

UCL Mechanical Engineering 2020/2021

MECH0010 Coursework 1

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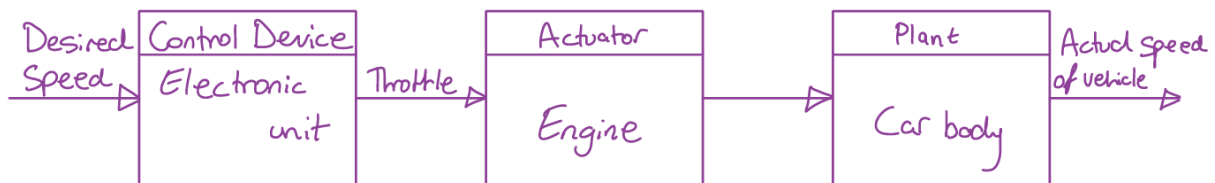
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Part I

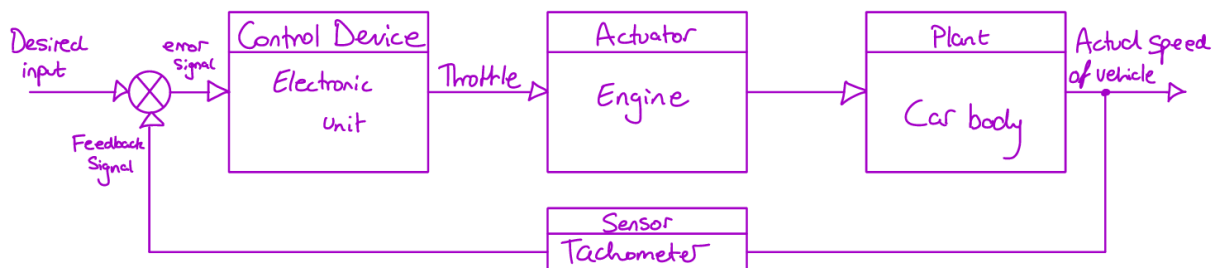
Control

1 Question A

Block diagram (open loop)



Block diagram (closed loop)



With closed loop control, we can better control the actual speed of the vehicle. For example, when in abnormal conditions, (bad weather, incline, decline) there will be a different resistive force acting against the motion of the vehicle. In the cases where the resistance force is changed, if power output is constant, the speed of the vehicle will be different (assuming throttle position is proportional to engine power output).

For example, on an incline not only does the engine force have to match the resistive forces but also the sine component of the weight of the vehicle ($\sum \text{opposing forces} = F_R + mg \sin \theta$, where θ is the incline angle.) The derivation for the equation linking power

output, force and velocity is below.

$$W = \int_{x_0}^x F \cdot dx \quad (1.1)$$

$$v = \frac{dx}{dt} \quad (1.2)$$

$$v dt = dx \quad (1.3)$$

$$W = \int_{t_0}^t Fv \cdot dt \quad (1.4)$$

$$P = \frac{dW}{dt} \quad (1.5)$$

$$P = \frac{d}{dt} \left(\int_{t_0}^t Fv \cdot dt \right) \quad (1.6)$$

$$P = Fv \quad (1.7)$$

If P is constant and F is increased/decreased, v must decrease/increase.

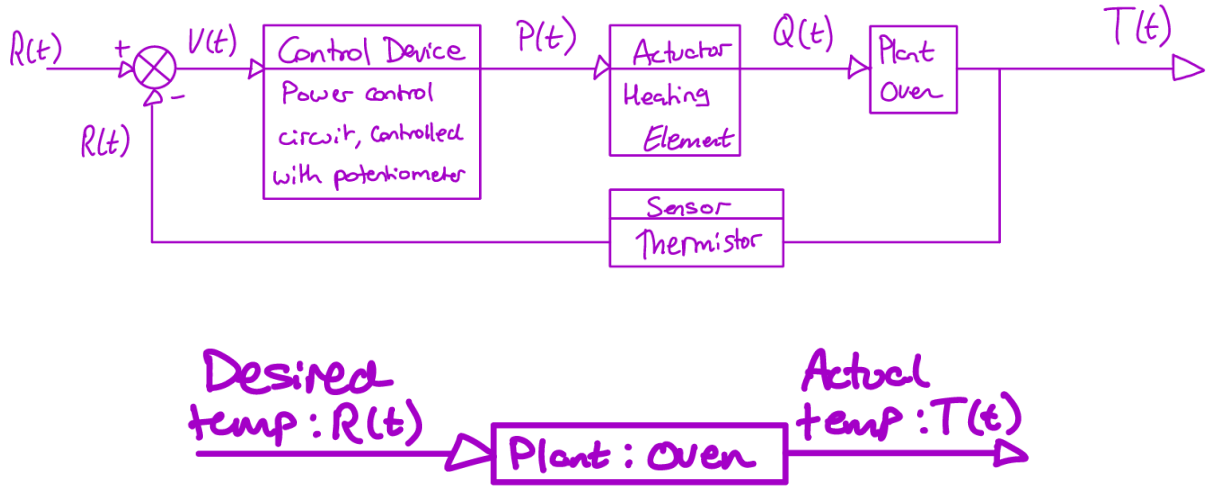
Proximity sensors

We can utilise a proximity sensor to measure the distance to the vehicle ahead. This can be used to 'track' the vehicle ahead. A simple cruise control system is unable to make changes to the throttle in response to changing road conditions. With a proximity sensor, we can observe whether the vehicle ahead is getting closer (vehicle ahead is braking) or moving further away (vehicle ahead is accelerating away). By setting a fixed 'following distance' (how far away the vehicle ahead should be kept), the driver can set a maximum speed for the vehicle to travel at and the vehicle will automatically slow down and speed up to the limits set in response to changing road conditions.

For example, in traffic vehicles are constantly slowing down and accelerating. With our previous system, we would have to disable the cruise control (by braking) and set it again once vehicles speed up again. Our new system will allow a user to simply set the cruise control once and no longer worry about colliding into the car in front, as the car will keep a safe distance from the vehicle.

2 Question B

- Sensor - circuit to measure resistance of heating element/thermistor to measure temperature of oven.
- Actuator - heating element
- Input signal(s) - Relay switch (heating element on and off, convection fan on and off)
- Output signal(s) - Oven temperature [T]
- Control signal(s) - Rotary switch (temperature control)



My first solution is to have an open-loop control system. Our input is simply the temperature desired. I have left the timing to an external device such as an oven timer, however this could be easily implemented with a simple timing circuit and relay. This system utilises a potentiometer as an analogue input. Each resistance setting corresponds to a certain desired temperature. Our heating element operates on the basis of Joule heating, whereby passing a current through a conductor produces a heating effect. Assuming that the heating element is a perfect resistor, we know that:

$$P = IV \quad (2.1)$$

$$V = IR \quad (2.2)$$

$$I = \frac{V}{R} \quad (2.3)$$

$$P = I^2 \cdot R \quad (2.4)$$

Whether we use a DC supply or an AC supply has a negligible effect on our system, as we can simply look at the RMS values for AC.

Our heating element will continue to heat up until it reaches a certain equilibrium (where the power in to the heating element matches the heat dissipation), however this equilibrium temperature will most likely not be the desired temperature. We have two options here: modulate our power supply to change the equilibrium temperature of the heating element or switch the power supply off and on at threshold values. Some of the advantages of power modulation include a stable temperature, but this may come at the cost of additional electronic complexity and cost.

On the other hand, on-off control is relatively simple. We can set our threshold values to be T_{set} and -5°C of the desired temperature. I have arbitrarily chosen this value, as the effects of a temperature oscillation of 5°C will have negligible effects on the food being cooked. The thermal inertia of the heating coil is also relatively high, thus switching will occur with low frequency and can be managed with a simple relay.

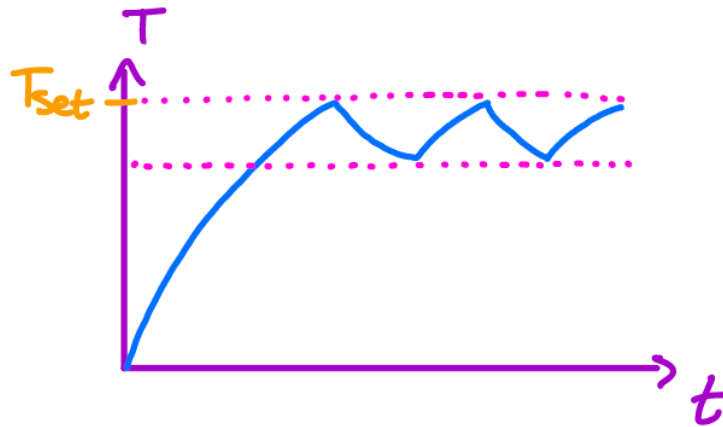


Figure 1: Temperature - time graph for an on-off control system. We can see the desired temperature as T_{set} and our threshold temperatures as the pink dotted lines. As the circuit is switched on and off, the temperature of the oven is controlled.

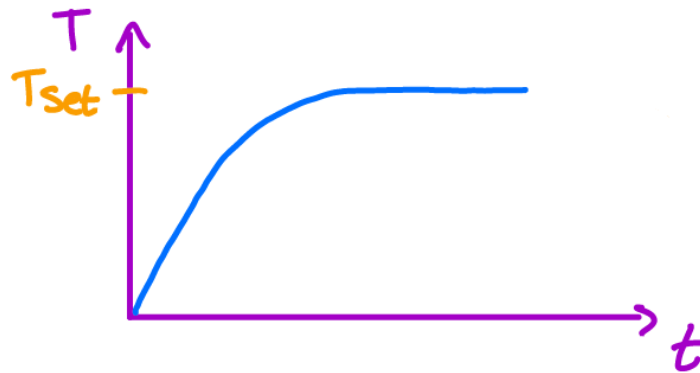
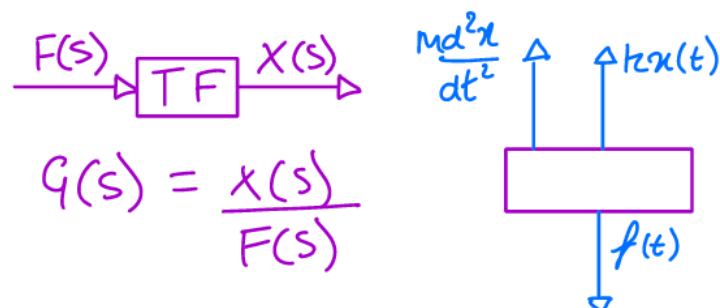


Figure 2: Temperature - time graph for a modulated power supply. Here we can see that the equilibrium temperature is our T_{set} value - the temperature at which power in = power dissipated.

3 Question C

Transfer function



$$G(s) = \frac{X(s)}{F(s)} \quad (3.1)$$

$$f(t) = kx(t) + m \frac{d^2 x}{dt^2} \quad (3.2)$$

$$F(s) = kX(s) + ms^2 X(s) \quad (3.3)$$

$$F(s) = X(s)(k + ms^2) \quad (3.4)$$

$$\frac{X(s)}{F(s)} = \frac{1}{k + ms^2} \quad (3.5)$$

$$\frac{X(s)}{F(s)} = \frac{\frac{1}{m}}{s^2 + \frac{k}{m}} \quad (3.6)$$

Time domain response

$$\frac{X(s)}{F(s)} = \frac{\frac{1}{m}}{s^2 + \frac{k}{m}} \quad (3.7)$$

$$\frac{X(s)}{F(s)} = \frac{\sqrt{\frac{k}{m}}}{s^2 + \frac{k}{m}} \times \frac{1}{\sqrt{mk}} \quad (3.8)$$

Where $\omega = \sqrt{\frac{k}{m}}$

$$x(t) = L^{-1} \left[\frac{\sqrt{\frac{k}{m}}}{s^2 + \frac{k}{m}} \cdot \frac{1}{\sqrt{mk}} \right] \quad (3.9)$$

$$x(t) = \frac{1}{\sqrt{mk}} \cdot L^{-1} \left[\frac{\sqrt{\frac{k}{m}}}{s^2 + \frac{k}{m}} \right] \quad (3.10)$$

$$x(t) = \frac{1}{\sqrt{mk}} \cdot \sin \left(\sqrt{\frac{k}{m}} \cdot t \right) \quad (3.11)$$

The frequency and period are:

$$\omega = 2\pi f = \sqrt{\frac{k}{m}} \quad (3.12)$$

$$f = \frac{1}{2\pi} \sqrt{\frac{k}{m}} \quad (3.13)$$

$$T = \frac{1}{f} = 2\pi \sqrt{\frac{m}{k}} \quad (3.14)$$

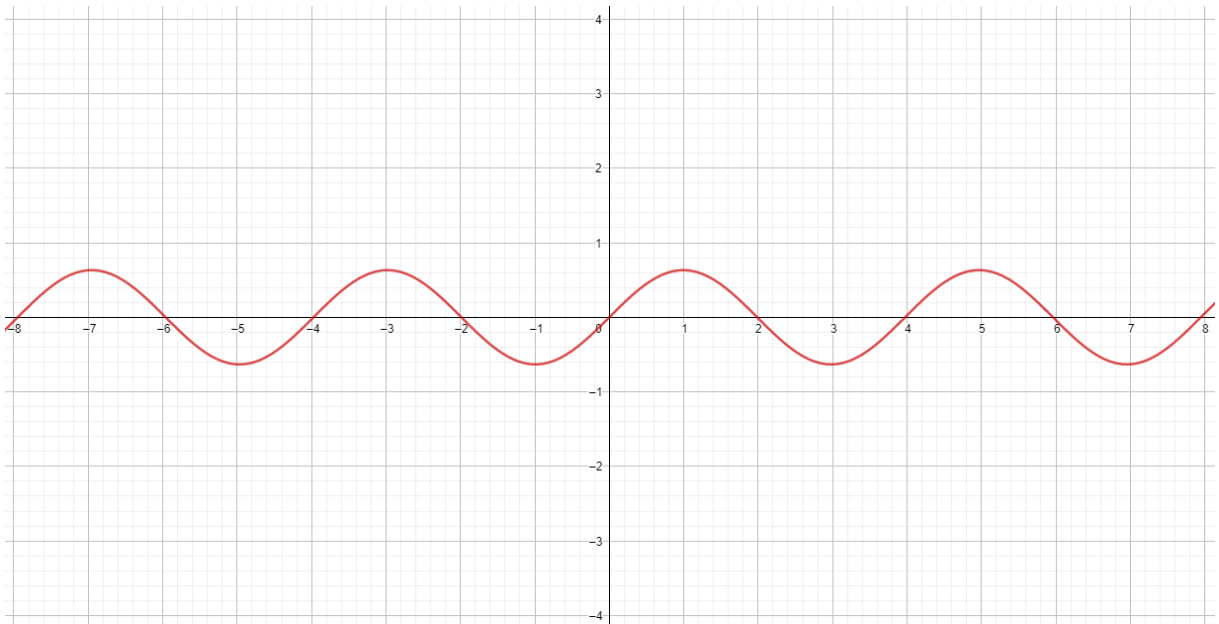


Figure 3: Graph to show plot of $x(t) = \frac{1}{\sqrt{mk}} \cdot \sin\left(\sqrt{\frac{k}{m}} \cdot t\right)$. m and k were arbitrarily selected as 1 and 2.5 respectively. y-axis: displacement, x-axis: time.

Discussion and improvements

After an hour, the system would still be oscillating as it were initially. This would not be the case in a real life system, as air resistance and losses would cause our system to stop oscillating. The system is missing this damping effect and an adding a damper will make our system more realistic.

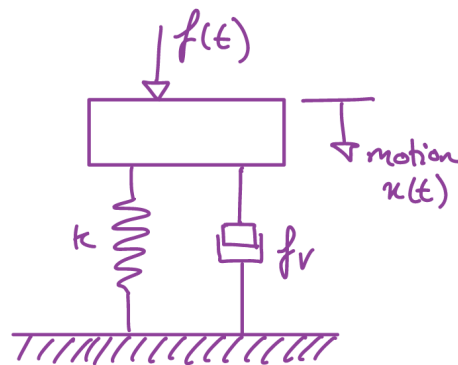


Figure 4: System diagram with a damper added.

$$f(t) = kx(t) + f_v \frac{dx}{dt} + m \frac{d^2 x}{dt^2} \quad (3.15)$$

$$F(s) = kX(s) + f_v sX(s) + ms^2 X(s) \quad (3.16)$$

$$\frac{X(s)}{F(s)} = \frac{1}{ms^2 + f_v s + k} \quad (3.17)$$

$$G(s) = \frac{1}{m} \cdot \frac{1}{s^2 + \frac{f_v}{m}s + \frac{k}{m}} \quad (3.18)$$

$$= \frac{1}{m} \cdot \frac{1}{\left(s + \frac{f_v}{2m}\right)^2 + \frac{4km - f_v^2}{4m^2}} \quad (3.19)$$

$$= \frac{1}{m \cdot \sqrt{\frac{4km - f_v^2}{4m^2}}} \cdot \frac{\sqrt{\frac{4km - f_v^2}{4m^2}}}{\left(s + \frac{f_v}{2m}\right)^2 + \frac{4km - f_v^2}{4m^2}} \quad (3.20)$$

$$= \frac{2}{\sqrt{4km - f_v^2}} \cdot \frac{\sqrt{\frac{4km - f_v^2}{4m^2}}}{\left(s + \frac{f_v}{2m}\right)^2 + \frac{4km - f_v^2}{4m^2}} \quad (3.21)$$

From Laplace tables:

$$e^{-at} \sin \omega t = \frac{\omega}{(s + a)^2 + \omega^2} \quad (3.22)$$

$$\omega = \sqrt{\frac{4km - f_v^2}{4m^2}} \quad (3.23)$$

$$a = \frac{f_v}{2m} \quad (3.24)$$

$$x(t) = L^{-1} \left[\frac{2}{\sqrt{4km - f_v^2}} \cdot \frac{\sqrt{\frac{4km - f_v^2}{4m^2}}}{\left(s + \frac{f_v}{2m}\right)^2 + \frac{4km - f_v^2}{4m^2}} \right] \quad (3.25)$$

$$x(t) = \frac{2e^{-\frac{f_v}{2m}t} \sin \left(\frac{\sqrt{4km - f_v^2}}{2m} \cdot t \right)}{\sqrt{4km - f_v^2}} \quad (3.26)$$

This was plotted:

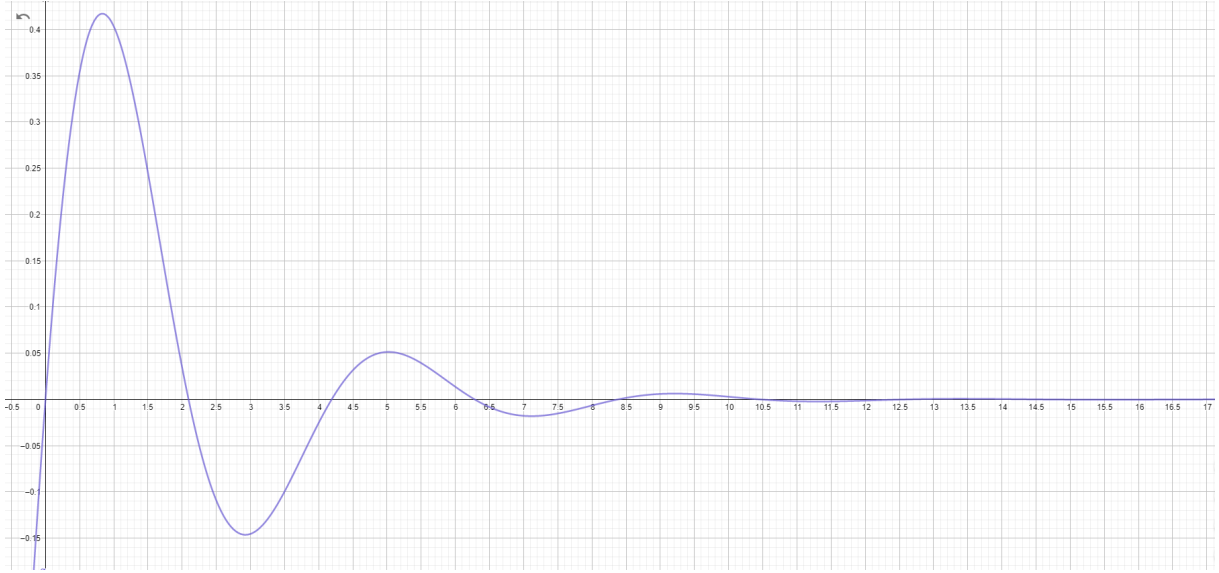


Figure 5: Graph to show plot of $x(t) = \frac{2e^{-\frac{f_v \cdot t}{2m}} \sin\left(\frac{\sqrt{4km - f_v^2}}{2m} \cdot t\right)}{\sqrt{4km - f_v^2}}$ m , k and f_v were arbitrarily selected as 1, 2.5 and 1 respectively. y-axis: displacement, x-axis: time.

We can also utilise another standard form to derive some characteristics of our system:

$$G(s) = \frac{1}{ms^2 + f_v s + k} \quad (3.27)$$

$$= \frac{1}{k} \cdot \frac{\frac{k}{m}}{s^2 + \frac{f_v}{m}s + \frac{k}{m}} \quad (3.28)$$

Standard form:

$$= \gamma \cdot \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \quad (3.29)$$

Where, ω_n is the natural frequency, ζ is the damping ratio and γ is the gain. For our system these are, $\omega_n = \sqrt{\frac{k}{m}}$, $\zeta = \frac{c}{2\sqrt{mk}}$ and $\gamma = \frac{1}{k}$.

Part II

Instrumentation

4 Question 1

Wheatstone bridge

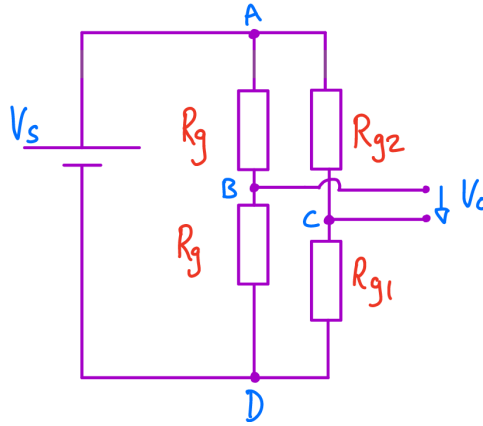


Figure 6: Circuit diagram to show a Wheatstone bridge

Resistances of our two strain gauges are:

$$R_{g1} = R(1 + x) \quad (4.1)$$

$$R_{g2} = R(1 - x) \quad (4.2)$$

We know that:

$$V_B = V_S \cdot \frac{R}{2R} = \frac{V_S}{2} \quad (4.3)$$

$$V_C = V_S \cdot \frac{R_{g1}}{R_{g1} + R_{g2}} \quad (4.4)$$

$$V_0 = V_C - V_B \quad (4.5)$$

$$V_0 = V_S \left(\frac{R_{g1}}{R_{g1} + R_{g2}} - \frac{1}{2} \right) \quad (4.6)$$

$$V_0 = V_S \frac{R(1 + x)}{R(1 + x) + R(1 - x)} - \frac{V_S}{2} \quad (4.7)$$

$$V_0 = \frac{1}{2} V_S \cdot x \quad (4.8)$$

We know that $x = eG$, hence:

$$\frac{V_0}{e} = \frac{1}{2} V_S \cdot G \quad (4.9)$$

Identify specification

- $G = 2.1$
- $R_g = 120\ \Omega$
- $\frac{V_0}{e} = 5\ \text{V}$

$$5 = \frac{1}{2}V_S \times 2.1 \quad (4.10)$$

$$V_S = 4.76\ \text{V} \quad (3\text{sf}) \quad (4.11)$$

$$(4.12)$$

To balance the bridge, we can set R to be the same value as our nominal gauge resistance.

$$R = R_g = 120\ \Omega \quad (4.13)$$

Longitudinal stretch

Since both strain gauges are undergoing tension, we will see their resistance change in the same way, rather than opposite to each other. This will lead to an approximately 0 V_S .

$$V_0 = V_S \left(\frac{R_{g1}}{R_{g1} + R_{g2}} - \frac{1}{2} \right) \quad (4.14)$$

$$R_{g1} = R_{g2} = R(1 + x) \quad (4.15)$$

$$V_0 = V_S \frac{R(1 + x)}{2R(1 + x)} - \frac{V_S}{2} \quad (4.16)$$

$$V_0 = 0 \quad (4.17)$$

Method to increasing sensitivity

We can increase the sensitivity of the system by adding two more gauges to our system. The arrangement shown below increases both the sensitivity whilst also compensating for temperature.

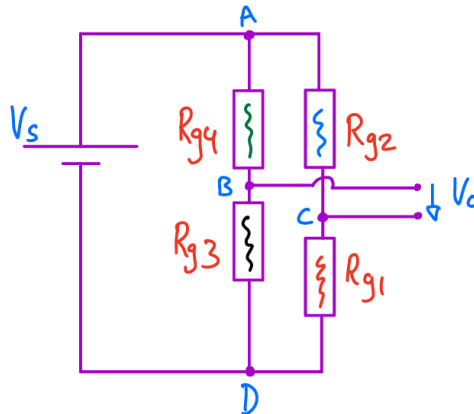


Figure 7: Circuit diagram to show a Wheatstone bridge with increased sensitivity.

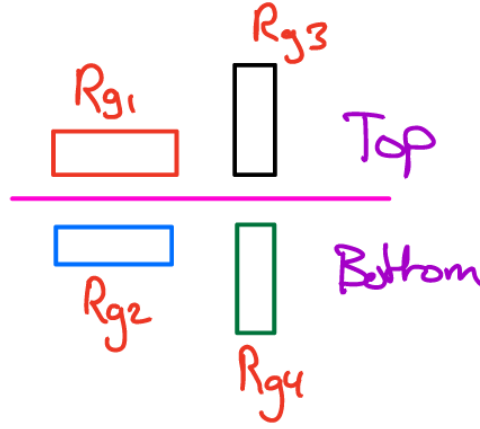


Figure 8: Diagram to show arrangement of strain gauges for increased sensitivity.

$$V_0 = V_S \left(\frac{R_{g1}}{R_{g1} + R_{g2}} - \frac{R_{g3}}{R_{g3} + R_{g4}} \right) \quad (4.18)$$

$$V_0 = V_S \left(\frac{R(1+x)}{R(1+x) + R(1+\nu x)} - \frac{R(1-\nu x)}{R(1-\nu x) + R(1+\nu x)} \right) \quad (4.19)$$

$$V_0 = V_S \left(\frac{1}{2}(1+x) - \frac{1}{2}(1-\nu x) \right) \quad (4.20)$$

$$V_0 = \frac{V_S \cdot x}{2} (1 + \nu) \quad (4.21)$$

$$\frac{V_0}{e} = \frac{4.8}{2} \times 2.1 \times (1 + 0.3) \quad (4.22)$$

$$\frac{V_0}{e} = 6.6 \text{ V} \quad (4.23)$$

$6.6 > 5$, hence we can see that this arrangement has increased sensitivity.

5 Question 2

The effect of ambient temperature has three primary effects:

- Dimensions of the specimen and hence the gauge will change due to thermal expansion
- Dimensions of the gauge itself (particularly the thickness) will change
- Resistivity of the gauge material will change

We can tackle the first two points by actually utilising a strain gauge which is of the same material as the test specimen. The thermal expansion of the material and the strain gauge will now be the same, cancelling out the effect of the contraction/expansion of the test specimen.

A second solution for temperature compensation is incorporating a dummy gauge. We have two options here: we can place the dummy gauge on an unstrained part of the same beam (placing it normal to the principal stress axis), or we can place it on a separate, unloaded beam (with identical properties) in the same environment as our test specimen. This dummy gauge will act as a 'reference' for our strain gauge and due to the ratiometric nature of the Wheatstone bridge, we can place the dummy gauge in another arm of the bridge to achieve this.

The system in Figure 3 of the coursework brief does compensate for temperature, as two gauges are being utilised. We can show this by:

$$R_{g1} = R(1 + x)(1 + y) \quad (5.1)$$

$$R_{g2} = R(1 - x)(1 + y) \quad (5.2)$$

$$(5.3)$$

Where $R(1 + y)$ is the resistance change due to temperature.

$$V_0 = V_S \frac{R(1 + x)(1 + y)}{R(1 + x)(1 + y) + R(1 - x)(1 + y)} - \frac{V_S}{2} \quad (5.4)$$

$$V_0 = V_S \frac{R(1 + x)}{R(1 + x) + R(1 - x)} - \frac{V_S}{2} \quad (5.5)$$

We can see that we have a factor of $(1 + y)$ in our fraction, which cancels out the effects of temperature.

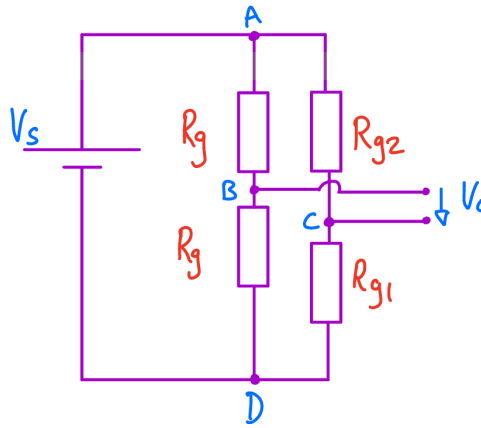


Figure 9: Circuit diagram to show a Wheatstone bridge which compensates for temperature.

V_{amp} variation with beam vibration

$$V_{amp} = \frac{R_2}{R_1}(V_2 - V_1) = \frac{R_2}{R_1}V_0 \quad (5.6)$$

$$V_0 = V_S \frac{R_{g1}}{R_{g1} + R_{g2}} - \frac{V_S}{2} \quad (5.7)$$

$$V_0 = V_S \frac{R(1+x)}{R(1+x) - R(1-x)} - \frac{V_S}{2} \quad (5.8)$$

$$V_0 = \frac{1}{2}V_S \cdot x \quad (5.9)$$

$$V_{amp} = \frac{R_2}{2R_1}V_S \cdot e \cdot G \quad (5.10)$$

- $e = 0.01 \sin(4\pi \cdot t)$
- $R_1 = 1000 \Omega$
- $R_2 = 200\,000 \Omega$
- $G = 2.1$
- $V_S = 4.76 \text{ V}$

$$V_{amp}(t) = \frac{200000}{2000} \cdot 4.76 \cdot 2.1 \cdot 0.01 \cdot \sin(4\pi \cdot t) \quad (5.11)$$

$$V_{amp}(t) = \frac{2499}{250} \sin(4\pi \cdot t) \quad (5.12)$$

- Maximum amplitude: 10 V (2sf)
- Frequency: 4π Hz

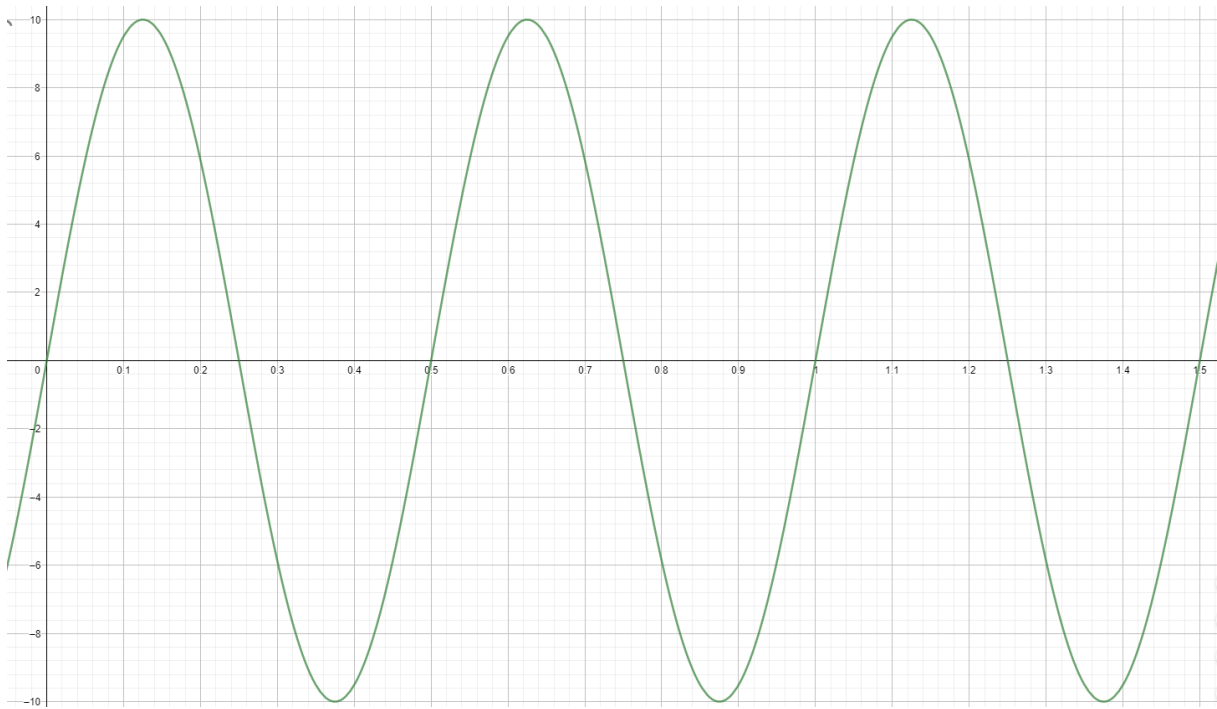


Figure 10: Graph to show V_{amp} as the beam vibrates.

6 Question 3

Amplified output voltage

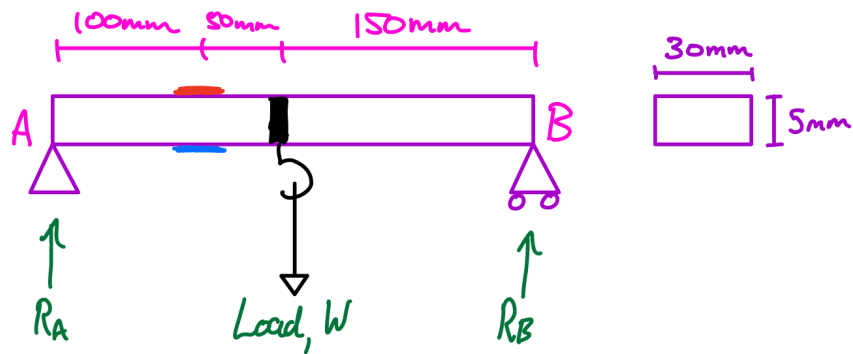


Figure 11: Beam setup.

Maximum weight system can measure:

$$\sum F_y : R_A = R_B = \frac{W}{2} \quad (6.1)$$

$$\epsilon_{max} = 0.005 \quad (6.2)$$

$$\sigma = \frac{Mh}{I} \quad (6.3)$$

$$h_{max} = 0.0025 \text{ m} \quad (6.4)$$

$$I = \frac{bd^3}{12} = \frac{0.03 \cdot 0.005^3}{12} = 3.125 \times 10^{-10} \text{ m}^4 \quad (6.5)$$

$$M(x) = R_A \cdot x - \langle Wx - 0.15 \rangle \quad (6.6)$$

$$M(I = 0.1) = \frac{0.1 \cdot W_{max}}{2} = 0.05 \cdot W_{max} \quad (6.7)$$

$$\sigma = \frac{0.05 \cdot W_{max} \cdot 0.0025}{3.125 \times 10^{-10}} = E\epsilon = 70 \times 10^9 \cdot 0.005 \quad (6.8)$$

$$W_{max} = 875 \text{ N} \quad (6.9)$$

Amplified output voltage V_{amp} , when 10 kg weight is placed on beam:

$$\sigma = \frac{Mh}{I} \quad (6.10)$$

$$\frac{0.05 \cdot 10g \cdot 0.0025}{3.125 \times 10^{-10}} = 70 \times 10^9 \cdot \epsilon \quad (6.11)$$

$$\epsilon = 5.6057 \times 10^{-4} \quad (6.12)$$

$$V_{amp} = 1000\epsilon = 0.56 \text{ V}(2sf) \quad (6.13)$$

Semiconductor gauges

Semiconductor gauges can have extremely high relative gauge factors (100-300). This makes them very sensitive and they also have a strong output signal due to this high gauge factor.

However, semiconductor gauges also have a smaller elastic limit (typically a factor of 10 less). At high strain levels, the gauge factor also varies, meaning there is a non-linear relationship. The gauge factor also varies with temperature and the temperature coefficient of resistivity is also large. Making them unsuitable in applications with wide-varying temperatures.