

TTK4190 - Assignment 3

Group 27

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1 Part 2: Heading Autopilot

1.1 Problem 1 - Environmental Disturbances

1.1.1 Problem 1.a

With $\beta_{V_c} = 45^\circ$ and $V_c = 1$, and from using the following equations from [1]

$$u_c = V_c \cos(\beta_{V_c} - \psi)$$

$$v_c = V_c \sin(\beta_{V_c} - \psi)$$

```

125 for i=1:Ns+1
126
127     t = (i-1) * h;                % time (s)
128     R = Rzyx(0,0,eta(3));
129
130     % current (should be added here)
131     nu_c(1) = Vc*cos(betaVc - eta(3)); % surge current
132     nu_c(2) = Vc*sin(betaVc - eta(3)); % sway current
133
134     nu_r = nu - nu_c;

```

Listing 1: MATLAB code for 2D irrotational ocean current in surge and sway

and the resulting plots become

1.1.2 Problem 1.b

With zero ocean currents (and wind) affecting the vessel, there will be no crab or sideslip angle. This will result in the vessel following a straight path.

1.1.3 Problem 1.c

The angle of attack between the wind and the boat is given as

$$\gamma_w = \psi - \beta_{V_w} - \pi \quad (1)$$

The wind-coefficients are approximated as in [1] in sway and yaw direction as

$$C_Y(\gamma_w) \approx c_y \sin(\gamma_w) \quad (2)$$

$$C_N(\gamma_w) \approx c_n \sin(2\gamma_w) \quad (3)$$

The moment

$$\tau_{wind} = \begin{bmatrix} \tau_{Y_{wind}} \\ \tau_{N_{wind}} \end{bmatrix} = \begin{bmatrix} \frac{\rho_a V_{ra}^2}{2} C_Y(\gamma_{rw}) A_{L_w} \\ \frac{\rho_a V_{ra}^2}{2} C_N(\gamma_{rw}) A_{L_w} L_{oa} \end{bmatrix} \quad (4)$$

```

137 if t > 200
138     % UNSURE ABOUT RELATIVE VS ABSOLUTE WIND SPEED AND ANGLE-OF-ATTACK
139     gamma_w = eta(3) - betaVw - pi;
140     C_Ywind = cy*sin(gamma_w); % sway wind coefficient
141     C_Nwind = cn*sin(2*gamma_w); % yaw wind coefficient
142     Ywind = 1/2*rho_a*Vw^2*C_Ywind*A_Lw; % expression for wind moment in ...
143     Nwind = 1/2*rho_a*Vw^2*C_Nwind*A_Lw*L; % expression for wind moment in ...
144     else
145         Ywind = 0;
146         Nwind = 0;
147     end
148     tau_env = [0 Ywind Nwind]';

```

Listing 2: Wind movement in sway and yaw.

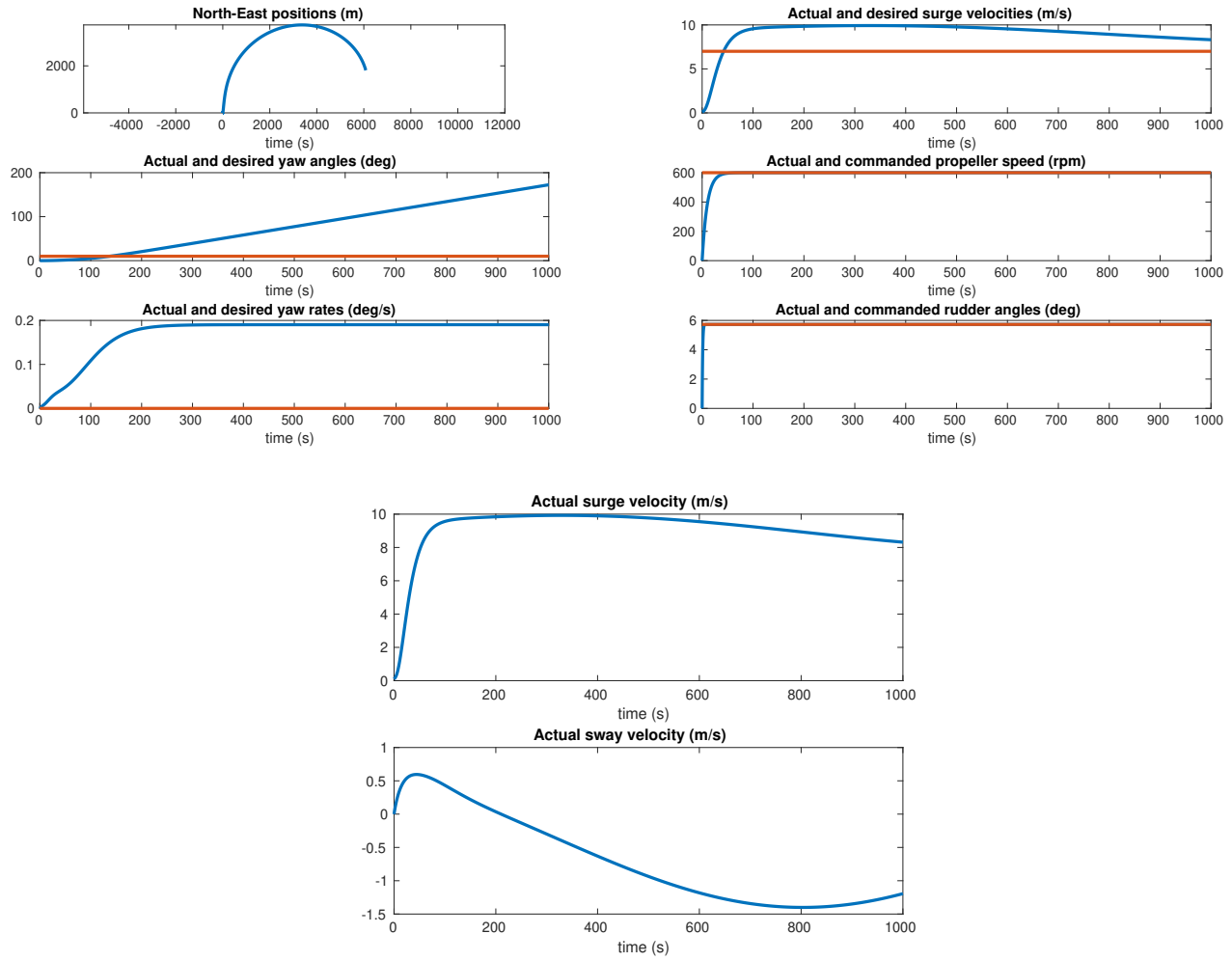


Figure 1: Boat behaviour with current disturbance

1.2 Problem 2 - Heading Autopilot

1.2.1 Problem 2.a

From the given matlab code we get that

$$\mathbf{C}_A = \begin{bmatrix} 0 & 0 & Y_{\dot{v}}v + Y_{\dot{r}}r \\ 0 & 0 & -X_{\dot{u}}u \\ -Y_{\dot{v}}v - Y_{\dot{r}}r & X_{\dot{u}}u & 0 \end{bmatrix} \quad (5)$$

$$\mathbf{C}_{RB} = \begin{bmatrix} 0 & -mr & -mx_g r \\ mr & 0 & 0 \\ mx_g r & 0 & 0 \end{bmatrix} \quad (6)$$

and the nonlinear coriolis forces become

$$\mathbf{C}_A(\nu)\nu = \begin{bmatrix} Y_{\dot{v}}vr + Y_{\dot{r}}r^2 \\ -X_{\dot{u}}ur \\ -Y_{\dot{v}}vu - Y_{\dot{r}}ru + X_{\dot{u}}uv \end{bmatrix} \quad (7)$$

$$\mathbf{C}_{RB}(\nu)\nu = \begin{bmatrix} -mrv - mx_g r^2 \\ mru \\ mx_g ru \end{bmatrix} \quad (8)$$

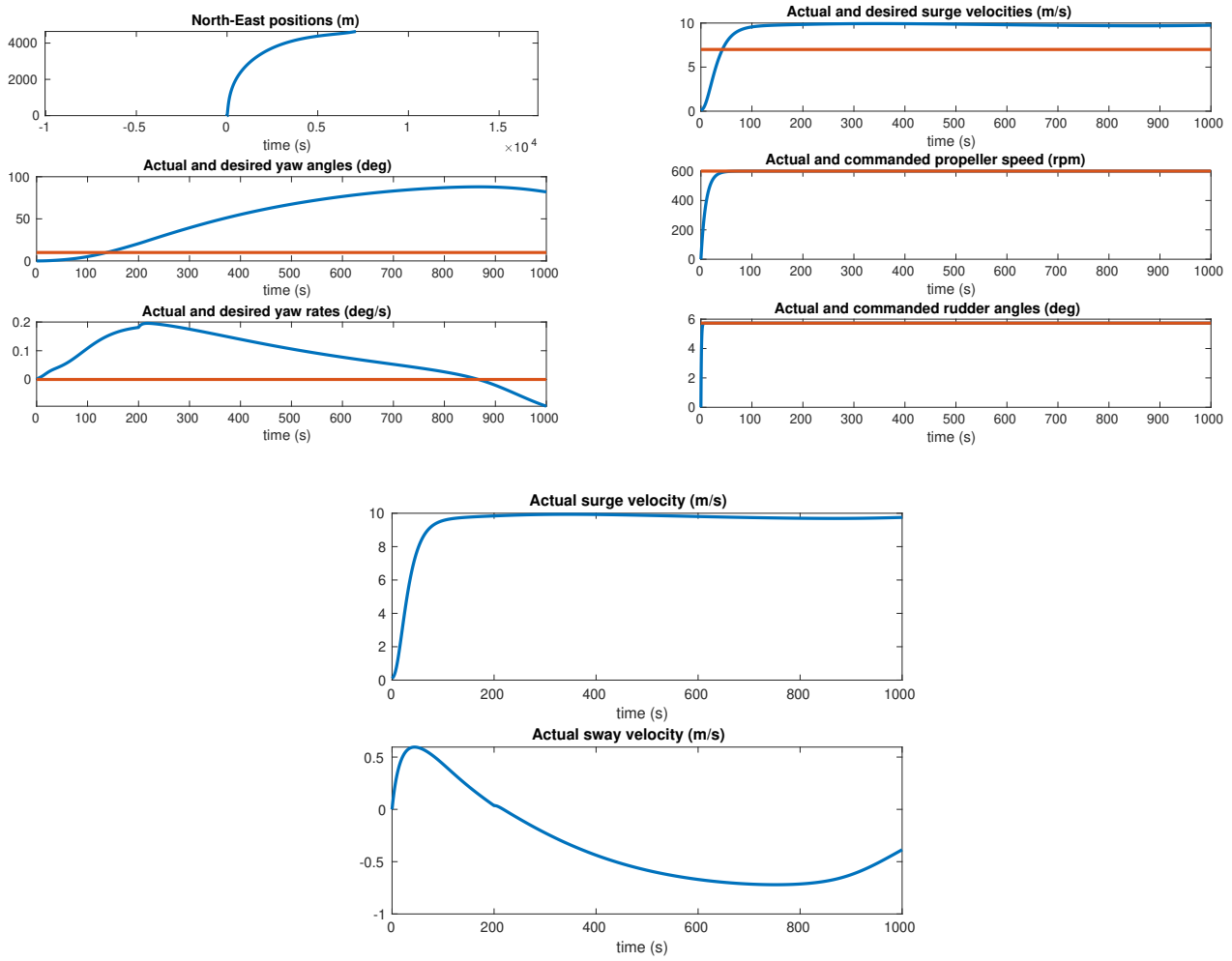


Figure 2: Boat behaviour with current disturbance and wind disturbance after 200 seconds

Linearized at $\nu = \nu^*$ we get

$$\mathbf{C}_A^* = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -X_{\dot{u}}U_d \\ 0 & -Y_{\dot{v}}U_d + X_{\dot{u}}U_d & -Y_{\dot{r}}U_d \end{bmatrix} \quad (9)$$

$$\mathbf{C}_{RB}^* = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & mU_d \\ 0 & 0 & mx_gU_d \end{bmatrix} \quad (10)$$

1.2.2 Problem 2.b

The model is given in [1] as

$$M\dot{\nu}_r + N\nu_r = b\delta \quad (11)$$

and by rearranging we get that

$$\dot{\nu}_r = -M^{-1}N\nu_r + M^{-1}b\delta \quad (12)$$

The maneuvering model can be expressed as a state-space model as

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u},$$

$$\mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}\mathbf{u}$$

where

$$\mathbf{A} = -\mathbf{M}^{-1}\mathbf{N}$$

$$\mathbf{B} = \mathbf{M}^{-1}\mathbf{b}$$

$$\mathbf{C} = \begin{bmatrix} 0 & 1 \end{bmatrix}$$

$$\mathbf{D} = 0$$

The resulting transfer function can be approximated by using `ss2tf(.)`

$$\frac{r}{\delta}(s) = \frac{8.04 \cdot 10^{-5}s + 6.15 \cdot 10^{-6}}{s^2 + 0.151s + 8.20 \cdot 10^{-4}} = \frac{K(T_3s + 1)}{(T_1s + 1)(T_2 + 1)} \quad (13)$$

With the numerical values yielding $K = 7.50 \cdot 10^{-3}$, $T_3 = 14.054$, $T_2 = 117.70$ and $T_1 = 6.90$.

1.2.3 Problem 2.c

Going from Equation (13) to the Nomoto model

$$\frac{r}{\delta}(s) = \frac{K}{Ts + 1} \quad (14)$$

where $T := T_1 + T_2 - T_3 = 169.55$ and $K = 7.50 \cdot 10^{-3}$

1.2.4 Problem 2.d

A PID controller for heading autopilot is created using pole placement. We can also see that the actual yaw angle and yaw rate is following the desired yaw angle and yaw rate. From the north east position we can see that the course angle is different from zero, however this is because we have created a heading controller, not a course controller. Because of the slow dynamics, the controller use some time to reach the desired angle. That is what causes the north east angle.

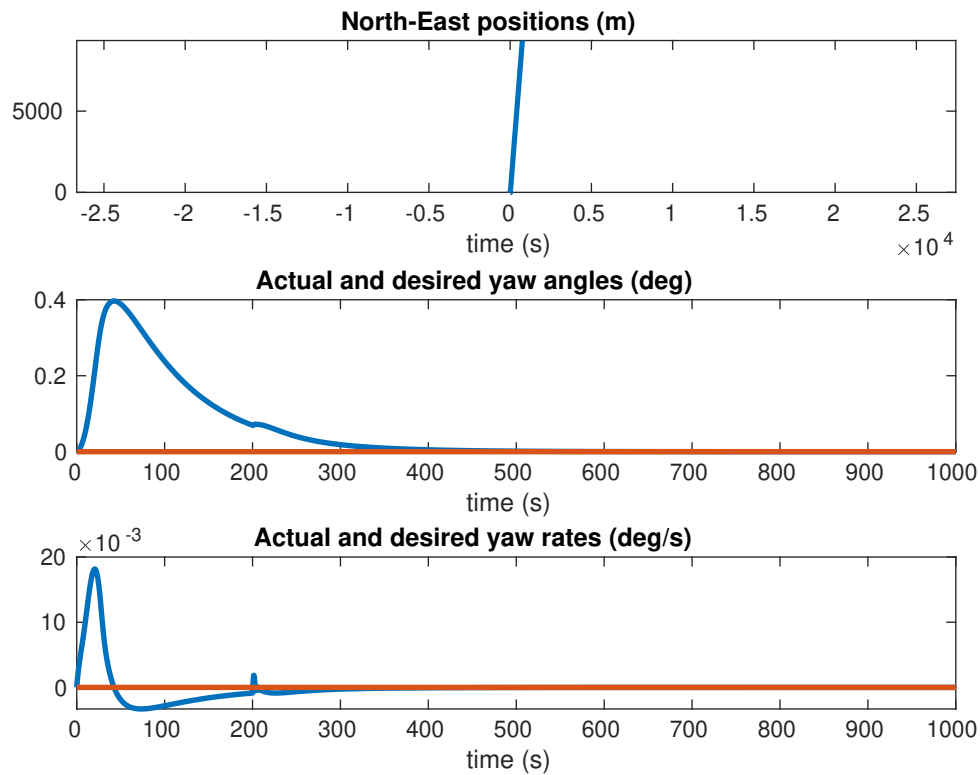


Figure 3: Boat behaviour with PID controller for heading autopilot.

1.2.5 Problem 2.e

The heading autopilot can compensate for the 10 deg to 20 deg maneuver, as can be seen in the North East position plot in fig. 4. Integral windup is not necessary in this case because we don't have saturation in the actuator. However it is good practice to implement.

```

110 M_lin = MRB + MA;
111 M_lin = M_lin(2:3,2:3);
112 M_lin_inv = inv(M_lin);
113
114 CRB_lin = [ 0      m*U_d
115             0      m*xg*U_d ];
116
117 CA_lin = [ 0      -Xudot*U_d
118            -Yvdot*U_d+Xudot*U_d  -Yrdot*U_d ];
119
120 N_lin = CRB_lin + CA_lin + D(2:3,2:3);
121 b_lin = [-2*U_d*Y_Δ -2*U_d*N_Δ]';
122
123 [num, den] = ss2tf(-M_lin_inv*N_lin, M_lin_inv*b_lin, [0 1], 0);
124
125 K_lin = num(3)/den(3); % gain can be found by using the steady-state ...
126                        value (s=0) of the transfer function
127
128 poles_lin = roots(den);
129 zeros_lin = roots(num);
130
131 T3_lin = -1/zeros_lin;
132 T1_lin = -1/poles_lin(1);
133 T2_lin = -1/poles_lin(2);

```

```

134 T_lin = T1_lin+T2_lin-T3_lin; % nomoto first-order time constant, eq. (7.24)
135
136 % controller gains (example 15.7)
137 Kp = wn^2*T_lin/K_lin;
138 Kd = (2*zeta*wn*T_lin-1)/K_lin;
139 Ki = wn/10*Kp;

```

Listing 3: Linearized C_a and C_{RB} and the gains for PID controller using pole placement

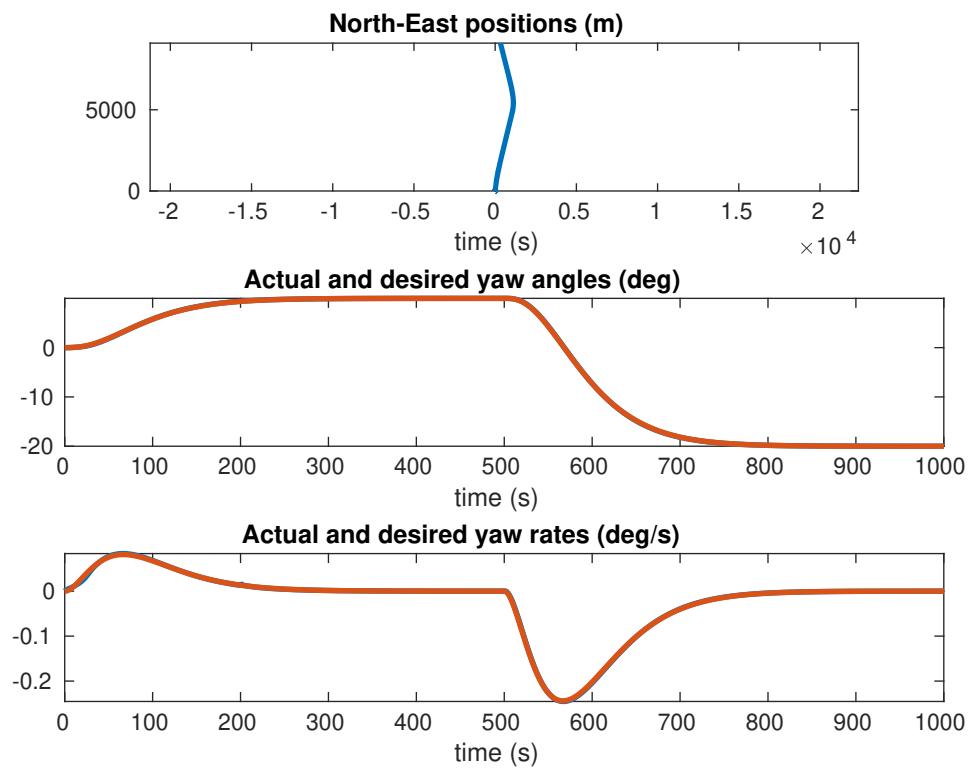


Figure 4: Boat behaviour when doing a 10-20 degree maneuver.

2 Part 3: Speed Control

2.1 Problem 1: Propeller Revolution and Speed Control

2.1.1 Problem 1a

After using the wageningen-function, the updated values are $K_T = 0.6367$ and $K_Q = 0.1390$

2.1.2 Problem 1b

The code is updated to include the dynamics of the prime mover system with the following equations.

$$\frac{Q_m}{Y}(s) = \frac{K}{Ts + 1} \quad (15)$$

$$\dot{Q}_m = -\frac{Q_m}{T} + \frac{K}{t}(m_c - n) \quad (16)$$

The shaft speed dynamics \dot{n} is also updated, and the code is shown below.

$$\dot{n} = \frac{Q_m - Q - Q_f}{I_m} \quad (17)$$

2.1.3 Problem 1c

```

256 % propeller dynamics
257 Im = 100000; Tm = 10; Km = 0.6; % propulsion parameters
258
259 % added feedforward
260 Td = U_d*Xu/(t_thr-1); % desired thrust (N)
261
262 n_term = Td/(rho * Dia^4 * KT);
263 n_d = sign(n_term) * sqrt(abs(n_term)); % desired propeller speed (rps)
264
265 Qf = 0; % friction torque (Nm)
266 Qd = rho * Dia^4 * KQ * abs(n_d) * n_d; % desired propeller moment (Nm)
267 Y = Qd/Km; % control input to main motor
268
269 Qm_dot = -Qm/Tm + Km/Tm*Y;
270 n_dot = (Qm-Q-Qf)/Im;
271
272 % store simulation data in a table (for testing)
273 simdata(i,:) = [t n_d Δ_c n Δ eta' nu' u_d psi_d r_d z];
274
275 % Euler integration
276 xd = euler2(xd_dot,xd,h); % reference model
277 z = euler2(e.psi,z,h); % integral state
278 Qm = euler2(Qm_dot,Qm,h);
279 eta = euler2(eta_dot,eta,h);
280 nu = euler2(nu_dot,nu,h);
281 Δ = euler2(Δ_dot,Δ,h);
282 n = euler2(n_dot,n,h);

```

Listing 4: Full code propeller dynamics

2.1.4 Problem 1d

Assuming $u_r = u$, $\dot{u} = \dot{u}_r = 0$, $u = U$, $x_{\delta\delta} = 0$, U is given by

$$U = \frac{(t-1)T}{x_u} \quad (18)$$

2.2 Problem 1e

With the equation from the last problem we get

$$T_d = \frac{U_d x_u}{t - 1} \quad (19)$$

and

$$n_{term} = \frac{T_d}{\rho * d^4 * K_T} \quad (20)$$

$$n_d = \text{sign}(n_{term}) \sqrt{\text{abs}(n_{term})} \quad (21)$$

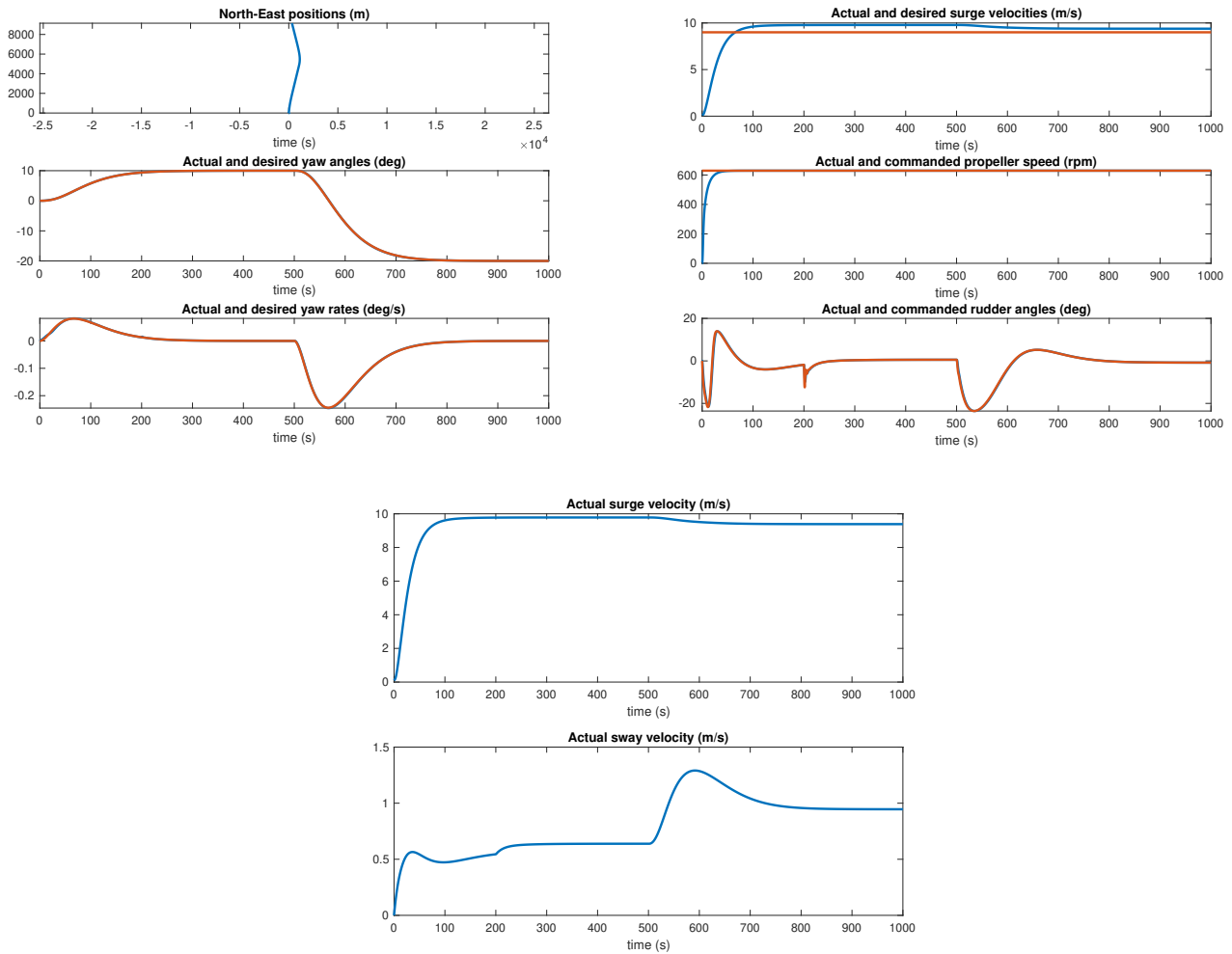


Figure 5: Ship behaviour with Propeller Revolution and Speed Control

No, we do not achieve the desired speed with constant heading angle. The surge velocity has a stationary error and does not reach the reference speed of $U_d = 9[m/s]$. This is because of the assumptions in the equation in problem 1d. To fix this we can add a feed-forward controller:

```
1 %Problem 1e: added feedforward
2 Td = (U_d-nu_c(1))*Xu/(t_thr-1);
```

Listing 5: Feed-forward

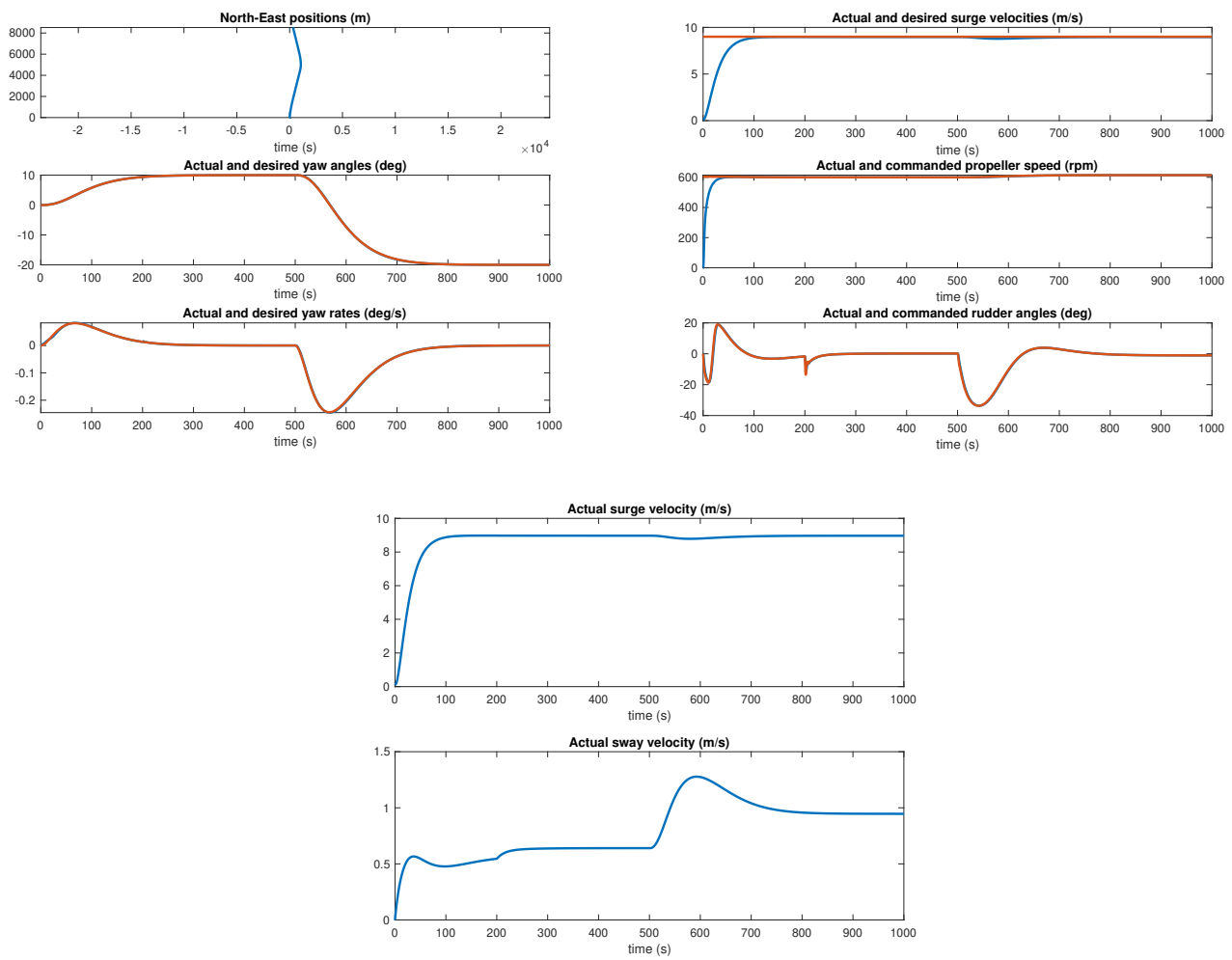


Figure 6: Ship behaviour with feed-forward

2.3 Problem 1f

Yes, with a $20[deg]$ setpoint change in heading, the speed will drop for a small period. This can be seen in figure 6. This is because of the sway velocity during the turn, so it need to be decomposed to be corrected to reference again.

References

- [1] T.I. Fossen. *Handbook of Marine Craft Hydrodynamics and Motion Control*. John Wiley & Sons, 2021.