

MPCC Documentation

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1 Problem Formulation

The MPCC problem as described in [LMG10] and [LDM15] is reformulated in continuous time (for the solver) as follows

$$\begin{aligned} \text{minimize} \quad & \int_0^T [\epsilon_c^{lin}(t) \quad \epsilon_l^{lin}(t)] \begin{bmatrix} Q^c & 0 \\ 0 & Ql \end{bmatrix} \begin{bmatrix} \epsilon_c^{lin}(t) \\ \epsilon_l^{lin}(t) \end{bmatrix} - Q_\theta \dot{\theta}(t) + u^T(t) R u(t) dt \\ \text{subject to} \quad & \dot{x} = f(x, u, \Phi) \\ & b_{lower} \preceq x(t) \preceq b_{upper} \\ & l_{lower} \preceq u(t) \preceq l_{upper} \\ & h(x, \Phi) \leq 0 \end{aligned} \tag{1}$$

given the system dynamics f and the arclength parametrization of the contour (in our case the track) Φ . Here $x(t)$ denotes the system state, $u(t)$ the inputs to the system, b the box constraints on the state, l the box constraints on the input and h captures the track boundary constraints.

The state of the system is augmented with the advancing parameter θ

$$x = \begin{bmatrix} x_{model} \\ \theta \end{bmatrix} \tag{2}$$

and the virtual input $\dot{\theta}$ is appended to the inputs from the original system dynamics.

$$u = \begin{bmatrix} u_{model} \\ \dot{\theta} \end{bmatrix} \tag{3}$$

The track boundary constraint is realized as a convex disk constraint.

$$h(x, \Phi) = (x - x_t^{lin}(\theta))^2 + (y - y_t^{lin}(\theta))^2 - r_\Phi(\hat{\theta})^2 \tag{4}$$

Here $r_\Phi(\hat{\theta})$ is the half-width of the track at the last predicted arc length.

The linearized contouring error ϵ_c^{lin} and lag error are computed as shown in fig. 1. To make the problem realtime feasible the are approximated by linearizing the both them and the track around the previous solution $\hat{\theta}$ as:

$$\Phi(\theta) = \begin{bmatrix} x_t(\theta) \\ y_t(\theta) \end{bmatrix} \approx \Phi(\hat{\theta}) + \partial_{\theta}\Phi(\hat{\theta})(\theta - \hat{\theta}) \quad (5)$$

$$\Rightarrow \Phi^{lin}(\theta) = \begin{bmatrix} x_t(\hat{\theta}) + \cos(\phi(\hat{\theta}))(\theta - \hat{\theta}) \\ y_t(\hat{\theta}) + \sin(\phi(\hat{\theta}))(\theta - \hat{\theta}) \end{bmatrix} \quad (6)$$

this allows us to compute the errors

$$x_t^{lin}(\theta) = x(t) + \epsilon_l^{lin} \cos(\phi(\hat{\theta})) + \epsilon_c^{lin} \sin(\phi(\hat{\theta})) \quad (7)$$

$$y_t^{lin}(\theta) = y(t) + \epsilon_l^{lin} \sin(\phi(\hat{\theta})) - \epsilon_c^{lin} \cos(\phi(\hat{\theta})) \quad (8)$$

$$\Leftrightarrow \begin{cases} \epsilon_l^{lin} = \cos(\phi(\hat{\theta}))(x_t^{lin}(\theta) - x(t)) + \sin(\phi(\hat{\theta}))(y_t^{lin}(\theta) - y(t)) \\ \epsilon_c^{lin} = \sin(\phi(\hat{\theta}))(x_t^{lin}(\theta) - x(t)) - \cos(\phi(\hat{\theta}))(y_t^{lin}(\theta) - y(t)) \end{cases} \quad (9)$$

These approximations are especially good if $\epsilon_l \approx 0$ and $\hat{\theta} - \theta \approx 0$. In practice the first can be incentivized by increasing Q_l and the second by warmstarting the problem correctly.

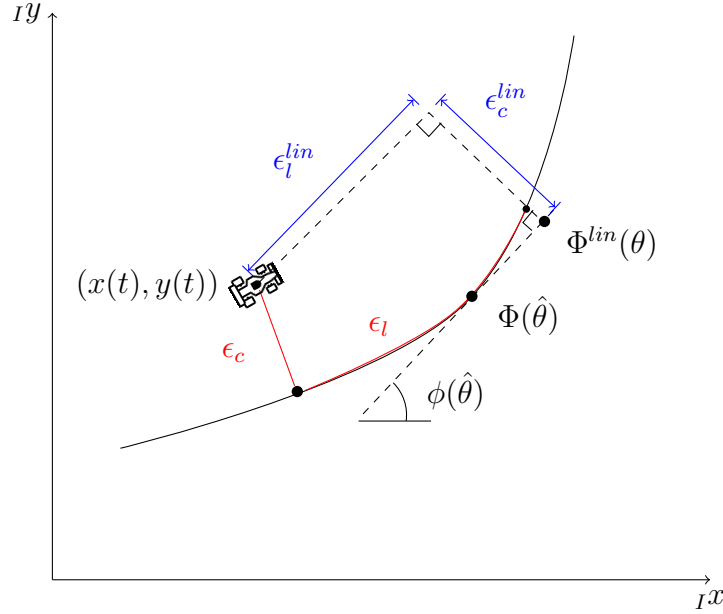


Figure 1: To compute the linearized contouring error ϵ_c^{lin} and lag error ϵ_l^{lin} the track is linearized around the last solution as the estimate of the arclength progress $\hat{\theta}$ of the MPCC. The coordinate frame (I_x, I_y) is the inertial reference frame the track is described in.

Taking a step back we see that MPCC basically is optimizing to move the position of a virtual point given by $\theta(t)$ as far along the track as possible while steering the model of the

system to keep the contouring and lag errors small. The solver then discretizes this problem and finds approximate solutions.

2 Models

The simulation is built around the dynamic model from [LDM15] is implemented with adapted parameters. It is a single track bicycle model with pajecka tire forces.

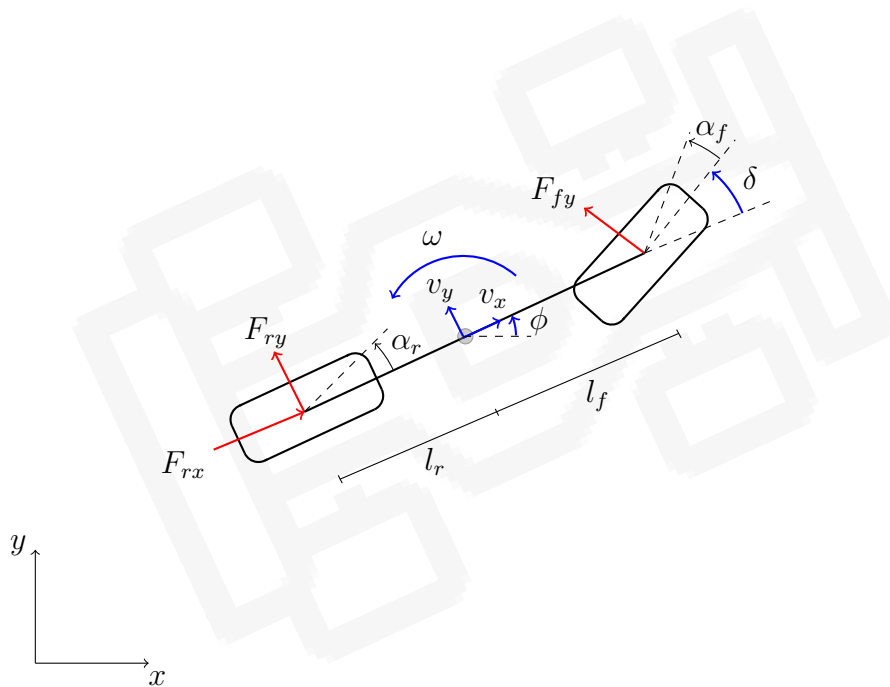


Figure 2: A single track model with pajecka tire forces is used as dynamic model for the controller.

The dynamics read

$$\dot{X} = v_x \cos(\phi) - v_y \sin(\phi) \quad (10)$$

$$\dot{Y} = v_x \sin(\phi) + v_y \cos(\phi) \quad (11)$$

$$\dot{\phi} = \omega \quad (12)$$

$$\dot{v}_x = \frac{1}{m}(F_{rx} - F_{fy} \sin(\delta) + mv_y \omega) \quad (13)$$

$$\dot{v}_y = \frac{1}{m}(F_{ry} + F_{fy} \cos(\delta) - mv_x \omega) \quad (14)$$

$$\dot{\omega} = \frac{1}{I_z}(F_{fy} l_f \cos(\delta) - F_{ry} l_r) \quad (15)$$

$$\dot{d} = u_d \quad (16)$$

$$\dot{\delta} = u_\delta \quad (17)$$

$$\dot{\theta} = u_\theta \quad (18)$$

with

$$\alpha_f = -\arctan\left(\frac{\omega l_f + v_y}{v_x}\right) + \delta \quad (19)$$

$$\alpha_r = \arctan\left(\frac{\omega l_r - v_y}{v_x}\right) \quad (20)$$

$$F_{fy} = D_f \sin(C_f \arctan(B_f \alpha_f)) \quad (21)$$

$$F_{ry} = D_r \sin(C_r \arctan(B_r \alpha_r)) \quad (22)$$

$$F_{rx} = (C_{m1} - C_{m2} v_x) d - C_{roll} - C_d v_x^2 \quad (23)$$

In the code the state is arranged in the following way

$$x = \begin{bmatrix} x \\ y \\ \phi \\ v_x \\ v_y \\ \omega \\ d \\ \delta \\ \theta \end{bmatrix} \quad (24)$$

where x , y are the position of the center of gravity, ϕ is the yaw angle of the car w.r.t. the world frame, v_x is the longitudinal velocity, v_y is the lateral velocity, ω is the yaw rate, θ the advancing parameter and lastly d is the integrated motor torque and δ is the steering angle.

The input u is arranged as

$$u = \begin{bmatrix} u_d \\ u_{\dot{\delta}} \\ u_{\dot{\theta}} \end{bmatrix} \quad (25)$$

The inputs we optimize for are chosen to be the derivatives of the commandable inputs to the system such that we can penalize their smoothness.

(For now only a the standard kinematic single track model is working with the ACADOS solver.)

3 Track Parametrization

For the track the centerline is given in waypoints. To implement MPCC we need an arclength parametrization Φ . This is realized by interpolating the waypoints using cubic splines (previously bezier curves) with a cyclic boundary condition, and creating a dense lookup table with the track location and the linearization parameters. Note that in the optimization it is not practical to pass the full parametrization since it contains functions that the used solver has difficulties dealing with such as floor and modulo. Instead the linearization parameters are precomputed offline and passed at every stage.

References

- [LDM15] Alexander Liniger, Alexander Domahidi, and Manfred Morari. Optimization-based autonomous racing of 1: 43 scale rc cars. *Optimal Control Applications and Methods*, 36(5):628–647, 2015.
- [LMG10] Denise Lam, Chris Manzie, and Malcolm Good. Model predictive contouring control. In *49th IEEE Conference on Decision and Control (CDC)*, pages 6137–6142. IEEE, 2010.