

UNIT-II**Highway Geometric Design****Introduction:**

Highway geometric design deals with the dimensions and layout of the visible features of a road such as horizontal alignment, vertical alignment, sight distance etc.,

Proper geometric design will help to reduce accidents.

Role of geometric design

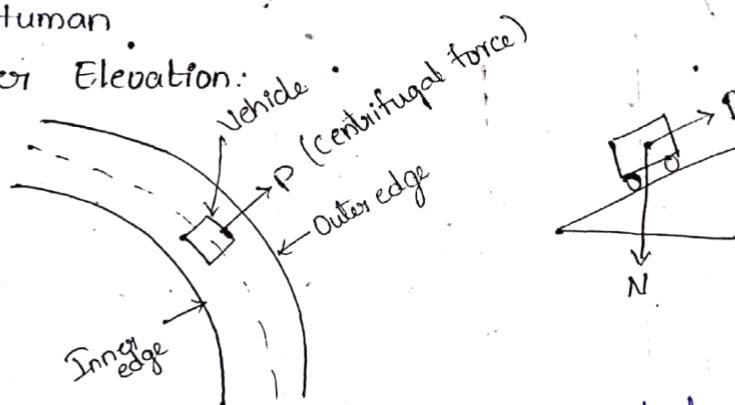
To provide comfort and safety,

Economical facilities

To minimize environmental impacts

factors affecting on Geometric design

1. Design speed
2. Traffic
3. Topography
4. Vehicle properties (width of vehicle, weight of vehicle)
5. Environmental factors
6. Human

Super Elevation:

The Transverse slope is provided at horizontal curve to counteract the centrifugal force by raising the outer edge of the pavement with respect to inner edge throughout the length of the horizontal curve is

Called Super elevation UNIT →

Elements Of Geometric Design

1. Cross sectional elements

2. Site distance (stopping side Distance & overtaking side Distance)

3. Horizontal alignment

4. Vertical alignment

5. Intersections

Cross Sectional elements:

1. Pavement Surface characteristics

2. Carriage way / width of pavement

3. Cross slope/Camber

4. Traffic separator / Median

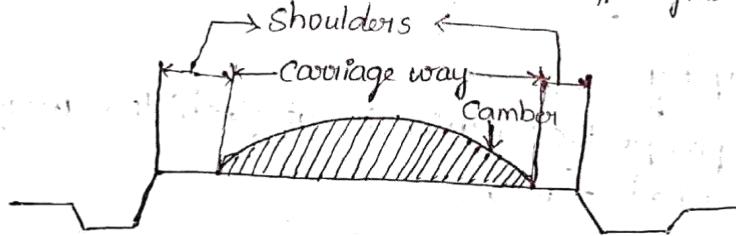
5. Kerbs

6. Road margins

7. formation width

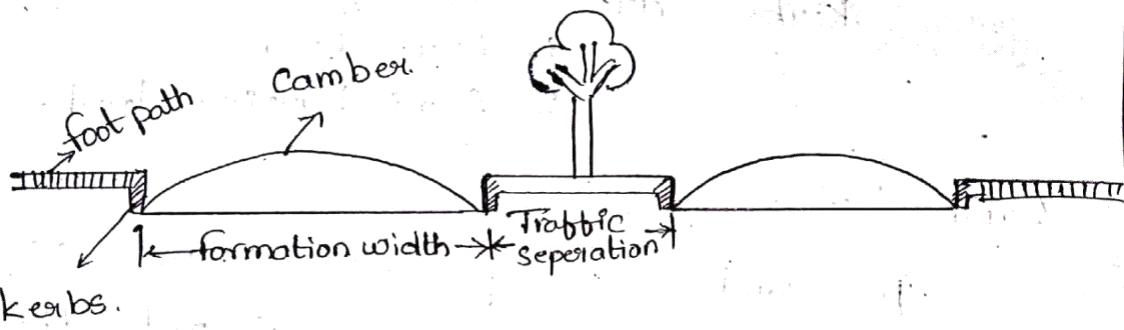
Right of way ← →

Margins ← → formation width ← → Margins →



Pavement Surface characteristics:

1. Unevenness
2. Light reflection on road
3. Drainage



Carriage way:

The width of pavement on which vehicles travel is called carriage way.

For single lane : 3.75m

for Two lanes : 7.5m

(The width of vehicle is 2.5m)

Shoulders:

shoulders are provided beside the

carriage way.

In emergency cases the shoulders are used as emergency lane for ambulance, fire engines e.t.c.,

As per IRC the minimum width of shoulder is 2.5m.

Formation width:

The width of pavement including shoulders is called formation width

i.e., formation width = Carriageway + shoulder way

kerb: The kerb is provided between footpath and pavement.

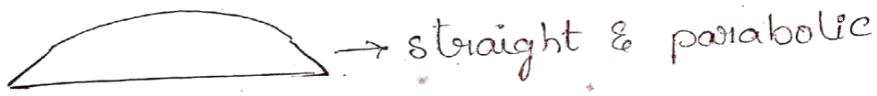
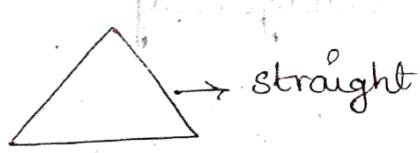
Kerbs are two types

1. Low kerbs

2. Semi-barrier kerbs

Camber cross-slope:

A camber is provided on the top surface of the road to drain out the rain water on the road surface.



Traffic separator (or) Median:

The Traffic separator is provided between two opposite lanes.

The Traffic is controlled by providing traffic separators.

Road Margins:

shoulders

footpath

Cycle track

Parking lane

embankment slope etc.,

Right of way:

The total width between "boundary lines" beside the road is called "Right Of Way". i.e., $ROW = \{Carriage\} + \{shoulders\} + \{Margins\}$.

Sight Distance

The actual distance along the road surface which a driver from a specified height above the carriage way has visibility of stationary objects or moving objects is called sight distance.

These are classified into two types.

1. Stopping sight distance (SSD) (8)

2. Absolute minimum sight distance (8)

Non-passing sight distance.

2. Overtaking sight distance (OSD) (8)

Non-passing sight distance

Apart from above two additional sight distances are considered by IRC

a) Intermediate sight distance (ISD)

= Two times of stopping sight distance

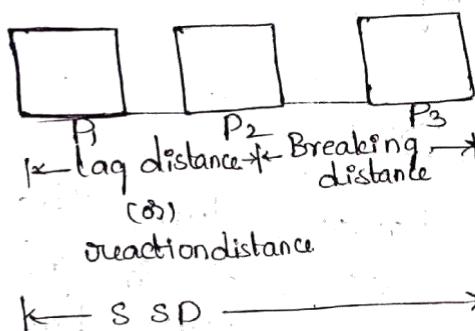
When overtaking sight distance can't be provided to given limited overtaking opportunity for fast vehicles. The intermediate sight distance is provided.

b) Head light sight distance

Distance visible to driver during the

night drivers under the vehicle headlights.
this is critical at ascending gradients

1. Stopping sight distance



It is defined as the distance needed for drivers to see an object on the road way and bring their vehicles to safe stop before colliding with the object is called stopping sight distance.

As per IRC height of drivers eye

$$= @ 1.2 \text{ m}$$

$$\left. \begin{array}{l} \text{height of object about road} \\ \text{Surface} \end{array} \right\} = 0.15 \text{ m}$$

Total Reaction time: (t)

$$t = \text{Perception time} + \text{Break reaction time}$$

As per IRC,

The total reaction time is split into four types (PIEV theory)

P - Perception (time to see an object)

I - Interpretation Intelection (time to understand about the object)

E - Emotion (time to react)

V - Violation (time to initiate the action)
i.e., for applying brakes

As per IRC

The total reaction time = 2.5 sec

Lag distance or Reaction distance

The distance covered by a vehicle during total reaction time is called lag distance

Breaking distance

The distance travelled (or) covered after application of breaks to a dead stop position is known as Breaking distance

Lag distance

$L = \text{Velocity (speed)} \times \text{time (reaction time)}$

$$\text{i.e. } L = v \times t$$

where v = velocity of vehicle in m/sec

| |
|---|
| Speed = $\frac{\text{Dist}}{\text{time}}$ |
| $D = v \times t$ |

$$t = ?$$

$$L = 0.278 v t \quad (v = 0.278 V)$$

where V = speed in KMPH

Breaking distance

On a level ground it is obtained by equating workdone to stopping a vehicle to kinetic energy loss.

Let w = weight of the vehicle

f = longitudinal co-efficient of friction (range 0.35 to 0.4)

work done = frictional force \times Breaking distance

$$= f \times B$$

$$= f \times w \times B \rightarrow ①$$

$$\begin{aligned} K.E &= \frac{1}{2} m v^2 \\ &= \frac{1}{2} \times \frac{w}{g} v^2 \end{aligned}$$

$$\left\{ \begin{array}{l} w = m g \\ m = \frac{w}{g} \end{array} \right.$$

$$K.E = \frac{w v^2}{2g} \rightarrow ②$$

from equation ① & ②

work done = kinetic energy

$$f \cdot \varphi B = \frac{w v^2}{2g}$$

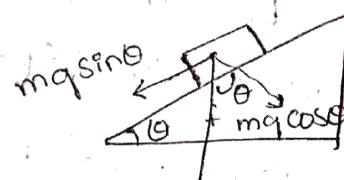
$$B = \frac{v^2}{2g f}$$

$$v = 0.278 \sqrt{f}$$

$$B = \frac{(0.278 \sqrt{f})^2}{2 \times 9.81 f}$$

$$B = \frac{v^2}{253.8 f}$$

Effect of gradient on breaking distance. (B)



$$K.E = w \cdot D$$

$$\frac{1}{2} m v^2 = \underbrace{\left[f \times mg (\cos \theta) + mg \sin \theta \right]}_{\text{friction force}} \times B \quad w = mg$$

$$\frac{1}{2} m v^2 = \cancel{m g} B (f \cos \theta + \sin \theta)$$

(If θ is small, $\cos \theta = 1$, $\sin \theta = \tan \theta$)

$$\frac{v^2}{2} = g B [f + \tan \theta]$$

$$B = \frac{v^2}{2g(f + \tan \theta)}$$

Note: In case stopping sight distance provided is not sufficient, a moving vehicle will collide with the object or another parked vehicle. The principle to be used in this analysis is as follows. for Breaking distance.

i.e., Kinetic energy lost = Work done in Skidding before collision.

$$\frac{1}{2} m v^2 = f \times B$$

$$\frac{1}{2} m [v_{fi}^2 - v_f^2] = f \times w \times B$$

where v_i = initial velocity before application of breaks

v_f = final velocity before collision

| S.No | Surface | SSD (v in m/s) tag distance + Breaking | SSD (V in KMPH) distance |
|------|---|--|---|
| 1. | level surface | $vt + \frac{v^2}{2gf}$ | $0.278vt + \frac{V^2}{254f}$ |
| 2. | Inclined surface (n.v. of Gradient) | $vt + \frac{v^2}{2g(F \pm 0.01n)}$ | $0.278vt + \frac{V^2}{254f(F \pm 0.01n)}$ |
| 3. | level surface with breaking efficiency (η) | $vt + \frac{v^2}{2g \cdot f \eta}$ | $0.278vt + \frac{V^2}{254f \eta}$ |

Note: For single lane \rightarrow 1-way traffic
 $SSD = SSD$
 \rightarrow 2-way traffic $SSD_1 = 2SSD_2$
 $(SSD_2 = SSD \text{ calculated with})$
 single lane

for two lanes with 6:12

\rightarrow 1-way traffic $SSD = SSD$
 \rightarrow 2-way traffic $SSD' = SSD$

Q) If the coefficient of longitudinal friction is 'f', The retardation (acceleration) developed during the process of breaking.

$$\text{Equation : } a = fg$$

Eq : Assuming a longitudinal co-efficient of friction to be 0.4, The resulting retardation of a vehicle being brought to a stop is ?

Given : Coefficient of friction = 0.4
 we know that retardation $a = fxg$

$$g = 9.81 \text{ [acceleration due to gravity]}$$

$$g = 9.81 \text{ m/sec}^2$$

$$a = 0.4 \times 9.81$$

$$= 3.92 \text{ m/sec}^2$$

standard value

(1g) gravitational

Determine the minimum stopping sight distance of a design speed of 50 kmph with longitudinal co-efficient of friction 0.36 when

- Two lane two way traffic
- Single lane two way traffic

$$S.S.D = Vt + \frac{V^2}{2gF}$$

Given that

$$\begin{aligned} V &= 50 \times \frac{1000}{3600} \\ &= 50 \times \frac{10}{36} \\ &= 13.88 \text{ m/sec.} \end{aligned}$$

we know that
The total reaction time IRC

$$t = 2.5 \text{ sec.}$$

$$f = 0.36$$

$$g = 9.81 \text{ m/sec.}$$

$$\begin{aligned} S.S.D &= 13.88 \times 2.5 + \frac{13.88^2}{2 \times 9.81 \times 0.36} \\ &= 61.97 \text{ m} \end{aligned}$$

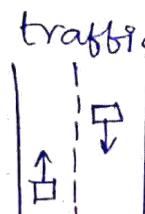
(or)

$$\begin{aligned} S.S.D &= 0.278 V t + \frac{V^2}{254 f} \\ &= 0.278 \times 50 \times 2.5 + \frac{50^2}{254 \times 0.36} \end{aligned}$$

$$S.S.D = 62 \text{ m}$$

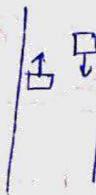
- Min SSD for two lane two way traffic

$$\begin{aligned} \text{Min SSD} &= \text{SSD} \\ &= 62 \text{ m.} \end{aligned}$$



b) Min SSD for single lane two way traffic

$$\begin{aligned} \text{Min SSD} &= 2 \times S.S.D \\ &= 2 \times 62 \\ &= 124 \text{ m} \end{aligned}$$



2. Calculate the min SSD on a highway at an ascending gradient 3% with a design speed of 40 KMPH

Given

ascending gradient $n = 3\%$

Coeff of friction $f = 0.35$ (range 0.35 - 0.4)

$$v = 40 \times \frac{1000}{60 \times 60} = 11.11 \text{ m/sec.}$$

$$t = 25 \text{ sec.}$$

$$g = 9.81 \text{ m/sec}^2$$

we know that

$$\begin{aligned} S.S.D &= vt + \frac{v^2}{2g(f+n)} \\ &= 11.11 \times 25 + \frac{11.11^2}{2 \times 9.81 (0.35 + 0.01(3))} \\ &= 44.33 \text{ m.} \end{aligned}$$

$$S.S.D = 0.278 Vt + \frac{V^2}{254(f+n)}$$

$$= (0.278 \times 40 \times 25) + \left[\frac{40^2}{254(0.35 + 0.03)} \right]$$

$$= 44.37 \text{ m.}$$

Determine a minimum stopping sight distance of a moving vehicle with a design speed of 60 KMPH with a break efficiency 80% and a co-efficient of friction is 0.4

Given

$$V = 60 \text{ KMPH} \quad \vartheta = 60 \times \frac{1000}{60 \times 60} = 16.67$$

Break efficiency $\eta = 80\%$

$$\begin{aligned} S.S.D &= \vartheta t + \frac{\vartheta^2}{2qf\eta} \\ &= 16.67(2.5) + \frac{16.67^2}{2 \times 9.81 \times 0.35 \times 0.80} = 42.18 \end{aligned}$$

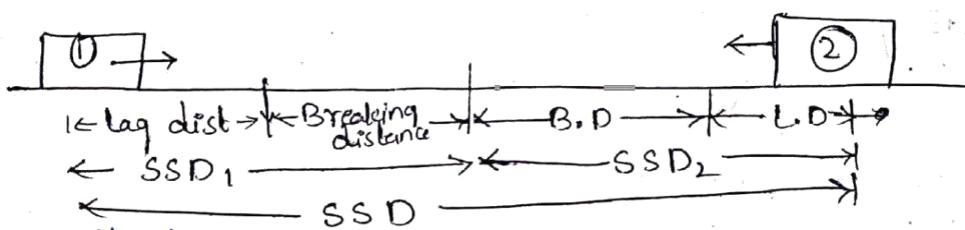
$$\left[\begin{array}{l} \text{From IRC} \\ t = 2.5 \\ f = 0.35 \\ q = 9.81 \end{array} \right]$$

(2)

$$S.S.D = 0.278 V t + \frac{V^2}{254 f \eta}$$

$$= 85.936 \text{ m.}$$

Calculate the minimum sight distance required to avoid a collision of two cars approaching from the opposite directions at 90 KMPH and 60 KMPH. Assume a reaction time of 2.5 sec, co-efficient of friction of 0.7 and break efficiency of 50% in both the cases.



Given that

Speed of first vehicle $V_1 = 90 \text{ KMPH}$

$$= 90 \times \frac{1000}{3600} = 25 \text{ m/s}$$

$$V_2 = 60 \text{ KMPH} = 60 \times \frac{1000}{3600} = 16.67 \text{ m/s}$$

SSD for 1st vehicle

$$SSD_1 = v_1 t + \frac{v_1^2}{2g f \eta}$$

$$= (25 \times 2.5) + \left[\frac{25^2}{2 \times 9.81 \times 0.7 \times 0.5} \right]$$

$$= 153.51 \text{ m.}$$

$$SSD_2 = v_2 t + \frac{v_2^2}{2g f \eta}$$

$$= [16.67 \times 2.5] + \left[\frac{16.67^2}{2 \times 9.81 \times 0.7 \times 0.5} \right]$$

$$= 82.12 \text{ m.}$$

S.S.D req to avoid collision b/w two vehicles

$$S.S.D = S.S.D_1 + S.S.D_2$$

$$= 153.51 + 82.12$$

$$= 235.63 \text{ m.}$$

* Breaking is applied on a vehicle which skids a distance of 16m before coming to stop. If the developed average co-efficient of friction b/w the tyres and pavement is 0.4 then calculate the speed of vehicle & before skidding.



$$Dist = 16 \text{ m} = 0.016 \text{ km.}$$

$$\text{Breaking dist} = \frac{V^2}{2g f}$$

0.016 = $\frac{V^2}{25.4 \times 0.35}$

$V = 11.27 \text{ m/sec}$

$$\text{Breaking dist} = \frac{V^2}{2g f}$$

$$13 + 16 = \frac{V^2}{2 \times 9.81 \times 0.35}$$

$$V = 11.27 \text{ km/PH}$$

Overtaking sight distance:

The minimum distance open to the vision of a driver of a overtaking vehicle intending to overtake slow vehicle with safety against the traffic in the opposite direction.

The min. dist required to overtake a vehicle against the vehicle coming from the opposite direction is called "asp" (O) "safe passing side distance."

As per IRC, the distance measure along the center line of a road which a driver with his eye level at 1.2m above the road surface can see the top of an object 1.2m above road surface



factors affecting on OSD

1. The speed of overtaking vehicle, overtaken vehicle (slow vehicle) and vehicle coming from the opposite direction.
2. Skill of driver, Reaction time of driver
3. Rate of acceleration of overtaking vehicle

Note:

- * Break efficiency does not effect on OSD
- * Gradient does not effect on OSD

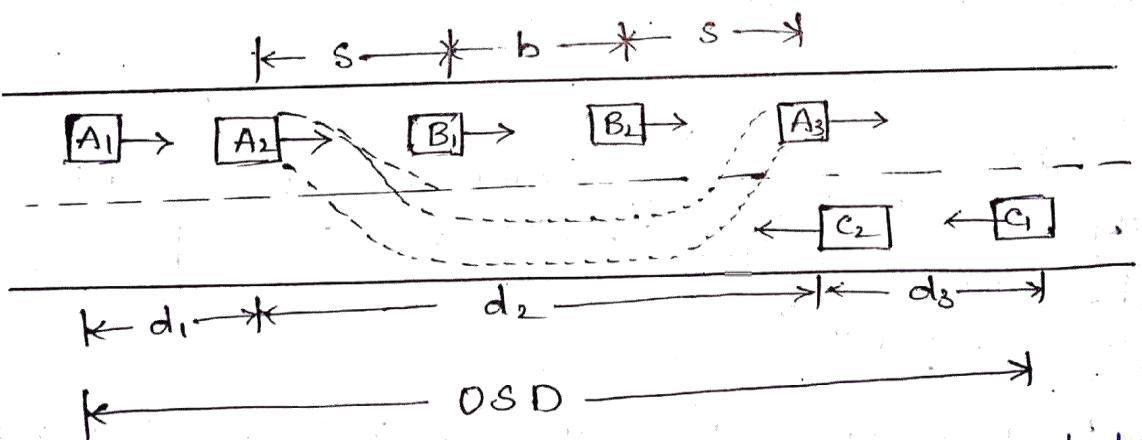
Calculation of "OSD"

Let, $\star = \text{constant}$

A = Overtaking Vehicle

B = Overtaken Vehicle (Slow Vehicle)

C = Opposite vehicle coming in the opposite direction.



d_1 = distance travelled by overtaking vehicle (A) during reaction time

from position A₁ to A₂



- d_2 = distance travelled by vehicle A from A_2 to A_3 during overtaking operation of vehicle A
- d_3 = distance travelled by oncoming vehicle or opposite vehicle (c) from C_1 to C_2 during the overtaking operation of vehicle 'A'

Now Let v = design speed in m/sec

V = design speed in KMPH

v_b = Speed of slow moving vehicle in m/sec

V_b = Speed of slow moving vehicle in KMPH

- (a) Distance travelled by overtaking vehicle A from position A_1 to A_2 during reaction time 't'

$$d_1 = v_b \times t, \text{ metres}$$

where t = reaction time of a driver in overtaking sight distance = 2 sec

[as per DRC]

$$(b) d_2 = s + b + s$$

$$d_2 = b + 2s$$

$$\text{where } b = v_b T$$

$$d_2 = v_b T + 2s \rightarrow ①$$

where T = Overtaking time in sec

but we know that

$$d_2 = ut + \frac{1}{2} at^2$$

$$u = v_b \quad t = T$$

$$d_2 = v_b T + \frac{1}{2} a T^2 \rightarrow ②$$

equate ① & ② equations

$$v_b T + 2S = v_b T + \frac{1}{2} a T^2$$

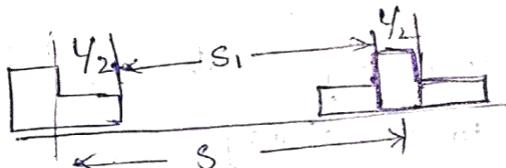
$$2S = \frac{1}{2} a T^2$$

$$a T^2 = 4S$$

$$T = \sqrt{\frac{4S}{a}} \text{ sec}$$

$$T = 2\sqrt{\frac{S}{a}}$$

where S = Spacing between vehicles
(center to center)



where S_1 = lag distance while overtaking
i.e., $S_1 = \text{Speed} \times \text{time}$

$$= v_b \times T$$

$$S_1 = 0.7 v_b$$

[Reaction time $T = 0.7 \text{ sec}$ as per IRC]

$$\therefore S = \frac{L}{2} + S_1 + \frac{L}{2}$$

$$S = S_1 + L$$

$$S = 0.7 v_b + 6, \text{ m}$$

where $L = 6 \text{ m}$, as per IRC [length of vehicle]

(c) $d_3 \Rightarrow$ dist. travelled by own coming vehicle or opposite vehicle (c) from position C_1 to C_2 during overtaking operation.

$$d_3 = v \times T$$

$$\therefore \text{OSD} = d_1 + d_2 + d_3$$

$$\boxed{\text{OSD} = [v_b t] + [v_b T + 2s] + [v T], \text{metres}}$$

(Or)

$$\boxed{\text{OSD} = [0.278 v_b t] + [0.278 v_b T + 2s] + [0.278 v T] \text{ KMPH}}$$

where, $T \Rightarrow$ Overtaking time $= \sqrt{\frac{4s}{a}} \text{ Sec}$

$$\boxed{T = \sqrt{\frac{14.4s}{A}}}$$

where $a = \text{acceleration in } m/\text{sec}^2$

$A = \text{acceleration in KMPH/sec}$

Note:

If speed of slow moving vehicle is not given

take speed of slow moving vehicle $\} \boxed{v_b = v - 4.5 \text{ m/sec}}$ (or)

$$\boxed{v_b = 0.45V - 5 \text{ KMPH}}$$

$$\boxed{v_b = V - 16 \text{ KMPH}}$$

Note:

for two lanes one way traffic

$\text{Safe OSD} = d_1 + d_2$ [$d_3 = 0$ there is no opposite vehicle]

for two lanes, two way traffic

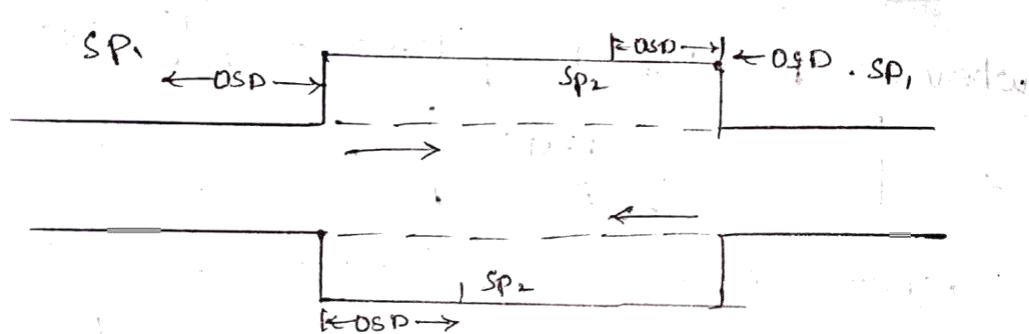
$$\text{Safe OSD} = d_1 + d_2 + d_3$$

for four lanes, two way traffic

$$\text{Safe OSD} = d_1 + d_2$$

"except, in two lanes, two way traffic
in all other cases, d_3 should not be
considered for calculating OSD"

Overtaking zone:



where SP_1 = Overtaking zone ahead
(information sign)

SP_2 = End of Overtaking zone

(mandatory sign.)

The minimum length of overtaking

zone = $3 \times \text{OSD}$

The desirable length of overtaking
zone = $5 \times \text{OSD}$ (as per IRC)

These zones are provided in single lane
roads to facilitate overtaking.

1. The speed of overtaking and overtaken vehicles are 80 KMPH and 50 KMPH respectively. The acceleration of overtaking vehicle is 0.98 m/sec^2 . for two lanes two way traffic i) Determine DSD
 ii) Determine min length of overtaking zone
 iii) Draw overtaking zone diagram and mention sign posts

Given

Speed of overtaking vehicle $V = 80 \text{ KMPH}$
 $v = 22.2 \text{ m/sec}$

Speed of overtaken vehicle $V_b = 50 \text{ KMPH}$
 $v_b = 13.8 \text{ m/sec}$

Acceleration $a = 0.98 \text{ m/sec}^2$
 of overtaking vehicle

DSD reaction time $t = 2 \text{ sec}$ (as per IRC)

Spacing b/w vehicles $S = 0.7 v_b + 6$

$$\boxed{S = 15.72 \text{ m}}$$

(Q1).

$$S = 0.2 v_b + 6$$

$$= 16 \text{ m}$$

Overtaking time $T = \sqrt{\frac{4S}{a}}$

$$= \sqrt{\frac{4 \times 15.72}{0.98}}$$

$$T = 8.01 \text{ sec}$$

∴ Overtaking sight distance
 $\text{OSD} = d_1 + d_2 + d_3$

$$OSD = V_b t + V_b T + 2s + \theta T$$

$$\text{pass}t = (13.8 \times 2) + (13.8 \times 8.01) + 2(15.72) + (22.22 \times 8.01)$$

$$\text{now } OSD = 347.56 \text{ m}$$

(or) ~~length of overtaking zone~~

$$OSD = (0.278 \times V_b t) + (0.278 V_b T) + (2s) + \sqrt{0.278}$$

$$= (0.278 \times 50 \times 2) + (0.278 \times 50 \times 8.01) + (2 \times 15.72) + (0.278 \times 80 \times 8.01)$$

$$\therefore OSD = 348.72 \text{ m}$$

ii) Min length of Overtaking zone

$$= 3 \times OSD$$

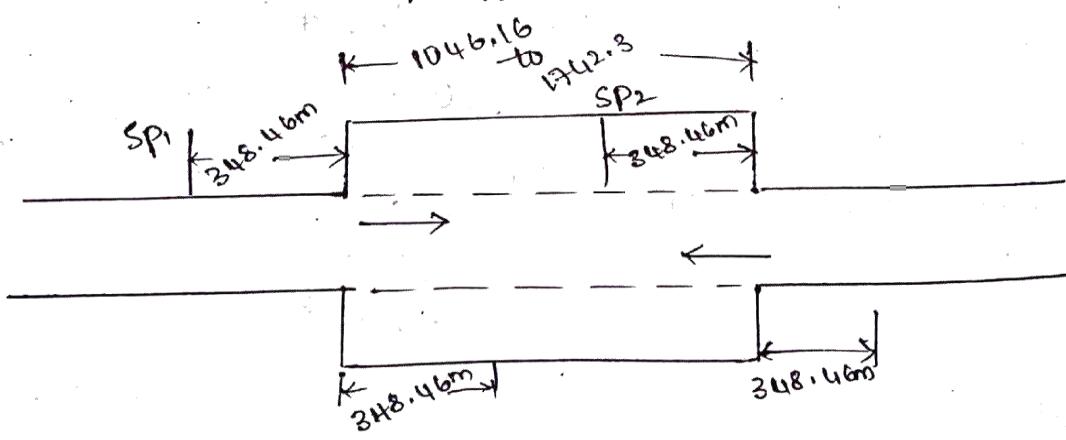
$$= 3 \times 348.72$$

$$= 1046.16 \text{ m}$$

The Desirable length of overtaking zone

$$= 5 \times 0.5D$$

$$= 1743.6$$



* If the speed of overtaking vehicle is 90 KMPH and acceleration is 2.5 KMPH/sec. Determine min. length of OSD and draw overtaking zone.

Speed of overtaking vehicle $V = 89.5 \text{ KMPH}$

$$\theta = 25 \text{ m/sec.}$$

$$\text{Acceleration } a = \frac{2.5 \times \frac{1000}{60 \times 60}}{= 0.69 \text{ m/sec}^2}$$

Speed of overtaken vehicle $V_b = V - 4.5$

$$V_b = 25 - 4.5$$

$$V_b = 20.5 \text{ m/sec}$$

$$V_b = \sqrt{16}$$

$$= 90 - 16$$

$$V_b = 74 \text{ KMPH}$$

OSD reaction time $t = 2 \text{ sec}$

$$\text{Spacing } S = 0.2 V_b + 6$$

$$= 0.2 \times 74 + 6$$

$$= 20.8 \text{ m.}$$

$$S = 0.7 V_b + 6$$

$$= 0.7 (20.5) + 6$$

$$= 20.35 \text{ m.}$$

Overtaking time $T = \sqrt{\frac{4S}{a}}$

$$T = \sqrt{\frac{14.45}{2.5}}$$

$$= 10.8$$

$$T = \sqrt{\frac{4 \times 20.35}{0.69}}$$

$$= 10.86$$

$$\text{OSD} = V_b t + V_b T + 2s + V T$$

$$= (20.5 \times 2) + (20.5 \times 10.86) + 2(20.35)$$

$$+ 0(10.86) - 25$$

DSD = 575.83 m

$$\text{OSD} = (0.278 \times V_b t) + (0.278 V_b T) + 2 s + V_0 \cdot 278 T$$

$$= (0.278 \times 74 \times 2) + (0.278 \times 74 \times 10.86) + (2 \times 20.35)$$

$$+ (0.278 \times 96 \times 10.86)$$

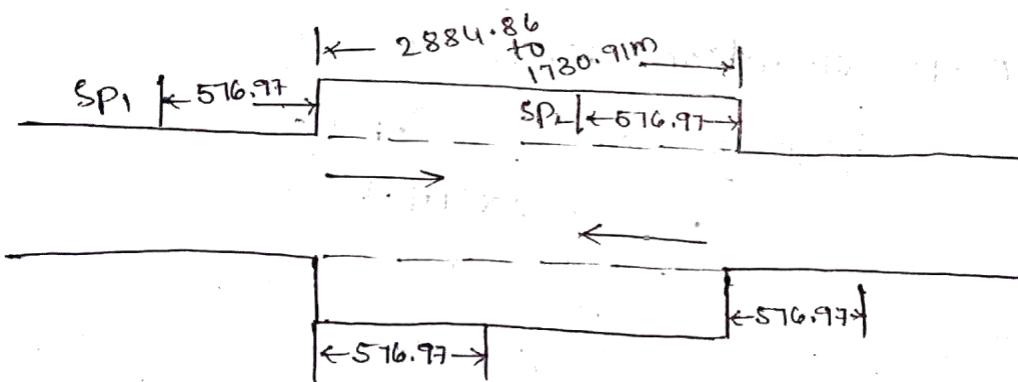
OSD = 576.97 m

Min length of OSD = 3 × OSD

Desirable = 1730.91 m.

Max length of OSD = 5 × OSD.

= 2884.8656



Design of horizontal alignment

The alignment changes from horizontal to curve in horizontal direction is called horizontal alignment.

Design elements to be considered in Horizontal alignment.

1. Design speed
2. horizontal curve
3. Super elevation
4. Type & length of transition curves
5. widening of width of pavement on curves
6. Set back distance

1. Design Speed

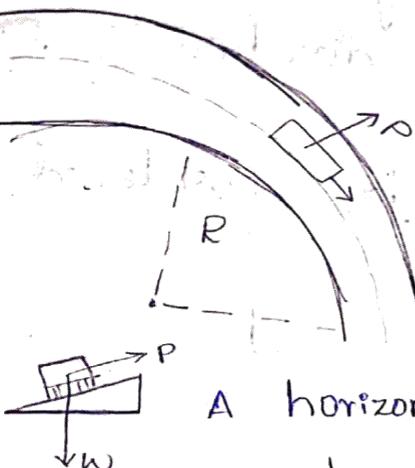
The design speed is the main factor on which geometric design elements depends.

The design speed of roads depends upon → class of the road
→ Terrain

Class of country

| Terrain classification | Gross slope in % |
|------------------------|------------------|
| plan | 0 - 10 |
| Rolling | 10 - 25 |
| Mountainous | 25 - 60 |
| Steep | > 60 |

b. Horizontal curve



The centrifugal force is given by the following eqn

$$P = \frac{w v^2}{R}$$

P = centrifugal force in kg

w = weight acting download of vehicle in kg

v = velocity speed in m/sec.

R = Radius of the curve (m)

A horizontal highway curve is a curve in plane to provide change in direction to the central line of road.

when a vehicle traverses a horizontal curve, the centrifugal force acts horizontally outwards through the center of gravity of the vehicle (C.G)

g = acceleration due to gravity = 9.81 m/sec^2

* The ratio between centrifugal force (P) to weight of a vehicle is called the centrifugal ratio (α) impact factor

i.e., centrifugal ratio $\frac{P}{w} = \frac{v^2}{gR}$

* The centrifugal force acting on a vehicle negotiating a horizontal curve has two effects

1. Tendency to overturn the vehicle outwards about the outer wheels

2. Tendency to skid the vehicle laterally outward.

(a) Over turning

from equilibrium eqn's

(Moment about outer wheel)

$$\sum M_{(O.w)} = 0$$

$$(P \times h) - (w \times \frac{b}{2}) = 0$$

$$Ph = w \frac{b}{2}$$

$$\frac{P}{w} = \frac{b}{2h}$$

\therefore the over turning will occur when the centrifugal ratio ($\frac{P}{w}$) is equal to

$$\frac{b}{2h}$$

(OR)

$$\frac{v^2}{gR} = \frac{b}{2h}$$

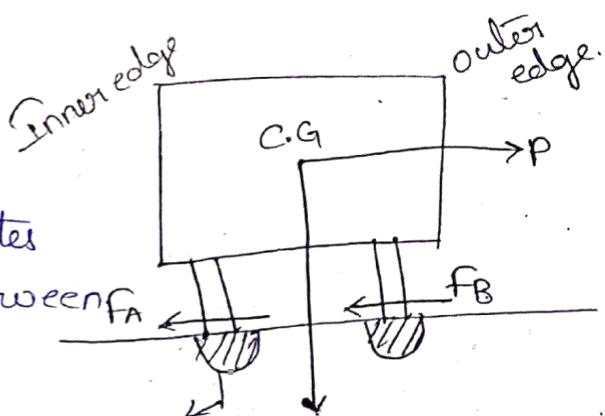
(b) Skidding

from the figure the shaded area will indicates the contact pressure between wheels and pavement

Let

R_A & R_B = Reactions at both wheels

Contact pressure between wheels & pavement w



f_A & f_B is frictional force at both wheels
from equilibrium equations

$$\sum H = 0$$

$$P - f_A - f_B = 0$$

$$P = f_A + f_B$$

$$\{ f_A = f R_A, f_B = f R_B \}$$

$$P = f R_A + f R_B$$

$$P = f (R_A + R_B)$$

$$P = f (R_A + R_B) \rightarrow ①$$

$$\sum V = 0$$

$$R_A + R_B - w = 0$$

$$w = R_A + R_B \rightarrow ②$$

From ①

$$P = f w$$

$$\boxed{\frac{P}{w} = f}$$

where f = frictional co-efficient in lateral direction

∴ If the centrifugal ratio is equal to the lateral coefficient of friction, the skidding will occur (i.e.) $\frac{b}{2h} = f$,

skidding will occur

if $f < \frac{b}{2h}$ → Skidding will occur but not over turning.

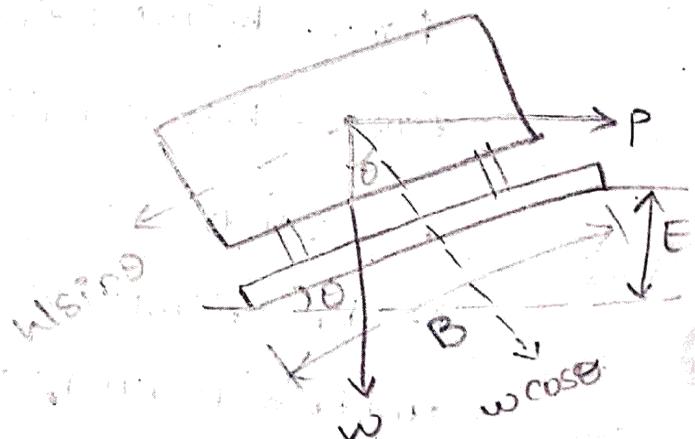
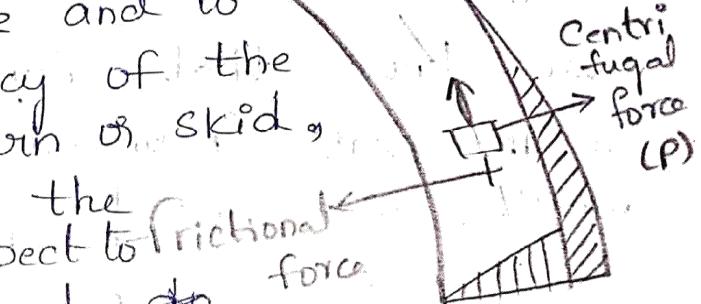
if $f > \frac{b}{2h}$ → Vehicle will over turn before skidding on the outer side.

Super Elevation:

In order to counteract the centrifugal force and to reduce the tendency of the vehicle to overturn or skid,

The outer edge of the pavement with respect to frictional inner edge is raised, do providing a transverse slope

Thus providing a transverse slope throughout the length of the curve. This transverse slope is known as Super elevation (S) cant (S) & banking



Analysis Of Super Elevation.

Let W = weight of vehicle

R_A & R_B = Normal reactions
frictional forces

f = frictional coefficient
in lateral direction.

F_A & F_B = frictional forces

$$F_A = f R_A \quad F_B = f R_B$$

We know that Super elevation (e) is defined as the ratio between outer edge to horizontal width i.e., $e = \frac{BC}{AC}$ but from figure

$$\tan \theta = \frac{BC}{AC}$$

$$e = \tan \theta$$

from equilibrium equations

$$\sum H = 0$$

$$P \cos \theta - W \sin \theta - F_A - F_B = 0$$

$$P \cos \theta = W \sin \theta + f R_A + f R_B$$

$$P \cos \theta = W \sin \theta + f (R_A + R_B) \rightarrow ①$$

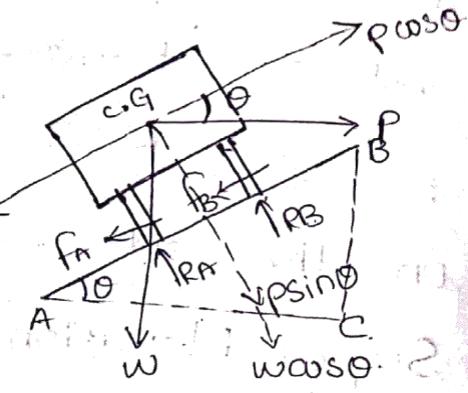
$$\sum V = 0$$

$$R_A + R_B - P \sin \theta - W \cos \theta = 0$$

$$R_A + R_B = P \sin \theta + W \cos \theta \rightarrow ②$$

from eqn's ① & ②

$$P \cos \theta = W \sin \theta + f (P \sin \theta + W \cos \theta)$$



$$P \cos \theta = w \sin \theta + f p \sin \theta + f w \cos \theta.$$

$$P \cos \theta - f p \sin \theta = w \sin \theta + f w \cos \theta.$$

$$P(\cos \theta - f \sin \theta) = w (\sin \theta + f \cos \theta)$$

$$\frac{P}{w} (\cos \theta - f \sin \theta) = (\sin \theta + f \cos \theta)$$

dividing with $\cos \theta$ on both sides

$$\frac{P}{w} (1 - f \tan \theta) = (\tan \theta + f)$$

$$\frac{P}{w} (1 - f e) = (e + f)$$

$$\frac{P}{w} = \left(\frac{e + f}{1 + ef} \right)$$

As per IRC for design purpose take
the co-efficient of friction in lateral direction
is equal to 0.15

$$\tan \theta = e = \text{Super-elevation } (\theta)$$

transverse slope exceeds 7 to 10 %.

∴ from the above equation

$$\frac{P}{w} (1 - f \tan \theta) = (1 + \tan \theta + f)$$

$$1 - f \tan \theta = 1 - (0.15 \times 0.07)$$

$$= 0.98 \cong 1$$

$$\frac{P}{w} \times 1 = \tan \theta + f$$

$$\frac{P}{w} = e + f$$

w.k.t. that $\frac{P}{w} = \frac{v^2}{gR}$

$$e + f = \frac{v^2}{gR} \quad v \text{ in m/sec}$$

$$e + f = \frac{(0.278 V)^2}{9.81 \times R}$$

$$e + f = \frac{V^2}{\frac{9.81}{(0.278)^2} \times R}$$

$$e + f = \frac{V^2}{127 R} \quad V \text{ in KMPH}$$

where e = rate of super elevation
 $= \tan \theta$ (exceeds 7 to 10%)

f = coefficient of value of lateral frictional co-efficient 0.15

g = acceleration due to gravity = 9.81 m/s^2

V = (velocity) speed of the vehicle
 in m/s

V = speed of the vehicle in KMPH

R = Radius of curve horizontal curve
 in m

if no super elevation is provided ($e=0$)

$$f = \frac{V^2}{127 R}$$

for the pressure on inner and outer wheels to be equal (or) for equilibrium Super elevation counteracting centrifugal force fully i.e., $f = 0$

$$e = \frac{V^2}{127 R}$$

The Radius of a horizontal curve is 95m. The design speed is 50 kmph and the design co-efficient of lateral friction 0.15.

1. Calculate the super elevation required if full lateral friction is assumed to develop

2. Calculate the coefficient of friction needed if no super elevation is provided

3. Calculate the equilibrium super elevation if the pressure on inner & outer wheel should be equal.

Given data:

design speed $V = 50 \text{ kmph}$

$$\text{centrifugal force} = \frac{V^2}{R} = \frac{50 \times 1000}{60 \times 60}$$

$$\text{Centrifugal force} = \frac{50^2}{60 \times 60} \times 1000 = 13.889 \text{ m/sec.}$$

horizontal curve $R = 95 \text{ m.}$

Co-efficient of lateral friction $f = 0.15$

Acceleration due to gravity $g = 9.81 \text{ m/sec}^2$

$$i) e + f = \frac{V^2}{127R}$$

$$e + 0.15 = \frac{50^2}{127 \times 95}$$

$$e = 0.057$$

$$e = 5.7\%$$

ii) No super elevation is provided.

$$f = \frac{V^2}{127R}$$

$$= \frac{50^2}{127 \times 95}$$

$$f = 0.207$$

iii) The pressure at inner wheels and outer wheels are equal, then

$$f = 0, e = \frac{V^2}{127R}$$

$$e = 0.207$$

* Design the super elevation required at a horizontal curve of radius 300m for speed 60 kmph. Assume suitable data.

Given data

Horizontal radius curve $R = 300\text{mm}$

Speed $V = 60\text{ kmph}$

Co-efficient $f = 0.15$

acceleration due to gravity $g = 9.81$

$$e + f = \frac{V^2}{127R}$$

$$e + 0.15 = \frac{60^2}{127 \times 300}$$

$$e = -0.055$$

Maximum Super elevation.

- * As per IEC the maximum limit of Super elevation in plane, rolling terrains and Snow bound areas is 7%.
- * AB hill roads not bound by snow; a maximum Super elevation upto 10% is recommended.
- * An urban road stretches with frequent intersections it may be necessary to limit the maximum Super elevation to 4%.

Minimum Super elevation.

- * from drainage considerations it is necessary to have a minimum cross slope to drain off the surface water.
- * If the calculated Super elevation is equal (or) less than camber of road surface then the minimum Super elevation to be provided on horizontal curve may be limited to the camber of the surface.

steps for design of Super elevation.

Step-1 The super elevation for 75% design speed is calculated, neglecting the friction

$$e = \frac{(0.75)^2}{gR} \text{ in m/sec}$$

$$e = \frac{(0.75)^2 (0.278)^2 V^2}{9.81 \times R}$$

$$e = \frac{9.81 V^2}{(0.75)^2 (0.278)^2 \cdot R}$$

$$e = \frac{V^2}{225R} \quad V \text{ in KMPH}$$

Step -2 If calculated value of Super elevation (e) less than 7% (0.07) the value so obtained is provided.

If the value of Super elevation (e)

exceeds 7%, then provide maximum super elevation equal to 7% and provide with step NO 3 & 4.

Step -3 check the co-efficient of friction

developed for the maximum value of Super elevation is 7% at the full value of design speed.

$$e+f = \frac{V^2}{gR}$$

$$e = 0.07 \text{ (7%)}$$

$$f = \frac{V^2}{gR} = 0.07 \quad V \text{ in m/sec}$$

$$f = \frac{V^2}{127R} \Rightarrow 0.07 \quad V \text{ in KMPH}$$

If the value of (f) calculated is less than 0.15. The super elevation of 0.07 is safe for design speed. If not calculate the required speed as given in

Step 4.

Step 4

The allowable speed at the curve is calculated by considering the design coefficient of lateral friction and the maximum super elevation.

Maximum Super Elevation = $e + f_g = \frac{V_a^2}{gR}$

$$e = 0.07, f_g = 0.15$$

$$0.07 + 0.15 = \frac{V_a^2}{gR}$$

$$0.07 + 0.15 = \frac{V_a^2}{gR}$$

$$\frac{V_a^2}{gR} = 0.22 \quad v \text{ in m/sec}$$

$$\frac{V_a^2}{127 R} = 0.22 \quad v \text{ in km/PH}$$

allowable speed is calculated as

$$V_a = \sqrt{0.22 g R} \quad \text{in m/sec}$$

$$V_a = \sqrt{0.22 \times 127 R} \quad \text{in km/PH}$$

$$V_a = \sqrt{27.9 R}$$

If the allowable speed as calculated above is greater than design speed of the design speed is adequate and provide a Super elevation of 0.07.

If the allowable speed is less than the design speed, the speed limit is limited to

the allowable speed calculated above and appropriate warning sign and speed limit regulation signs are installed to restrict and regulate the speed.

- * The design speed of horizontal curve is 80 kmph and the radius is 200 m. required. Super elevation and also calculate the allowable speed at horizontal curve if there is no possibility to increase the radius of the curve.

Given that

The design speed of a road $V = 80 \text{ kmph}$

The radius of horizontal curve $R = 200 \text{ m}$

There is no possibility to increase the

radius.

\therefore To avoid this problem determine the

Super elevation for 75% design speed.

$$e = \frac{V^2}{225R} = \frac{80^2}{225 \times 200}$$

$$e = 0.14$$

The value of super elevation is greater than 0.07 (7%) as per SRC the super elevation should not exceed 7%.

Provide the super elevation $e = 0.07$.

check the value of friction developed

$$f = \frac{V^2}{127R} = 0.07$$

$$= \frac{80^2}{127 \times 200} \rightarrow 0.07$$

$$f = 0.181$$

The value is greater than the maximum allowable safe friction coefficient of 0.15 and also the radius can't be increased, the speed has to be restricted.

∴ The maximum allowable speed is determined

by $V_a = \sqrt{27.94 \times R}$

$$= \sqrt{27.94 \times 200}$$

∴ $V_a = 74.75 \text{ KMPH}$

The maximum allowable speed at that particular location is 74.75 KMPH.

Extra widening:

On horizontal curves especially when there is $< 300\text{m}$ Radius, it is common to widen the pavement slightly more than the normal width.

The widening is needed for the following reasons

→ An automobile has a rigid wheel base and only the front wheel can be turned, when the vehicle takes a turn to negotiate the horizontal curve, the rear wheels do not follow the same path as that of the front wheels. This phenomenon is called as tracking.

- while two vehicles cross or over take on horizontal curve, there is psychological tend.ency to maintain a greater clearance b/w the vehicles for safety.
- Greater visibility occurs, this driver has tendency not to follow the central path of the lane but to use the outer side at the begining of the curve.
- At higher super elevation and lateral friction can't counteract centrifugal force and skidding may occur.

Analysis of extra widening on horizontal curve

It is divided into two parts

i) Mechanical widening (W_m)

ii) Psychological widening (W_{ps})

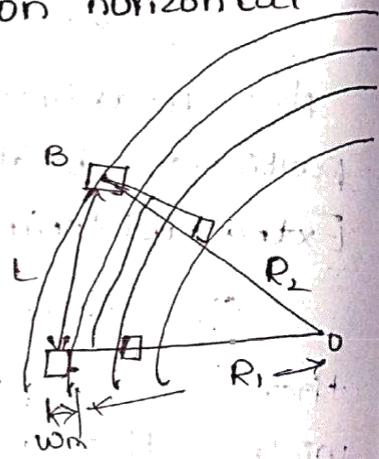
i) Mechanical widening (W_m) - Off tracking

let R = Radius of horizontal curve

$R_1 = OA$ = Radius of the path traversed by the outer rear wheels in m.

$R_2 = OB$ = Radius of the path traversed by the outer front wheels in m.

W_m = mechanical widening due to off tracking in m



L = length of wheelbase in m. $L = 6m$ as per IRC for commercial vehicles.

From figure

$$W_m = OB - OA$$

$$= R_2 - R_1$$

$$R_1 = R_2 - W_m$$

$$R_2^2 = L^2 + R_1^2$$

$$R_2^2 = L^2 + (R_2 - W_m)^2$$

$$R_2^2 = L^2 + R_2^2 + W_m^2 - 2R_2 W_m$$

$$L^2 + W_m^2 - 2R_2 W_m = 0$$

$$L^2 + W_m (W_m - 2R_2) = 0$$

$$W_m = \frac{L^2}{2R_2 - W_m} \quad \{W_m \text{ is neglected}\}$$

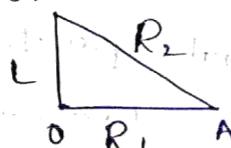
$$W_m = \frac{L^2}{2R_2}$$

Apparently

$$W_m = \frac{L^2}{2R}$$

If road having n traffic lanes and n vehicles can travel simultaneously, mechanical widening is given by $W_m = \frac{nL^2}{2R}$

(B.)



b) Psychological widening

Extra width of pavement is also provided for psychological reasons, such as to provide greater clearance for crossing and overtaking vehicles on the curves.

An empirical formula has been recommended by IRC for deciding the additional psychological widening and is given by the following formula

$$\text{i.e., } W_{ps} = \frac{V}{9.5\sqrt{R}}$$

\therefore Without the total extra widening

$$W_e = W_m + W_{ps}$$

$$W_e = \frac{nL^2}{2R} + \frac{V}{9.5\sqrt{R}}$$

where n = no. of traffic lanes

L = Length of the wheel base
 $= 6\text{m}$ (recommended by IRC)
 for commercial vehicles)

V = Design speed in KMPH

R = Radius of horizontal curve in m

Note:

- for sharp curves of radius less than 50m
 The extra widening shall be provided on inner side of the curve

2. for curves of R greater than 50m, the widening shall be equally distributed on both sides of the curves.
- * calculate the extra widening required for a pavement of 7m on a horizontal curve of radius 250m if the longest wheel base of vehicle expected on the road is 7m. The design speed is 70 KMPH

Given

$$\text{width of pavement} = 7\text{m}$$

i.e it is double lane

$$\text{So } n=2$$

$$R = 250\text{ m}$$

$$V = 70 \text{ KMPH}$$

$$L = 7\text{ m}$$

$$We = \frac{nL^2}{2R} + \frac{V}{9.5\sqrt{R}}$$

$$= \frac{2(7)^2}{2 \times 250} + \frac{70}{9.5\sqrt{250}}$$

extra widening in addition to the width of the road. $We = 0.66\text{ m}$ (using leftmost position of outer wheel) which is less than the maximum width of 7m and will be acceptable. So the design width is 7.66m. The transition from the straight section to the curve is made by a circular curve with a diameter equal to the sum of the design widths of the two sections.

* Radius of Horizontal Curves

we know all that

$$e+f = \frac{V^2}{gR}$$

then we can get the formula for ruling radius based on minimum design speed

$$R_{\text{ruling}} = \frac{V^2}{g(e+f)}$$

$$R_{\text{ruling}} = \frac{V^2}{127(e+f)}$$

Absolute minimum radius based on minimum design speed.

$$R_{\min} = \frac{(V')^2}{g(e+f)}$$

$$R_{\min} = \frac{(V')^2}{127(e+f)}$$

* find the total width of pavement on a horizontal curve for a new national highway to be aligned along a rolling terrain with a rolling minimum radius assumed for National highways on rolling terrain - the ruling design speed = 80 kmph

$$n = 2$$

$$V = 80 \text{ kmph}$$

$$L = 7 \text{ m}$$

$$R_{\min} = \frac{V^2}{127(e+f)}$$

The value of super elevation is 0.07
co-efficient of friction, $f = 0.15$

$$R_{min} = \frac{(80)^2}{127(0.07 + 0.15)}$$

$$R_{min} = 229.06 \text{ m}$$

$$w_e = \frac{n t^2}{2R} + \frac{V}{9.5 \sqrt{R}}$$

$$w_e = \frac{2 \times 7^2}{2 \times 229.06} + \frac{80}{9.5 \sqrt{229.06}}$$

$$w_e = 0.77 \text{ m.}$$

The total width of pavement is $= 7 + 0.77$
 ± 7.77

~~Horizontal Transition Curve~~: (The curve having different radius)

A transition curve has a radius which decreases from infinity at the tangent point to a designed radius of the circular curve.

→ The rate of change of radius of the transition curve will depend on the shape of the curve adopted and the equation of the curve.

The transition curve is introducing between a straight and circular curve.

Types:

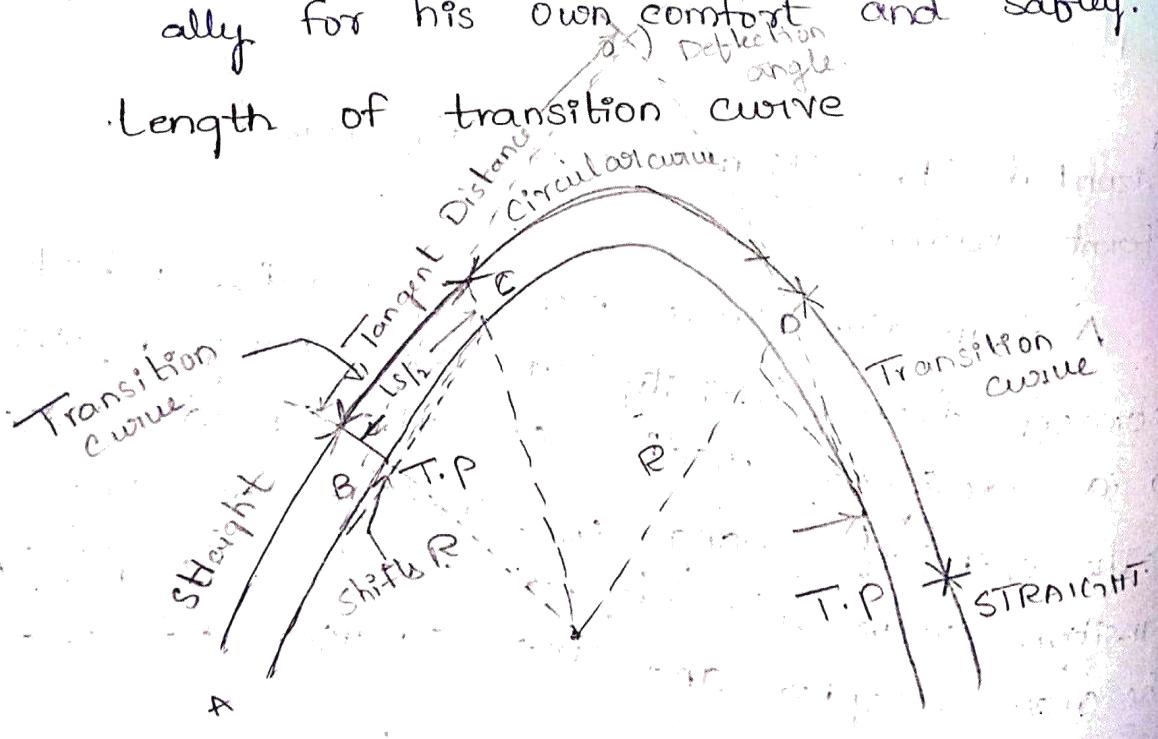
1. Spiral or clothoid or clover's spiral
2. Bernoulli's Lemni scale
3. Cubic parabola or Froude's transition curve or easement curve.

Object of providing transition curve.

A transition curve which is introduced between straight and a circular curve will help in:

- a) Gradually introducing centrifugal force.
- b) Gradually introducing design super elevation.
- c) Gradually introducing extra widening.
- d) To enable the driver twin steering gradually for his own comfort and safety.

Length of transition curve



The length of transition curve is designed to fulfill three conditions.

- Rate of change of centrifugal acceleration to be developed gradually.
- Rate of introduction of design super-elevation
- Minimum length by centrifugal acceleration.

a. Rate of change of transition curve is calculated as

$$\rightarrow \text{The length of transition curve is calculated as}$$

$$L_s = \frac{V^3}{CR} \quad (\text{if } V \text{ is in m/sec})$$
~~$$L_s = \frac{0.0215 V^3}{CR} \quad (V \text{ is in kmph})$$~~

where

$$C = \frac{80}{75 + V} \text{ m/sec}^3 \quad (0.5 < C < 0.8)$$

L_s = length of transition curve in m.

C = allowable rate of change of centrifugal acceleration (δ) jerk

V = design speed in m/sec

V = design speed in kmph

R = Radius of curve in m

b. Rate of introduction of Super elevation

\rightarrow If the pavement is rotated about the center line

$$L_s = EN/2 = \dot{\epsilon} N/2 (w + w_c)$$

→ If the pavement is rotated about the inner edge

$$L_s = EN = e \cdot N (W + w_e)$$

$$\tan \theta = \frac{E}{B}$$

where $W = \text{width of pavement}$

$w_e = \text{extra widening}$

$E = \text{total raised pavement} = e \cdot B$

$B = \text{total width of pavement} = (W + w_e)$

Rate of change of super elevation of

min N

$N = 150, 100, 60$

c) Minimum length by IRC empirical formula.

According to IRC standards

a. for plane and rolling terrain

$$L_s = \frac{2.7 V^2}{R}$$

b. for mountainous and steep terrain

$$L_s = \frac{V^2}{R}$$

Note: The length of transition curve shall be the highest of the above (centrifugal acceleration & empirical formula)

Super elevation & empirical formula

shift of transition curve:

If the length of transition curve is L_s & the radius of the circular curve is R . The shift 's' of transition curve is given by

$$S = \frac{L_s^2}{24R}$$

Calculate the length of transition curve and shift using the following data

- * Design speed = 65 KMPH
- * Radius of circular curve $R = 220\text{m}$
- * allowable rate of introduction of super elevation [Pavement is rotated above the central line] $= 1^\circ\text{min} = 1^\circ\text{in } 50$
- * Pavement width including extra widening $= 7.5\text{m}$

Given that

design speed $V = 65\text{ KMPH}$

Radius of circular curve $R = 220\text{m}$

Rate of introduction of super elevation $= 1^\circ\text{in } 50$
 $N = 150$

a) Length of transition curve as per allowable rate of centrifugal acceleration.

$$L_s = \frac{0.0215 V^3}{C R}$$

C = allowable centrifugal acceleration for jerk

$$= \frac{80}{75 + V}$$

$$= \frac{80}{75 + 65}$$

$$C = 0.57 \text{ m}^3/\text{sec}^3 \quad \{0.5 < C < 0.8\}$$

$$L_s = \frac{0.0215 \times 65^3}{0.57 \times 220} = 47.08\text{m.}$$

b) length of transition by introducing S.E.:
 Rate of introducing Super-elevation S.E.:

For smooth transition the rate will be constant

initial gradient and passing through

$$L_S = EN/2$$

where E = sum of forces perpendicular to the road
 weight of car = $\frac{eN}{2} (w + w_e)$

at H. point, $\frac{V^2}{R} = \frac{w + w_e}{2}$

$$e = \frac{w_e}{w} = \frac{V^2}{225R} = \frac{V^2}{225 \times 65} = 0.085$$

$\left\{ \frac{V^2}{127R} \text{ for design speed of } 70 \text{ m/s} = \frac{V^2}{225R} \right\}$

therefore $e = 0.085 > 0.07$

The value is greater than 0.07

as per IRC the maximum super elevation is equal to 0.07 i.e., 7%

∴ provide the super elevation with 0.07

* Check for frictional co-efficient with design speed

$$\text{i.e., } f = \frac{V^2}{127R} - e$$

$$f = 0.08 \times 0.15$$

∴ The value obtained is less than 0.15

(The max. value of lateral frictional co-efficient is 0.15 as per IRC)

∴ Provide the super elevation 0.07 with frictional co-efficient 0.15

for the design speed of 65 KM PH

The length of transition curve for pavement rotated with respect to the center line

$$\text{i.e., } l_s = \frac{EN}{2} = \frac{EN}{2}(w + w_e) \text{ m}$$

in case of uniform rotation the value of

$$N = 150$$

$$E = EB = e(w + w_e)$$

$$E = 0.525$$

$$l_s = \frac{0.07(7.5) \times 150}{2} = 39.375 \text{ m}$$

c) Length of transition curve by empirical formulae

for plane & rolling terrain

$$l_s = \frac{2.7 V^2}{R} = \frac{2.7 \times 65^2}{220} = 51.85 \text{ m}$$

\therefore Adopt the highest value from above 3 ($a = 47.08, b = 39.375;$

$$c = 51.85$$

length of transition curve is

$$\boxed{\therefore l_s = 51.85 \text{ m}} \cong 52 \text{ m}$$

$$\text{Shift } S = \frac{l_s^2}{24R} = \frac{52^2}{24 \times 220} = 0.512 \text{ m}$$

* Design the length of horizontal transition curve for a two lane N.H. having a radius of 300m and for a design speed of 100KMPH for rolling terrain. The average length of rigid

wheel base of vehicle is considered as 6m. Take the co-efficient of friction $\mu_s = 0.38$ in soft soil before setting. So,

Given that $\mu_s = 0.38$ Design speed $V = 100 \text{ kmph}$

Radius of circular curve $R = 300 \text{ m}$

Co-efficient of friction $f = 0.38$

The length of original wheel base $L = 6 \text{ m}$

for 2 lanes National Highway

The width of pavement $= 7 \text{ m}$

we know that

The total extra widening $W_e = W_{mt}$

$$W_{mt} + W_{pys} = \frac{nL^2}{2R} + \frac{V}{9.5\sqrt{R}}$$

$$= \frac{2 \times 6^2}{2 \times 300} + \frac{100}{9.5\sqrt{300}}$$

$$W_e = 0.73 \text{ m}$$

\therefore The total width of pavement including extra widening $= w + w_e$

$$= 7 + 0.73$$

$$B = 7.73 \text{ m}$$

Given that the co-efficient of lateral friction is 0.38

"f" is greater than 0.15 so we have to change the design speed

The maximum allowable speed is

$$V_a = \sqrt{27.94 \times R}$$

$$= \sqrt{27.94 \times 300}$$

$$= 91.5 \text{ KMPH}$$

$$V_a = 25.437 \text{ m/sec}$$

* length of transition curve by rate of change of centrifugal acceleration.

$$L_s = \frac{V^3}{CR} = \frac{(25.437)^3}{0.79 \times 300}$$

$$= 69.44 \text{ m}$$

$$L_s = \frac{0.0215 V^3}{CR} = \frac{0.0215 (91.5)^3}{0.79 \times 300} = 69.49 \text{ m}$$

$$C = \frac{80}{75 + V} = \frac{80}{75 + 25.43} = 0.79$$

* Minimum length for a rolling terrain

$$L_s = \frac{2.7V^2}{R}$$

$$= 75.35 \text{ m}$$

* length of transition curve by the rate of introduction of super elevation.

~~$$L_s = EN/2$$~~

$$= \frac{EN}{2} (w + 2e)$$

$$e = \frac{V^2}{225R} = \frac{(91.5)^2}{225 \times 300} = 0.124$$

The value of e is greater than 0.07
So consider 0.07 as the max. Super
elevation as per IRC

$$L_s = \frac{0.07 \times 150}{2} (7 + 0.73)$$

$$= 40.5 \text{ m.}$$

Adopt the highest of above three values.

\therefore Consider length of transition curve $l_s = 75.35 \text{ m}$

$$\text{Curve } l_s = 75.35 \text{ m}$$

station

station

station

~~If "S" is greater than 0.15 so we have to change the design speed.~~

The maximum allowable speed is

$$V_a = \sqrt{27.94 \times R}$$

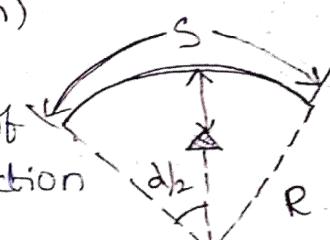
$$= \sqrt{27.94 \times 300}$$

$$= 91.5 \text{ KMPH}$$

Set back^{distance} (or) Elevation clearance (m)

the distance between center of an horizontal curve to an obstruction @ inner side of the curve.

It is required to provide an adequate sight distance on the horizontal curve
(Obstructions are buildings, trees, cut slopes etc.)



Note:

1. On Narrow road the sight distance is measured along the center line of the road.
2. On wider roads the sight distance is measured along the center line of inner side lane.
3. Set back distance is depends on sight distance (S), Radius of Curve (R), length of curve (L_c)

Analysis of set back distance:

Case-1 \Rightarrow If the length of curve is greater than sight distance ($L_c > S$).

Case-2 \Rightarrow If the length of curve is less than sight distance ($L_c < S$)

Case-i If the length of curve is $>$
Sight distance Set back distance

$$\text{The length of curve} = R - R \cos \frac{\alpha}{2}$$

where $\alpha = \text{Subtended angle} = \frac{s}{R}$ radians

$$\alpha = \frac{s}{R} \times \frac{180}{\pi} \text{ degrees}$$

for multilane roads if, d is the distance between center line of a road and center line of an inner lane.

Set back distance $m = R - (R-d) \cos \frac{\alpha}{2}$

$$\alpha = \frac{s}{(R-d)} \text{ rad}$$

$$\alpha = \frac{s}{(R-d)} \times \frac{180}{\pi} \text{ degrees}$$

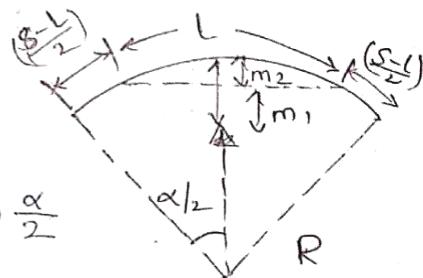
Case-ii If the length of curve is $<$ than the sight distance.

for narrow road.

$$m = R - R \cos \frac{\alpha}{2} + \frac{(s-l_c)}{2} \sin \frac{\alpha}{2}$$

$$\alpha = \frac{l_c}{R} \text{ rad.}$$

$$= \frac{l_c}{R} \times \frac{180}{\pi} \text{ degrees.}$$



for wider road

$$m = R - (R-d) \cos \left(\frac{\alpha}{2} \right) + \frac{(s-l_c)}{2} \sin \frac{\alpha}{2}$$

$$\alpha = \frac{l_c}{(R-d)} \text{ rad.}$$

$$\alpha = \frac{L_c}{R-d} \times \frac{180}{\pi} \text{ degrees.}$$

* There is a horizontal curve of 360m & length 180m. The clearance required from the center line of the inner side of the curve so as to provide

- a) SSD of 80m
- b) OSD of 250m

Given : $L_c = 180\text{m}$

$$R = 360\text{m}$$

a) $S = 80\text{m}$ ($L_c > S$) Stopping sight distance

The length of Curve $L_c = 180\text{m}$

$$\therefore \text{The set } \left. \begin{array}{l} \text{back dist} \\ \text{dist} \end{array} \right\} (\text{m}) = R - R \cos \frac{\alpha}{2}$$

$$= 360 - 360 \cos \left(\frac{12^\circ 43' 56.62''}{2} \right)$$

$$\alpha = \frac{S}{R} \times \frac{180}{\pi} \quad m = 2.219\text{m}$$

$$= \frac{80}{360} \times \frac{180}{\pi}$$

$$= 12^\circ 43' 56.62''$$

b) OSD $S = 250\text{m}$ ($L_c < S$)

$$L_c = 180\text{m.}$$

$$m = R - R \cos \frac{\alpha}{2} + \frac{(S - L_c) \sin \frac{\alpha}{2}}{2}$$

$$= 360 - 360 \cos \left(\frac{28^\circ 32' 52.4''}{2} \right) + \frac{(250 - 180)}{2} \frac{\sin \left(\frac{28^\circ 32' 52.4''}{2} \right)}{\sin \left(\frac{28^\circ 32' 52.4''}{2} \right)}$$

$$\alpha = 28^\circ 32' 52.4''$$

$$m = 19.74\text{m}$$

Vertical Alignment:

The elevations along the profile of the center line of a road is called Vertical Alignment. It consists of grades (slopes) and vertical curves.

Gradients:

The state of rise or fall along the length of the road with respect to the horizontal.

It is expressed as the ratio of linear or percentage (%).

Types of Gradients:

1. Ruling gradient (or) Design Gradient
2. Limiting gradient
3. exceptional gradient
4. Minimum gradient

1. Design Gradient (or) Ruling Gradient

The maximum gradient within that the vertical profile is designed.

2. Limiting Gradient

In hilly roads it may be frequently necessary to exceed ruling gradient and adopt limiting gradient (topography).

3. Exceptional Gradient:

It is provided in extreme situations and it is steeper than Limit gradient.

4. Minimum Gradient:

This is important only at locations where surface drainage is important.

It depends on Rainfall, Type of Soil, and other site conditions.

Grade Compensation:

When a sharp horizontal curve is to be introduced on a road which has already maximum permissible gradient, then the gradient should be decreased to compensate for loss of tractive efforts due to curve.

This reduction in gradient at horizontal curve is called Grade compensation.

$$\text{Grade compensation} = \frac{3D + R}{R} \%$$

The Grade compensation gives the following specifications

IRC gave the following specifications for grade compensation.

1. Grade compensation is not required for grades flatter than 4% because the loss of tractive force is negligible.

2. The maximum grade compensation is limited to $\frac{75}{R}$ % (which ever is less).

$$\text{Compensated Gradient} = \text{Gradient} - \text{Grade compensation}$$

Eg: The longitudinal gradient on a road is 1 in 20 if the radius is 150 m. Then what is the grade compensation.

The grade compensation is $\frac{30+R}{R}$

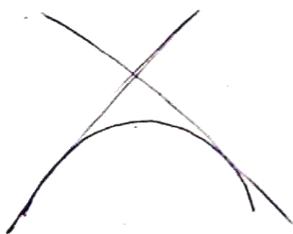
$$= \frac{30+150}{150} = 1.2\%$$

$$\Rightarrow \frac{75}{R} = \frac{75}{150} = 0.5\%$$

Vertical curves

* Vertical curves are provided in elevation at change of gradients.

* These curves are convex when two curves (two grades) meet at summit and when they meet at sag they ~~sag~~ is called as concave.



Summit curves



Valley curves

Summit curves

* These are provided when

- Positive grades meets negative grades
- Positive grades meets another milder positive grades

iii. Positive grades meet level stretch

iv. Negative grade meets steeper negative grades

Assumptions:

1. The length of curve is equal to the length of chord. (the curve is so flat)
2. The portions of curve along the two tangents on either side of the point of intersection are equal.
3. The angle subtended by the tangents are so small that the tangents of these angles are equal to the angles in radius themselves.

Length of Summit curve:

While designing the length of parabolic summit curves it is necessary to consider SSD and OSD separately

The length of summit curve for S.S.D

Case i: $L > SSD$

Case ii: $L < SSD$

L is greater than SSD [$L > SSD$]

General equation for length of curve is given by $L = \frac{NS^2}{(\sqrt{2H} + \sqrt{2h})^2}$

where S = Stopping sight Distance

N = Deviation angle (or) Algebraic difference in grades

H = Height of eye level of driver as per DRC

$h = \text{object height} = 0.15\text{m}$ (as per IRC)

$$L = \frac{Ns^2}{(\sqrt{2 \times 1.2} + \sqrt{2 \times 0.15})^2} = \frac{Ns^2}{4.4}$$

$$L = 0.227 N S^2$$

Case-1: L is less than SSD ($L < S.S.D$)

$$L = 2S - \frac{(\sqrt{2H} + \sqrt{2h})^2}{N}$$

$$L = 2S - \frac{4.4}{N}$$

length of summit curves for OSD

Case-1: L is greater than OSD ($L > OSD$)

$$L = \frac{Ns^2}{8H}$$

$$L = \frac{Ns^2}{8(1.2)} = \frac{Ns^2}{9.6} = 0.104 NS^2$$

Case-2: L is less than OSD ($L < OSD$)

$$L = 2S - \frac{8H}{N}$$

$$L = 2S - \frac{9.6}{N}$$

Length of Valley curves:

factors considered in the design of valley curves.

1. Impact free movement of vehicles at design speed. (or) Comfort to passengers
2. Providing adequate sight distance under headlights of vehicles. for night driving
3. Locating lowest point of valley curve for providing suitable cross drainage facilities.
4. The valley curve and its length are designed as transition curves to fulfill to criterias
 - i. Allowable rate of change of centrifugal acceleration
 - ii. Headlight sight distance for night driving.

length of valley curve for comfort conditions

$$L = 2 \sqrt{\frac{NV^3}{C}} \quad V \text{ in m/sec.}$$

$$L = 0.38 \sqrt{NV^3} \quad V \text{ in kmph.}$$

$$L = 2L_s \quad (L_s = \text{length of Transition curve})$$

where $C = \text{centrifugal acceleration. (or)}$

$$\begin{aligned} \text{jerk} &= @ = \{0.5 < C < 0.8\} \text{ m/sec}^3 \\ &= 0.6 \text{ m/sec}^3 \end{aligned}$$

Length of Valley curve from head light sight distance.

(i) ($L > SSD$) L is greater than S.S.D

$$\text{Length of } L = \frac{N S^2}{(1.5 + 0.35S)}$$

$$\left\{ \begin{array}{l} L = \frac{N S^2}{2h_1 + 2 \tan \alpha} \\ h_1 = \text{height of headlight} = 0.75 \text{ m (as per IRC)} \\ \alpha = \text{beam angle} = 1^\circ \end{array} \right.$$

(ii) ($L < SSD$) L is smaller than S.S.D

$$L = 2S - \frac{2h_1 + 2 \tan \alpha}{N}$$

$$L = 2S - \frac{1.5 + 0.35S}{N}$$

Note: 1. Overtaking Sight distance is not a problem for valley curve as during night, other vehicles with head lights can be seen from a considerable distance.

∴ Headlight sight distance available shall be atleast equal to S.S.D

2. Impact factor - The ratio of centrifugal force (P) and the weight of the vehicle (W) is called impact factor (α) centrifugal ratio.

$$\text{i.e. } I = \frac{P}{W} = \frac{1.6 N V^2}{L}$$

It is expressed in percentage (%) and should not greater than 17%.

3. Minimum radius of valley curve for

cubic parabola $R = \frac{L_s}{N}$

$$L_s = \frac{L}{2}$$

$$R = \frac{L}{2N}$$

Example

A vertical curve is formed by intersecting two gradients ~~+3%~~ +3% and -5%. Determine the length of ^{Summit} ~~valley~~ curve to a stopping sight distance for a design speed of 90 KMPH.

Given that

ascending gradient = +3%

descending gradient = -5%

design speed $V = 90 \text{ KMPH}$

$\therefore N = \text{algebraic difference b/w two gradients}$

i.e., $N = (+3) - (-5)$

$$= 8\% = 0.08$$

The stopping sight distance $SSD =$

$$= 0.218 V t + \frac{V^2}{254 f}$$

$$= 0.218 (90) 2.5 + \frac{90^2}{254 (0.35)}$$

where $t = \text{total reaction time} = 2.5 \text{ sec}$

$f = \text{longitudinal friction} = 0.35$

$$\therefore \text{SSD} = 153.66 \text{ m.}$$

Assume, The length of Summit curve
is $>$ S.S.D

$$L = \frac{Ns^2}{4.4}$$

$$= \frac{0.08 \times (153.66)^2}{4.4}$$

$$L = 429.29 \text{ m.}$$

* An ascending gradient of $\frac{1}{100}$ in 100m, a descending gradient of $\frac{1}{120}$ in 120. A Summit curve is to be designed for a speed of 90KMPH so as to have an overtaking sight distance of 460m. Determine length of Summit curve.

Given that

$$\text{An ascending gradient} = \frac{1}{100} = 0.01$$

$$\text{descending gradient} = \frac{1}{120} =$$

The algebraic difference b/w two gradients

$$= \frac{1}{100} - \left(-\frac{1}{120} \right) = \frac{11}{600}$$

