Control Theory State feedback

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Feedback control

Consider the system

$$\dot{x} = A\underline{x} + B\mathbf{u} \qquad \mathbf{X} \in \mathbb{R}^{h}$$

$$u \in \mathbb{R}^{m}$$

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- We have conditions for controllability and observability
- ▶ Q: How to design a controller to make the system behave the way we want?

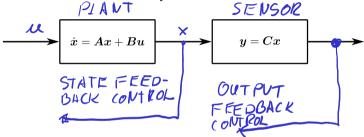
Assume the con observe the

The only way is to give feedback.

$$u = \varphi(x,X)$$

Types of feedback

We will look at linear control of linear systems



- ► This video: state-feedback controller
- Next video: output-feedback controller

Suppose that $\underline{x}^* = \underline{0}$ is an unstable equilibrium of the system

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u} \tag{1}$$

for u = 0.

Now choose N×1

$$\underline{ \mathbf{u}(t) = -\mathbf{K} \mathbf{x}(t) }$$

Toward pendulum

Pole placement control (2)

▶ Substitute (2) into (1) to get the closed-loop system

$$\overset{\dot{x} = Ax}{\stackrel{\cdot}{\times} = (A - B | K) \times} \Longrightarrow$$

▶ If we choose matrix **K** such that:

then we ensure that the equilibrium $x^* = 0$ is assymptotically stable.

▶ We will see: if the system is controllable, we can place the eigenvalues (poles) anywhere.

Pole placement control (3)

Question: How to tune matrix **K**? For example

- 1. Direct inspection A=LA-BK
- Using Ackermann's formula (MATLAB 'place')

Inverted pendulum (linearised)

Control
$$\dot{x} = \begin{pmatrix} 0 & 1 \\ \kappa & 0 \end{pmatrix} + \begin{pmatrix} 0 \\ 1 \end{pmatrix} U$$

$$M = \mathcal{U} \qquad \lambda^2 - d = 0 \qquad \lambda_{1,2} = \pm \sqrt{\kappa} \qquad \lambda^2 - tr(A)\lambda + det(A) = 0$$

$$\mathcal{U} = -K \times_1 \text{ where } K = (k_1 \ k_2)$$

$$\mathcal{A} = A - BK = \begin{pmatrix} 0 & 1 \\ \lambda & 0 \end{pmatrix} - \begin{pmatrix} 0 \\ 1 \end{pmatrix} \begin{pmatrix} k_1 \ k_2 \end{pmatrix} \qquad \lambda_{1,2} = -\frac{k_2 + \sqrt{k_1 - k_1}}{2} + \sqrt{k_1 - k_2}$$

$$= \begin{pmatrix} 0 & 1 \\ k_1 & k_2 \end{pmatrix} \qquad \qquad k_1 - \lambda \neq 0 = -\sqrt{k_1 + k_2}$$

$$= \begin{pmatrix} 0 & 1 \\ k_2 - k_1 - k_2 \end{pmatrix} \qquad \qquad \text{For negative}$$

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Inverted pendulum (direct inspection) (1)

(hoosings
$$k_17d_1k_270$$

$$= \sum_{\alpha \neq 0} x^{\alpha} = 0 \text{ is}$$

$$\alpha \neq 0 \text{$$

Ackermann's formula (1)

- Direct inspection cannot be automated, not viable for large systems
- ► For single-input, single-output (SISO) systems
- Using Ackermann's formula (MATLAB 'place')

Ackermann's formula (1)

We aim to tune the controller

$$u = -Kx$$

such that the characteristic polynomial of $A_{\frac{1}{2}}BK$ is

$$p(\lambda) = a_0 + a_1 \lambda + a_2 \lambda^2 + \dots + a_{n-1} \lambda^{n-1} + \lambda^n$$

Theorem

Row vector K is given by

$$\rho(A) = a_0 I + a_1 A + a_2 A^2 + \cdots + a_{n-1} A^{n-1} + A^n$$

Row
$$K = (0 \ 0 \cdots 0 \ 1) \ W_r^{-1} p(A)$$
.

Reacthability matrix

Wikipedia has a nice proof using the Cayley-

Hamilton theorem

Inverted pendulum (Ackermann's formula) (1)

$$A = \begin{pmatrix} 0 & 1 \\ 0 & 0 \end{pmatrix}$$

$$B = \begin{pmatrix} 0 \\ 1 \end{pmatrix}$$

$$M - M \quad \lambda_{1} = -1 \quad \lambda_{2} = -2$$

$$M = -K \times$$

$$A - BK$$

$$K = \begin{bmatrix} 0 & 1 \end{bmatrix} \quad W_{1}^{-1} \quad p(A)$$

$$W_{r} = \begin{bmatrix} B & AB \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix}$$

$$P(\lambda) = (\lambda + 1)(\lambda + 2) = \lambda^{2} + 3\lambda + 2$$

$$= A^{2} + 3A + 2I$$

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$$= \begin{pmatrix} 0 & 1 \\ \lambda & 0 \end{pmatrix} \begin{pmatrix} 0 & 1 \\ \lambda & 0 \end{pmatrix} + \begin{pmatrix} 0 & 3 \\ 2\lambda & 0 \end{pmatrix} + \begin{pmatrix} 2 & 0 \\ 0 & 2 \end{pmatrix} = \begin{pmatrix} \lambda + 2 & 3 \\ 3\lambda & \lambda + 2 \end{pmatrix}$$

$$= \begin{pmatrix} 0 & 1 \\ \lambda & 0 \end{pmatrix} \begin{pmatrix} 0 & 1 \\ \lambda & 0 \end{pmatrix} + \begin{pmatrix} 0 & 3 \\ 2\lambda & 0 \end{pmatrix} + \begin{pmatrix} 2 & 0 \\ 0 & 2 \end{pmatrix} = \begin{pmatrix} \lambda + 2 & 3 \\ 3\lambda & \lambda + 2 \end{pmatrix}$$

Inverted pendulum (Ackermann's formula) (2)
$$k = (0 \ 1) \begin{pmatrix} 0 \ 1 \end{pmatrix} \begin{pmatrix} 2 \ 3 \ 2 \end{pmatrix} \begin{pmatrix} 2 \ 3 \ 2 \end{pmatrix} \begin{pmatrix} 2 \ 2 \ 2 \end{pmatrix}$$

$$= (2 \ 2) \begin{pmatrix} 2 \ 2 \ 2 \end{pmatrix} \begin{pmatrix} 2 \ 2 \ 2 \end{pmatrix} \begin{pmatrix} 2 \ 2 \ 2 \end{pmatrix}$$

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$$= 2 \begin{pmatrix} 2 \ 2 \ 2 \end{pmatrix} \begin{pmatrix} 2 \ 2 \end{pmatrix} \begin{pmatrix} 2 \ 2 \ 2 \end{pmatrix} \begin{pmatrix} 2$$

assymptotic stability $\lambda_1 = -1 \quad \lambda_2 = -2$ $\mathcal{M} = -K \times = -(L+23) \begin{pmatrix} t_1 \\ t_2 \end{pmatrix} = -(2+L) t_1 - 3t_2$ STATE FEEDBACK