

UCL Mechanical Engineering 2020/2021

MECH0013 Final Assessment

NCWT3

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1 Question 1

1.1 i

Straight section analysis:

Equilibrium conditions:

$$\sum F_x : R_{Ax} = 0 \quad (1.1)$$

$$\sum F_y : R_{Ay} + R_B + P = W \quad (1.2)$$

$$\sum M_A : -M_A + M_C + 4WL = 2R_B L + 3PL \quad (1.3)$$

Using Macaulay's method:

$$M = -M_A + R_{Ay}x + R_B \langle x - 2L \rangle + P \langle x - 3L \rangle \quad (1.4)$$

Slope:

$$\theta = \frac{1}{EI} \int (M) dx \quad (1.5)$$

$$\theta = \frac{1}{EI} \int (-M_A + R_{Ay}x + R_B \langle x - 2L \rangle + P \langle x - 3L \rangle) dx \quad (1.6)$$

$$\theta = \frac{1}{EI} \left[-M_A x + \frac{R_{Ay}x^2}{2} + \frac{R_B \langle x - 2L \rangle^2}{2} + \frac{P \langle x - 3L \rangle^2}{2} \right] + \theta_0 \quad (1.7)$$

Deflection:

$$y = \int (\theta) dx \quad (1.8)$$

$$y = \int \left(\frac{1}{EI} \left[-M_A x + \frac{R_{Ay}x^2}{2} + \frac{R_B \langle x - 2L \rangle^2}{2} + \frac{P \langle x - 3L \rangle^2}{2} \right] + \theta_0 \right) dx \quad (1.9)$$

$$y = \frac{1}{EI} \left[-\frac{M_A x^2}{2} + \frac{R_{Ay}x^3}{6} + \frac{R_B \langle x - 2L \rangle^3}{6} + \frac{P \langle x - 3L \rangle^3}{6} \right] + \theta_0 x + y_0 \quad (1.10)$$

Curved section analysis:

$$M(\theta) = WR(1 - \cos \theta) + H_0 R \sin \theta + M_0 \quad (1.11)$$

$$\frac{\partial M}{\partial M_0} = 1 \quad (1.12)$$

$$\varphi_A = \int_0^L \left(\frac{M}{EI} \frac{\partial M}{\partial M_0} \right) dx = 0 \quad (1.13)$$

Converting to polar ($dx = R d\theta$):

$$\varphi_A = \frac{1}{EI} \int_0^{\frac{\pi}{2}} \left((WR(1 - \cos \theta) + H_0 R \sin \theta + M_0) (1) (R) \right) d\theta \quad (1.14)$$

M_0 and H_0 represent dummy loads and can be neglected:

$$\varphi_A = \frac{1}{EI} \int_0^{\frac{\pi}{2}} (WR^2 (1 - \cos \theta)) d\theta \quad (1.15)$$

$$\varphi_A = \frac{1}{EI} \left[WR^2 (\theta - \sin \theta) \right]_0^{\frac{\pi}{2}} \quad (1.16)$$

$$\varphi_A = \frac{WR^2 \left(\frac{\pi}{2} - 1 \right)}{EI} \quad (1.17)$$

Boundary conditions:

$$x = 0, y = 0 \therefore y_0 = 0 \quad (1.18)$$

$$x = 0, \theta = 0 \therefore \theta_0 = 0 \quad (1.19)$$

$$x = 2L, y = 0 \quad (1.20)$$

$$x = 4l, \theta = \frac{WR^2 \left(\frac{\pi}{2} - 1 \right)}{EI} \quad (1.21)$$

From 1.20:

$$0 = \frac{1}{EI} \left[-\frac{M_A (2L)^2}{2} + \frac{R_{Ay} (2L)^3}{6} + \frac{R_B < 2L - 2L >^3}{6} \right] \quad (1.22)$$

$$0 = \frac{1}{EI} \left[-2M_A L^2 + \frac{4R_{Ay} L^3}{3} + 0 \right] \quad (1.23)$$

$$0 = -2M_A L^2 + \frac{4R_{Ay} L^3}{3} \quad (1.24)$$

$$M_A = \frac{2R_{Ay} L}{3} \quad (1.25)$$

From 1.21:

$$\frac{WR^2 \left(\frac{\pi}{2} - 1 \right)}{EI} = \frac{1}{EI} \left[-M_A (4L) + \frac{R_{Ay} (4L)^2}{2} + \frac{R_B < 4L - 2L >^2}{2} + \frac{P < 4L - 3L >^2}{2} \right] \quad (1.26)$$

$$WR^2 \left(\frac{\pi}{2} - 1 \right) = -4M_A L + 8R_{Ay} L^2 + 2R_B L^2 + \frac{PL^2}{2} \quad (1.27)$$

Substituting 1.25:

$$WR^2 \left(\frac{\pi}{2} - 1 \right) = -\frac{8R_{Ay} L^2}{3} + 8R_{Ay} L^2 + 2R_B L^2 + \frac{PL^2}{2} \quad (1.28)$$

$$WR^2 \left(\frac{\pi}{2} - 1 \right) = \frac{16R_{Ay} L^2}{3} + 2R_B L^2 + \frac{PL^2}{2} \quad (1.29)$$

From 1.2:

$$R_B = W - P - R_{Ay} \quad (1.30)$$

Substituting 1.30:

$$WR^2 \left(\frac{\pi}{2} - 1 \right) = \frac{16R_{Ay} L^2}{3} + 2(W - P - R_A) L^2 + \frac{PL^2}{2} \quad (1.31)$$

$$WR^2 \left(\frac{\pi}{2} - 1 \right) = \frac{16R_{Ay} L^2}{3} + 2WL^2 - 2PL^2 - 2R_A L^2 + \frac{PL^2}{2} \quad (1.32)$$

$$WR^2 \left(\frac{\pi}{2} - 1 \right) = \frac{10R_{Ay} L^2}{3} + 2WL^2 - \frac{3PL^2}{2} \quad (1.33)$$

$$R_{Ay} = \frac{3}{4} \left(P - \frac{WR}{L} - 2W \right) \quad (1.34)$$

Substituting 1.34:

$$WR^2 \left(\frac{\pi}{2} - 1 \right) = \frac{10L^2}{3} \left(\frac{3}{4} \left(P - \frac{WR}{L} - 2W \right) \right) + 2WL^2 - \frac{3PL^2}{2} \quad (1.35)$$

$$WR^2 \left(\frac{\pi}{2} - 1 \right) = \frac{5PL^2}{2} - \frac{5WRL}{2} - 5WL^2 + 2WL^2 - \frac{3PL^2}{2} \quad (1.36)$$

$$WR^2 \left(\frac{\pi}{2} - 1 \right) = PL^2 - 3WL^2 - \frac{5WRL}{2} \quad (1.37)$$

Rearranging for P :

$$PL^2 = 3WL^2 + \frac{5WRL}{2} + WR^2 \left(\frac{\pi}{2} - 1 \right) \quad (1.38)$$

$$P = 3W + \frac{5WR}{2L} + \frac{WR^2}{L^2} \left(\frac{\pi}{2} - 1 \right) \quad (1.39)$$

$$P = 3(42.67) + \frac{5(42.67)(0.3)}{2(0.35)} + \frac{(42.67)(0.3)^2}{(0.35)^2} \left(\frac{\pi}{2} - 1 \right) \quad (1.40)$$

$$P = 237.34 \text{ N} \quad (1.41)$$

1.2 ii

2 Question 2

2.1 ii

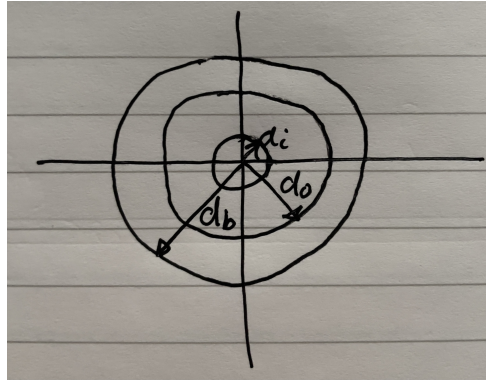


Figure 1: Sketch of cylinder arrangement.

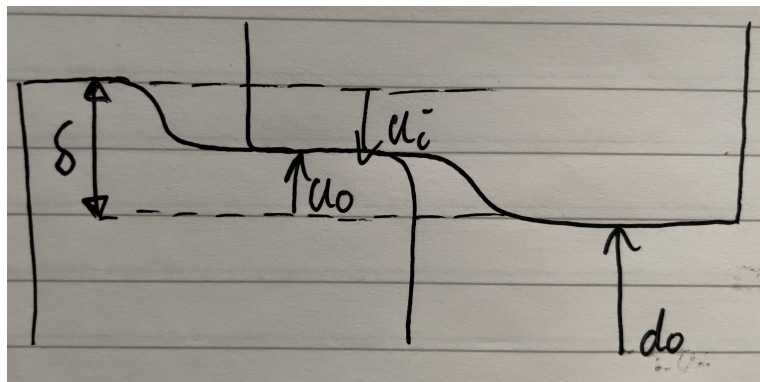


Figure 2: Diagram to show interference.

Hence:

$$\delta = u_o - u_i \quad (2.1)$$

Finding u_i . Let us start with the general equation for u :

$$u = \frac{r}{E} (\sigma_\theta - v\sigma_r) \quad (2.2)$$

$$\rightarrow u_i = \frac{d_0}{E_i} [\sigma_{\theta,i}(d_0) - v_i\sigma_{r,i}(d_0)] \quad (2.3)$$

Lame's equations:

$$\sigma_r = A - \frac{B}{r^2} \quad (2.4)$$

$$\sigma_\theta = A + \frac{B}{r^2} \quad (2.5)$$

Boundary conditions:

$$\sigma_{r,i}(d_i) = 0 = A - \frac{B}{d_i^2} \quad (2.6)$$

$$\sigma_{r,i}(d_0) = -p_{int} = A - \frac{B}{d_0^2} \quad (2.7)$$

$$A = -p_{int} \frac{d_0^2}{d_0^2 - d_i^2} \quad (2.8)$$

$$B = -p_{int} \frac{d_i^2 \cdot d_0^2}{d_0^2 - d_i^2} \quad (2.9)$$

Substituting:

$$\sigma_{\theta,i} = A + \frac{B}{r^2} = -p_{int} \frac{d_0^2}{d_0^2 - d_i^2} \left(1 + \frac{d_i^2}{r^2} \right) \quad (2.10)$$

Therefore:

$$\sigma_{r,i}(d_0) = -p_{int} \quad \sigma_{\theta,i}(d_0) = -p_{int} \frac{d_0^2 + d_i^2}{d_0^2 - d_i^2} \quad (2.11)$$

$$u_i = -p_{int} \frac{d_0}{E_i} \left(\frac{d_0^2 + d_i^2}{d_0^2 - d_i^2} - v_i \right) \quad (2.12)$$

Repeating to find u_o :

$$u_o = \frac{d_0}{E_o} [\sigma_{\theta,o}(d_0) - v_o\sigma_{r,o}(d_0)] \quad (2.13)$$

Lame's equations:

$$\sigma_r = A - \frac{B}{r^2} \quad (2.14)$$

$$\sigma_\theta = A + \frac{B}{r^2} \quad (2.15)$$

Boundary conditions:

$$\sigma_{r,o}(d_0) = -p_{int} = A - \frac{B}{d_0^2} \quad (2.16)$$

$$\sigma_{r,o}(d_b) = 0 = A - \frac{B}{d_b^2} \quad (2.17)$$

$$A = -p_{int} \frac{d_0^2}{d_0^2 - d_b^2} \quad (2.18)$$

$$B = -p_{int} \frac{d_0^2 \cdot d_b^2}{d_b^2 - d_0^2} \quad (2.19)$$

Substituting:

$$\sigma_{\theta,o} = A + \frac{B}{r^2} = -p_{int} \frac{d_0^2}{d_b^2 - 2} \left(1 + \frac{d_b^2}{r^2} \right) \quad (2.20)$$

Therefore:

$$\sigma_{r,o}(d_b) = 0 \quad \sigma_{\theta,o}(d_0) = p_{int} \frac{d_b^2 + d_0^2}{d_b^2 - d_0^2} \quad (2.21)$$

$$u_o = p_{int} \frac{d_0}{E_o} \left(\frac{d_b^2 + d_0^2}{d_b^2 - d_0^2} + v_o \right) \quad (2.22)$$

Finding δ :

$$u_i = -p_{int} \frac{d_0}{E_i} \left(\frac{d_0^2 + d_i^2}{d_0^2 - d_i^2} - v_i \right) \quad (2.23)$$

$$u_o = p_{int} \frac{d_0}{E_o} \left(\frac{d_b^2 + d_0^2}{d_b^2 - d_0^2} + v_o \right) \quad (2.24)$$

$$\delta = p_{int} d_0 \left[\frac{1}{E_o} \left(\frac{d_b^2 + d_0^2}{d_b^2 - d_0^2} + v_o \right) + \frac{1}{E_i} \left(\frac{d_0^2 + d_i^2}{d_0^2 - d_i^2} - v_i \right) \right] \quad (2.25)$$

For a single material: $E_i = E_o = E$, and $v_i = v_o = v$:

$$\delta = p_{int} \frac{d_0}{E} \left[\frac{2d_0^2 (d_b^2 - d_i^2)}{(d_b^2 - d_0^2) (d_0^2 - d_i^2)} \right] \quad (2.26)$$

$$p_{int} = E \delta \left[\frac{(d_b^2 + d_0^2) (d_0^2 - d_b^2)}{2d_0^3 (d_b^2 - d_i^2)} \right] \quad (2.27)$$