

UNIVERSIDADE DO ESTADO DO RIO DE JANEIRO  
FACULDADE DE ENGENHARIA  
DEPARTAMENTO DE ENGENHARIA MECÂNICA

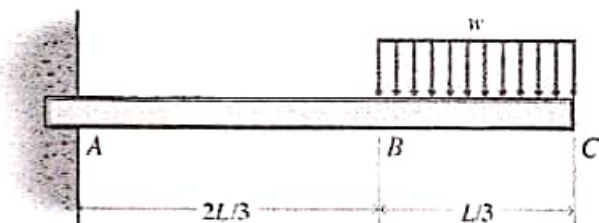
**2ª PROVA DE RESISTÊNCIA DOS MATERIAIS VII – 2020-1**

1ª Questão (2,5 pontos) – Para a viga da figura, determinar a equação da curva elástica e o deslocamento e rotação no ponto C. (Figura 1)

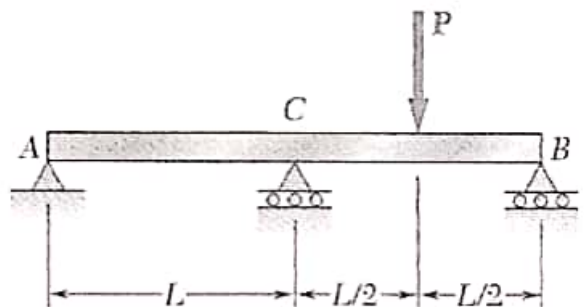
2ª Questão (2,5 pontos) – Determine as reações nos apoios e trace os diagramas de esforço cortante e momento fletor.  $EI = \text{constante}$ . (Figura 2)

3ª Questão (2,5 pontos) – O suporte ABC da figura suporta um carregamento vertical  $W$  na junta B. Cada membro é um tubo circular esbelto de aço ( $E = 200 \text{ GPa}$ ) com diâmetro externo de 100 mm e espessura da parede de 6,0 mm. A distância entre os suportes é 7,0 m. Determine o valor crítico de  $W$ . (Figura 3)

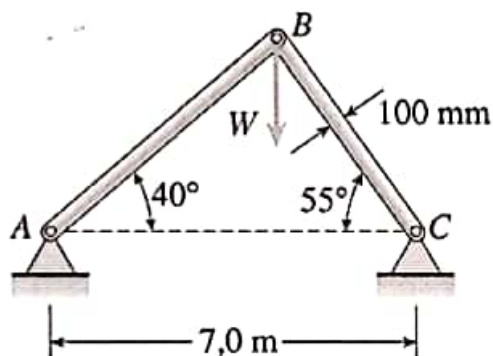
4ª Questão (2,5 pontos) – A estrutura com a forma de L é constituída de dois segmentos, cada um com comprimento  $L$  e rigidez à flexão  $EI$ . Se ela é submetida a uma carga uniformemente distribuída conforme a figura determine o deslocamento horizontal da extremidade C. (Figura 4)



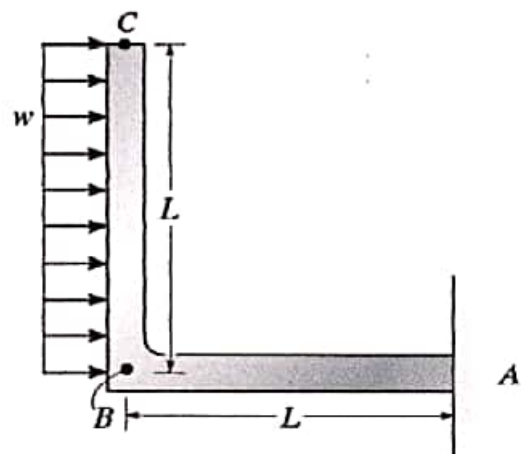
**Figura 1**



**Figura 2**

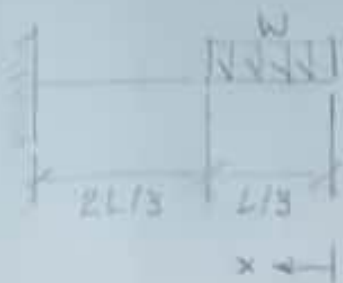


**Figura 3**



**Figura 4**

1)



$$w(x) = w \langle x - 0 \rangle^0 - w \langle x - L/3 \rangle^0$$

$$EI \frac{d^2 v}{dx^2} = M$$

$$V = -w \langle x - 0 \rangle^1 + w \langle x - L/3 \rangle^1$$

$$M = -\frac{w}{2} \langle x - 0 \rangle^2 + \frac{w}{2} \langle x - L/3 \rangle^2$$

$$EI \frac{dv}{dx} = -\frac{w}{2} x^2 + \frac{w}{2} \langle x - L/3 \rangle^2$$

$$EI \frac{dv}{dx} = -\frac{w}{6} x^3 + \frac{w}{6} \langle x - L/3 \rangle^3 + C_1$$

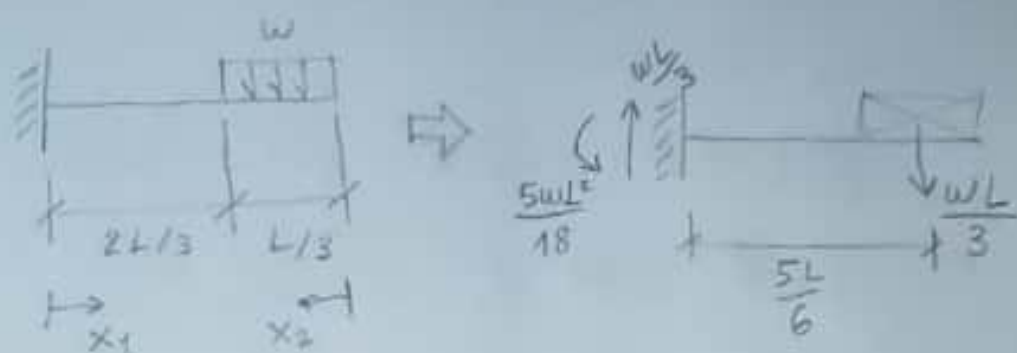
$$EI v = -\frac{w}{24} x^4 + \frac{w}{24} \langle x - L/3 \rangle^4 + C_1 x + C_2$$

$$\frac{dv}{dx} = 0 \rightarrow x = L \rightarrow C_1 = -\frac{4wL^3}{84} + \frac{w}{6} L^3$$

$$v = 0 \rightarrow x = L \rightarrow C_2 = -\frac{2wL^4}{243} + \frac{wL^4}{24} - C_1 L$$

$$\text{Rotação} = \left. \frac{dv}{dx} \right|_0 = C_1 = 0,117 \frac{wL^3}{EI}$$

$$\text{deslocamento} = v(0) = C_2 = -8,38 \cdot 10^{-2} \frac{wL^4}{EI}$$



$$M(x_1) = -\frac{5wL^2}{18} + \frac{wL}{3}x_1$$

$$M(x_2) = -\frac{wL^2}{2}$$

$$EI \frac{d^2v}{dx_1^2} = -\frac{5wL^2}{18} + \frac{wL}{3}x_1$$

$$EI \frac{d^2v}{dx_2^2} = -\frac{wL^2}{2}$$

$$EI \frac{dv}{dx_1} = -\frac{5wL^2}{18}x_1 + \frac{wL}{6}x_1^2 + C_1$$

$$EI \frac{dv}{dx_2} = -\frac{wL^2}{6}x_2 + C_1'$$

$$EI v = -\frac{5wL^2}{36}x_1^2 + \frac{wL}{18}x_1^3 + C_1x_1 + C_2$$

$$EI v = -\frac{wL^2}{24}x_2^2 + C_1'x_2 + C_2'$$

$$\frac{dv}{dx_1} = 0 \rightarrow x_1 = 0 \rightarrow C_1 = 0$$

$$v = 0 \rightarrow x_1 = 0 \rightarrow C_2 = 0$$

$$\frac{dv}{dx_1} = -\frac{dv}{dx_2} \rightarrow x_1 = \frac{2L}{3} \quad \wedge \quad x_2 = \frac{L}{3}$$

$$-\frac{5wL^2}{18} \cdot \frac{2L}{3} + \frac{wL}{6} \cdot \frac{4L^2}{9} = \frac{wL^3}{6 \cdot 27} - C_1' \rightarrow C_1' = \frac{19wL^3}{162}$$

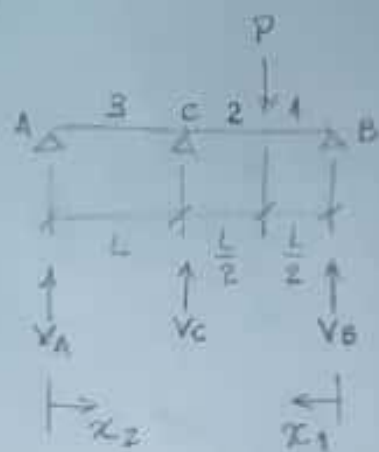
$$v(x_1) = v(x_2) \rightarrow x_1 = \frac{2L}{3} \quad \wedge \quad x_2 = \frac{L}{3}$$

$$-\frac{5wL^2}{36} \cdot \frac{4L^2}{9} + \frac{wL}{18} \cdot \frac{8L^3}{27} = -\frac{wL^4}{24 \cdot 81} + \frac{19wL^3}{162} \cdot \frac{L}{3} + C_2' \rightarrow C_2' = \frac{-163}{1944} wL^4$$

$$\text{rotaco} = \left. \frac{dv}{dx_2} \right|_0 = C_1' = 0,117 wL^3/EI$$

$$\text{deslocamento} = v(x_2=0) = C_2' = -8,38 \cdot 10^{-2} wL^4/EI$$

2)



Método da integração

$$P = V_A + V_B + V_C \quad (1)$$

$$\sum M_A = 2L \cdot V_B + L \cdot V_C - P \cdot \frac{3L}{2} = 0 \quad (2)$$

$$M(x_1)_1 = V_B \cdot x_1$$

$$M(x_1)_2 = V_B \cdot x_1 - P(x_1 - L/2)$$

$$M(x_2)_3 = V_A \cdot x_2$$

Trecho 1:

$$EI \frac{dv_1^2}{dx_1^2} = V_B \cdot x_1$$

$$EI \frac{dv_1}{dx_1} = V_B \cdot \frac{x_1^2}{2} + C_1$$

$$EI v_1 = V_B \cdot \frac{x_1^3}{6} + C_1 x_1 + C_2$$

Trecho 2:

$$EI \frac{dv_2^2}{dx_1^2} = V_B \cdot x_1 - P x_1 + \frac{PL}{2}$$

$$EI \frac{dv_2}{dx_1} = V_B \cdot \frac{x_1^2}{2} - \frac{P x_1^2}{2} + \frac{PL x_1}{2} + C_1'$$

$$EI v_2 = V_B \cdot \frac{x_1^3}{6} - \frac{P x_1^3}{6} + \frac{PL x_1^2}{4} + C_1' x_1 + C_2'$$

Trecho 3:

$$EI \frac{dv_3^2}{dx_2^2} = V_A \cdot x_2$$

$$EI \frac{dv_3}{dx_2} = V_A \cdot \frac{x_2^2}{2} + C_1''$$

$$EI v_3 = V_A \cdot \frac{x_2^3}{6} + C_1'' x_2 + C_2''$$

$$v_1 = 0 \rightarrow x_1 = 0 \rightarrow C_2 = 0$$

$$\left\{ \begin{aligned} \frac{dv_1}{dx_1} &= \frac{dv_2}{dx_1} \text{ at } x_1 = L/2 \rightarrow V_B \cdot \frac{L^2}{8} + C_1 = V_B \cdot \frac{L^2}{8} - \frac{PL^2}{8} + \frac{PL^2}{4} + C_1' \\ v_1 &= v_2 \text{ at } x_1 = L/2 \rightarrow V_B \cdot \frac{L^3}{48} + \frac{L}{2} C_1 = V_B \cdot \frac{L^3}{48} - \frac{PL^3}{48} + \frac{PL^3}{16} + \frac{L}{2} C_1' + C_2' \\ v_2 &= 0 \text{ at } x_1 = L \rightarrow 0 = V_B \frac{L^3}{6} - \frac{PL^3}{6} + \frac{PL^3}{4} + L C_1' + C_2' \end{aligned} \right.$$

$$C_1 = \frac{-8L^2 \cdot V_B + PL^2}{48}, \quad C_1' = \frac{-8L^2 \cdot V_B - 5PL^2}{48}, \quad C_2' = \frac{PL^3}{48}$$

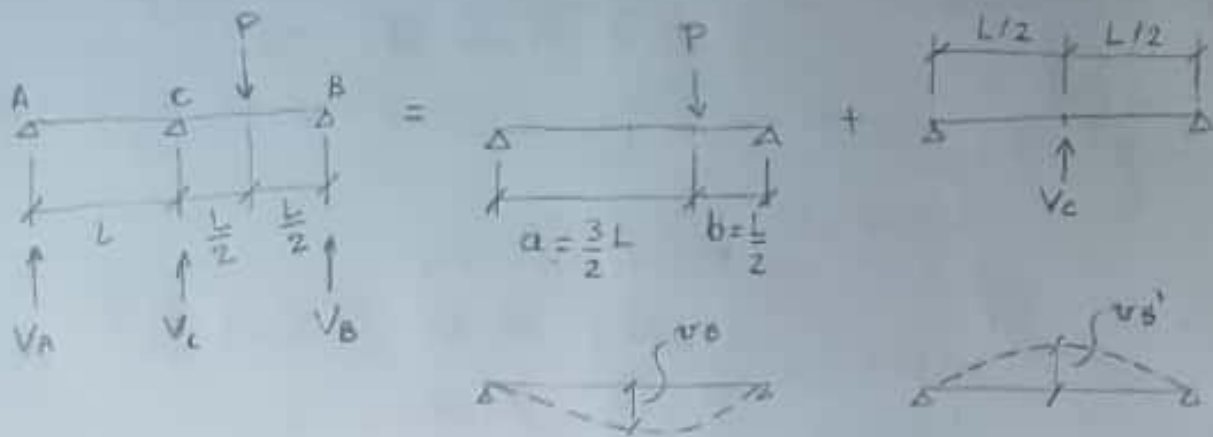
$$v_3 = 0 \rightarrow x_2 = 0 \rightarrow C_2'' = 0$$

$$v_3 = 0 \rightarrow x_2 = L \rightarrow 0 = V_A \cdot \frac{L^3}{6} + C_1'' \cdot L \rightarrow C_1'' = -V_A \frac{L^2}{6}$$

$$\frac{dv_2}{dx_1} = -\frac{dv_3}{dx_2} \text{ at } x_1 = L \wedge x_2 = L \rightarrow V_A = \frac{-16V_B + 5P}{16} \quad (3)$$

Resolvendo o sistema de equações dado por (1), (2) e (3), tem-se que:  $V_A = -\frac{3P}{32}$ ,  $V_B = \frac{13P}{32}$  e  $V_C = \frac{11P}{16}$

## Método da superposição



Equação de compatibilidade:  $v_B + v_B' = 0$

$$v_B = -\frac{P \cdot b \cdot x}{6EI(2L)} [(2L)^2 - b^2 - x^2] = -\frac{11PL^3}{96EI} \text{ para } x = L$$

$$v_B' = \frac{V_C(2L)^3}{48EI} = \frac{V_C L^3}{6EI}$$

De (1) tem-se que:  $\frac{V_C L^3}{6EI} - \frac{11PL^3}{96EI} = 0 \rightarrow V_C = \frac{11}{16}P$  (2)

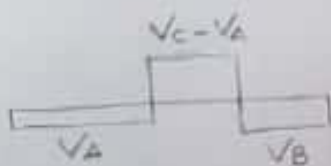
Equações de equilíbrio:

$$P = V_A + V_B + V_C \quad (3)$$

$$\sum M_A = 2LV_B + LV_C - 3PL/2 = 0 \quad (4)$$

Das equações (2), (3) e (4) tem-se:  $V_A = -\frac{3P}{32}$  e  $V_B = \frac{13P}{32}$

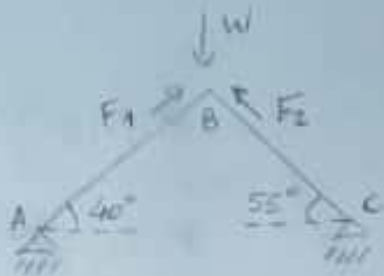
DEC



DMF



3)



$$\begin{cases} W = F_1 \cos 50^\circ + F_2 \cos 35^\circ \\ F_1 \sin 50^\circ = F_2 \sin 35^\circ \end{cases} \rightarrow \begin{aligned} F_1 &= 0,576 W \\ F_2 &= 0,769 W \end{aligned}$$

$$d_e = 100 \text{ mm}$$

$$d_i = 100 - 2 \cdot 6 = 88 \text{ mm}$$

$$I = \frac{\pi}{64} (d_e^4 - d_i^4) = 1963934,64 \text{ mm}^4$$

$$\begin{cases} \overline{AB} \cos 40^\circ + \overline{BC} \cos 55^\circ = 7000 \\ \overline{AB}^2 + \overline{BC}^2 - 2 \overline{AB} \cdot \overline{BC} \cos 85^\circ = 7000^2 \end{cases}$$

$$\begin{aligned} \overline{AB} &= 5756 \text{ mm} \\ \overline{BC} &= 4516,7 \text{ mm} \end{aligned}$$

Barra AB

Barra BC

$$P_{cr} = \frac{\pi^2 EI}{(KL)^2} = 116892,87 \text{ N}$$

$$P_{cr} = 189839,73 \text{ N}$$

K = 1 mas duas barras + bi-notulada

E = 200000 MPa

$$F_1 = 116892,87 = 0,576 W$$

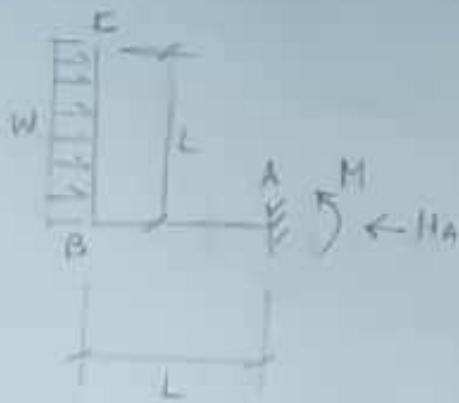
$$W = 202939,01 \text{ N}$$

$$F_2 = 189839,73 = 0,769 W$$

$$W = 246865,70 \text{ N}$$

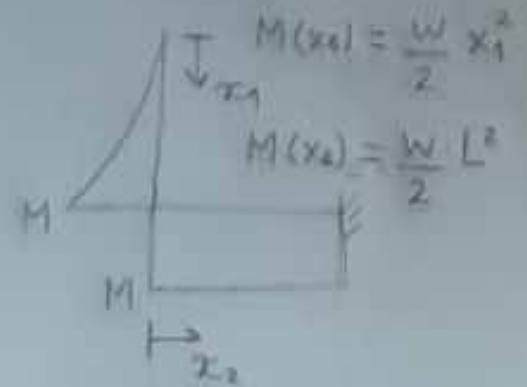
→ Por caso  
(Resposta)

4)



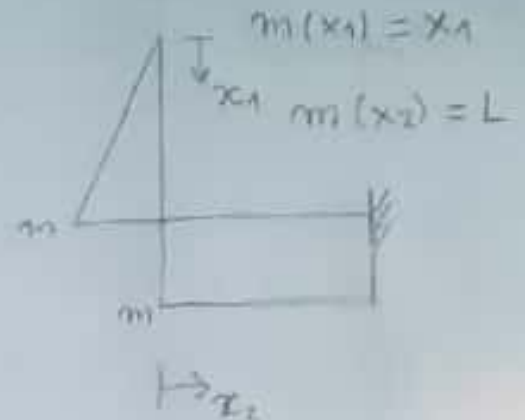
$$H_A = wL$$

$$M = \frac{wL^2}{2}$$



$$h_A = 1$$

$$m = L$$



Barra CB

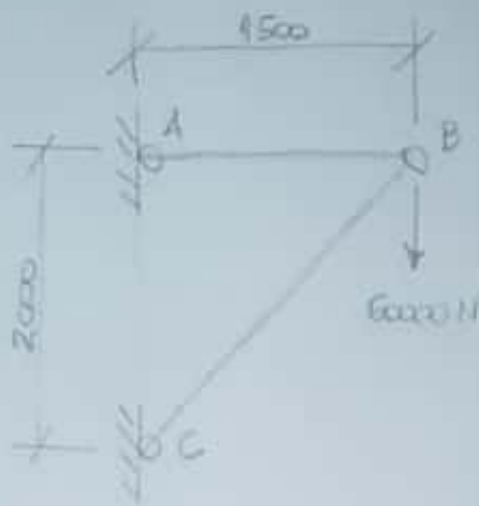
$$\Delta_{CB} = \int_0^L \frac{m(x_1)M(x_1)}{EI} dx_1 = \frac{1}{EI} \int_0^L \frac{w}{2} x_1^3 dx_1 = \frac{wL^4}{8EI}$$

Barra BA

$$\Delta_{BA} = \int_0^L \frac{m(x_2)M(x_2)}{EI} dx_2 = \frac{1}{EI} \int_0^L \frac{w}{2} L^2 \cdot L dx_2 = \frac{wL^4}{2EI}$$

$$\Delta_C = \Delta_{CB} + \Delta_{BA} = \frac{5wL^4}{8EI}$$





$$\begin{aligned} \bar{AB} \rightarrow d &= 30 \text{ mm} \\ E &= 210000 \text{ MPa} \\ \sigma_e &= 360 \text{ MPa} \end{aligned} \quad \left\{ \begin{aligned} A &= \frac{\pi d^2}{4} = 706,86 \text{ mm}^2 \end{aligned} \right.$$

$$\begin{aligned} \bar{BC} \rightarrow d_i &= 50 \text{ mm}, t = 15 \text{ mm} \\ E &= 73000 \text{ MPa} \\ \sigma_e &= 280 \text{ MPa} \end{aligned} \quad \left\{ \begin{aligned} A &= \frac{\pi}{4} [(d_i + 2t)^2 - d_i^2] \\ &= 3063,05 \text{ mm}^2 \\ I &= \frac{\pi [(d_i + 2t)^4 - d_i^4]}{64} \\ &= 1703823,14 \end{aligned} \right.$$

\* Tensão normal desenvolvida em  $\bar{AB}$  por conta da tração:

$$\sigma = \frac{45000}{706,86} = 63,66 \text{ MPa} < 360 \text{ MPa (OK)}$$

\* Tensão normal desenvolvida em  $\bar{BC}$  por conta da compressão:

$$\sigma = \frac{75000}{3063,05} = 24,48 \text{ MPa} < 280 \text{ MPa (OK)}$$

\* Carga crítica da barra comprimida

$$P_{cr} = \frac{\pi^2 EI}{(KL)^2} = \frac{\pi^2 \cdot 73000 \cdot 1703823,14}{2500^2} = 196411,6 \text{ N} > 75000 \text{ (OK)}$$

$K=1 \rightarrow$  barra bi-articulada

\* Fator de segurança

$$F.S. = \frac{\sigma}{\sigma_{adm}}$$

$$\rightarrow \text{Barra } \bar{AB}: 63,66 / 360 = 0,18$$

$\rightarrow$  Barra  $\bar{BC}$ :

- compressão:  $24,48 / 280 = 0,08$

- flambagem:  $75000 / 196411,6 = 0,38$

Fator de segurança é o maior valor

$$F.S. = 0,38$$

(flambagem)