

Experiment #01 - STRESS DISTRIBUTION IN A TOWER BRIDGE

Aim: Calculating and visualizing the eigenvalues of stress matrix for a simply supported beam.

Problem statement: Find the principal stresses for a two-dimensional simply supported beam by finding the eigenvalues of the stress matrix with variable components.

Mathematical background: The principal stresses are eigenvalues of the stress matrix.

The 2×2 stress matrix is given by:

$$S = \begin{bmatrix} \sigma_x & \tau_{xy} \\ \tau_{yx} & \sigma_y \end{bmatrix}$$

The principal stress at the point will be the eigenvalues of the stress matrix S .

Properties of eigenvalues and eigenvectors:

1. The sum of the eigenvalues of a matrix equals the trace of the matrix.
2. A matrix is singular if and only if it has a zero eigenvalue.
3. The eigenvalues of an upper (or lower) triangular matrix are the elements on the main diagonal.

MATLAB Syntax

<code>p = poly(A)</code>	Where A is an $n \times n$ matrix returns an $n+1$ element row vector whose elements are the coefficients of the characteristic polynomial, $\det(A - \lambda I)$, which are stored in p .
<code>r = roots(p)</code>	Returns a column vector r whose elements are the roots of the polynomial p .
<code>[V,D] = eig(A)</code>	D = diagonal matrix with eigenvalues on its diagonal; V = modal matrix whose columns are the corresponding eigenvectors.
<code>eye(n)</code>	Returns an $n \times n$ identity matrix.

Example #1:

```
A =  
  
    3    0   -1  
    0    1    0  
    2    0    0  
  
>> poly(A)  
  
ans =  
  
    1   -4    5   -2  
  
>> roots(poly(A))  
  
ans =  
  
    2.0000 + 0.0000i  
    1.0000 + 0.0000i  
    1.0000 - 0.0000i  
  
>> eig(A)  
  
ans =  
  
    2  
    1  
    1  
  
>> [V,D] = eig(A)  
  
V =  
  
    0.7071    0.4472    0  
         0         0    1.0000  
    0.7071    0.8944    0  
  
D =  
  
    2    0    0  
    0    1    0  
    0    0    1  
  
>> eig(trace(A))  
  
ans =  
  
    4  
  
>> det(A)  
  
ans =  
  
    2  
  
>> [V,D] = eig(inv(A))
```

```
V =  
-0.7071  -0.4472  0  
0         0      1.0000  
-0.7071  -0.8944  0
```

```
D =  
0.5000  0  0  
0      1.0000  0  
0       0  1.0000
```

```
>> [V,D] = eig(A')
```

```
V =  
0.8944  -0.7071  0  
0         0      1.0000  
-0.4472  0.7071  0
```

```
D =  
2  0  0  
0  1  0  
0  0  1
```

```
>> I = eye(3)
```

```
I =  
1  0  0  
0  1  0  
0  0  1
```

```
>> B = A^2 + 3*A + 2*I
```

```
B =  
18  0  -6  
0   6  0  
12  0  0
```

```
>> [V,D] = eig(B)
```

```
V =  
0.7071  0.4472  0  
0         0      1.0000  
0.7071  0.8944  0
```

```
D =  
12  0  0  
0   6  0  
0   0  6
```

```
>> eig(B)

ans =

    12
     6
     6

>>
```

Example #2

```
>> A = [2 0 -1; 0 2 -2; 1 -1 2]

A =

     2     0    -1
     0     2    -2
     1    -1     2

>> eig(A)

ans =

    3.0000
    2.0000
    1.0000

>> roots(poly(A))

ans =

    3.0000
    2.0000
    1.0000
```

Questions

1. Perform basics matrix tasks using MATLAB.

```
>> A = zeros(3)

A =

     0     0     0
     0     0     0
     0     0     0

>> A = [1,2,3]

A =

     1     2     3

>> B = [1;2;3]

B =

     1
     2
     3

>> C = [1 2 3; 10 20 30; 3 6 9]

C =

     1     2     3
    10    20    30
     3     6     9

>> length(C)

ans =

     3

>> D = [1 2 3; 4 5 6]

D =

     1     2     3
     4     5     6

>> length(D)
```

```
ans =  
  
      3  
  
>> size(D)  
  
ans =  
  
      2      3
```

2. Find the following for A :
- Characteristic polynomial of A .
 - Roots of characteristic polynomial of A .
 - Eigenvalues for A .
 - Eigenvectors for A .
 - Eigenvalues of A^{-1} .
 - Eigenvalues of A^T .
 - Eigenvalues of $B = A^2 + 3A + 2I$.

```
>> A = [1 2 1; 6 -1 0; -1 -2 -1]  
  
A =  
  
      1      2      1  
      6     -1      0  
     -1     -2     -1  
  
>> poly(A)  
  
ans =  
  
      1.0000      1.0000     -12.0000      0.0000  
  
>> roots(poly(A))  
  
ans =  
  
     -4.0000  
      3.0000  
      0.0000  
  
>> eig(A)  
  
ans =  
  
     -4.0000  
      3.0000  
      0.0000  
  
>> [V,D]=eig(A)
```

```
V =  
    0.4082   -0.4851   -0.0697  
   -0.8165   -0.7276   -0.4180  
   -0.4082    0.4851    0.9058  
  
D =  
   -4.0000    0    0  
    0    3.0000    0  
    0    0    0.0000  
  
>> eig(trace(A))  
  
ans =  
    -1  
  
>> det(A)  
  
ans =  
    0  
  
>> [V,D] = eig(inv(A))  
Warning: Matrix is singular to working precision.  
Error using eig  
Input to EIG must not contain NaN or Inf.  
  
>> [V,D] = eig(A')  
  
V =  
   -0.7252    0.9117    0.7071  
    0.6447    0.3419    0.0000  
    0.2417    0.2279    0.7071  
  
D =  
   -4.0000    0    0  
    0    3.0000    0  
    0    0   -0.0000  
  
>> eig(A')  
  
ans =  
   -4.0000  
    3.0000  
   -0.0000  
  
>> B = A^2 + 3*A + 2*eye(3)  
  
B =  
    17    4    3  
    18   12    6
```

```

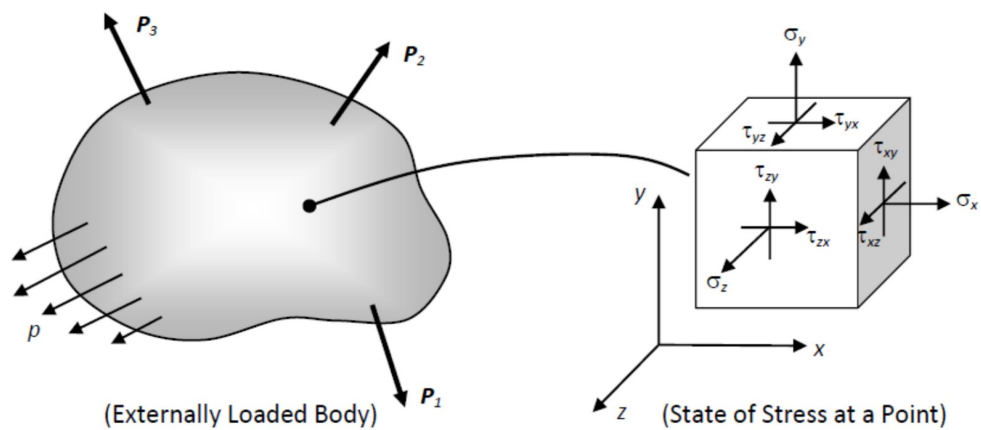
-15    -4    -1
>> [V,D] = eig(B)
V =
-0.4851    -0.4082    -0.0697
-0.7276     0.8165    -0.4180
 0.4851     0.4082     0.9058

D =
20.0000     0     0
 0     6.0000     0
 0     0     2.0000

>> eig(B)
ans =
20.0000
 6.0000
 2.0000

```

Stress Analysis



The number of components and some other transformation properties, the stress can be expressed as a 3×3 matrix:

$$\sigma = \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{bmatrix}$$

Since the shearing stresses have the equalities, $\tau_{xy} = \tau_{yx}, \tau_{yz} = \tau_{zy}, \tau_{zx} = \tau_{xz}$, the stress matrix is symmetric.

If we changed the orientation of a particular plane the normal stress component σ_x will vary.

Total surface force \bar{p} per unit volume is:

$$\bar{p} = \frac{\delta p}{\delta x} + \frac{\delta p}{\delta y} + \frac{\delta p}{\delta z}$$

\bar{p}_x, \bar{p}_y and \bar{p}_z are vectors which can be decomposed into the components perpendicular to each surface elements, i.e., normal surface σ and by giving the direction of normal stresses as index for normal stresses. The components in the plane of surface elements are called tangential stresses τ . They acquire the double index. The first position indicates to which axis the surface element is perpendicular and second position states in which direction the stress τ is pointing.

In notation, τ is pointing:

$$\bar{p}_x = \sigma_x \bar{e}_x + \tau_{xy} \bar{e}_y + \tau_{xz} \bar{e}_z$$

$$\bar{p}_y = \tau_{yx} \bar{e}_x + \sigma_y \bar{e}_y + \tau_{yz} \bar{e}_z$$

$$\bar{p}_z = \tau_{zx} \bar{e}_x + \tau_{zy} \bar{e}_y + \sigma_z \bar{e}_z$$

The normal stresses are known as principal stresses:

$$\sigma_l = \sigma_x l + \tau_{xy} m + \tau_{xz} n \dots (1)$$

$$\sigma_m = \tau_{yx} l + \sigma_y m + \tau_{yz} n \dots (2)$$

$$\sigma_n = \tau_{zx} l + \tau_{zy} m + \sigma_z n \dots (3)$$

Where, σ_l, σ_m and σ_n are components of principle stress σ .

$$\text{We know that } l^2 + m^2 + n^2 = 1 \dots (4)$$

Thus above system of homogenous equations does not admit a trivial solution. {a trivial solution would be $(l = m = n = 0)$ } as (4) is true.

Since, $\Delta = 0$, the characteristic equation is,

$$\begin{vmatrix} \sigma_x - \sigma & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y - \sigma & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z - \sigma \end{vmatrix} = 0$$

Example 1:

Q: Consider the bridge to be equivalent to a two-dimensional simple supported beam of thickness 1 meter. The bridge carries a concentrated force $P = 100\text{N}$ at the mid-point. The length of the beam is 10 times its thickness. If the individual stresses at the point $(0.5, 0.5)$ are given by $\sigma_x = -4$, $\sigma_y = 0$ and $\sigma_{xy} = -2.5$, then write the stress matrix and hence calculate the principal stresses.

A: The stress matrix at the point $(0.5, 0.5)$ will be:

```
>> S = [-4 -2.5, -2.5 0]
S =
    -4.0000    -2.5000    -2.5000         0
>> S = [-4 -2.5; -2.5 0]
S =
    -4.0000    -2.5000
    -2.5000         0
>> eig(S)
ans =
    -5.2016
     1.2016
>> [V,D] = eig(S)
V =
    -0.9013     0.4332
    -0.4332    -0.9013
D =
    -5.2016         0
         0     1.2016
```

Example 2:

Q: If the individual stresses in the beam at any point (x,y) are given by:

$$\sigma_x = \frac{-3P}{4c^3}, \sigma_y = 0 \text{ and } \tau_{xy} = \frac{-3}{4c^3}(c^2 - y^2)$$

When P denotes the force $2c$ denotes the height and $2L$ denotes the length of the beam. Write the stress matrix and hence calculate the principal stresses. Draw the stress distribution in the beam using contour plot.

A:

```
clc;
clear all;
c = 0.5; L = 5; P = 1;
x = [0:0.1:L]; y = [-c:0.01:c];
[X,Y] = meshgrid(x,y);
sx = -(3*P/(4*c^3))*(L-X).*Y;
sy = zeros(length(y),length(x));
txy = -(3/(4*c^3))*(c^2-Y.^2);
for i = 1:length(y)
    for j = 1:length(x)
        s = [sx(i,j),txy(i,j);txy(i,j),sy(i,j)]
        z = eig(s);
        s1(i,j) = z(2);
        s2(i,j) = z(1);
    end
end

figure(1)
contour(X,Y,s1,[0.01,0.05,0.1,0.5,1,3,5,7,9,11]);

figure(2)
contour(X,Y,s2,[-0.01,-0.05,-0.1,-0.5,-1,-3,-5,-7,-9,-11]);
```

Figure 1:

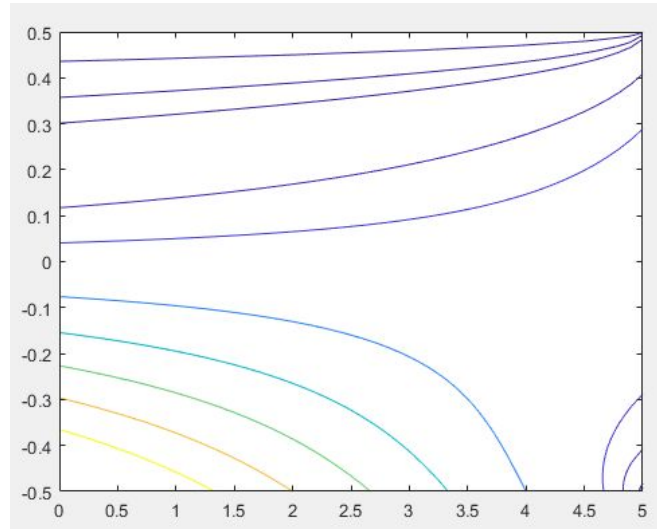
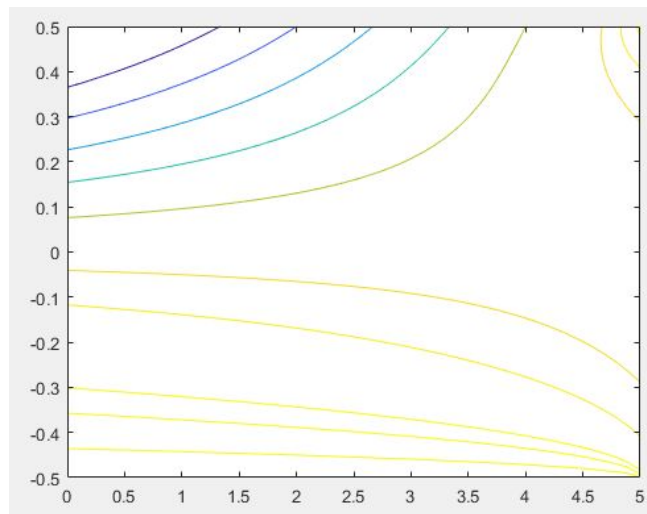


Figure 2:



Exercise - 1

Q: Create a 3 x 3 matrix:

1. Find the eigenvalues and eigenvectors.
2. Verify the properties of eigenvalues.

A:

```
>> A = [2 0 4;0 6 0;4 0 2]
```

A =

```
    2    0    4
    0    6    0
    4    0    2
```

```
>> [V,D] = eig(A)
```

V =

```
    0.7071    0.7071    0
         0         0   -1.0000
   -0.7071    0.7071    0
```

D =

```
   -2    0    0
    0    6    0
    0    0    6
```

```
>> A = [2 0 4;0 6 0;4 0 2];
>> %Eigen values of A and A transpose are the same
>> eig(A)
```

ans =

```
   -2
    6
    6
```

```
>> eig(transpose(A))
```

ans =

```
   -2
    6
    6
```

```
>> %trace of A = sum of eigenvectors
>> trace(A)
```

```
ans =  
    10  
  
>> sum(eig(A))  
ans =  
    10  
  
>> %det = product of eigenvalues of A  
>> det(A)  
ans =  
   -72  
  
>> prod(eig(A))  
ans =  
   -72  
  
>> %lamda^2 are the eigenvectors of A^2  
>> A^2  
ans =  
    20     0    16  
     0    36     0  
    16     0    20  
  
>> eig(A^2)  
ans =  
     4  
    36  
    36  
  
>> eig(A)  
ans =  
    -2  
     6  
     6  
  
>> %The diagonal elements are the eigen vectors of a diagonal matrix  
>> A = [1 0 0;0 2 0;0 0 3];  
>> A  
A =  
     1     0     0  
     0     2     0  
     0     0     3  
  
>> eig(A)
```

ans =

- 1
- 2
- 3

Attestation:

