Amme 3500 : System Dynamics & Control

Design via Frequency Response

Dr Stefan B Williams

Frequency Response

- In week 7 we looked at modifying the transient and steady state response of a system using root locus design techniques
 - Gain adjustment (speed, steady state error)
 - Lag (PI) compensation (steady-state error)
 - Lead (PD) compensation (speed, stability)
- We will now examine methods for designing for a particular specification by examining the frequency response of a system
- We still rely on approximating CL behaviour as 2nd Order

Course Outline

Week	Date	Content	Assignment Notes
1	1 Mar	Introduction	
2	8 Mar	Frequency Domain Modelling	
3	15 Mar	Transient Performance and the s-plane	
4	22 Mar	Block Diagrams	Assign 1 Due
5	29 Mar	Feedback System Characteristics	
6	5 Apr	Root Locus	Assign 2 Due
7	12 Apr	Root Locus 2	
8	19 Apr	Bode Plots	No Tutorials
	26 Apr	BREAK	
9	3 May	Bode Plots 2	
10	10 May	State Space Modeling	Assign 3 Due
11	17 May	State Space Design Techniques	
12	24 May	Advanced Control Topics	
13	31 May	Review	Assign 4 Due
14		Spare	

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Time vs. Freq. Domain Analysis

- Control system performance generally judged by time domain response to certain test signals (step, etc.)
 - Simple for < 3 OL poles or ~2nd order CL systems.
 - No unified methods for higher-order systems.
- Freq response easy for higher order systems
 - Qualitatively related to time domain behaviour
 - More natural for studying sensitivity and noise susceptibility

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Frequency Response Specifications

- Crossover frequency: $|G(j\omega_c)|=1$
- · Gain Margin:

$$K$$
 (dB) s.t. $|KG(j\omega)| = 1$ at ω where $\angle G(j\omega) = 180^{\circ}$

· Phase Margin:

$$PM = 180^{\circ} - \angle G(j\omega_c)$$

· Bandwidth (CL specification)

$$|G(j\omega_{RW})| = -3 \,\mathrm{dB}$$

$$\omega_c \le \omega_{BW} \le 2\omega_c$$

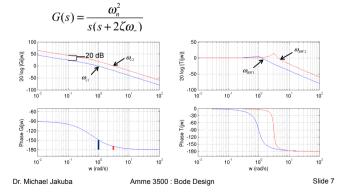
 Less intuitive than RL, but easier to draw for high order systems.

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Gain Adjustment and the Frequency Response



Transient Response via Gain

- The root locus demonstrated that we can often design controllers for a system via gain adjustment to meet a particular transient response
- We can effect a similar approach using the frequency response by examining the relationship between phase margin and damping

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Design using Phase Margin

- Recall that the Phase Margin is closely related to the damping ratio of the system
- For a unity feedback system with openloop function

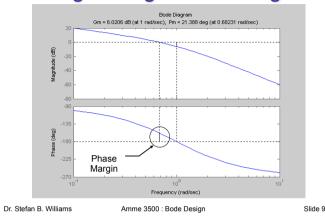
$$G(s) = \frac{\omega_n^2}{s(s + 2\zeta\omega_n)}$$

 We found that the relationship between PM and damping ratio is given by

$$PM = \tan^{-1} \frac{2\zeta}{\sqrt{-2\zeta^2 + \sqrt{1 + 4\zeta^4}}} \qquad \zeta \approx \frac{PM}{100}, \ PM < 70^{\circ}$$
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Design using Phase Margin



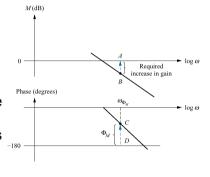
Design using Phase Margin

- The design procedure therefore consists of
 - Draw the Bode Magnitude and phase plots
 - Determine the required phase margin from the percent overshoot
 - Find the frequency on the Bode phase diagram that yields the desired phase margin
 - Change the gain to force the magnitude curve to go through 0dB

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Design using Phase Margin

- Given a desired overshoot, we can convert this to a required damping ratio and hence PM
- Examining the Bode plot we can find the frequency that gives the desired PM



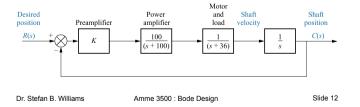
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Phase Margin Example

 For the following position control system shown here, find the preamplifier gain K to yield a 9.5% overshoot in the transient response for a step input



Phase Margin Example

- · Draw the Bode plot
- For 9.5% overshoot, z=0.6 and PM must be 59.2°
- Locate frequency with the required phase at 14.8 rad/s
- The magnitude must be raised by 55.3dB to yield the cross over point at this frequency
- This yields a K = 583.9

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Designing Compensation

- As we saw previously, not all specifications can be met via simple gain adjustment
- We examined a number of compensators that can bring the root locus to a desired design point
- A parallel design process exists in the frequency domain

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Designing Compensation

- In particular, we will look at the frequency characteristics for the
 - PD Controller
 - Lead Controller
 - PI Controller
 - Lag Controller
- Understanding the frequency characteristics of these controllers allows us to select the appropriate version for a given design

PD Controller

 The ideal derivative compensator adds a pure differentiator, or zero, to the forward path of the control system

$$U(s) = K(s + z_c)$$

- The root locus showed that this will tend to stabilize the system by drawing the roots towards the zero location
- We saw that the pole and zero locations give rise to the break points in the Bode plot

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PD Controller

- The Bode plot for a PD controller looks like this
- The stabilizing effect is seen by the increase in phase at frequencies above the break frequency
- However, the magnitude grows with increasing frequency and will tend to amplify high frequency noise

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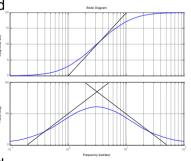
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Lead Compensation

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- The Bode plot for a Lead compensator looks like this
- The frequency of the phase increase can be designed to meet a particular phase margin requirement
- The high frequency magnitude is now limited



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Lead Compensation

Introducing a higher order pole yields the lead compensator

$$U(s) = \frac{K(s+z_c)}{(s+p_c)} z_c < p_c$$

· This is often rewritten as

$$U(s) = \frac{T_{s+1}}{\alpha T_{s+1}} = \frac{1}{\alpha} \frac{s + \frac{1}{T}}{s + \frac{1}{T}}, \alpha < 1$$

where $1/\alpha$ is the ratio between pole-zero break points

 The name Lead Compensation reflects the fact that this compensator imparts a phase lead

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Lead Compensation

- The lead compensator can be used to change the damping ratio of the system by manipulating the Phase Margin
- · The phase contribution consists of

$$\phi = \tan^{-1} T\omega - \tan^{-1} \alpha T\omega$$

· The peak occurs at

$$\omega_{\text{max}} = \frac{1}{T\sqrt{\alpha}}$$

with a phase shift and magnitude of

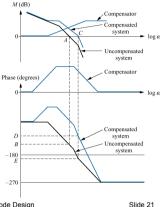
$$\phi_{\text{max}} = \sin^{-1} \frac{1-\alpha}{1+\alpha}, |U(j\omega_{\text{max}})| = \frac{1}{\sqrt{\alpha}}$$

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Lead Compensation

 This compensator allows the designer to raise the phase of the system in the vicinity of the crossover frequency



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Lead Compensation Design

5. Determine the new crossover frequency $\left|U(j\omega_{\max})\right| = \frac{1}{\sqrt{\alpha}}$

6. Determine the value of T such that ω_{max} lies at the new crossover frequency $\omega_{\text{max}} = \frac{1}{2\pi\sqrt{-1}}$

- 7. Draw the compensated frequency response and check the resulting phase margin.
- Check that the bandwidth requirements have been met.
- 9. Simulate to be sure that the system meets the specifications (recall that the design criteria are based on a 2nd order system).

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Lead Compensation Design

- The design procedure consists of the following steps:
 - 1. Find the open-loop gain K to satisfy steady-state error or bandwidth requirements
 - 2. Evaluate phase margin of the uncompensated system using the value of gain chosen above
 - 3. Find the required phase lead to meet the damping requirements
 - 4. Determine the value of α to yield the required increase in phase

$$\phi_{\text{max}} = \sin^{-1} \frac{1 - \alpha}{1 + \alpha}$$

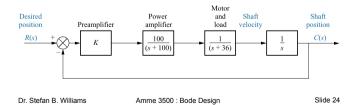
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Lead Compensation Example

 Returning to the previous example, we will now design a lead compensator to yield a 20% overshoot and K_v=40, with a peak time of 0.1s



Lead Compensation Example

- From the specifications, we can determine the following requirements
 - For a 20% overshoot we find ζ=0.456 and hence a Phase Margin of 48.1°
 - For peak time of 0.1s with the given ζ , we can find the require closed loop bandwidth to be 46.6rad/s

$$\omega_{BW} = \frac{\pi}{T_n \sqrt{1 - \zeta^2}} \sqrt{(1 - 2\zeta^2) + \sqrt{4\zeta^4 - 4\zeta^2 + 2}}$$

– To meet the steady state error specification $K_{\nu} = 40 = \lim_{s \to 0} sG(s) = \frac{K100}{3600}$

$$K_v = 40 = \lim_{s \to 0} sG(s) = \frac{K100}{3600}$$

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Lead Compensation Example

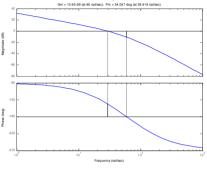
- We require a phase margin of 48.1°
- The lead compensator will also increase the phase margin frequency so we add a correction factor to compensate for the lower uncompensated system's phase angle
- The total phase contribution required is therefore

$$48.1^{\circ} - (34^{\circ} - 10^{\circ}) = 24.1^{\circ}$$

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Lead Compensation Example

- From the Bode plot, we evaluate the PM to be 34° for a gain of 1440
- We can't simply increase the gain without violating the other design constraints
- · We use a Lead Compensator to raise the PM



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Lead Compensation

· Based on the phase requirement we find

$$\phi_{\text{max}} = \sin^{-1} \frac{1 - \dot{\alpha}}{1 + \alpha}$$

$$\alpha = 0.42$$
 for $\phi_{\text{max}} = 24.1^{\circ}$

· The resulting magnitude is

$$U(j\omega) = \frac{1}{\sqrt{\alpha}} = 3.77 dB$$

- $|U(j\omega)| = \frac{1}{\sqrt{\alpha}} = 3.77 dB$ Examining the Bode magnitude, we find that the frequency at which the magnitude is -3.77dB is ω_{max} =39rad/s
- The break frequencies can be found at 25.3 and 60.2

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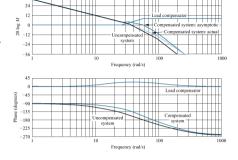
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Lead Compensation Example

The compensator is

$$U(s) = 2.38 \frac{s + 25.3}{s + 60.2}$$

· The resulting system Bode plot shows the impact of the phase lead



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· We need to verify

· The simulation

assumption

the performance of

the resulting design

appears to validate

our second order

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Lead Compensation Example

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PI Controller

 We saw that the integral compensator takes on the form

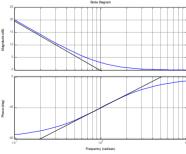
$$U(s) = K \frac{s + z_c}{s}$$

- This results in infinite gain at low frequencies which reduces steady-state error
- A decrease in phase at frequencies lower than the break will also occur

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PI Controller

- · The Bode plot for a PI controller looks like this
- The break frequency is usually located at a frequency substantially lower than the crossover frequency to minimize the effect on the phase margin



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Lag Compensation

Lag compensation approximates PI control

$$U(s) = \frac{K(s+z_c)}{(s+p_c)} z_c > p_c$$

This is often rewritten as

$$U(s) = \frac{T_{s+1}}{\alpha T_{s+1}}, \alpha > 1$$

where α is the ratio between zero-pole break points

• The name Lag Compensation reflects the fact that this compensator imparts a phase lag

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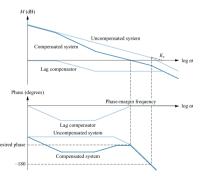
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Lag Compensation

· In this case we are trying to raise the gain at low frequencies without affecting the stability of the system



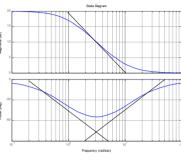
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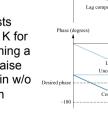
Lag Compensation

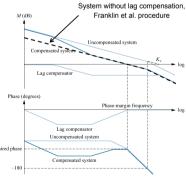
- The Bode plot for a Lag compensator looks like this
- · This compensator effectively raises the magnitude for low frequencies
- The effect of the phase lag can be minimized by careful selection of the centre frequency



Lag Compensation

- · Nise suggests setting the gain K for s.s. error, then designing a lag network to attain desired PM.
- · Franklin suggests setting the gain K for PM, then designing a lag network to raise the low freq. gain w/o affecting system stability.





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Lag Compensation Design

- The design procedure (Franklin et al.) consists of the following steps:
 - 1. Find the open-loop gain K to satisfy the phase margin specification without compensation
 - 2. Draw the Bode plot and evaluate low frequency gain
 - 3. Determine a to meet the low-frequency gain error requirement
 - 4. Choose the corner frequency $\omega=1/T$ to be one octave to one decade below the new crossover frequency
 - 5. Evaluate the second corner frequency $\omega = 1/\alpha T$
 - 6. Simulate to evaluate the design and iterate as required

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Lag Compensation Example

- In the first example, we found the gain K=583.9 would vield our desired 9.5% overshoot with a PM of 59.2° at 14.8rad/s
- · For this system we find that

$$K_{v} = \lim_{s \to 0} sG(s)$$
$$= 583.9 \frac{100}{3600} = 16.22$$

- We therefore require a K, of 162.2 to meet our specification
- We need to raise the low frequency magnitude by a factor of 10 (or 20dB) without affecting the PM

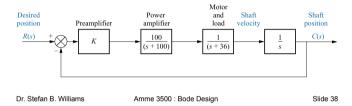
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Lag Compensation Example

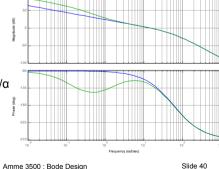
 Returning again to the previous example, we will now design a lag compensator to yield a ten fold improvement in steady-state error over the gaincompensated system while keeping the overshoot at 9.5%



Lag Compensator Example

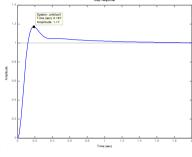
- · First we draw the Bode plot with K=583.9
- · Set the zero at one decade, 1.48rad/s. lower than the PM frequency
- The pole will be at 1/α relative to this so $U(s) = \frac{s + 1.483}{s}$ s + 0.1483

 $=10\frac{0.674s+1}{}$ Dr. Stefan B. Williams



Lag Compensator Example

- The resulting system has a low frequency gain K_v of 162.2 as per the requirement
- The overshoot is slightly higher than the desired
- Iteration of the zero and pole locations will yield a lower overshoot if required



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Conclusions

- We have looked at techniques for designing controllers using the frequency techniques
- There is once again a trade-off in the requirements of the system
- By selecting appropriate pole and zero locations we can influence the system properties to meet particular design requirements

Lag-Lead Compensation

- As with the Root Locus designs we considered previously, we often require both lead and lag components to effect a particular design
- This provides simultaneous improvement in transient and steady-state responses
- In this case we are trading off three primary design parameters
 - Crossover frequency ω_c which determines bandwidth, rise time and settling time
 - Phase margin which determines the damping coefficient and hence overshoot
 - Low frequency gain which determines steady state error characteristics

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Further Reading

- Nise
 - Sections 11.1-11.5
- Franklin & Powell
 - Section 6.7

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