Mathemathical Modelling

isakhammer

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Contents

1	Lecture 1	2					
	1.1 Practical Information	2					
	1.2 Dimensional Analysis	2					
	1.3 Fundamental Units	2					
	1.4 Trinity of the first atomic blast	3					
	1.5 Steady-state single phase flow in a uniform straight pipeline	4					
2	Lecture 2	5					
	2.1 Practical Information	5					
	2.2 Recall	5					
	2.3 Scaling	5					
	2.4 Buckinghams Pi-Theorem	5					
	2.5 Scaling	6					
3	Lecture 3	8					
	3.1 Recall	8					
	3.2 Sinking Ball	8					
	3.2.1 Scaling	10					
	3.3 Let Analyze The equation	12					
	3.3.1 Case B	12					
	3.3.2 Case C: High V and high friction	12					
	3.4 Turbulence	13					
4	Lecture 31/08	14					
	4.1 Turbulence	14					
	4.2 Regular Perturbation Theory	16					
	4.3 The Projectile Problem	16					
5	Lecture 02/09						
	5.1 Modelling how the kidney disposes salt and water	21					
	5.1.1 Scaling the model	24					
6	References 28						

1 Lecture 1

1.1 Practical Information

You need to know

- Separable 1. order equations.
- Linear 1. order equations.
- 2. order linear equations with constant coefficients.

1.2 Dimensional Analysis

Basic facts

- Any physical relation has to make sense dimensionally.
- Any physical relation must be valid for any choice of fundamental units.

Remark.

Make sure remark looks better

- Forbidden 3m + 2kg = ?
- m = f(x, t) is legal
- e^{-t} and $s = 5t^2$, is nonsense
- Dimension is length, mass, energy, etc.
- Unit is meter, feet, year, etc

numerical value

Given a variable R, we write R =

$$(R)$$
 (R)

If we have a physical relation that is dimensionall correct that

$$f(R_1, R_2, ..., R_n) = 0 \rightarrow f(v(R_1), v(R_2), ..., v(R_n)) = 0$$

1.3 Fundamental Units

Given units F_1, F_2, \ldots, F_m for fundamental if

$$F_1^{\alpha_1}, F_2^{\alpha_2}, \dots, F_m^{\alpha m} = 0 \quad \rightarrow \quad \alpha_1 = \alpha_2 = \dots = 0$$

This units are then independent. **Example.** The units kg, m, s are independent.

Example. In a right angle triangle with angle α and hypothenus c. We know the area A is uniquely determined by α and c

$$A = f(c, \alpha)$$

 α is dimensialless since $\alpha = \frac{s}{r}$. Since A scales as the square of the length, then is

$$f\left(ac,\alpha\right) = a^{2} f\left(c,\alpha\right)$$

$$c = 1 \to f\left(a,\alpha\right) = a^{2} f\left(1,\alpha\right) = a^{2} h\left(\alpha\right)$$

Which then ends up with the relation

$$A = a^2 h\left(\alpha\right)$$

Make corollary environmet

Lets derive $A=a^2h\left(\alpha\right)$ somwhat differently. We know there is a relation $f\left(A,c,\alpha\right)=0$. We want to introduce new variables.

$$\Pi_1 = \frac{A}{c^2}, \quad c = c_1, \quad \alpha = \alpha_1$$

which means $f(c^2\Pi_1, c, \alpha) = 0$ and $h(\Pi_1, \alpha, c) = 0$. h must be dimensially consistent $\to h$ must be independent of c.

$$h\left(\Pi_{1},\alpha\right) = 0 \leftrightarrow \Pi_{1} = k\left(\alpha\right)$$
$$\rightarrow \frac{A}{c^{2}} = k\left(\alpha\right) \quad \leftrightarrow \quad A = c^{2}k\left(\alpha\right).$$

1.4 Trinity of the first atomic blast

We assume there is a relation

$$f(E, \rho, r, t) = 0$$

- Energy: $E, [E] = kgm^2s^{-2}$
- Mass density of air: ρ , $[\rho] = kg^{-3}$
- Radius: r, [r] = m
- Time: t, [t] = s

We choose 3 independent variables, say r, t, ρ . Also we call r, t, ρ **core variables**. Let is define a dimensionalless number Π_1 such that

$$[\Pi_1] = 0$$

The relation is now given by $h\left(\Pi,t,r,\rho\right)=0$, where h is independent of t, r and ρ . Which in fact is $h\left(\Pi\right)=0$, where $\Pi_{1}=c$ s.t. [c]=1.

Given by the definition is

$$\frac{Et^2}{\rho r^5} = c \quad \to \quad E = \frac{c\rho r^5}{t^2}$$

Using $\rho = 12kgm^{-3}$, r = 110m, $t = 6 \cdot 10^{-3}$ do we end up with the relation

$$E = c \cdot 7.5 \cdot 10^{13} J$$

1.5 Steady-state single phase flow in a uniform straight pipeline

Figure of a pipe

Pipe with flow u, length L and pressure drop Δp Then there is a relation between

- L: length, [L] = m
- D: diameter [D] = m
- u: flow rate $[u] = ms^{-1}$
- Δp : Pressure drop, $\left[\Delta kgm^{-1}s^{-2}\right]$
- μ : (Shear) viscousity $[\mu] = kgm^{-1}s^{-1}$
- ρ : mass density: $[\rho] = kgm^{-3}$
- E: Wall roughness: [E] = m

We have to choose 3 core variables and they are not unique. Since we have 3 independent units ρ , u, D are independent such that it can be a core variable:

$$\Pi_1 = \frac{L}{D}$$
 , $\Pi_2 = \frac{\Delta p}{\rho u^2}$, $\Pi_3 = \frac{\rho}{\mu}$, $\Pi_4 = \frac{E}{D}$

Then the relation is

$$f\left(\Pi_{1},\Pi_{2},\Pi_{3},\Pi^{4},\rho,D,u\right)=0 \quad \Pi_{2}=h\left(\Pi_{1},\Pi_{3},\Pi_{4}\right) \leftrightarrow \frac{\Delta p}{\rho u^{2}}=h\left(\Pi_{1},\Pi_{3},\Pi_{4}\right)$$

$$\rightarrow \frac{\Delta p}{u^{2}\rho}=\Pi_{1}k\left(\Pi_{3},\Pi_{4}\right)$$

$$\Delta p=u^{2}\rho\frac{L}{D}k\left(\frac{\rho Du}{\mu},\frac{E}{D}\right)$$
 measure
$$\frac{\rho D\mu}{\mu} \quad , \quad k=\frac{\Delta pD}{u^{2}\rho}$$

2 Lecture 2

2.1 Practical Information

Ask for zoom meeting. ola.mahlen@ntnu.no, wednesday 13-14.

2.2 Recall

Last time did we consider steady-state single phase in a flow in a pipe.

• Assuming $f(L, \Delta p, u, \mu, D, E, \rho) = 0$ we arrive with this formula

$$\frac{\Delta pD}{u^2 \rho L} = k \begin{pmatrix} \text{Reynhold number} \\ \hline \frac{\rho uD}{\mu} \\ \\ \text{Relative wall roughness} \end{pmatrix}$$

• Dimensionless numbers are often called **dimensionless groups**. Such numbers are independent of choice of fundamental units. They have real physical meaning. **Reynholds number** R_e essentially define what type of flow. Usually $R_e < 2000$ is it laminar flow and $R_e > 4000$ turbulent flow.

2.3 Scaling

Let a pipe have diameter D and flow rate u such that $t_v = \frac{D}{u}$. Then can we describe

$$t_{\alpha} = \frac{D^2}{\frac{\mu}{e}}$$

where μ is the kinematic viscosity. Then is R_e defined such that

$$R_e = \frac{t_\alpha}{t_v}$$

Assume we have the relation

$$R_1 = f\left(R_2, \dots, R_m\right)$$

Such that it exist an

$$\Pi_1 = g(\Pi_2, \Pi_2, \dots, \Pi_{m-k}).$$

2.4 Buckinghams Pi-Theorem

Assume we have a dimensionally valid relation $f(R_1, \ldots, R_m) = 0$ and a set of fundemental units F_1, F_2, \ldots, F_n such that

$$[R_j] = F_1^{a_{j1}} F_2^{a_{j2}} \dots F_n^{a_{jn}} \quad j = 1, 2, \dots, m$$

This then defines the dimension matrix A given by

Table 1:								
	F_1	F_2		F_n				
R_1 R_2	a_{11}	a_{11}		a_{1n}				
R_2	a_{21}	a_{21}		a_{2n}				
:		٠						
R_n	a_{m1}			a_{mn}				

Fix better table environment

Let rank(A) = dim(row(A)) = k. This translates to that we have k dimensionally independent variables. Choosing k linearly independent row vectors, corresponds to choosing core variables. Let this basis be $\mathbf{a}_{i1}, \mathbf{a}_{i2}, \ldots, \mathbf{a}_{ik}$. Let the rest of the row vectors be

$$\mathbf{a}_{j_1}, \mathbf{a}_{j_2}, \dots, \mathbf{a}_{j_{m-k}}$$

Then is $\mathbf{a}_{j_r} = \sum_{s=1}^k C_{j_r,s} \mathbf{a}_{\mathbf{i}_s}$ where $r=1,\ldots,m-k$. We end up with the equation

$$\Pi_r = \frac{R_{j_r}}{R_{i_1}^{r_{j_r,1}} R_{i_2}^{a_{j_r,2}} \dots R_{j_k}^{a_{j_r,k}}}$$

Are dimensionally numbers.

Our relation becomes

$$g(\Pi_1, \dots, \Pi_{m-k}) = 0, \quad \begin{cases} i_1, i_2, \dots, i_k \\ j_1, \dots, j_{m-k} \end{cases}$$

Example. Swinging pendulum

Assume there is a relation

$$f(w, \alpha_0, L, M, g,) = 0$$

where w is the frequency, g gravitational acceleration, M mass, α_0 the swinging angle. We can set L, M, g as core variables such that

$$\begin{bmatrix} \frac{L}{g} \end{bmatrix} = s^2 \quad \rightarrow \quad \begin{bmatrix} \frac{L}{g} w^2 \end{bmatrix} = 1$$

$$f\left(w, \alpha_0, L, M, g\right) = 0 \implies \quad g\left(\alpha_0, \frac{Lw^2}{g}\right) = 0$$

2.5 Scaling

We have a problem at hand, usually differential equations. Then we tru to find representative scales for the various variables, and then write the equation on so-called fimensionless form. This has several advantages

- Our dimensionless variables are of order 1 .
- We get rid of a lot of physical constants.
- It makes us able to see what terms are "small" in the equation. The idea is to introduce dimensionless variables by introducing appropriate scales. If we have a stick of length L, we choose L as length scale i.e

 $x^* = Lx$ Where x is dimensionless

Example. Heat flow in a rod with length L. Let $u^*(x^*, t^*)$ be the temperatur with the boundary conditions

$$u^*(0,t^*) = 0$$
 $u^*(L,t^*) = 0$

If we let the model be

$$\frac{\partial u^*}{\partial t^*} = D \cdot \frac{\partial^2 u^*}{\partial x^{*2}}, \quad u^* \left(0, t^* \right) = 0 \quad u^* \left(L, t^* \right) = 0$$
$$u^* \left(x^*, 0 \right) = u_0 \sin \left(\pi \frac{x^*}{L} \right)$$

We fund the tune scale T by scales **balancing the equation**. Let $x^* = Lx$, and $t^* = Tt$, where T is to be determined $u^* = u_0u$. If we find u(x,t), then the physical temperature is given by

$$u^*(x^*, t^*) = u_0 u\left(\frac{x^*}{L}, \frac{t^*}{T}\right)$$

We have u(0,t) = u(1,t) = 0

$$\begin{split} \frac{\partial u^*}{\partial t^*} &= D \frac{\partial^2 u^*}{\partial x^{*2}} \quad \Longrightarrow \quad \frac{u_0}{T} \frac{\partial u}{\partial t} = \frac{u_0}{L^2} D \frac{\partial^2}{\partial x^2} \\ & \leftrightarrow \frac{\partial u}{\partial t} = \left(\frac{TD}{L^2}\right) \frac{\partial^2 u}{\partial x^2} \quad \text{Balancing the equation} \\ \frac{TD}{L^2} &= 1 \quad \Longrightarrow \quad T = \frac{L^2}{D} \\ u^*\left(x^*,0\right) &= u_0 \sin\left(\pi \frac{x^*}{L}\right) \\ u\left(x,0\right) &= \sin\left(\pi x\right) \end{split}$$

which fulfills the condition

$$\frac{\partial u}{\partial t} = \frac{\partial^2 u}{\partial x^2}$$
, $u(0,t) = u(1,t) = 0$

3 Lecture 3

3.1 Recall

$$\frac{\partial u^*}{\partial t^*} = D \frac{\partial^2 u^*}{\partial x^{*2}}$$
$$0 \le x^* \le L$$
$$x^* = Lx$$
$$t^* = Tt$$
$$u^* = u_0$$

We can also recall

$$u^*\left(x^*,t^*\right) = u_0 u\left(\frac{x^*}{L},\frac{t^*}{T}\right)$$

$$\frac{u_0}{T}\frac{\partial u}{\partial t} = D\frac{u_0}{L^2} \implies \frac{\partial u}{\partial t} = \frac{TD}{L^2}\frac{\partial^2 u}{\partial x^2}$$
 Require
$$\frac{TD}{L^2} = 1 \implies T = \frac{L^2}{D}$$

This can be generelized to

$$\frac{\partial u}{\partial t} = \frac{\partial^2 u}{\partial x^2}, \quad 0 \le x \le 1$$

3.2 Sinking Ball

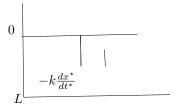


Figure 1: sinkingball

Let

- ρ_b e mass density of ball
- ρ_f mass density of fluid
- \bullet V Volume of ball

Then is the equation

$$\rho_b V g - \rho_f V g = V g \rho_b \left(1 - \frac{\rho_f}{\rho_b} \right)$$
$$= m \hat{g} \implies \hat{g} = g \left(1 - \frac{\rho_f}{\rho_b} \right)$$

And we then end up with the newtions law

$$m\frac{dx^{*2}}{dt^{*2}} = m\hat{g} - k\frac{dx^{*}}{dt}, \quad \text{Friction coefficient} \quad k$$

where

$$x^*(0) = 0, \quad \frac{dx^*}{dt^*}(0) = V$$

The cases can be described as follows

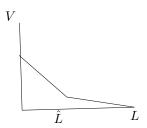


Figure 2: highV

- 1. High friction, not so high V. Ball will sink at constant speed most of the time.
- 2. Friction is low, and C not "too high". ("Free fall with V=0")
- 3. High V, and high friction $m \frac{d^2 x^*}{dt^{*2}} = m \hat{g} k \frac{dx^*}{dt^*}$

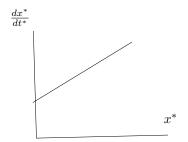


Figure 3: frefall

For this problem there is 3 characteristic speeds

- 1. V: initial velocity
- 2. v_0 : equilibrium speed in case A $v_0 = \frac{m\hat{g}}{k}$
- 3. v_f : free fall $v_f = \sqrt{2\hat{g}L}$

Let us put

$$\frac{d^2x^*}{dt^{*2}} = 0 \implies k\frac{dx^*}{dt} = \hat{g}m$$
$$\implies \frac{dx^*}{dt^*} = \hat{g}\frac{m}{k} = v_0$$

and put

$$x^* (0) = \frac{dx^*}{dt^*} (0) = 0$$
$$k = 0$$

3.2.1 Scaling

- 1. Case A: The ball sinks at constant speed "most" of the time.
 - (a) Length scale $L: x^* = Lx$. Since the ball falls with speed most of the time, a timescale would be $T = \frac{L}{v_0}$. v is not much larger than v_0

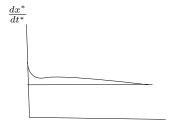


Figure 4: sinking

 \implies it is not so that $v \gg v_0$

$$\begin{split} m\frac{L}{T^2}x^{''} &= m\hat{g} - k\frac{L}{T}x^{'} & \text{Divide by } L \\ \Longrightarrow m\frac{1}{kT}x^{''} &= \frac{Tm\hat{g}}{KL} - x^{'} \\ \frac{m}{k\frac{L}{v_0}}x^{''} &= \frac{\frac{k}{v_0}m\hat{g}}{kL} - x^{'} \\ \Longrightarrow \frac{mv_0}{Lk}x^{''} &= \frac{Lm\hat{g}}{KLv_0} - x^{'} \end{split}$$

We can then derive

$$\frac{m\frac{m\hat{g}}{k}}{Lk}x'' = 1 - x'$$

$$\implies \frac{m^2\hat{g}}{Lk^2}x'' = 1 - x'$$

$$\implies \frac{m^2\hat{g}^2}{\hat{g}Lk^2}x'' = 1 - x'$$

$$\epsilon x'' = 1 - x' \quad \text{Where} \quad \epsilon = 2\left(\frac{v_0}{v_f}\right)^2$$

The condition are $x\left(0\right)=0,\,\frac{L}{T}x^{'}\left(0\right)=V$ which can be rewritten to

$$x^{'}(0) = \frac{TV}{L} \frac{\frac{L}{v_0 V}}{L} = \frac{V}{v_0} = \mu$$

3.3 Let Analyze The equation

In case A is the

$$\epsilon \ddot{x} = 1 - \dot{x}$$

An approximation we can do is to put $\epsilon = 0$ such that

$$0 = 1 - \dot{x}$$
 $x(0) = 0$, $\dot{x}(0) = \mu$ $\dot{x} = 0$

unless $\mu = 1$, we cant find a solution.

3.3.1 Case B

Small friction, V is not too high. Let the lengthscale be L.

$$\frac{d^2}{dt^{*2}}x^{*2} = \hat{g}, \quad x^*(0) = \frac{dx^*}{dt^*}(0) = 0$$
$$x^*(t^*) = \frac{1}{2}\hat{g}(t^*)^2$$

Hit the bottom with speed \mathcal{V}_f . We can choose time scale T such that

$$T = \frac{L}{v_f}$$

So gain

$$\frac{mL}{T^2}\ddot{x} = m\hat{g} - \frac{kL}{T}\dot{x}$$

What you can observe is that gravity dominates so we modify the equation to be

$$\begin{split} \frac{L}{\hat{g}T^2}\ddot{x} &= 1 - \frac{kL}{gmT}\dot{x} \\ \Longrightarrow & 2\ddot{x} = 1 - \left(\frac{v_F}{v_0}\right), \quad \frac{K}{T}\dot{x}\left(0\right) = 0 \\ & 2\ddot{x} = 1 - \epsilon\dot{x} \quad \dot{x}\left(0\right) = \frac{V}{v_f} = \mu \end{split}$$

3.3.2 Case C: High V and high friction

Let us consider

$$m\frac{d^2x^*}{dt^{*2}} = -kV \quad \frac{dx^*}{dt^*} = V - \frac{kV}{m}t^* = 0$$

Where we choose the scales $t^* = \frac{m}{k} = T$, $L = \frac{Vm}{k}$, where TV = L.

$$\implies \ddot{x} = \epsilon - \dot{x}, \quad x(0) = 1, \quad \dot{x} = 1, \quad \epsilon = \frac{v_0}{V}$$

Example. Let

$$a\frac{d^2x^*}{dt^{*2}} + b\frac{dx^*}{dt^*} + cx^* = 0$$
$$x^*(0) = x_0, \quad \frac{dx^*}{dt^*}(0) = 0$$

Three waus to scale by balancing the equation. Last term "small"

$$x^* = x_0 x, \quad t^* = Tt$$

Where T is to be determined.

$$a\frac{x_0}{T^2}\ddot{x} + b\frac{x_0}{T}\dot{x} + cx_0 = 0$$

$$\ddot{x} + \frac{bT}{a}\dot{x} + \frac{cT^2}{a} = 0$$

If we are smart can we choose the timescale $T = \frac{a}{b}$ then we get

$$\ddot{x} + \dot{x} + \frac{ca^2}{b^2a} = 0.$$

$$\implies \ddot{x} + \dot{x} + \left(\frac{ca}{b^2}\right)x = 0$$

3.4 Turbulence

Reynold number

$$R_e = \frac{u\rho L}{\mu} = \frac{uL}{\frac{mu}{\rho}} = \frac{uL}{\mathcal{V}}$$

Then we have

$$\frac{\partial v}{\partial t} = \mathcal{V} \frac{\partial^2 v}{\partial x^2}$$

4 Lecture 31/08

4.1 Turbulence

Kolmogorvs Microscales .

$$\rho \frac{du}{dt} = \mu \frac{\partial^2 u}{\partial x^2}$$

Time svale for convitive flow over a distance L

$$t_c = \frac{L}{U}, \quad U$$
 is velocity.

This can be rearranged such that

$$\frac{\partial u}{\partial t} = \left(\frac{\mu}{\rho} \frac{\partial^2 u}{\partial x^2}\right).$$

We also define $\mathcal{V} = \frac{\mu}{\rho}$ where $[\mathcal{V}] = m^2 s^{-1}$, which is the time for dispersion of velocity.

Let $t_d = \frac{L^2}{\mathcal{V}}$ such that the Reynolds number can be written

$$R_e = \frac{v\rho L}{\mu} = \frac{UL}{\left(\frac{\mu}{\rho}\right)} = \frac{UL}{\mathcal{V}} = \frac{t_d}{t_0}$$

For water is $V = 10^{-6} m^2 s^{-1}$. So for a river , put L = 100m with $U = 1ms^{-1}$

$$R_e = \frac{1ms^{-1} \cdot 100m}{10^{-6}m^2s^{-1}} = 10^8$$

Assume the generation of new whrils stops when $t_d \approx t_c \to R_e \approx 1$. Let

$$E = \frac{\text{Energy}}{\text{time per unit mass}}$$

$$[E] = kqm^2s^{-2}s^{-1}kq$$

Let l be bthe scale of the smallest whirls and u the unit velocity then is

$$E = E(l, u, \mathcal{V}).$$

We assume that E is proportional to u^2 .

$$f\left(\frac{E}{u^2}, l, \mathcal{V}\right) = 0$$

$$\begin{array}{c|c} \text{Table 2:} \\ m & s \\ \frac{E}{n^2} & 1 & 0 \\ l & 1 & 0 \\ v & 2 & -2 \end{array}$$

$$\begin{bmatrix} \frac{E}{u^2} \\ \overline{\mathcal{V}} \end{bmatrix} = m^{-2}$$

$$\Pi = \frac{\frac{E}{u^2}}{\mathcal{V}} l^2$$

$$\text{choose } \Pi = 1$$

$$\rightarrow E = \mathcal{V} (\frac{u^2}{l})^2$$

$$ul = \mathcal{V}$$

$$\implies k = \left(\mathcal{V}^3 \frac{1}{E} \right)^{\frac{1}{4}}, \quad u = (VE)^{\frac{1}{4}}$$

 $\mathbf{Example}$. Let us have 1kg what in a mixma ster and apply 100W power. then is

$$l = \left(\frac{\left(10^{-6}m^2s^{-1}\right)^3}{100m^2s^{-3}}\right)^{\frac{1}{4}} = 0.01mm$$

4.2 Regular Perturbation Theory

Assume we have an equation s.t.

$$D(x,\varepsilon) = 0$$
 where $\varepsilon \ll 1$

meaning that ε is small.

We have a solution $x(\varepsilon)$ to the problem $D(x,\varepsilon)$. The perturbation problem is regular if $\lim_{\varepsilon\to 0} x(\varepsilon)$ is a solution to D(x,0)=0. The idea is

1. Put $x(\varepsilon) = x_0 + \varepsilon x_1 + \varepsilon^2 x_2 + \dots$

$$x(\varepsilon) \approx x_0$$
 in 0. order $x(\varepsilon) \approx x_0 + \varepsilon x_1$ to 1. order

- 2. Insert $x(\varepsilon) = x_0 + \varepsilon x_1 + \dots$ into $D(x, \varepsilon)$.
- 3. Collect all terms of order 0, all terms of order 1 so that

$$D(x,\varepsilon) = 0 \leftrightarrow \overbrace{()}^{=0} + \overbrace{()\varepsilon^2}^{=0} + \dots = 0$$

Example. Let

$$x^3 + x^2 + \varepsilon x - 2 = 0$$
, $\varepsilon \ll 1$

For $\varepsilon=0$ we have x=1 as a solution. To find a solution "close to" 1 when $\varepsilon\neq 0$ we put

$$x = 1 + \varepsilon x_1 + \varepsilon^2 x_2 + O(\varepsilon)$$

Want an approximation to 2. order. We get

$$(1 + \varepsilon x_1 + \varepsilon^2 x_2)^3 + (1 + \varepsilon x_1 + \varepsilon^2 x_2)^2 + \varepsilon (1 + \varepsilon x_1 + \varepsilon^2 x_2) - 2 = 0$$

$$\implies \varepsilon (5x_1 + 1) + \varepsilon^2 (\dots) = 0$$

$$x(\varepsilon) \approx 1 - \frac{\varepsilon}{5} + \frac{\varepsilon^2}{125}$$

4.3 The Projectile Problem

Let v_0 be the vertical velocity and v_e be escape velocity such that $v_0 \ll v_e$.

Newton gravitational law

$$\mathbf{F} = -m\frac{R^2 g}{\left(R + x^*\right)^2}$$

Where g is the gravitational constand at $x^* = 0$.

Energy to move to $x^* = \infty$

$$-\int_0^\infty \mathbf{F} dx^* = mgR^2 \int_0^\infty \frac{dx^*}{(R+x^*)^2}$$
$$= mgR^2 \left[-\frac{1}{(R+x^2)} \right]_0^\infty$$
$$= mgR = \frac{1}{2} mv_e^2$$
$$\implies v_e = \sqrt{2gR}$$

We have

$$m\frac{d^2x^*}{dt^{*2}} = -m\frac{gR^2}{(R+x^*)^2}$$

Such that

$$\frac{d^2}{dt^{*2}} = -\frac{R^2 g}{(R x^*)^2}, \quad x^* (0) = 0, \quad \frac{dx^*}{dt^*} (0) = v_0$$

and $v_0 \ll v_e$, when $x^* \ll R$ (a consequence of $v_0 \ll v_e$)

$$\frac{d^2x^*}{dt^{*2}} \approx -g \quad \frac{dx^*}{dt^*} = v_0 - t^*g = 0 \quad \leftrightarrow t^* = \frac{v_0}{g} = T = \text{timescale}$$

$$X^* = v_0t^* - \frac{1}{2}t^*g \quad x^*\left(T\right) = \frac{v_0^2}{g} - \frac{1}{2}\frac{v_0^2}{g} = \frac{1}{2}\frac{v_0^2}{g}$$

Let $L = \frac{v_0^2}{g}$ and scale the equation $\left(\frac{L}{T}\right) = v_0$ and $x^* = Lx$.

$$\begin{split} \frac{L}{T^2}\ddot{x} &= \frac{-gR^2}{\left(R + Lx\right)^2} \leftrightarrow \frac{L}{T^2}\ddot{x} = -\frac{gR^2}{R^2\left(1 + \frac{L}{R}x\right)^2} \\ &\to \ddot{x} = \frac{-T^2\frac{g}{L}}{\left(1 + \frac{L}{R}x^2\right)} \to \ddot{x} = \frac{-1}{\left(1 + \varepsilon x\right)^2} \end{split}$$

Where

$$\varepsilon = \frac{L}{R} = \frac{v_0^2}{Rg} = 2\frac{2v_0^2}{v_e^2}$$

Following problem

$$\ddot{x} = \frac{-1}{(1 + \varepsilon x)^2}, \quad x(0) = 0, \quad \dot{x}(0) = 1$$

Recall that

$$f(u) = \frac{1}{(1+u)^2} \to \int f(u) = \frac{1}{1+u} + C$$
$$= C - (1 - u + u^2 - u^3 + \dots)$$
$$\implies f(u) = 1 - 2u03u^2 + O(u^3)$$

Then to second order

$$\ddot{x} = -\left(1 - 2\varepsilon x + 3\varepsilon x^2\right), \quad x\left(0\right) = 0, \quad \dot{x}\left(0\right) = q$$

Next et

$$x(t) = x_0(t) + \varepsilon x_1(t) + \varepsilon x_2(t) + O(\varepsilon)$$

So let

$$x_{j}(0) = 0 \quad \text{for} \quad j = 0, 1, 2$$

$$\ddot{x_{0}}(0) = 1, \quad \dot{x_{1}}(0) = \dot{x_{2}}(0) = 0$$

$$\rightarrow \ddot{x_{0}} + \varepsilon \ddot{x_{1}} + \varepsilon^{2} \ddot{x_{2}} = -1 + 2\varepsilon \left(x_{0}0\varepsilon x_{1}\right) - 3\varepsilon^{2}x_{0}^{2}$$

$$\rightarrow (\ddot{x_{0}} + 1) + \varepsilon \left(\ddot{x_{1}} - 2x_{0}\right) + \varepsilon^{2} \left(\ddot{x_{2}} + 2x_{1} + 3x_{0}^{2}\right) = 0$$

$$\ddot{x_{0}} = -1 \quad x_{0}(0) = 0, \quad \dot{x_{0}} = 1$$

$$\ddot{x_{1}} = 2x_{0}, \quad \dot{x_{1}}(0) = \dot{x_{i}}(0) = 0$$

$$\ddot{x_{2}} = 2x_{1} - 3x_{0}^{2}, \quad x_{2}(0) = \dot{x_{2}}(0) = 0$$

$$\rightarrow x_{0}(t) = t - \frac{1}{2}tst$$

$$\ddot{x_{1}}(t) = 2t - t^{2}$$

$$\dot{x_{1}}(t) = t^{2} - \frac{1}{3}t^{3}$$

$$x_{1}(t) = \frac{1}{3}t^{3} - \frac{1}{12}t^{4}$$

Where

$$\ddot{x_2} = \frac{2}{3}t^3 - \frac{1}{6}t^4 - 3\left(t^2 - t^3 + \frac{1}{4}t^4\right)$$
$$x_2 = -\frac{1}{4}t^4 + \frac{11}{60}t^5 - \frac{11}{360}t^6$$

Which end up with

$$x\left(t\right) = t - \frac{1}{2}t^{2}0\varepsilon\left(\frac{1}{3}t^{3} - \frac{1}{12}\right) + \varepsilon^{2}\left(-\frac{t^{4}}{4}0\frac{11}{60}t^{5} - \frac{11}{360}t^{6}\right)$$

Gives the diea of how to approx the time to the maximum height. $\dot{x}\left(t\right)=0$ is a 5. degree equation containing ε .

Lets put

$$t = 1 + \varepsilon t_2 \varepsilon^2 t_2$$

Into the 5. degree edition and to regular perturabation

$$\rightarrow t = 1 + \frac{2}{3}\varepsilon + 2/5\varepsilon^2 + O(\varepsilon)$$

such that

$$\ddot{x} = \frac{-1}{(1+\varepsilon x)^2} \to \ddot{x}\dot{x} = \frac{\dot{x}}{(1+\varepsilon x)^2}$$

$$\to \frac{d}{dt}\left(\frac{1}{2}\dot{x}^2\right) = \frac{d}{dt}\left(\frac{-1}{\varepsilon}\frac{1}{1+\varepsilon x}\right)$$

$$\frac{1}{2}\dot{x}^2 = \frac{-1}{\varepsilon}\frac{1}{1+\varepsilon x} + C$$

$$\frac{1}{2} = \frac{-1}{\varepsilon}$$

$$C = \frac{1}{2} + \frac{1}{\varepsilon}$$

where

$$\frac{1}{2}\dot{x}^2 = \frac{-1}{\varepsilon}\frac{1}{1+\varepsilon x} + \frac{1}{2} + \frac{1}{\varepsilon}$$

At maximum height $\dot{x} = 0$

$$0 = -\frac{1}{\varepsilon}.$$

5 Lecture 02/09

Let Newtons Law be

$$\frac{d^2s^*}{dt^{*2}} = g\sin\left(\alpha^*\right) \implies \frac{d^2\alpha^*}{dt^{*2}} = -\frac{g}{L}\sin\left(\alpha^*\right)$$

scaling:

$$\begin{split} \alpha^* &= \varepsilon \alpha, \quad t^* = Tt \\ \frac{\varepsilon}{T^2} \ddot{\alpha} &= \frac{-g}{L} \sin \left(\varepsilon \alpha \right) \implies \ddot{\alpha} = -\left(T^2 g \frac{1}{L} \right) \frac{\sin \left(\varepsilon \alpha \right)}{\varepsilon} \\ T &= \sqrt{\frac{L}{g}} \implies \ddot{\alpha} = -\frac{\sin \left(\varepsilon \alpha \right)}{\varepsilon} \\ \alpha \left(0 \right) &= 1 \quad \dot{\alpha} \left(0 \right) = 0 \end{split}$$

Let put $\alpha = \alpha_0(t) + \varepsilon^2 \alpha_2(t) + O(\varepsilon^4)$. where $\alpha(t)$ is an even function of ε due to symmetry.

$$\alpha_0(0) = 1$$
, $\dot{\alpha}_0(0) = 0$, $\alpha_2(0) = \dot{\alpha}_2(0) = 0$

Inserted into the equation

$$\ddot{\alpha_0} + \varepsilon^2 \ddot{\alpha_2} = -\frac{\sin\left(\varepsilon\left(\alpha_0 + \varepsilon^2 \alpha_2\right)\right)}{\varepsilon} \implies \ddot{\alpha_0} + \varepsilon^2 \ddot{\alpha_2}$$
$$= \frac{-1}{3} \left(\varepsilon\underbrace{\left(\alpha_0 + \varepsilon^2 \alpha_2\right)}_{u} \frac{\varepsilon^2}{6} \left(\alpha_0 + \alpha \varepsilon^2\right)\right)$$

Let

$$\begin{aligned} &\alpha_0\left(t\right) = A\cos t + B\sin t\\ &\alpha_0\left(0\right) = 1, \quad \dot{\alpha}\left(0\right) = 0 \quad \Longrightarrow \quad \alpha_0\left(t\right) = \cos t\\ &\alpha_2\left(t\right) = A\cos t + B\sin t + \alpha_{2,f}\left(t\right)\\ &\cos^3 t = \left(\frac{1}{2}\left(e^{it} - e^{it}\right)\right)^3 = \frac{1}{8}\left(e^{i3t} + 3e^{it}03e^{-i3t}\right)\\ &= \frac{1}{4}\left(\cos 3t + 3\cos t\right)\\ &\alpha_{20}\left(t\right) = A\cos 3t + B\sin 3t + Ct\cos t + Dt\sin t\\ &\alpha_2\left(t\right) = \frac{1}{192}\left(\cos t + \cos 3t\right) + \frac{1}{16}t\sin t\\ &\alpha\left(t\right) = \alpha_0\left(t\right) + \varepsilon_2^2\left(t\right) \quad \text{is not periodic} \end{aligned}$$

Poincare-Lin Stel Method . Instead let

$$\alpha\left(t\right) = \alpha_{0}\left(\omega\left(\varepsilon\right)t\right) + \alpha_{2}\left(\omega\left(\varepsilon\right)t\right)\varepsilon^{2} + O\left(\varepsilon^{4}\right)$$

Where $\omega\left(\varepsilon\right)=1+\omega_{2}\varepsilon^{2}~O\left(\varepsilon^{4}\right)$. See exercise.

5.1 Modelling how the kidney disposes salt and water.

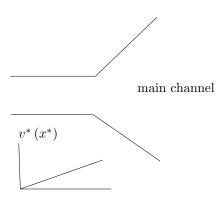


Figure 5: watermodell

Assumptions

- 1. Secondary channel is fed water by osmosis from the sorrouinding tissue.
- 2. Ions are transported down the channel by connection and diffusion.
- 3. Ions are fed into the channel be a chemical ppump-

We want the steady-state profiles of ion concenstration $C^*(x^*)$ and the velocity $v^*(x^*)$ of the ion water solution.

The ion concentration is written as

$$[C^*] = \frac{ions}{m^3} = \frac{osmol}{m^3}$$

One mole salt give two moles ions

 ${\bf Osomosis}:$

$$J^* = P\left(c^* - c_0\right)$$



Figure 6: molefig

Is flux density of water entering the secondary channel. J^* is volume water in per area per time. c_0 ion concentration is tissue and main channel. P is called membrance permeability.

$$[P] = \frac{[J^*]}{[c^*]} = \frac{ms^{-1}}{osmol \cdot m^{-3}} = \frac{m^4}{s \cdot osmol}$$

Ion flux density

$$N^* = \begin{cases} N_0, & 0 \le x^* \le \delta \\ 0, & \delta \le x^* \le L \end{cases}$$

Where $[N_0] = \frac{osmol}{m^2 \cdot s}$. The toal rate of salt entering the channel

$$N_0 \cdot c \cdot \delta$$

Where c is the area of pump.

• The flux density of ions in the secondary channel

$$F^* = F_c^* + F_\alpha^*$$

$$[F^*] = \frac{osmol}{m^2 \cdot s}$$

• Convective flow

$$F_c^* = c^* v^*$$

• Diffusion: Ficus law

$$F_1^* = -D\frac{dc^*}{dx^*}.$$

where D is the diffusion of salt in water.

Conservation of water

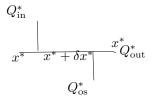


Figure 7: conssswater

$$Q_{\text{out}}^* = Q_{\text{in}}^* + Q_{\text{os}}^*$$

$$v^* (x^* + \Delta x^*) = v^* A + P \left(c^* \left(\hat{x} \right) - c_0 \right) c \Delta x^* \quad \text{where} \quad \hat{x^*} \in \langle x^*, x^* + \Delta x^* \rangle$$

$$\implies \frac{v^* (x^* + \Delta x^*) - v^* (x^*)}{\Delta x^*} = \frac{c}{A} P \left(c^* \left(\hat{x^*} \right) - c_0 \right)$$

$$\Delta x^* \to 0 \quad \implies \frac{dv^*}{dx^*} = \left(\frac{cP}{A} \right) (c^* - c_0)$$

COnservation of salt

$$F^*\left(x^* + \Delta x^*\right)A = F^*\left(x^*\right)A + N^*\left(\hat{x^*}\right)c\Delta x^*$$

This ends up with

$$\Rightarrow \frac{dF^*}{dx^*} = \frac{c}{A}N^*(x^*)$$
or
$$\frac{dF^*}{dx^*} = \frac{c}{A} \cdot \begin{cases} N_0, & 0 < x^* < \delta \\ 0, & \delta < x^* < L \end{cases}$$

$$F^*(0) = 0 \Rightarrow F(x^*) = \begin{cases} \frac{N_0 c}{A}x^*, & 0 < x^* < \delta \\ \frac{N_0 \delta c}{A}, & \delta < x^* < L \end{cases}$$

$$\Rightarrow v^*c^* - D\frac{dc^*}{dx^*} = F^*(x^*)$$

$$\frac{dv^*}{dt^*} = \frac{cP}{A}(c^* - c_0)$$

$$v^*(0) = 0$$

$$c^*(L) = c_0$$

Also same that v^* and c^* are continious at $x^* = \delta$.

5.1.1 Scaling the model

Two length scales δ and L. Choose δ as length svale. Natural to use c_0 as scale for c^* . The rate salt supplied is

$$N_0 \delta c = c_0 U A$$

Ions supplied is convective flux with c^* such that $U = \frac{N_0 \delta c}{c_0 A}$.

$$x^* = \delta,$$

$$c^* = c_0 c$$

$$v^* = U v$$

1. $(Uc_0) cv - \frac{Dc_0}{\delta} \dot{c} = F^*$ such that

$$\implies vc - \frac{Dc}{\delta Uc_0}\dot{c} = \frac{1}{Uc} \cdot \begin{cases} \frac{N_0c\delta x}{AUc_0}, & 0 < x\delta < \delta \\ \frac{N_0c\delta}{Auc_0}, & \delta < x\delta < L \end{cases}$$
$$vc - \varepsilon \dot{c} = \begin{cases} x & 0 < x < 1 \\ 1 & 1 < x < \lambda \end{cases}$$

where $\varepsilon = \frac{D}{\delta u}$, and $\lambda = \frac{L}{\delta}$

$$\implies U = \frac{N_0 \delta c}{c_0 A}$$

$$2. \ \frac{U}{\delta}\dot{v} = \frac{cP}{A}c_0\left(c - 1\right)$$

6 References