

# **AERO 632: Design of Advance Flight Control System**

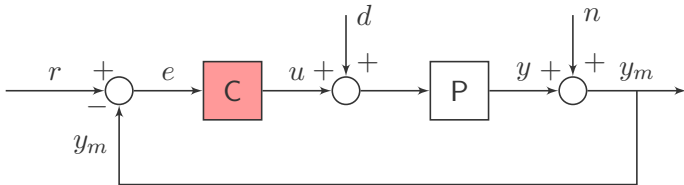
Preliminaries

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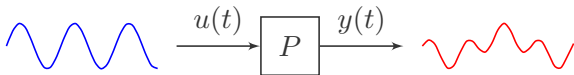
# Preliminaries

- Signals & Systems
- Laplace transforms
- Transfer functions – from ordinary **linear** differential equations
- System interconnections
- Block diagram algebra – simplification of interconnections
- General feedback control system interconnection.



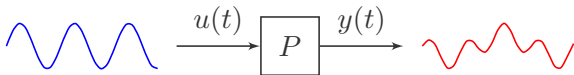
# **Signals & Systems**

# Signals & Systems



- Actuator applies  $u(t)$
- Sensor provides  $y(t)$
  
- Feedback controller takes  $y(t)$  and determines  $u(t)$  to achieve desired behavior
- The controller is typically implemented as software, running in a micro controller
  
- Imperfections exist in real world
  - ▶ *sensors have noise*
  - ▶ *actuators have irregularities*
  - ▶ *plant  $P$  is not fully known*

# System Response to $u(t)$



**Given plant  $P$  and input  $u(t)$ , what is  $y(t)$ ?**

- $P$  is defined in terms of **ordinary differential equations**
- $y(t)$  is the forced + initial condition response.

## Linear Dynamics

$$m\ddot{x} + c\dot{x} + kx = u(t) \text{ dynamics}$$

$$y(t) = x(t) \text{ measurement}$$

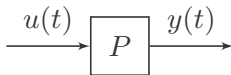
## Nonlinear Dynamics

$$\ddot{x} - \mu(1 - x^2)\dot{x} + x = u(t) \text{ dynamics}$$

$$y(t) = x(t) \text{ measurement}$$

**In this class we focus on linear systems**

# Linear Systems



- Dynamics is defined by linear ordinary differential equation
- Super position principle applies

$$\begin{aligned} u_1(t) &\mapsto y_1(t) \\ u_2(t) &\mapsto y_2(t) \end{aligned} \implies (u_1(t) + u_2(t)) \mapsto (y_1(t) + y_2(t))$$

# Laplace Transforms

# Laplace Transforms

**Given** signal  $u(t)$ , Laplace transform is defined as

$$\mathcal{L}\{u(t)\} := \int_0^{\infty} u(t)e^{-st}dt$$

**Exists** when

$$\lim_{t \rightarrow \infty} |u(t)e^{-\sigma t}| = 0, \text{ for some } \sigma > 0$$

Very useful in studying linear dynamical systems and designing controllers



# Properties Laplace Transforms

## Linear operator

### ■ Additive

$$\begin{aligned}\mathcal{L}\{u_1(t) + u_2(t)\} &= \int_0^{\infty} (u_1(t) + u_2(t)) e^{-st} dt \\ &= \int_0^{\infty} u_1(t) e^{-st} dt + \int_0^{\infty} u_2(t) e^{-st} dt \\ &= \mathcal{L}\{u_1(t)\} + \mathcal{L}\{u_2(t)\}\end{aligned}$$

### ■ Superposition

$$\mathcal{L}\{au(t)\} = a\mathcal{L}\{u(t)\}, \text{ } a \text{ is a constant}$$

# Properties (contd.)

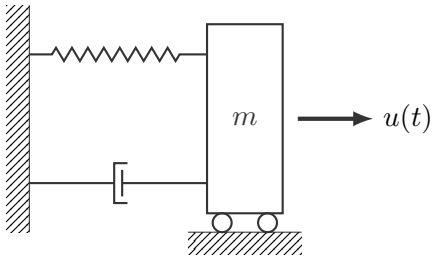
1.  $U(s) := \mathcal{L}\{u(t)\}$
2.  $\mathcal{L}\{au_1(t) + bu_2(t)\} = a\mathcal{L}\{u_1(t)\} + b\mathcal{L}\{u_2(t)\} = aU_1(s) + bU_2(s)$
3.  $\frac{1}{s}U(s) \iff \int_0^t u(\tau)d\tau$
4.  $U_1(s)U_2(s) \iff u_1(t) * u_2(t)$  Convolution
5.  $\lim_{s \rightarrow 0} sU(s) \iff \lim_{t \rightarrow \infty} u(t)$  Final value theorem
6.  $\lim_{s \rightarrow \infty} sU(s) \iff u(0^+)$  Initial value theorem
7.  $-\frac{dU(s)}{ds} \iff tu(t)$
8.  $\mathcal{L}\left\{\frac{du}{dt}\right\} \iff sU(s) - su(0)$
9.  $\mathcal{L}\{\ddot{u}\} \iff s^2U(s) - su(0) - \dot{u}(0)$

# Important Signals

1.  $\mathcal{L}\{\delta(t)\} = 1$   $\delta(t)$  is impulse function
2.  $\mathcal{L}\{1(t)\} = \frac{1}{s}$   $1(t)$  is unit step function at  $t = 0$
3.  $\mathcal{L}\{t\} = \frac{1}{s^2}$
4.  $\mathcal{L}\{\sin(\omega t)\} = \frac{\omega}{s^2 + \omega^2}$
5.  $\mathcal{L}\{\cos(\omega t)\} = \frac{s}{s^2 + \omega^2}$

# Transfer Functions

# Spring Mass Damper System



## Equation of Motion

$$m\ddot{x} + c\dot{x} + kx = u(t)$$

Take  $\mathcal{L}\{\cdot\}$  on both sides

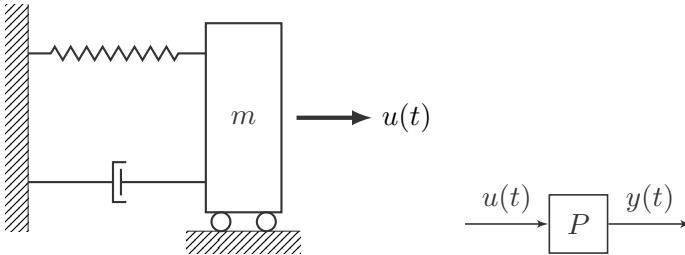
$$\mathcal{L}\{m\ddot{x} + c\dot{x} + kx\} = \mathcal{L}\{u(t)\}$$

$$m\mathcal{L}\{\ddot{x}\} + c\mathcal{L}\{\dot{x}\} + k\mathcal{L}\{x\} = \mathcal{L}\{u(t)\}$$

$$m(s^2X(s) - sx(0) - \dot{x}(0)) + c(sX(s) - x(0)) + kX(s) = U(s)$$

$$(ms^2 + cs + k)X(s) = U(s) \quad \dot{x}(0) \text{ and } x(0) \text{ are assumed to be zero}$$

# Transfer Function



$$(ms^2 + cs + k)X(s) = U(s) \implies \frac{X(s)}{U(s)} = \frac{1}{ms^2 + cs + k}$$

Choose output  $y(t) = x(t) \implies Y(s) = X(s)$ .

Therefore

$$P(s) := \frac{Y(s)}{U(s)} = \frac{1}{ms^2 + cs + k} \quad \text{Transfer function}$$

# Transfer Function (contd.)

In general

$$P(s) = \frac{N(s)}{D(s)}$$

where  $N(s)$  and  $D(s)$  are polynomials in  $s$

- Roots of  $N(s)$  are the **zeros**
- Roots of  $D(s)$  are the **poles** – determine stability

# Response to $u(t)$

## Given

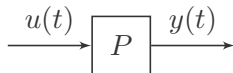
- input signal  $u(t)$  and transfer function  $P(s)$ .

## Determine

- output response  $y(t)$

### 1. Laplace transform

$$U(s) := \mathcal{L}\{u(t)\}$$



### 2. Determine $Y(s) := P(s)U(s)$

### 3. Laplace inverse

$$y(t) := \mathcal{L}^{-1}\{Y(s)\} = \mathcal{L}^{-1}\{P(s)U(s)\}$$



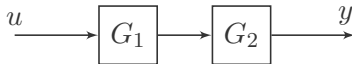
# **System Interconnection**

# Block Diagram

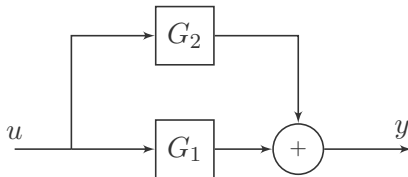
*Representation of System Interconnections*

- Series
- Parallel
- Feedback
- A simple example
- A complex example

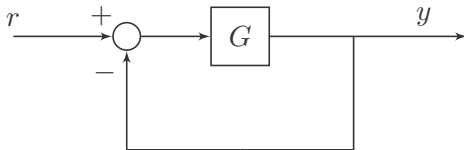
# Series Connection



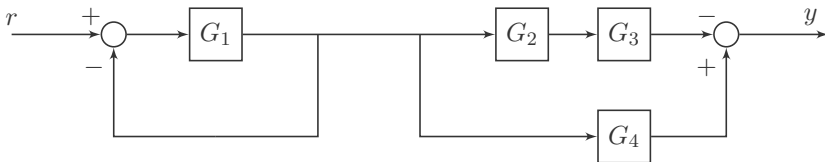
# Parallel Connection



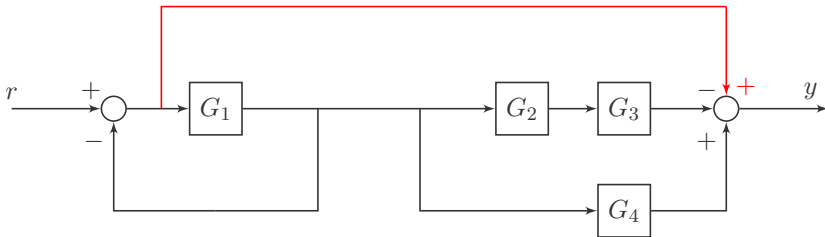
# Feedback Connection



# Simple Example



# Complex Example



# Frequency Response



# Response to Sinusoidal Input



- Let  $u(t) = A_u \sin(\omega t)$
- Vary  $\omega$  from 0 to  $\infty$

A linear system's response to sinusoidal inputs is called the system's frequency response

# Response to Sinusoidal Input

*Example*

- Let  $P(s) = \frac{1}{s+1}$ ,  $u(t) = \sin(t)$

$$\begin{aligned} y(t) &= \frac{1}{2}e^{-t} - \frac{1}{2}\cos(t) + \frac{1}{2}\sin(t) \\ &= \underbrace{\frac{1}{2}e^{-t}}_{\text{natural response}} + \underbrace{\frac{1}{\sqrt{2}}\sin(t - \frac{\pi}{4})}_{\text{forced response}} \end{aligned}$$

- Forced response has form  $A_y \sin(\omega t + \phi)$
- $A_y$  and  $\phi$  are functions of  $\omega$

# Response to Sinusoidal Input

## Generalization

In general

$$\begin{aligned} Y(s) &= G(s) \frac{\omega_0}{s^2 + \omega_0^2} \\ &= \frac{\alpha_1}{s - p_1} + \cdots + \frac{\alpha_n}{s - p_n} + \frac{\alpha_0}{s + j\omega_0} + \frac{\alpha_0^*}{s - j\omega_0} \\ \Rightarrow y(t) &= \underbrace{\alpha_1 e^{p_1 t} + \cdots + \alpha_n e^{p_n t}}_{\text{natural}} + \underbrace{A_y \sin(\omega_0 + \phi)}_{\text{forced}} \end{aligned}$$

Forced response has **same** frequency, **different** amplitude and phase.

# Response to Sinusoidal Input

*Generalization (contd.)*

For a system  $P(s)$  and input

$$u(t) = A_u \sin(\omega_0 t),$$

forced response is

$$y(t) = A_u \textcolor{red}{M} \sin(\omega_0 t + \textcolor{red}{\phi}),$$

where

$$M(\omega_0) = |P(s)|_{s=j\omega_0} = |P(j\omega_0)|, \text{ magnitude}$$

$$\phi(\omega_0) = \angle P(j\omega_0) \text{ phase}$$

In polar form

$$P(j\omega_0) = M e^{j\phi}.$$

# Fourier Analysis

# Fourier Series Expansion

Given a signal  $y(t)$  with periodicity  $T$ ,

$$y(t) = \frac{a_0}{2} + \sum_{n=1,2,\dots} a_n \cos\left(\frac{2\pi nt}{T}\right) + b_n \sin\left(\frac{2\pi nt}{T}\right)$$

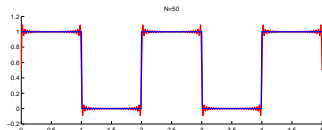
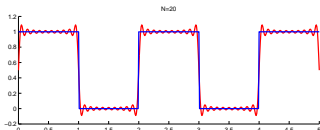
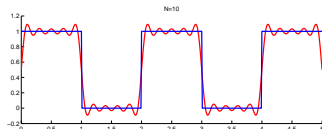
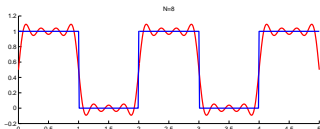
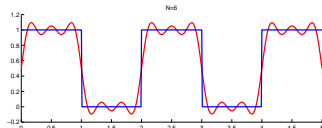
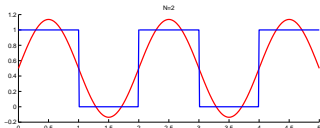
$$a_0 = \frac{2}{T} \int_0^T y(t) dt$$

$$a_n = \frac{2}{T} \int_0^T y(t) \cos\left(\frac{2\pi nt}{T}\right) dt$$

$$b_n = \frac{2}{T} \int_0^T y(t) \sin\left(\frac{2\pi nt}{T}\right) dt$$

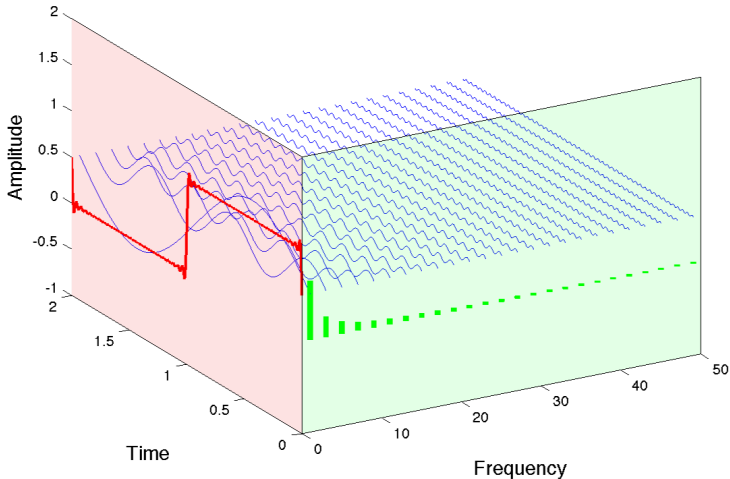
# Fourier Series Expansion

*Approximation of step function*



# Fourier Transform

*Step function*

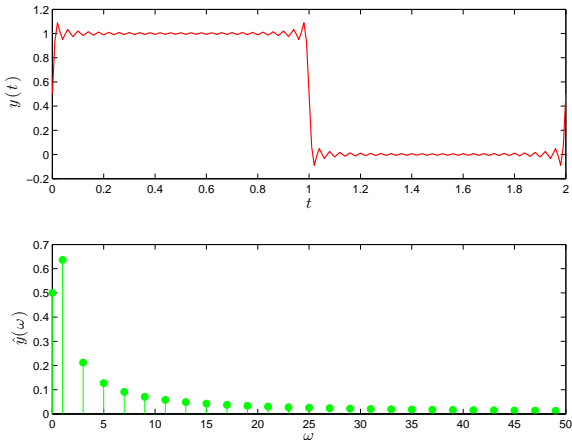


Fourier transform reveals the frequency content of a signal



# Fourier Transform

*Step function – frequency content*



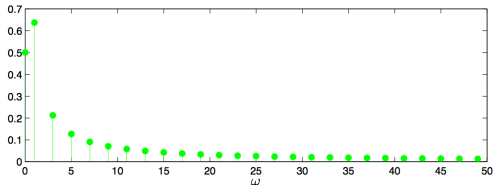
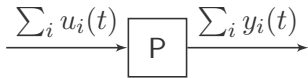
# Signals & Systems

## Input Output

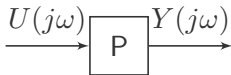


## Fourier Series Expansion

superposition principle



## Fourier Transform



$$u_i(t) = a_i \sin(\omega_i t)$$

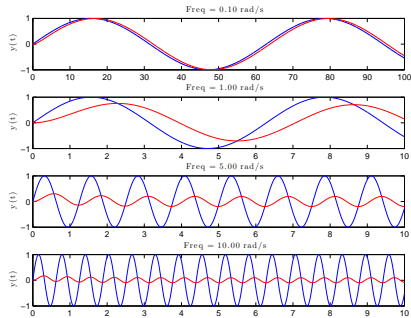
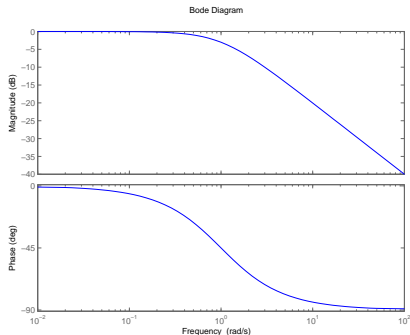
$$y_{i\text{forced}}(t) = a_i M \sin(\omega_i t + \phi)$$

$$Y(j\omega) = P(j\omega)U(j\omega)$$

Suffices to study  $P(j\omega)$   $|P(j\omega)|$ ,  $\angle P(j\omega)$

# Bode Plot

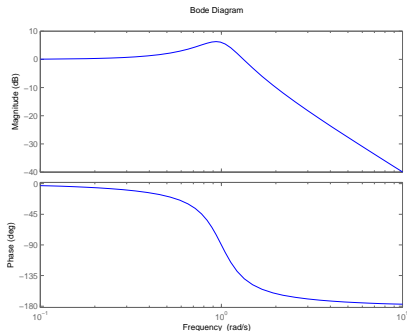
# First Order System



- $P(s) = 1/(s + 1)$
- loglog scale
- $\text{dB} = 10 \log_{10}(\cdot)$
- $20\text{dB} = 10 \log_{10}(100/1)$

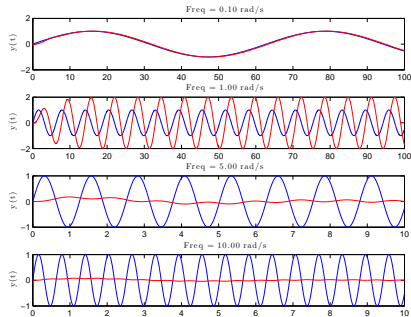
- $u(t) = A \sin(\omega_0 t)$
- $y_{\text{forced}}(t) = A M \sin(\omega_0 t + \phi)$

# Second Order System



■  $P(s) = 1/(s^2 + 0.5s + 1)$

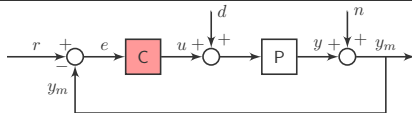
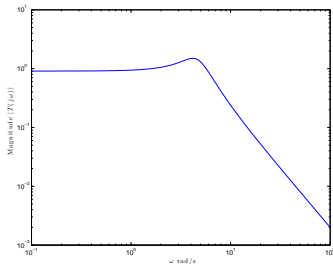
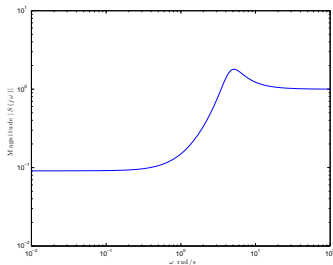
■  $\omega_n = 1 \text{ rad/s}$



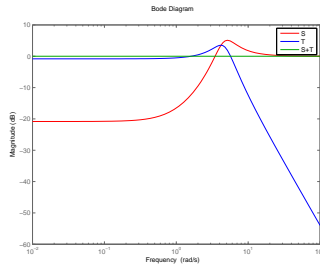
■  $u(t) = A \sin(\omega_0 t)$

■  $y_{\text{forced}}(t) = A \mathbf{M} \sin(\omega_0 t + \phi)$

$$S(j\omega) + T(j\omega) = 1$$

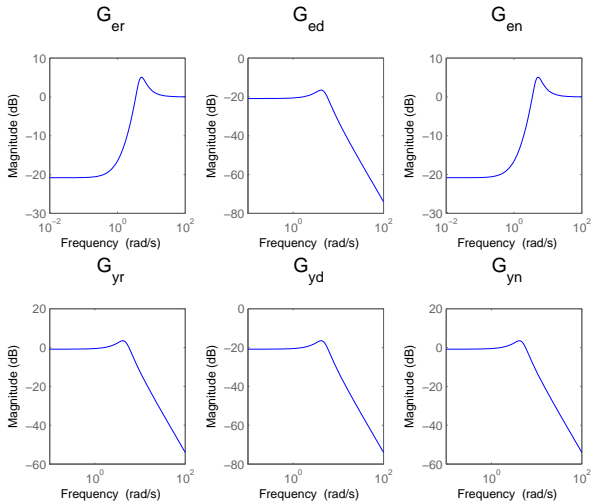


- $P(s) = \frac{1}{(s+1)(s/2+1)}$
- $C(s) = 10$
- $S = G_{er} = \frac{1}{1+PC} = \frac{1}{1+10P}$
- $T = G_{yr} = \frac{PC}{1+PC} = \frac{10P}{1+10P}$



# All transfer functions

*With proportional controller*



# **Controller Design Considerations**



# Design Using Bode Plot of $P(j\omega)C(j\omega)$

## Loop Shaping

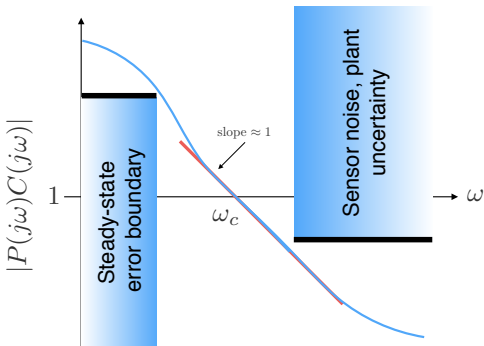
Develop conditions on the Bode plot of the open loop transfer function

- Sensitivity  $\frac{1}{1+PC}$
- Steady-state errors: slope and magnitude at  $\lim_{\omega} \rightarrow 0$
- Robust to sensor noise
- Disturbance rejection
- Controller roll off  $\implies$  not excite high-frequency modes of plant
- Robust to plant uncertainty

Look at Bode plot of  $L(j\omega) := P(j\omega)C(j\omega)$

# Frequency Domain Specifications

*Constraints on the shape of  $L(j\omega)$*

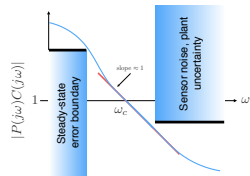
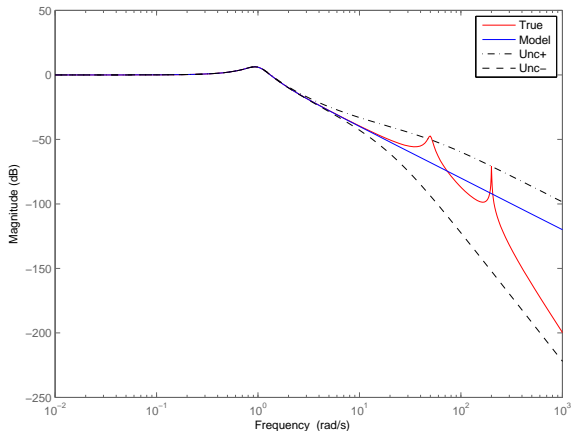


- Choose  $C(j\omega)$  to ensure  $|L(j\omega)|$  does not violate the constraints
- Slope  $\approx -1$  at  $\omega_c$  ensures  $PM \approx 90^\circ$   
stable if  $PM > 0 \implies \angle PC > -180^\circ$

# Plant Uncertainty

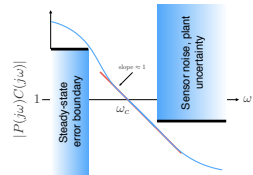
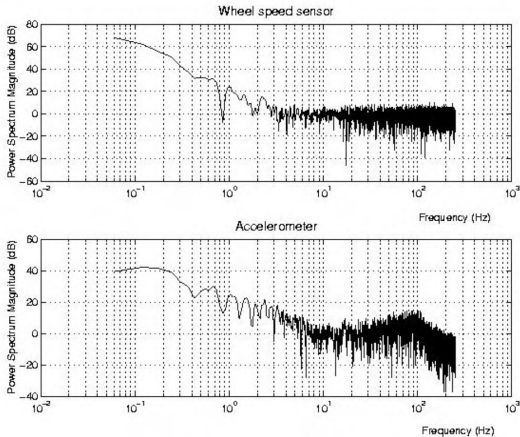
$$P(j\omega) = P_0(j\omega)(1 + \Delta P(j\omega))$$

Bode Diagram



# Sensor Characteristics

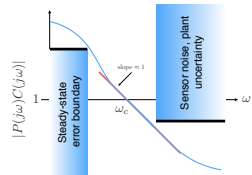
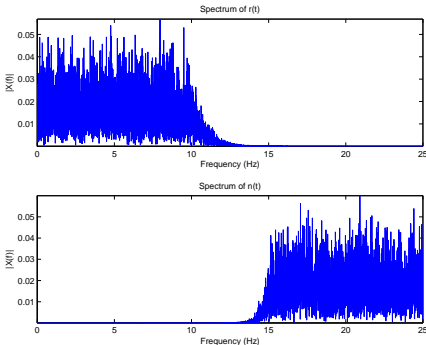
## Noise spectrum



$$G_{yn} = -\frac{PC}{1 + PC}$$

# Reference Tracking

*Bandlimited* else conflicts with noise rejection

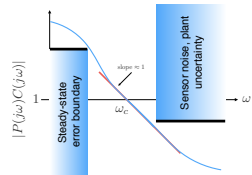
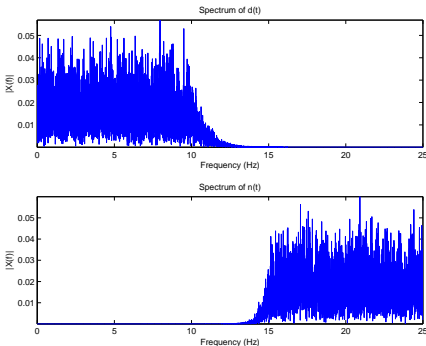


$$G_{yr} = \frac{PC}{1 + PC}$$

$$G_{yn} = -\frac{PC}{1 + PC}$$

# Disturbance Rejection

*Bandlimited else conflicts with noise rejection*



$$G_{yd} = \frac{P}{1 + PC}$$

$$G_{yn} = -\frac{PC}{1 + PC}$$