1.8. Fluid dynamics

mf23 - Magnus effect

Witchcraft?

$\Rightarrow$ Study **fluids in motion** and their effects

*from **wikipedia**, **Attribution-Share Alike 4.0 International***
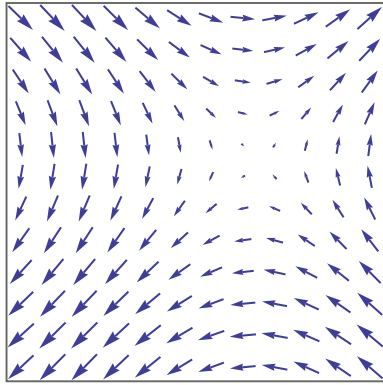
Types of Fluid Flow

Fluid motion is described by a **velocity vector field**:

$$\vec{v}(x, y, z, t) = v_x(x, y, z, t)\,\hat{i} + v_y(x, y, z, t)\,\hat{j} + v_z(x, y, z, t)\,\hat{k}.$$

$\rightarrow$ Every point in the fluid has a local velocity vector that may **vary in space and time**
Flow types:

- Steady vs. Unsteady: **steady (stationary)** $\rightarrow$ velocity field does not change with time; **unsteady (time-dependent)** $\rightarrow$ velocity field evolves in time
- Laminar vs. Turbulent: **laminar** $\rightarrow$ smooth, ordered motion, no mixing of layers; **turbulent** $\rightarrow$ chaotic motion with eddies and fluctuations
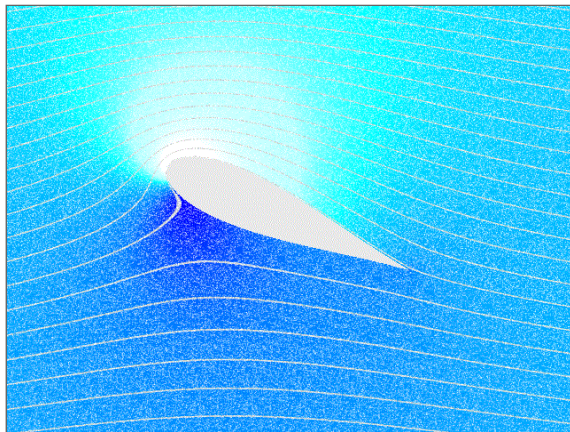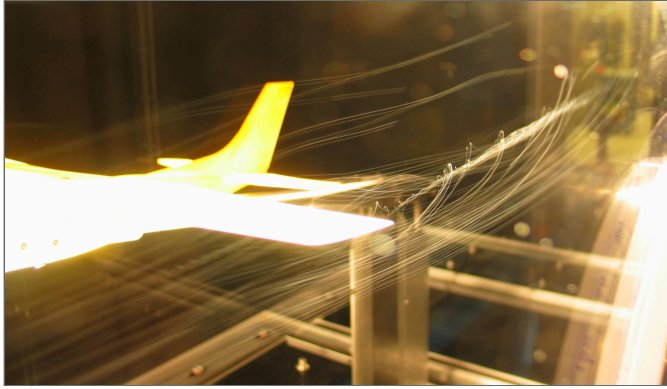
*from **wikipedia**, public domain*

Streamlines

`mf28 - Wellenwanne`

A **streamline** is a curve everywhere **tangent** to the instantaneous velocity field:

- Streamlines never intersect.
- Dense spacing indicates higher speed.
- In **steady** flow, streamlines coincide with particle paths; in **unsteady** or turbulent flow, streamlines exist but change with time.

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Volume & Mass Flow Rate

Given A the cross-sectional area, v the average speed, and ρ the density, we define:

Volume flow rate

$$Q = \frac{dV}{dt} = Av$$

Mass flow rate

$$\frac{dm}{dt} = \rho\,\frac{dV}{dt} = \rho Av$$

$$\dot{m} = \rho Q$$

For **incompressible fluids** ($\rho = \text{constant}$) $\rightarrow Q$ and $\dot{m}$ **remain constant**

Continuity Equation

Mass conservation in steady flow requires

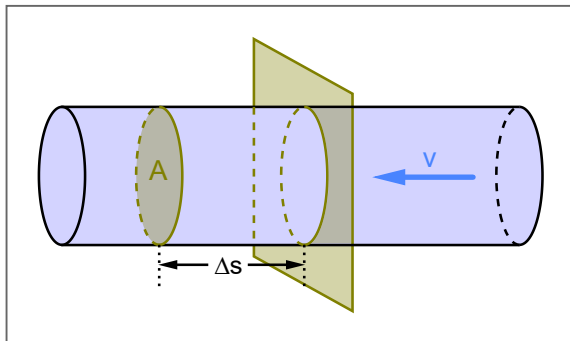$$\dot{m} = \rho A v = \text{constant}$$

Between two cross-sections:

$$\rho_1 A_1 v_1 = \rho_2 A_2 v_2$$

For **incompressible flow** ($\rho_1 = \rho_2$):

$$A_1 v_1 = A_2 v_2 = Q = \text{constant}$$

$\rightarrow$ Narrow sections (small A) $\rightarrow$ higher velocity

Bernoulli's Equation: Assumptions & Energy of a Fluid Element

Assumptions

- Steady flow (no time dependence)

- Incompressible fluid ($\rho = \mathrm{const}$)

- Non-viscous (no friction losses $\rightarrow$ **energy conserved**)

- Motion along a streamline

Energy of a small volume element (dV, $m = \rho dV$)

- Kinetic energy: $\frac{1}{2}\rho v^2\, dV$

- Potential energy: $\rho g h\, dV$

- Pressure energy (work done **on** dV): $P\, dV$

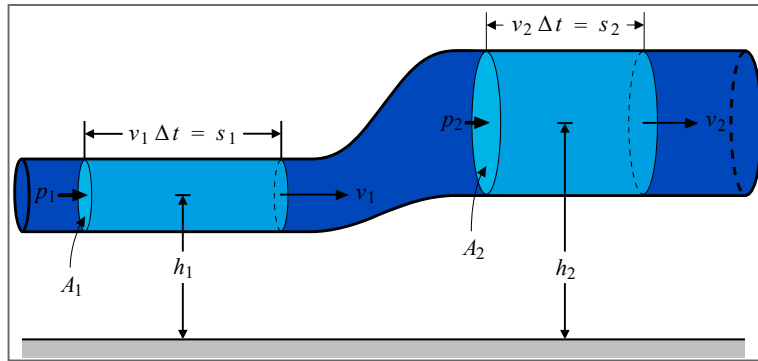# Bernoulli's Equation: From Energy to Formula

`sim - Bernoulli`

Energy conservation for volume element dV in an ideal fluid flow along a streamline:

Total mechanical energy per unit volume is constant:

$$P\,dV + \tfrac{1}{2}\rho v^2 dV + \rho g h\,dV = \text{const.}$$

Divide by dV (incompressible $\to dV$ constant) gives **Bernoulli's equation**:

$$\underbrace{P}_{\text{pressure energy density}} + \underbrace{\tfrac{1}{2}\rho v^2}_{\text{kinetic energy density}} + \underbrace{\rho g h}_{\text{gravitational energy density}} = \text{constant}$$

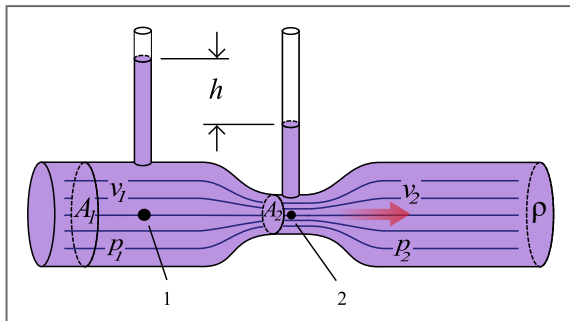*from **wikipedia**, **Attribution-Share Alike 3.0 Unported***

Venturi Effect (Horizontal Flow)

mf34 - Druck in strömenden Gasen

- Horizontal flow: $h_1 = h_2$

- Bernoulli: $P_1 + \frac{1}{2}\rho v_1^2 = P_2 + \frac{1}{2}\rho v_2^2$

- Continuity: $A_1 v_1 = A_2 v_2 \rightarrow v_2 > v_1$ in constriction

- Pressure drop:

$$\Delta P = \frac{1}{2}\rho(v_2^2 - v_1^2)$$

- Narrow section → higher speed → lower pressure

Torricelli's Theorem (Efflux Speed)

`mf06 - Schweredruck vs. ausströmende Flüssigkeit`

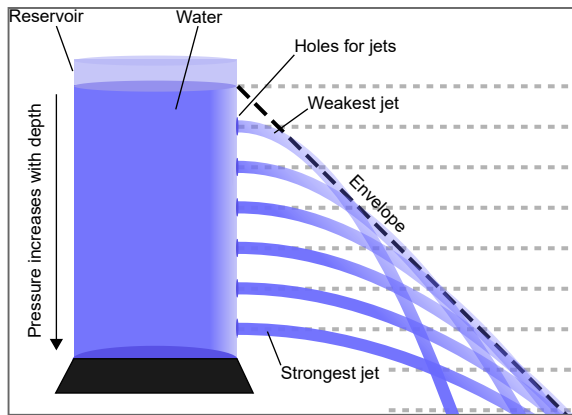- Apply Bernoulli between the liquid surface and the hole:

$$P_{\text{atm}} + \rho g h = P_{\text{atm}} + \tfrac{1}{2}\rho v^2$$

$$\rho g h = \tfrac{1}{2}\rho v^2$$

- Efflux speed: **Speed of the outflowing jet equals the speed a body would acquire when falling freely through the height h.**

$$v = \sqrt{2gh}$$

(Note: h is the height of liquid above the hole, not the height above ground.)

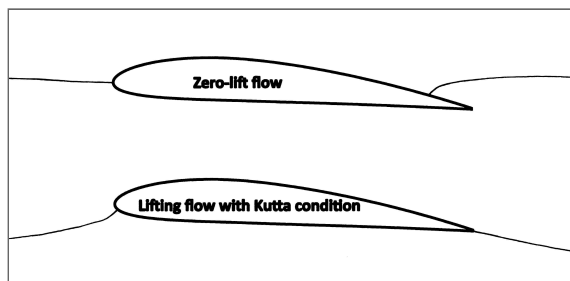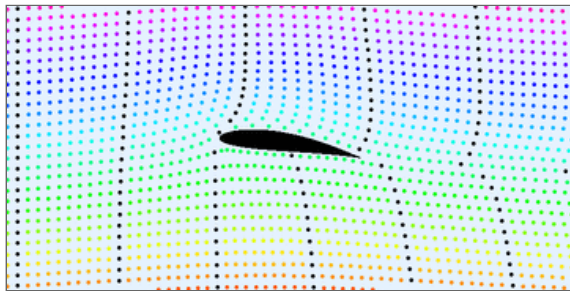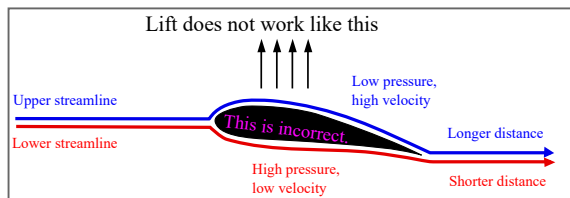*from **wikipedia**, **Attribution-Share Alike 4.0 International***

Hydrodynamic Paradox

`mf30 - Hydrodynamisches Paradoxon`

- Faster flow → lower pressure (Bernoulli).
- A fast jet can **pull** objects toward it instead of pushing them away.
- Thus, paper lifted toward a blowing stream, balls "sticking" in an air jet.

Aside: Why Bernoulli Cannot Explain Lift

- Bernoulli's "faster flow → lower pressure → lift" is **not** the full explanation
- Real lift requires viscosity & boundary layer, circulation (Kutta condition), asymmetric flow due to wing shape / angle of attack

Lift does not work like this

Upper streamline

Low pressure, high velocity

This is incorrect.

Lower streamline

Longer distance

High pressure, low velocity

Shorter distance

Zero-lift flow

Lifting flow with Kutta condition

Viscosity

`mf22 - Viskosität`

- Fluid between lower stationary plate and upper plate moving with v under tangential force $F_\parallel$

- **Viscosity = internal friction** between fluid layers

- Shear stress is proportional to the velocity gradient: $\dfrac{F_\parallel}{A} = \eta \dfrac{dv}{dy}$

- Required force to keep the plate moving:

$$F_\parallel = \eta A \, \frac{v}{d}$$

- Units of viscosity: $[\eta] = \mathrm{Pa} \cdot \mathrm{s}$ (Poise: $1\,\mathrm{P} = 0.1\,\mathrm{Pa} \cdot \mathrm{s}$)

*[left] from **wikipedia**, **Attribution-Share Alike 4.0 International**; [right]from **wikipedia** by Duk, **Attribution-Share Alike 3.0 Unported***

Parabolic Laminar Flow (Poiseuille flow)

- Tube wall is stationary $\rightarrow$ **no-slip condition**: $v = 0$ at $r = R$
- Fluid at the center moves fastest $\rightarrow$ velocity decreases toward the wall
- **Viscosity couples layers**: inner layers drag outer layers $\rightarrow$ smooth velocity variation
- Cylindrical geometry + viscous shear $\rightarrow$ **parabolic profile**

$$v(r) = v_{\max}\left(1 - \frac{r^2}{R^2}\right)$$

*from **wikipedia**, public domain*

Poiseuille's Equation (1): Setup, Assumptions, and Pressure Force

- Steady, incompressible, Newtonian fluid flow in rigid cylindrical tube of radius R

- **Viscosity causes energy loss $\Rightarrow$ pressure must drop along the tube**

Pressure acts on the cross-section $A_\perp = \pi r^2$ giving use the differential pressure force:

$$dF_P = -\, dP\, \pi r^2$$

Per unit length:

$$\frac{dF_P}{dx} = -\frac{dP}{dx}\, \pi r^2$$

Note: $-\frac{dP}{dx} > 0$ because pressure decreases downstream

Poiseuille's Equation (2): Viscous Force on the Cylindrical Shell

Viscous shear acts on the lateral surface $dA_\parallel = 2\pi r \cdot dx$ (circumference · length)

Differential viscous force:

$$dF_v = \eta\, dA_\parallel\, \frac{dv}{dr} = 2\pi r\, dx\, \eta\, \frac{dv}{dr}$$

Per unit length:

$$\frac{dF_v}{dx} = 2\pi r\, \eta\, \frac{dv}{dr}$$

Poiseuille's Equation (3): Force Balance and Velocity Profile

Set pressure and viscous forces equal (steady flow):

$$-\frac{dP}{dx}\,\pi r^2 = 2\pi r\,\eta\,\frac{dv}{dr}$$

Solve for dv by integrating:

$$\int_{v(r)}^{0} dv = -\frac{1}{2\eta}\,\frac{dP}{dx}\,\int_{r}^{R} r\,dr$$

$$v(r) = \frac{1}{4\eta}\,\frac{dP}{dx}\,(R^2 - r^2)$$

With constant pressure gradient $\frac{dP}{dx} = -\frac{\Delta P}{L}$, final velocity profile:

$$v(r) = \frac{\Delta P}{4\eta L}(R^2 - r^2)$$

Poiseuille's Equation (4): Volume Flow Rate

Sum contributions from all concentric shells with area element $dA = 2\pi r\, dr$:

$$Q = \int_0^R v(r)dA = \int_0^R v(r)\, 2\pi r\, dr.$$

Insert $v(r)$ give **Poiseuille's equation** for laminar flow through a circular tube:

$$Q = \frac{\pi R^4 \Delta P}{8\eta L}.$$

Poiseuille's Equation (5): Demo

`mf24 - Gesetz von Bernoulli und Hagen – Poiseuille`

$$v(r) = \frac{\Delta P}{4\eta L}(R^2 - r^2)$$

$$Q = \frac{\pi R^4 \Delta P}{8\eta L}.$$

$\rightarrow$ Poiseuille's equation links **pressure**, **radius**, **viscosity**, and **flow rate** in laminar flow.
$\rightarrow$ Flow depends very strongly on tube radius $\propto R^4$.
$\rightarrow$ Velocity profile is parabolic; $v_{\max} = 2v_{\mathrm{avg}}$.
$\rightarrow$ Viscosity causes linear pressure drop along the tube.
$\rightarrow$ Breaks down for turbulent or non-Newtonian flow.
Application: Blood flow regulation relies strongly on vessel diameter.

Reminder: Bernoulli's equation $P + \frac{1}{2}\rho v^2 + \rho g h = \text{constant}$

Navier–Stokes Equation: Motivation

- Real fluid flow can be **unsteady**, **accelerating**, or even **turbulent**
- Described by **incompressible Navier–Stokes equation**:

$$\frac{\partial \vec{v}}{\partial t} + (\vec{v} \cdot \nabla)\vec{v} = -\frac{1}{\rho}\nabla p + \nu\nabla^2\vec{v} + \vec{g}$$

- Instead of deriving it, we will **build intuition** using an **analogy from Prof. René Matzdorf**
- Imagine: **you are a person inside a dense moving crowd** → a small **fluid element** whose motion is influenced by surroundings.

Navier–Stokes Equation: Local Acceleration

- You can change your own speed $\Rightarrow$ **Velocity changes at a point because time passes, not because you moved elsewhere**

$$\frac{\partial \vec{v}}{\partial t}$$

Navier–Stokes Equation: Convection (Advection)

- Speed changes because you move into a region where the crowd flows differently
- Convective acceleration from spatial changes in the velocity field
- Nabla ∇ collects all spatial derivatives of a vector field

$$\nabla \vec{v} = \begin{pmatrix} \frac{\partial v_x}{\partial x} & \frac{\partial v_x}{\partial y} & \frac{\partial v_x}{\partial z} \\[2mm] \frac{\partial v_y}{\partial x} & \frac{\partial v_y}{\partial y} & \frac{\partial v_y}{\partial z} \\[2mm] \frac{\partial v_z}{\partial x} & \frac{\partial v_z}{\partial y} & \frac{\partial v_z}{\partial z} \end{pmatrix} \quad \Rightarrow \quad (\vec{v} \cdot \nabla)\vec{v} = \begin{pmatrix} v_x \frac{\partial v_x}{\partial x} + v_y \frac{\partial v_x}{\partial y} + v_z \frac{\partial v_x}{\partial z} \\[2mm] v_x \frac{\partial v_y}{\partial x} + v_y \frac{\partial v_y}{\partial y} + v_z \frac{\partial v_y}{\partial z} \\[2mm] v_x \frac{\partial v_z}{\partial x} + v_y \frac{\partial v_z}{\partial y} + v_z \frac{\partial v_z}{\partial z} \end{pmatrix}$$

- $(\vec{v} \cdot \nabla)\vec{v}$ = how $\vec{v}$ changes **in the direction of** $\vec{v}$

- **Change in velocity (i.e. acceleration) caused by moving through the flow**

Navier–Stokes Equation: Example of $(\vec{v} \cdot \nabla)\vec{v}$

- Example field $\vec{v} = (x, \ 2y, \ 0)$
- Insert $v_x = x$, $v_y = 2y$, $v_z = 0$ and all derivatives directly into the component formula

$$(\vec{v} \cdot \nabla)\vec{v} = \begin{pmatrix} v_x \frac{\partial v_x}{\partial x} + v_y \frac{\partial v_x}{\partial y} + v_z \frac{\partial v_x}{\partial z} \\[2ex] v_x \frac{\partial v_y}{\partial x} + v_y \frac{\partial v_y}{\partial y} + v_z \frac{\partial v_y}{\partial z} \\[2ex] v_x \frac{\partial v_z}{\partial x} + v_y \frac{\partial v_z}{\partial y} + v_z \frac{\partial v_z}{\partial z} \end{pmatrix} = \begin{pmatrix} x \frac{\partial x}{\partial x} + 2y \frac{\partial x}{\partial y} + 0 \frac{\partial x}{\partial z} \\[2ex] x \frac{\partial (2y)}{\partial x} + 2y \frac{\partial (2y)}{\partial y} + 0 \frac{\partial (2y)}{\partial z} \\[2ex] x \frac{\partial 0}{\partial x} + 2y \frac{\partial 0}{\partial y} + 0 \frac{\partial 0}{\partial z} \end{pmatrix}$$

- Evaluate derivatives

$$(\vec{v} \cdot \nabla)\vec{v} = \begin{pmatrix} x \cdot 1 + 2y \cdot 0 + 0 \cdot 0 \\ x \cdot 0 + 2y \cdot 2 + 0 \cdot 0 \\ x \cdot 0 + 2y \cdot 0 + 0 \cdot 0 \end{pmatrix} = \begin{pmatrix} x \\ 4y \\ 0 \end{pmatrix}$$

Navier–Stokes Equation: Gravity

- Crowd pulled downhill on a slope
- **Gravity acts as a body force everywhere in the fluid**

$$\vec{g}$$

Navier–Stokes Equation: Pressure Gradient

- Crowd denser on one side and looser on the other
- Net shove toward the emptier low-pressure region
- **Pressure force drives fluid from high to low pressure**
- ∇p uses the same spatial derivatives as before

$$-\frac{1}{\rho}\nabla p$$

Navier–Stokes Equation: Viscous Forces

- Shoulder-to-shoulder interactions pull or brake you $\Rightarrow$ **Faster regions drag slower ones, slower regions resist faster ones**
- Laplacian detects where a component of $\vec{v}$ is **higher or lower than its neighborhood**
- High relative values decrease, low relative values increase $\Rightarrow$ momentum diffusion
- Viscous forces **smooth out** velocity differences $\Rightarrow$ **viscous diffusion** term with $\nu = \eta/\rho$ as the kinematic viscosity:

$$\nu \nabla^2 \vec{v}$$

- Laplacian of a vector field $\vec{v} = (v_x, v_y, v_z)$ (each component uses the scalar Laplacian)

$$\nabla^2 \vec{v} = \left(\frac{\partial^2 v_x}{\partial x^2} + \frac{\partial^2 v_x}{\partial y^2} + \frac{\partial^2 v_x}{\partial z^2}, \; \frac{\partial^2 v_y}{\partial x^2} + \frac{\partial^2 v_y}{\partial y^2} + \frac{\partial^2 v_y}{\partial z^2}, \; \frac{\partial^2 v_z}{\partial x^2} + \frac{\partial^2 v_z}{\partial y^2} + \frac{\partial^2 v_z}{\partial z^2} \right.$$

Navier–Stokes Equation: Summary

$$\underbrace{\frac{\partial \vec{v}}{\partial t}}_{\text{local acceleration}} + \underbrace{(\vec{v} \cdot \nabla)\vec{v}}_{\text{convective acceleration}} = \underbrace{-\frac{1}{\rho}\nabla p}_{\text{pressure force}} + \underbrace{\nu \nabla^2 \vec{v}}_{\text{viscous diffusion}} + \underbrace{\vec{g}}_{\text{gravity}}$$
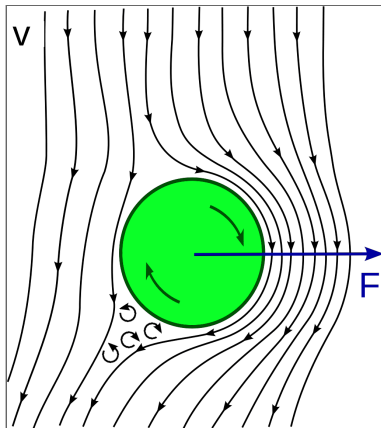
with incompressibility condition $\nabla \cdot \vec{v} = 0$

- **Local acceleration:** you change your own speed
- **Convective acceleration:** you move into a region with a different flow
- **Pressure force:** crowd pushes you from dense to less dense regions
- **Viscous diffusion:** neighbors slow you down or pull you forward
- **Gravity:** downhill pull on the whole crowd

Revisit initial experiment: The Magnus Effect
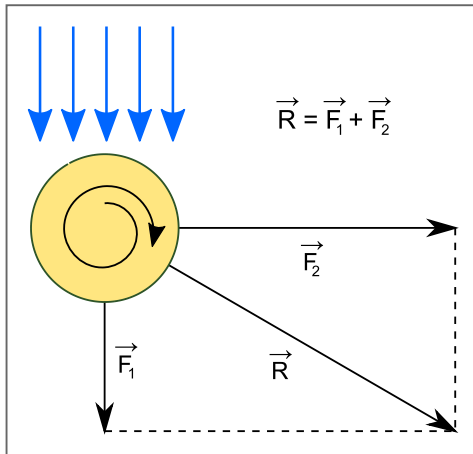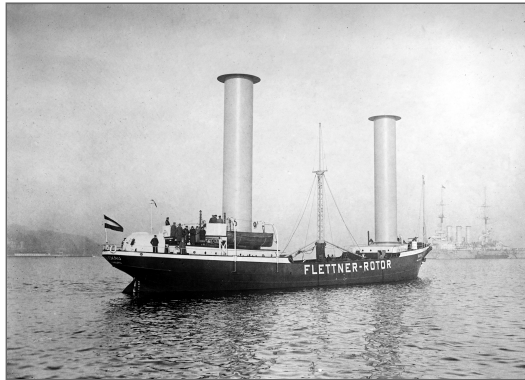
`mf23 - Magnus effect`

- A spinning object **drags fluid** in its boundary layer (viscosity)
- One side: faster airflow → **lower pressure**
- Other side: slower airflow → **higher pressure**
- Pressure difference creates a **sideways force**
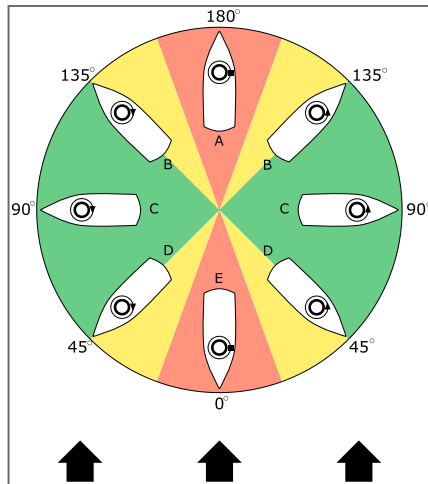- Explains curved paths of spinning balls; **requires viscosity**

Magnus Effect in Action: Flettner Rotor

`mf32 - Flettner`

$$\vec{R} = \vec{F_1} + \vec{F_2}$$

$\vec{F_2}$

$\vec{F_1}$

$\vec{R}$

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