MECH0010 Assignment Report 1 - Group 13

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Abstract—**3D printers allow the rapid development of CAD modelled objects. 3D printers are also very versatile, being able to print a wide range of materials and at varying resolutions and sizes. This paper will look at the control systems and some of the measurement systems employed by 3D printers.**

INTRODUCTION

In this paper, we shall be investigating velocity and position control of a servo using Simulink, relating to the motion of the printer head. We shall also investigate circuits and design systems to measure the position of the printer head to a suitable degree of accuracy.

QUESTION 1 - IMPLEMENT OPEN AND CLOSED LOOP VELOCITY AND POSITION CONTROL...

In the real world, the operation of DC motor is the transfer from electrical energy to the mechanical rotation of motor. Therefore, the electric circuit of motor operation is composed by the input voltage (v (V)), terminal resistance (R (Ω)), Inductance (L (H)), and the electromotive force (em (V)). In this case the formulas for the time domain and frequency domain are (i is the current of the circuit):

$$V = Ri + L\frac{\mathrm{d}i}{\mathrm{d}t} + em \tag{1}$$

$$V(s) = RI(s) + LsI(s) + Em(s) \tag{2}$$

For the mechanical rotation, the relationship among torque (T (N m)), moment of inertia (Im (kg m^2)), angular displacement (θ) and the friction coefficient (b) in time domain and frequency domain are given:

$$T = Im\frac{\mathrm{d}^2\theta}{\mathrm{d}t^2} + b\frac{\mathrm{d}\theta}{\mathrm{d}t} \tag{3}$$

$$T(s) = Ims^2\theta(s) + bs\theta(s) \tag{4}$$

In this case, angular velocity is proportional to the electromotive force and current is proportional to the torque of the motor. (K_e the electrical motor constant, K_t is the mechanical motor constant):

$$em = K_e\frac{\mathrm{d}\theta}{\mathrm{d}t} \tag{5}$$

$$Em(s) = K_es\theta(s) \tag{6}$$

$$T = K_ti \tag{7}$$

$$T(s) = K_tI(s) \tag{8}$$

To find the response between angular position (θ) and input voltage (v) in time domain, the Laplace transfer function by combination of Eq.2, 4, 7, 8 is shown below:

$$\frac{\theta(s)}{v(s)} = \frac{K_t}{bR_s\left(\frac{K_eK_t}{bR}\right) + \tau_e\tau_ms^2 + \tau_ms + \tau_es + 1} \tag{9}$$

Note: τ_m is the mechanical time constant of motor ($\tau_m = \frac{Im}{b}$), τ_e is the electrical time constant ($\tau_e = \frac{L}{R}$).

Assumption: Because the electrical time constant (τ_e) is much smaller compared to the mechanical time constant of the motor (τ_m), term $\tau_e\tau_ms^2$ and τ_es are approximately equal to zero compared with other terms. Therefore, the equation will become:

$$\frac{\theta(s)}{v(s)} = \frac{K}{s(\tau s+1)} \quad \left(K = \frac{K_t}{bR+K_tK_e}, \ \tau = \frac{\tau_mRb}{KK_t+Rb}\right) \tag{10}$$

Due to the angular velocity (ω) is the first derivative of angular position related to time ($\omega(s) = s \cdot \theta(s)$). In this case, the transfer function of angular velocity and input voltage is:

$$\frac{\omega(s)}{v(s)} = \frac{k}{\tau s + 1} \tag{11}$$

According to the specification data of Motor C42-L50 winding code 10 Therefore, the constant K and overall time constant

TABLE I
SPECIFICATION DATA

Specification	Value
Torque sensitivity (K_t)	$0.1412\,\mathrm{N\,m\,A^{-1}}$
Back e.m.f. (K_e)	$0.1413\,\mathrm{V\,rad^{-1}\,s^{-1}}$
Rotor inertia (Im)	$6.3354 \times 10^{-4}\,\mathrm{kg\,m^{-2}}$
Mec. time constant (τ_m)	$0.0223\,\mathrm{s}$
Terminal resistance (R)	$0.7\,\Omega$
Friction coefficient $\left(\frac{Im}{\tau_m}\right)$	0.0285

τ can be calculated as:

$$K = \frac{K_t}{bR + K_tK_e} = 3.54 \tag{12}$$

$$\tau = \frac{\tau_mRb}{KK_t + Rb} = 0.0112 \tag{13}$$

In this case, the transfer function of angular velocity and angular displacement of motor with voltage input is:

$$\frac{\omega(s)}{v(s)} = \frac{K}{(\tau s + 1)} = \frac{3.54}{0.0112s + 1} \tag{14}$$

$$\frac{\theta(s)}{v(s)} = \frac{K}{s(\tau s + 1)} = \frac{3.54}{0.0112s^2 + s} \tag{15}$$

where 3.54 is the gain of the system.

According to the results show in Graphs.7, .9, .11, the response of output angular displacement will intersect with input and then overshoot to infinity (undesirable behaviour), which means these open loops cannot meet the requirement to control the angular displacement of the motor. Under this circumstance, the unit feedback closed loop with a gain of H
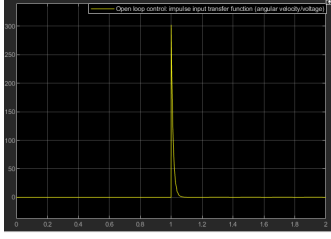
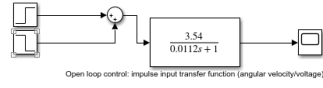Fig. 1. Results analysis: the response can reach desired target exponentially without oscillation.

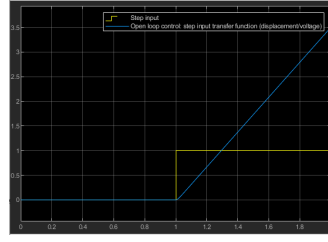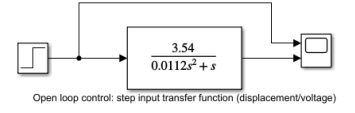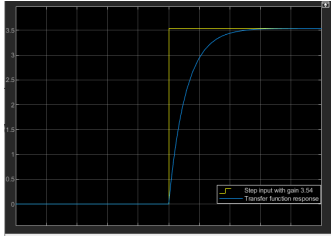Fig. 2. Model of open loop transfer function of angular velocity with impulse input.

Fig. 3. Results analysis: the response can reach desired step input exponentially without oscillation.

Fig. 4. Model of open loop transfer function of angular velocity with step input.

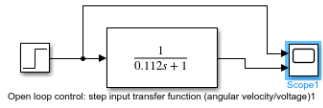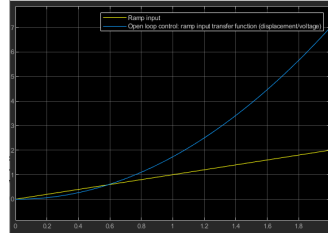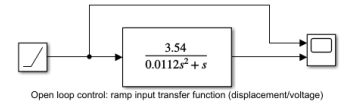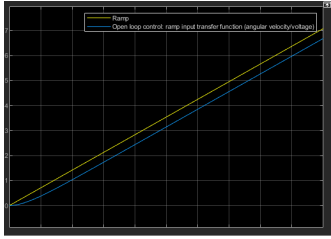Fig. 5. Results analysis: the response cannot reach the desired input and has a steady state lag.

Fig. 6. Model of open loop transfer function of angular velocity with ramp input.

Fig. 7.

Fig. 8. Model of open loop transfer function of angular displacement with impulse input.

Fig. 9.

Fig. 10. Model of open loop transfer function of angular displacement with step input.
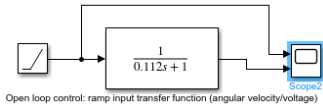

Fig. 11.

Fig. 12. Model of open loop transfer function of angular displacement with ramp input.

is required to overcome this undesirable behaviour of open loop control system for angular velocity. The formula for the feedback loop is:

$$G(s) = \frac{\theta(s)}{v(s)} = \frac{3.54}{0.0112s^2 + s} \tag{16}$$

$$H(s) = H \tag{17}$$

$$G'(s) = \frac{G(s)}{1 + G(s) \cdot H(s)} \tag{18}$$

Therefore, the new transfer function is:

$$G'(s) = \gamma \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \tag{19}$$

$$G'(s) = \frac{316.07H}{H\left(s^2 + 2 \cdot 44.64 \cdot \sqrt{\frac{1}{316.07H}} \cdot \sqrt{316.07H}s + 316.07H\right)} \tag{20}$$

Note: $\gamma = \frac{1}{H}$, $\zeta = 44.64\sqrt{\frac{1}{316.07H}}$ and $\omega_n = \sqrt{316.07H}$

In this case, we can use ζ to determine the response wanted as overdamped, critically damped, underdamped or undamped to control the loop without undesired behaviour, and the table below shows the range of H for the overdamped, critically damped, underdamped or undamped response cases: Therefore, the transfer function of unit feedback loop is:

TABLE II
TRANSFER FUNCTION RESPONSES

Response	Value of ζ	Value of gain H
Critical damping	$\zeta > 1$	$0 < H < 6.304$
Overdamped	$\zeta = 1$	$H = 6.304$
Underdamped	$0 < \zeta < 1$	$H > 6.304$
Undamped	$\zeta = 0$	$H = \infty$

$$G(s) = \frac{3.54}{0.0122s^2 + s} \tag{21}$$

The unit feedback loop servo control system is type 1 system ($P = 1$), which means there is no steady state position error for the step input and there must be a steady velocity lag for the ramp input.

For the impulse input, we want the response reach desired value fast without any oscillation, which refers to the overdamped case. Hence, H=6.304 for the impulse input situation. The Simulink model of impulse input of closed loop transfer response is shown in Graph 7. For the step input situation, underdamped response has the fastest speed to reach the desired input value. In our case, the desired overshoot percentage is 5%, which means our servo system can reach desired position within 5% overshoot. Therefore, gain H can be calculated by:

$$\frac{A}{100} = e^{\frac{-\zeta\pi}{\sqrt{1-\zeta^2}}} = 0.05, \ \zeta = 0.69 \tag{22}$$

In this case, $H = 13.24$ for the step input situation according to ($\zeta = 44.64\sqrt{\frac{1}{316.1H}}$). Simulink model of step input of closed loop transfer response is shown in Fig.15. For the ramp input, there must be a steady state velocity lag due to type 1 system ($P = 1$), therefore, for better performance, we set the lag to $0.1\,\text{s}$. In this case, H can be calculated by the formula:

$$e = \frac{a}{kv} \ (kv = \lim_{s\to 0} G(s)H(s)) \tag{23}$$

Hence, $H = \frac{1}{e\times3.54} = 2.82$ for $0.1\,\text{s}$ velocity lag, which means the response is critical dump case. Simulink model of ramp input of closed loop transfer response in Fig.17
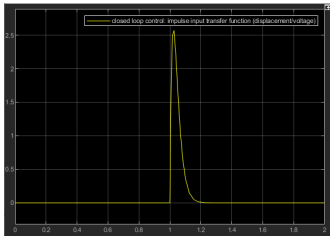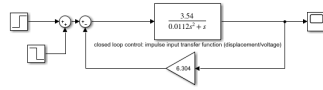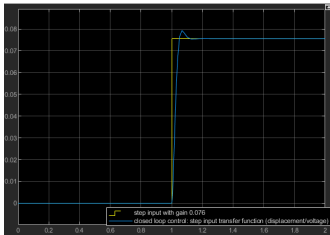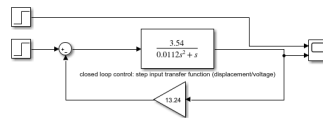

Fig. 13.

Fig. 14.
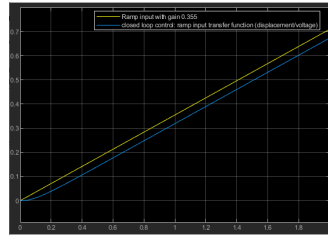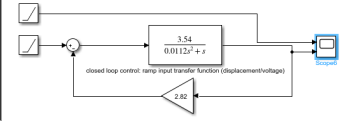

Fig. 15.

Fig. 16.

Fig. 17.

Fig. 18.

QUESTION 2

2.1 - Create a light-sensing circuit...

Building the voltage divider circuit in TinkerCAD, the light level slider was moved left and right and the range of voltages displayed were $1.49\,\text{V}$ - $0.504\,\text{V}$. Using the formula:

$$V_{meter} = \frac{R_{LDR}}{R_R + R_{LDR}} \times V_{in} \tag{24}$$

$$R_{LDR} = \frac{V_{meter}R_R}{V_{in} - V_{meter}} \tag{25}$$

$$R_{LDR,max} = \frac{1.49 \times 1000}{1.5 - 1.49} = 149 \times 10^3\,\Omega \tag{26}$$

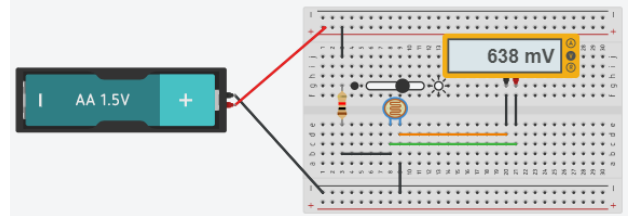$$R_{LDR,min} = \frac{0.504 \times 1000}{1.5 - 0.504} = 506.02\,\Omega \tag{27}$$

Fig. 19. LDR on breadboard.

2.3 - Investigate linear position transducers...

To measure the position of the printer head, we can utilise a variety of sensors. Some technologies that have been used to measure the position of the printer head include [2]:

A microswitch placed at the "home" position. This is actuated when the printer head is at the origin coordinate, hence the printer head is calibrated to this position. This method may become unreliable because if the printer head is knocked during a print, it must reset back to the home position to recalibrate, which must be done manually.

A laser system can also be used. The time-of-flight is calculated for the laser beam. This changes depending on the distance from the origin point to the printer head. This method is more accurate and also self correcting i.e. during a print, a misalignment can be corrected. However, they must also be checked and calibrated in cases where they may be a systematic error.

Hall Effect sensors consist of a thin piece of semiconductor material with a current being passed through it. When this

is placed in a magnetic field, a force is exerted on the charge carriers, producing a voltage proportional to the strength of the magnetic flux density [3]. Hence, placing hall effect sensors along the rails of the 3D printer and placing a small magnet on the printer head itself will allow for the position of the printer head to be calculated to a high degree of accuracy. When the printer head approaches a sensor, the voltage will increase linearly to a peak (when the head is directly overhead) and then decrease linearly again. This allows for the position the be calculated to extremely high accuracies. Typically, the error values for a hall effect sensor are much less than one percent [4].

2.4 - Design a linear encoder...

Considering a FDM (fused deposition modelling) PLA (polyactic acid) 3D printer, typical printing speeds [1] for a medium end model are $100\,\mathrm{mm\,s^{-1}}$. The accuracy for a good printer would be around $\pm 0.2\,\mathrm{mm}$. The sampling rate of an Arduino's analogue input port is roughly $9600\,\mathrm{Hz}$, as tested here. This would allow us to have a stripe with a blocked-transparent pattern in $0.5\,\mathrm{mm}$ blocks (i.e. $0.25\,\mathrm{mm}$ of blocked out and then $0.25\,\mathrm{mm}$ of transparent). Assuming our sample rate is stable at $9600\,\mathrm{Hz}$, if the head travels at $100\,\mathrm{mm\,s^{-1}}$, we will traverse 192 patterned blocks. This relates to 48 samples per block traversed. This should provide adequate information to measure the intensity of light from the LED. When the printer head moves along the encoder, the intensity of the light reaching the LDR will form a sinusoidal intensity signal. We can utilise a comparator to remove noise from our signal by inverting one of the outputs and running both signals to the inputs of an op-amp. This will be beneficial in reducing error signals, a necessary requirement for a high-accuracy system.

For the signal sent to the Arduino, we can measure which direction the head is moving in and the position of the head by using a quadrature sine/cosine signal. This is where we take the voltage signal from our LDR (a sinusoid) and compare it with a signal with a $\frac{\pi}{2}$ phase shift. Plotting the intensity of light on perpendicular axis' on an xy oscilloscope will produce a plot called a Lissajous figure. Under perfect conditions, our Lissajous figure will be a circle centred on the origin. The radius of the Lissajous is based on the amplitude and the direction in which the point is traced relates to whether our linear encoder is being read in the positive or negative direction. However, we may see that our Lissajous is not a perfect circle and this can be amended with trimming the signal and calibrations. The Lissajous figure tells us the position of the printer head, but we will need an absolute reference in order for our printer head to know where it is, e.g. utilising hall effect sensors. We can count the number of times our Lissajous traverses 2π times in the oscilloscope as the head moves along the rails. We can use information such as whether the axis' was crossed clockwise or anticlockwise as a reference as to whether the number should be increased or decreased. As we also know the size of our stripe pattern, we can derive the position of the head.
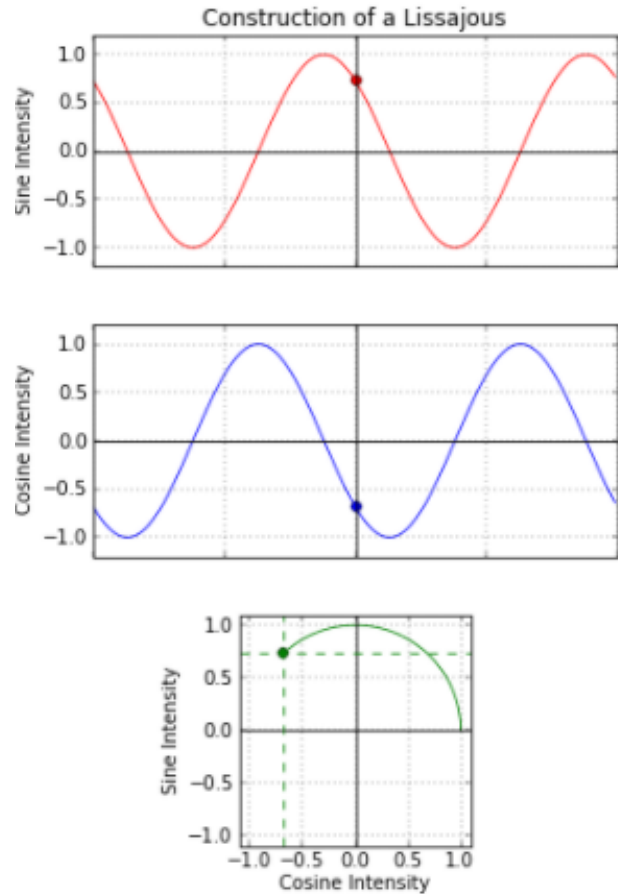

Fig. 20. Lissajous plot

Conclusion

According to our Simulink modelling and TinkerCAD circuits, we can successfully control the printer head and measure its position to a suitable standard. After researching the status quo, we believe that the control methods (e.g. closed loop displacement control system) we have devised are acceptable for use in a 3D printer.

References

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