

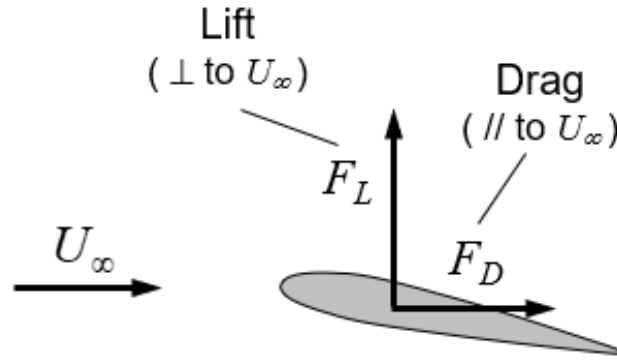
0.1 Lift and drag

Typical forces of interest for bodies in a flow are **drag** and **lift**. We can represent these in dimensionless form:

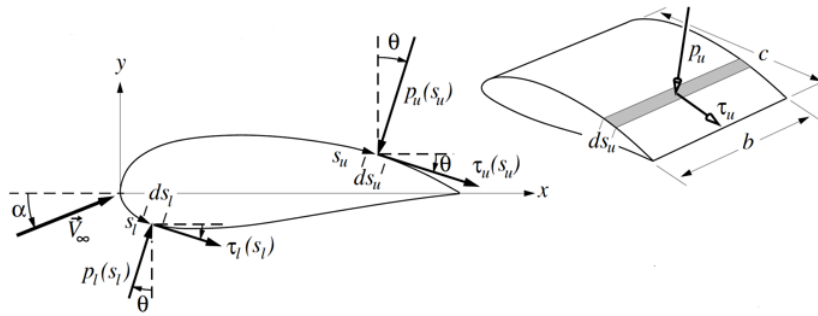
$$\text{Drag coefficient: } c_D = \frac{F_D}{\frac{1}{2}\rho U_\infty^2 S} \quad (1)$$

$$\text{Lift coefficient: } c_L = \frac{F_L}{\frac{1}{2}\rho U_\infty^2 S} \quad (2)$$

Where S is a representative area for the body, determined by convention.



0.2 Pressure and frictional force distribution



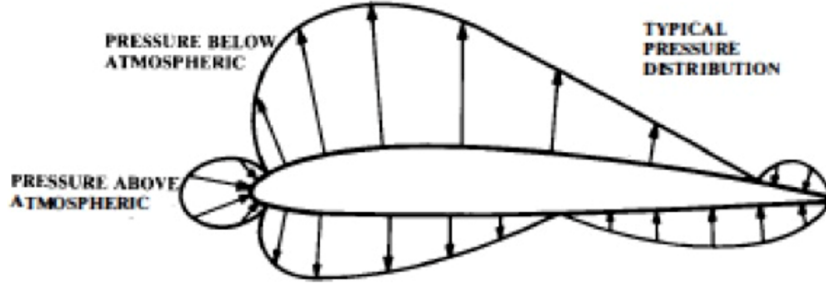
$$L = - \int_S \left(p(\hat{n} \cdot \hat{j}) \right) dS + \int^S \left(\vec{\tau} \cdot \hat{j} \right) dS \quad (3)$$

$$D = - \int \left(p(\hat{n} \cdot \hat{j}) \right) dS + \int \left(\vec{\tau} \cdot \hat{i} \right) dS \quad (4)$$

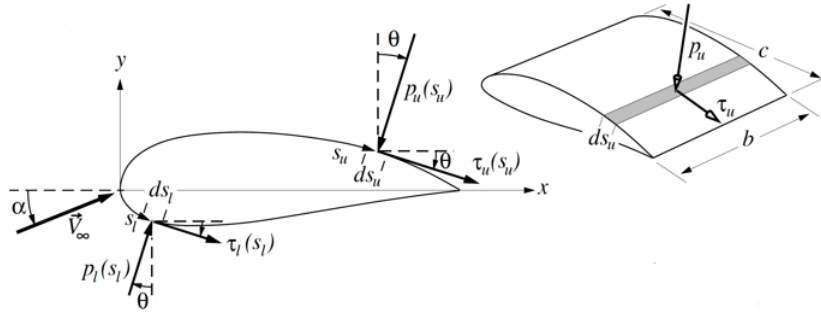
To determine the lift and drag coefficients c_L and c_D , we are interested in the pressure distribution over the airfoil, or more specifically in the local pressure difference from the stream pressure p_∞ .

$$c_p = \frac{p - p_\infty}{\frac{1}{2}\rho V_\infty^2} \quad (5)$$

Free stream pressure and velocity are p_∞ and V_∞ .



- Local suction (depression): $c_p < 0$ Vectors point away from the airfoil surface
- Local pushing: $c_p > 0$ Vectors point towards the airfoil surface



$$L = - \int_S (p \hat{n} \cdot \hat{j}) dS = \quad (6)$$

$$c_L = -\frac{1}{S} \int_S \left(\frac{p - p_\infty}{\frac{1}{2}\rho V_\infty^2} \hat{n} \cdot \hat{j} \right) dS = -\frac{1}{S} \int_S (c_p \hat{n} \cdot \hat{j}) dS \quad (7)$$

The lift coefficient per unit of span-wise length is:

$$c'_L = \frac{1}{c} \int_c^0 (c_p \hat{n} \cdot \hat{j}) dx \quad (8)$$

0.3 Rearrangement of momentum equation - x direction

$$\rho \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = -\frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) \quad (9)$$

$$\left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) + v \frac{\partial u}{\partial y} - v \frac{\partial u}{\partial y} \quad (10)$$

$$= -v \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) + \frac{1}{2} \left(\frac{\partial u^2}{\partial x} + \frac{\partial v^2}{\partial x} \right) \quad (11)$$

$$= -v \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) + \frac{1}{2} \frac{\partial}{\partial x} (u^2 + v^2) \quad (12)$$

$(u^2 + v^2)$ is the total kinetic energy of the fluid particle. The derivative is the element that takes into the account the variation of this kinetic energy. $\left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right)$ relates to the rotation of the particle. This rotation is related to the difference of velocity gradient.

$$\rho \left[-v \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) + \frac{1}{2} \frac{\partial}{\partial x} (u^2 + v^2) \right] = -\frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) \quad (13)$$

$$-v \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) = -\frac{\partial}{\partial x} \left(\frac{p}{\rho} + \frac{u^2 + v^2}{2} \right) + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) \quad (14)$$

Our Bernoulli term in the above equation is $\left(\frac{p}{\rho} + \frac{u^2 + v^2}{2} \right)$, gravitational energy is negligible. $\left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right)$ is an anti-clockwise rotation. Hence, the vorticity component in the z direction is:

$$\omega_z = \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \quad (15)$$

Our final momentum equations in x and y are:

$$-v\omega_z = -\frac{\partial}{\partial x} \left(\frac{p}{\rho} + \frac{u^2 + v^2}{2} \right) + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) \quad (16)$$

$$u\omega_z = -\frac{\partial}{\partial y} \left(\frac{p}{\rho} + \frac{u^2 + v^2}{2} \right) + \nu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right) \quad (17)$$

We can make some assumptions:

- Inviscid flow - $\nu = 0$ (this may be realistic in some parts of a fluid domain but in real life, inviscid fluids do not exist)
- Irrotational flow - $\omega_z = 0$

This reduces our equations to:

$$0 = -\frac{\partial}{\partial x} \left(\frac{p}{\rho} + \frac{u^2 + v^2}{2} \right) + 0 \quad (18)$$

$$0 = -\frac{\partial}{\partial y} \left(\frac{p}{\rho} + \frac{u^2 + v^2}{2} \right) + 0 \quad (19)$$

0.4 Application of Bernoulli

$$p_\infty + \frac{1}{2}\rho V_\infty^2 = p + \frac{1}{2}\rho(u^2 + v^2) = \text{constant} \quad (20)$$

$$c_p = \frac{p - p_\infty}{\frac{1}{2}\rho V_\infty^2} = 1 - \frac{u^2 + v^2}{V_\infty^2} = 1 - \frac{||V||^2}{V_\infty^2} \quad (21)$$