

AERO 632: Design of Advance Flight Control System

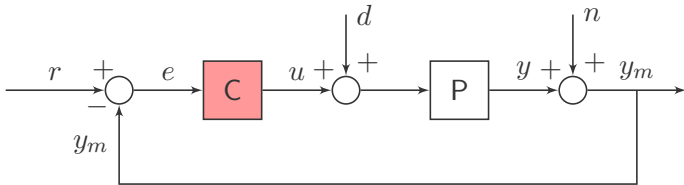
Preliminaries

Raktim Bhattacharya

Laboratory For Uncertainty Quantification
Aerospace Engineering, Texas A&M University.

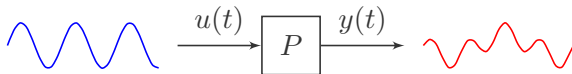
Preliminaries

- Signals & Systems
- Laplace transforms
- Transfer functions – from ordinary **linear** differential equations
- System interconnections
- Block diagram algebra – simplification of interconnections
- General feedback control system interconnection.



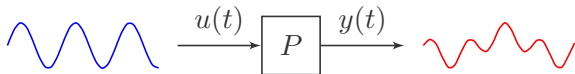
Signals & Systems

Signals & Systems



- Actuator applies $u(t)$
- Sensor provides $y(t)$
- Feedback controller takes $y(t)$ and determines $u(t)$ to achieve desired behavior
- The controller is typically implemented as software, running in a micro controller
- Imperfections exist in real world
 - ▶ sensors have noise
 - ▶ actuators have irregularities
 - ▶ plant P is not fully known

System Response to $u(t)$



Given plant P and input $u(t)$, what is $y(t)$?

- P is defined in terms of **ordinary differential equations**
- $y(t)$ is the forced + initial condition response.

Linear Dynamics

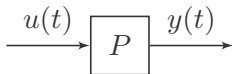
$$m\ddot{x} + c\dot{x} + kx = u(t) \text{ dynamics}$$
$$y(t) = x(t) \text{ measurement}$$

Nonlinear Dynamics

$$\ddot{x} - \mu(1 - x^2)\dot{x} + x = u(t) \text{ dynamics}$$
$$y(t) = x(t) \text{ measurement}$$

In this class we focus on linear systems

Linear Systems



- Dynamics is defined by linear ordinary differential equation
- Super position principle applies

$$\begin{aligned} u_1(t) &\mapsto y_1(t) \\ u_2(t) &\mapsto y_2(t) \end{aligned} \implies (u_1(t) + u_2(t)) \mapsto (y_1(t) + y_2(t))$$

Laplace Transforms

Laplace Transforms

Given signal $u(t)$, Laplace transform is defined as

$$\mathcal{L}\{u(t)\} := \int_0^{\infty} u(t)e^{-st} dt$$

Exists when

$$\lim_{t \rightarrow \infty} |u(t)e^{-\sigma t}| = 0, \text{ for some } \sigma > 0$$

Very useful in studying linear dynamical systems and designing controllers

Properties Laplace Transforms

Linear operator

■ Additive

$$\begin{aligned}\mathcal{L}\{u_1(t) + u_2(t)\} &= \int_0^{\infty} (u_1(t) + u_2(t)) e^{-st} dt \\ &= \int_0^{\infty} u_1(t) e^{-st} dt + \int_0^{\infty} u_2(t) e^{-st} dt \\ &= \mathcal{L}\{u_1(t)\} + \mathcal{L}\{u_2(t)\}\end{aligned}$$

■ Superposition

$$\mathcal{L}\{au(t)\} = a\mathcal{L}\{u(t)\}, \quad a \text{ is a constant}$$

Properties (contd.)

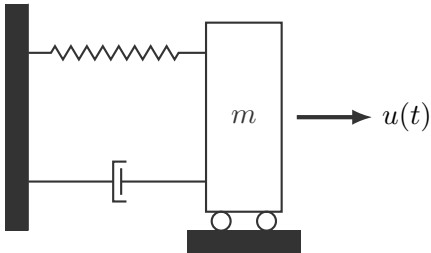
1. $U(s) := \mathcal{L}\{u(t)\}$
2. $\mathcal{L}\{au_1(t) + bu_2(t)\} = a\mathcal{L}\{u_1(t)\} + b\mathcal{L}\{u_2(t)\} = aU_1(s) + bU_2(s)$
3. $\frac{1}{s}U(s) \iff \int_0^t u(\tau)d\tau$
4. $U_1(s)U_2(s) \iff u_1(t) * u_2(t)$ Convolution
5. $\lim_{s \rightarrow 0} sU(s) \iff \lim_{t \rightarrow \infty} u(t)$ Final value theorem
6. $\lim_{s \rightarrow \infty} sU(s) \iff u(0^+)$ Initial value theorem
7. $-\frac{dU(s)}{ds} \iff tu(t)$
8. $\mathcal{L}\left\{\frac{du}{dt}\right\} \iff sU(s) - su(0)$
9. $\mathcal{L}\{\ddot{u}\} \iff s^2U(s) - su(0) - \dot{u}(0)$

Important Signals

1. $\mathcal{L}\{\delta(t)\} = 1$ $\delta(t)$ is impulse function
2. $\mathcal{L}\{1(t)\} = \frac{1}{s}$ $1(t)$ is unit step function at $t = 0$
3. $\mathcal{L}\{t\} = \frac{1}{s^2}$
4. $\mathcal{L}\{\sin(\omega t)\} = \frac{\omega}{s^2 + \omega^2}$
5. $\mathcal{L}\{\cos(\omega t)\} = \frac{s}{s^2 + \omega^2}$

Transfer Functions

Spring Mass Damper System



Equation of Motion

$$m\ddot{x} + c\dot{x} + kx = u(t)$$

Take $\mathcal{L}\{\cdot\}$ on both sides

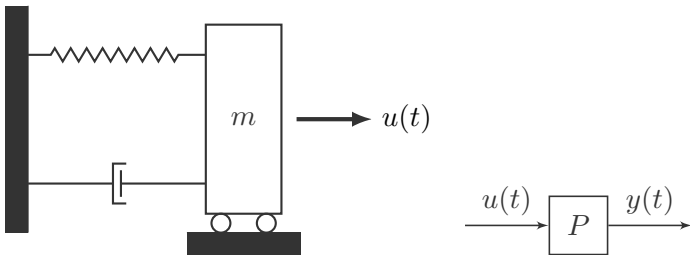
$$\mathcal{L}\{m\ddot{x} + c\dot{x} + kx\} = \mathcal{L}\{u(t)\}$$

$$m\mathcal{L}\{\ddot{x}\} + c\mathcal{L}\{\dot{x}\} + k\mathcal{L}\{x\} = \mathcal{L}\{u(t)\}$$

$$m(s^2X(s) - sx(0) - \dot{x}(0)) + c(sX(s) - x(0)) + kX(s) = U(s)$$

$$(ms^2 + cs + k)X(s) = U(s) \quad \dot{x}(0) \text{ and } x(0) \text{ are assumed to be zero}$$

Transfer Function



$$(ms^2 + cs + k)X(s) = U(s) \implies \frac{X(s)}{U(s)} = \frac{1}{ms^2 + cs + k}$$

Choose output $y(t) = x(t) \implies Y(s) = X(s)$.

Therefore

$$P(s) := \frac{Y(s)}{U(s)} = \frac{1}{ms^2 + cs + k} \text{ Transfer function}$$

Transfer Function (contd.)

In general

$$P(s) = \frac{N(s)}{D(s)}$$

where $N(s)$ and $D(s)$ are polynomials in s

- Roots of $N(s)$ are the **zeros**
- Roots of $D(s)$ are the **poles** – determine stability

Response to $u(t)$

Given

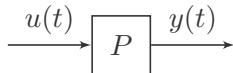
- input signal $u(t)$ and transfer function $P(s)$.

Determine

- output response $y(t)$

1. Laplace transform

$$U(s) := \mathcal{L}\{u(t)\}$$



2. Determine $Y(s) := P(s)U(s)$

3. Laplace inverse

$$y(t) := \mathcal{L}^{-1}\{Y(s)\} = \mathcal{L}^{-1}\{P(s)U(s)\}$$

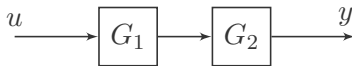
System Interconnection

Block Diagram

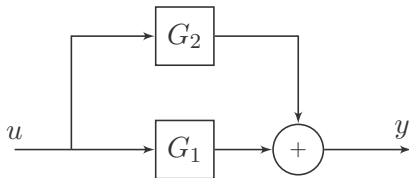
Representation of System Interconnections

- Series
- Parallel
- Feedback
- A simple example
- A complex example

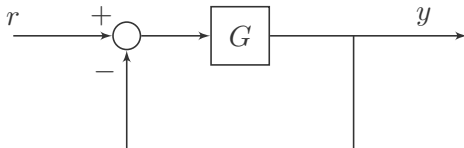
Series Connection



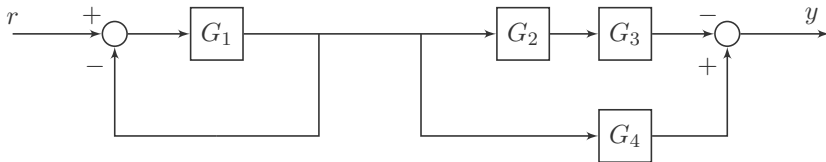
Parallel Connection



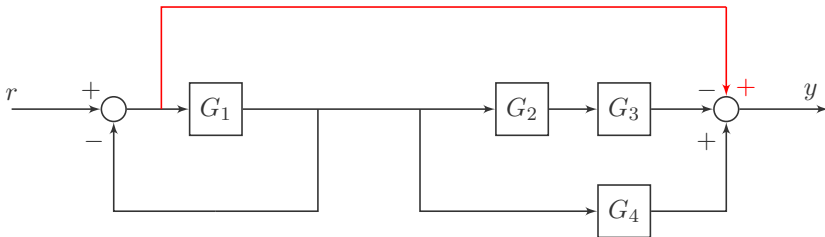
Feedback Connection



Simple Example

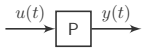


Complex Example



Frequency Response

Response to Sinusoidal Input



- Let $u(t) = A_u \sin(\omega t)$
- Vary ω from 0 to ∞

A linear system's response to sinusoidal inputs is called the system's frequency response

Response to Sinusoidal Input

Example

- Let $P(s) = \frac{1}{s+1}$, $u(t) = \sin(t)$

$$\begin{aligned}
 y(t) &= \frac{1}{2}e^{-t} - \frac{1}{2}\cos(t) + \frac{1}{2}\sin(t) \\
 &= \underbrace{\frac{1}{2}e^{-t}}_{\text{natural response}} + \underbrace{\frac{1}{\sqrt{2}}\sin(t - \frac{\pi}{4})}_{\text{forced response}}
 \end{aligned}$$

- Forced response has form $A_y \sin(\omega t + \phi)$
- A_y and ϕ are functions of ω

Response to Sinusoidal Input

Generalization

In general

$$\begin{aligned} Y(s) &= G(s) \frac{\omega_0}{s^2 + \omega_0^2} \\ &= \frac{\alpha_1}{s - p_1} + \cdots + \frac{\alpha_n}{s - p_n} + \frac{\alpha_0}{s + j\omega_0} + \frac{\alpha_0^*}{s - j\omega_0} \\ \Rightarrow y(t) &= \underbrace{\alpha_1 e^{p_1 t} + \cdots + \alpha_n e^{p_n t}}_{\text{natural}} + \underbrace{A_y \sin(\omega_0 + \phi)}_{\text{forced}} \end{aligned}$$

Forced response has **same** frequency, **different** amplitude and phase.

Response to Sinusoidal Input

Generalization (contd.)

For a system $P(s)$ and input

$$u(t) = A_u \sin(\omega_0 t),$$

forced response is

$$y(t) = A_u \textcolor{red}{M} \sin(\omega_0 t + \textcolor{red}{\phi}),$$

where

$$M(\omega_0) = |P(s)|_{s=j\omega_0} = |P(j\omega_0)|, \text{ magnitude}$$

$$\phi(\omega_0) = \angle P(j\omega_0) \text{ phase}$$

In polar form

$$P(j\omega_0) = M e^{j\phi}.$$

Fourier Analysis

Fourier Series Expansion

Given a signal $y(t)$ with periodicity T ,

$$y(t) = \frac{a_0}{2} + \sum_{n=1,2,\dots} a_n \cos\left(\frac{2\pi nt}{T}\right) + b_n \sin\left(\frac{2\pi nt}{T}\right)$$

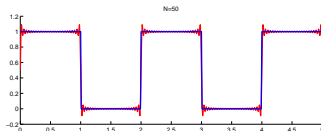
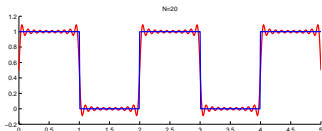
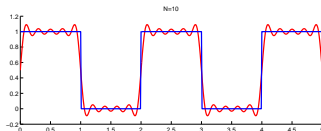
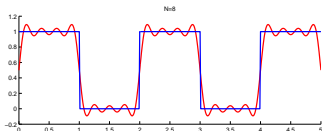
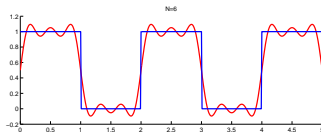
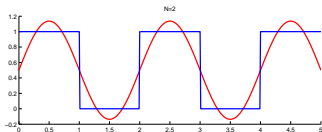
$$a_0 = \frac{2}{T} \int_0^T y(t) dt$$

$$a_n = \frac{2}{T} \int_0^T y(t) \cos\left(\frac{2\pi nt}{T}\right) dt$$

$$b_n = \frac{2}{T} \int_0^T y(t) \sin\left(\frac{2\pi nt}{T}\right) dt$$

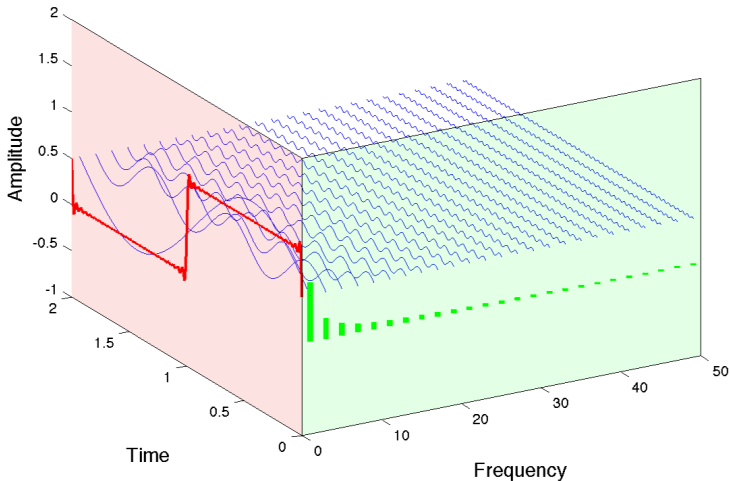
Fourier Series Expansion

Approximation of step function



Fourier Transform

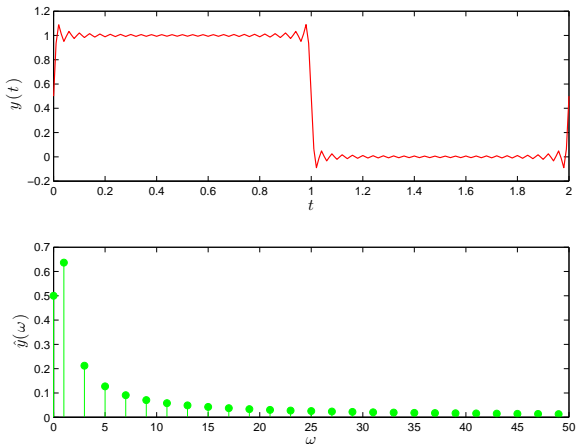
Step function



Fourier transform reveals the frequency content of a signal

Fourier Transform

Step function – frequency content



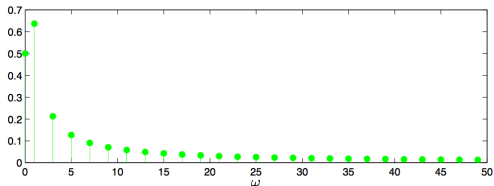
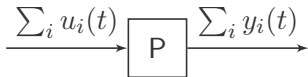
Signals & Systems

Input Output

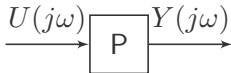


Fourier Series Expansion

superposition principle



Fourier Transform



$$u_i(t) = a_i \sin(\omega_i t)$$

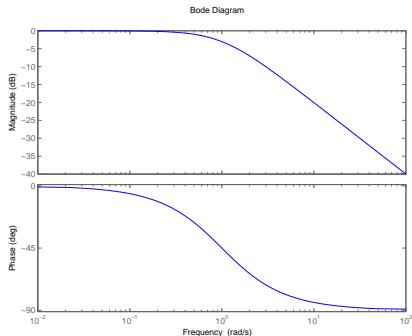
$$y_{i\text{forced}}(t) = a_i M \sin(\omega_i t + \phi)$$

$$Y(j\omega) = P(j\omega)U(j\omega)$$

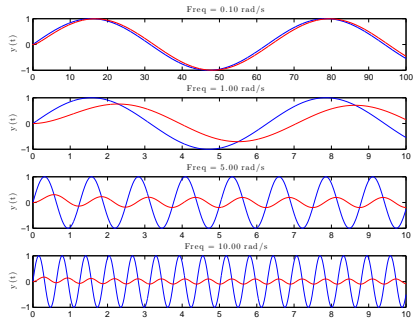
Suffices to study $P(j\omega)$ $|P(j\omega)|$, $\angle P(j\omega)$

Bode Plot

First Order System

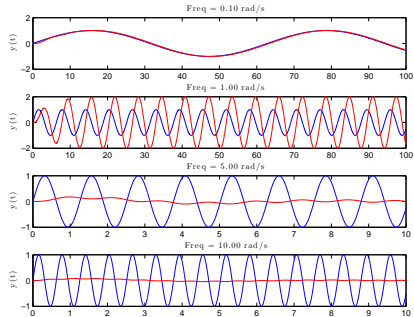
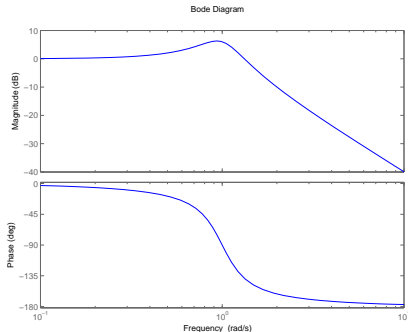


- $P(s) = 1/(s + 1)$
- loglog scale
- $\text{dB} = 10 \log_{10}(\cdot)$
- $20\text{dB} = 10 \log_{10}(100/1)$



- $u(t) = A \sin(\omega_0 t)$
- $y_{\text{forced}}(t) = A M \sin(\omega_0 t + \phi)$

Second Order System



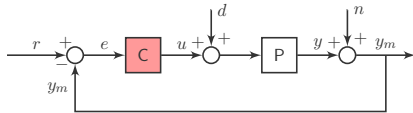
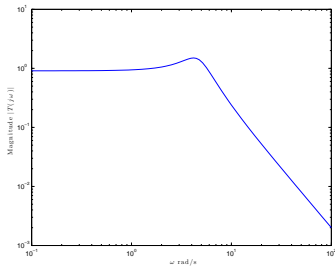
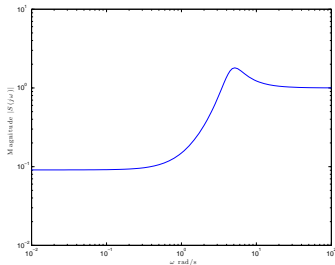
■ $P(s) = 1/(s^2 + 0.5s + 1)$

■ $\omega_n = 1 \text{ rad/s}$

■ $u(t) = A \sin(\omega_0 t)$

■ $y_{\text{forced}}(t) = A \mathbf{M} \sin(\omega_0 t + \phi)$

$$S(j\omega) + T(j\omega) = 1$$

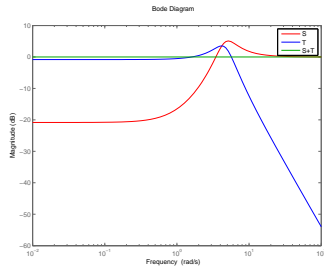


$$\blacksquare P(s) = \frac{1}{(s+1)(s/2+1)}$$

$$\blacksquare C(s) = 10$$

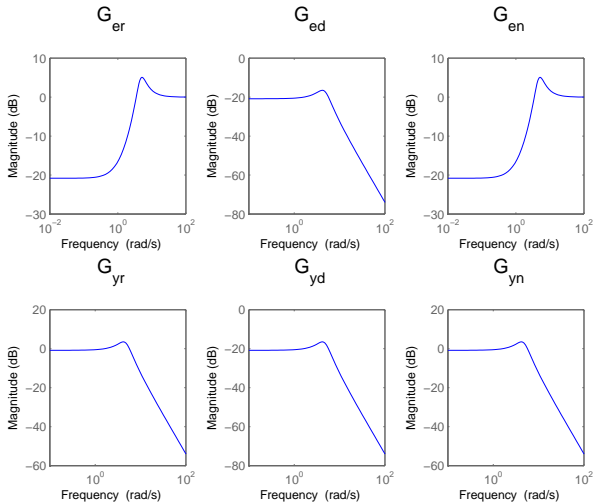
$$\blacksquare S = G_{er} = \frac{1}{1+PC} = \frac{1}{1+10P}$$

$$\blacksquare T = G_{yr} = \frac{PC}{1+PC} = \frac{10P}{1+10P}$$



All transfer functions

With proportional controller



Controller Design Considerations

Design Using Bode Plot of $P(j\omega)C(j\omega)$

Loop Shaping

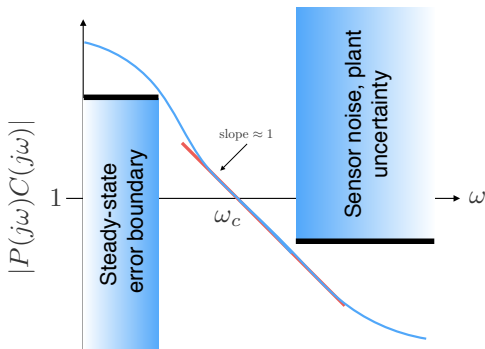
Develop conditions on the Bode plot of the open loop transfer function

- Sensitivity $\frac{1}{1+PC}$
- Steady-state errors: slope and magnitude at $\lim_{\omega} \rightarrow 0$
- Robust to sensor noise
- Disturbance rejection
- Controller roll off \Rightarrow not excite high-frequency modes of plant
- Robust to plant uncertainty

Look at Bode plot of $L(j\omega) := P(j\omega)C(j\omega)$

Frequency Domain Specifications

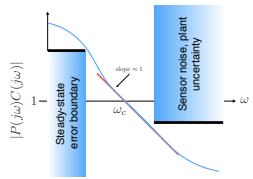
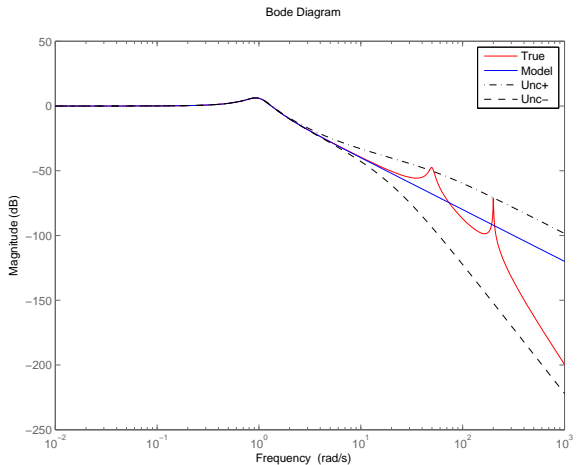
Constraints on the shape of $L(j\omega)$



- Choose $C(j\omega)$ to ensure $|L(j\omega)|$ does not violate the constraints
- Slope ≈ -1 at ω_c ensures $PM \approx 90^\circ$
stable if $PM > 0 \implies \angle PC > -180^\circ$

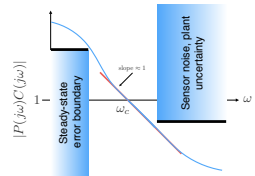
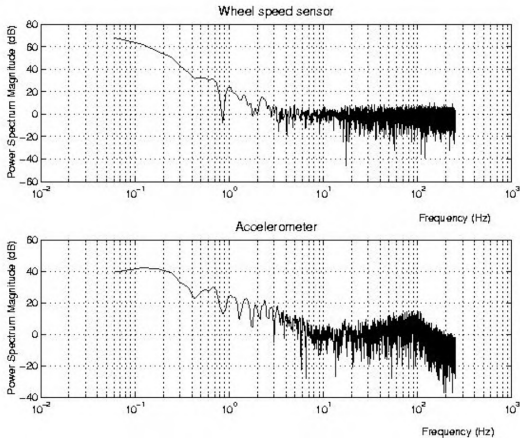
Plant Uncertainty

$$P(j\omega) = P_0(j\omega)(1 + \Delta P(j\omega))$$



Sensor Characteristics

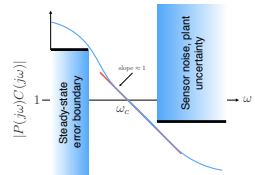
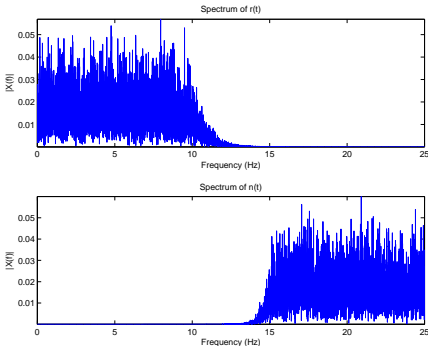
Noise spectrum



$$G_{yn} = -\frac{PC}{1 + PC}$$

Reference Tracking

Bandlimited else conflicts with noise rejection

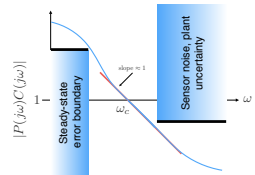
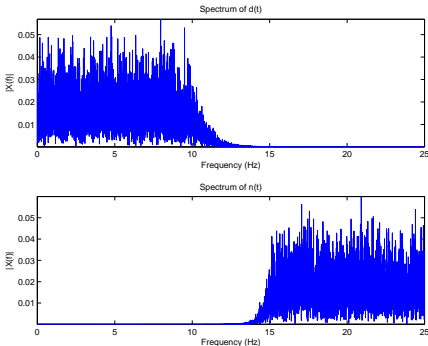


$$G_{yr} = \frac{PC}{1 + PC}$$

$$G_{yn} = -\frac{PC}{1 + PC}$$

Disturbance Rejection

Bandlimited else conflicts with noise rejection



$$G_{yd} = \frac{P}{1 + PC}$$

$$G_{yn} = -\frac{PC}{1 + PC}$$