

UNIVERSITY OF COLORADO - BOULDER

ASEN 3200 ATTITUDE AND ORBITAL MECHANICS

Lab 3 - Section 013

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I. Abstract

This experiment focuses on the design and implementation of active control systems to cause a spacecraft model to follow a reference trajectory. The reference trajectory is defined by one degree of angular freedom. The closed loop tracking behavior is achieved by designing a feedback controller with proportional and derivative gains to achieve a desired system response, with respect to the spacecraft's moment of inertia and rigid body properties.

II. Preliminary questions

1. Description of Spacecraft

The moment of inertia can be calculated using $\tau = I \alpha$, τ being the applied torque, and α being the angular rate of the spacecraft in rad/s with respect to time. Experimentally, we can apply a torque to the spacecraft perpendicular to its axis of free motion, and capture data for its angular rate at the time we apply the torque. Using this, we can easily calculate our value for the moment of inertia by computing $I = \tau \alpha$.

2. Block Diagram

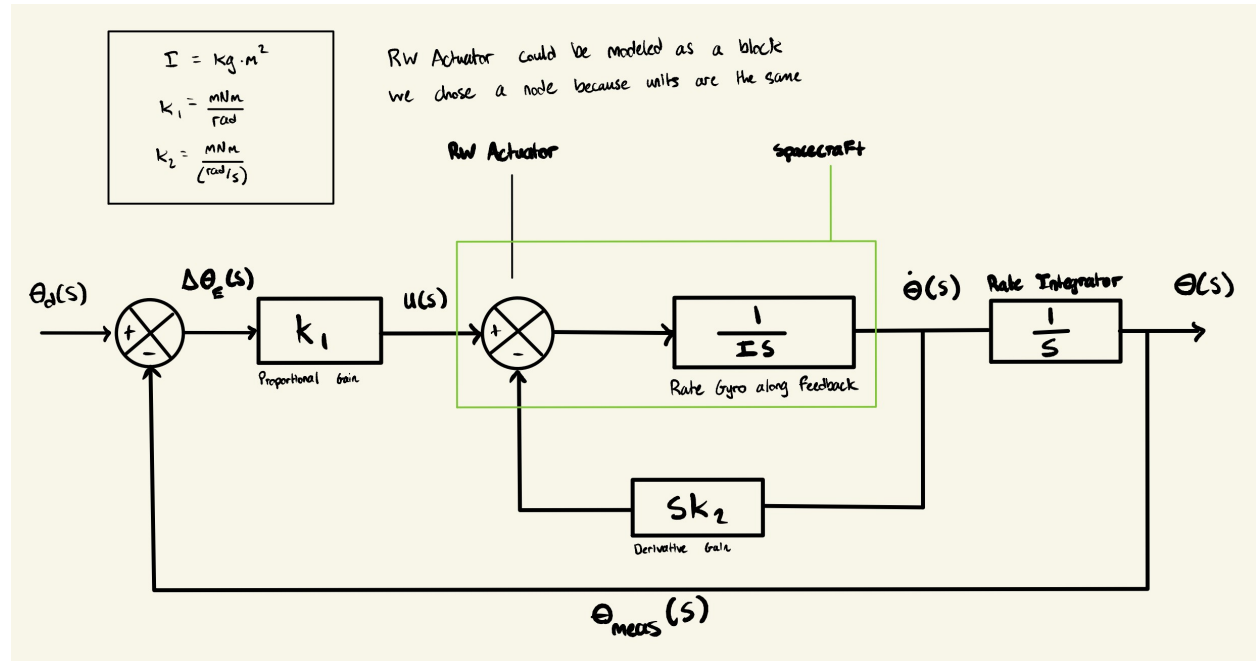


Fig. 1 Single DOF Control Block Diagram

3. Poles and gains of closed loop system

Using our given values, we can calculate the damping coefficient and the natural frequency. We can calculate our damping coefficient using our max overshoot equation, which is given by:

$$M_p = \exp\left(-\frac{\zeta}{\sqrt{1 - (\zeta)^2}}\pi\right) \quad (1)$$

We can re-arrange to solve for ζ , which provides us with:

$$\zeta = \frac{-\ln(0.1)}{\sqrt{(\pi)^2 + \ln(0.1)^2}} \quad (2)$$

Evaluating this equation gives us a damping ratio of $\zeta = 0.591$.

Next, we can evaluate the natural frequency by using the percent overshoot equation, which has the form of:

$$P = \exp(-\zeta \omega_n t_s) \quad (3)$$

Re-arranging to solve for ω_n , we get:

$$\omega_n = \frac{\ln(0.1)}{\zeta t_s} \quad (4)$$

Evaluating, we get a natural frequency of $\omega_n = 3.379$ rad/s.

Subsequently, if we evaluate our transfer function from the above block diagram, we get that our damping coefficients are:

$$I \omega_n^2 = K_1 = 11.4461I \quad (5)$$

$$2\zeta \omega_n I = K_2 = 4I \quad (6)$$

III. Experiment and Analysis

A. Inertia

Listed below are our average values of the moment of inertia for each trial.

- Case 1: $I = 9.4687 \text{ [kgm}^2\text{]}$
- Case 2: $I = 8.3246 \text{ [kgm}^2\text{]}$
- Case 3: $I = 7.5829 \text{ [kgm}^2\text{]}$
- Case 4: $I = 7.8828 \text{ [kgm}^2\text{]}$
- Average I : $8.2998 \text{ [kgm}^2\text{]}$

These values were derived in MATLAB by utilizing the known equation for the torque, $T = [I]\alpha$ and solving for I . Each case represents the inertia for a particular amount of torque and corresponding angular acceleration, represented as the slope of the angular velocity data. Case 1 is 4 mNm of torque, and Case 2 is 6 mNm, of torque, up to 10 mNm of torque. It is important to note that each trial is simulated for the same amount of time. The only difference between them is the inputted value of torque. We see from our values that the higher the torque, the higher the magnitude of inertia. However, the value for inertia seems to stabilize towards 8.2998 kgm^2 . This value is positive because the reaction wheel angular velocity convention is negative for our dataset. Taking into account the maximum torque of the spacecraft to be 20 mNm, it is possible that these values are stabilizing as such since the spacecraft can only handle so much load.

Sources of error could range between human and mechanical errors. Depending on how the data is parsed and collected, we could face some human error with slight deviations in results. Additionally, the machine has an inherent error with things such as mechanical friction.

B. Control Design

Our closed-loop step response needed to have a 5% settling time of less than 1.5s, with an overshoot of less than 10%. From their respective equations, we found that: $\zeta = 0.591$, $\omega_n = 3.379 \frac{\text{rad}}{\text{s}}$. Our proportional and derivative gains were calculated by solving the denominator of the closed-loop transfer function, with respect to the moment of inertia. Once these values were found, we set the denominator equal to zero and solved for the s-values, which are the poles.

The poles of our system are: $\{-2.0000 + 2.7288i, -2.0000 - 2.7288i\}$. Both real components of our poles are negative, therefore we can assume the spacecraft is stable when influenced by our closed-loop system. The analytical step and impulse responses of our closed-loop system can be seen below:

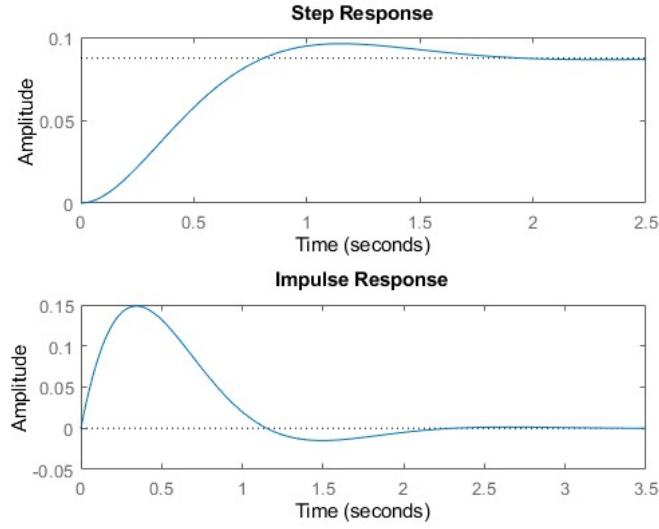


Fig. 2 Analytical Step and Impulse Response

C. Control Results

The experimental results from $K1 = 95$ and $K2 = 33.1667$ are shown below. We were given a reference angle of 0.5 rad, shown by both the horizontal line and the reference square wave. The integrated gyro output is referring to our measured square waveform. This measured waveform is the behavior of the spacecraft when provided the aforementioned gains and reference values.

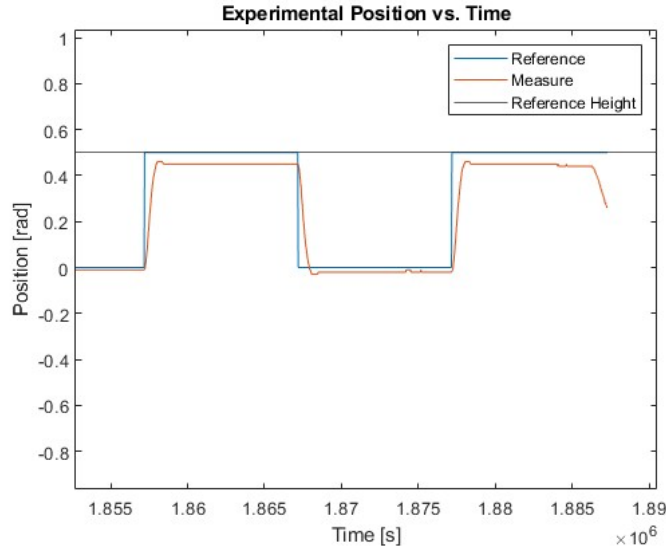


Fig. 3 Experimental Position vs. Time Response

When referencing this plot to our step reference, we see that the measured vs reference position data follows closely to our analytical step response. We see a relative undershoot when we reach our first reference angle, and immediately see an overshoot shortly after we return to an angle of zero, while subsequently reaching just short of zero when our measured data settles. Overall, we do see similar behavior when referenced to our analytical data. Within the short period of overshoot, we reach an overshoot of roughly 2%, which falls within our acceptable range.

D. Control Analysis/Comparison

By inspection, we can see that the settling time of the step response is 1.24s when the response reaches 5% of the steady-state value, which is less than the design maximum of 1.5s. The maximum overshoot relative to the response-steady state is 2.22%, which is less than the design maximum of 10%.

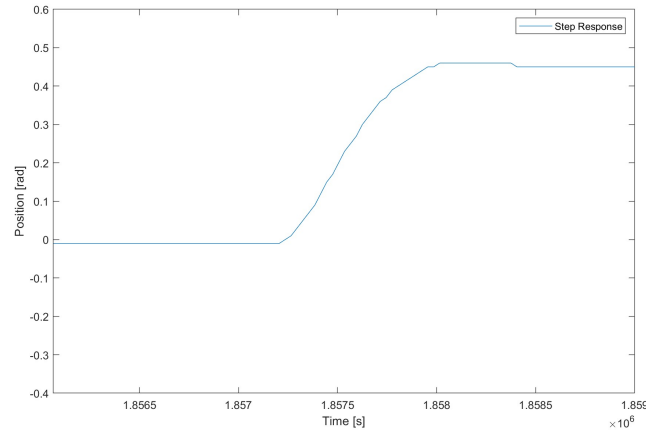


Fig. 4 Reaction Wheel Position / System Step Response

E. Disturbance Response

Varying the proportional gain within the experiment, we notice a trend when increasing or decreasing the value between 0 and 200. At a value of 50, we saw a relatively fast response to the disturbance, with the rotator returning back to zero within a couple of seconds. Increasing K_1 provides us with a higher time to decay to zero, resulting in a longer time of oscillation.

If we provide a derivative gain between 0 and 30, we see that our motion ceases a linear motion of oscillation towards zero, and now behaves quadratically, or rather exponentially, towards zero. With the addition of the derivative gain in addition to a linear gain, we see a rapid decrease in oscillation time, with the amplitude of each oscillation exponentially decreasing with time. We know to expect this response since an increase of K_1 increases the relative error or "stiffness" our system experiences, while K_2 represents the damping of our system. Varying values for K_1 and K_2 , we can see a trend in different system conditions, such as being critically damped (determined from our system) or underdamped (K_2 equal to zero).

F. PID vs. PD empirical comparison

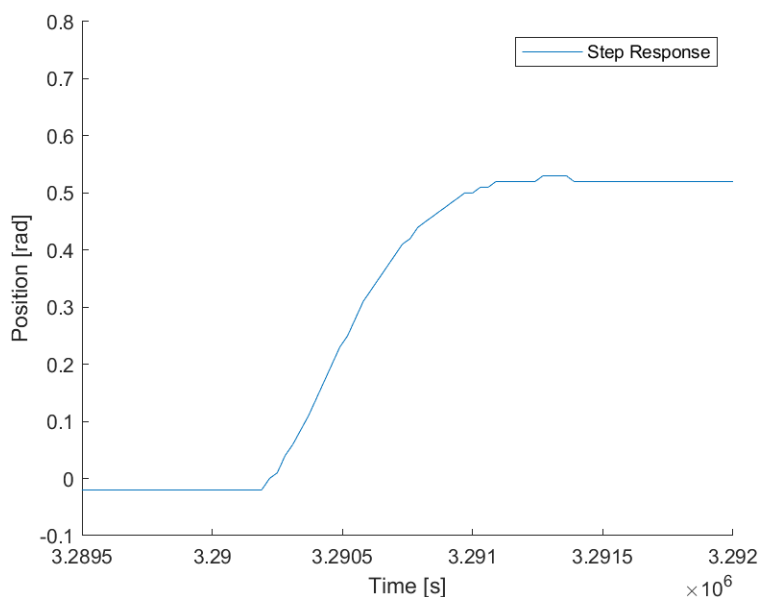


Fig. 5 Analytical Step and Impulse Response with Integral Gain

By inspection, the settling time to reach within 5% of the settling time is 1.2 seconds, which is also below our max design requirement of 1.5 seconds. Our max overshoot relative to the steady state is 1.92%, also less than the design max of 10%. Based on this, we can see that the integral gain removes some long-term errors. In comparison to the PD controller, we have a shorter setting time and a smaller max overshoot, which shows that the integral gain does in fact reduce the systems error relative to the steady state.

IV. Conclusion and Recommendations

This experiment focused on the design and implementation of active control systems with the intention of causing a spacecraft model to follow a reference trajectory. The closed loop tracking behavior was achieved by designing a feedback controller with gains that we calculated by determining the spacecraft's experimental moment of inertia. The proportional and derivative gains were achieved by tailoring the system's closed loop transfer function to the required damping coefficient and natural frequency we desired in our reaction response. In synthesis, we achieved a new depth of knowledge of generating methods and models that allow spacecraft to autonomously perform attitude corrections.

V. Appendix

Participation Report

Name	Contribution Factor	Signature
Hayden Gebhardt	Code and Report	HG
Alan Jaffe	Code and Report	AJ
Michael Lauver Lopez	Code and Report	MLL

We would like to acknowledge the TAs and the Orbital Lab Teaching Team that helped us answering conceptual questions and any misunderstandings we had regarding the lab.

References

The initial version of this lab was created by Prof Dale Lawrence. Modifications have been made by P. Axelrad, H. Schaub, D. Scheeres, Bobby Hodgkinson, Trudy Schwartz.

A. MATLAB Code

```
1 % Hayden Gebhardt, Alan Jaffe, Michael Lauver Lopez
2 % ASEN 3200 – Lab 3
3
4 %% Housekeeping
5
6 clc; clf;
7 clear all
8
9 %l rpm → 0.1047 rad/s
10 %l = L/alpha
11
12 data_4 = load("5sec4torque_Lab3GROUP17");
13 data_4 = data_4(10:end, :);
14 data_6 = load("5sec6torque_Lab3GROUP17");
15 data_6 = data_6(10:end, :);
16 data_8 = load("5sec8torque_Lab3GROUP17");
17 data_8 = data_8(10:end, :);
18 data_10 = load("5sec10torque_Lab3GROUP17");
19 data_10 = data_10(10:end, :);
20
21 time1 = data_4(:, 1) / 1000;
22 time2 = data_6(:, 1) / 1000;
23 time3 = data_8(:, 1) / 1000;
24 time4 = data_10(:, 1) / 1000;
25
26 %% torque = 4 mNm
27
28 t_4 = data_4(:, 2); % mNm
29 omega_4 = data_4(:, 3) * 0.1047; % RPM → rad/s
30 line_4 = polyfit(time1, omega_4, 1);
31 alpha_4 = line_4(1);
32
33 moi1 = t_4 / alpha_4;
34 moi1(any(isinf(moi1), 2), :) = [];
35 moi1final = rmmissing(moi1);
36 MOI1 = mean(moi1final);
37
38 %% torque = 6 mNm
39
40 t_6 = data_6(:, 2); % mNm
41 omega_6 = data_6(:, 3) * 0.1047; % RPM → rad/s
42 line_6 = polyfit(time2, omega_6, 1);
43 alpha_6 = line_6(1);
44
45 moi2 = t_6 / alpha_6;
46 moi2(any(isinf(moi2), 2), :) = [];
47 moi2final = rmmissing(moi2);
48 MOI2 = mean(moi2final);
```



```

49
50 %% torque = 8 mNm
51
52 t_8 = data_8(:, 2); % mNm
53 omega_8 = data_8(:, 3) * 0.1047; % RPM -> rad/s
54 line_8 = polyfit(time3, omega_8, 1);
55 alpha_8 = line_8(1);
56
57 moi3 = t_8/alpha_8;
58 moi3(any(isinf(moi3),2), :) = [];
59 moi3final = rmmissing(moi3);
60 MOI3 = mean(moi3final);
61
62 %% torque = 10 mNm
63
64 t_10 = data_10(:, 2); % mNm
65 omega_10 = data_10(:, 3) * 0.1047; % RPM -> rad/s
66 line_10 = polyfit(time4, omega_10, 1);
67 alpha_10 = line_10(1);
68
69 moi4 = t_10/alpha_10;
70 moi4(any(isinf(moi4),2), :) = [];
71 moi4final = rmmissing(moi4);
72 MOI4 = mean(moi4final);
73
74 %% Average MOI
75
76 MOI_sc = -(MOI1 + MOI2 + MOI3 + MOI4) / 4
77
78 %% Gain calculations
79
80 k1 = 11.4461 * MOI_sc
81 k2 = 4 * MOI_sc
82
83 %%
84
85 %% Closed-loop response
86
87 sys = tf(1, [1 (k2/MOI_sc) (k1/MOI_sc)])
88
89 subplot(2,1,1)
90 step(sys)
91 subplot(2,1,2)
92 impulse(sys)
93
94 pole(sys)
95 h = pzplot(sys);
96
97 %% Control Plots
98
99 clc
100 clear
101 close
102

```

```

103 %I rpm → 0.1047 rad/s
104 %I = L/W
105
106 %% torque = 4 mNm
107
108 dataControl = load("K1_95_K2_33control20sLABTASK4");
109
110 time = dataControl(:,1); %ms
111 refpos = dataControl(:,2); %rad
112 measpos = dataControl(:,3); %rad
113 current = dataControl(:,4); %Amp
114
115 figure()
116 plot(time, refpos)
117 hold on
118 plot(time, measpos)
119 legend('Reference', 'Measured')
120 axis([1856065.000 1887290.000 -1 1])
121 title('')
122 xlabel('Time [s]')
123 ylabel('Position [rad]')
124
125 figure()
126 plot(time, measpos)
127 legend('Step Response')
128 axis([1856065.000 1859000.000 -0.4 0.6])
129 title('')
130 xlabel('Time [s]')
131 ylabel('Position [rad]')

```