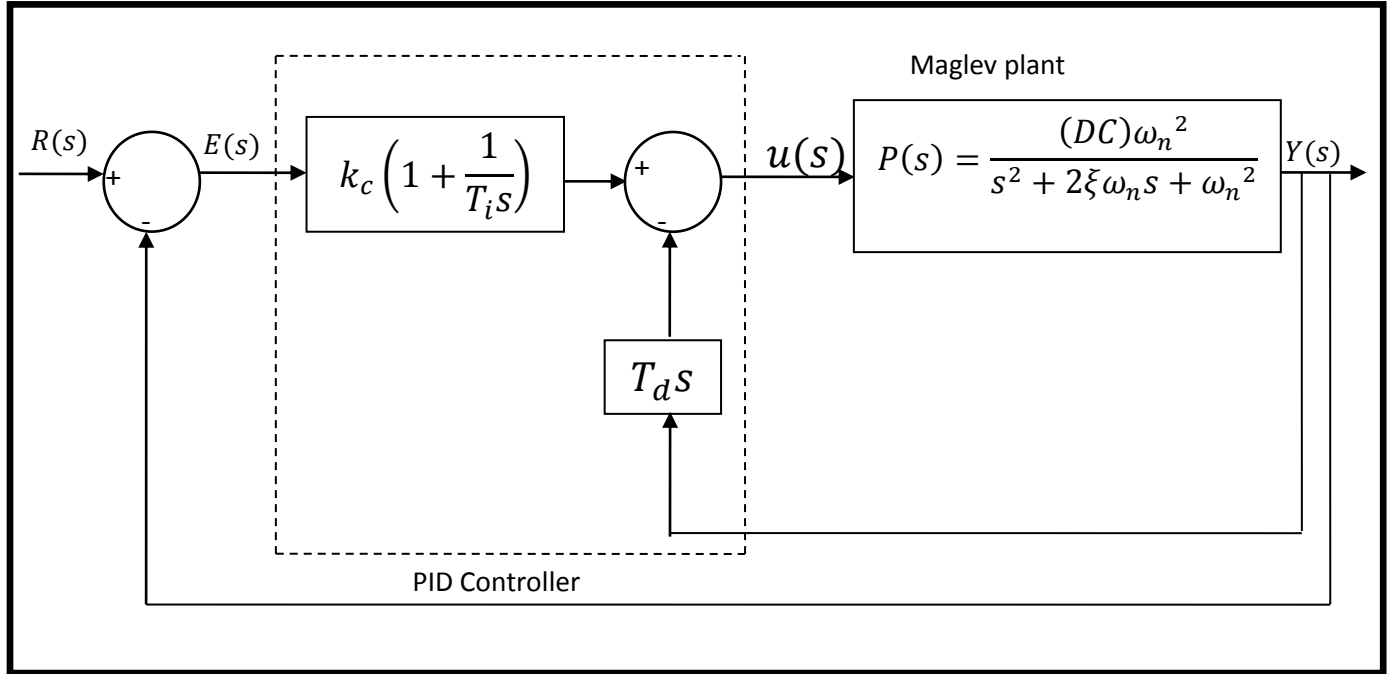


### Theory

Consider the PID controller configuration with the Maglev plant shown in Figure:



The PID controller in above figure has the derivative mode acting on the output  $Y(s)$  instead of the error  $E(s)$ . This is commonly done in industry, to prevent the “derivative kick” from happening for step changes in set point signal  $R(s)$ .

$$u(s) = k_c \left( 1 + \frac{1}{T_i s} \right) E(s) - T_d s Y(s) , \quad (1)$$

We can derive the closed loop transfer function CLTF between  $Y(s)$  and  $R(s)$  as

$$CLTF = \frac{Y(s)}{R(s)} = \frac{(DC)\omega_n^2 K_c (s + \frac{1}{T_i})}{s^3 + (2\xi\omega_n + T_d(DC)\omega_n^2)s^2 + \omega_n^2(1 + K_c(DC))s + K_c\omega_n^2(DC)/T_i} , \quad (2)$$

**Remark 1:** A word of caution in using  $T_d$ : In most PID implementation, we use the form

$$k_c \left( 1 + \frac{1}{T_i s} \right) E(s) - K_c T_d s Y(s) ,$$

so our  $T_d$  value may have to be suitably adjusted.

Let us compare the denominator of CLTF in (2) in terms of the standard second order system along with a pole  $p_{CLTF}$ . That is, let us express

$$\begin{aligned} s^3 + (2\xi\omega_n + T_d\omega_n^2(DC))s^2 + \omega_n^2(1 + K_c(DC))s + \frac{K_c\omega_n^2(DC)}{T_i} \\ = (s + p_{CLTF})(s^2 + 2\xi_{CLTF}\omega_{n_{CLTF}}s + \omega_{n_{CLTF}}^2) , \end{aligned} \quad (3)$$

Where the term  $(s + p_{CLTF})$  is chosen (or designed) to cancel the numerator term

$(s + 1/T_i)$  In equation (2) (leading to a standard second order form that is easy to use for design).

The RHS of (3) can be expressed as

$$s^3 + (2\xi_{CLTF}\omega_{n_{CLTF}} + p_{CLTF})s^2 + (\omega_{n_{CLTF}}^2 + 2\xi_{CLTF}\omega_{n_{CLTF}}p_{CLTF})s + \omega_{n_{CLTF}}^2p_{CLTF} , (4)$$

From (3) and (4)

$$\begin{aligned} s^3 + (2\xi\omega_n + T_d\omega_n^2(DC))s^2 + \omega_n^2(1 + K_c(DC))s + K_c\omega_n^2(DC)/T_i \\ = s^3 + (2\xi_{CLTF}\omega_{n_{CLTF}} + p_{CLTF})s^2 + (\omega_{n_{CLTF}}^2 + 2\xi_{CLTF}\omega_{n_{CLTF}}p_{CLTF})s + \omega_{n_{CLTF}}^2p_{CLTF} , \end{aligned} \quad (5)$$

Comparing the coefficients of corresponding powers of s on both sides gives

$$\circ \text{ Constant: } \omega_{n_{CLTF}}^2p_{CLTF} = K_c\omega_n^2(DC)/T_i , \quad (6)$$

$$\circ \text{ s term: } \omega_n^2(1 + K_c(DC)) = \omega_{n_{CLTF}}^2 + 2\xi_{CLTF}\omega_{n_{CLTF}}p_{CLTF} , \quad (7)$$

$$\circ \quad s^2 \text{term:} \quad 2\xi\omega_n + T_d(DC)\omega_n^2 = 2\xi_{CLTF}\omega_{n_{CLTF}} + p_{CLTF} , \quad (8)$$

Note that  $p_{CLTF}$  is chosen such that the pole cancels the zero at  $s=-1/T_i$ .

Thus,  $p_{CLTF} = \frac{1}{T_i} \rightarrow p_{CLTF} * T_i = 1$ .

From equations (6), (7), (8), we obtain

$$K_c = \frac{\omega_{n_{CLTF}}^2}{\omega_n^2(DC)} , \quad (9)$$

$$T_i = \frac{2\xi_{CLTF}\omega_{CLTF}}{\omega_n^2} , \quad (10)$$

$$T_d = \frac{2\xi_{CLTF}\omega_{n_{CLTF}} - 2\xi\omega_n + 1/T_i}{\omega_n^2(DC)} , \quad (11)$$

From the given PO and using the well-known relation

$$\xi_{CLTF} = \sqrt{\frac{(\ln PO)^2}{(\ln PO)^2 + \pi^2}} , \quad (12)$$

we can obtain the corresponding  $\xi_{CLTF}$ . From the given  $t_s$  and using the approximate relation

$$t_{s \approx} \frac{4}{\xi_{CLTF}\omega_{n_{CLTF}}} , \quad (13)$$

we get

$$\omega_{n_{CLTF}} = \frac{4}{(t_s)(\xi_{CLTF})} , \quad (14)$$

where  $\xi_{CLTF}$  is found from (12).

Having obtained  $\xi_{CLTF}$  from (12) and  $\omega_{n_{CLTF}}$  from (14), we can find the required  $K_c, T_i, T_d$ , using equations (9), (10), (11). The steady state error (or offset) to step input is zero, due to the presence of integral action.

**Remark 2:** *It is important to note that the relation (13) is quite approximate. See any standard book on control systems, or the references link on the main page of this site.*