Phone Case Heat Dissipation Analysis





Introduction: Heat Dissipation in Phones

- Phones are designed with heat dissipation in mind
- Cases and covers affect the designed dissipation abilities



Objective

 To study how the addition of a phone case or skin affects the rate of heat loss from the phone via all three modes of heat transfer

 Doing so helps in understanding whether adding a case/skin benefits or hinders heat dissipation



Method: Heating the Phone

The phone was heated screen-side-up using a heat lamp until the back side measured 60 °C.

This was to make sure that every trial would have a consistent amount of thermal energy added to the system.

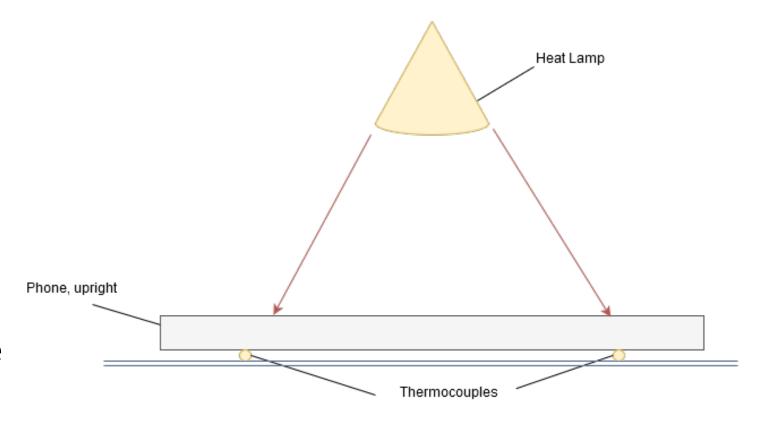
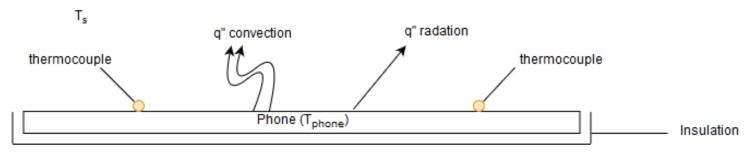


Figure: Diagram of experimental setup for heating up the phone.

Method: Monitoring Rate of Head Dissipation

- The phone was then placed screen down in the foam case, insulating it from all sides except the back.
- Thermocouples on the back of the phone monitor the temperatures at multiple points on the phone during the cooling process, until it reached steady state.

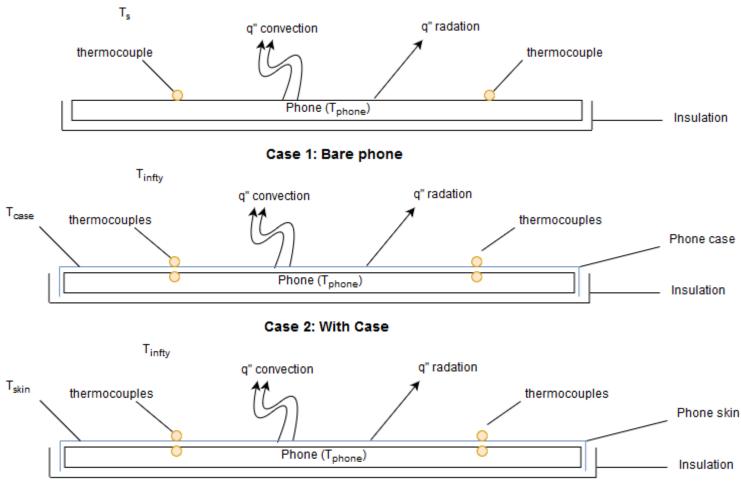


Case 1: Bare phone

Method: Three Different Cases

- The process was repeated with the following individual additions to the phone:
 - Silicon case
 - Matte black vinyl skin
- Extra thermocouples were added to the system to monitor the temperatures of the case and the skin in the separate cases.

Analysis: Schematics



Case 3: With Skin

Analysis: Governing Equation (Bare Phone)

$$\dot{E}_{stored} = \dot{E}_{in} - \dot{E}_{out} + \dot{E}_{generated}$$

$$V_{phone} C_{phone} \frac{dT}{dt} = -q_{conv} - \sigma \varepsilon A (T_S^4 - T_\infty^4)$$

Assumptions

- Insulated on sides
- No heat generation
- Square plate
- Uniform temperature
- Gray body

Knowns

- Heat Capacity of phone
- Dimensions
- Emissivity
- Temperature data

Analysis: Governing Equation (w/Case, w/Skin)

$$\begin{split} \dot{E}_{stored} &= \dot{E}_{in} - \dot{E}_{out} + \dot{E}_{generated} \\ V_{phone} C_{phone} \frac{dT}{dt} &= q_{cond,exp} \\ V_{case} C_{case} \frac{dT}{dt} &= q_{cond,exp} - q_{conv,exp} - \sigma \varepsilon A (T_s^4 - T_\infty^4) \\ q_{cond,th} &= k_{case} A (T_{phone} - T_s) \end{split}$$

Assumptions

- Insulated on sides
- No heat generation
- Square plate
- Uniform temperatures
- Gray body
- 1-D Conduction

Knowns

- Heat Capacity of phone
- Dimensions
- Emissivity
- Temperature data
- Convective coefficient
- Case properties

Result Validation: Calculating h and \mathcal{C}_{phone}

- To calculate C_{phone} , Colorado University Study was used
 - Provided data of multiple phone components capacities
- Calculated the aluminum heat capacity
- Averaged everything
 - Around the same magnitude
- $C_{phone} = \sim 2 \times 10^6 \frac{J}{K \times m^3}$
- Is validated by test without case

• To calculate theoretical h, assumption of natural convection over flat plate

•
$$Ra = \frac{|\Delta T|gL_c^3\rho^2c_{p,air}}{T\mu k_{air}}$$

- Air properties known, taken at 318K
- $Nu = 0.59Ra^{\frac{1}{4}}$
- $h = Nu\frac{k}{L}$
- Will be calculated at every temperature point and compared to experimental
- $q_{conv,th} = hA_{phone}(T_s T_{\infty})$

Analysis: Sample Calculations (Bare)

- $Area = 0.0105 m^2$
- $Volume = 8.26E(-5)m^3$
- $C_{phone} = 2.4E(6) \frac{J}{m^3 K}$
- $T_{phone} = 55$ °C
- $T_{\infty} = 24.5^{\circ}C$
- $\bullet \frac{dT}{dt} = -0.024 \frac{K}{s}$
- $q_{rad} = 1.827W$
- $L_c = 0.024m$

$$-(8.26E(-5)m^{3})\left(2.4E(6)\frac{J}{m^{3}K}\right)\left(-0.024\frac{K}{s}\right)$$

$$= q_{conv} + 1.815W$$

$$q_{conv,exp} = 2.93W$$

$$Ra = 29421.46$$

$$Nu = (0.54)Ra^{\frac{1}{4}} = 7.07$$

$$h = \frac{Nu(k_{air})}{L_c} = \frac{(7.07)\left(0.02744\frac{W}{mK}\right)}{0.024m} = 8.09\frac{W}{m^2K}$$

$$q_{conv,th} = \left(8.09\frac{W}{m^2K}\right)(0.0105 m^2)(55^{\circ}\text{C} - 24.5^{\circ}\text{C})$$

$$q_{conv,th} = 2.59W$$

Analysis: Sample Calculations (w/ Case, w/Skin)

•
$$t = 0.002m$$

•
$$k = 0.11 \frac{W}{mK}$$

•
$$Vol_{case} = 2.1E(-05)m^3$$

•
$$T_{case} = 46.7$$
°C

•
$$\frac{dT}{dt_{phone}} = -0.02118 \frac{K}{s}$$

•
$$c_{p,case} = 1350 \frac{J}{kgK}$$

•
$$\rho_{case} = 1100 \frac{kg}{m^3}$$

•
$$q_{rad} = 1.46W$$

$$q_{cond,exp} = -(8.26E(-5)m^3) \left(2.4E(6)\frac{J}{m^3K}\right) \left(-0.02118\frac{K}{s}\right)$$

$$q_{cond,exp} = 4.2W$$

$$q_{cond,th} = \frac{\left(0.11\frac{W}{mK}\right)}{0.002m} (0.0105 m^2)(55 - 46.7) = 4.81W$$

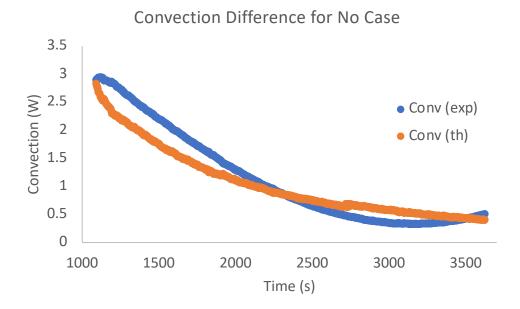
$$(2.1E(-05)m^3) \left(1350\frac{J}{kgK}\right) \left(1100\frac{kg}{m^3}\right) \left(-0.01525\frac{K}{s}\right)$$

$$= 4.2W - q_{conv,exp} - 1.46W$$

 $q_{conv,exp} = 3.21W$

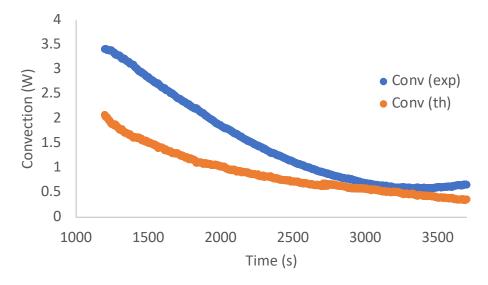
 $q_{conv,th} = 1.75W$

Results: Convection

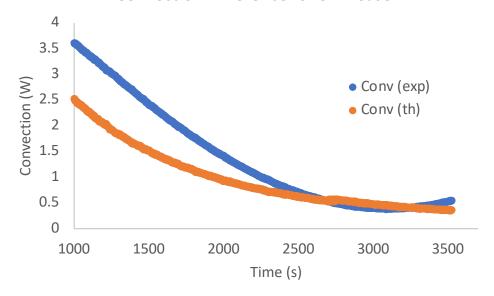


- Observable differences
- Error possibilities:
 - Properties at wrong temperatures
 - Steady State

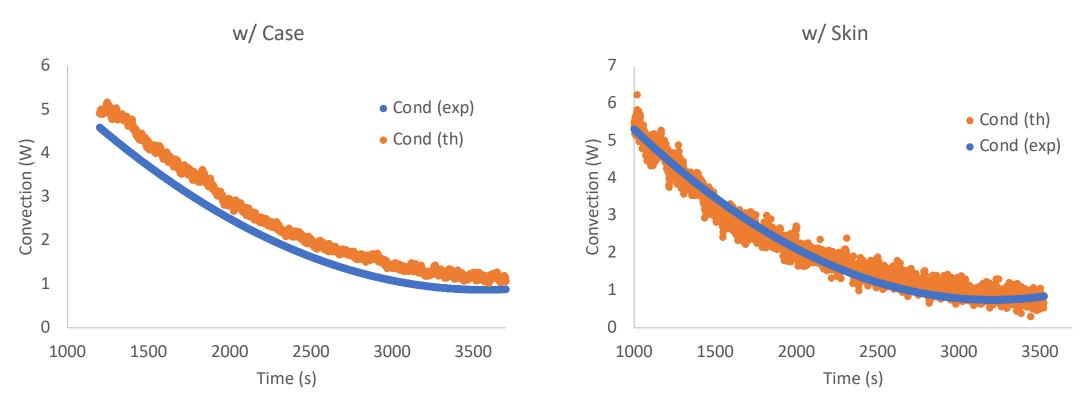
Convection Difference for Silicone Case



Convection Difference for Skin Case

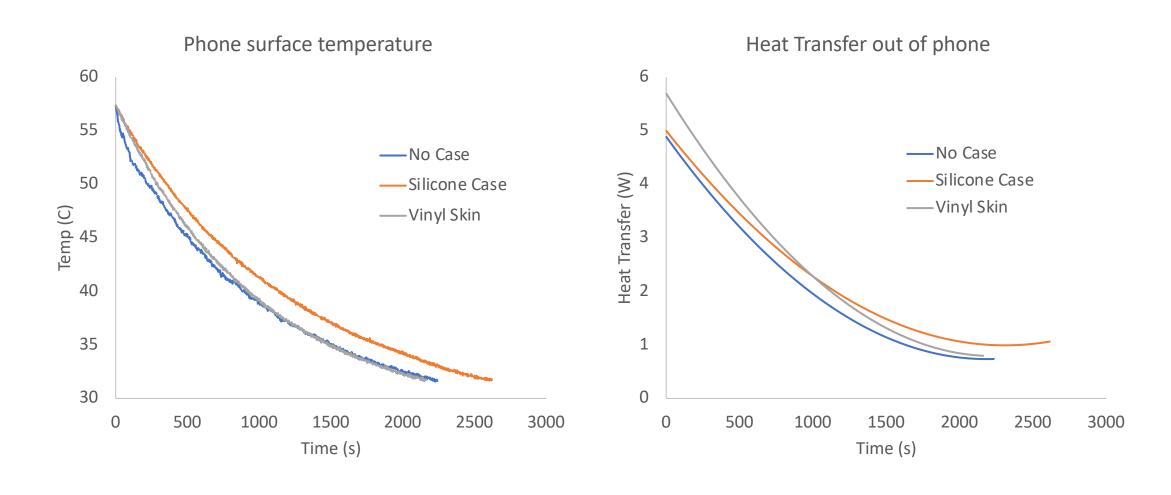


Results: Conduction



- Small differences
- Conduction matches expected results
- Possible disparities are thermal conductivity and thickness

Results: Phone Temp. and Heat Transfer vs t



Results: Overall

- Original estimate for phone heat capacity was not too far off
 - However, heat capacity directly influences results
- Theoretical plate results prove more accurate for steadier state
 - As observed on graphs, results approach at lower temperatures
- Conduction results proved accurate
 - Depending on k and t
- Silicone case definitely slows cooling of phone down
 - Took over 6 minutes to reach same temperature
 - Significant temperature differences (55°C vs 46.7°C)
- Results with skin proved faster than without a case
 - Reached same temperature a minute under phone
 - Maybe due to difference in emissivity (0.82 vs 0.94)

Uncertainty Analysis

• Absolute uncertainty of k-type thermocouple $=\pm 2.2~^{\circ}C~or~\pm 2.2~K$

- At 45 °C (318 K):
 - Relative uncertainty = $\frac{2.2 \text{ K}}{318 \text{ K}} = 0.692\%$
 - Therefore, uncertainty in $r_{rad}=2.77\%$ and $r_{conv}=2.77\%$
- Uncertainty well within 5%

Discussion

- As expected, silicone case slowed cooling of phone
 - Due to the conduction aspect and thickness
 - Does not seem damaging to the phone
 - Use material with higher thermal conductivity for heat dissipation
 - Lowering thickness lowers protection
- Skin did not have same effects since much thinner
 - Cooled down faster than phone by itself
 - Provides less protection
 - Radiation probably the cause

Sources of Error

- Phone was assumed flat plate
- Phone had a section in the back not covered by skin/case
- Thermocouples taped
- Heat Capacity approximation
- Using trend line to find dT/dt
- Thermal conductivity and thickness data
- Perfect insulation assumption
- Constant ambient temperature

Conclusion & Recommendations

- Heat capacity must be investigated further for accuracy
 - Directly impacts both convection and conduction results
- Silicone case protects better but reduces heat dissipation
- Use higher conducting materials to help
- Phone won't blow up because of it, might reduce performance
- Skin slightly helps phone cool
- Skin does not offer protection, only versus scratches
- Possibly has higher absorptivity due to color

Phone Case Heat Dissipation Analysis

Q&A







References

- Slide 1 image: https://www.zteusa.com/axon-7mini
- Slide 2 image: https://www.techadvisor.co.uk/how-to/mobile-phone/phone-overheating-3647780/
- Slide 3 image: https://mobileayes.com/cases/pc-cooling-heat-dissipation-mobile-phone-case/
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- Thermodynamic properties of air, and relevant equations: Fundamentals of Heat and Mass transfer [7th ed.] (Bergman, Lavine, Incropera & Dewitt)
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- Silicone case thermal properties: https://www.electronics-cooling.com/2001/11/the-thermal-conductivity-of-rubbers-elastomers/