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SPH Modelling of Water Waves

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Abstract: Advances in computational power have permitted the use of Lagrangian particle methods to model fluids, particularly separated flows and free surface flows with splashing. Here we present a smoothed particle hydrodynamics (SPH) approach to a numerical wave tank, capable of studying several types of waves impinging on structures and walls.

INTRODUCTION

Computer modeling of the details of wave breaking and the turbulence in the surf zone can only be done by intricate computational models, including such techniques as Direct Numerical Simulation (DNS) of the Navier-Stokes equations, Large Eddy Simulation (LES), and Volume of Fluid (VOF). All of these methods are Eulerian and have complicated algorithms to determine the instantaneous location of the free surface. The situation becomes more difficult when splashing and air entrainment becomes important.

Recently, increases in computational power have led to the use of Lagrangian methods and, in particular, particle methods, which do not require any special techniques at the free surface or for rotational flows. For these methods, wave nonlinearities and wave breaking are naturally captured.

There are a variety of particle methods that have been developed—all involving applying Newton's Second Law to particles that represent large parcels of water molecules. One such method is Smoothed Particle Hydrodynamics (SPH), developed primarily by Lucy (1977) and Monaghan and colleagues over the last 30 years for astrophysics. More recently Monaghan (1994) has begun applying the technique to 2D free surface flows, including the dam break problem, a bore, and waves shoaling on a beach. Monaghan and Kos (1999) have examined the run-up of a solitary wave (in

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the context of tsunami inundation) and, in follow-up work, they (Monaghan and Oka 2000) have also looked in detail at the flow around the wavemaker in Scott Russell's original experimental apparatus for studying solitary waves in the laboratory.

A similar particle technique has been used by Koshizuka, Tomako and Oka (1998), Koshizuka, Nobe and Oka (1998), and Gotoh and Sakai (2000) to examine the runup of waves on a beach and the flow as the wave hits a seawall. Their technique is called the Moving Particle Semi-implicit (MPS) method. The major difference between the method and the SPH is that they solve the Poisson equation for the fluid pressure while in SPH an equation of state is used for the pressure—that is, the fluid is treated as slightly compressible.

Here we discuss the development of a 2-D numerical wave tank utilizing SPH to study large amplitude waves and wave breaking.

METHODOLOGY

The numerical development of SPH schemes is based on a particle representation of field quantities. Particle representations are generally based on an *approximate* identity, which is typically expressed as:

$$\zeta(r) \approx \int_V \zeta(r') W_\delta(r-r') dr' \quad (1)$$

Here, $\zeta(r)$ is a generic field quantity, r is the position vector, δ is a numerical parameter called the core radius, comparable to the particle size, while W_δ is a rapidly-decaying, radial smoothing function. The dependence of the smoothing function on the core radius is given by:

$$W_\delta(x) = \frac{1}{\delta^n} W\left(\frac{|x|}{\delta}\right) \quad (2)$$

where n is the dimension of the space. Eq. 1 is an approximate identity since it converges to the Dirac delta function, $\delta(r)$ in the sense of distributions as the radius δ decreases to zero. In this limit one recovers the original function ζ and the equality in Eq. 1 is obtained.

The particle representation of $\zeta(r)$ is then obtained by approximating the integral in Eq. 1 using numerical quadrature with a cell size $h \ll \delta$, resulting in:

$$\zeta(r) = \sum_{i=1}^N \zeta_i h^n W_\delta(r-r_i) \quad (3)$$

where N is the total number of particles, ζ_i is the value of $\zeta(r)$ for the i -th particle, r_i is the position of the i -th particle. In application, h is about 20% larger than the radius of the computational particle radius.

Application of the SPH to free surface flows is based on using this particle

representation scheme to represent the density and velocity fields. This is accomplished using a distribution of Lagrangian particles that are described in terms of their positions r_i , densities ρ_i , masses m_i , and velocities v_i . The evolution of the solution is then described in terms of a coupled system of ODEs that express the mass and momentum conservation laws. As discussed in Gingold and Monaghan (1977), Monaghan (1982), and Monaghan (1992), this system can be expressed as:

$$\begin{aligned} \frac{dm_i}{dt} &= 0 \\ \frac{dr_i}{dt} &= v_i \\ \frac{d\rho_i}{dt} &= \sum_{j=1}^N m_j (v_i - v_j) \cdot \nabla_i W_\delta(r_i - r_j) \\ \frac{dv_i}{dt} &= - \sum_{j=1}^N m_j \left(\frac{\rho_i + \rho_j}{\rho_i^2 \rho_j^2} \right) \nabla_i W_\delta(r_i - r_j) + F_i \end{aligned} \quad (4)$$

where t is time, ρ_{ij} is the viscosity term, and F_i is a body force term (including gravity). The quantity p_i is the pressure at the i th particle. The last two equations represent the conservation of mass and the conservation of momentum equations in SPH format. These equations show the advantage of the kernel smoothing functions W as derivatives of the kernel function. The viscosity term is important for viscous flows (see, e.g., Cleary and Monaghan, 1999); however, in the examples below, it has been neglected.

For nearly incompressible free surface flows, the fluid pressure can be obtained from an equation of state (Batchelor, 1967; Monaghan, 1994):

$$p = B \left[\left(\frac{\rho}{\rho_0} \right)^\gamma - 1 \right] \quad (5)$$

where B and γ are constants, and ρ_0 is the density at the free surface; here we use $B = 200 \rho g H / \gamma$ and $\rho_0 = 1000 \text{ kg/m}^3$. Monaghan uses $B = 200 \rho g H / \gamma$, where H is a representative depth. This modified equation of state actually gives a sound speed $c = \sqrt{dp/d\rho}$ that is much slower than the actual speed of sound in the water. This is important as the numerical time stepping is determined by the apparent speed of sound in the fluid.

The kernel function used here is due to Monaghan and Lattanzio (1985):

$$W(r, h) = \frac{10}{7\pi h^2} \begin{pmatrix} 1 - \frac{3}{2} \left(\frac{r}{h}\right)^2 + \frac{3}{4} \left(\frac{r}{h}\right)^3; 0 \leq \left(\frac{r}{h}\right) \leq 1 \\ \frac{1}{4} \left(2 - \left(\frac{r}{h}\right)\right)^3; 1 \leq \left(\frac{r}{h}\right) \leq 2 \\ 0; 2 < \left(\frac{r}{h}\right) \end{pmatrix} \quad (6)$$

This spline kernel has continuous first and second derivatives. Notice that for distances greater than $2h$, the kernel is zero. This means that the kernel has compact support and the interactions between particles are only important in the neighborhood of a particle. These leads to immense computational time savings in computing the sum in Eqn. 4. Further, Eqn 4 shows that for isolated particles, the equation of motion is forced simply by gravity.

NUMERICAL SIMULATION

The time stepping of the governing equations is done with a predictor-corrector scheme utilized by Monaghan (1994). For an equation of the form:

$$\frac{dv}{dt} = -\gamma v + F \quad (7)$$

The predictor step for a midpoint calculation is:

$$v^{n+1/2} = \frac{v^n + 0.5 \Delta t F^n}{1 + 0.5 \gamma \Delta t} \quad (8)$$

The corrector step gives

$$v^{n+1/2} = \frac{v^n + 0.5 \Delta t F^{n+1/2}}{1 + 0.5 \gamma \Delta t} \quad (9)$$

and the final calculation for time level n is

$$v^{n+1} = 2v^{n+1/2} - v^n \quad (10)$$

Other time-stepping schemes are possible (e.g. Fulk and Quinn, 1995)

Time Step size:

Monaghan (1992) provides several criteria for choosing the step size; one of which is that the time step is chosen to be less than the travel time of the speed of sound in the medium to traverse the particle size h . Since the speed of sound is much faster than the particle speeds, this gives small time steps. For most problems, resolution requires the use of many particles, this means that the size of h is small, decreasing with the number of particles. So, the higher the resolution required, the smaller the time step.

time step and the longer computation time. In our calculations, time steps are in the range of 10^{-4} s.

Neighbors list

It is crucial that the compact support of the kernel be recognized in the numerical model and link (or neighbor) lists be used. This is essentially a list maintained during the computations that keeps track of all the particles that are within $2h$ of each other. The procedure for doing this is to divide the computational domain into square cells of size $2h$. A linear array is maintained that refers to all the particles in a given square cell. Each particle, the numerical calculations in Eqn. 4 are carried out only over the cells of particles located in the 9 cells centered near the particle position. This leads to a large saving in computational time.

The symmetry in the kernels W_{ij} can be further exploited to effectively halve the number of computations.

Boundary conditions

Monaghan and colleagues have used several boundary conditions either to ensure slip at the boundary or to have a no-slip condition. The first technique was to model the fluid-Jones forces between stationary boundary particles and the fluid (Monaghan, 1994) but these radial forces produced a bumpy bottom with fluid particles bouncing off each boundary particle. Monaghan and Kos (1999) developed a second condition to smooth the boundary repulsive force such that it was always acting normal to the boundary, regardless of the fluid particle position. Here we take a third approach, which is to establish several rows of boundary particles, which enter into the force calculations but are not allowed to move. By experimentation we found that two rows of stationary particles on all boundaries was sufficient to contain the fluid. This technique provides several advantages: there is no additional programming needed to treat the boundaries and it is far easier to specify irregular boundaries by placing multiple rows of particles than to specify analytic boundaries as in Monaghan and Kos (1999).

In application, this boundary condition provides a check on the time step. Should the particles fail after a number of iterations, it is likely due to particles penetrating the boundary—implying a time step that is too large.

NUMERICAL WAVETANK (2-D)

The SPH model that we have developed can be considered a two-dimensional wave tank. We have a variable left-lateral boundary consisting of either a fixed wall or a piston, producing a variety of wave forms, including a solitary wave or a periodic wave train. The periodic wave is created by moving the wall back and forth with a constant velocity in piston or flap motion. The solitary wave is created by Goring's method (1978). The other end of the tank is an arbitrary slope beach. In between is a horizontal bottom. Objects can be placed anywhere within the tank to represent obstacles and the forces on these objects can be obtained from the fluid pressure.

RESULTS

Prior to the development of the numerical wave tank, a few tests were carried out to ensure that we could replicate Mongaghan's (1994) results. These tests included the dam break problem (along with a triangular jump on the bed) and wave breaking on a slope. In addition, we performed a variety of test cases, such as dropping columns of fluid and examining the sloshing of a closed rectangular tank.

For the dropping fluid column, several cases were carried out. First, discrete columns of liquid were dropped onto the bottom boundary to determine the number of fixed rows of particles necessary in the boundary. Interestingly, when the number of particles comprising the column is large, the behavior of the falling fluid column impinging on the bottom is realistic; however, when the number of particles is small, including the case of a single particle, the falling particles bounce elastically from the bottom boundary! This is the continuum hypothesis at work.

Containers onboard ships can be excited into resonant sloshing modes by wave action. Faltinsen *et al.* (2000) develop an analytical eigenfunction approach to nonlinear sloshing of rectangular containers. Their experiments included such things as amplitude sloshing that the fluid impinged on the roof of the container; however, their analytical model did not include the presence of the roof. Figure 1 shows our approach to the problem. The forcing of the tank is accomplished, not by moving the tank, but by fixing the coordinate system with the tank and modifying the acceleration of gravity to either include horizontal accelerations associated with a periodic sliding of the tank horizontally or by rotating the gravitational vector in a pendulum like motion which mimics the rolling of a ship.

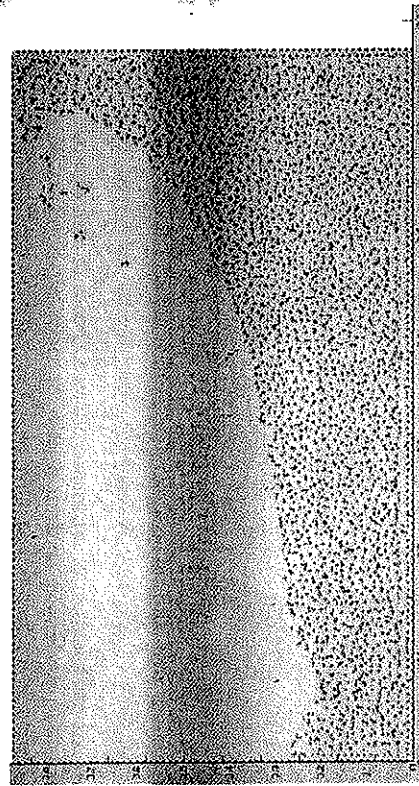


Figure 1 Nonlinear sloshing in a tank.

the breaking of a solitary wave is shown in Figure 2. This wave was initialized in the basin using the velocities and displacements of a solitary wave; however, its initial wave height was greater than the stable wave height for that depth. Within a few seconds, the initially symmetric wave form deformed into the plunging breaker shown in the figure. Further computations, after plunger touch-down shows the development of a turbulent bore.



Figure 2 Breaking solitary wave

The last two figures (3 and 4) show the propagation of a solitary wave generated by Li's wavemaker approach. This wave propagates on a flat bottom section prior to impinging at a sloping beach (1/19.85). At the end of the tank, the wave impacts and splashes up on the end wall, with splashing. This models the laboratory experiments carried out by Li, Raichlen and Lee (2000), who were examining the splash-up at the

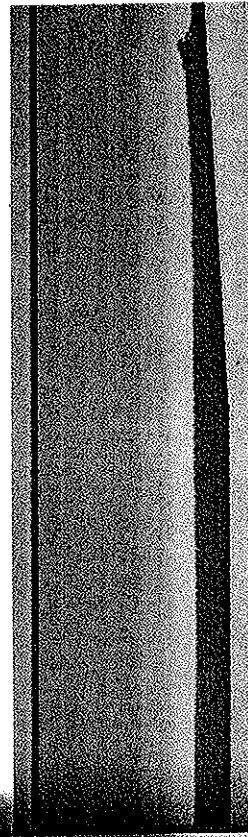


Figure 3 Solitary wave on beach (right side)

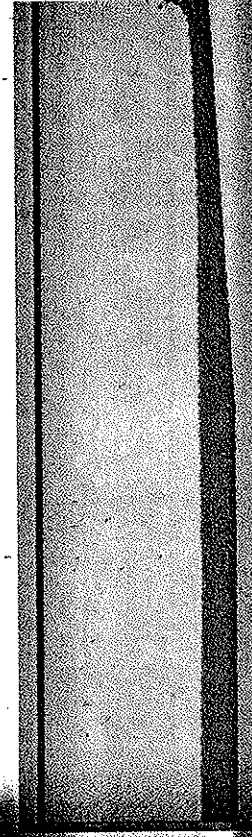


Figure 4 Splash-up of solitary wave on a wall (at right)

CONCLUSIONS

Particle methods are useful in predicting wave motions, particularly for the cases where conventional models fail, such as when wave splash occurs or within turbulent bores after wave breaking.

There are a variety of issues still to be addressed on the topic of SPH modeling. For appropriate sound speeds, fluid viscosity, and particle number densities are all issues to be addressed by numerical experimentation. Quantitative comparisons with a wide range of laboratory experiments needs to be completed.

The long computational times, associated with the large numbers of particle needed for high resolution, dictate the use of parallel computing and large computing clusters due to the immense amount of computation required. However, it is likely in the future that computation times will not be an impediment.

The numerical wave tank provides a useful tool for testing when two-dimensional studies are sufficient.

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