- **2.1 a.** Obtain the global stiffness matrix \underline{K} of the assemblage shown in Figure P2–1 by superimposing the stiffness matrices of the individual springs. Here k_1, k_2 , and k_3 are the stiffnesses of the springs as shown.
 - **b.** If nodes 1 and 2 are fixed and a force *P* acts on node 4 in the positive *x* direction, find an expression for the displacements of nodes 3 and 4.
 - c. Determine the reaction forces at nodes 1 and 2.

(*Hint:* Do this problem by writing the nodal equilibrium equations and then making use of the force/displacement relationships for each element as done in the first part of Section 2.4. Then solve the problem by the direct stiffness method.)

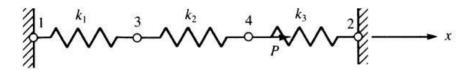


Figure P2-1

3.12 Solve for the axial displacement and stress in the tapered bar shown in Figure P3–12 using one and then two constant-area elements. Evaluate the area at the center of each element length. Use that area for each element. Let $A_0 = 2$ in², L = 20 in., $E = 10 \times 10^6$ psi, and P = 1000 lb. Compare your finite element solutions with the exact solution.

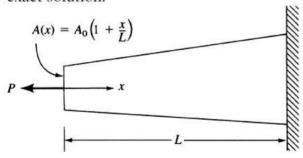


Figure P3-12

$$K'' = \begin{bmatrix} F_1 & O - F_1 & O \\ O & O & O & O \\ -K_1 & O & O & O \end{bmatrix}$$

$$\begin{bmatrix}
-k_1 & 0 & k_1 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}$$

$$\begin{bmatrix}
-k_1 & 0 & k_1 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}$$

$$\begin{bmatrix}
-k_1 & 0 & -k_1 & 0 \\
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}$$

$$\begin{bmatrix}
-k_1 & 0 & -k_1 & 0 \\
0 & k_3 & 0 & -k_3 \\
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}$$

$$\begin{bmatrix}
-k_1 & 0 & -k_1 & 0 \\
0 & k_3 & 0 & -k_3 \\
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}$$

$$\begin{bmatrix}
-k_1 & 0 & -k_1 & 0 \\
0 & k_3 & 0 & -k_3 \\
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}$$

$$\begin{bmatrix}
-k_1 & 0 & -k_1 & 0 \\
0 & k_3 & 0 & -k_3 \\
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}$$

b.
$$\begin{cases} k_1 & 0 - k_1 & 0 \\ 0 & k_3 & 0 - k_3 \\ -k_1 & 0 & k_1 k_2 - k_2 \\ 0 & -k_3 & -k_2 & k_2 + k_3 \end{cases} \begin{pmatrix} d_1 x \\ d_2 x \\ d_3 x \end{pmatrix} = \begin{pmatrix} F_1 x \\ 0 \\ F_2 x \end{pmatrix} = \begin{pmatrix} 0 \\ P \\ F_3 x \end{pmatrix} = \begin{pmatrix} k_1 + k_2 & -k_2 \\ -k_3 & k_2 + k_3 \end{pmatrix} \begin{pmatrix} u_{3} x \\ u_{4} x \end{pmatrix}$$

$$(F)=(k)(d) \rightarrow [k-1](k)=(k)(d)=(d)$$

$$d_{1x} = 0$$

$$F_{2x} = -k_3 d_{4x} = \frac{-k_3(k_1+k_2)p}{k_1k_2+k_1k_3+k_2k_3}$$

3.12
$$S = \frac{F \cdot L}{EA} : J = E \cdot E$$

$$J = \frac{F}{A}, E = \frac{G}{L}$$

$$\int = \int \frac{f}{EA} dx = \int \frac{P dx}{EA \cdot (1+\frac{2}{c})} = \int \frac{P \cdot L \cdot dx}{EA \cdot (1+\frac{2}{c})} dx$$

$$= \int \frac{P \cdot L \cdot dx}{EA \cdot (1+\frac{2}{c})} = \int \frac{P \cdot L \cdot dx}{EA \cdot (1+\frac{2}{c})} dx$$

$$\frac{P \cdot l \, du}{EA_0 \, u} = \frac{Pl}{EA_0} \cdot \ln u = \frac{Pl}{EA_0} \ln \left(l + x \right) = \frac{-1000 \cdot 20}{10 \times 10^6 \times 2} \ln \left(20 + x \right) = +0^{-3} \ln \left(20 + x \right)$$

.
$$\chi = 0$$
 $\rightarrow -10^{-3} \ln(20) = -2.996 \times 10^{3}$ in.

.
$$\chi = 0$$
 $\rightarrow -10^{-3} \ln(20) = -2.996 \times 10^{3} \text{ in.}$
. $\chi = 10^{-3} \ln(20 + 10) = -3.40 \times 10^{3} \text{ in.}$

$$P \leftarrow \begin{cases} \frac{1.5EA_0}{L} & \frac{1.5EA_0}{L} & \frac{1.5EA_0}{L} \\ \frac{1.5EA_0}{L} & \frac{1.5EA_0}{L} & \frac{1.5EA_0}{L} \end{cases} \qquad \begin{cases} V_{1\chi} = 0 \\ V_{2\chi} = 0 \end{cases}$$

$$= \frac{-PL}{1.5 E \cdot A_0} = \frac{-1000 \times 20}{1.5 \times 10 \times 10^6 \times 2} = -0.667 \times 10^{-3} \text{ in}$$

$$\frac{1.25 \pm A_{0}}{L/2} U_{1X} - \frac{1.25 \pm A_{0}}{L/2} U_{0X} = -P$$

$$\frac{-1.25 \pm A_{0}}{L/2} U_{1X} + \frac{3 \pm A_{0}}{L/2} U_{2X} = O$$

$$= -\frac{2}{\eta} \chi \frac{|000 \times 20|}{2 \times |0 \times |0|} = -0.285 \eta \times |0|^{-3} in.$$

$$U_{1X} = \frac{12}{5}U_{2X} = -0.68110^3$$
 in