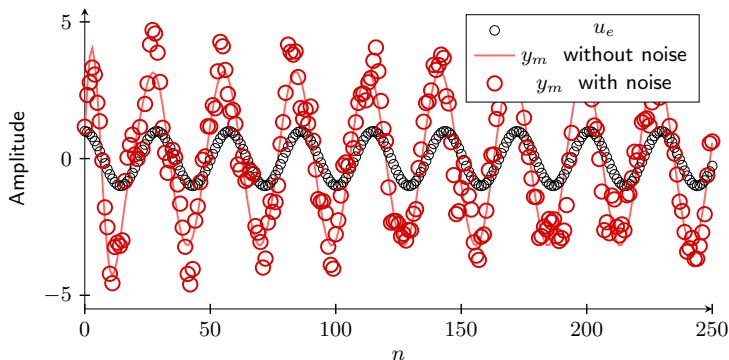
We now revisit the previous example using the identical system, given by

$$H(z) = \frac{0.9474 + 0.2105z^{-1} - 0.7368z^{-2}}{1 - 1.579z^{-1} + 0.7895z^{-2}},$$

and where the system is subject to both the process noise  $u_d$  and the measurement noise  $y_d$ .

- 1 Compared to the previous example, we **increase the noise** such that the standard deviation is increased by a **factor of 5**, resulting in  $\sigma_y^2 = \sigma_u^2 = \frac{25}{3}$ .
- 2 We perform system identification by applying sinusoidal inputs at  $N$  different frequencies, by choosing  $l = 0, 1, \dots, (N - 1)/2$ .

The recorded input  $u_e$  and output  $y_m$  for  $N_T = 50$ ,  $N = 201$ ,  $A = 1$ , and  $l = 7$  are as follows:

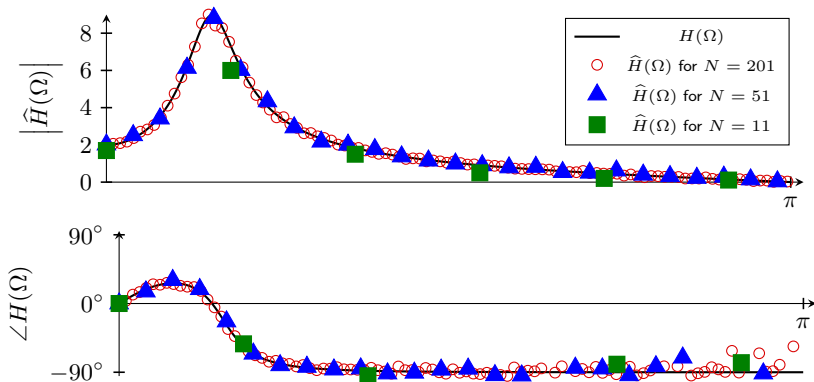


## Observations

1. Ideal output of system is: shifted and scaled version of input sinusoid.
2. Process noise and measurement noise corrupt the signals and their effects are noticeable.
3. By picking  $N_T = 50$ , we let the transient decay.



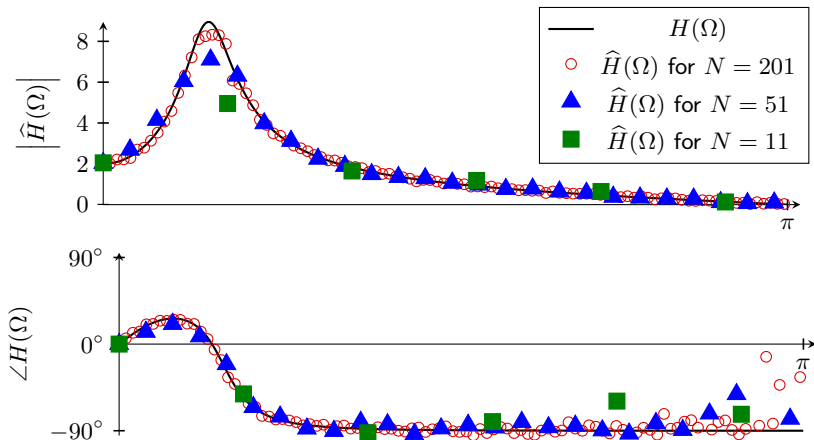
The estimate of the frequency response is as follows:



**Observations:** This method performs very well in the presence of noise. The above plot highlights the two main benefits of picking a larger  $N$ :

- 1 the effect of noise is further reduced, since more samples are averaged,
- 2 a larger  $N$  results in a finer frequency resolution, allowing more frequencies to be tested.

We now demonstrate what happens if we do not wait for the transient to decay, i.e. we pick  $N_T = 0$ .



We see that, as long as  $N$  is much larger than the transient decay time, the effect of the transient is small. However, for small  $N$  the transient significantly affects the estimate of the frequency response.

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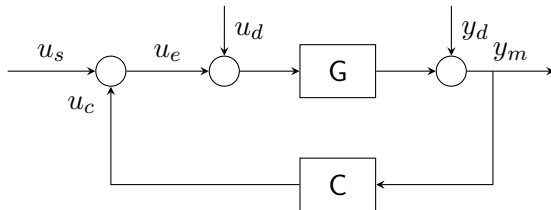
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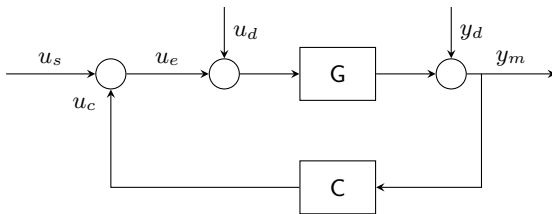
## Closed-loop system identification

Often, the system to be identified is part of a closed-loop system.

This is the case, for example, if the open-loop system is unstable.  
 $\Rightarrow$  In this case, an appropriate, stabilizing controller  $C$  is required before performing the identification.

Consider the following stable closed-loop system (where  $G$  can be unstable)





Let  $u_e = u_c + u_s$ . Then,  $y_m = Gu_e + Gu_d + y_d$ .

### Discussion :

- if  $u_e$  can be measured, the open loop method described in the previous section can be used to identify the frequency response of the plant.
- If  $G$  and  $C$  have no poles at  $z = e^{j\Omega_l}$ , and under some mild assumptions on  $u_d$  and  $y_d$ , it can be shown that the bias and mean-squared error of the estimate  $\hat{H}(\Omega_l) := \frac{Y_m[l]}{U_e[l]}$ , both approach 0 as  $N \rightarrow \infty$ .

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## Identifying the Transfer Function: Method

After estimating the frequency response at a number of distinct frequencies  $\Omega_l$ , we can identify the transfer function of the system.

Given the **two design parameters**:

- $A$  (number of  $a_k$  coefficients, although  $a_0$  is fixed at 1)
- and  $B$  (number of  $b_k$  coefficients),

**the goal is to identify the unknown parameters:**  $a_k$  and  $b_k$  of the system's transfer function, given by

$$H(z) = \frac{\sum_{k=0}^{B-1} b_k z^{-k}}{1 + \sum_{k=1}^{A-1} a_k z^{-k}},$$

with frequency response

$$H(\Omega) = \frac{\sum_{k=0}^{B-1} b_k e^{-j\Omega k}}{1 + \sum_{k=1}^{A-1} a_k e^{-j\Omega k}}.$$

### As a result of the identification experiments

we obtained estimates  $\hat{H}(\Omega_l)$  of  $H(\Omega)$  at  $L$  frequencies  $\Omega_l$  for  $l = 0, 1, \dots, L - 1$ .

Setting  $\hat{H}(\Omega_l) = H(\Omega_l)$  at all measurement frequencies, we obtain the system of equations

$$\begin{aligned} \left(1 + a_1 e^{-j\Omega_l} + \dots + a_{A-1} e^{-j(A-1)\Omega_l}\right) \hat{H}(\Omega_l) = \\ = b_0 + b_1 e^{-j\Omega_l} + \dots + b_{B-1} e^{-j(B-1)\Omega_l} \quad \text{for } l = 0, 1, \dots, L-1. \end{aligned}$$



These equations must hold for both real and imaginary components.

We therefore let :

- $\hat{H}(\Omega_l) = R_l e^{j\phi_l}$  and,
- using  $e^{j\theta} = \cos \theta + j \sin(\theta)$ ,

rewrite the system of equations for  $l = 0, 1, \dots, L - 1$  as

$$\begin{aligned} R_l \cos(\phi_l) + a_1 R_l \cos(\phi_l - \Omega_l) + \dots + a_{A-1} R_l \cos(\phi_l - (A-1)\Omega_l) \\ = b_0 + b_1 \cos(\Omega_l) + \dots + b_{B-1} \cos((B-1)\Omega_l) \end{aligned} \quad \text{Real}$$

$$\begin{aligned} R_l \sin(\phi_l) + a_1 R_l \sin(\phi_l - \Omega_l) + \dots + a_{A-1} R_l \sin(\phi_l - (A-1)\Omega_l) \\ = -b_1 \sin(\Omega_l) - \dots - b_{B-1} \sin((B-1)\Omega_l) \end{aligned} \quad \text{Imaginary}$$

This system of equations can be converted to the least squares problem of minimizing

$$(F\Theta - G)^T(F\Theta - G),$$

where

$$\Theta = [a_1 \quad a_2 \quad \dots \quad a_{A-1} \quad b_0 \quad b_1 \quad \dots \quad b_{B-1}]^T$$

contains the unknown parameters and where

- $\Theta$ : size  $(A + B - 1)$  vector, *unknown*.
- $F$ : size  $(2L) \times (A + B - 1)$  regression matrix, *known*.
- $G$ : size  $(2L)$  output measurement vector, *known*.

The least-squares solution

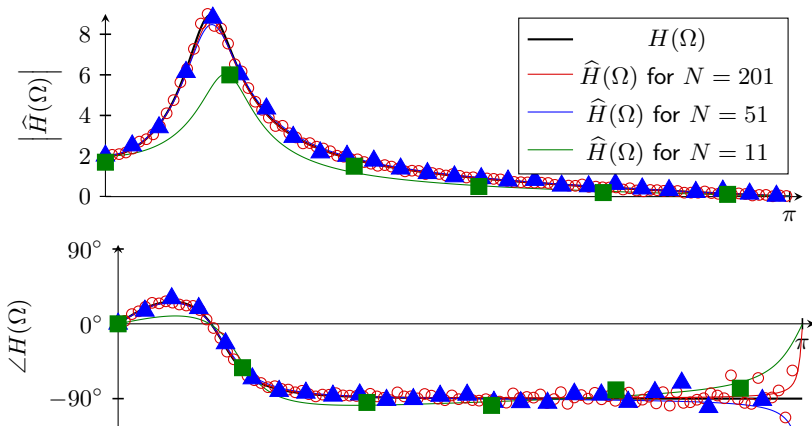
$$\Theta^* = (F^T F)^{-1} F^T G$$

yields the estimated transfer function coefficients  $\Theta^*$ .





The resulting frequency responses are the following:



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- **Weighted least squares**

A useful variant for system identification is weighted least squares:  $F\Theta = G$  is replaced by  $WF\Theta = WG$ , where

$$W = \begin{bmatrix} w_0 & & & & \\ & w_0 & & & \\ & & w_1 & & \\ & & & w_1 & \\ & & & & \ddots \\ & & & & & w_L \end{bmatrix}$$

is a  $(2L \times 2L)$  square, diagonal matrix. Its elements  $w_l$  are weights which capture the confidence in the measurements. We obtain the solution using the substitutions  $\overline{F} = WF$ ,  $\overline{G} = WG$ :

$$\begin{aligned}\Theta^* &= (\overline{F}^\top \overline{F})^{-1} \overline{F}^\top \overline{G} \\ &= (F^\top W^\top W F)^{-1} F^\top W^\top W G.\end{aligned}$$

Higher weights result in a better fit at the corresponding frequencies.  
For example, it is beneficial to use low weights at frequencies where a system rolls off. There, the measured amplitudes become small compared to the noise and the measurements are less reliable.