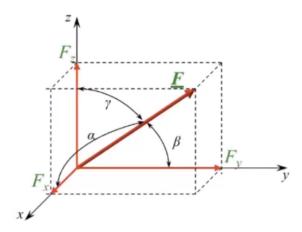
#### **Actions and Deformations** 0.1

#### Vector Quantities 0.1.1

A vector is a quantity defined by **magnitude** and **direction**. Mechanical actions (forces and moments) can be represented as **vectors**.

Vector quantities can be decomposed in components, that can be conveniently oriented with the Cartesian reference system.

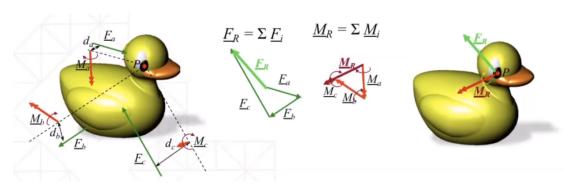


$$F_x = F \cdot \cos \alpha \qquad M_x = M \cdot \cos \alpha \tag{1}$$

$$F_y = F \cdot \cos \beta \qquad M_y = M \cdot \cos \beta \tag{2}$$

$$F_x = F \cdot \cos \alpha$$
  $M_x = M \cdot \cos \alpha$  (1)  
 $F_y = F \cdot \cos \beta$   $M_y = M \cdot \cos \beta$  (2)  
 $F_z = F \cdot \cos \gamma$   $M_z = M \cdot \cos \gamma$  (3)

On the other hand, a set of vector forces can be composed in a resultant force applied to any point P, and the moment they produce about P.



$$\vec{F_R} = \sum \vec{F_i} \qquad \vec{M_R} = \sum \vec{F_i} \tag{4}$$

$$\vec{F_R} = \sum \vec{F_i} \qquad \vec{M_R} = \sum \vec{F_i}$$

$$F_R = \sqrt{(F_x)^2 + (F_y)^2 + (F_z)^2}$$
(5)

$$M_R = \sqrt{(M_x)^2 + (M_y)^2 + (M_z)^2}$$
 (6)

## 0.1.2 Equilibrium State

If a configuration is in equilibrium, the resultant of all external forces and moments is zero. This can be expressed mathematically in the following 6 equations:

$$\sum_{x} F_x = 0 \qquad \sum_{x} F_y = 0 \qquad \sum_{x} F_z = 0$$
$$\sum_{x} M_x = 0 \qquad \sum_{x} M_y = 0 \qquad \sum_{x} M_z = 0$$

These equations have to be valid for the entire body, and for any of its portions.

## 0.1.3 Deformations

Mechanical actions produce **deformations** in the body. These can be:

- Tension
- Compression
- Bending
- Twisting

These deformations translate into local strains and are opposed and balanced by internal reaction forces (and stresses), that guarantee the structural congruence of the body.

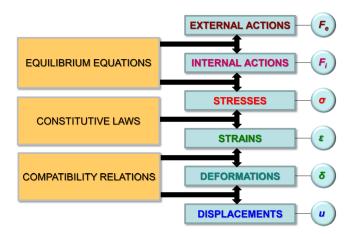
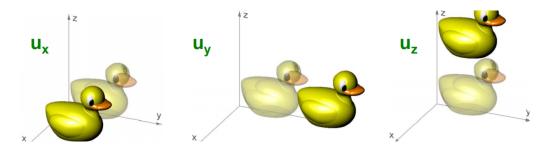


Figure 1: Solid Mechanics Equation: When dealing with mechanical action problems, the actions listed in the flowchart above occur, starting with external/internal forces and ending with displacements/deformations

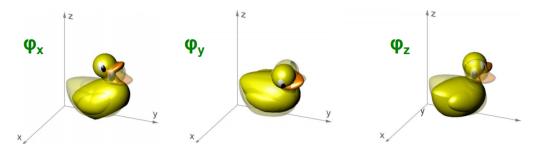
# 0.2 Degree of Freedom and Supports

We define **degree of freedom** of a system as all the basic kinematical parameters (or all the forms of movement) allowed. A rigid body in the space, in a coordinate system, has 6 degrees of freedom:

3 translations along the coordinate axes x, y and z



3 rotations about the coordinate axes x, y and z



The total translational and rotational movement of an object can be shown with the following expression:

$$ec{u} = \left[ egin{array}{c} u_x \\ u_y \\ u_z \\ \phi_x \\ \phi_y \\ \phi_z \end{array} 
ight]$$

In a 2D plane, the degree of freedom reduces to pnly 3 variables:

$$\vec{u} = \begin{bmatrix} u_x \\ u_y \\ \phi_z \end{bmatrix}$$

## 0.2.1 Constraint

We define **constraint** as a limitation of the degree of freedom of the system. The most common constraints are:

- Supports providing the required reacting forces to maintain overall equilibrium
- Connections providing reaction forces between two components of the system

The table below summarizes the different types of supports (constraints) that will be used throughout the course:

Fixed	Rotating	Roller	Sliding
$ec{u} = \left[ egin{array}{c} 0 \ 0 \ 0 \end{array}  ight]$	$\vec{u} = \begin{bmatrix} 0 \\ 0 \\ \phi \end{bmatrix}$	$ec{u} = \left[ egin{array}{c} u_x \ 0 \ \phi \end{array}  ight]$	$ec{u} = \left[egin{array}{c} u_x \ 0 \ 0 \end{array} ight]$
$\vec{R} = \begin{bmatrix} R_x \\ R_y \\ M_z \end{bmatrix}$	$\vec{R} = \begin{bmatrix} R_x \\ R_y \\ 0 \end{bmatrix}$	$\vec{R} = \begin{bmatrix} 0 \\ R_y \\ 0 \end{bmatrix}$	$\vec{R} = \begin{bmatrix} 0 \\ R_y \\ M \end{bmatrix}$

# 0.3 Beams and Sign Conventions

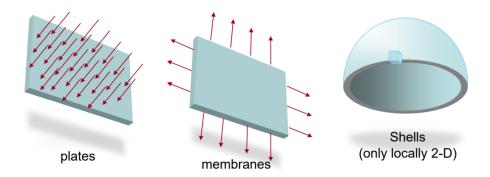
Structures are sets of solid bodies components with the function of carrying loads. All solid bodies are 3-dimensional, however, often it is possible to identify some dimension that is more relevant. Many structures can be analysed as bi-dimensional (2D) or mono-dimensional (1D).

## 0.3.1 Types of Structures

### **Bi-dimensional Structures**

If one of the dimensions is negligible compared to the other two, the structure can be studied as bi-dimensional. Some examples are:

- Plates
- Membranes
- Shells (Only locally 2D)

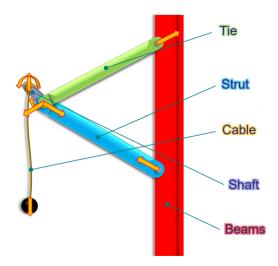


For shells to be considered bi-dimensional, they need to be looked at very closely where they can resemble a plate. The difference between plates and membranes is the bending rigidity; force is required to bend plates while membranes are really floppy.

#### Mono-dimensional Structures

If two of the dimensions are negligible compared to the other one, the structure can be studied as mono-dimensional. Some examples are:

- Tie Prevents two parts of the structure from moving away
- Strut Prevents two parts of the structure from moving forward
- Cable Flexible string that stands only tensile loads
- Shaft Is used for the transmission of torque
- Beams Can carry also transverse loads



## 0.3.2 Beams

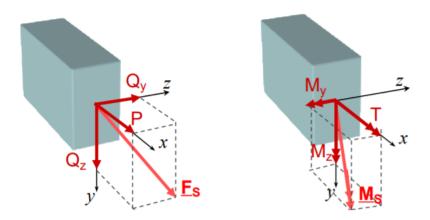
The generic mono-dimensional components of structures, able to carry also transverse loads are called **beams**. Beams are between the most common and important components in structures. In order to study beams as mono-dimensional structures, all mechanical actions have to act on the **centre of gravity (CG)** of the beam section. If there is a case where a force isn't acting on the CG, it will be converted to act on it, so that the beam can be analysed in a simple manner.



Figure 2: On the left: Simply supported beam (rotating support + roller support), On the right: Cantilever beam (with a fixed end)

### 0.3.3 Internal Forces

Each point of the beam is characterised by a specific set of internal forces. We consider a cross section of the beam to investigate these forces.

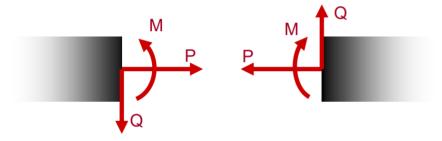


- $\bullet~P$  Longitudinal force
- $Q_y$  y shear forces
- $Q_z$  z shear forces T Torque
- $M_y$  y bending moment
- $M_z$  z bending moment

The internal forces at every point of the beam can be characterised with the following expression:

$$ec{F} = \left[ egin{array}{c} P \ Q_y \ Q_z \ T \ M_y \ M_z \end{array} 
ight]$$

In a 2D plane, it simplifies to:



- $\bullet$  P Longitudinal force
- ullet Q Shear forces
- ullet M Bending moment

$$\vec{F} = \left[ \begin{array}{c} P \\ Q \\ M \end{array} \right]$$

## 0.3.4 Sign Conventions

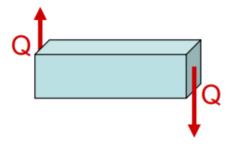
## Longitudional Force



The direction of pulling is considered to be positive.

The direction of compression is considered to be negative.

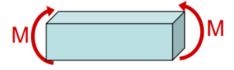
#### **Shear Force**



The left side pointing upwards and right side pointing downwards are taken as positive.

The left side pointing downwards and right side pointing upwards are taken as negative.

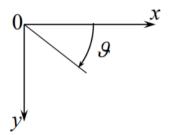
## **Bending Moment**



If the bending moment is making the beam a upwards concave shape (like U), it is considered to be positive.

If the bending moment is making the beam a downwards concave shape, it is considered to be negative.

### Reference System



x axis (horizontal direction) to the right is taken as positive.

y axis (vertical direction) downwards is taken as positive.