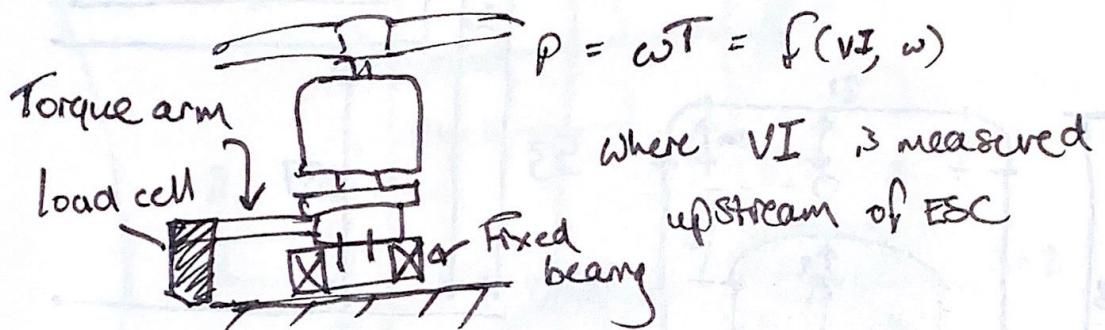


- Check motor power/voltage
- Power draw ~ each motor?
~ ESC?
- Motor rpm ~ via ESC? or Auto pilot?

→ Calibrate efficiency of each motor with static torque, ω , η test.



|| → Load cell, get sensitive.
Look at megs, more sensitive than that Amazon?

BUY

→ Light weight pressure scanners
≥ Evolution Measurements.
→ postage stamp sized Sch?
→ ↳ Own branding
↳ store data?
↳ serial? Output?
↳ Arduino maybe?

→ Check motor power/voltage

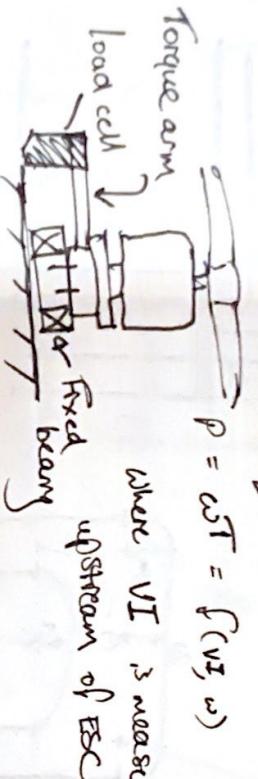
→ Power draw ~ each motor? ESC?

→ Motor NPM ~ via ESC? or Auto pilot?

→ Calibrate efficiency of each motor with static torque, ω_0 , 1 test.

$$P = \omega T = f(VI, \omega)$$

where VI is measured upstream of ESC



→ Load cell, get sensitive

Look at megas, more sensitive than that

Amazon?

→ Light weight pressure scanners

→ Evolution Measurements.

→ Postage stamp sized sen?

→ Own branding

↳ Store data?

↳ serial? Output?

↳ Arduino maybe?

→ Consider pressure sensing wrt trapings

WT

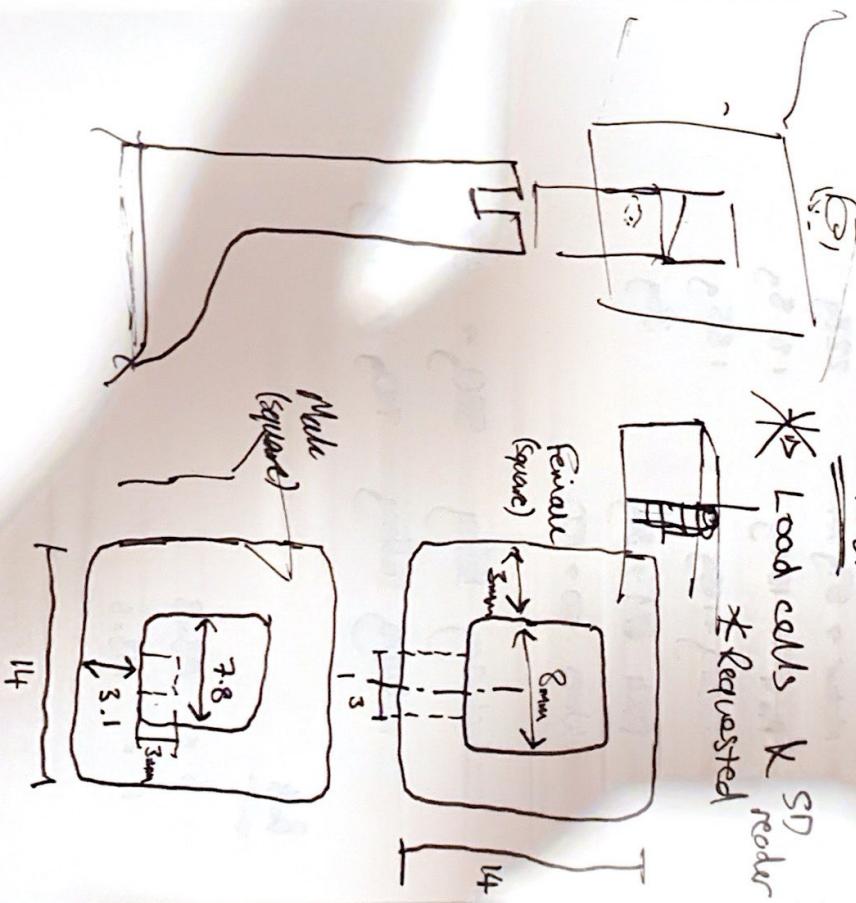
→ Make test mating joint. Use assembly

→ Bingo. Asst Megatron.

→ Assembly in CURA to change

in films.

* Load cells & SD reader
* Requested



Weights

Adapter \rightarrow ~5g \times 4 20g
Arm \rightarrow 15g \times 4 60g
Base \rightarrow ~30g 30g

Electronics \rightarrow

ESC \rightarrow 57g \times 4 228g
Motor \rightarrow 57g \times 4 228g
Pix H \rightarrow 15.8g 15.8g
Battery \rightarrow 188g 188g
PXL4 FIM \rightarrow 36g 36g
Wires 50 \rightarrow 100

$$(4 \times 20g = 80)$$

\rightarrow Load cell on arm
 \rightarrow Tension Load cell

Charger

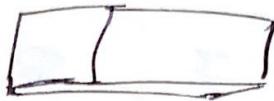
\therefore Excluding Battery $520g \rightarrow 570g$
including battery $710g \rightarrow 760g$

Bolts

16 \times M3x8
12 \times M3x6

Inventory

3x3cm



- WOOD
- ① $2.5m \times 2.4m \times 6$
 - ② $2.18m \times 4$
 - ③ $1.90m \times 6 \times 3$
 - ④ $0.90m \times 3$
- $14 \times 2.4m$
 $3 \times 3m$

SCREWS

$\sim 5cm$ wood $\sim \times 80$
Hinges $\times 3.$

CHICKEN WIRE $\sim 1.2m$ grid

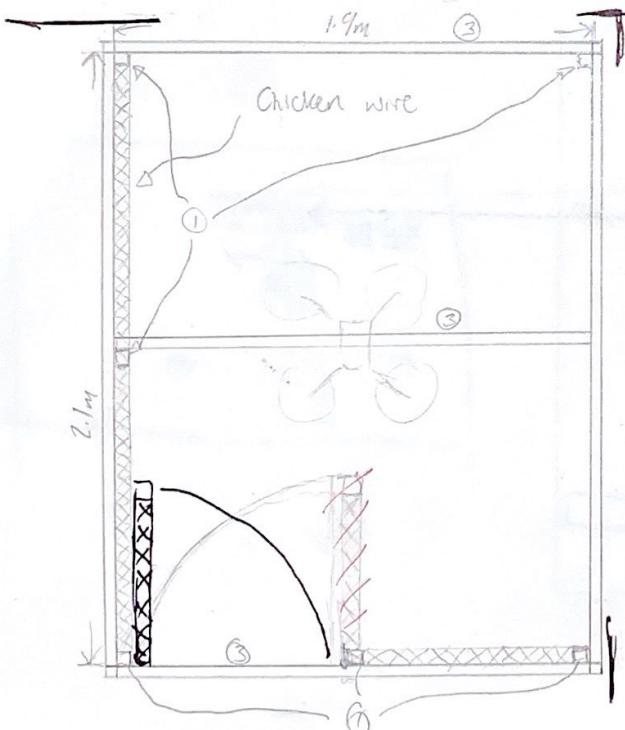
- | | | |
|------------------------|------------|-------------------------------------|
| Ⓐ $2.16m \times 2.5m$ | $\times 1$ | $(1.08 \times 2.5 \times 2)$ |
| Ⓑ $0.96m \times 0.5m$ | $\times 1$ | ⑤ $\sim 1.1m \times 1.96m \times 2$ |
| Ⓒ $0.96m \times 2.5m$ | $\times 1$ | |
| Ⓓ $0.76m \times 1.96m$ | $\times 1$ | |

$\} 6x$
8ft x 4ft
1" holes

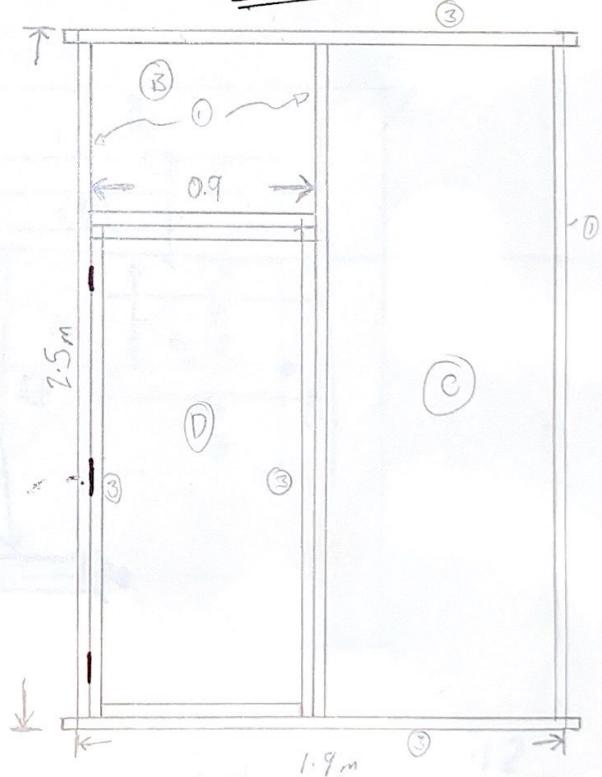
~ 3777 Amazon

Cage

TOP VIEW



FRONT



Chat with Pawel @ RR

GCS \rightarrow Q Ground Control
Sensor

RPM

- > optical sensors on ab tape stuck on half of motor

Power

- > use on board V & A measurements
- > How accurate?
- > How to get end power out, shaft power?

Pressure

- > Mini pressure sensor, talk to RP_i

Position Hold

- > use Maxbotix I^c XL Max sonar ultrasonic sensor
- > How do I incorporate the sensors into the FC?
- > Tuning PID?

RP_i

- > sensible to use RP_i for sensor management?

- > synchronizing data?

- > log rate on FC vs log rate on

Paved

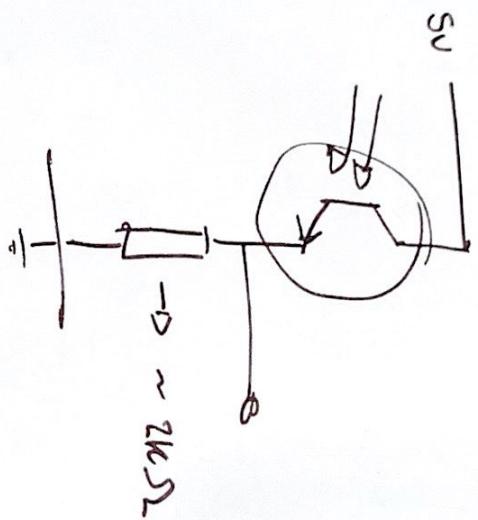
- ENSTOL
- Sensor integration
- Plug in from Simulink
- Simulink
 - long time to debug
 - Need to develop
 - Contact Matlab to get help.
 - can use
 - Sensors
 - Use PWM, 6 analogue output sensors on PWM
 - Write control software on RP; can incorporate with simulink.
 - Camera pictures?

Power

- calibrated on my
- Power / input to speed to power.

3-6 MHz range for serial comm.

Photo transistor



|||

$$r = f \times 60$$

$$\bar{g}_k \frac{1}{\tau} = f$$

WILHELM FRIEDRICH WILHELM STUDE

an old white
in California

Wetmore 1934
Hawkins

→ *Wetzel*

time

FIXED QUANTITIES

$$\phi, \varphi \quad \phi = \frac{Vx}{u}$$

$$\varphi = \frac{1}{2} \frac{Vx^2}{u^2}$$

T_f

$$T_f = \frac{Mg}{4}$$

$$T_f = T_{in} + T_{fan} + T_{diff}$$

Others

6

Diffusion ratio

$$= \frac{A_e}{A_s} = \frac{A_4}{A_3}$$

RPM/POWER

$$\text{Weight} = \rho_n (\tilde{\sigma}, \tilde{r}_c, \tilde{r}_n)$$

($T - w$) fixed at ≈ 5

$$\frac{A_2}{A_1} \text{ vs } \frac{L}{h_i}$$

$$h_i = r_c - r_h$$

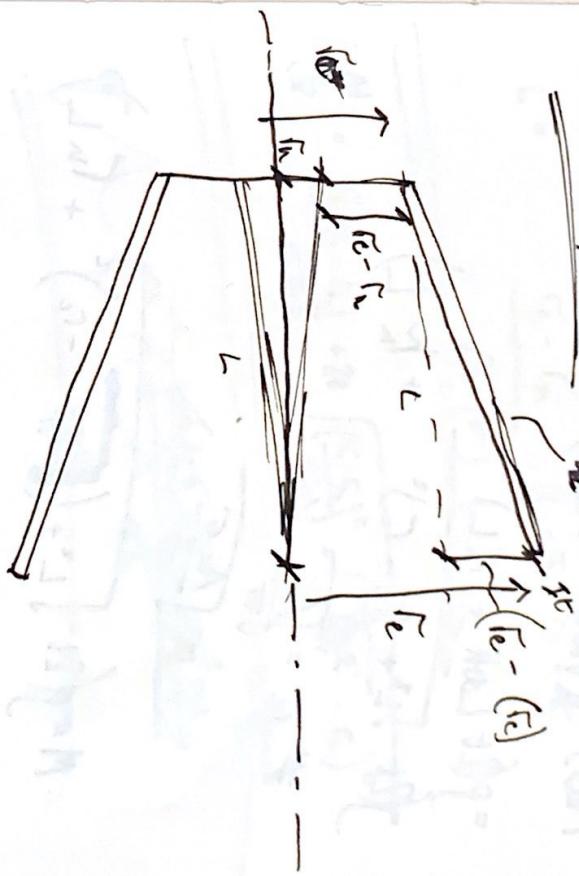
L = length of diffuser

Data Points

	$\frac{A_2}{A_1}$	$\frac{L}{h_i}$
0, 9	1.33	
1	1.35	
2	1.48	
3	1.59	
4	1.68	
5	1.76	
6	2.0	
10, 21		

Mass Model

$$L_{diff} = \sqrt{L^2 + (r_c - r_h)^2}$$



\therefore Volume of end cone $\sim \frac{1}{3}\pi r^2 h$

$$\text{Vcone} = \frac{2\pi}{3} \frac{r_h L}{2} = \frac{\pi r_h L}{3}$$

\therefore Volume of diffuser

$$\Delta V_{diff} = L \times \dots$$

$$A_e = \pi r_c^2$$

$$\therefore \Delta V_{diff} = L \times A_e \times \Delta x$$

$$\frac{A_2}{A_1} = \frac{r_c^2}{r_h^2}$$

$$\left(\frac{A_2}{A_1}\right)^{1/3} L = (r_c - r_h)$$

30, 3

1) Thrust weight

$$\text{Mass} = \rho (\text{Width} + \text{Core})$$

$$= \rho (2t L \text{Width} + r_h L)$$

$$= \rho (2t \sqrt{L^2 + (r_c - r_h)^2} + r_h L)$$

$$\# r_c = \sqrt{\frac{Ae}{\pi}}$$

$$= \sqrt{\frac{Ae \sigma}{\pi}}$$

$$\therefore M = \rho (2t \sqrt{L^2 + \left(\sqrt{\frac{Ae \sigma}{\pi}} - r_c\right)^2} + r_h L)$$

$$\frac{M}{\rho} = 2t \sqrt{L^2 + \left(\sqrt{\frac{Ae \sigma}{\pi}} - r_c\right)^2} + r_h L$$

$$\# \underbrace{\left(\frac{r_c - r_h}{Ae}\right)^{-1/2}}_{DF} \cdot L + r_h = r_c$$

$$\therefore \frac{M}{\rho} = 2t \sqrt{L^2 + \left(\sqrt{\frac{Ae \sigma}{\pi}} - r_c - L \cdot DF\right)^2}$$

$$+ r_h L$$

$$\frac{M}{\rho} = r_h L + 2t \sqrt{L^2 + \left(\frac{Ae \sigma}{\pi} - r_h - L \cdot DF\right)^2}$$

$$\# \cancel{BVT} \quad \begin{cases} r_h - (r_c - r_h) \\ -r_h - r_c + r_h \end{cases}$$

$$L = \frac{r_c - r_h}{DF(\sigma)}$$

$$\therefore \frac{M}{\rho} = \frac{r_h (r_c - r_h)}{DF(\sigma)} + 2t \sqrt{\left(\frac{r_c - r_h}{DF(\sigma)}\right)^2 + \left(\sqrt{\frac{Ae \sigma}{\pi}} - r_c\right)^2}$$

$$\sigma = \sqrt{r_c^2 - \frac{4r_h^2}{(r_c + r_h)^2} \Omega^2}$$

$$\frac{\rho}{\rho} = 2 \sigma^2 \Rightarrow \sigma = \sqrt{2\rho}$$

In the limit of separation, airbus mass can be written wrt ϕ , φ , r_c where $DF = f_a(\rho, \phi)$

$$\left(\frac{M}{\rho}\right) = \frac{(r_h(r_c - r_h))}{DF} + 2t \sqrt{\left(\frac{r_c - r_h}{DF}\right)^2 + \left(\sqrt{\frac{Ae \sigma}{\pi}} - r_c\right)^2}$$

$$\frac{M}{J} = \frac{(r_c - r_h)}{DF} + 2t \sqrt{\left(\frac{(r_c - r_h)}{DF}\right)^2 + \left(\frac{(r_c^2 - r_h^2)}{\sqrt{2\phi}} - r_c\right)^2}$$

$$A(r_c - r_h)^2 + (Br_c^2 - r_h^2B - r_c)^2$$

$$= A(r_c - r_h)^2 + r_c^4$$

$$Ae = \pi r_e^2$$

$$A_{xc} = \frac{\pi r_e^2}{6}$$

$$\therefore r_e = \sqrt{\frac{A_{xc} \cdot 6}{\pi}}$$

$$A_{xc} = \pi(r_c^2 - r_h^2)$$

$$\sigma = \frac{\phi^8}{\sqrt{2\phi}}$$

$$L_{dth}^2 = L^2 + (r_e - r_c)^2$$

$$L_{dth}^2 = L^2 + (r_e - \sqrt{L^2 + \left(\frac{A_{xc} \cdot 6}{\pi} - r_c\right)^2})^2$$

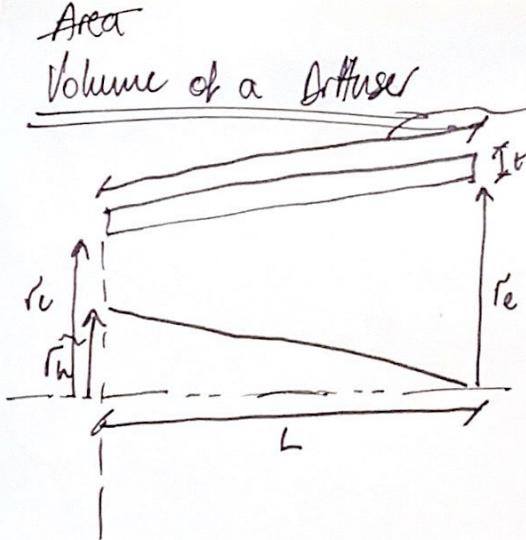
$$= L^2 + \left(\frac{\phi^8(r_c^2 - r_h^2)}{\sqrt{2\phi}} - r_c\right)^2$$

$$\therefore V_{dth} \approx L_{dth} \cdot t \cdot 2\pi r_e$$

$$\approx L_{dth} \cdot t \cdot 2\pi \sqrt{\frac{A_{xc} \cdot 6}{\pi}} \cdot 2\pi \sqrt{\frac{\phi^8(r_c^2 - r_h^2)}{\sqrt{2\phi}}}$$

$$= L_{dth} \cdot t \cdot \cancel{\sqrt{A_{xc} \cdot 6 \cdot \pi}}$$

$$= 2\pi t \sqrt{\frac{\phi^8(r_c^2 - r_h^2)}{\sqrt{2\phi}}} \cdot \sqrt{\left(\frac{r_c - r_h}{DF}\right)^2 + \left(\frac{\phi^8(r_c^2 - r_h^2)}{\sqrt{2\phi}} - r_c\right)^2}$$



$$V_{cone} = \frac{1}{3} h \pi r^2$$

$$= \frac{1}{3} L \pi r_h^2$$

$$= \frac{1}{3} \frac{\pi r_h^2 (r_c - r_h)}{DF}$$

PDI 3 A+ Headboard	
RPM + Vcc (Red)	PPI
Column 2 Blue (Red)	/down/
Column 2 Red (Black)	use pin sensors, outputs
Column 2 Green (Green)	either pins at each
Column 2 Blue (Blue)	or point sensor strips
RPM (Yellow)	use pin sensors, outputs
RPM GND (Purple)	either or fine strips
GND	use gop measurement
LC (Yellow)	to determine resu#:
LC (Purple)	replace by rpm - c-
LC (Yellow)	test setup for
LC (Purple)	waveform module
RPM Motor 1	WAV
RPM Motor 2	and long #/j
RPM Motor 3	waveform
RPM Motor 4	short

$$\Delta h_o = \bar{u} \Delta V_o = \frac{\bar{u}}{A} \Delta A$$

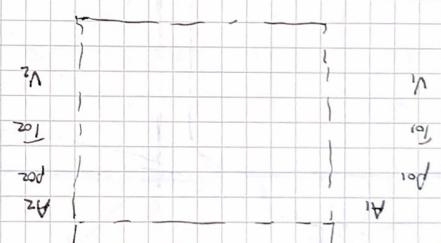
$$\Delta h_o = \bar{u} \Delta V_o = \frac{\bar{u}}{A} \Delta A$$

$$dh = \frac{dp}{\rho g}$$

$$ds = dh - \frac{dp}{\rho g}$$

$$\therefore \Delta u \sim \frac{dp}{\rho g}$$

Assume $T_{o1} = T_{o2}$ i.e. isothermal expansion



$$\Delta u = \int P dV$$

Assume 1:1 diffuser $\therefore A_1 = A_2 = A$

$$\Delta u = \frac{u^2}{2}$$

$$\begin{aligned} f &= \bar{u}_2 V_2 - \bar{u}_1 V_1 + (P_2 - P_1) A \\ &= \bar{u} V_2 + g(P_2 - P_1) A \end{aligned}$$

$$P_1 \text{ at end} = P_{o1}$$

SFM

$$\phi = \frac{u}{\bar{u}_{o1}}$$

$$f = \Delta h_o$$

$$T = \frac{Mg}{4} \approx 2.981$$

$$= 4.905$$

$$T = g A V_2 + (P_2 - P_{o1}) A / \rho g$$

$$P_2 = P_{o1} + \frac{1}{2} g A V_2^2$$

$$\therefore T = A g V_2^2 + \frac{1}{2} A g V_2^2$$

$$= \frac{3}{2} A g V_2^2$$

$$\therefore V_2^2 = \frac{2T}{3A g}$$

$$V_2 = \sqrt{\frac{2T}{3A g}}$$

WORK

$$V_2 = \int \frac{dV}{dt}$$

$$V_2 = \int \frac{dV}{dt}$$

see see where in code & change.
Free vertex const. NV at exit.



- steps. ↗
- ↗ lattice compressor design ↗
- Turbo cap ↗
- THIS EFFECT

BL wave 13L Jarry

→ check things ↗
jumps after ↗
↗ turbos ↗

→ flow coeff? lower?

→ blade count

PROJECT MAPPING

$$SL = 4000 \text{ m}$$

$$\frac{60}{4000 \cdot 2\pi} =$$

$$\Delta = \frac{\Delta h_{stagn}}{\Delta h_{stagn}} = 0.5$$

$$\phi = \frac{u}{U}$$

$$\phi = \frac{u}{U}$$

$$\therefore \phi \sim \frac{u}{U}$$

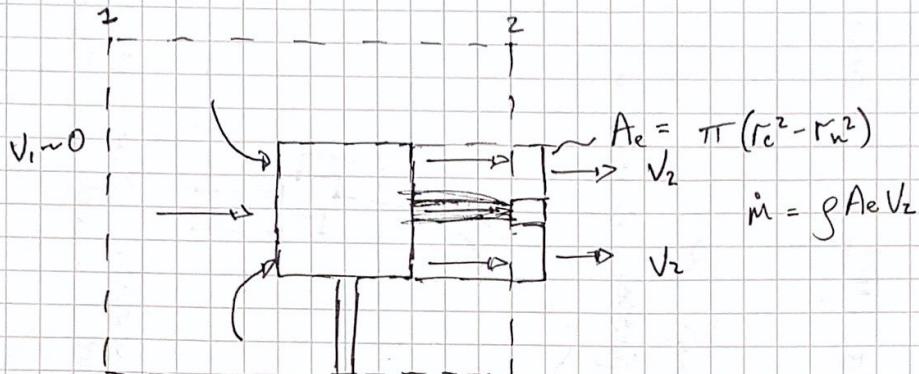
$$\therefore \Delta h \approx \frac{u}{U}$$

$$ALe \sim \frac{f}{\Delta h}$$

$$= \frac{60}{4000 \cdot 2\pi}$$

CONTROL VOLUME ANALYSIS

Upstream velocity & pressure \sim zero, atmospheric respectively.



$T_f \sim$ on Axn \therefore equal opposite on fluid

$$g = \text{const} \quad (\text{Low } V, M \ll 1)$$

$$\therefore p_0 = p_2 + \frac{1}{2} \rho V^2$$

CV large enough such that $m_m V_m \approx 0$ for all circumferences excluding A_e .

$$\therefore T_f = m (V_2 - V_1) + \cancel{p_2 A_e} (p_2 - p_1) A_e \quad p_2 + \frac{1}{2} \rho V_2^2 = p_1 + \frac{1}{2} \rho V_1^2$$

$$T_f = m (V_2 - V_1) + A_e (p_2 - p_1) \quad p_{02} - \frac{1}{2} \rho V_2^2 - (p_{01} - \frac{1}{2} \rho V_1^2)$$

$$V_1 \approx 0; \quad p_1 \approx p_{01} \approx p_{0j}; \quad p_e = \cancel{p_{02} + \frac{1}{2} \rho V_2^2}$$

$$p_2 + \frac{1}{2} \rho V_2^2 = p_1 + \frac{1}{2} \rho V_1^2 \quad = \Delta h_o g - \frac{1}{2} \rho (V_2^2 - V_1^2)$$

$$= \frac{1}{2} V_2^2 g - \frac{1}{2} \rho V_2^2 + \frac{1}{2} \rho V_1^2$$

$$\Delta h_o = \frac{\Delta P_o}{\rho g}$$

$$\Delta P_o = p_{02} - p_{01}$$

$$= p_2 + \frac{1}{2} \rho V_2^2 - p_{01}$$

$$\Delta h_o = \frac{1}{2} V_2^2$$

$$p_{02} = p_2 + \frac{1}{2} \rho V_2^2$$

$$p_2 = p_{01} = p_a$$

If assume exit static = inlet static

$$p_2 - p_1 = 0$$

$$p_{02} - \frac{1}{2} \rho V_2^2$$

$$T_F = m(V_2 - V_1) + A_e(p_2 - p_1)$$

$$\begin{aligned} T_F &= dh - \frac{dp}{g} \\ dh &= \frac{dp}{g} \end{aligned}$$

$$\therefore \Delta h_o = \frac{\Delta p_o}{g}$$

$$\Delta p_o = p_{o2} - p_{o1}$$

$$= p_2 + \frac{1}{2} \rho V_2^2 - p_1$$

$$= \frac{1}{2} \rho V_2^2 + (p_2 - p_1)$$

and V_2

and accelerate
 $\therefore \text{Max } P_{o2}$
 $\therefore \text{Max } V_2$

\therefore To maximise thrust, maximise $p_2 - p_1$, ie diffuse flow downstream
 or fan. Most efficient when flow leaves pressure matched

↓

$$p_2 = p_1$$

$$\therefore \Delta p_o = \frac{1}{2} \rho V_2^2 \Rightarrow \Delta h_o = \frac{1}{2} V_2^2$$

$$\therefore T_F = m(V_2 - V_1)$$

$$= g A_e V_2^2$$

$$\therefore V_2^2 = \sqrt{\frac{T_F}{g A_e}}$$

$$\phi = \frac{V_2}{U}$$

$$\varphi = \frac{\frac{1}{2} V_2^2}{U^2}$$

$$\phi = \frac{V_L}{U}$$

$$V_2 = \sqrt{\frac{T_F}{g A_e}} = \sqrt{\frac{T_F}{g \pi (r_c^2 - r_n^2)}}$$

$$U = \sqrt{\frac{r_c + r_n}{2}}$$

$$\phi = \sqrt{\frac{T_F}{g \pi (r_c^2 - r_n^2)}} \circ \frac{2}{\sqrt{\pi} (r_c + r_n)}$$

$$\phi = \frac{T_F}{g \pi (r_c^2 - r_n^2)} \circ \frac{2}{\pi r^2 (r_c + r_n)^2}$$

Man

r.c

r.h

r.m

Span

r.radius

HTR

Al-rotor

AR-stator

tip-gap-percentage

r.11

s.12

ac-r

$$DF \approx 1 - \frac{V_2}{V_1} + \frac{1}{2} \frac{\Delta V_{\theta}}{V_1} \frac{s}{c}$$
$$\Delta V_{\theta-m} = |V_{\theta 2} - V_{\theta 1}|$$

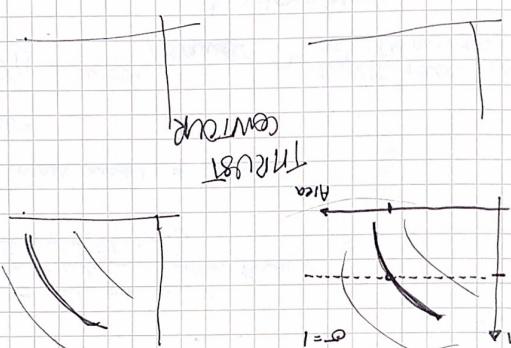
Jonny's CV Analysse:

$$p_0 = p_{00} - \frac{1}{2} \rho V_0^2$$

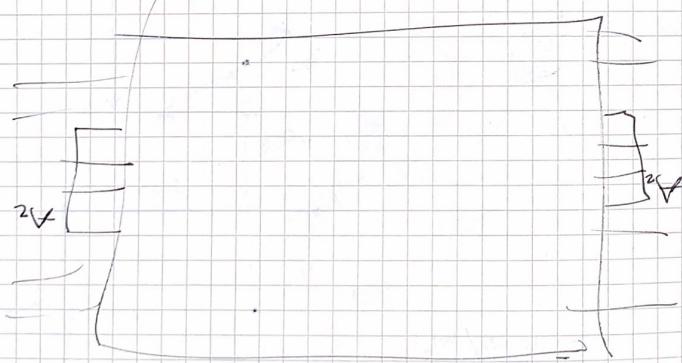
$$p_4 = p_{04} - \frac{1}{2} \rho V_4^2$$

$$p_4 - p_0 = \frac{1}{2} \rho (V_0^2 - V_4^2)$$

+ fees by authority.



- Only quadratic equations -
 - So $b + d$ are per-se
 - Get powers of the below.



$$\frac{1}{\phi} = \frac{1}{0.7} = 1.4$$

$$\Delta V_0 = \frac{S}{\lambda^2} \cdot \frac{1}{2} \cdot \frac{1}{V^2} \cdot \frac{1}{2} \cdot \frac{1}{V^2} = \frac{S}{\lambda^2} \cdot \frac{1}{V^4}$$

(21) χ (11) Mean μ (21) χ (11) χ (21) χ (11)

the new draw (like as luster)

Various dead quadratics drawing programs

• 1 •

"... > W" 12

$$P(A \cap B) = P(A)P(B) \quad \therefore P(A) = 1$$

which is \neq zero.

$$m \cdot n = ?$$

CU Analysis → from design

6. accelerated to thrust by pressure change

Small mass model reflects to BL separation

$$\frac{d}{dt} \left(\frac{1}{2} \rho A U^2 \right) = P_A U^2$$

How does thickness

$$U = \frac{d}{dt}$$

$$[14] \quad R = \frac{R}{\delta}$$

$$H = 1.8 + 3.35 \delta$$

H = shape factor

$$\frac{\delta}{R}$$

$$[15]$$

operating weight vs pressure loss due diff.

With weight? more at bleed flow rate

bleed flow rate more at

length/diameter ~ aspect ratio measure

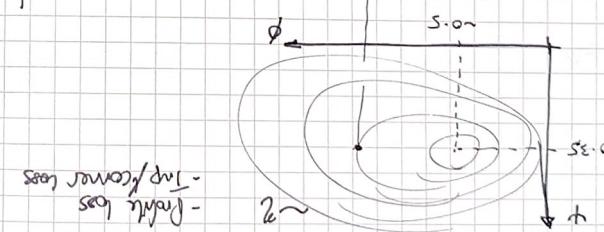
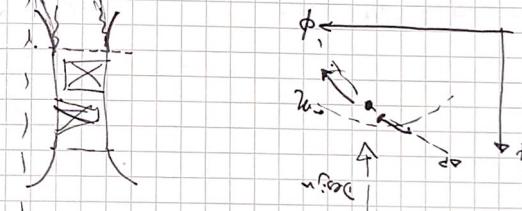
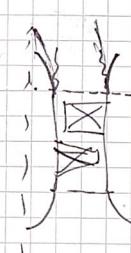
Diffusers

length/diameter

shape mass model of diffuser

- contour maps of flow field
- 2D power analysis on wall shear stress
- let's do it in 3D

$$C = 1.5 \Leftrightarrow P_2 - P_1 \text{ across boundary}$$



Sin. Curve

Fixed

RPM (or Power...)

ρ, ω

$T_r \sim \begin{cases} \text{Intable} \\ \text{For Ext} \end{cases}$

r_h

Torque vs RPM
no work eqn.
(Matlab)

linked as $f_n(r)$ once it is included.

See Sam's email

Vary

σ

put in σ , spits out r_c (via area)

Then from ESDU (expressed analytically)

$\sigma \rightarrow$ length from r_i & r_h as above.

\hookrightarrow length modelled to weight

$(T - w)$

s

σ

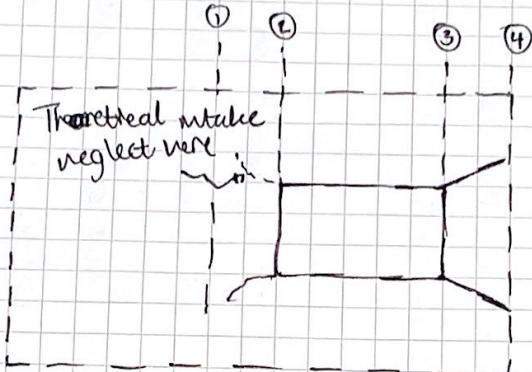
\rightarrow Want PONTEK optimised
w/ no constraint on $(T - w)$

\rightarrow Then fix

$L \& r \dots \sigma$ limited by
mass model

\therefore Vary r (hence σ) to optimise power.

CV Analyses with σ , OPTIMISE FOR MIN POWER wrt $r_c, (r_h)$



① $p_1 = p_a = P_{01}^*$ A_1 large, $V_1 = 0$

② $p_2 = p_a$ (no intake) $A_2 = A_{ac}$

$$③ p_3 = p_3 \neq p_{03} = p_{03} - \frac{1}{2} g V_3^2$$

$$A_3 = A_{ac}, V_3 = V_{ac}$$

$$④ p_4 = p_a = p_{04} - \frac{1}{2} g V_4^2$$

$$A_4 = \sigma A_3 = \sigma A_{ac}$$

$$V_4 \neq V_{ac}$$

Relating ③ & ④

$$p_3 = p_{03} - \frac{1}{2} g V_3^2 ; \quad p_4 = p_{04} - \frac{1}{2} g V_4^2 ; \quad p_{03} = p_{04} ; \quad p_4 = p_a$$

$$p_3 + \frac{1}{2} g V_3^2 = p_4 + \frac{1}{2} g V_4^2$$

By continuity

$$A_3 V_3 = A_4 V_4$$

$$\therefore \left(\frac{V_3}{V_4} \right)^2 = \sigma = \left(\frac{V_{ac}}{V_4} \right)^2 \quad \therefore V_{ac}^2 = \sigma V_4^2$$

SFME

$$T_T = m_4 V_4 + A_4 p_4 - A_1 p_1$$

$$p_4 = p_1 \quad \therefore p_4 - p_1 = 0$$

$$= m_4 V_4$$

$$= g A_4 V_4^2$$

$$= g A_4 \frac{V_4^2}{\sigma}$$

$$= g A_{ac} V_{ac}^2$$

$$A_{ac} = \frac{A_4}{\sigma}$$

$$\therefore V_{ac} = \sqrt{\frac{T_T}{g A_{ac}}}$$



$$g A_4 V_4^2 = \frac{g A_{ac} V_{ac}^2}{\sigma}$$

$$m V_4 = g A_{ac} V_{ac} V_4$$

$$= g A_{ac} V_{ac}^2$$

Flow coefficients; stage loading

$$\phi = \frac{V_{ac}}{U}; \quad \varphi = \frac{\Delta h_o}{U^2}$$

Near isentropic pressure rise

$$\therefore T ds = dh - \frac{g}{s} dp$$

$$\therefore dh_o = \frac{dp}{s}$$

$$\Delta h_o = \frac{\Delta P_o}{s} \quad \therefore \varphi = \frac{\Delta P_o}{g U^2}$$

$$\Delta P_o = P_{o3} - P_{o2}$$

$$\begin{cases} P_{o2} = P_a \\ P_{o3} = P_3 + \frac{1}{2} g V_3^2 \end{cases}$$

$$\Delta P_o = P_3 + \frac{1}{2} g V_3^2 - P_a$$

$$P_3 = P_a + \frac{1}{2} g (V_4^2 - V_3^2) \quad // \text{from Bernoulli}$$

$$= P_a + \frac{1}{2} g \left(\frac{V_3^2}{6} - V_3^2 \right)$$

$$V_3 = V_x$$

$$P_3 = P_a + \frac{1}{2} g \left(\frac{V_x^2}{6} - V_x^2 \right)$$

$$\therefore \Delta P_o = \frac{1}{2} g V_x^2 \left(\frac{1}{6^2} - 1 \right) + \frac{1}{2} g V_x^2$$

$$= \cancel{\frac{1}{2} g V_x^2 \left(\frac{1}{6^2} - 1 \right)} = \frac{1}{2} g V_x^2 \left(\frac{1}{6^2} \right)$$

$$\Delta P_o = \frac{1}{2} g \left(\frac{V_x^2}{6} - V_x^2 \right) + \frac{1}{2} g V_x^2$$

$$= \frac{1}{2} g \frac{V_x^2}{6}$$

$$\therefore \varphi = \cancel{\frac{V_x^2 (1 - \frac{1}{6^2})}{g U^2}} = \frac{\frac{1}{2} g V_x^2}{g U^2 \cdot 6^2}$$

$$\therefore \varphi = \frac{V_x^2}{2 \cdot 6 \cdot U^2}$$

$$\therefore \varphi = \frac{V_x^2}{2 U^2 \cdot 6^2}$$

$$\varphi = \frac{V_x}{U}$$

$$V_x = \sqrt{\frac{T_f}{\rho A_x}}$$

$$\frac{\varphi^2}{26} = \rho$$

$$\frac{\varphi^2}{\varphi} = 26$$

Redefine σ in terms of A_x & T_f

$$\varphi = \sqrt{\frac{T_f}{\rho A_x U^2}}; \quad \varphi = \frac{T_f}{2 \rho A_x U^2 \sqrt{6}}$$

Rearrange for σ

$$\varphi^2 \cdot \frac{\rho A_x U^2}{T_f} = \sigma$$

$$\frac{T_f}{2 \rho A_x U^2 \varphi} = \sigma$$

$$\left. \begin{array}{l} \varphi^2 \in 26^2 \\ \varphi \end{array} \right\}$$

$$A_x = \pi (r_c^2 - r_n^2)$$

\therefore Limit of diffusional factor can be determined by flow co-eff & stage loading

$$\sigma = \frac{\varphi}{\sqrt{26}}; \quad M = \rho \left[\frac{(r_c - r_n)}{DF} + 2t \int \left(\frac{(r_c - r_n)}{DF} \right)^2 + \left(\frac{A_x \varphi}{T_f \sqrt{26}} - r_c \right)^2 \right]$$

Optimize for mass wrt r_c Assume fixed: r_h, φ, ϕ

$$DF = \left(\frac{\sigma - c}{A}\right)^{1/B} = \left(\frac{\phi}{2A\sqrt{2\varphi}} - \frac{c}{A}\right)^{1/B}$$

$$\begin{aligned} A &= 0.2863 \\ B &= 0.5627 \\ C &= 1.06 \end{aligned}$$

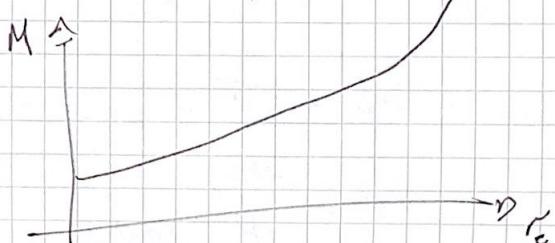
Empirically from
ESDU 75026

$$\therefore M = \rho_{air} \left[\frac{r_h(r_c - r_h)}{DF} + 2\pi \sqrt{\left(\frac{r_c - r_h}{DF}\right)^2 + \left(\frac{\phi(r_c^2 - r_h^2)}{\sqrt{2\varphi}} - r_c\right)^2} \right]$$

Total Diffuser Volume ($V_{cone} + V_{thin shell}$)
 \downarrow
 $t \times 2\pi r \times L$

$$V_{total} = \frac{\pi r_h^2 (r_c^2 - r_h^2)}{3DF} + 2\pi t \sqrt{\frac{\phi(r_c^2 - r_h^2)}{\sqrt{2\varphi}}} \cdot \sqrt{\left(\frac{r_c - r_h}{DF}\right)^2 + \left(\frac{\phi(r_c^2 - r_h^2)}{\sqrt{2\varphi}} - r_c\right)^2}$$

$$\therefore \text{Mass} = V_{total} * \rho_{PLA}$$



$$\phi \varphi = \frac{T_f}{2\rho A x U^2 c}$$

$$\sigma = \frac{\phi}{\sqrt{2\varphi}}$$

$$= \frac{T_f \sqrt{2\varphi}}{2\rho A x U^2 \phi}$$

$$A_x = \pi (r_c^2 - r_h^2)$$

$$U = 2 \sqrt{(r_c + r_h)}$$

$$U^2 = \frac{2}{4} (r_c + r_h)^2$$

$$\varphi =$$

$$\phi \sqrt{\rho} \cdot \sqrt{2\varphi} \rho A x U^2 = T_f$$

$$\phi \sqrt{\rho} \pi (r_c^2 - r_h^2) \frac{\sqrt{2}}{\sqrt{2}} (r_c + r_h) \sqrt{\rho} = T_f$$

$$\phi \sqrt{\rho} \pi \frac{\sqrt{2}}{\sqrt{2}} (r_c^2 - r_h^2) (r_c + r_h) = T_f$$

$$\phi \sqrt{\rho} \pi \frac{\sqrt{2}}{\sqrt{2}} (r_c - r_h) (r_c + r_h)^3 = T_f$$

$$L = \frac{r_c - r_h}{DF}$$

$$\frac{\phi}{\sqrt{2\varphi}} - C$$

$$\therefore \varphi = \frac{T_f \cdot \sqrt{2\varphi} \cdot 4}{2\rho \pi (r_c^2 - r_h^2) \sqrt{2} (r_c + r_h)^2 \phi}$$

$$\therefore T_f = \frac{4 \rho \sqrt{2\varphi} (r_c^2 - r_h^2) \sqrt{2} (r_c + r_h)^2 \phi}{4 \sqrt{2\varphi}}$$

$$= \sqrt{\rho} \phi \sqrt{\pi} (r_c^2 - r_h^2) \sqrt{2} (r_c + r_h)^2$$

SPEC

$$m(h_0 + g\frac{r^2}{2} + \frac{V_0^2}{2}) + \dot{Q} = m(h_4 + g\frac{r^2}{4} + \frac{V_4^2}{2}) + \dot{W} \quad m^{-1}$$

$$h_4 - h_0 + \frac{1}{2}(V_4^2 - V_0^2) = \dot{Q} - \dot{W} \quad \text{Assume } \dot{Q} = 0$$

$$\Delta h_0 + \frac{1}{2}V_4^2 = -\dot{W}$$

$$+ \cancel{V_0} + \cancel{V_0} = -\dot{W}$$

$$\omega = -re$$

i.e. work done on Rod

$$\frac{\Delta h_0}{\rho} = \frac{1}{2} \frac{V_0^2}{6}$$

$$V_4^2 = \frac{V_0^2}{6}$$

$$P = \frac{1}{2} \rho A V_x^3$$

$$P_Q = \frac{1}{2} \frac{V_x^2}{\sigma^2}$$

$$= \frac{1}{2} \rho A T$$

$$= \frac{1}{2} \frac{T \rho}{\rho A \sigma^2}$$

$$\therefore \Delta h_0 + \frac{1}{2}V_4^2 = -\dot{W}$$

$$-\dot{W} = \frac{V_0^2}{26} + \frac{V_0^2}{26} = \frac{V_{26}}{52} = \frac{T_f}{g A r e \sigma} = \frac{T_f}{T_p A r e \sigma}$$

$$P = \frac{T^2}{1450 \sigma}$$

w/kg

$$\therefore -\dot{W} = \frac{\sqrt{2} \rho \sigma L^2}{26} = 2 \varphi \sigma L^2 (\bar{r}_c + \bar{r}_h)^2$$

$$\therefore \bar{W} = \frac{4 \varphi \sigma L^2 (\bar{r}_c + \bar{r}_h)^2}{2}$$

Omega vs r_c ✓

Fix omega, \bar{r}_c vs \bar{r}_h ?

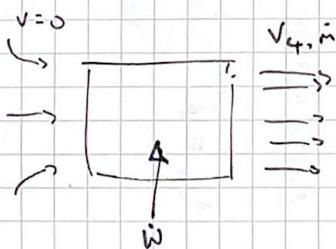
HTR

effects

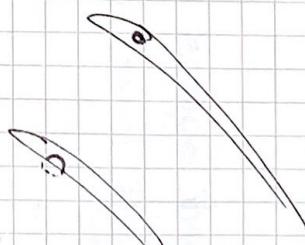
?

119.4
89.57

~ 0.4 as a maximum



$$\dot{W} = \frac{1}{2} \bar{m} V_4^2 = \frac{1}{2} \bar{m} \frac{V_x^2}{6}$$



Using loss models produce SxS of ϕ vs ϕ for loss.

Use Smith chart.

A. Dicksen → Check out appendix as well.

Ch 9

→ well written

Not beyond 4A3

→ Check against notes same sent!!

3 control volumes.

CV

→ efficiency.

2. ΔP_{rotor}

3. ΔP_{stator} ✓ $A_3 = A_{\infty}$

$$5 \quad V_4 = \frac{\sqrt{P_3}}{\rho} \frac{\sqrt{3}}{\sqrt{6}}$$

~~H G -> D G~~ ²

2.1 Mach # < ✓

2.2 (1) P_{atm}

2.3

2.4 (18)/(17) replace constants with a or b

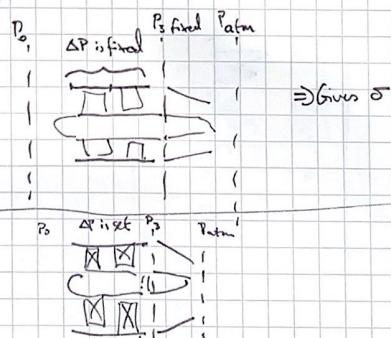
2.4.1 (21) Model of rotors & stators as flat plates & relate to $N \rightarrow$

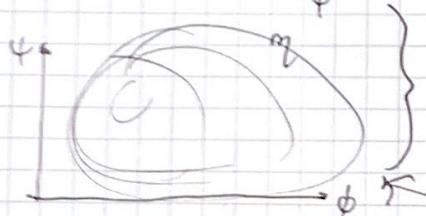
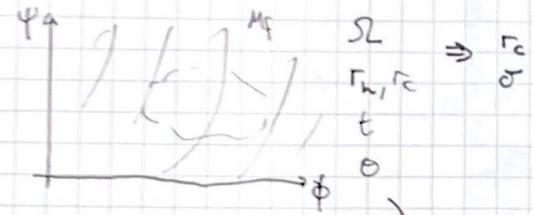
.. relates to P/C - pitch chord.

Trade off between weight + N

(22) Want thickness of diffuser also defined as predetermined variable.

(21) ~~compute~~ Combine motor weight with rest of drone instead of propulsor





See if combining M_f with η gives any optimum ~~systems~~ in OP.
Use existing data for η , to prove concept before using Loss model.

$$M_f = \frac{I}{P} \sqrt{\frac{M}{2gA}}$$

$$\frac{1}{2} \frac{gAVx^3}{\sigma^2} = \frac{1}{4} \frac{gAT^{3/2} \sigma^{3/2}}{g^{3/2} A^{3/2} \sigma^{3/2}}$$

$$= \frac{T^{3/2}}{\sqrt{4gAxG}} = \frac{M^{3/2}}{\sqrt{4gAxG}}$$

$$\omega = \frac{T_f}{2gAxG}$$

$$T_f = \sqrt{2\varphi} \sqrt{gAx} U^2$$

T_f



$$= \frac{\sqrt{4\varphi} \sqrt{gAx} U^2}{\sqrt{2gAxG}} = \frac{\sqrt{g} \sqrt{\varphi} U^2}{\sqrt{2} \sqrt{G}}$$

$$\sigma = \frac{\varphi^2}{2\varphi}$$

$$= \frac{\sqrt{2} \varphi^{3/2} \sigma U^2}{\varphi}$$

P.

$$= \eta \sqrt{2} \varphi^{3/2} U^2 \cdot gAxVx$$

\Rightarrow

$$M_f = \frac{I}{P} \sqrt{\frac{T}{2gA}}$$

$$= \frac{I}{P} \sqrt{\frac{\varphi}{2gA}} \ln \left(\frac{R}{\varphi} \right)$$

4A3 Turbomachinery - Turbine Cascade

1.1

Free-stream Δp

$$\begin{aligned} P_2 - P_1 &= (\rho_{01} - \rho_1) - (\rho_{02} - \rho_2) - (\rho_{02} - P_2) \\ &\approx -100 \text{ mm H}_2\text{O} \end{aligned}$$

$$\frac{P_1}{P_2} = \frac{P_2''}{P_2}$$

$$\begin{aligned} \sigma^{1/2} &= \frac{\phi}{\sqrt{2\psi}} \quad \frac{\phi^2}{2\psi} \\ P &= \frac{\sqrt{2\psi} T^3}{\sqrt{4g A_x} \phi} \\ &= \sqrt{\frac{\phi \cdot T^3}{g \phi^2 ((r_e + r_n) \pi (r_e^2 - r_n^2))}} \\ &= \sqrt{\frac{\phi T^3}{2g \phi \pi (r_e^2 - r_n^2)}} \\ &= \sqrt{2^3 \rho^5 \cdot \phi \cdot g A_x U^3} = \sqrt{2^3 \rho^5 \phi} \cdot g \pi (r_e^2 - r_n^2) \frac{\pi r^3}{8} (r_e + r_n)^3 \end{aligned}$$

$$\frac{P_1}{P_2} = \left(\frac{\rho_1}{\rho_2}\right)^{\gamma} \quad \rho_2 = \rho_{\text{stagn}} = 100126 - 100130$$

$$\begin{aligned} |\rho_2 - \rho_1| &= 200 \times 10^{-3} / (1000) / (9.81) \\ &= 200 (9.81) \text{ Pa} \\ \therefore \rho_1 &= 100130 + 200 (9.81) \\ &= 102092 \end{aligned}$$

$$\therefore \frac{P_1}{P_2} = 1.0196$$

$$\delta = 1.4 \quad \# \text{ Air uncombusted}$$

$$\therefore \frac{P_1}{P_2} = 1.0140$$

$\therefore \text{Change in } \rho \sim 1\%$

little content of turbomachinery 1% changes
should not be neglected.

$$gAV_x = gAV_4 \quad A_x = \frac{A_4}{6} \quad \therefore$$

$$\frac{A_x}{A_4} = \frac{V_4}{V_x} = \frac{1}{6} \quad \therefore \frac{V_{bc}}{V_4} = 6$$

$$T_f = \frac{gA_{bc}V_x^2}{\sigma^2}$$

$$\therefore V_{bc}^2 = \sqrt{\frac{4\sigma^2}{gA_x}}$$

$$\Delta p_0 = p_3 + \frac{1}{2}\rho V_{bc}^2 - p_a$$

$$= \left[p_a + \frac{1}{2}\rho \left(\frac{V_x^2}{\sigma^2} - V_{bc}^2 \right) \right] + \frac{1}{2}\rho V_{bc}^2 - p_a$$

$$= \frac{\rho V_x^2}{2\sigma^2}$$

$$\varphi = \frac{\Delta p_0}{\rho U^2} = \frac{V_x^2}{2U^2\sigma^2}$$

$$\sigma^2 = \frac{\phi^2 g A_x U^2}{T_f}$$

$$\sigma^2 = \frac{T_f}{2\rho A_x U^2 \varphi}$$

$$2\rho^2 A_x^2 U^4 \phi^2 \varphi = T_f^2$$

$$g A_x \phi U^2 \sqrt{2\varphi} = T_f$$

$$\phi^2 = \frac{T_f \sigma^2}{g A_x U^2} \quad \varphi = \frac{T_f}{2\rho A_x U^2 \sigma^2}$$

$$\frac{\phi}{\phi^2} = \frac{1}{\sigma^4}$$

$$\frac{2\varphi \sigma^4}{\sigma^4} = \left(\frac{\phi^2}{2\varphi} \right)$$

$$\sigma^4 = \frac{\phi^2}{\varphi} \quad \sigma = \sqrt{\frac{\phi}{\varphi}} = \left(\frac{\phi^2}{\varphi} \right)^{1/4}$$

$$\phi^2 = \frac{T_f g^2}{\rho A_2 v^2}$$

$$\psi = \frac{T_f}{2 \rho A_2 v^2}$$

$$\phi^2 = \frac{2\psi}{M} \frac{g^2}{v^2} v^2$$

$$g^2 = \frac{\phi^2}{2\psi}$$

$$\frac{T_f}{\rho A_2} = \psi \frac{2\rho v^2}{M} = \frac{\phi^2 v^2}{g^2}$$

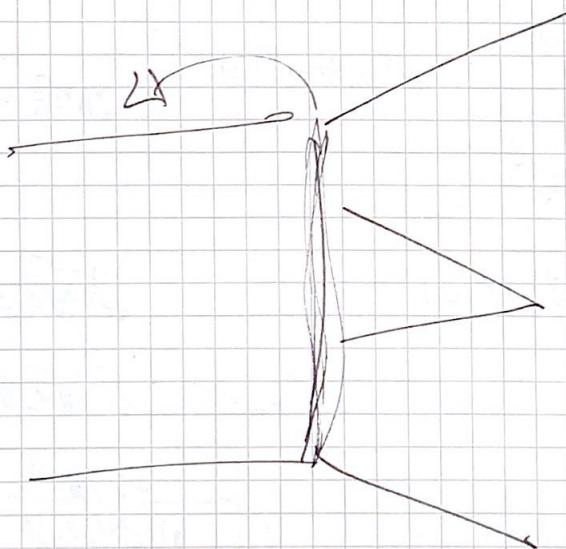
$$\phi^2 = \frac{T_f \phi^2}{\rho A_2 v^2 2\psi}$$

$$T_f = 2\psi \rho A_2 v^2$$

$$T_f = \frac{\phi^2 \rho A_2 v^2}{g^2}$$

$$T_f = \frac{\phi^2 \rho A_2 v^2}{g^2}$$

sigh
figh



SAM'S EQUATIONS

$$P_i = \frac{T_{tot}^{3/2}}{\sqrt{4\sigma g A_x}}$$

$$= \left[\frac{\phi^6 g^3 A_x^3 U^6}{4 \sigma g A_x} \right]^{1/2}$$

$$= \frac{\phi^3 g A_x U^3}{2 \sqrt{6}} \cdot \sigma^{-3}$$

$$= \frac{\phi^3 g A_x U^3}{2 \sigma^{3/2}}$$

$$T_r = \frac{\phi^2 g A_x U^2}{\sigma^2} = \underbrace{\phi^2 g A_x U^2}_{kg m^{-3} m^2 m^2 s^{-2}} \cdot \sigma^{-2}$$

$$kg m^{-3} m^2 m^2 s^{-2} = kg m s^{-2}$$

$$\Rightarrow \frac{(kg m s^{-2})^{3/2}}{(kg m^{-3} m^2)^{1/2}} = kg m^2 s^{-3}$$

$$\therefore P_i = \frac{T_r \cdot \phi}{2 \sigma^{3/2}}$$

$$= \frac{T_r \cdot \phi U}{2 \sigma^{3/2}}$$

~~cancel~~

$$P_f = \frac{1}{2} \bar{m} V_4^2$$

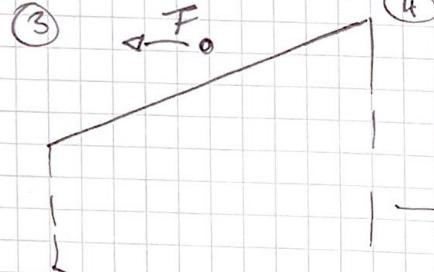
$$\phi^3 g^{3/2} A_x^{3/2} U^3$$

$$M_f = \cancel{\omega} \cancel{A_x} \sqrt{\frac{2 A_x \sigma}{A_x}} = \sqrt{2 \sigma}$$

$$M_C = \frac{I}{P} \sqrt{\frac{I}{2 \sigma A_x}}$$

$$= \frac{2 \sigma^{3/2}}{\phi U} \sqrt{\frac{\phi A_x U^2}{2 \sigma^2}} = \sqrt{2 A_x \sigma^{1/2}}$$

$$h_{out} - h_{in} = \frac{1}{2} (V_4^2 - V_{in}^2)$$



$$M = \frac{40}{\sqrt{R T}}$$

adiabatic $\Rightarrow T_0 = const$
isentropic $\Rightarrow T_0 = const$

$$\sqrt{C_p T_0} = 0.2633$$

$$T = 293 \quad \frac{1}{T_0} = 0.9971$$

$$T_0 = 293.85$$

$$\therefore \frac{\bar{m} (V_3^2 - V_4^2)}{2 V_3 f} = \left(\frac{\sigma + 1}{2 \sigma} \right)$$

Energy: ~~cancel~~

Force/mass

$$F + \cancel{\frac{1}{2} \bar{m} V_3^2} = \bar{m} V_3$$

Mass

$$\rho A_3 V_3 = \rho A_4 V_4 \quad \checkmark$$

Energy

$$\frac{1}{2} \bar{m} V_3^2 = \frac{1}{2} \bar{m} V_{in}^2 + \int_{in}^{out} \rho \int dV \frac{dV}{dx}$$

$$V(x) = V_3 - (V_3 - V_4) \frac{x}{L}$$

$$\frac{\sqrt{C_p T_{in}}}{A_2 P_{in}} = \frac{\sqrt{C_p T_{out}}}{A_1 P_{out}} \cdot \frac{A_1}{A_2} \approx V_3 - (V_3 - V_4) \cdot \frac{x}{L}$$

$$M = 0.06 \quad = 0.316 \\ \frac{1}{T_0} = 0.9993 \quad L \therefore \frac{1}{2} \bar{m} (V_3^2 - V_4^2) = \left[x - (1 - \frac{1}{e}) \frac{x^2}{2L} \right]_0^L$$

$$\sqrt{3} \left(1 - (1 - \frac{1}{e}) \frac{x}{L} \right)$$

$$= L - (1 - \frac{1}{e}) \frac{L}{2}$$

$$= L \left(\frac{1}{2} + \frac{1}{e} \right) = \frac{L}{2e} (e + 1)$$

$$\frac{\dot{m} (V_3^2 - V_4^2)}{2V_3 F} = \frac{\sigma + 1}{2\sigma}$$

$$\frac{\dot{m} V_3}{2F} \left(V_3 - \frac{V_3}{\sigma^2} \right) = \frac{\sigma + 1}{2\sigma}$$

$$\frac{\dot{m} V_3}{2\sigma F} (\sigma - 1) = \frac{\sigma + 1}{2\sigma}$$

$$\frac{\dot{m} V_3}{F} = \frac{\sigma + 1}{\sigma - 1} \quad \therefore \sigma > 1$$

$$F + \dot{m} V_u = \dot{m} V_3$$

$$F + \dot{m} \left(\frac{V_3}{\sigma} - V_3 \right) = 0$$

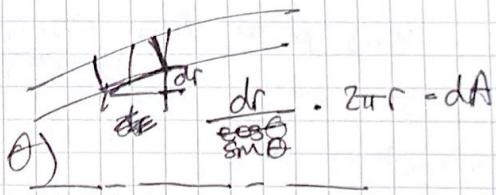
$$F + \frac{\dot{m} V_3}{\sigma} (1 - \sigma) = 0$$

$$F = \frac{\dot{m} V_3}{\sigma} (\sigma - 1)$$

\therefore More diffusion, more resistive force on propulsor from flow deceleration

Now assuming $\rho_4 = \rho_a$

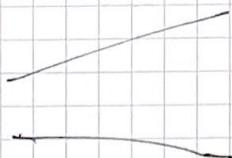
$$\begin{aligned} \text{Total Pressure free} &= \int p dA \\ &= \int_{r_n}^{r_e} p(r) \cdot \frac{2\pi r}{\sin\theta} dr \end{aligned}$$



$$\rho_4 + \frac{1}{2} \rho V_4^2 = \rho_r + \frac{1}{2} \rho V_r^2$$

$$\int A_r V_r = \int A_4 V_4$$

$$V_r = \frac{A_4 V_4}{A_r}$$



$$A_r =$$

QUESTIONS FOR SAM

- Having corrected the σ issue we obtained an expression for thrust which has correct dimensions.

$$\begin{aligned} G^2 &= \frac{\phi^2}{2\phi} \\ P &= \frac{(2\phi)^{3/2} \sigma^{-3} g A c u^3}{2\sigma^{3/2}} \\ &= \frac{(2\phi)^{3/2} g A c u^3}{2\sqrt{\sigma}} \\ &= \frac{12\phi^3 \cdot 12\phi g A c u^3}{\phi} = 2\phi^3 A c \end{aligned}$$

→ Tip velocity limit wrt stress →

→ What geometry & features drive figure of merit.

"Reasons its happening is A B C

Don't want to go there because

→ Efficiency changes across map and we

assumed $\gamma = 0.9$.

→ Flip to Smith chart explain why not
in bottom no.

→ Ideal prop $M_f = 1$

→ Ideal fan $M_f = \sqrt{2}$ FDR $\leftarrow = 1$

We are $\sigma > 1 \therefore M_f$ can be greater.

→ Weight UROP chat towards 6th yr
or UROP Proj.

→ presentation

→ Mass model, outline EXACTLY what the
Set-up looks like.

Build date - Instrument
- cage - Get flying

- Get data.

Replace V_0 with V_m

2. Leave out (\dots)

3. Consider exit swirl

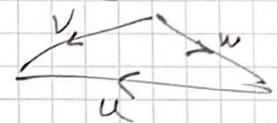
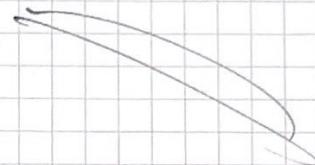
$$V_3 \cos \alpha_3 = V_x$$

May talk later about

how exit deviation

affects thrust re how well

are stators working



Mass Model

\rightarrow Solid cone?

(19) look at how payload changes

$$V_m$$

$$V_m$$

\tilde{T}_E inc on M_f .

$$\sigma^2 = \frac{\phi^2}{2\psi}$$

$$\phi^3 = (\sigma^2 2\psi)^{3/2}$$

$$\frac{\sigma^3}{\sigma^{7/2}} = \sigma^{-\frac{6}{2}-\frac{7}{2}}$$

$$= \sigma^{-\frac{13}{2}}$$

$$= \frac{1}{\sqrt{50}}$$

$$\sigma^{\frac{1}{2}} = \frac{\phi^{\frac{1}{2}}}{(2\psi)^{\frac{1}{2}}}$$

$$P = \frac{c A V_m^3}{2\sigma^2}$$

$$\sigma^2 = \frac{\phi^2}{24}$$

$$P = \frac{\phi^3 g A U^3}{2\sigma^{7/2}} = \frac{\sigma^3 (2\psi)^{3/2} g A U^3}{2\sigma^{7/2}}$$

$$= \frac{(2\psi)^{3/2} g A U^3}{2\sqrt{50}}$$

$$= \frac{\sqrt{2} \sqrt{3/2} \rho A U^3}{2\sqrt{50}}$$

$$= \frac{2\sqrt{2} \rho A U^3}{2\sqrt{50}}$$

$$in = \frac{g A c \sigma^2 V_m^3}{g A c \sigma^2 V_m^3}$$

$$in = \frac{g A c \sigma^2 V_m^3}{g A c \sigma^2 V_m^3}$$

$$V_{xc} = \sqrt{\frac{T_f \sigma}{g A x}}$$

$$\phi = \sqrt{\frac{T_f \sigma}{g A x^2}} \quad \varphi = \frac{T_f}{2 g A x u^2 \sigma}$$

$$T_f = \frac{\phi^2 g A x u^2}{\sigma}. (12)$$

$$\frac{T_f}{g A x^2} = \frac{\phi^2}{\sigma}$$

$$2\varphi \sigma = \frac{T_f}{g A x u^2}$$

$$2\varphi \sigma = \frac{\phi^2}{\sigma}$$

$$\sigma^2 = \frac{\phi^2}{2\varphi}$$

$$M_f = \sqrt{2\sigma}$$

$$P = \frac{g A x V_{xc}^3}{2\sigma^2}$$

$$V_{xc}^3 = \frac{T_f^{3/2} \sigma^{3/2}}{g^{3/2} A x^{3/2}}$$

$$P = \frac{g A x}{2\sigma^2} \cdot T_f^{3/2} \sigma^{3/2}$$

$$= \frac{T_f^{3/2}}{\sqrt{4\sigma^2 g A x}}$$

$$= \frac{\phi^3 g^{3/2} A x^{3/2} u^3}{2\sigma^{3/2} \sigma^{1/2} g^{1/2} A x^{1/2}}$$

$$= \frac{\phi^3 g A x u^3}{2\sigma^2}$$

$$T_f = \frac{\phi^2 g A x u^2}{\sigma}$$

$$0.4 = \frac{\Delta P}{\frac{1}{2} g u^2}$$

$$\frac{1}{2} 0.4 = \frac{\Delta P_0}{g u^2}$$

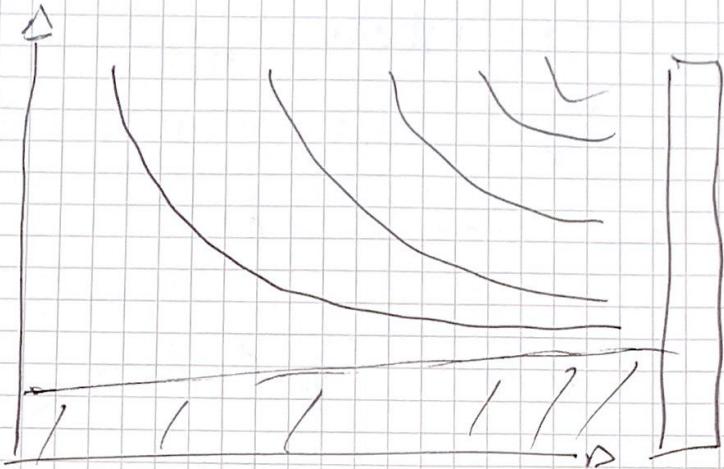
$$\Delta P \frac{u^3}{u^3} = \text{same}$$

$$\phi = \frac{g A x^3 \phi \varphi}{\sigma}$$

Ask about $\phi = \frac{\Delta P_0}{\frac{1}{2} g u^2}$ vs $\frac{\Delta P_0}{g u^2}$
 Plots range 0:0.5
 vs 0:1

→ Have 3 plots each at a different speed.
 How does it effect path load

$$T_f = \phi \sqrt{2\varphi} g A x u^2$$



$$P = \bar{T}_T \sim M$$

$$\bar{T}_T = M$$

$$M_F$$

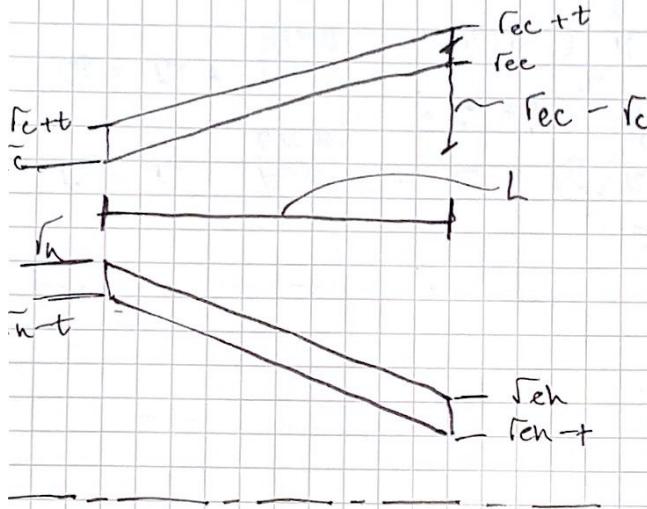
- Make rotor lighter
- Reduce AR a bit.
- Be more generous with fllets.
- Change connector to motor.

- Get flying, get measurements.

Fix jobs!
Apply?

- What does it look like? ✓
- Bar chart of component weights ✓
- Thrust vs RPM at design ✓
- RDM required for Thrust = ~~Weight~~ weight? ✓
- Max payload for theoretical max RPM?
- Thrust vs Power at design ✓
- Power required for Thrust = weight ✓
- Max payload at max power?
- Sensitivity with efficiency.

DIFFUSER VOLUME



$$L = \frac{r_c - r_n}{DF}$$

$$(r_c + t_w)(r_c - t_w)$$

$$r_c^2 - (r_c t_w + t_w r_c) - r_n^2$$

$$r_{ec} = r_c (1 + \sigma) + r_n (1 - \sigma)$$

$$= r_n + \frac{A_c}{4 r_m \pi} \quad A_c = \pi (r_c^2 - r_n^2) \sigma$$

$$= \frac{r_c + r_n}{2} + \frac{2\pi (r_c^2 - r_n^2) \sigma}{24 (r_c + r_n) \pi}$$

$$= \frac{r_c + r_n}{2} + \frac{(r_c + r_n)(r_c - r_n) \sigma}{2 (r_c + r_n)}$$

$$= \frac{r_c + r_n + (r_c - r_n) \sigma}{2}$$

$$= \frac{r_c + r_c \sigma + r_n - r_n \sigma}{2}$$

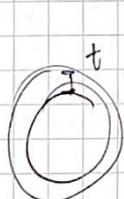
$$= \frac{r_c (1 + \sigma) + r_n (1 - \sigma)}{2}$$

$$r_{ec} - r_e = \frac{r_c (1 + \sigma) + r_n (1 - \sigma)}{2} - r_c$$

$$= \frac{r_c (1 + \sigma) - 2r_c + r_n (1 - \sigma)}{2}$$

$$= \frac{r_c (\sigma - 1) + r_n (1 - \sigma)}{2}$$

$$\therefore L_{diff} = \sqrt{\left(\frac{(r_c - r_n)}{DF}\right)^2 + \left(\frac{r_c (\sigma - 1) + r_n (1 - \sigma)}{2}\right)^2} \times t \times 2\pi$$



$$dA = t \, d\theta \cdot \sqrt{L_{diff}^2 - (r_{ec} - r_c)^2}$$

$$d\theta^2 = dr_c^2 - (r_{ec} - r_c)^2$$

$$\frac{1}{3} \pi r^2$$

$$\frac{L}{r_{ec} - r_c} \cdot r_{ec} \stackrel{r_{ec} - t}{=} h$$

$$r^2 = r_{ec}^2 - (r_{ec} - t)^2$$

$$\therefore A = \frac{L r_{ec}^3 \pi}{3(r_{ec} - r_c)} - \frac{L(r_{ec} - t)^3 \pi}{3(r_{ec} - r_c)} - \left[\frac{L}{(r_{ec} - r_c)} \cdot r_{ec} - h \right] \pi$$

$$h_1^* = \frac{L}{r_{ec} - r_c} \cdot (r_{ec} + t) \quad h_2^* = \frac{L}{r_{ec} - r_c} \cdot (r_{ec})$$

$$\begin{aligned} V &= \frac{h_1 \pi (r_{ec} + t)^2}{3} - \frac{h_2 \pi (r_{ec})^2}{3} - (h_1 - L) \pi (r_c + t) \\ &= \frac{h_1 \pi (r_{ec} + t)^2}{3} - \frac{(h_1 - L) \pi (r_{ec} + t)^2}{3} - \left[\frac{h_2 \pi r_{ec}^2}{3} - \frac{(h_2 - L) \pi r_c^2}{3} \right] \end{aligned}$$

$$h_1 = \frac{L}{r_{ec} - r_c} \cdot (r_{ec} + t)$$

$$h_2 = \frac{L}{r_{ec} - r_c} \cdot (r_{ec})$$

$$V = k_B T (r_{ec} + t)^2$$

$$V_{case} = \frac{\pi}{3} \left[h_1 (r_{ec} + t)^2 - (h_1 - L) (r_c + t)^2 - h_2 r_{ec}^2 + (h_2 - L) r_c^2 \right]$$

$$V_{hub} = \frac{\pi}{3} \left[h_1 r_h^2 - (h_1 - L) (r_{eh})^2 - h_2 (r_h - t)^2 + (h_2 - L) (r_{eh} - t)^2 \right]$$

COVER LETTER

UPDATE CU ←

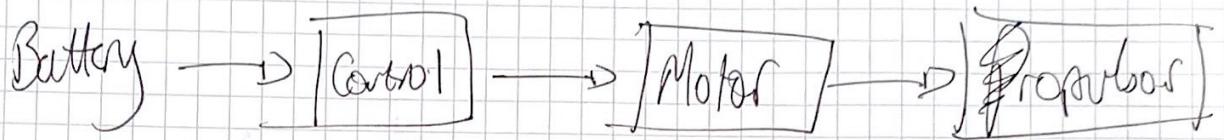
$$n_r V = rpm$$

$$V = 14.8 \text{ V}$$

$$V = \frac{rpm}{KV}$$

$$P = 1481V$$

$$\therefore I = \frac{P}{V} = \frac{PKv}{rpm}$$



$$P_E = V I \xrightarrow{\text{gear ratio}} \frac{\text{gear ratio}}{\text{kv}} \frac{\text{rpm}}{\text{kv}} = \text{inlet} \times \text{dust} \times \text{blades}$$

$$\therefore P = \eta_c \eta_m \eta_p P_E$$

$$P = \eta_T V I = \eta_T \frac{\text{rpm}}{\text{kv}} I = P$$

$$\frac{\text{rpm}}{\text{kv}} I = \eta_T P_E \quad I = \frac{\text{kv} P}{\eta_T \text{rpm}}$$

~~Efficiency~~

$$P_T$$

$$P_T = \eta_T P_E$$

$$= \eta_T \frac{\text{rpm}}{\text{kv}} I$$

$$\therefore I = \frac{P_T \text{kv}}{\eta_T \text{rpm}}$$

- // Prop - RPM power characteristics
- APC props with known calibration
- same prop diameter

Sigma limit due to geometric constraints of diffuser becoming cone.

$$\sigma \Rightarrow r_{th} = 0$$

$$\therefore 0.5 [r_c(1-\sigma) + r_h(1+\sigma)] = 0 \quad M_f = \sqrt{2\sigma}$$

$$r_c(\sigma - 1) = r_h(1 + \sigma)$$

$$\sigma(r_c - r_h) = \frac{r_h + r_c}{r_c - r_h} = \frac{\rho' c}{2\varphi}$$

C_D

$$\therefore \varphi = \frac{\rho' c^2}{2\rho_{max}}$$

→ Update MESH SECTION.

→ Need to relate φ , φ , C_D , r_h , r_c , u

4H Computational Fluid Dynamics

flow-guess

Exit velocity & ϕ known
isentropic flow with unknown
exit & exit area.

$$\rho_{in} = \rho_{out} \quad A_e = \alpha \text{flow (ini)}$$

$$T_{in} = T_{out}$$

$$\rho_e = \rho_{down}$$

$$\left(\frac{\rho_e}{\rho_{out}}\right) = \left(1 + \frac{\gamma-1}{2} M^2\right)^{-\frac{2}{\gamma-1}}$$

$$\therefore M^2 = \frac{2}{\gamma-1} \left(\left(\frac{\rho_{out}}{\rho_e}\right)^{\frac{\gamma-1}{\gamma}} - 1 \right)$$

$$M = \sqrt{\text{ans}}$$

further

$$\frac{T}{T_0} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{-1}$$

$$T = T_0 \left(1 + \frac{\gamma-1}{2} M^2\right)^{-1}$$

Assuming uniform flow

$$m = \rho A U$$

$$\therefore V = M \sqrt{RT}$$

density (ideal gas)

$$\frac{f_k}{f_g} = R T_e \quad \therefore \rho = \frac{R}{R T_e}$$

$$\dot{m} = \frac{R}{R T} \cdot A \cdot M \sqrt{RT_e} = \dot{V} \rho A M \sqrt{\frac{RT}{RT_e}}$$

$$\phi^2 = \frac{T_f \sigma}{g A_{xc} U^2}$$

$$\varphi = \frac{T_f}{2 g A_{xc} U^2 \sigma}$$

$$\text{Equating } \phi^2 \quad \cancel{\phi^2 g A_{xc} U^2} = \cancel{\varphi^2 g A_{xc}}$$

$$\frac{\phi^2 g A_{xc} U^2}{T_f} = \frac{T_f}{2 g A_{xc} U^2 \varphi}$$

$$T_f^2 = 2 \varphi \phi^2 g^2 A_{xc}^2 U^4$$

$$T_f = \sqrt{2 \varphi \phi^2 g^2 A_{xc}^2 U^2}$$

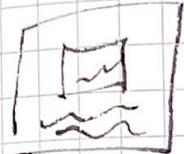
$$T_f = M$$

$$- \sqrt{2 \varphi g_a \pi} (r_e^2 - r_u^2) \cdot \Omega^2 \left(\frac{r_e + r_u}{2}\right)^2$$

$$M = f_p (V_D + V_1 + V_c)$$

$$f_p(r_e, r_u, t, \phi, p).$$

Presentation Chat



base in Physics

w

Done in | Program | Physics | Line Out

Introduce

Explain
what's on
it

Explain
why

Sum-up.



① Analytical model. Slide 11

- Have details on backup slide only

Intro to section

- Model to static threat

- Slides 19/20 became bullet points in an introduction.

② Slide 12 is intro slide to 2nd Q.

- More like culture work.

- Slide 14, more interested in outputs as in poster

Combine 13 + 14

4 - Animation to replace drone PIC with schematic
+ explain where M_F comes from.

Modular

- Made to compare engines

14 - 2 problems

Control ke used

Measured live developed...

(2)

4 slots

in

①

These 11 \rightarrow 4

}

No control over M_F for ideal prop.

With ducted fan, diffuser adds Dof hence $M_F =$

$f_n(5)$.

How do I design it to give good M_F ?

Then produce sleek with ONLY

New fixed ...

To close problem need to match weight to thrust. To do this mass model.

$$\boxed{\frac{F}{T} = \frac{m}{M_F} g}$$

$$\Delta m =$$

$$\boxed{T = M_F \cdot \rho \cdot V}$$

12, 13, 14 & results of payload &

power.